



university of
groningen

faculty of science
and engineering

Mechatronics

Week 2 Day 1



Introduction to modeling of dynamical systems



Motivation of Dynamical modeling

- **Analysis** - gain a deeper understanding of the underlying mechanisms and behavior of the system
- **Prediction** - forecast the future behavior of a system and anticipate outcomes
- **Control** - regulate system's behavior to achieve desired outcomes



Representation of Dynamical Systems



There are 3 ways to represents dynamical systems:

- Ordinary differential equations,
- Transfer functions,
- State-space equations,

which give us the relationship between the input and output of the system.



Representation of systems by differential equations

$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$

- u is the input
- y is the output
- a_1, \dots, a_{n-1}, a_n and $b_0, b_1, \dots, b_{n-1}, b_n$ are constants
- The left hand-side of the equation is related to the output of the system
- The right hand-side of the equation is related to the inputs of the system



$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$

The aim is to transform the differential equation into a transfer function, to do this we first have to take the Laplace transform:

$$s^n \hat{Y}(s) + a_1 s^{n-1} \hat{Y}(s) + \dots + a_{n-1} s \hat{Y}(s) + a_n \hat{Y}(s) = b_0 s^n \hat{U}(s) + b_1 s^{n-1} \hat{U}(s) + \dots + b_{n-1} s \hat{U}(s) + b_n \hat{U}(s)$$

The transfer function of the system is:

$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$

Zeros of the system

Characteristic equation → poles of the system



REMARK

$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$

$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{1 * s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$

1. Differential equation $\xrightarrow{\text{Laplace Transform}}$ Transfer Function

2. The coefficient of **highest degree term (s^n)** in the characteristic equation is **1**

$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$

The **transfer function** of the system is:

$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{\text{Zeros of the system}}{\text{Characteristic equation} \rightarrow \text{poles of the system}} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$

- The **left** hand-side of the equation is related to the **output** of the system and gives the **poles** of the system
- The **right** hand-side of the equation is related to the **inputs** of the system and it gives the **zeros** of the system



State-Space Representation of Dynamical Systems

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

- $x \in \mathbb{R}^n$, $u \in \mathbb{R}$, $y \in \mathbb{R}$, $A \in \mathbb{R}^{n \times n}$, $B \in \mathbb{R}^{n \times 1}$, $C \in \mathbb{R}^{1 \times n}$, and $D \in \mathbb{R}$
- $u \in \mathbb{R}$ and $y \in \mathbb{R}$ means that we are considering a single-input-single-output (SISO) system.
- The case of multi-input-multi-output (MIMO) systems will be treated later on in the course



$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

By taking the **Laplace transform** of the above state-space equation, we arrive at:

$$\begin{aligned}s\hat{X}(s) &= A\hat{X}(s) + B\hat{U}(s) \Rightarrow \hat{X}(s) = (sI - A)^{-1}B\hat{U}(s) \\ \hat{Y}(s) &= C\hat{X}(s) + D\hat{U}(s)\end{aligned}$$

Plugging $\hat{X}(s) = (sI - A)^{-1}B\hat{U}(s)$ in the equation above, we obtain:

$$\hat{Y}(s) = [C(sI - A)^{-1}B + D]\hat{U}(s)$$

$$\frac{\hat{Y}(s)}{\hat{U}(s)} = C(sI - A)^{-1}B + D$$

REMARK: State Space $\xrightarrow{\text{Laplace Transform}}$ Transfer Function



Transformation of State Equations

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

Original
system

$$z = Tx$$

$$\begin{aligned}\dot{z} &= T\dot{x} = TAx + TBu \\ &= TAT^{-1}z + TBu \\ y &= Cx + Du = CT^{-1}z + Du\end{aligned}$$

Transformed
system

where:

- $T \in \mathbb{R}^{n \times n}$ is an invertible transformation matrix
- $TAT^{-1} = \bar{A}$, $TB = \bar{B}$, $CT^{-1} = \bar{C}$, $D = \bar{D}$

The Transfer Function of the transformed state equation:

$$\bar{G}(s) = \bar{C}(sI - \bar{A})^{-1}\bar{B} + \bar{D} = CT^{-1}(sI - TAT^{-1})^{-1}TB + D = C(sI - A)^{-1}B + D$$

Original system

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

Transformed system

$$\begin{aligned}\dot{z} &= \bar{A}z + \bar{B}u \\ y &= \bar{C}z + \bar{D}u\end{aligned}$$

