

Project 2: *Design of a cooling rib*

Wenjing KE

Christopher Reinartz

General

- This is to solve a heat exchanger model in which the ribs are cooled down by water and heated by hot gases. We need to analyze the distribution of temperature in the rib and water.

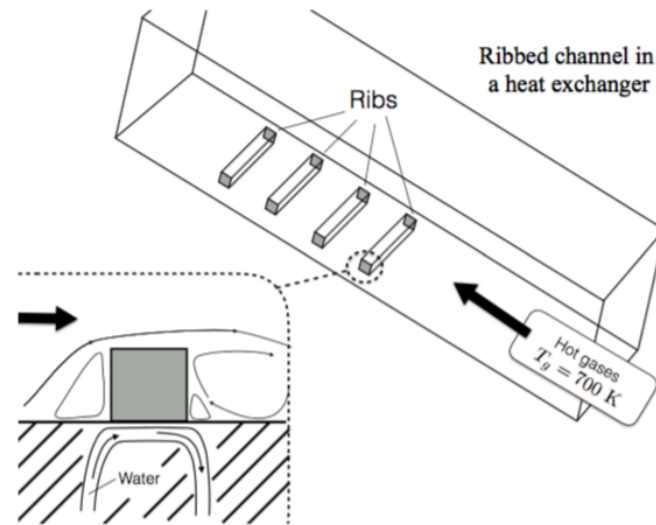
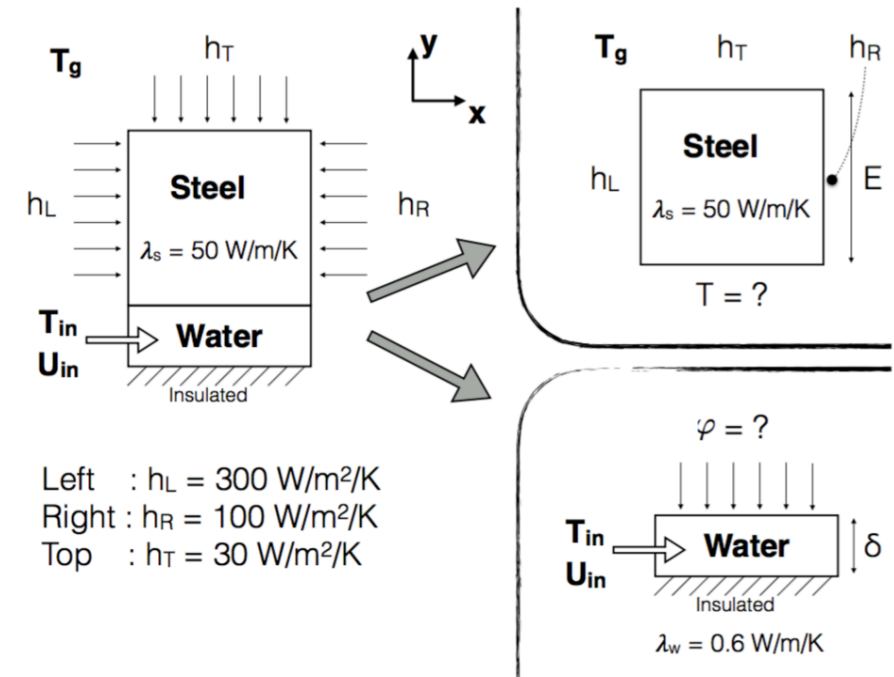


Figure 1: Water-cooled ribs in contact with hot gases

Modelization

- Separate the system into two parts, solve the rib and water part separately.
- For the steel part, suppose the temperature of steel bottom is constantly 300K, solve the distribution of temperature and calculate the heat flux transferred to the water part.
- For the water part, get the heat flux from the upper side, suppose the heat flux of right side is zero, solve the distribution of temperature and calculate the temperature maximum.

