Osamu Katagiri - A01212611@itesm.mx - CEM

```
# Futures
      %matplotlib inline
      # from __future__ import unicode_literals
# from __future__ import print_function
      # Generic/Built-in
      import datetime
      import argparse
      # Other Libs
      from IPython.display import display, Image
      import matplotlib.pyplot as plt
      import numpy as np
      # Owned
      # from nostalgia_util import log_utils
      # from nostalgia_util import settings_util
       __authors__ = ["Osamu Katagiri - A01212611@itesm.mx"]
      __copyright__ = "None"
      __credits__ = ["Marcelo Videa - mvidea@itesm.mx"]
      __license__ = "None"
      = "Under Work"
```

# **Exercise 1**

```
In [6]: display(Image(filename='./directions/1.jpg'))
```

- Calculate the work that 1 mol of a perfect gas (\$\overline{C}\_V = 25 \text{ J/mol K}\$) produces when it expands from an initial volume of 10 L and 31.5°C to:
  - (a) a final volume of 50 L, reversibly and isothermically.
  - (b) irreversibly and isothermically against an external pressure of 0.5 atm.
  - (c) a final volume of 50 L reversibly and adiabatically.
  - (d) irreversibly and adiabatically against an external pressure of 0.5 atm.
  - (e) For all cases above, determine the final state after the expansion.

#### For 1.a & 1.e (Isothermal Reversible Expansion)

$$W_{ ext{REV}} = -RnT ext{ln}\left(rac{V_f}{V_i}
ight)$$

where:

n=1mol

 $R = 8.314JK^{-1}mol^{-1}$ 

 $T=31.5\degree C$ 

 $V_i = 10L$ 

 $V_f = 50L$ 

$$W_{\rm REV} = -4076.481J$$

$$P_f = P_i V_i V_f^{-1} = 50657.202 Pa$$
 
$$V_f = 50 L$$
 
$$T_f = T_i = 304.65 K$$

#### For 1.b & 1.e (Isothermal Irreversible Expansion)

$$W_{
m IRREV} = -P_{
m ext} \left( V_f - V_i 
ight)$$

where:

 $P_{\mathrm{ext}} = 0.5 atm$ 

 $V_f = 50L$ 

 $V_i = 10L$ 

 $W_{
m IRREV} = -20 atm L$ 

as: 1atmL=101.3J

$$W_{\rm IRREV} = -2027J$$

$$P_f = P_{ext} = 0.5atm = 50700Pa$$
 
$$V_f = 50L$$
 
$$T_f = T_i = 304.65K$$

### For 1.c & 1.e (Adiabatic Reversible Expansion)

$$P_{i} = \frac{nRT_{i}}{V_{i}} = \frac{(1mol)(8.314JK^{-1}mol^{-1})(31.5^{\circ}C)}{(10L)}$$

$$P_{i} = 253286.01Pa$$

$$\gamma = \frac{\overline{C_{p}}}{\overline{C_{v}}} = \frac{\overline{C_{v}} + R}{\overline{C_{v}}} = \frac{(25JK^{-1}mol^{-1}) + (8.314JK^{-1}mol^{-1})}{25JK^{-1}mol^{-1}}$$

$$\gamma = 1.33256$$

$$P_{f} = \frac{P_{i}V_{i}^{\gamma}}{V_{f}^{\gamma}} = \frac{(253286Pa) (10L)^{1.33256}}{(50L)^{1.33256}}$$

$$P_{f} = 29661.406Pa$$

$$W_{\text{REV}} = \frac{P_{f}V_{f} - P_{i}V_{i}}{\gamma - 1} = \frac{(29661.406Pa)(50x10^{-3}m^{-3}) - (253286.01Pa)(10x10^{-3}m^{-3})}{1.33256 - 1}$$

$$W_{\rm REV} = -3156.693J$$

$$P_f = 29661.406 Pa$$

$$V_f = 50 L \ T_f = T_i V_i^{\gamma-1} {(V_f^{\gamma-1})}^{-1} = 178.382 K$$

### For 1.d & 1.e (Adiabatic Irreversible Expansion)

$$W_{
m IRREV} = \Delta {
m U} = -PdV = n \overline{C_v} dT$$

where:

n = 1mol

 $\overline{C_v} = 25JK^{-1}mol^{-1}$ 

 $dT = T_f - T_i$ 

 $T_i = 304.65K$ 

 $P_{i}^{'}=253286.01Pa$ 

 $P_f = P_e xt = 0.5atm = 50700Pa$ 

$$T_f = rac{T_i \left(\overline{C_v} + rac{RP_f}{P_i}
ight)}{\overline{C_p}} \ T_f = 243.839 K \ W_{
m IRREV} = (1mol)(25JK^{-1}mol^{-1})(T_f - T_i)$$

$$W_{\rm IRREV} = -1520.275 J$$

$$V_f = rac{P_f = P_{ext} = 0.5 atm = 50700 Pa}{(1 mol)(8.314 JK^{-1} mol^{-1})(243.839 K)} = 40 L \ T_f = 243.839 K$$

