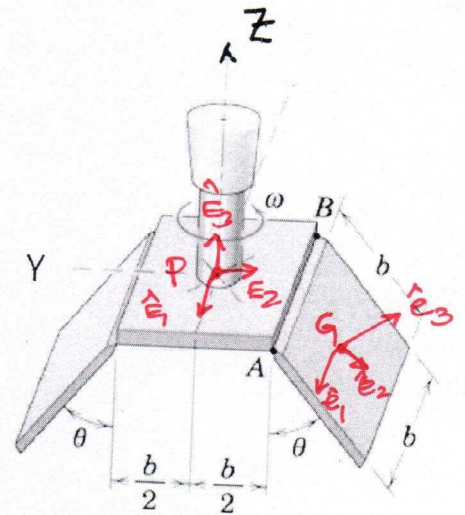
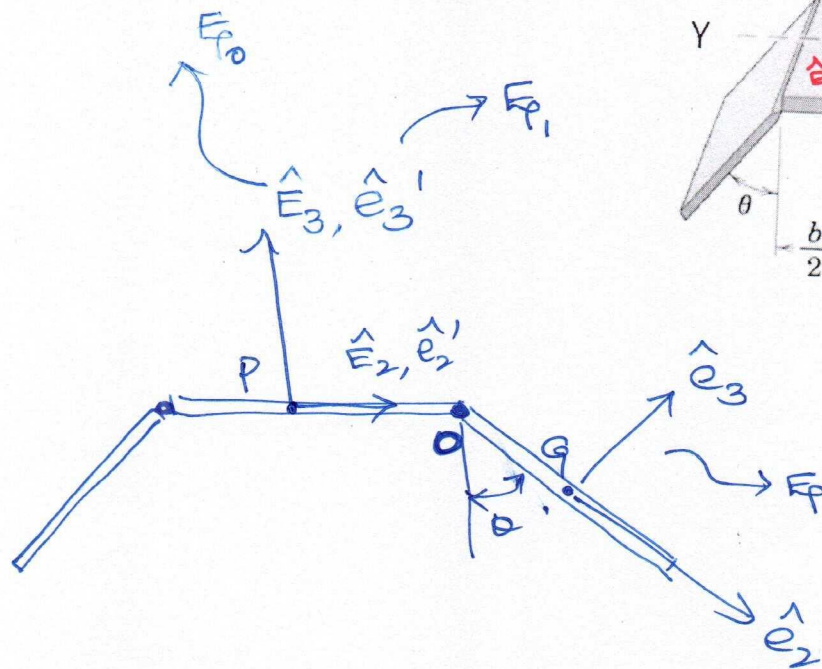


ESO 209.  
Tutorial 8.

①

Problem 6

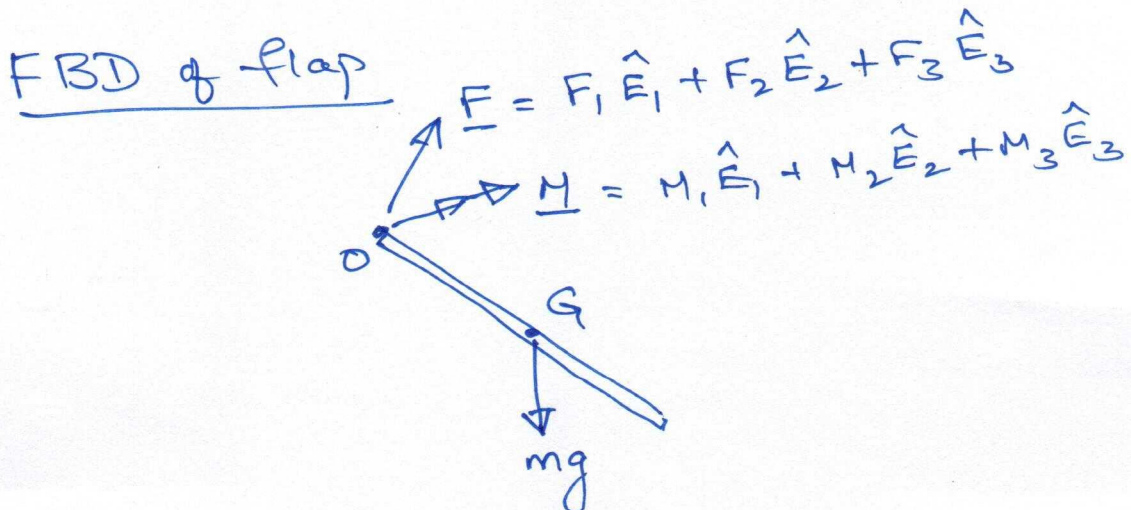


Three coordinate system.