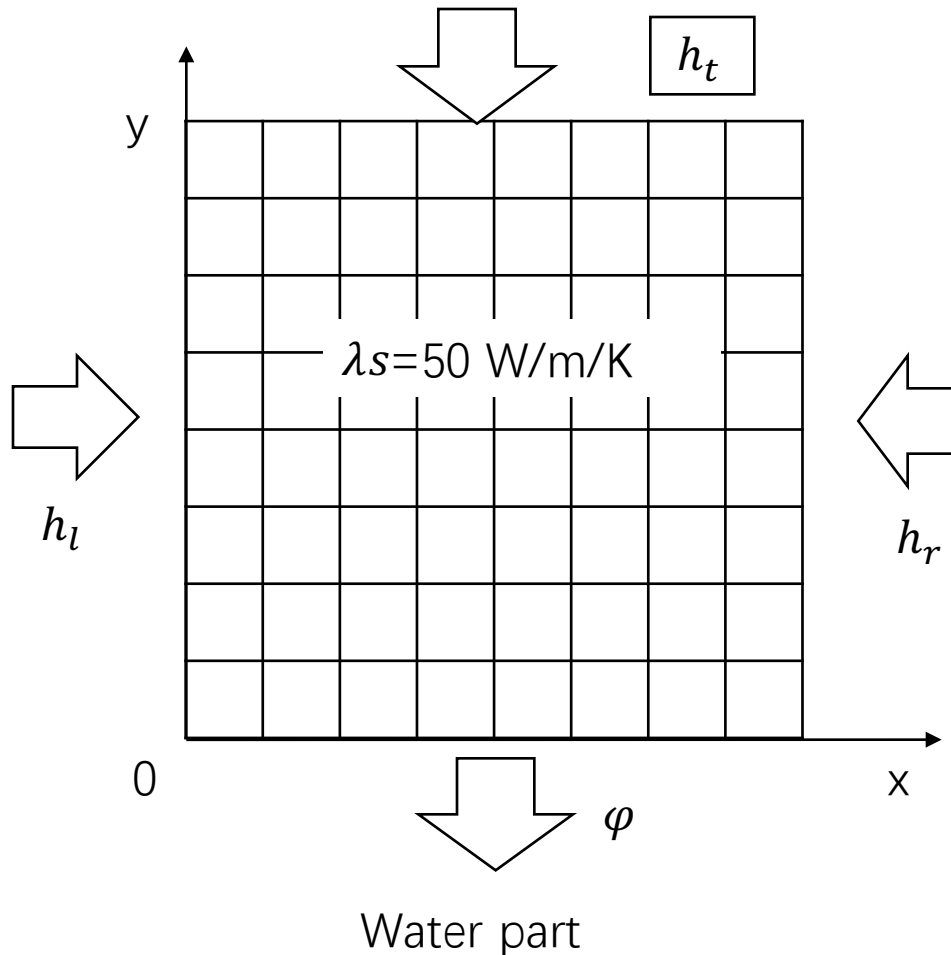


Steel part

- The steel applies the conductive heat transfer with hot gas on the left, right and top surface with $T_g = 700K$, contacts water in the bottom with $T_{bottom} = 300K$.
- Heat function
 - $\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 0$
- Boundary condition
 - $T = 300K$, *on the bottom*
 - $h(T - T_g) = \lambda_s \Delta T$, *on the left, right and top surface*

Steel part

- Discretization



- Discretize the rib to $M \times N$
- Boundary condition (1st order)
 - $$\begin{cases} T_{0,i} = 300 \\ h_l(T_g - T_{0,j}) = \lambda_s \frac{T_{0,j} - T_{1,j}}{dx} \\ h_r(T_g - T_{M,j}) = \lambda_s \frac{T_{M,j} - T_{M-1,j}}{dx} \\ h_t(T_g - T_{N,j}) = \lambda_s \frac{T_{i,N} - T_{i,N-1}}{dy} \end{cases}$$
- Heat flux: $\varphi_i = \lambda_s \frac{T_{i,1} - T_{i,0}}{dy}$

Steel part

- Heat function(SOR)

- $$\frac{T_{i+1,j} - 2T_{i,j} + T_{i-1,j}}{\Delta x^2} + \frac{T_{i,j+1} - 2T_{i,j} + T_{i,j-1}}{\Delta y^2} = 0$$

- $$T_{i,j}^{k+1} = (1 - \omega)T_{i,j}^k + \frac{\omega}{\frac{2}{\Delta x^2} + \frac{2}{\Delta y^2}} \left(\frac{T_{i+1,j}^k + T_{i-1,j}^{k+1}}{\Delta x^2} + \frac{T_{i,j+1}^k + T_{i,j-1}^{k+1}}{\Delta y^2} \right)$$

- With $\omega = 2 * (1 - \frac{\pi}{N} + \frac{\pi^2}{N^2})$

Water part

- The input temperature from left side is homogeneous $T_{in} = 300K$, flux transferred from the upper surface to heat water.

