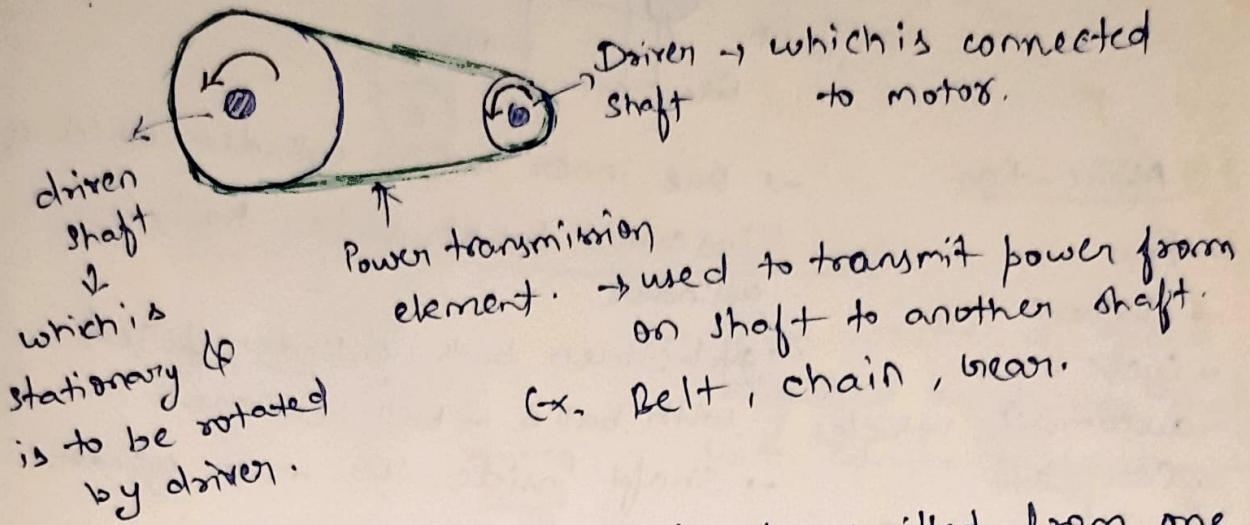


## Power transmission system



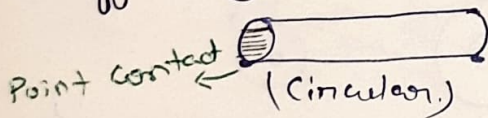
System in which power is transmitted from one driver shaft to driven shaft.

### Different type of belt

(i) • Circular belt  
(rope belt) (pulley)

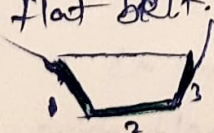
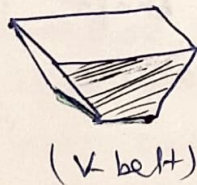
$\rightarrow$  It has point contact  
 $\rightarrow$  So friction is less.  
Slip is more.

$\rightarrow$  less power transmitted.  
efficiency less.

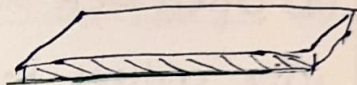


(iii) • V-belt

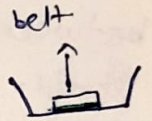
3 line contact  
friction is more.  
Slip  $\downarrow$   
Power transmission is more.  
efficiency  $\uparrow$  > flat belt.



(ii) • Flat belt

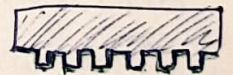


$\rightarrow$  a line contact  
 $\rightarrow$  friction is more  
 $\rightarrow$  Slip is less.  
 $\rightarrow$  more power transmitted.  
efficiency is more than Circular.



(iii) • Ribbed belt

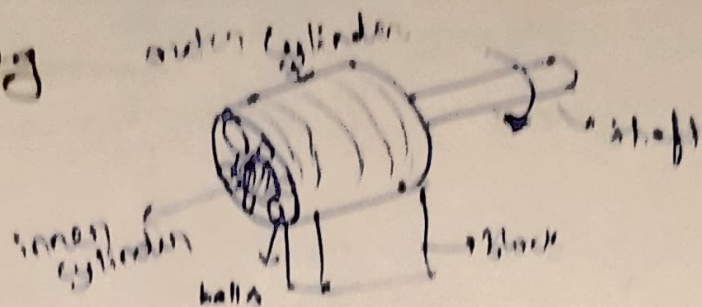
• Timing belt



$\rightarrow$  Internal surface is constructed by slots  
 $\rightarrow$  Surface area contact is max.  
 $\rightarrow$  friction max, Slip min.  
 $\rightarrow$  Power transmission max.  
efficiency max.



