

# Lecture 11

*Rigid body kinematics: Velocity analysis examples;  
Five-term acceleration formula; Applications*

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*1-7 September, 2021*

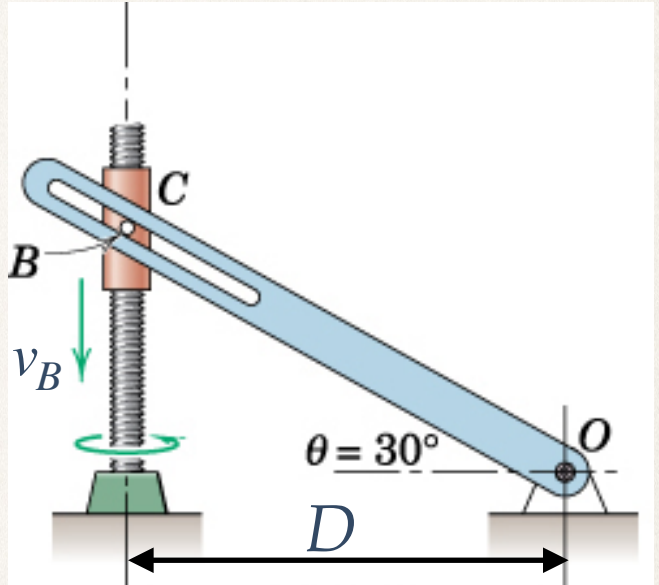


# Example 1

*Find the angular velocity of the slotted arm*

**General strategy** for systems with connected rigid bodies:

1. Connections impose kinematic constraints.
2. Usually partial kinematic information is given for each rigid body. *For example*, slotted arm rotates in plane; point  $O$  on arm is fixed; collar  $C$  moves vertically.
3. Find a convenient point whose motion is found in two ways, say *w.r.t.* two rigid bodies, and equate. *For example*, point  $B$  is on collar  $C$  and velocity  $\mathbf{v}_B$  is known. But  $\mathbf{v}_B$  can also be found by sitting in the slotted arm's *BFC*S. Comparing the two expressions of  $\mathbf{v}_B$  solves the problem.

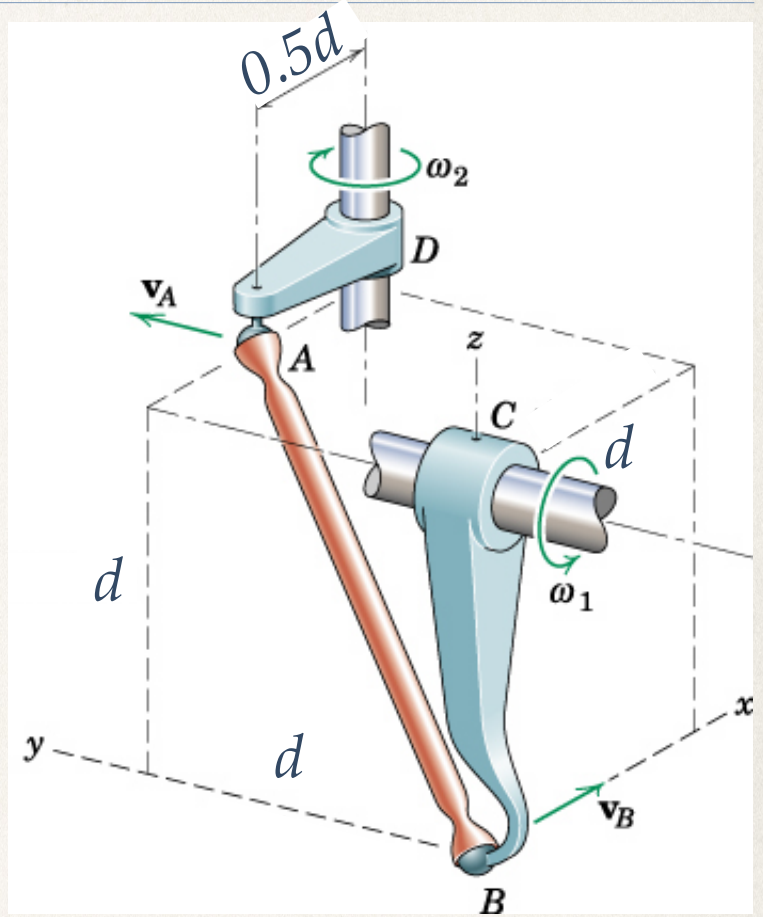




# Example 2

*Given  $\omega_1$ , find  $\omega_2$  and  $\omega^{AB}$  in this configuration.*

**Note:** Even when stationary, link  $AB$  can rotate in an arbitrary manner about  $AB$ , because of spherical joints at hinges  $A$  and  $B$ .



