

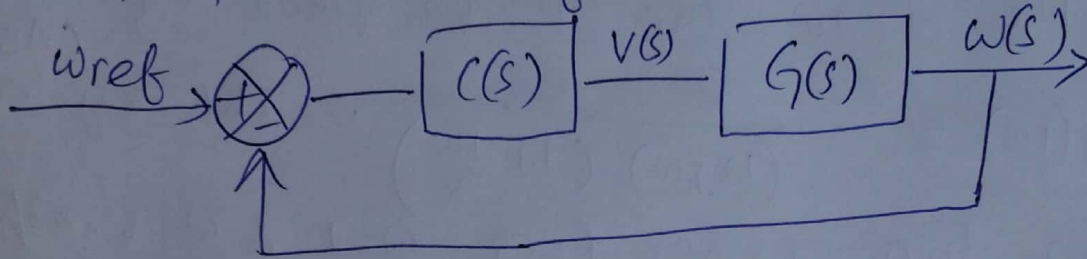
# Tutorial sheet 8

EE 250

8 April 2021

Q1

A certain DC Motor has the transfer function  $G(s) = \frac{\omega(s)}{V(s)} = \frac{10}{(s+1)(s+5)}$ .  
~~The~~ A speed controller is to be designed so that  $\omega(t)$  follows  $\omega_{ref}$  as given:



Design a lag compensator that would ensure  $70^\circ$  of phase margin. What is your observation from the unit step response of the compensated system? Redesign the compensator for  $PM = 50^\circ$ . Comment on the speed of response.

Solution Without the compensator

$$\frac{\omega(s)}{\omega_{ref}(s)} = \frac{G(s)}{1+G(s)} = \frac{10}{s^2 + 6s + 5}$$

$$\omega_n = \sqrt{5} = 3.87 \text{ rad/sec.}$$

$$\zeta = 6/2\omega_n = 0.774.$$

$$t_s = \frac{4}{\zeta\omega_n} = 1.33 \text{ sec.}$$

The system has fast response.

