# **Summary | Dynamics**

## Introduction

A branch of mechanics, which deals with motion of bodies.

### 2 parts:

- Kinematics: the study of geometric aspects of motion (not referencing the forces)
- Kinetics: the analysis of the forces that cause the motion

## Kinematics of a particle

A particle has a mass and negligible size.

## (i) Note

When bodies of finite size is of interest, the body might be considered as particles **provided** motion of the body is characterized by motion of its center of mass and any rotation of the body is neglected.

#### **Rectilinear motion**

When the motion of a particle is along a straight line.

Suppose x is the distance to the particle from a fixed point on its motion path.

- $\dot{x}$  is its instantaneous velocity.
- $\ddot{x}$  is its instantaneous acceleration.

#### Curvilinear motion

When the motion of a particle is along a curve.

Suppose  $\overline{r}$  is the position vector of the particle.

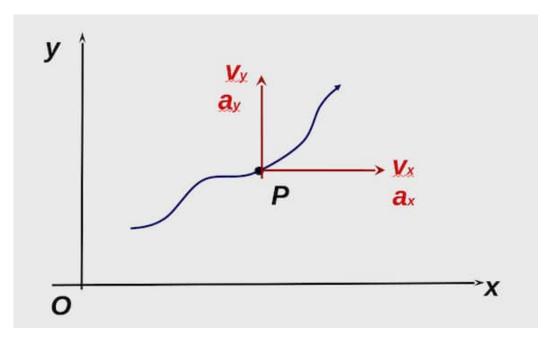
- ullet Instantaneous velocity  $v=rac{\mathrm{d}r}{\mathrm{d}t}$
- ullet Instantaneous speed  $|v|=rac{\mathrm{d}s}{\mathrm{d}t}$
- ullet Instantaneous acceleration  $a=rac{\mathrm{d}v}{\mathrm{d}t}$

## (i) Note

Right hand rule is used here to denote the direction of any rotary motions.

## 2D motion of a particle

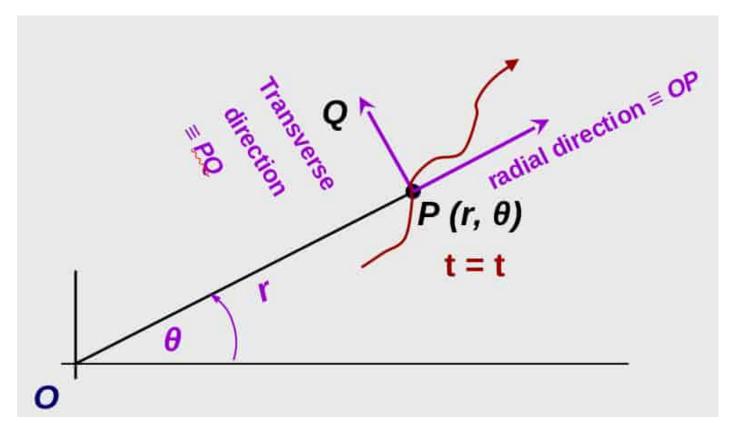
### Rectangular form



$$v_y = rac{\mathrm{d}y}{\mathrm{d}t} = \dot{y} \ \wedge \ v_x = rac{\mathrm{d}x}{\mathrm{d}t} = \dot{x}$$

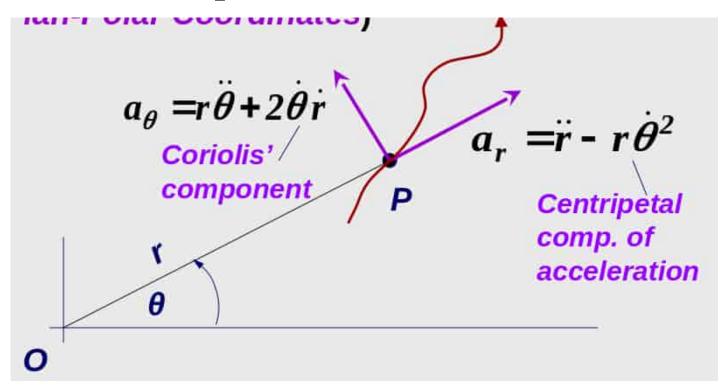
$$a_y = rac{\mathrm{d}^2 y}{\mathrm{d}t^2} = \ddot{y} \ \wedge \ a_x = rac{\mathrm{d}^2 x}{\mathrm{d}t^2} = \ddot{x}$$

#### Polar form



Velocity have a transverse and radial components.