**Implication:** We can write

$$\omega^{AB} = \underbrace{(\omega^{AB} \cdot \hat{\mathbf{n}}_{AB}) \hat{\mathbf{n}}_{AB}}_{\omega_a^{AB}, \text{ along } AB} + \underbrace{\hat{\mathbf{n}}_{AB} \times (\omega^{AB} \times \hat{\mathbf{n}}_{AB})}_{\omega_n^{AB}, \text{ normal to } AB}$$

1. The **Note** above says that we can not expect to find  $\omega_a^{AB} = (\omega_{AB} \cdot \hat{\mathbf{n}}_{AB}) \hat{\mathbf{n}}_{AB}$ .
2. Will only find  $\omega_n^{AB} = \hat{\mathbf{n}}_{AB} \times (\omega_{AB} \times \hat{\mathbf{n}}_{AB})$ .



# Acceleration analysis

## Relating acceleration of point $P$ in two CS.

1.  $P$ 's acceleration is  $\mathbf{a}_{\mathcal{E}}^P (=:\mathbf{a}_{rel}^P)$  w.r.t. rigid body  $\mathcal{B}$  with BFCS  $\{\mathcal{E}(t), G, \hat{\mathbf{e}}_i(t)\}$ .
2.  $\mathcal{B}$  rotates at  $\boldsymbol{\omega}_{\mathcal{B}} := \boldsymbol{\omega}_{\mathcal{E}/\mathcal{E}_0}$  and  $\boldsymbol{\alpha}_{\mathcal{B}} = \boldsymbol{\alpha}_{\mathcal{E}/\mathcal{E}_0}$  w.r.t. CS  $\{\mathcal{E}_0, O, \hat{\mathbf{E}}_i\}$ .

Find  $P$ 's acceleration w.r.t.  $\mathcal{E}_0$ , i.e.  $\mathbf{a}_{\mathcal{E}_0}^P =: \mathbf{a}^P$

In general,  $\mathcal{E}_0$  and  $\mathcal{E}$  have different origins:

- a.  $\mathcal{E}$ 's origin is at  $\mathbf{r}^{G/O}(t)$  w.r.t.  $O$ , and
- b.  $\mathcal{E}$ 's origin has acceleration  $\mathbf{a}_{\mathcal{E}_0}^G =: \mathbf{a}^G$ :

$$\mathbf{a}^P = \mathbf{a}_{rel}^P + \boldsymbol{\omega}_{\mathcal{B}} \times (\boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{r}^{P/G}) \dots$$
$$\dots + \boldsymbol{\alpha}_{\mathcal{B}} \times \mathbf{r}^{P/G} + 2\boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{v}_{rel}^P + \mathbf{a}^G.$$

- c.  $\mathbf{a}^G = 0$ , if  $\mathcal{E}_0$  and  $\mathcal{E}$  have same origins.



# “Five-term” formula

$$\mathbf{a}^P = \mathbf{a}_{rel}^P + \boldsymbol{\omega}_{\mathcal{B}} \times \left( \boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{r}^{P/G} \right) + \boldsymbol{\alpha}_{\mathcal{B}} \times \mathbf{r}^{P/G} + \dots$$
$$\dots + 2\boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{v}_{rel}^P + \mathbf{a}^G.$$

Acceleration  $\mathbf{a}^P$  of  $P$  in  $\{\mathcal{E}_0, O, \hat{\mathbf{E}}_i\}$  is found by measuring  $P$ 's motion in  $\{\mathcal{E}(t), G, \hat{\mathbf{e}}_i(t)\}$  which rotates at  $\boldsymbol{\omega}_{\mathcal{B}}$  and  $\boldsymbol{\alpha}_{\mathcal{B}}$  and translates w.r.t.  $\mathcal{E}_0$ .