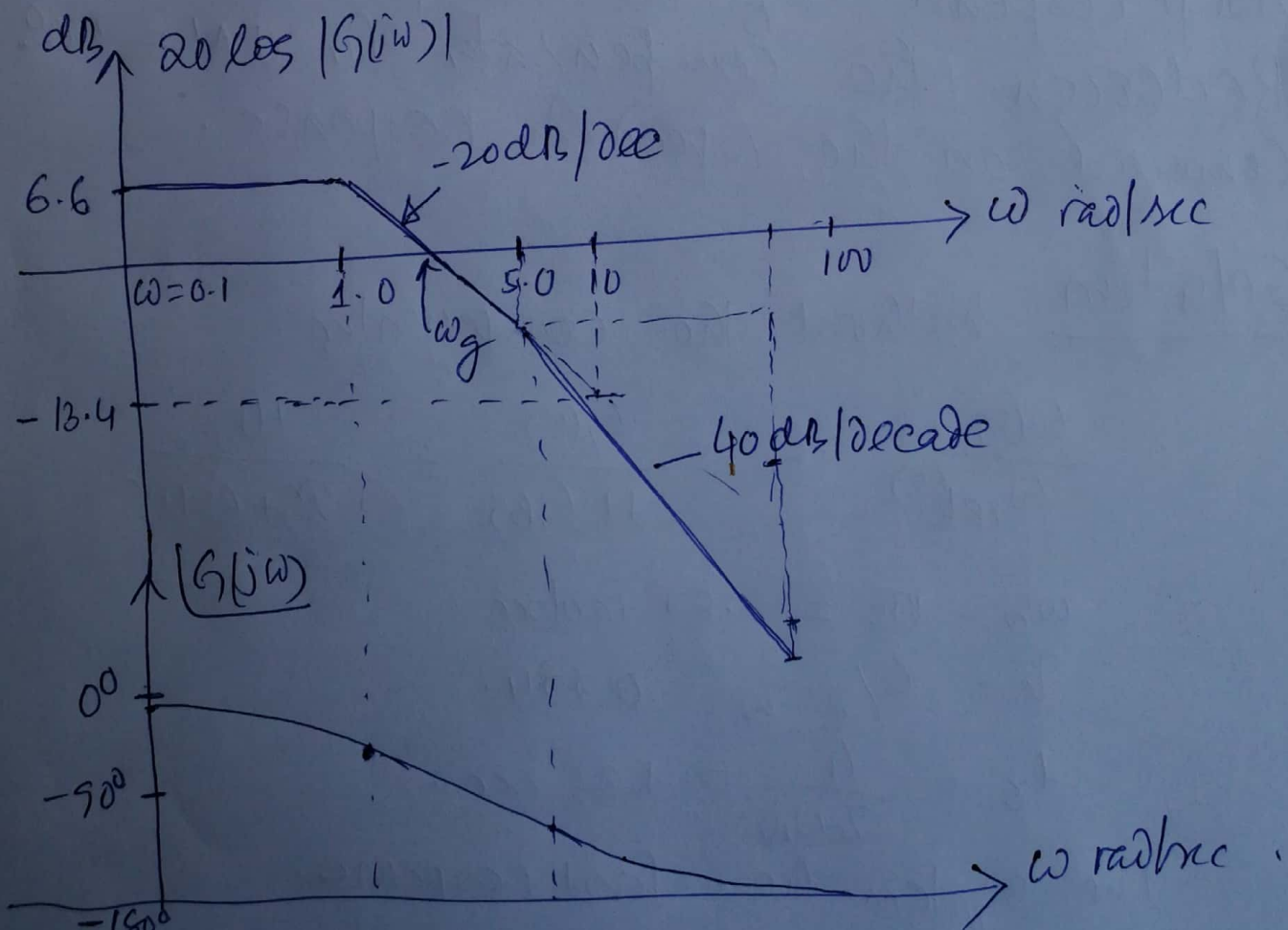
But DC gain =  $\frac{10}{15}$   
 Steady state error =  $\left(1 - \frac{10}{15}\right) 100\%$   
 $= 33.3\%$

Thus we need a lag compensator to improve the steady state response.

Let's draw the Bode plot of  $G(s)$  (Asymptotic)

$G(j\omega) = \frac{2}{(1+j\omega)(1+\frac{j\omega}{5})}$ . The system has two corner freq  $\omega=1$  &  $\omega=5$  rad/sec

$\angle G(j\omega) = -\tan^{-1}\omega - \tan^{-1}\omega/5$   
 $\omega=0.1 \quad \omega=1 \quad \omega=5 \quad \omega=\infty$   
 $\angle G(j\omega) \quad -6.8^\circ \quad -56.3^\circ \quad -123.7^\circ \quad -180^\circ$





For the Bode plot

$$\text{At } \omega_g, \frac{2}{\sqrt{\omega^2+1} \sqrt{1+\frac{\omega^2}{25}}} = 1$$

$$(\omega^2+1)(1+\frac{\omega^2}{25}) = 4$$

$$\omega^4 + 26\omega^2 - 75 = 0$$

$$x^2 + 26x - 75 = 0 \Rightarrow x = 2.62 \leftarrow \omega^2$$

$$\omega = \sqrt{2.62} = 1.62 \text{ rad/sec.}$$

The uncompensated system has PM 10° at  $\omega_g = 1.62 \text{ rad/sec}$

For the lag compensator design, the general structure is:

$$C(s) = \alpha \frac{Ts+1}{\alpha Ts+1}, \quad \alpha > 1$$

As the compensator adds high gain ( $\alpha$ ) at steady state (low freq / zero freq), the PM is retained from the  $G(j\omega)$ .  
Find  $\omega$  for which  $|G(j\omega)| = -110^\circ$  such that  $\phi_m = 70^\circ$ .