## \* Ball Bearing



(9)

### Application

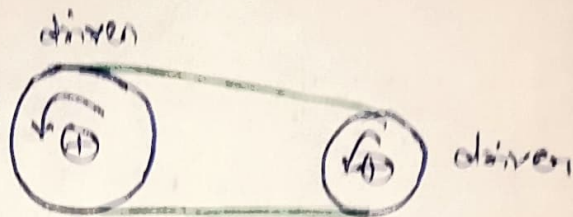
- Fan
- Mixture
- Cycle
- automobile vehicles
- Cars etc

- One inner hollow cylinder which is inserted inside outer hollow cylinder
- gap between both cylinder filled with balls → ball bearings.
- Shaft inside the inner cylinder.

## \* Different type of belt Drive

### (1) Open belt drive -

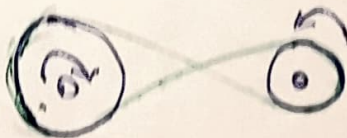
- One endless belt is mounted between driven & driven shaft.



- Power transmission in same direction, in which driven shaft is rotating.
- Used when axis of both shafts are parallel.

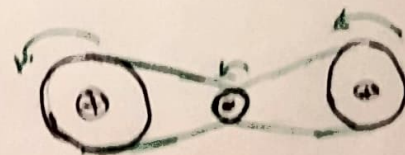
### (2) Crossed belt drive -

- Similar to open belt drive.
- power transmission in opposite direction in which driver shaft is rotating.
- Used when axis are parallel.

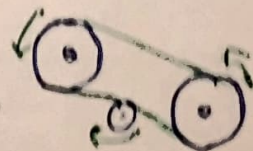


### (3) Intermediate belt drive

- Used when 2 shafts are placed at large distance.
- One more pulley is placed between 2 shaft.



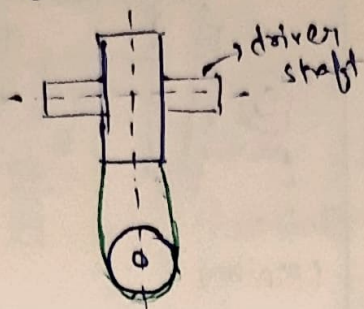
- Used for Id is also known as Idler Pulley belt drive (Jockey)





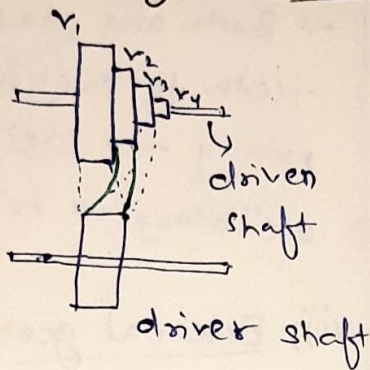
#### ④ Quarterly (Right angled) belt drive -

(3)



two shafts are arranged at right angle.

#### (5) Cone Pulley / Step Pulley belt drive -



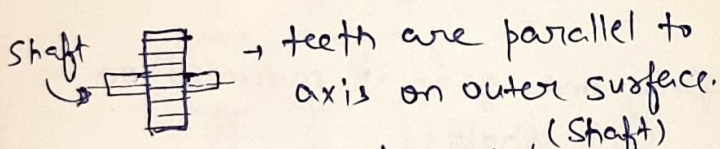
→ When varieties of speed is required on driven shaft. with constant speed motor (driver)

→ It has more than one pulley (Step)

→ Shape  $\Rightarrow$  Cone like

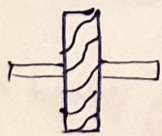
#### • Gears :- Power transmission element.

##### Types (i) Spur Gear



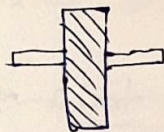
→ Used in wrist watch.  
→ vernier caliper (Precision measuring instrum.)  
→ for low power transmission.

