Thermodynamics III Cheatsheet

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l Conduction

1.1 Definition

Heat transfer in a body due to temperature difference

1.2 Fouriers Law

$$\vec{q}^{\prime\prime} = -\lambda \nabla T$$

$$\vec{q}^{\prime\prime} = -k \nabla T$$

$$1\text{D:} \quad \vec{q}^{\prime\prime\prime} = \frac{\dot{Q}}{A} = -\lambda \frac{dT}{dx}$$

$$\lambda = \text{thermal conductivity, } \left[\frac{W}{mK}\right]$$

3-Dimensional equations follow the fouriers and fick's law:

$$\begin{split} \vec{q}^{\prime\prime} &= \underbrace{-\lambda \frac{\partial T}{\partial x} \hat{x}}_{\dot{q}_x^{\prime\prime}} - \lambda \frac{\partial T}{\partial y} \hat{y}}_{\dot{q}_y^{\prime\prime}} - \lambda \frac{\partial T}{\partial z} \hat{z}}_{\dot{q}_z^{\prime\prime}} \\ \vec{q}^{\prime\prime} &= \underbrace{-\lambda \frac{\partial T}{\partial r} \hat{r}}_{\dot{q}_x^{\prime\prime}} - \lambda \frac{\partial T}{r \partial \phi} \hat{\phi}}_{\dot{q}_y^{\prime\prime}} - \lambda \frac{\partial T}{\partial z} \hat{z}}_{\dot{q}_z^{\prime\prime}} \\ \vec{q}^{\prime\prime} &= \underbrace{-\lambda \frac{\partial T}{\partial r} \hat{r}}_{\dot{q}_r^{\prime\prime}} - \lambda \frac{\partial T}{r \partial \theta} \hat{\phi}}_{\dot{q}_y^{\prime\prime}} - \lambda \frac{\partial T}{r \sin(\theta) \partial \phi} \hat{\phi}}_{\dot{q}_y^{\prime\prime}} \end{split}$$

Koordinatensysteme Zylindrisch und Sphärisch

Some common thermal conductivities

Material	thermal conductivity $\lambda \left[\frac{W}{mK} \right]$	
Aluminium:	240	
Copper:	400	
Carbon steels:	40-60	
Stainless steels:	15	
Helium:	0.152	
Air or N_2 :	0.026	
Steam (100°C):	0.025	
Plastic:	0.2	
Rubber:	0.16	

1.3 Heat Conduction Equation

Remember:
$$\nabla = \begin{bmatrix} \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} \end{bmatrix}$$

1.3.1 Solving the Heat Conduction Equation: Boundary conditions (Noch machen, Lecture 2 oder 3)

- Surface Temperature known
- Heat Flux known
- Convection known

2 Convection

2.1 Heat transfer by convection

3 Thermal Resistances

3.1 Thermal resistance model

assumption: $\lambda = const$

$$\dot{Q_x} = \dot{Q_x}^{"}A = \frac{1}{R_{th}}\Delta T$$

Geometry Planar	Conduction R_{th} $\frac{L}{\lambda A}$	Convection $R_{\frac{1}{\alpha A}}$
Cylinder	$\frac{\ln\left(\frac{r_2}{r_1}\right)}{2\pi\lambda L}$	$\frac{1}{2\pi r_1 L \alpha}$
Sphere	$\frac{\frac{1}{R_1} - \frac{1}{R_2}}{4\pi\lambda}$	$\frac{1}{4\pi r^2 \alpha}$

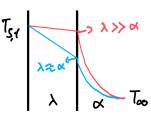
Note that for some pipes, there exists a critical thickness of the pipe insulation, for which the thermal insulation of a thin insulation layer is lower than without insulation layer.

$$R_{crit}: \frac{d}{dr_2}(R_{tot}) = 0, R_{tot} = R_{th,ins} + R_{th,conv}$$

4 Lumped capacitance method

4.1 Lumped capacitance

Idea: The temperature in a body is almost uniform, so we can assume it to be uniform. Temperature within body will now be T(t) instead of T(t,x,y,z). That means, the temperature difference inside the body $\Delta T_i = T_{s,1} - T_{s,2}$ must be much smaller than the temperature difference outside the body $\Delta T_o = T_{s,2} - T_{\infty}$.



$$\begin{split} \dot{q} &= \frac{\lambda A}{L} (T_{s,1} - T_{s,2}) = \alpha A (T_{s,2} - T_{\infty}) \\ &\Rightarrow \frac{(T_{s,1} - T_{s,2})}{(T_{s,2} - T_{\infty})} = \frac{\alpha L}{\lambda} = Bi \\ &L_{\text{cylinder}} = \frac{r}{2} \qquad L_{\text{sphere}} = \frac{r}{3} \end{split}$$