I. FIVE terms in above formula:

1.  $\mathbf{a}_{rel}^P$ :  $P$ 's acceleration *measured in*  $\mathcal{E}$ .
2.  $\boldsymbol{\omega}_{\mathcal{B}} \times \left( \boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{r}^{P/G} \right)$ : Centripetal acceleration
3.  $\boldsymbol{\alpha}_{\mathcal{B}} \times \mathbf{r}^{P/G}$ : Angular acceleration
4.  $2\boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{v}_{rel}^P$ : Coriolis acceleration
5.  $\mathbf{a}^G$ :  $G$ 's acceleration *measured in*  $\mathcal{E}_0$ .

II.  $\boldsymbol{\omega}_{\mathcal{B}} \times \left( \boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{r}^{P/G} \right) + \boldsymbol{\alpha}_{\mathcal{B}} \times \mathbf{r}^{P/G}$ : Measurement in rotating  $\mathcal{E}$  misses obvious rotational effects

III.  $2\boldsymbol{\omega}_{\mathcal{B}} \times \mathbf{v}_{rel}^P$ : **Two** sources — measurement of rate of change of  $\mathbf{v}_{rel}^P$  and  $\mathbf{r}^{P/G}$  in rotating  $\mathcal{E}$ .



# Application

I. **Example 1.** *Relating accelerations of two points on the same rigid body.* Let  $A$  and  $B$  be points on a rigid body  $\mathcal{B}$ , rotating at  $\omega_{\mathcal{B}}$  and  $\alpha_{\mathcal{B}}$  w.r.t.  $\{\mathcal{E}_0, O, \hat{\mathbf{E}}_i\}$ . Let  $\mathbf{a}^A$  be acceleration of  $A$  in  $\mathcal{E}_0$ .

Find  $\mathbf{a}^B$  measured w.r.t.  $\mathcal{E}_0$ .

**Answer:**  $\mathbf{a}^B = \mathbf{a}^A + \omega_{\mathcal{B}} \times (\omega_{\mathcal{B}} \times \mathbf{r}^{B/A}) + \alpha_{\mathcal{B}} \times \mathbf{r}^{B/A}$