##### (ii) Spiral Gear



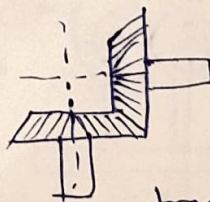
→ teeth are cut in spiral form on outer surface.  
→ when axis are not parallel but intersecting each other.

##### (iii) Helical Gear



teeth are cut in helical form.  
→ Used for high speed.  
→ Smooth operation.

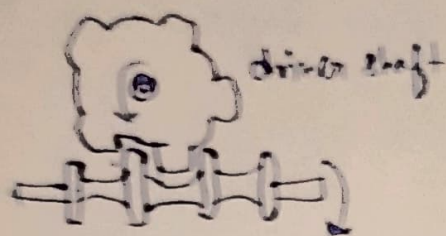
##### (iv) Bevel gear



It has two cone gear.  
It is used to transmit power at an angle  $90^\circ$ .  
It is used in auto mobile.



## (vi) Worm gear



driven shaft.

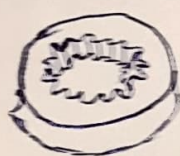
→ It has a large wheel on which teeth are constructed.

→ below that wheel, screw is placed.

→ when wheel will rotate, screw will also rotate.

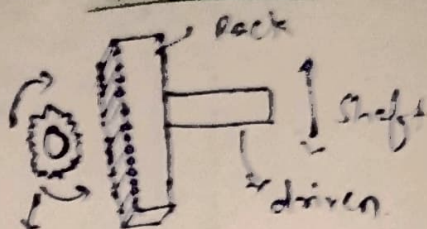
at  $90^\circ$  or  $180^\circ$

## (vii) Internal gear



Teeth are inside of gear ring.

## (vi) Rack gear



Driver (pinion)

→ It has one small gear and one slab with teeth.

→ Both are meshed with teeth to each other and rotary → oscillatory  
oscillatory → rotary motion.

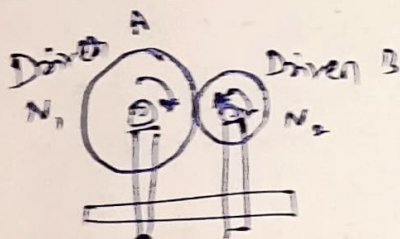
## (viii) External gear



teeth are outside of gear ring.

## \* Gear train

(i) Simple gear train → Only one gear is mounted on each shaft.



$$\text{Train value} = \frac{N_2}{N_1} = \frac{\text{Driven speed}}{\text{Driver speed}}$$

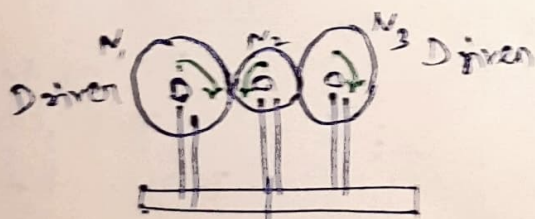
Note 
$$\frac{D_A}{D_B} = \frac{N_2}{N_1}$$

(ii) No of (Intermediate gear) is even

then direction of speed of driver & driven are opposite.

(iii) No of (Intermediate gear) is odd.

then direction of speed is same.



Intermediate gear.

Used when to cover the gap between driver &

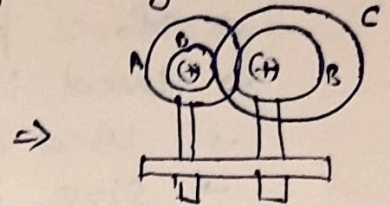
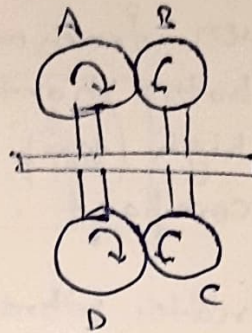
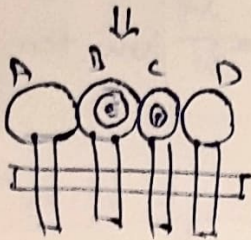
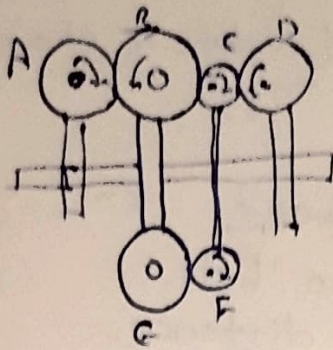
driven. as desired direction of velocity.