1)  $\{E_0, P, \hat{E}_i\}$  observer CS

2)  $\{E_1, P, \hat{E}_i^1\}$  BFCS of Horizontal panel.  
Aligned with  $E_P$  at instant of interest.

3)  $\{E_P, G, \hat{E}_i\}$  BFCS of flap (G is at CM of flap)  
 $\hat{E}_1$  aligned with  $\hat{E}_1$  at instant of interest.



## Linear Momentum Balance.

(2)

LMB states

$$\sum \underline{F}^i = m \underline{a}^G$$

$$\text{LHS} = F_1 \hat{E}_1 + F_2 \hat{E}_2 + F_3 \hat{E}_3 - mg \hat{E}_3$$

RHS

To find  $\underline{a}^G$ , we use the BFCS  $\mathcal{F}_P$ , in which P.O.G. is like a rigid body ( $\theta = \text{const}$ )

$$\therefore \underline{a}^G = \underline{a}^P + \underline{\omega}^B \times (\underline{\omega}^B \times \underline{r}^{G/P}) + \underline{\alpha}^B \times \underline{r}^{G/P}$$

Since P is fixed,  $\underline{a}^P = 0$

$$\underline{\alpha}^B = \frac{d}{dt} (\underline{\omega}^B) = \frac{d}{dt} (\omega \hat{E}_3) = 0$$

since both  $\omega$  &  $\hat{E}_3$  are constant

$$\underline{r}^{G/P} = \left( \frac{b}{2} + \frac{b}{2} \sin \theta \right) \hat{E}_2 - \frac{b}{2} \cos \theta \hat{E}_3$$

$$\begin{aligned} \therefore \underline{a}^G &= \omega \hat{E}_3 \times \left[ \omega \hat{E}_3 \times \left\{ \left( \frac{b}{2} + \frac{b}{2} \sin \theta \right) \hat{E}_2 - \frac{b}{2} \cos \theta \hat{E}_3 \right\} \right] \\ &= \omega \hat{E}_3 \times \left[ -\frac{\omega b}{2} (1 + \sin \theta) \hat{E}_1 \right] \end{aligned}$$

$$\text{Thus, } \underline{a}^G = -\frac{\omega^2 b}{2} (1 + \sin \theta) \hat{E}_2$$

Thus equating LHS & RHS we get

$$F_1 \hat{E}_1 + F_2 \hat{E}_2 + (F_3 - mg) \hat{E}_3 = -\frac{m \omega^2 b}{2} (1 + \sin \theta) \hat{E}_2$$