II. **Example 2.** Find acceleration of the rod's end.

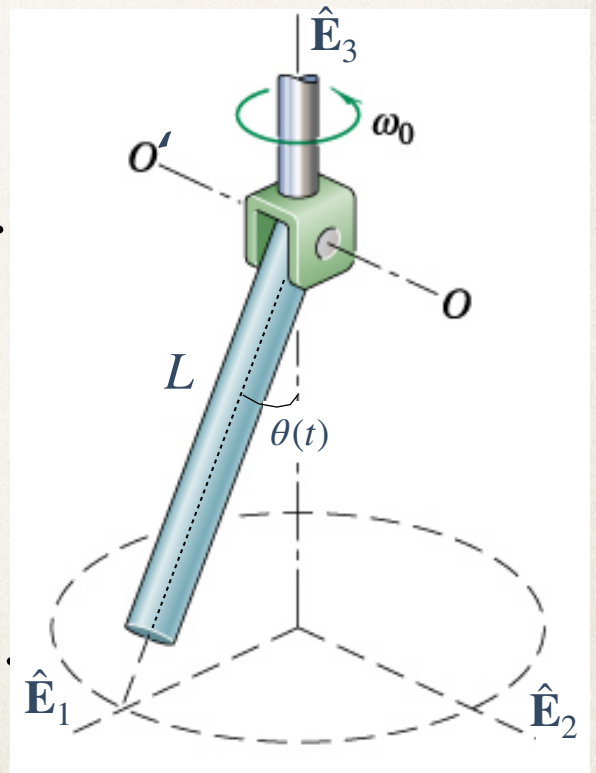
**Answer:**

$$\left\{ -(\omega_0^2 + \dot{\theta}^2) \sin \theta + \dots \right.$$

$$\left. \ddot{\theta} \cos \theta \right\} L \hat{\mathbf{E}}_1 + \dots$$

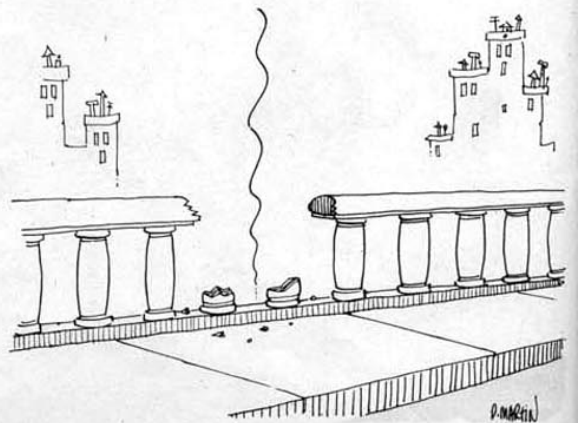
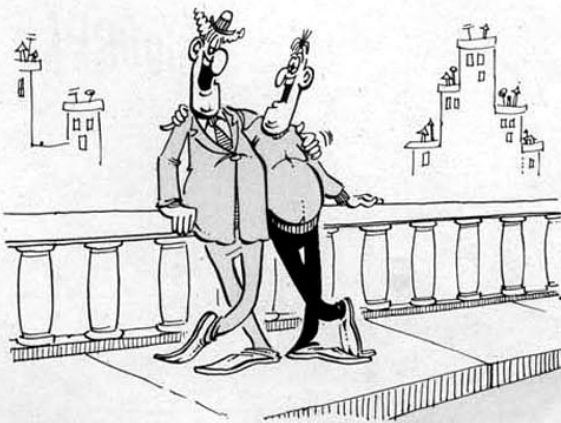
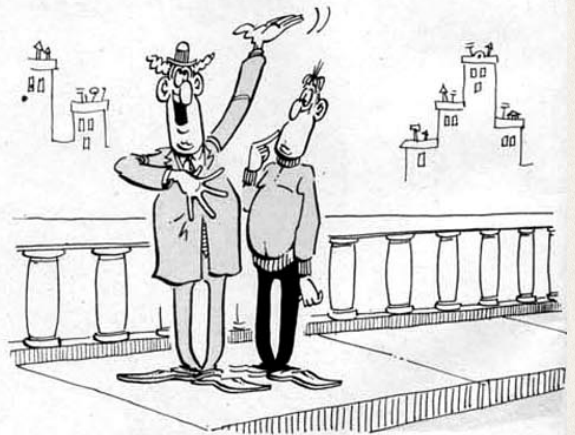
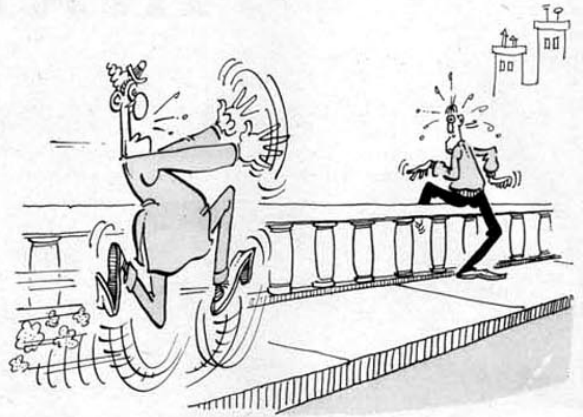
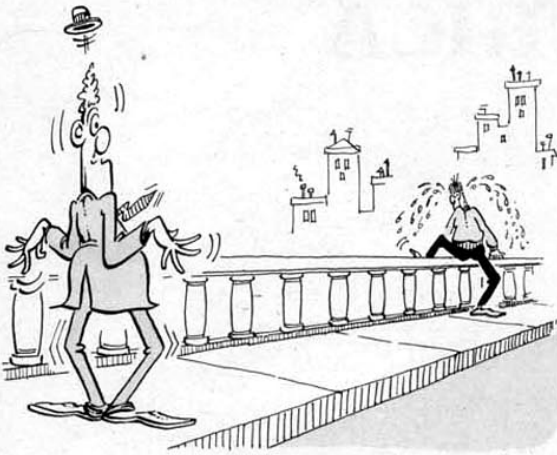
$$2\omega_0 \dot{\theta} L \cos \theta \hat{\mathbf{E}}_2 + \dots$$

$$(\dot{\theta}^2 \cos \theta + \ddot{\theta} \sin \theta) L \hat{\mathbf{E}}_3.$$





# ONE DAY ON THE BRIDGE



Don't cheat, nor allow others.  
Be careful who you help / know.