- ullet Transverse component  $v_{ heta} = \dot{ heta} imes oldsymbol{r}$
- ullet Radial component  $v_r=\dot{oldsymbol{r}}$



Acceleration also have a transverse and radial components.

• Transverse component

$$\circ ~~a_{ heta} = r\ddot{ heta} + 2\dot{ heta}\dot{r}$$

$$\circ$$
 In vector equation:  $\underline{a_{ heta}} = \underline{\ddot{ heta}} imes \underline{r} + 2(\underline{\dot{ heta}} imes \underline{\dot{r}})$ 

• Radial component

$$a_r = \ddot{r} - r\dot{ heta}^2$$

$$\circ \ \underline{a_r} = \underline{\ddot{r}} + \underline{\dot{\theta}} \times (\underline{\dot{\theta}} \times \underline{r})$$

In the acceleration:

- ullet Coriolis' component of acceleration:  $2\dot{ heta}\dot{r}$
- ullet Centripetal component of acceleration:  $-r\dot{ heta}^2=\dot{ heta} imes(\dot{ heta} imes r)$

### **Effects of Coriolis' component**

- Objects reflect to the right in the northern hemisphere
- Objects reflect to the left in the southern hemisphere
- Maximum deflections occur at the poles. No deflection at the equator.

#### **Unit vectors**

Unit vectors in transverse and radial directions are denoted by  $e_{ heta}$  and  $e_{r}$  respectively.

$$\dot{e}_r = \dot{ heta}e_ heta \ \wedge \ \dot{e}_ heta = -\dot{ heta}e_r$$

Velocity

$$v=rac{\mathrm{d}}{\mathrm{d}t}(re_r)=\dot{r}e_r+r\dot{ heta}_r=\dot{r}e_r+r\dot{ heta}e_{ heta}$$

Acceleration

$$a=rac{\mathrm{d}}{\mathrm{d}t}(r\dot{ heta}e_{ heta})=(\ddot{r}-r\dot{ heta}^2)e_r+(r\ddot{ heta}+2\dot{ heta}\dot{r})e_{ heta}$$

# 2D kinematics of a rigid body

### Rigid body

A solid body that doesn't deform.

## Degrees of freedom

In the motion of a rigid body in 2D kinematics, there are 3 degrees of freedom.

- ullet Movement along  $oldsymbol{x}$  direction
- ullet Movement along y direction
- ullet Rotation about z direction

In 3D, there are  $\bf 6$  degrees of freedom: movement and rotation along each direction.

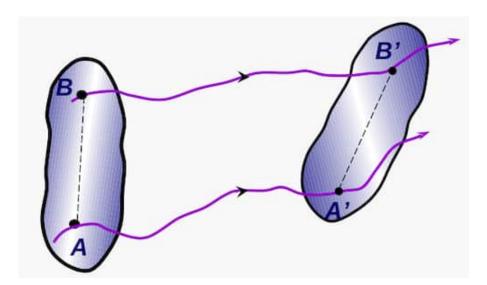
### **Translation**

Movement that changes the position of an object. Translation can be done through a rectilinear or curvilinear path. Axes of the body always stays parallel.

#### **Rotation**

Circular movement of an object about a fixed axis that is perpendicular to the plane.

### General 2D motion



Mixture of translation and rotation.

$$v_{
m B} = v_{
m A} + v_{
m B/A} = v_{
m A} + \dot{ extstyle heta} imes r_{
m B/A}$$

$$a_{
m B} = a_{
m A} + a_{
m B/A} = a_{
m A} + \ddot{ heta} imes r_{
m B/A} + \dot{ heta} imes (\dot{ heta} imes r_{
m B/A})$$

Here:

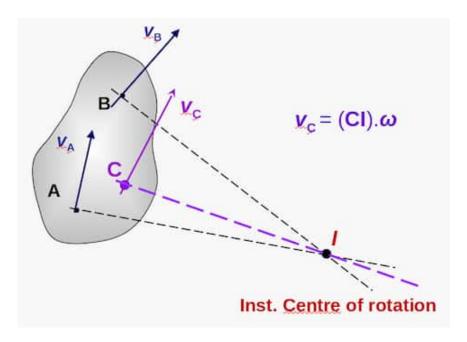
- $\dot{ heta}$  Angular velocity of  ${f B}$  relative to  ${f A}$
- ullet  $v_{
  m B/A}$  Velocity of  ${
  m B}$  relative to  ${
  m A}$
- ullet  $a_{\mathrm{B/A}}$  Acceleration of B relative to A
- $r_{
  m B/A}$  Position vector of m B relative to m A . It's constant.