### (ii) Compound gear train:-

(5)

When two or more gear are on one shaft & rpm of both shaft is same, then this type of arrangement is known as compound gear train.



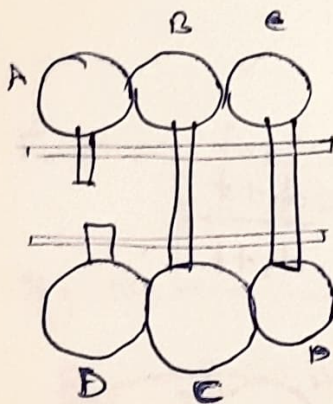
$$r_A + r_B = r_D + r_C$$

Speed ratio

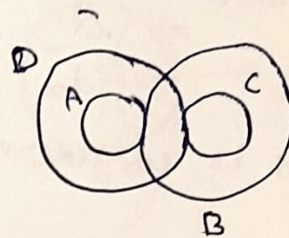
$$\frac{N_A}{N_B} = \frac{D_B}{D_A} \quad \& \quad \frac{N_C}{N_D} = \frac{D_D}{D_C}$$

### (iii) Reverted gear train:-

When driver & driven are mounted on ~~same~~ shaft, co-axial shafts.



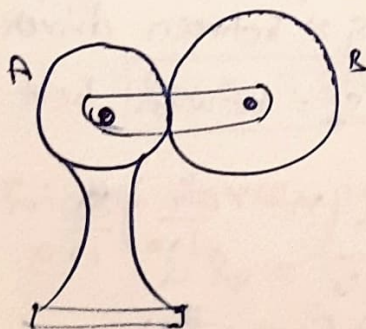
=>



$$r_A + r_B = r_C + r_D$$

### (iv) Epicyclic gear train:-

When one gear rotates in cyclic motion about another gear.



→ The axes of shaft, over which the gear move, mounted, may move relative to a fixed axis.



## \* Chain Drive

(6)

→ In belt drive some slip is present, but in chain drive, it is eliminated.

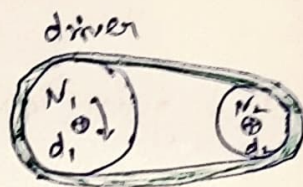
Advantage ⇒ No Slip, Accurate speed.

- More power transmission than belt.
- Used in both short & long distance.
- Used at high temp as well as low temp.
- Size is compact.

Disadvantage ⇒ → Creates vibration

- Production cost is high.
- Proper alignment is required.
- Required maintenance & lubrication.

\* Velocity Ratio =  $\frac{N_{\text{driver}}}{N_{\text{follower}}}$

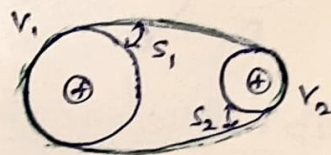


$$V_{\text{driver}} = \pi d_1 N_1$$

$$V_{\text{follower}} = \pi d_2 N_2 \Rightarrow \frac{d_1}{d_2} = \frac{N_2}{N_1}$$

if thickness is considered ⇒  $\boxed{\frac{N_2}{N_1} = \frac{d_1 + t}{d_2 + t}}$

\* Slip = some forward motion of the driver without carrying the belt with it (relative motion)



$$(\text{Speed})_{\text{belt}} = v_1 - v_1 \times \frac{s_1}{100}$$

$s_1$  ⇒ between driver & belt  
 $s_2$  ⇒ between belt & driven

driven speed

$$v_2 = \left( v_1 - v_1 \frac{s_1}{100} \right) - \left( v_1 - v_1 \frac{s_1}{100} \right) \frac{s_2}{100}$$

$$N_2 = \frac{v_1 (100 - s_1) (100 - s_2)}{10000}$$



$$\therefore V = \frac{\pi d N}{60}$$

$$\Rightarrow \frac{\pi d_2 N_2}{60} = \frac{\pi d_1 N_1}{60} \times \frac{(100 - s_1)(100 - s_2)}{10000}$$

$$\Rightarrow \frac{N_2}{N_1} = \frac{d_1}{d_2} \left[ 1 - \left( \frac{s_1 + s_2}{100} \right) \right] \quad \left( \because s_1 + s_2 \approx 8 \text{ total slip} \right)$$