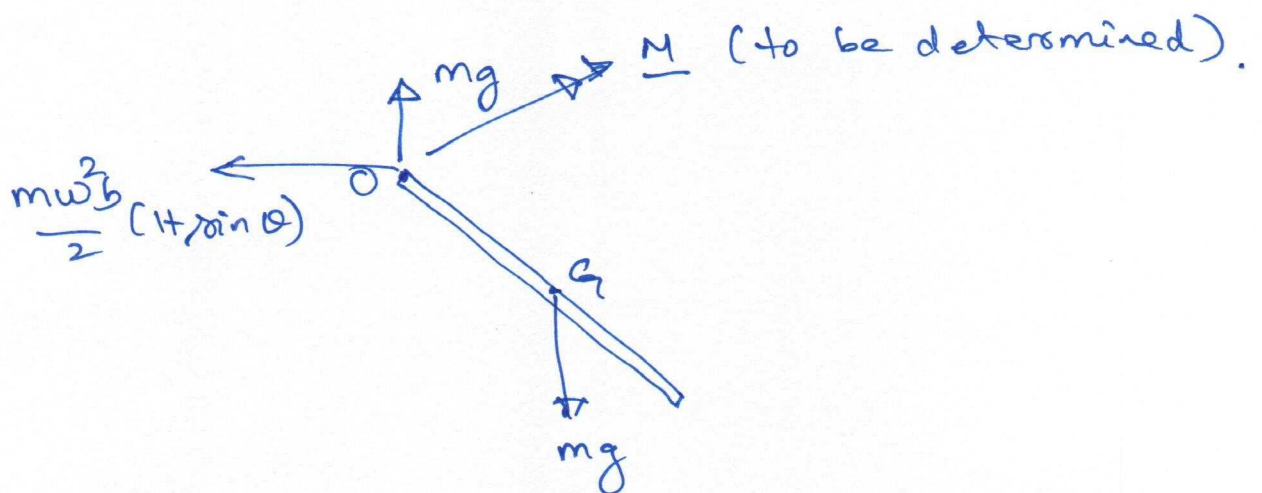
Thus

(3)

$$F_1 = 0$$

$$F_2 = -\frac{m\omega^2 b}{2} (1 + \sin \theta)$$

$$F_3 = mg.$$



Considering the AMB equation.

Considering AMB about P

$$\sum \underline{M}^P = \underline{r}^{G/P} \times m \underline{a}^G + \underline{\omega}^B \times (\underline{I}_G \cdot \underline{\omega}^B) + \underline{I}_G \cdot \underline{\alpha}^B$$

we have already established the following.

$$\underline{r}^{G/P} = \frac{b}{2} (1 + \sin \theta) \hat{E}_2 - \frac{b}{2} \cos \theta \hat{E}_3$$

$$\underline{a}^G = -\frac{\omega^2 b}{2} (1 + \sin \theta) \hat{E}_2$$

$$\underline{\omega}^B = \omega \hat{E}_3$$

$$\underline{\alpha}^B = 0$$

we need  $\underline{I}_G$

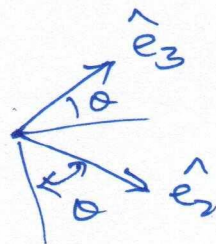
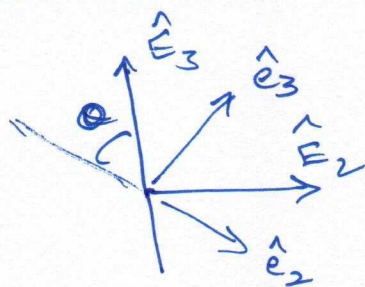
(4)

It is convenient to determine  $\underline{I}_A$  in the BFCS  $\underline{E}_P$  where the axes are along the principal axes of the inertial tensor.

Using established moments of inertia of a plate about its axes of symmetry, we can say

$$[\underline{I}_A]_{\underline{E}_P} = \begin{bmatrix} \frac{mb^2}{12} & 0 & 0 \\ 0 & \frac{mb^2}{12} & 0 \\ 0 & 0 & \frac{mb^2}{6} \end{bmatrix}$$

To find  $\underline{I}_A \cdot \underline{\omega}_B$ , it thus helps to have  $\underline{\omega}_B$  also in the BFCS  $\underline{E}_P$



$$\underline{\omega}_B = \omega \hat{E}_3$$

$$= \omega [-\cos\theta \hat{e}_2 + \sin\theta \hat{e}_3]$$

$$\therefore [\underline{I}_A \cdot \underline{\omega}_B]_{\underline{E}_P} = \begin{bmatrix} \frac{mb^2}{12} & 0 & 0 \\ 0 & \frac{mb^2}{12} & 0 \\ 0 & 0 & \frac{mb^2}{6} \end{bmatrix} \begin{bmatrix} 0 \\ -\omega \cos\theta \\ \omega \sin\theta \end{bmatrix}$$



(5)

$$\therefore [\underline{I}_A \cdot \underline{\omega}_B]_F = \begin{bmatrix} 0 \\ -\frac{mb^2\omega}{12} \cos\alpha \\ \frac{mb^2\omega}{6} \sin\alpha \end{bmatrix}$$

$$\text{i.e. } \underline{I}_A \cdot \underline{\omega}_B = -\frac{mb^2\omega}{12} \cos\alpha \hat{e}_2 + \frac{mb^2\omega}{6} \sin\alpha \hat{e}_3$$

