# Summary | Hydraulic Machinary

## Introduction |

| Positive Displacement | Piston pump, Rotary pump | Motors | Hydraulic Ram, Jack Press | | Rotodynamic | Pumps, Compressors | Turbines | Hydraulic coupling, Torque converter |



In s1, only rotodynamic <u>pumps</u> and rotodynamic turbines are studied.

## **Pumps**

Energy flow direction is machine to fluid.

#### Vane

A curved blade used in a pump.

### **Impeller**

Set of vanes attached to a disc or a cyllinder. Main rotating element in a pump.

In a pump, impeller is mounted on a shaft. The shaft is driven by an electric motor or IC engine.

### Direction of the fluid flow

#### **Axial flow**

Fluid enters and exits the impeller axially.

#### **Radial flow**

Fluid enters the impeller axially. Leaves radially. Aka. centrifugal pumps.

### Mixed flow

Fluid enters the impeller axially. Leaves in both axial and radial directions.

(i) Note

For s1, only centrifugal pumps are studied.

### **Parameters**

## Head provided

The head provided by a pump depends on the flow rate.

$$H = f(Q)$$

Here:

- ullet H provided head
- ullet Q flow rate

For a given pump running at a given speed, there is a unique variation of  $m{H}$  and  $m{Q}$ .

## **Power input**

Denoted by  $P_i$ . Varies with Q.

## **Efficiency**

Denoted by  $\mu$ . Varies with Q.

$$\mu = rac{P_o}{P_i}$$

(i) Note

$$ext{Energy per unit volume} = rac{P_{i_A}}{Q}$$

All these parameters, plotted vs Q, is known as **performance characteristic** of the pump. Will be given by the manufacturer. Can be found by laboratory testing.

## In a pipeline system

$$H = H_0 + KQ^2$$

 $m{H}$  is the head required (or received) to create the flow rate  $m{Q}$  in the pipeline system. The above equation is known as **system characteristic** or **system load curve**.

Here K is the loss coefficient and is given by:

$$K=rac{8}{\pi^2gD^4}igg(K_L+rac{\lambda L}{D}igg)$$

(i) Note

Working state of a pipeline system is given by the intersection of system characteristic and performance characteristic (of the pump) curves.

## **Resultant pumps**

#### In serial

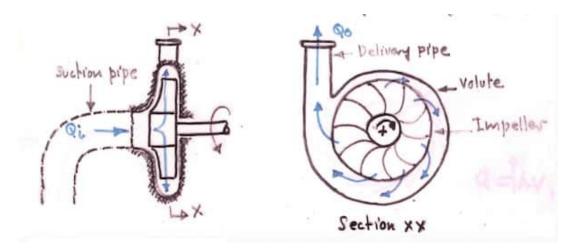
When 2 pumps are operating in a series, their head inputs are added.

### In parallel

When 2 pumps are operating in a parallel, their flow rates are added.

# **Centrifugal Pumps**

Most used pumps in engineering because they support wide range of heights and flow rates. Mixed flow, rotodynamic pump.



There can be a diffuser as well, which is optional.

#### Volute

Casing of the impeller. A passage with increasing area, to reduce velocity (to reduce losses).

i Note

Energy losses in a fluid flow is directly proportional to  $v^2$ .

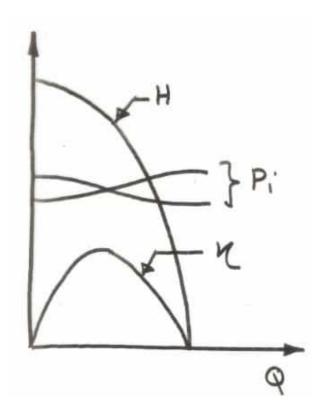
#### **Diffuser**

A fixed set of vanes added to the impeller. To direct the flow into the volute, to minimize impact losses.

## **Operation**

- Volute must be filled with fluid to start pumping
- Fluid enters through the eye of the impeller
- ullet v and P are increased when the fluid flows through the impeller

## Performance characteristic



## **Turbines**

Used to generate electricity. Direction of energy transfer is fluid to machine.

Rotating element is called as the runner.

## **Types of turbines**

### **Reaction turbines**

Aka. pressure turbines. Similar to pumps. Operating in reverse direction (direction of fluid flow and energy transfer). Guide vanes are placed to guide fluid flow onto the runner.

3 types of reaction turbines based on the direction of fluid flow.

### **Radial flow**

Aka. Francis turbine. Commonly used to get a head output of 30 to 500m.

#### **Axial flow**

Aka. Kaplan turbine. Commonly used to get a head output of 3 to 70m.

#### Mixed flow

A combination of radial flow and axial flow.

### Impulse turbines

Aka. velocity turbines. Used for high heads. Highly efficient. Includes a runner (a wheel with buckets attached) mounted on a shaft. High velocity jet is focused on the buckets.

Efficiency of an impulse turbine is given by:

$$\mu=rac{1}{v_1^2}(2u)(v_1-u)(1+k\coseta)$$

Here:

- ullet  $v_1$  velocity of the jet of fluid
- ullet u velocity of the bucket
- k loss coefficient (a little less than 1)
- $oldsymbol{ heta}$  angle of deflection of fluid inside the bucket

 $\mu$  can be considered as a function of u. And from that, the turbine works at maximum efficiency when  $2u=v_1$ .

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