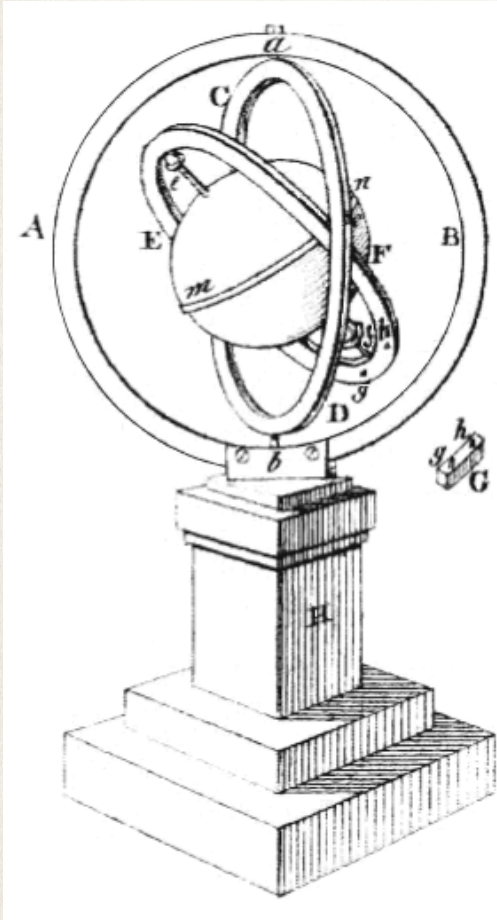


Lecture 13

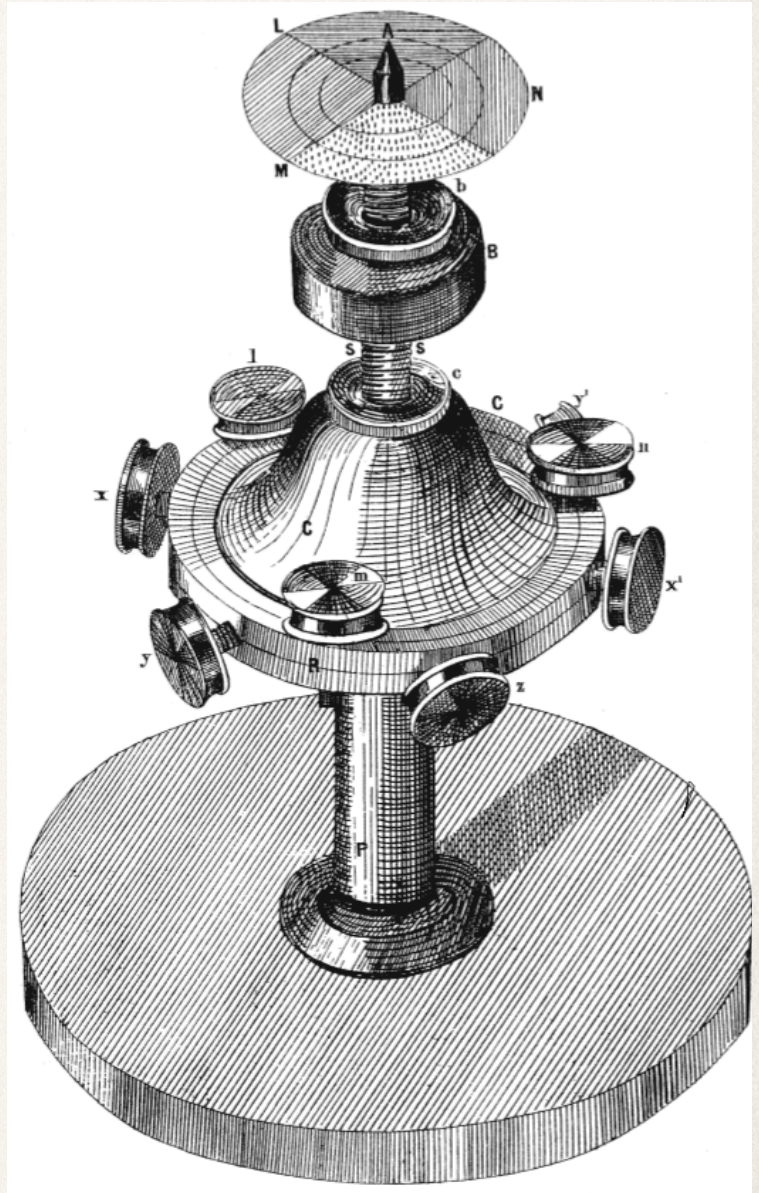
Rigid body kinetics: Rigid body motion; Kinetic quantities; Moment of inertia tensor.

22-28 September, 2021

Rigid body motion



Bohnenberger's machine to show Earth's rotation.