- Heat equation:

$$u(x, y) \frac{\partial T}{\partial x} = a_w \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right)$$

- With

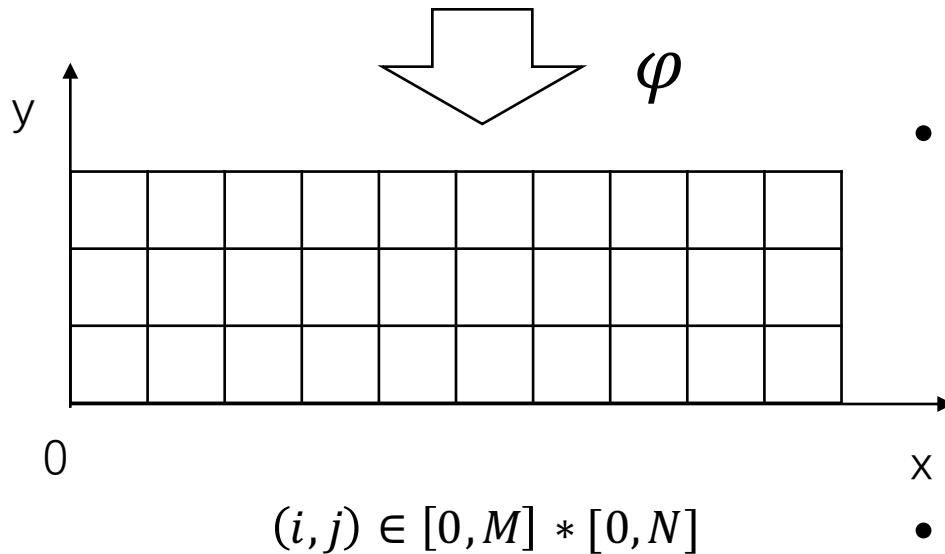
$$u(x, y) = 8U_{in} \frac{y}{\delta} \left(1 - \frac{y}{\delta} \right)$$

- Boundary condition:

$$\bullet \left\{ \begin{array}{l} T_{in} = 300K, \text{ left side} \\ -\lambda_{water} \frac{\partial T_{water}}{\partial y} = \varphi_{steel \rightarrow water}, \text{ up side} \\ \lambda_{water} \frac{\partial T_{water}}{\partial x} = 0, \text{ right side} \\ \lambda_{water} \frac{\partial T_{water}}{\partial y} = 0, \text{ bottom} \end{array} \right.$$

Water part

- Discretization



- Boundary condition(1st order):

$$\begin{cases} T_{0,j} = 300 \\ \lambda_w \frac{T_{i,N} - T_{i,N-1}}{\Delta x} = \varphi_i \\ T_{i,0} = T_{i,1} \\ T_{M,j} = T_{M-1,j} \end{cases}$$

- Heat equation(Gauss Siedel):

$$u_{i,j} \frac{T_{i,j} - T_{i-1,j}}{\Delta x} = a_w \left(\frac{T_{i+1,j} - 2T_{i,j} + T_{i-1,j}}{\Delta x^2} + \frac{T_{i,j+1} - 2T_{i,j} + T_{i,j-1}}{\Delta y^2} \right)$$