$s_1, s_2 \approx 0$

$$\boxed{\frac{N_2}{N_1} = \frac{d_1}{d_2} \left( 1 - \frac{s}{100} \right)}$$

\* Creep of Belt :- When the belt passes from slack side to tight side, a certain portion of belt extends. (also when it's tight to slack side)  
Due to length change, relative motion occurs known as creep.

effective turning force =  $T_1 - T_2$

$$\text{work done/sec} = P = (T_1 - T_2)v$$

Friction force

$$F = \mu R_N$$

By NLM,

$$R_N = T \sin \frac{\delta\theta}{2} + (T + \delta T) \sin \frac{\delta\theta}{2} \quad (\text{horizontal})$$

$$R_N = T \frac{\delta\theta}{2} + \cancel{\delta T \frac{\delta\theta}{2}} + \frac{T \delta\theta}{2} \quad (\delta\theta \approx \text{small})$$

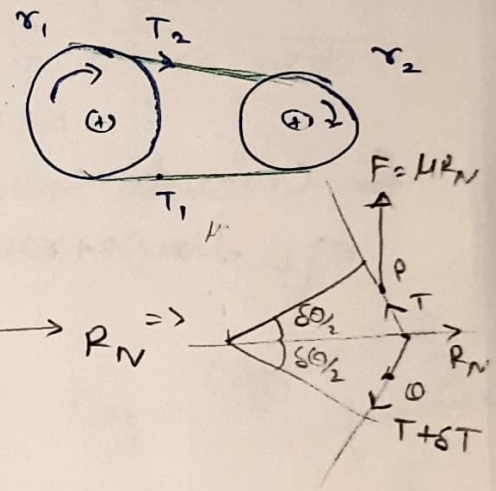
$$\boxed{R_N = T \delta\theta}$$

Friction force in vertical direction,

$$\mu R_N = (T + \delta T) \cos \frac{\delta\theta}{2} - T \cos \frac{\delta\theta}{2}$$

$$\mu R_N = \delta T$$

$$\boxed{R_N = \delta T / \mu}$$



$$\frac{\delta I}{T} = \mu \theta \quad (T_1 \text{ to } T_2)$$

$$\Rightarrow \left[ \frac{T_1}{T_2} = e^{\mu \theta} \right] \quad (\text{Tension Ratio})$$

### \* Length of Belt

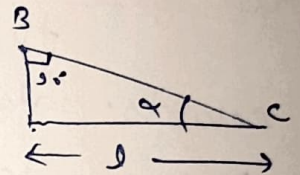
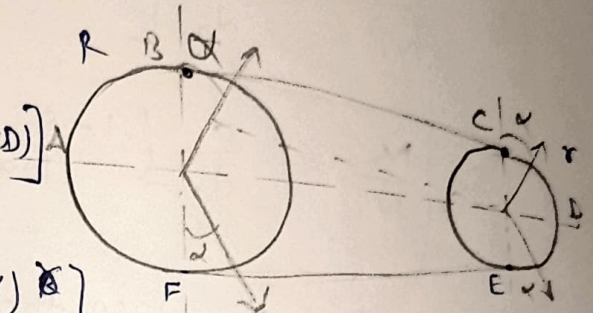
(Open drive)

$$\text{Length} = 2 [\text{Arc}(AB) + BC + \text{Arc}(CD)]$$

$$= 2 \left[ \left( \frac{\pi}{2} + \alpha \right) R + BC + \left( \frac{\pi}{2} - \alpha \right) r \right]$$

$$= 2 \left[ \pi(R+r) + 2\alpha(R-r) - \left( \frac{R-r}{1} \right)^2 + 2l \right]$$

$$L = \pi(R+r) + \frac{(R-r)^2}{1} + 2l$$



$$BC = 1 \left[ 1 - \left( \frac{R-r}{2l} \right)^2 \right]$$

$$\sin \alpha = \frac{R-r}{1}$$

$$(\alpha = \frac{R-r}{1})$$

for cross drive

$$\Rightarrow L = \pi(R+r) + \frac{(R+r)^2}{1} + 2l$$