Maxwell's top: Visualises path of instantaneous rotation axis. [Further reading](#) ; [Video link](#) .

Rigid body motion

I. General rigid body motion.

1. A rigid body can *translate* and *rotate*.
2. To track *translation*: Follow location of a point on the rigid body.
3. To track *rotation*: Generally use an Euler angle sequence.

II. To find location of a point needs its velocity as function of time. To find rigid body's orientation need its angular velocity as a function of time.

III. To find velocity and angular velocity integrate, respectively, acceleration and angular acceleration.

IV. Accelerations and angular accelerations depend on applied forces and moments through the **laws of motion**.

Rigid body: Kinetic quantities

Laws of motion are in terms of the following :

I. *Mass*: $m = \int_V \rho(\mathbf{r}) dV$.

II. *Center of mass*: $\mathbf{r}^{G/O} = \frac{1}{m} \int_V \rho(\mathbf{r}) \mathbf{r} dV$.

III. *Linear momentum*: $\mathbf{p} = \int_V \rho(\mathbf{r}) \mathbf{v}(\mathbf{r}) dV = m \mathbf{v}^G$.

IV. *Angular momentum* about a point P :

$$\mathbf{h}^P = \int_V \mathbf{r}^{/P} \rho(\mathbf{r}) \mathbf{v}(\mathbf{r}) dV = \mathbf{r}^{G/P} \times m \mathbf{v}^G + \underbrace{\mathbf{I}^G \cdot \boldsymbol{\omega}^{\mathcal{B}}}_{\mathbf{h}^G},$$

where, $\mathbf{I}^G = \int_V \rho(\mathbf{r}) \left(|\mathbf{r}^{/G}|^2 \mathbf{1} - \mathbf{r}^{/G} \otimes \mathbf{r}^{/G} \right) dV$

is the *Moment of Inertia* about G of the body.

V. *Kinetic energy*:
$$E_K = \frac{1}{2} \int_V \rho(\mathbf{r}) |\mathbf{v}(\mathbf{r})|^2 dV$$
$$= \frac{1}{2} m |\mathbf{v}^G|^2 + \frac{1}{2} \boldsymbol{\omega}^{\mathcal{B}} \cdot \mathbf{I}^G \cdot \boldsymbol{\omega}^{\mathcal{B}}$$

Moment of Inertia

I. **Definition.** *Moment of Inertia Tensor* about a point P of a rigid body is given by

$$\mathbf{I}^P = \int_V \rho(\mathbf{r}) \left(|\mathbf{r}^{/P}|^2 \mathbf{1} - \mathbf{r}^{/P} \otimes \mathbf{r}^{/P} \right) dV.$$

1. *Symmetric, positive definite* tensor.
2. Three *real, positive* principal values.
3. Three *orthogonal* principal axes which define the rigid body's *principal CS*.
4. Moment of inertia changes with P .
5. Estimates mass distribution *w.r.t.* P .

II. In CS $\{\mathcal{E}, P, \hat{\mathbf{E}}_i\}$, let $\mathbf{I}^P = I_{ij}^P \hat{\mathbf{E}}_i \otimes \hat{\mathbf{E}}_j$, then

$$I_{11}^P = \int_V \rho(\mathbf{r}) (X_2^2 + X_3^2) dV, \quad I_{22}^P = \int_V \rho(\mathbf{r}) (X_3^2 + X_1^2) dV$$

$$I_{33}^P = \int_V \rho(\mathbf{r}) (X_2^2 + X_1^2) dV, \quad I_{12}^P = I_{21}^P = - \int_V \rho(\mathbf{r}) X_1 X_2 dV,$$

$$I_{23}^P = I_{32}^P = - \int_V \rho(\mathbf{r}) X_2 X_3 dV, \quad I_{31}^P = I_{13}^P = - \int_V \rho(\mathbf{r}) X_3 X_1 dV.$$

Moment of Inertia

I. **Parallel axes theorem.** Let CM be at G:

$$\mathbf{I}^P = \mathbf{I}^G + m \left(\left| \mathbf{r}^{G/P} \right|^2 \mathbf{1} - \mathbf{r}^{G/P} \otimes \mathbf{r}^{G/P} \right).$$