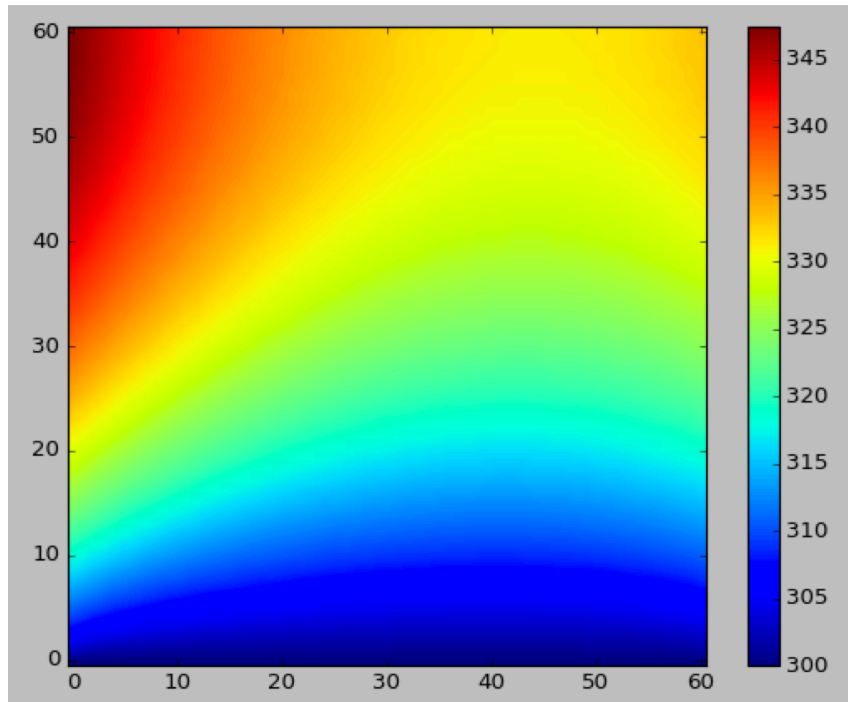
$$T_{i,j}^{k+1} = \frac{1}{\Delta x^2 + \Delta y^2 + \frac{u_{i,j}}{a_w \Delta x}} \left(\frac{T_{i+1,j}^k + T_{i-1,j}^{k+1}}{\Delta x^2} + \frac{T_{i,j+1}^k + T_{i,j-1}^{k+1}}{\Delta y^2} + \frac{u_{i,j}}{a_w} \frac{T_{i-1,j}^{k+1}}{\Delta x} \right)$$

- Velocity:

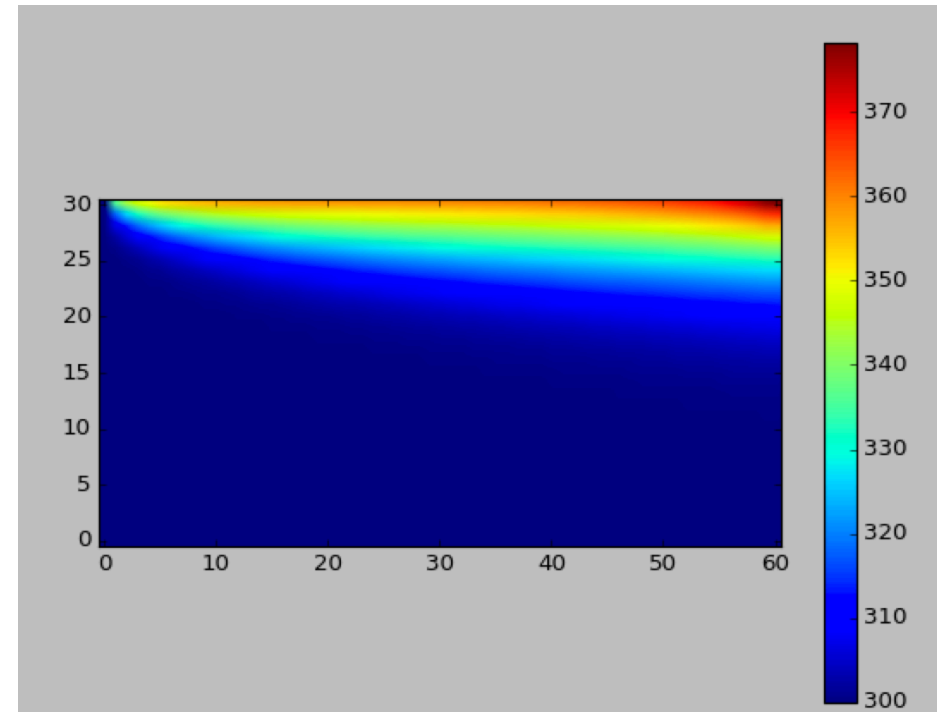
$$u_{i,j} = 8U_{in} \frac{j^* dy}{\delta} \left(1 - \frac{j^* dy}{\delta} \right)$$

Result of implementation

- Set $U_{in}=0.5$ m/s, temperature of the steel bottom $T_b = 300K$, so we get the temperature champs as below:



