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Aim: Finding the vertical deflection in a cantilever beam subjected to variable load and material properties and visualization of it.

Methodology: Solving governing equation of vertical deflection in swimming pool diving board using Laplace transform.

Laplace Transform : The Laplace Transform of a function f(t), defined for all real numbers t > 0 is the function F(s) defined by,

$$L[f(t)] = \int_0^\infty e^{-st} f(t) dt = F(s) \qquad \dots (1)$$

 $\label{eq:continuous_problem} \textbf{Inverse Laplace Transform of } F(s) \ is \ defined \\ by$

$$f(t) = L^{-1}[F(s)] = \int_0^\infty e^{-st} F(s) dt$$
 (2)

List of MATLAB Commands used:

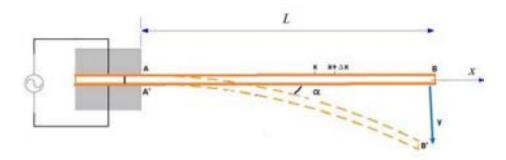
- **1.**] *laplace*(*f*): Returns the Laplace transform of f using the default independent variable t and the default transformation variable s.
- **2.**] laplace(f, transVar): Uses the specified transformation variable transVar
- **3.**] *laplace(f, var, transVar)* : Uses the specified independent variable var and transformation variable transVar instead of t and s respectively
- **4.**] ilaplace(F): Returns the inverse Laplace transform of F using the default independent variable s for the default transformation variable t. If F does not contain s, ilaplace uses symvar.
- **5.**] ilaplace(F, transVar): Uses the specified transformation variable transVar
- **6.**] *ilaplace*(*F*, *var*, *transVar*): Uses the specified independent variable var and transformation variable transVar instead of s and t respectively.
- **7.**] heaviside(t-a): To input the heaviside's unit step function H(t-a).
- **8.**] dirac(t-a): To input the dirac delta function $\delta(t-a)$.
- **9.**] laplace(diff(f(t), t), t, s) : s*laplace(f(t), t, s) f(0)

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Example 1
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Write the MATLAB code which computes the Laplace Transform of
            2 < t < 3
f(t)=\langle t-1,
     7, t > 3.
MATLAB code:
clear all
c1c
syms t
f=input('Enter the function in terms of t:');
F=laplace(f);
F=simplify(F);
OUTPUT:
Enter the function in terms of t:
t^2*(heaviside(t-0)-heaviside(t-2))+(t-1)*(heaviside(t-2)-heaviside(t-3)) +
7*(heaviside(t-3))
Y = 5/(s*exp(3*s))-3/(s*exp(2*s))-3/((s^2)*exp(2*s))-1/((s^2)*exp(3*s))-
2/((s^3*exp(2*s)) + 2/s^3
Example 2:
Solve y'' + 2y' + 10y = 1 + 5(t - 5), y(0) = 1, y'(0) = 2
MATLAB Code:
clc
clear all
syms t s Y
y2=diff(sym('y(t)'),2);
y1=diff(sym('y(t)'),1);
y0=sym('y(t)');
a = input('The Coefficient of D2y = ');
b = input('The Coefficient of Dy = ');
c = input('The Coefficient of y = ');
nh = input('Enter the non homogenous part = ');
eqn=a*y2+b*y1+c*y0-nh;
LTY=laplace(eqn,t,s);
if (a==0)
d = input('The initial value at 0 is ');
LTY=subs(LTY, 'laplace(y(t), t, s)', 'y(0)', Y, d)
else
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```
d = input('The initial value at 0 is ');
e = input('The initial value at 0 is ');
LTY=subs(LTY, 'laplace(y(t), t, s)', 'y(\theta)', 'D(y)(\theta)', Y, d, e)
eq=collect(LTY,Y);
Y=simplify(solve(eq,Y));
y=simplify(ilaplace(Y,s,t));
OUTPUT:
The Coefficient of D2y = 1
The Coefficient of Dy = 2
The Coefficient of y = 10
Enter the non homogenous part = 1 + 5*dirac(t-5)
The initial value at 0 is 1
The initial value at 0 is 2
LTY = 10 * Y - s - 5/exp(5 * s) + 2 * Y * s + Y * s 2 - 1/s - 4 y =
(\cos(3*t) - \sin(3*t)/3)/\exp(t) - (\cos(3*t) + \sin(3*t)/3)/(10*t)
\exp(t)) + (4 * \sin(3 * t))/(3 * \exp(t)) + (5 * heaviside(t - 5) * \exp(5))
- t) * sin(3 * t - 15))/3 + 1/10
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Mathematical Modelling:



- 1.] Stress is proportional to strain. Thus, the equation is valid only for beams that are not stressed beyond the elastic limit.
- 2.] The curvature is always small.
- 3.] Any deflection resulting from the shear deformation of the material or shear stresses is neglected.
- 4.] For the deflected shape of the beam, the slope α at any point is defined as $\tan \alpha = dy/dx$. Assuming $\tan \alpha = \alpha$ we can write $\alpha = dy/dx$.