II. In CS $\{\mathcal{E}, P, \hat{\mathbf{E}}_i\}$, let $\mathbf{r}^{P/G} = d\hat{\mathbf{n}}^{GP} = dn_i^{GP}\hat{\mathbf{E}}_i$:

$$I_{kk}^P = I_{kk}^G + md^2 n_k^{GP} n_k^{GP} \quad (\text{no sum on } k),$$

$$I_{ij}^P = I_{ij}^G - md^2 n_i^{GP} n_j^{GP} \quad (i \neq j).$$

III. **Application.** If a rigid body has a *fixed* point C, i.e. $\mathbf{v}^C = \mathbf{0}$, then

$$\mathbf{h}^C = \mathbf{I}^C \cdot \boldsymbol{\omega}^{\mathcal{B}}.$$

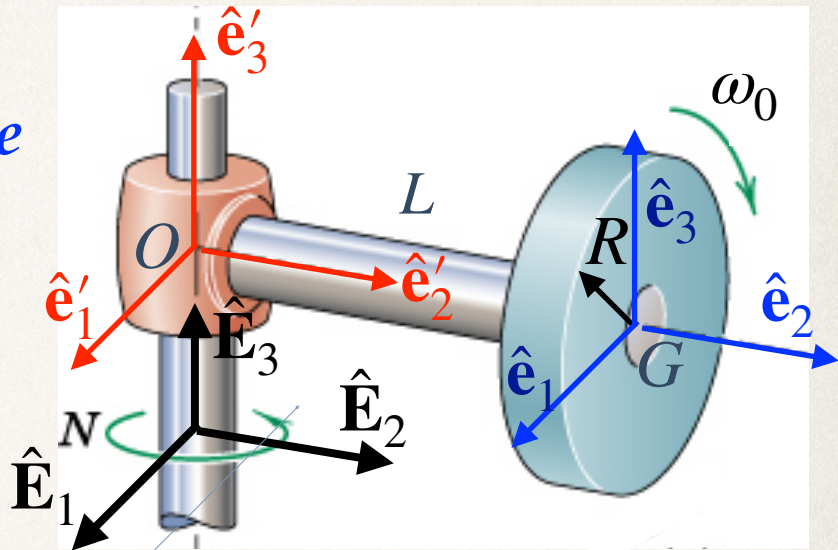
IV. **Perpendicular axes theorem.** If body is planar with normal $\hat{\mathbf{n}}$, then

$$\mathbf{I}^G = I_n^G \hat{\mathbf{n}} \otimes \hat{\mathbf{n}} + I_{a_1}^G \hat{\mathbf{a}}_1 \otimes \hat{\mathbf{a}}_1 + I_{a_2}^G \hat{\mathbf{a}}_2 \otimes \hat{\mathbf{a}}_2,$$

with $I_n^G = I_{a_1}^G + I_{a_2}^G$, and $\{I_n^G, \hat{\mathbf{n}}\}$ and $\{I_{a_i}^G, \hat{\mathbf{a}}_i\}$ being the *principal pairs* of \mathbf{I}^G .

Example 1

Find the angular momentum of the system about G and O . The arm OG is massless.



$$\text{I. } \{ \mathcal{E}_0, \hat{\mathbf{E}}_i \} \xrightarrow{R(\hat{\mathbf{E}}_3, \varphi_N)} \{ \mathcal{E}', \hat{\mathbf{e}}'_i \} \xrightarrow{R(\hat{\mathbf{e}}'_2, \varphi_{\omega_0})} \{ \mathcal{E}, \hat{\mathbf{e}}_i \}$$

$$\text{II. } \mathbf{h}_{\text{sys}}^G = \mathbf{h}_{\text{disk}}^G + \cancel{\mathbf{h}_{\text{arm}}^G}^0 = \mathbf{I}_{\text{disk}}^G \cdot \boldsymbol{\omega}^{\text{disk}}$$

$$\text{III. } [\mathbf{I}_{\text{disk}}^G]_{\mathcal{E}} = \frac{mR^2}{4} \begin{pmatrix} 1 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

$$\text{IV. } \mathbf{h}_{\text{sys}}^G = mR^2(-2\omega_0\hat{\mathbf{E}}_2 + N\hat{\mathbf{E}}_3)/4$$

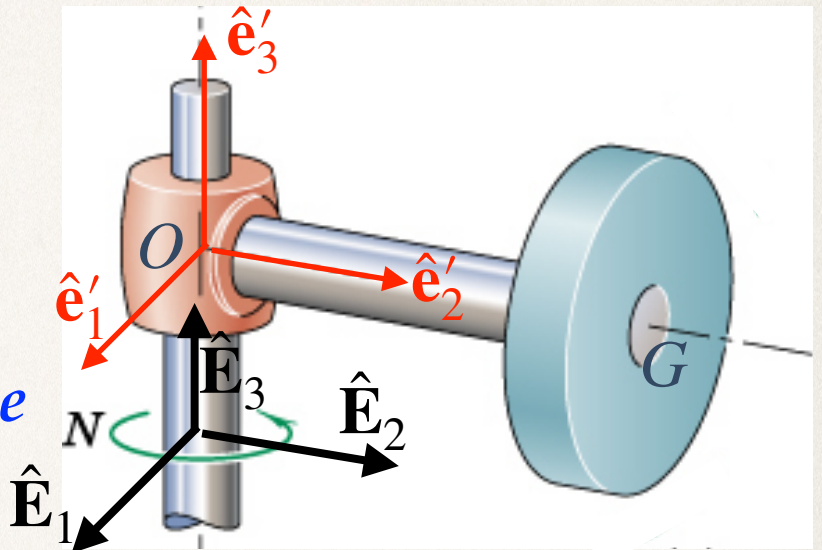
$$\text{V. } \mathbf{h}_{\text{disk}}^O = \mathbf{r}^{G/O} \times m\mathbf{v}^G + \mathbf{h}_{\text{disk}}^G$$