# **Exercise 2**

In [8]: | display(Image(filename='./directions/2.jpg'))

2. A solar collector is used as a heat source for a Carnot engine with a heat sink at 300 K. The efficiency of the solar collector ε is defined as the fraction of the energy reaching the collector that is actually absorbed. It is related to the temperature of the collector as follows:

$$\varepsilon = 0.75 - 1.75 \left( \frac{T}{300 K} - 1 \right)$$

Determine the best operating temperature of the collector.

Solar collector  $\varepsilon$  is given by:

$$arepsilon = 0.75 \, -1.75 \left(rac{T}{300K} -1
ight)$$

The Carnot engine efficiency  $\eta$  is given by:

$$\eta = \frac{T}{300K} - 1$$

The system's efficiency  $\epsilon$  is given by:

$$\epsilon=arepsilon\eta \ \epsilon=-rac{1.75T^2}{(300K)^2}+rac{4.25T}{300K}-1.5$$

Let's find  $\frac{\partial \epsilon}{\partial T}$ 

$$\frac{\partial \epsilon}{\partial T} = -\frac{2(1.75)}{300^2} T + \frac{4.25}{300}$$
$$\frac{\partial \epsilon}{\partial T} = \frac{17}{1200} - \frac{7}{180000} T$$

Let's find where the function is the maximum by calculating T for  $rac{\partial \epsilon}{\partial T}=0$ 

$$rac{17}{1200} - rac{7}{180000}T = 0 \ T = 354.286K \ arepsilon = 0.75 - 1.75 \left(rac{T}{300K} - 1
ight) ext{; with } T = 354.286K$$

The operating temperature shall be T=354.286K to yeild the maximum power from the Cranot engine, with an efficiency of  $\varepsilon=0.433$ 

# **Exercise 3**

In [9]: display(Image(filename='./directions/3.jpg'))

- 3. Produce a single graph of  $P_f/P_i$  vs.  $V_f/V_i$  for the following expansion processes:
  - (a) Reversible isothermic process.
  - (b) Reversible adiabatic processes for monoatomic, diatomic and polyatomic perfect gases.
  - (c) Irreversible adiabatic processes for monoatomic, diatomic and polyatomic perfect gases.

Compare the plots in the graph and discuss the different behaviors observed. Choose the representation of your axes wisely.

### Function to compute the "next" pressure value for REVERSIBLE processes

$$P=rac{nRT^{\gamma}}{V^{\gamma}}$$

# Function to compute the "next" pressure value for IRREVERSIBLE processes

$$egin{aligned} \mathrm{d}\mathrm{U} &= -P\mathrm{d}\mathrm{V} = n\overline{C_v}\mathrm{d}\mathrm{T} \ n\overline{C_v}\int_{T_i}^{T_f}dT &= -P_f\int_{V_i}^{V_f}dV \ n\overline{C_v}(T_f - T_i) &= -P_f(V_f - V_i) \ n\overline{C_v}\left(rac{P_fV_f}{nR} - rac{P_iV_i}{nR}
ight) &= -P_f(V_f - V_i) \ rac{n\overline{C_v}P_fV_f}{nR} - rac{n\overline{C_v}P_iV_i}{nR} &= -P_f(V_f - V_i) \ P_f\left(rac{n\overline{C_v}V_f}{nR} + V_f - V_i
ight) &= rac{n\overline{C_v}P_iV_i}{nR} \end{aligned}$$