$$-\tan^{-1} \omega - \tan^{-1} \frac{\omega}{5} = -110$$

$$\frac{\omega + \frac{\omega}{5}}{1 - \frac{\omega^2}{5}} = \tan 110 = -2.75$$

$$\frac{6\omega}{5 - \omega^2} = -2.75$$

$$2.75\omega^2 - 6\omega - 13.75 = 0$$

$$\omega = 3.58 \text{ rad/sec.}$$

$$\text{At this } \omega, |G(j\omega)| = \frac{10}{(3.58^2+1)^{1/2} (3.58^2+25)^{1/2}} = -7.2 \text{ dB.}$$

To ensure  $PM = 70^\circ$ ,  $\omega_g = \omega \Rightarrow$   
 a gain 7.2 dB must be added

$$20 \log k = 7.2$$

$$k = 10^{\frac{7.2}{20}} = 2.29.$$

$$C(s) = \alpha \frac{Ts+1}{\alpha Ts+1}$$

Let  $\alpha = 10$ . The higher corner freq  $\frac{1}{T}$  (that of zero) be placed at least a decade to the left of  $\omega_g$ .

$$\text{Let } \frac{1}{T} = 0.3 \text{ rad/sec.}$$

$$T = 3.33 \text{ sec.}$$

$$\alpha T = 33.3$$

$$C(s) = 10 \frac{3.33s+1}{33.3s+1}$$

Since we have added a gain of 2.29 to make  $\omega = 3.58 \text{ rad/sec}$  as  $\omega_g$ ,

$$C(s) = 10 \times 2.29 \frac{3.33s+1}{33.3s+1} \\ = 22.9 \frac{3.33s+1}{33.3s+1}$$

Using Matlab command `margin(C(s).GB)`, we can verify that the compensated system has  $PM = 65.5^\circ$  at  $3.58 \text{ rad/sec}$ . There is a slight reduction in required  $PM$  as  $C(s)$  has contributed negative phase at  $\omega_g$ .



This can be rectified by placing the corner freq  $\frac{1}{T}$  ~~over~~ left of  $0.3 \text{ rad/sec}$ .

$$T(s) = \frac{C(s)G(s)}{1+C(s)G(s)}$$

$$= \frac{762.6s + 229}{33.3s^3 + 200.8s^2 + 935.1s + 234}$$

from Matlab,  $\text{step}(T(s))$  shows that although the steady state error has been eliminated, the settling time has increased to 8 sec (too high as compared to 1.3 sec of the uncompensated system).

Speed of response is generally increased by increasing the bandwidth or  $\omega_g$ . This would imply that we reduce the gain margin. Let the revised gain margin is  $50^\circ$ .

$$\frac{6\omega}{5-\omega^2} = \tan 130^\circ = -1.19$$

$$1.19\omega^2 - 6\omega - 5.95 = 0 \Rightarrow \omega = 5.89 \text{ rad/sec}$$

$$|G(j\omega)|_{\omega=5.89} = 0.2166 = -13.28 \text{ dB}$$

The gain needs to be added 13.28 dB to make this  $\omega$  as  $\omega_g$ .

$$K = 10^{13.28/20} = 4.61$$

If we retain the same  $C(s)$ ,

$$C(s) = 4.61 \times 10 \frac{3.33s+1}{33.3s+1} = 46.1 \frac{3.33s+1}{33.3s+1}$$

From MATLAB,  $\text{margin}(C(s)G(s))$  show

$$\text{PM} = 47.3^\circ \text{ at } \omega_g = 5.89 \text{ rad/sec.}$$

$$T(s) = \frac{C(s)G(s)}{1 + C(s)G(s)}$$

$$= \frac{1535s + 461}{33.3s^3 + 200.8s^2 + 1708s + 466}$$

$\text{Step}(T(s))$  shows that the settling time is less than 3 sec.