In general motion, each particle of the body has a different velocity at every instance.

#### Instantaneous centre of rotation

The point that has 0 velocity at a particular instant of time. This point might be changing throughout the motion. Denoted by I.

It can be imagined that the object is momentarily having a pure rotation about this centre I.



I can be found by drawing a line perpendicular at the velocity vectors at 2 different points and finding their intersection point.

#### Centrode

The locus of instantaneous centres during the motion.

## **Mechanisms**

#### Mechanism

An assembly of rigid bodies or links designed to obtain a desired motion from an available motion while transmitting appropriate forces and moments. Motion of the links have definite relative motion with other links.

#### Simple mechanisms

- Lever
- Pulley
- Gear trains
- Belt and chain drive
- Four bar linkage

### Other complex mechanisms

- Lock stitch mechanism (used in sewing machine)
- Geneva mechanism
  - Constant rotational motion to intermittent rotational motion. mostly used in watches.
- Scotch yoke mechanism
  - Constant rotational motion to linear motion (vice versa.). Mainly used as valve actuators in high pressure gas pipelines.
- Slider crank mechanism
   Used in internal combustion engines

### 2D link mechanisms

#### Method of instantaneous centre of rotation

- Find the instantaneous centre of the rotation from known velocities at known points
- Use the instantaneous centre to find velocities at other points

#### Kinematic chain

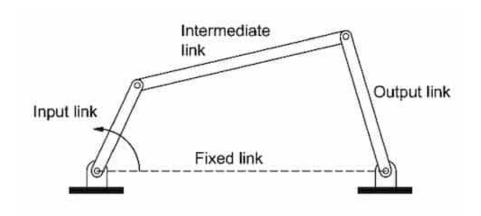
An arbitrary collection of links (forming a closed link) that is capable of relative motion and that can be made into a rigid structure by an additional single link.

## Four-bar Mechanism

Four bar-shaped members connected to each other in one plane.

## Usually:

- 1 fixed link + 3 moving links
- 4 pin joints
- 2 moving pivots + 2 fixed pivots
- 4 turning pairs



- input link usually denoted in the left.
- output link usually denoted in the right.
- coupler intermediate link
- frame fixed link

### Grashof's law

A four bar mechanism has at least one revolving link if  $l_0+l_3 \leq l_1+l_2$ .

Here:  $l_0, l_1, l_2, l_3$  are the length of four bars from shortest to longest.

## Modes of motions

Mechanism	Shortest link	Criteria
Crank rocker	Input link	s+l < p+q
Double crank	Fixed link	s+l < p+q
Double rocker	Coupler link	s+l < p+q
Change point	Any	s+l=p+q
Triple rocker	Any	s+l>p+q

**crank** means a link that makes a full revolution. **rocker** means a link that doesn't make a full revolution.

#### Crank rocker mechanism

Shortest link rotates a full revolution. Output link oscillates.

#### Double crank mechanism

Shortest link is fixed. Both input and output links rotates a full revolution.

### Double rocker mechanism

Shortest link make full resolution. Input and output links makes a full revolution.

# **Special cases**

$$l_0 + l_3 = l_1 + l_2$$
.

Mechanism	Orientation
Parallelogram linkage or anti- parallelogram linkage	Equal links are opposite to each other
Deltoid linkage	Equal links are adjacent to each other

### Parallelogram linkage

Double crank mechanism. Opposite links are equal and parallel. Angular velocity of input crank & output crank is same. Orientation of the coupler doesn't change during the motion.

### Anti-parallelogram linkage

Double crank mechanism. Angular velocity of input crank is different to output crank.

## **Deltoid linkage**

- Longest link is fixed: crank rocker mechanism
- Shortest link is fixed: double crank mechanism

### Non-Grashof's condition

A four bar mechanism with the property if  $l_0+l_3>l_1+l_2$ .

Here:  $l_0, l_1, l_2, l_3$  are the length of four bars from shortest to longest.

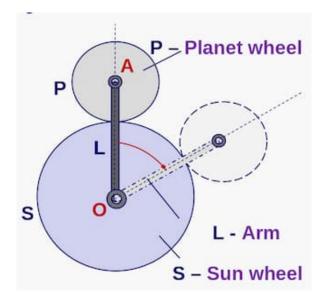
Three links are in oscillation.