Now to find  $\underline{\omega}_B \times (\underline{I}_A \cdot \underline{\omega}_B)$

$$\text{Again using } \underline{\omega}_B = -\omega \cos\alpha \hat{e}_2 + \omega \sin\alpha \hat{e}_3$$

$$\underline{\omega}_B \times (\underline{I}_A \cdot \underline{\omega}_B) = \begin{vmatrix} \hat{e}_1 & \hat{e}_2 & \hat{e}_3 \\ 0 & -\omega \cos\alpha & \omega \sin\alpha \\ 0 & -\frac{mb^2\omega}{12} \cos\alpha & \frac{mb^2\omega}{6} \sin\alpha \end{vmatrix}$$

$$= \left[ -\frac{mb^2\omega^2}{6} \sin\alpha \cos\alpha + \frac{mb^2\omega^2}{12} \sin\alpha \cos\alpha \right] \hat{e}_1$$

$$= -\frac{mb^2\omega^2}{12} \sin\alpha \cos\alpha \hat{e}_1$$

At the instant of interest  $\hat{e}_1$  is aligned to  $\hat{E}_1$

$$\therefore \underline{\omega}_B \times (\underline{I}_A \cdot \underline{\omega}_B) = -\frac{mb^2\omega^2}{12} \sin\alpha \cos\alpha \hat{E}_1$$

←  
2<sup>nd</sup> term of RHS.

To find the 1<sup>st</sup> term of RHS

(6)

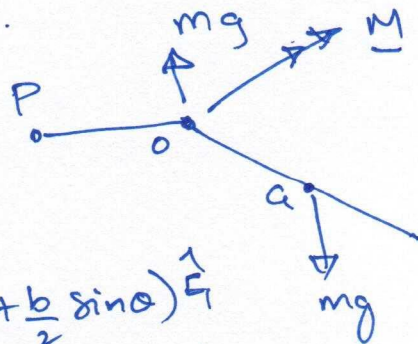
$$\underline{\tau}^{A/P} \times m \underline{a}^A = \begin{vmatrix} \hat{E}_1 & \hat{E}_2 & \hat{E}_3 \\ 0 & \frac{b}{2}(1+\sin\alpha) & -\frac{b}{2}\cos\alpha \\ 0 & -\frac{m\omega^2 b}{2}(1+\sin\alpha) & 0 \end{vmatrix}$$

$$= -\frac{m\omega^2 b^2}{4} (1+\sin\alpha)\cos\alpha \hat{E}_1$$

$$\therefore \text{RHS} \equiv -\frac{m\omega^2 b^2}{4} (1+\sin\alpha)\cos\alpha \hat{E}_1 - \frac{mb^2\omega^2}{12} \sin\alpha\cos\alpha \hat{E}_1$$

$$= \left[ -\frac{m\omega^2 b^2}{4} \cos\alpha - \frac{m\omega^2 b^2}{3} \sin\alpha\cos\alpha \right] \hat{E}_1$$

LHS  $\equiv \sum \underline{M}^i$  about P



$$= mg \cdot \frac{b}{2} \cdot \hat{E}_1 - mg \left( \frac{b}{2} + \frac{b}{2} \sin\alpha \right) \hat{E}_1 + M_1 \hat{E}_1 + M_2 \hat{E}_2 + M_3 \hat{E}_3$$

(because hinge)

$$= -\frac{mg b}{2} \sin\alpha \hat{E}_1 + M_2 \hat{E}_2 + M_3 \hat{E}_3$$



Equating LHS & RHS.

(7)

$$-\frac{mgb}{2} \sin \alpha \hat{E}_1 + M_2 \hat{E}_2 + M_3 \hat{E}_3 = \left[ -\frac{m\omega^2 b^2}{4} \cos \alpha - \frac{m\omega^2 b^2}{3} \sin \alpha \cos \alpha \right] \hat{E}_1$$

$$\Rightarrow M_2 = 0 ; M_3 = 0$$

$$\text{and } -\frac{mgb}{2} \sin \alpha = -m\omega^2 b^2 \cos \alpha \left( \frac{1}{4} + \frac{\sin \alpha}{3} \right)$$

$$\Rightarrow \frac{6g \tan \alpha}{b(3+4 \sin \alpha)} = \omega^2$$

$$\therefore \omega = \sqrt{\frac{6g \tan \alpha}{b(3+4 \sin \alpha)}}$$