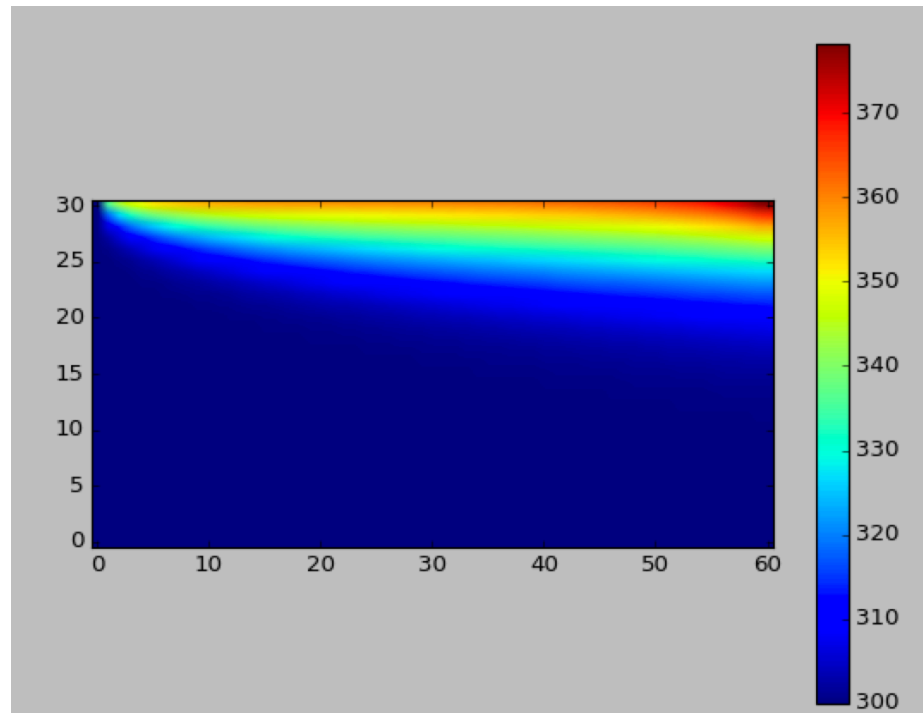
Steel part



Water part

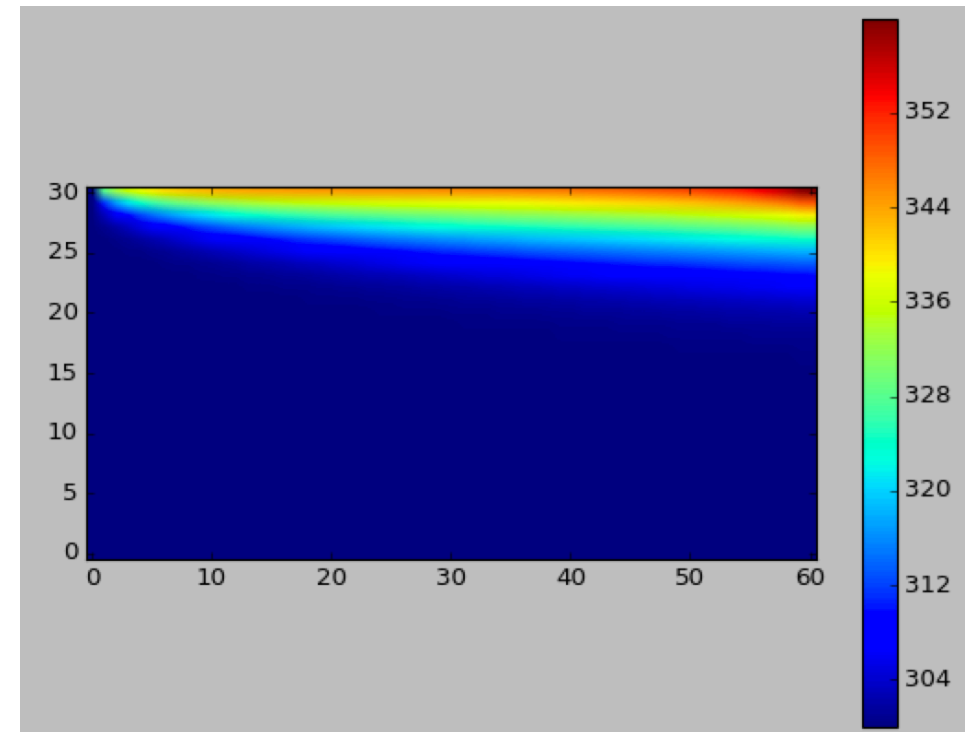
Specific questions

- Determine the bulk velocity U_{in} to cool the rib and prevent boiling of the water inside the channel: As a safety measure, water temperature is not to exceed 360 K.