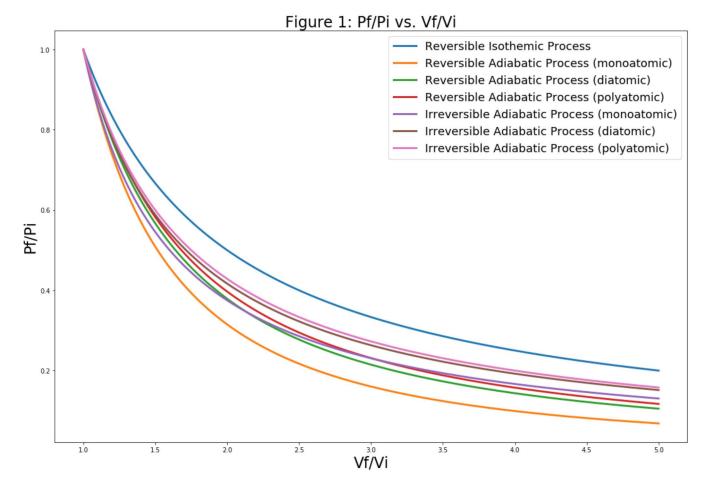
$$P_f = rac{\overline{C_v}P_iV_i}{\overline{C_v}V_f + V_fR - V_iR}$$

```
In [5]: | # Function to compute the "next" pressure value for REVERSIBLE processes
        def P_rev(n, R, T, V, gamma):
            return (n*R*(T**gamma))/(V**gamma);
        # Function to compute the "next" pressure value for IRREVERSIBLE processes
        def P_irrev(Cv, Pi, Vi, Vf, R):
            return (Cv*Pi*Vi)/(Cv*Vf+Vf*R-Vi*R);
        # Draw the plot's workspace
        n = 6:
        plt.subplots(figsize=(3*n, 2*n))
        # Initial State, Constants & Final Volume
        Ti = 304.65; #K
        Vi = 0.01; #10 L
        Vf = 0.05; #50 L
        V = np.linspace(Vi,Vf,1000);
        n = 1; #mol
        R = 8.314; \#J/K mol)
        # Reversible Isothemic Process
        gamma = 1;
        Vf = V;
        Pi = P_rev(n, R, Ti, Vi, gamma);
        Pf = P_rev(n, R, Ti, V, gamma);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Reversible Isothemic Process');
        # Reversible Adiabatic Process (monoatomic)
        Cv = (3/2)*R;
        Cp = Cv + R;
        gamma = Cp/Cv;
        Vf = V;
        Pi = P_rev(n, R, Ti, Vi, gamma);
        Pf = P_rev(n, R, Ti, V, gamma);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Reversible Adiabatic Process (monoatomic)');
        # Reversible Adiabatic Process (diatomic)
        Cv = (5/2)*R;
        Cp = Cv + R;
        gamma = Cp/Cv;
        Vf = V;
        Pi = P_rev(n, R, Ti, Vi, gamma);
        Pf = P_rev(n, R, Ti, V, gamma);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Reversible Adiabatic Process (diatomic)');
        # Reversible Adiabatic Process (polyatomic)
        Cv = 3*R;
        Cp = Cv + R;
        gamma = Cp/Cv;
        Vf = V;
        Pi = P_rev(n, R, Ti, Vi, gamma);
        Pf = P_rev(n, R, Ti, V, gamma);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Reversible Adiabatic Process (polyatomic)');
        # Irreversible Adiabatic Process (monoatomic)
        Cv = (3/2)*R;
        Cp = Cv + R;
        gamma = Cp/Cv;
        #Tf = 243.839; #K (from exercice 1.d)
        \#Vf = (n*R*Tf)/(50700); \#L
        \#V = np.linspace(Vi, Vf, 1000);
        Vf = V;
        Pi = P_rev(n, R, Ti, Vi, gamma);
        Pf = P irrev(Cv, Pi, Vi, Vf, R);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Irreversible Adiabatic Process (monoatomic)');
        # Irreversible Adiabatic Process (diatomic)
        Cv = (5/2)*R;
        Cp = Cv + R;
        gamma = Cp/Cv;
        Vf = V;
        Pi = P rev(n, R, Ti, Vi, gamma);
        Pf = P_irrev(Cv, Pi, Vi, Vf, R);
        plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Irreversible Adiabatic Process (diatomic)');
```

```
# Irreversible Adiabatic Process (polyatomic)
Cv = 3*R;
Cp = Cv + R;
gamma = Cp/Cv;
Vf = V;
Pi = P_rev(n, R, Ti, Vi, gamma);
Pf = P_irrev(Cv, Pi, Vi, Vf, R);
plt.plot(Vf/Vi, Pf/Pi, '-', linewidth=3, label='Irreversible Adiabatic Process (polyatomic)');

plt.xlabel('Vf/Vi', fontsize=24);
plt.ylabel('Pf/Pi', fontsize=24);
plt.title("Figure 1: Pf/Pi vs. Vf/Vi", size=24)
plt.legend(prop={'size': 18})
display(plt);
```

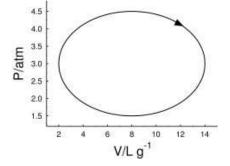
 $$$\mbox{}\mbox{$\mbox{}\box{$\mbox{$ 



### **Exercise 4**

```
In [10]: display(Image(filename='./directions/4.jpg'))
```

4. A PV diagram of an imaginary heat engine is represented in the following figure.



- (a) Find the work performed per cycle by 1 g of working fluid.
- (b) Find the engine efficiency if it rejects 5.7 kJ/g during each cycle

The work performed per cycle is given by the area within the PV curve.

