

1 Conduction

1.1 Definition

Heat transfer in a body due to temperature difference

1.2 Fouriers Law

$$\vec{q}'' = -\lambda \nabla T$$

1D: $q_x'' = \frac{\dot{Q}}{A} = -\lambda \frac{dT}{dx}$

$\vec{q}'' = \text{Heat flux, } \left[\frac{W}{m^2} \right]$ $\lambda = \text{thermal conductivity, } \left[\frac{W}{mK} \right]$

3-Dimensional equations follow the fouriers and fick's law:

$$\vec{q}'' = -\lambda \underbrace{\frac{\partial T}{\partial x} \hat{x}}_{q_x''} - \lambda \underbrace{\frac{\partial T}{\partial y} \hat{y}}_{q_y''} - \lambda \underbrace{\frac{\partial T}{\partial z} \hat{z}}_{q_z''}$$
$$\vec{q}'' = -\lambda \underbrace{\frac{\partial T}{\partial r} \hat{r}}_{q_r''} - \lambda \underbrace{\frac{\partial T}{r \partial \phi} \hat{\phi}}_{q_\phi''} - \lambda \underbrace{\frac{\partial T}{\partial z} \hat{z}}_{q_z''}$$
$$\vec{q}'' = -\lambda \underbrace{\frac{\partial T}{\partial r} \hat{r}}_{q_r''} - \lambda \underbrace{\frac{\partial T}{r \partial \theta} \hat{\theta}}_{q_\theta''} - \lambda \underbrace{\frac{\partial T}{r \sin(\theta) \partial \phi} \hat{\phi}}_{q_\phi''}$$

Koordinatensysteme Zylindrisch und Sphärisch

Some common thermal conductivities	
Material	thermal conductivity $\lambda \left[\frac{W}{mK} \right]$
Aluminium:	240
Copper:	400
Carbon steels:	40-60
Stainless steels:	15
Helium:	0.152
Air or N_2 :	0.026
Steam (100°C):	0.025
Plastic:	0.2
Rubber:	0.16

1.3 Heat Conduction Equation

Remember: $\nabla = \begin{bmatrix} \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} \\ \frac{\partial}{\partial z} \end{bmatrix}$

$$\frac{\partial}{\partial t} \left(\underbrace{\vec{\nabla} \cdot (\lambda \vec{\nabla} T)}_{\text{Change in thermal energy storage}} + \underbrace{\dot{Q}_{\text{source}}'''}_{\text{Thermal energy generation}} \right) + \frac{\partial}{\partial y} \left(\lambda \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left(\lambda \frac{\partial T}{\partial z} \right) + \dot{Q}_{\text{source}}'''$$
$$\lambda r \frac{\partial T}{\partial r} + \frac{1}{r^2} \frac{\partial}{\partial \varphi} \left(\lambda \frac{\partial T}{\partial \varphi} \right) + \frac{\partial}{\partial z} \left(\lambda \frac{\partial T}{\partial z} \right) + \dot{Q}_{\text{source}}'''$$
$$\lambda r^2 \frac{\partial T}{\partial r} + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \varphi} \left(\lambda \frac{\partial T}{\partial \varphi} \right) + \frac{1}{r^2 \sin(\theta)} \frac{\partial}{\partial \theta} \left(\lambda \sin(\theta) \frac{\partial T}{\partial \theta} \right) + \dot{Q}_{\text{source}}'''$$

1.3.1 Solving the Heat Conduction Equation: Boundary conditions (Noch machen, Lecture 2 oder 3)

- Surface Temperature known
- Heat Flux known
- Convection known

2 Convection

2.1 Heat transfer by convection

$$\vec{q}'' = \text{Heat flux, } \left[\frac{W}{m^2} \right]$$
$$q'' = \frac{\dot{Q}}{A} = \alpha (T_\infty - T_W)$$

$\alpha = \text{heat transfer coefficient, } \left[\frac{W}{mK} \right]$

3 Thermal Resistances

3.1 Thermal resistance model

assumption: $\lambda = const$

$$\dot{Q}_x = \dot{Q}_x'' A = \frac{1}{R_{th}} \Delta T$$

Geometry	Conduction R_{th}	Convection R_{th}
Planar	$\frac{L}{\lambda A}$	$\frac{1}{\alpha A}$
Cylinder	$\frac{\ln \left(\frac{r_2}{r_1} \right)}{2 \pi \lambda L}$	$\frac{1}{2 \pi r_1 L \alpha}$
Sphere	$\frac{\frac{1}{R_1} - \frac{1}{R_2}}{4 \pi \lambda}$	$\frac{1}{4 \pi r^2 \alpha}$

Note that for some pipes, there exists a critical thickness of the pipe insulation, for which the thermal insulation of a thin insulation layer is lower than without insulation layer.
 $R_{crit} : \frac{d}{dr_2} (R_{tot}) = 0, R_{tot} = R_{th,ins} + R_{th,conv}$
 $\Rightarrow r_{2,crit} = \frac{\lambda}{\alpha}$

4 Lumped capacitance method

4.1 Lumped capacitance

Idea: The temperature in a body is almost uniform, so we can assume it to be uniform. Temperature within body will now be $T(t)$ instead of $T(t, x, y, z)$. That means, the temperature difference inside the body $\Delta T_i = T_{s,1} - T_{s,2}$ must be much smaller than the temperature difference outside the body $\Delta T_o = T_{s,2} - T_\infty$.

$$\dot{q} = \frac{\lambda A}{L} (T_{s,1} - T_{s,2}) = \alpha A (T_{s,2} - T_\infty)$$
$$\Rightarrow \frac{(T_{s,1} - T_{s,2})}{(T_{s,2} - T_\infty)} = \frac{\alpha L}{\lambda} = Bi$$
$$L_{cylinder} = \frac{r}{2} \quad L_{sphere} = \frac{r}{3}$$