# **Epicyclic Gears**

In below equations:

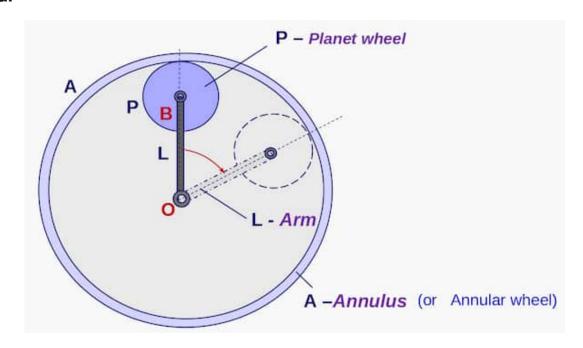
ullet  $\omega_p$  - Absolute angular speed of planet wheel  $\,P\,$ 

## **External**



$$\omega_p = \Big(1 + rac{r_S}{r_P}\Big)\omega_L - \Big(rac{r_S}{r_P}\Big)\omega_S$$

## Internal



$$\omega_p = \Big(1 - rac{r_A}{r_P}\Big)\omega_L + \Big(rac{r_A}{r_P}\Big)\omega_A$$

## **Mobility of Mechanisms**

### Independent object

Has 3 degrees of freedom.

#### **Lower Pair**

A pair of kinematic elements which share a surface of contact.

When a rigid body is constrained by a lower pair, which allows only rotational or sliding movement. It has  ${\bf 1}$  degree of freedom, and the  ${\bf 2}$  degrees of freedom are lost.

Some examples:

- Turning pair
- Sliding pair
- Helical thread

### **Higher Pair**

A pair of kinematic elements which share only a line or a point of contact.

When a rigid body is constrained by a higher pair, it has 2 degrees of freedom: translating along the curved surface and turning about the instantaneous contact point.

Gear is an example.

When 2 independent objects are brought together to create a link, some degree of freedom will be lost.

"You lose some freedom when you become a couple." — Our Dynamics Lecturer

## **Grubler's Equation**

Suppose N kinematic elements are brought together. 1 of them is fixed. The remaining elements have 3(N-1) degrees of freedom. Each lower pairs loses 2 degrees of freedom. Each higher pairs loses 1 degree of freedom. For a workable mechanism, resultant degrees of freedom must be 1.

$$F=3(N-1)-2L-H=1 \implies 3N-2L+H=4$$

#### Here:

- ullet F degree of freedoms
- ullet N number of kinematic elements
- ullet L number of lower pairs
- ullet H number of higher pairs

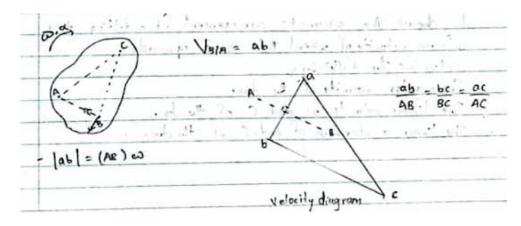
# **Velocity & Acceleration Diagram**

O is a fixed point.

## Velocity diagram

#### **Notation**

- ullet oa Absolute velocity of point  ${f A}$
- ab Velocity of point  ${\bf B}$  relative to point  ${\bf A}$



The above illustration is from Ruththiragayan, one of my friends.

## **Acceleration diagram**

#### **Notation**

- ullet  $o_1a_1$  Absolute acceleration of point  ${f A}$
- ullet  $a_1b_1$  Velocity of point  ${f B}$  relative to point  ${f A}$

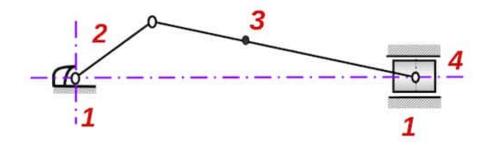
$$a_1b_1=a_1x_1+x_1b_1= ext{(AB)}\,\omega_{AB}^2+ ext{(AB)}\,lpha_{AB}$$

#### Here:

- ullet  $a_1x_1$  the radial component of the relative acceleration between A and B
- ullet  $x_1b_1$  the transverse component of the relative acceleration between A and B

## Inversions of a mechanism

The inversions are obtained by making different kinematic element stationary (one at a time) while keeping the same set of kinematic pairs.



For example, in slider crank mechanism:

- When link 2 is fixed: Whitworth quick-return mechanism
- When link 3 is fixed: The oscillating cylinder engine
- When link 4 is fixed: Hand pump

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