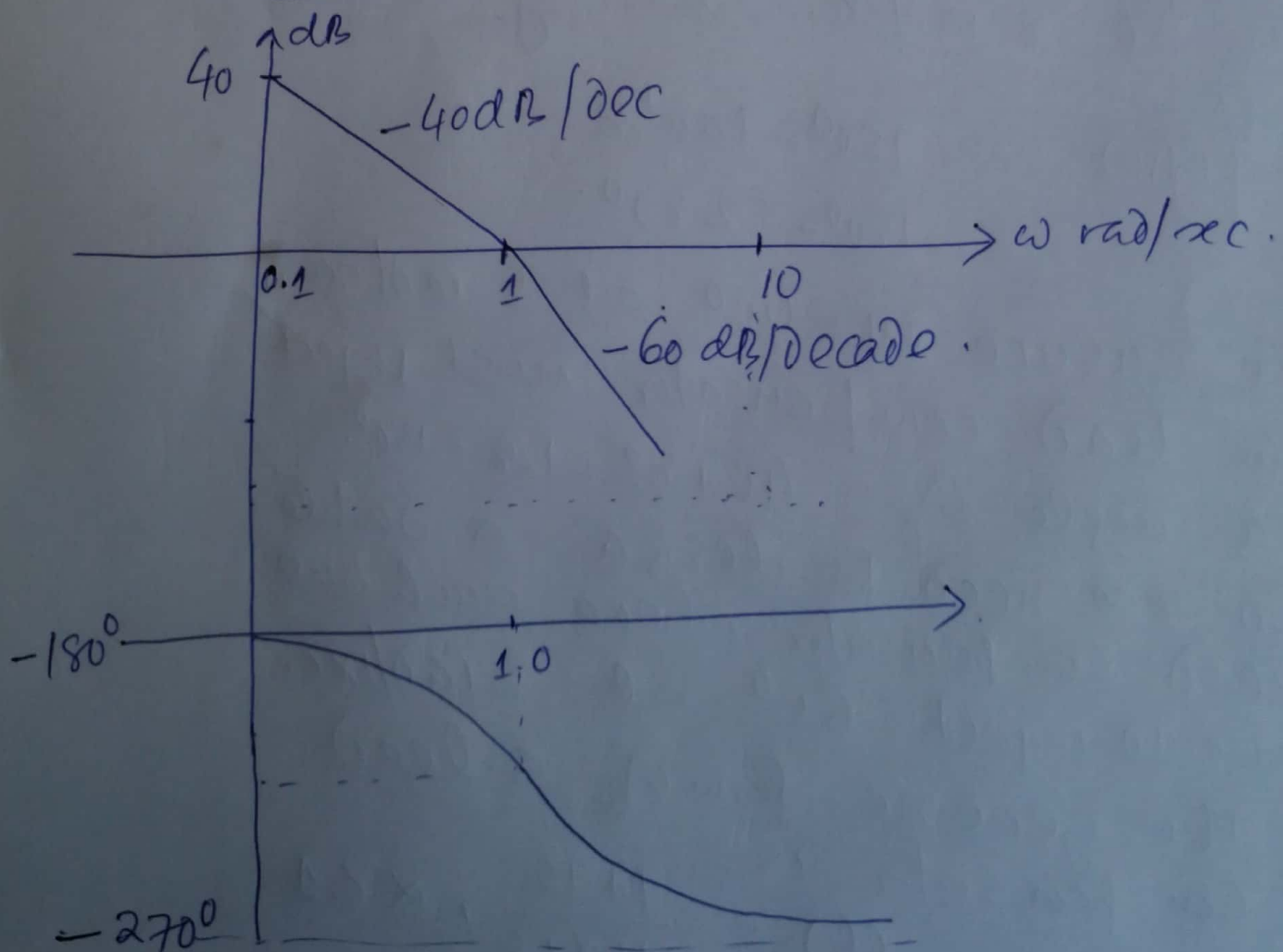
Q2 Design a lead compensator for a system in unity feedback configuration whose open loop transfer function is

$$G(s) = \frac{1}{s^2(s+1)} \quad \text{The}$$

compensator should ensure a phase margin of  $45^\circ$ .