$$\text{VI. } \mathbf{h}_{\text{sys}}^O = -\frac{m\omega_0 R^2}{2}\hat{\mathbf{E}}_2 + \frac{mN(R^2 + 4L^2)}{4}\hat{\mathbf{E}}_3$$

Example 2

The disc is now welded to the massless arm.

Find the angular momentum of the system about G and O .



$$\text{I. } \{ \mathcal{E}_0, \hat{\mathbf{E}}_i \} \xrightarrow{R(\hat{\mathbf{E}}_3, \varphi_N)} \{ \mathcal{E}', \hat{\mathbf{e}}'_i \}$$

$$\text{II. } \mathbf{h}_{\text{sys}}^G = \mathbf{I}_{\text{sys}}^G \cdot \boldsymbol{\omega}^{\text{sys}} = mR^2 N \hat{\mathbf{E}}_3 / 4$$

$$\text{III. } \mathbf{h}_{\text{sys}}^O = \mathbf{I}_{\text{sys}}^O \cdot \boldsymbol{\omega}^{\text{sys}}$$

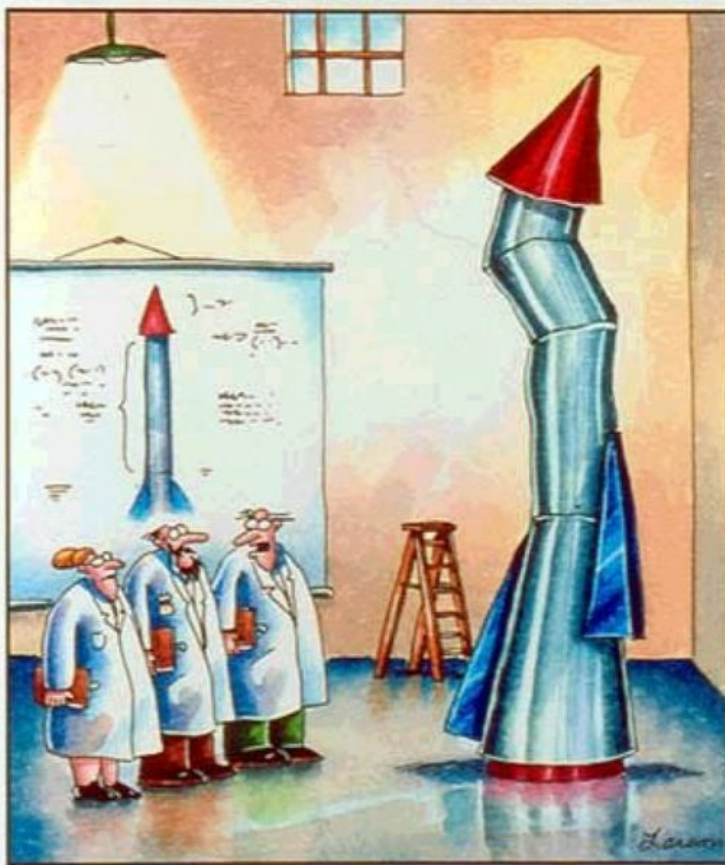
$$\text{IV. } \mathbf{I}^O = \mathbf{I}^G + m \left(\left| \mathbf{r}^{G/O} \right|^2 \mathbf{1} - \mathbf{r}^{G/O} \otimes \mathbf{r}^{G/O} \right)$$

$$[\mathbf{I}_{\text{disk}}^O]_{\mathcal{E}'} = \frac{m}{4} \begin{pmatrix} R^2 + 4L^2 & 0 & 0 \\ 0 & 2R^2 & 0 \\ 0 & 0 & R^2 + 4L^2 \end{pmatrix}$$

$$\text{V. } \mathbf{h}_{\text{sys}}^O = \frac{mN(R^2 + 4L^2)}{4} \hat{\mathbf{E}}_3$$

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by GARY LARSON



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"It's time we face reality, my friends...
We're not exactly rocket scientists."