Simulation of an Ideal Internal Combustion Engine Using MATLAB

Aditya Khandelwal

Indian Institute of Technology Kanpur

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Outline

- Introduction to IC Engine
- Simulation Results
- Enhancements and Comparisons
- Conclusion

What is an Internal Combustion Engine?

- An Internal Combustion (IC) Engine converts chemical energy in fuel into mechanical energy through combustion within a cylinder.
- It powers various applications, including automobiles, aircraft, and industrial machines.
- Two main types: Spark Ignition (SI) Engines and Compression Ignition (CI) Engines.

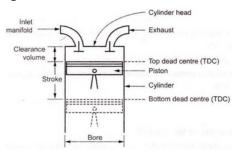
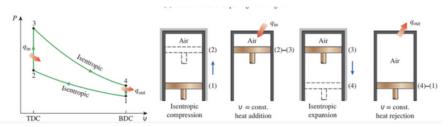


Figure: Ideal Otto Cycle parameters

Stages of an Otto Cycle

- Intake Stroke: Air-fuel mixture is drawn into the cylinder.
- Compression Stroke: Mixture is compressed, increasing temperature and pressure.
- Power Stroke: Combustion pushes the piston downward, generating work.
- Exhaust Stroke: Burnt gases are expelled from the cylinder.



Ideal cycle in spark ignition engine (Otto cycle) and it's P-v diagram.

Initial Conditions for Simulation

- User Input Parameters:
 - Initial Temperature, Initial Pressure, and Maximum Temperature.
 - Engine geometry: Stroke, Bore, Compression Ratio.
- Derived Parameters:
 - Swept Volume, Clearance Volume, and Mass of Air.
 - Specific heat capacities and specific heat ratio as functions of temp erature.

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Provide the following initial conditions to simulate an ideal otto cycle: 
Enter Initial Temperature (in K): 380 
Enter Initial Pressure (in Pa): 180800 
Enter Maximum Temperature (in K):1500 
Enter connecting ond length (in m):0.14 
Enter stroke (in m): 0.1 
Enter stroke (in m): 0.09 
Enter connecting (in m): 0.09 
Enter compression ratio: 8.5
```

Figure: Parameters taken as input from the user

PV Diagram of an Ideal Otto Cycle

- Process 1-2 (Compression):
 Isentropic compression with increasing pressure and decreasing volume.
- Process 2-3 (Heat Addition):

Pressure rises sharply at constant volume.

- Process 3-4 (Expansion): Isentropic expansion with decreasing pressure and increasing volume.
 - Process 4-1 (Heat
 Rejection): Pressure drops
 sharply at constant volume.

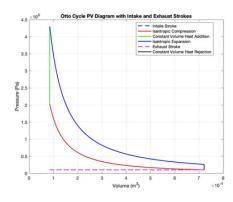


Figure: PV diagram illustrating the processes of an ideal Otto cycle.

Log(PV) Curve

- Linear Representation:
 The log(PV) plot highlights isentropic processes as linear.
- Constant Volume Processes: Heat addition and rejection appear as vertical transitions.

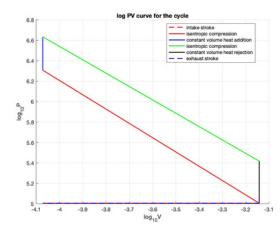


Figure: Log(PV) curve showing linear transitions of processes.

Compression Ratio vs Efficiency

- Trend: Efficiency increases with higher compression ratios.
- Diminishing Returns: Gains reduce at high compression ratios due to practical constraints.
- Real-world Limits:
 Knocking and material limitations restrict maximum compression ratios.

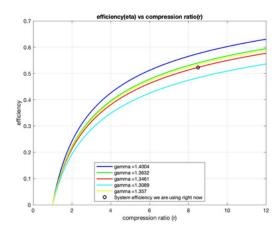


Figure: log(PV) curve showing linear process transitions.

Pressure vs Crank Angle

- Pressure Rise: Compression stroke shows a steep pressure increase.
- Expansion: Pressure decreases sharply during the power stroke.
- Heat Transfer Effects: Heat addition and rejection cause rapid pressure changes.

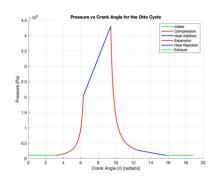


Figure: Pressure variation with crank angle.

Modifications to the Model

- Implemented variable specific heats for realistic gas behavior.
- Simulated piston-cylinder dynamics using crank angle calculations.
- Added dual input conditions for comparative performance analysis.

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Results for the simulation Theoretical Efficiency of the system: 0.523198 | Theoretical Efficiency of the system: 0.523198 | State variables [P1, V1, T1]: 1180.000000 kPs, 0.000721 m^3, 300.000000 kT |
State variables [P2, V2, T2]: [2002.35735 kPs, 0.000005 m^3, 706.73345 kT |
State variables [P3, V3, T3]: [4250.000000 kPs, 0.000005 m^3, 1500.000000 kT |
State variables [P4, V4, T4]: [258.155867 kPs, 0.000721 m^3, 774.467602 kT |
Heat (input) of cycle: 16.507200 kT |
Heat (input) of cycle: 3.663915 kT |
Mork done of the cycle: 7.863375 kT |
Mork done of the cycle: 7.863375 kT |
```

Figure: Results obtained from the simulation

Comparison: Adiabatic vs Non-Adiabatic Processes

- Compression: Adiabatic processes show steeper pressure-volume curves.
- Expansion: Non-adiabatic processes exhibit smoother curves due to heat losses.
- Efficiency: Adiabatic assumptions predict higher efficiencies compared to real-world constraints.

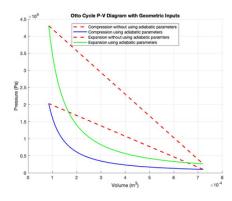


Figure: Adiabatic vs non-adiabatic PV curves.

Conclusion

Key Takeaways:

- Summarized the behavior of an ideal Otto cycle and its thermodynamic principles.
- Highlighted performance characteristics of internal combustion engines.

Proposed Enhancements:

- Further refinements will include real gas behavior, heat losses, and frictional effects.
- Optimizing the model for more practical simulations will improve accuracy.

Thank you! Questions?