Soln  $G(j\omega) = \frac{1}{(j\omega)^2(1+j\omega)}$

$$\angle G(j\omega) = -180^\circ - \tan^{-1}\omega$$





$$|G(j\omega)| = \frac{1}{\omega^2 (1+\omega^2)^{1/2}} = 1$$

$$\omega^4 (1+\omega^2) = 1$$

$$\omega = 0.8688 \text{ rad/sec}$$

$$\begin{aligned} \angle G(j\omega) &= -180^\circ - \tan^{-1} 0.8688 \\ &= -180^\circ - 41^\circ \end{aligned}$$

So the system has  $PM = -41^\circ$  at  $\omega_g = 0.8688 \text{ rad/sec}$ . (Please validate through MATLAB as well)

If we keep  $\omega_g = 0.87 \text{ rad/sec}$ , then the system response will be very sluggish. So let's fix  $\omega_g$  at  $8 \text{ rad/sec}$ . At  $\omega_g = 8 \text{ rad/sec}$ ,

$$\begin{aligned} \angle G(j\omega) &= -180^\circ - \tan^{-1} 8 \\ &= -180^\circ - 82.87^\circ \end{aligned}$$

To ensure  $PM = 45^\circ$  at  $8 \text{ rad/sec}$ , the lead compensator must inject a angle of  $45 + 83 = 128 \approx 130^\circ$ .

So ~~we~~ need to design a double lead compensator where each one would inject  $65^\circ$  at  $8 \text{ rad/sec}$ .

The generic form of a lead compensator is

$$C(s) = \frac{1+TS}{1+\alpha TS}, \quad \alpha < 1$$



where the max angle contribution takes place at

$$\frac{1}{\sqrt{\alpha} T} = 8$$

$$\alpha = \frac{1 - \sin \phi_m}{1 + \sin \phi_m}, \quad \phi_m = 65^\circ$$

$$\approx 0.05$$

$$\frac{1}{\sqrt{0.05} T} = 8 \Rightarrow T = 0.56$$

$$\text{Thus } C(s) = \frac{0.56s + 1}{0.028s + 1} \quad \leftarrow \frac{1 + TS}{1 + \alpha TS}$$

We have to design K such that

$$\left| K C(s) C(s) G(s) \right|_{\omega=8 \text{ rad/sec}} = 1$$

$$\left| \tilde{C}(j\omega) G(j\omega) \right| = \frac{((0.56\omega)^2 + 1)}{((0.028\omega)^2 + 1)} \left[ \frac{1}{\omega^2 (1 + \omega^2)^{1/2}} \right]_{\omega=8 \text{ rad/sec}}$$

$$= \frac{21.06}{1.0502} \times \frac{1}{8^2 \times 8.06}$$

$$= 0.0389$$

$$= -28.2 \text{ dB}$$

We have to add 28.2 dB to achieve  
 $\omega_g = 8 \text{ rad/sec}$ .

$$20 \log k = 28.2$$

$$k = 10^{28.2/20}$$

$$= 25.7$$

Thus the compensator is

$$C_D(s) = 25.7 \left( \frac{0.56s+1}{0.028s+1} \right)^2$$

In MATLAB, one can verify that the phase margin of the compensated system is  $PM = 46.7^\circ$  at  $8 \text{ rad/sec}$ .

The closed loop transfer function is

$$T(s) = \frac{8.06s^2 + 28.78s + 25.7}{0.000784s^5 + 0.05678s^4 + 1.056s^3 + 9.06s^2 + 28.78s + 25.7}$$

The step response using MATLAB shows that  $t_s = 1 \text{ sec}$  with peak overshoot of 34%. If the desired peak overshoot is 20%, then we have to reduce  $\omega_g$  and redesign.