$U_{in} = 0.5$, $T_{max} = 378K$

Increase U_{in}



$U_{in} = 1.04$ m/s, $T_{max} = 360.0K$

Specific questions

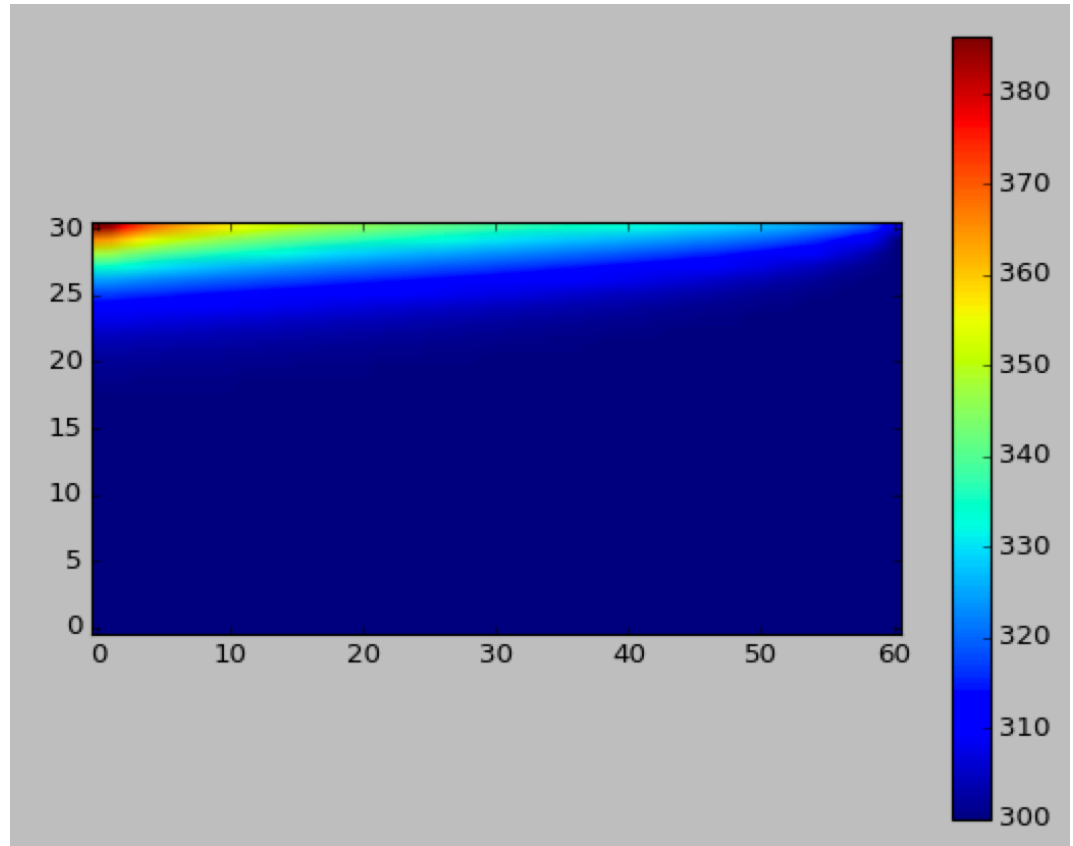
- Determine the bulk velocity U_{in} to cool the rib and prevent boiling of the water inside the channel: As a safety measure, water temperature is not to exceed 360 K.
- Response: while $U_{in} > 1.04 \text{ m/s}$, water temperature would not exceed 360K.

Specific questions

- Estimate computationally the corresponding heat power that can be extracted.
- Response:
 - we have calculated the φ on the steel-water interface, the heat power $\phi = \int \varphi ds = \int \varphi dx \cdot L_{rib}$
 - Since $\varphi_i = \lambda_s \frac{T_{i,1} - T_{i,0}}{dy}$, so $\phi = (\sum_i \varphi_i dx) \cdot L_{rib}$
 - After calculation, we get: while $U_{in}=1.04$ m/s, $\phi = (3589 \cdot L_{rib})W$

Specific questions

- The case of flow reversal:
 - While $U_{in}=1.04$ m/s, normal we get the water $T_{max}=360K$.
 - In the case of flow reversal, $T_{max}= 386K$ in the left side.
- So, if it happens a wrong setting of the water pumping system, the water will boil more easily than the normal case.



Resume

- The minimum velocity to avoid the boiling of water is $U_{in}=1.04\text{m/s}$
- While $U_{in}=1.04\text{ m/s}$, the corresponding heat power is $\phi = (3589 \cdot L_{rib})W$
- In the case of flow reversal, $T_{max}=386K$, the water boil more easily.