$$W_{performed} = \pi (14-8)(4.5-3.0) atm Lg^{-1} \ W_{performed} = 9\pi atm Lg^{-1}$$

as: 1atmL = 101.3J

$$W_{performed} = rac{9117}{10} \pi J g^{-1} = 2864.190 J g^{-1}$$

$$\eta_{engine} = rac{W_{performed}}{W_{performed} + W_{rejected}}$$

$$\eta_{engine} = rac{2864.190 J g^{-1}}{2864.190 J g^{-1} + 5700 J g^{-1}} = 0.334$$

#### **Exercise 5**

In [3]: display(Image(filename='./directions/5.jpg'))

5. A heat engine, of which all steps in a cycle are reversible, absorbs thermal energy from a high-temperature reservoir, performs an amount of net work w<sub>net</sub>, and rejects thermal energy into a lowtemperature reservoir. Initially, the reservoirs are at temperatures T<sub>1</sub> and T<sub>2</sub>. Their heat capacities are constant with values C<sub>1</sub> and C<sub>2</sub>, respectively. Calculate what will be the final temperatures for the heat reservoirs and the maximum amount of work produced by the engine.

The following is extracted from: Mungan, C. E. (2009). Heat engine with finite thermal reservoirs and nonideal efficiency. Latin-American Journal of Physics Education, 3(2), 239–242.

Let's denote some variables:

 $dQ_H$  = heat transfer out of the hot reservoir

 $dQ_C$  = heat transfer into the cold reservoir

dW = work output by the engine

 $dT_H$  = change in temperature of the hot reservoir ( $dT_H < 0$  as it's cooling down)

 $T_{Ho}$  =  $T_1$  = initial temperature of the hot reservoir

 $dT_C$  = change in temperature of the cold reservoir ( $dT_C>0$  as it warms up)

 $T_{Co}$  =  $T_2$  = initial temperature of the cold reservoir

T = final temperature of both reservoirs

 $\eta$  = 2nd law efficiency given

As both reservoirs have temperature-independent heat capacities  $C_1$  and  $C_2$ , then it follows that:

$$dQ_H = -C_1 T_H \ dQ_C = C_2 T_C$$

The engine efficiency can be defined as:

$$\varepsilon = \frac{dW}{dQ_H} = 1 - \frac{dQ_C}{dQ_H} = 1 + \frac{dT_C}{dT_H}$$

With the following Carnot limiting efficiency:

$$arepsilon_c = 1 - rac{T_C}{T_H}$$

The 2nd law efficiency  $\eta$  is assumed to be constant.

$$\eta = rac{W_{net,out}}{Q_H} = rac{Q_H - Q_C}{Q_H} = 1 - rac{Q_C}{Q_H}$$

Rearranging terms,  $T_{C}$  can be defined as a function of  $T_{H}$ :

$$T_H rac{dT_C}{dT_H} + \eta T_C = (\eta - 1) T_H$$

The complementary solution of the previous is  $AT_H^{-\eta}$  where A is a constant to taylor the initial conditions. On the other hand, by trying a particular solution of the equation that is proportional to  $T_H$ , the following is obtained  $(\eta-1)T_H/(\eta+1)$ . Adding together these complementary and particular solutions and fitting to the initial temperatures of the two reservoirs gives:

$$rac{T_C}{T_{
m Ho}} = \left(rac{T_{
m Ho}}{T_H}
ight)^{\eta} \left\{rac{T_{
m Co}}{T_{
m Ho}} + rac{1-\eta}{1+\eta} \Biggl(1-\left(rac{T_H}{T_{
m Ho}}
ight)^{\eta+1}\Biggr)
ight\}$$

Let's set  $T_C=T_H=T$  to find the common final temperature of the two reservoirs:

$$rac{T}{T_{Ho}} = \left(rac{(T_{Co} + T_{Ho}) + \eta(T_{Co} - T_{Ho})}{2T_{Ho}}
ight)^{rac{1}{1+\eta}}$$

$$T = T_{Ho} igg( rac{(T_{Co} + T_{Ho}) + \eta (T_{Co} - T_{Ho})}{2T_{Ho}} igg)^{rac{1}{1+\eta}}$$

where:

 $T_{
m min} = \sqrt{T_{
m Co} T_{
m Ho}}$  is the geometric average of the initial temperatures

 $T_{
m max} = rac{Y_{Co} + T_{Ho}}{2}$  is their aithmetic average

The total work output by the engine is the difference in heat transfers for the two reservoirs, as follows:

$$W = Q_H - Q_C = C_1(T_{Ho} - T) - C_2(T - T_{Co})$$

And the maximum amount of work is given by:

$$W_{max} = C_1 C_2 (T_{max} - T_{min})$$

The maximum work is achieved for reversible operation of the engine when  $\eta=1$ . On the other hand, zero work is output for maximally irreversible operation of the engine when  $\eta=0$ .