5.] The curvature of a plane curve at a point can be expressed as

$$\frac{1}{\rho} = \frac{\frac{d^2y}{dx^2}}{\left(1 + \frac{dy}{dx}^2\right)^{\frac{3}{2}}}$$

- 6.] In the elastic curve of beam dy/dx is very small so we can neglect its higher order terms. $\frac{1}{\rho} = \frac{d^2y}{dx^2}$
- 7.] From the theory of elasticity, if x is the distance of the section from the left end of the beam then $1/\rho = M(x)/EI$ where M=Bending moment, E=Modulus of Elasticity & I=Moment of inertia of the cross section.
- 8.] The governing equation for an elastic curve: $\frac{d^2y}{dx^2} = \frac{-M(x)}{EI}$
- 9.] When a beam supports a distributive load w(x) then dM/dx = V(Shear force) and dV/dx = -w. Therefore,

$$\frac{d^4y}{dx^4} = \frac{w(x)}{EI}$$

Vertical deflection in a swimming pool diving board subjected to a distributed load can be seen as the deflection in a cantilever beam of length L subjected to a distributed load w(x) which is the solution of the differential equation:

$$\frac{d^4y}{dx^4} = \frac{w(x)}{EI}$$

subjected to the boundary conditions y(0) = y'(0) = y''(L) = y'''(L) = 0. Note: y''(L) = 0 because there is no bending moment and y'''(L) = 0 because there is no shear at that point.

Problem:

Find the deflection in a cantilever beam subjected to the following conditions y(0) = y'(0) = y''(L) = y'''(L) = 0 by taking L = 3, $E = 2.1 * 10^{11} N/mm^2$, $I = 4.5 * 10^{-11} mm^4$ and w(x) = x.

```
MATLAB Code:
clc
clear all
syms x s C D Y
y4=diff(sym('y(x)'),4);
v0=svm('v(x)');
L=input('Enter the length of the beam:');
E=input('Enter Modulus of elasticity:');
I=input('Enter Moment of inertia of the cross section:');
w=input('Enter distributive load w(x):');
eqn=E*I*y4-w;
LTY=laplace(eqn,x,s);
a=input('Enter y(0):');
b=input('Enter Dy(0):');
c=input('Enter D2y(L):');
d=input('Enter D3y(L):');
LTY=subs(LTY, {'laplace(y(x),x,s)','y(0)','D(y)(0)','D(D((y)))(0)','D(D((y)))(0)','D(D((y)))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D(
(D((y)))(0)',{Y, a, b, C, D});
eq=collect(LTY,Y);
Y=simplify(solve(eq,Y));
y=simplify(ilaplace(Y,s,x));
eq1=subs(diff(y,x,2),x,L);
eq2=subs(diff(y,x,3),x,L);
[C, D]=solve(eq1, eq2);
gen=subs(y);
def=subs(def,heaviside(x-L),0);
fzplot(-def,[0,L])
title('Vertical deflection in cantilever beam')
xlabel('Length of the beam')
vlabel('Deflection')
OUTPUT:
Enter the length of the beam: 3
                                           Length of the beam = 3
Enter Modulus of elasticity: 2.1 * 10^(-11)
                                           Modulus of elasticity = 2.1 * 10^{(-11)}
Enter Moment of inertia of the cross section: 4.5 * 10^{(-11)}
                                           Moment of inertia = 4.5 * 10^{(-11)}
Enter distributive load w(x): x*(heaviside(x-L)-heaviside(x))
                                           Distributive load = x*(heaviside(x-L)-heaviside(x))
Enter y(0):0
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$$y(0) = 0$$

$$Dy(0) = 0$$

Enter D2y(L):0

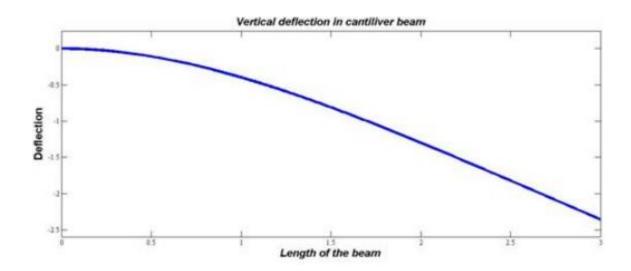
$$D2y(L) = 0$$

Enter D3y(L):0

$$D3y(L) = 0$$

LTY =-
$$(189/20)*D-(189/20)*s*C+(1-(3*s+1)*exp(-3*s))/s^2+ 189/20*s^4*Y$$
 C =-20/21 D =10/21

$$\mathsf{def} \ = (5/63) * x^3 - (10/21) * x^2 - (1/1134) * x^5$$



Exercise:

```
Find the deflection in a cantilever beam subjected to the following conditions y(0) = y'(0) = y''(L) = y'''(L) = 0 by taking L = 2, E = 2.1 * 10^{11} N/mm^2, I = 4.5 * 10^{-11} mm^4 and w(x) = \begin{cases} x, & x < 1, \\ x - L, & 1 < x < L. \end{cases}
```

```
INPUT (MATLAB Code):
clc
clear all
syms x s C D Y
y4=diff(sym('y(x)'),4);
y\theta = sym('y(x)');
L=input('Enter the length of the beam:');
E=input('Enter Modulus of elasticity:');
I=input('Enter Moment of inertia of the cross section:');
w=input('Enter distributive load w(x):');
eqn=E*I*y4-w;
LTY=laplace(eqn,x,s);
a=input('Enter y(0):');
b=input('Enter Dy(0):');
c =input('Enter D2y(L):');
d = input('Enter D3y(L):');
LTY=subs(LTY, {'laplace(y(x),x,s)','y(0)','D(y)(0)','D(D((y)))(0)','D(D((y)))(0)','D(D((y)))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D(
(D((y)))(0)',{Y, a, b, C, D});
eq=collect(LTY,Y);
Y=simplify(solve(eq,Y));
v=simplify(ilaplace(Y,s,x));
eq1=subs(diff(y,x,2),x,L);
eq2=subs(diff(y,x,3),x,L);
[C, D]=solve(eq1, eq2);
gen=subs(y);
def=subs(def,heaviside(x-L),0);
fzplot(-def,[0,L])
title('Vertical deflection in cantilever beam')
xlabel('Length of the beam')
ylabel('Deflection')
```

```
OUTPUT:
Enter the length of the beam: 2
               Length of the beam = 2
Enter Modulus of elasticity: 2.1 * 10^(-11)
               Modulus of elasticity = 2.1 * 10^{(-11)}
Enter Moment of inertia of the cross section: 4.5 * 10^{(-11)}
               Moment of inertia = 4.5 * 10^{(-11)}
Enter distributive load w(x): x*(heaviside(x-L)-heaviside(x))
               Distributive load = x*(heaviside(x-L)-heaviside(x))
Enter y(0):0
               y(0) = 0
Enter Dy(0):0
               Dy(0) = 0
Enter D2y(L):0
               D2y(L) = 0
Enter D3y(L):0
               D3y(L) = 0
LTY =-(189/20)*D-(189/20)*s*C+(1-(3*s+1)*exp(-3*s))/s^2+ 189/20*s^4*Y
C = -20/21
D = 10/21
def = (5/63)*x^3-(10/21)*x^2-(1/1134)*x^5
```

Exercise:

Solve the differential equation

```
y'' + 2y' + 10y = 1 + 5(t - 5), y(0) = 1, y'(0) = 2
MATLAB Code:
clc
clear all
syms x s C D Y
y4=diff(sym('y(x)'),4);
y0=sym('y(x)');
L=input('Enter the length of the beam:');
E=input('Enter Modulus of elasticity:');
I=input('Enter Moment of inertia of the cross section:');
w=input('Enter distributive load w(x):');
eqn=E*I*y4-w;
LTY=laplace(eqn,x,s);
a=input('Enter y(0):');
b=input('Enter Dy(0):');
c =input('Enter D2y(L):');
d = input('Enter D3y(L):');
LTY=subs(LTY, \{'laplace(y(x),x,s)','y(0)','D(y)(0)','D(D((y)))(0)','D(D((y)))(0)','D(D((y)))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D((y))(0)','D(
))(0)'},{Y, a, b, C, D});
eq=collect(LTY,Y);
Y=simplify(solve(eq,Y));
y=simplify(ilaplace(Y,s,x));
eq1=subs(diff(y,x,2),x,L);
eq2=subs(diff(y,x,3),x,L);
[C, D]=solve(eq1, eq2);
gen=subs(y);
def=subs(def,heaviside(x-L),0);
fzplot(-def,[0,L])
title('Vertical deflection in cantilever beam')
xlabel('Length of the beam')
ylabel('Deflection')
Output:
Enter the length of the beam: 2
                 L = 2
Enter Modulus of elasticity: 2.1 * 10^(-11)
                 E = 2.1000e - 11
```

```
I=input('Enter Moment of inertia of the cross section:')
 Enter Moment of inertia of the cross section: 4.5 * 10^(-11)
          I = 4.5000e-11
 w=input('Enter distributive load w(x):')
 Enter distributive load w(x): x*(heaviside(x-L)-heaviside(x))
     w = x*(heaviside(x - 2) - heaviside(x))
 a=input('Enter y(0):')
 Enter y(0):1
   a = 1
 b=input('Enter Dy(0):')
 Enter Dy(0): 2
   b = 2
 y=simplify(ilaplace(Y,s,x))
  y =Y*dirac(x)
 eq1=subs(diff(y,x,2),x,L)
  eq1 =0
 eq2=subs(diff(y,x,3),x,L)
  eq2 =0
 gen=subs(Y)
  gen = Y
            Vertical deflection in cantilever beam
    2
  1.5
Deflection
  0.5
    0
     -5
                                                     5
```

Length of the beam