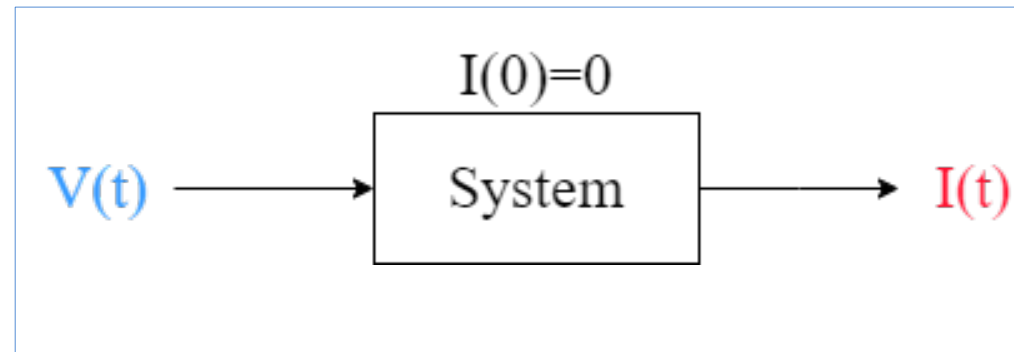
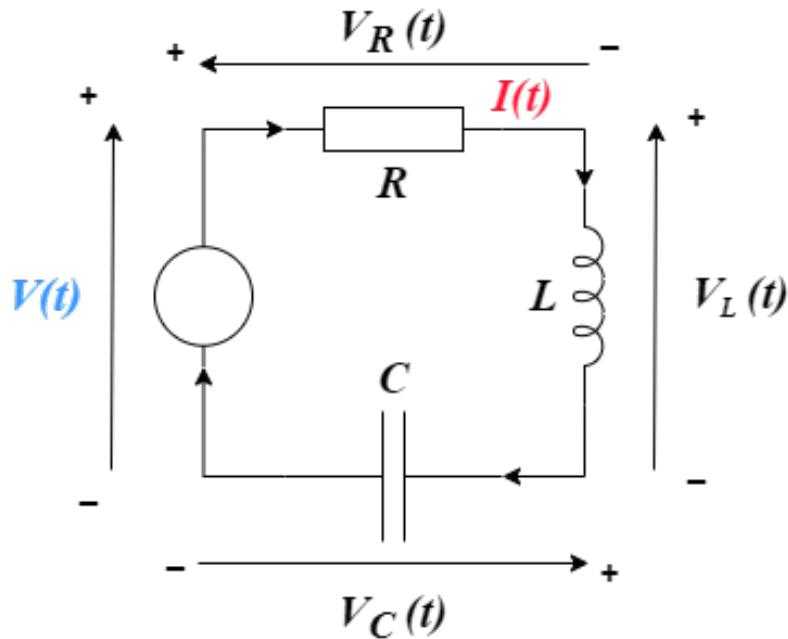
$C(sI - A)^{-1}B + D$
Same Transfer Function

REMARK:

- The transfer function remain the same, i.e., the state-space is not unique
- There are infinite possibilities to express a given transfer function in state-space form



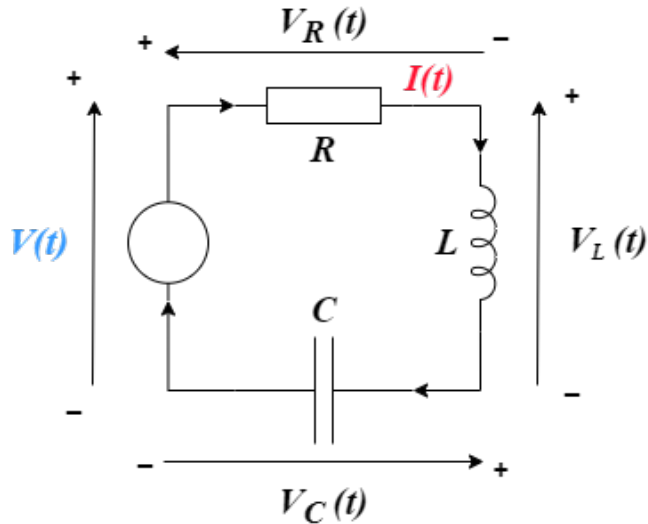
Example (Series RLC Circuit)



By **Kirchoff's voltage law**, the **sum** of voltages around the loop must be **zero**, i.e.,

$$V(t) - V_R(t) - V_L(t) - V_C(t) = 0 \quad (1).$$

$$V(t) - V_R(t) - V_L(t) - V_C(t) = 0 \quad (1)$$



Capacitor:

$$\frac{CdV_C(t)}{dt} = I(t) \Rightarrow V_C(t) = \frac{1}{C} \int_0^t I(t) dt \quad (2)$$

Inductor:

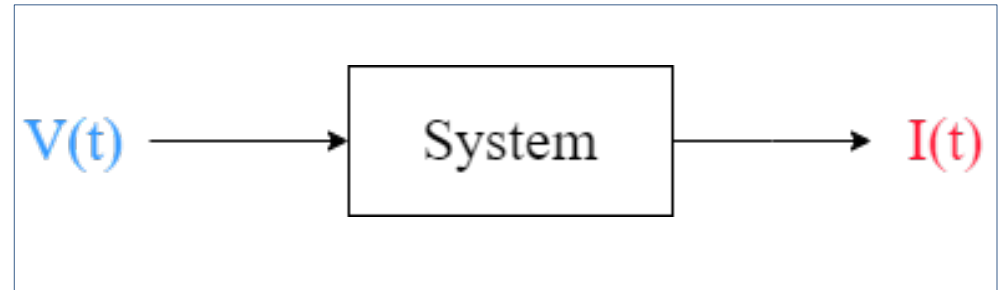
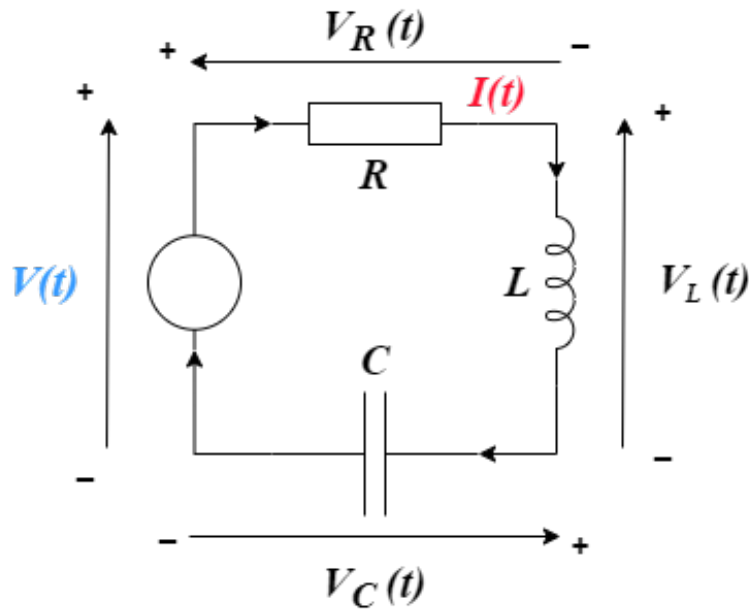
$$V_L(t) = L \frac{dI(t)}{dt} \quad (3)$$

Resistor:

$$V_R(t) = RI(t) \quad (4)$$

We obtain an **integral-differential** equation by plugging equations (2) through (4) in (1),

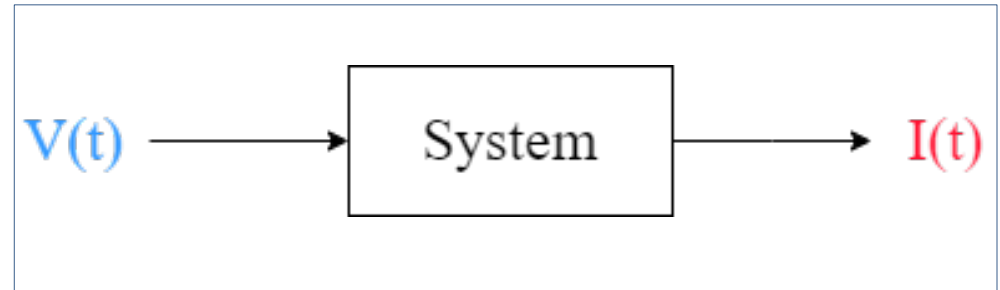
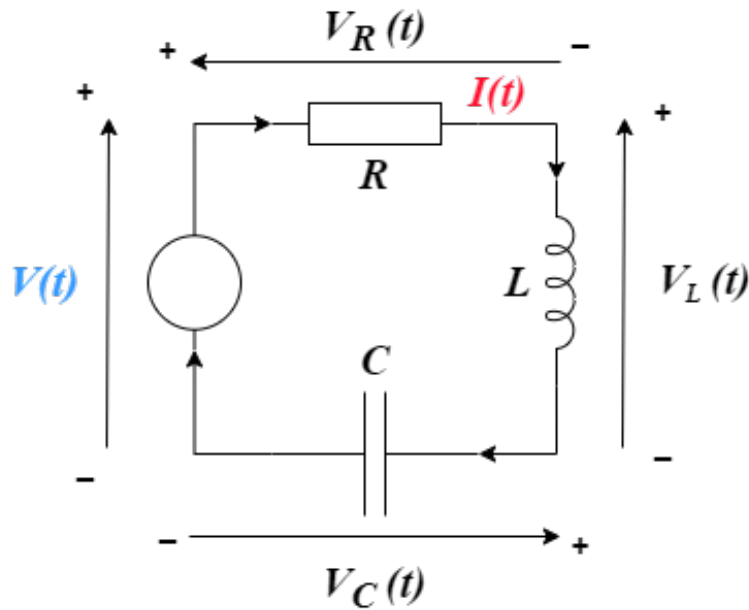
$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (5)$$



Consider the series RLC circuit model and its state space representation:

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0$$

- $x_1 = \int_0^t I(t) dt$
 - $x_2 = I(t)$
-
- $\dot{x}_1 = x_2$
 - $\dot{x}_2 = -\frac{1}{LC} x_1 - \frac{R}{L} x_2 + \frac{1}{L} V(t),$



Now consider different state variables :

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0$$

- $z_1 = -\frac{1}{LC} \int_0^t I(t) dt$
- $z_2 = I(t)$



- $\dot{z}_1 = -\frac{1}{LC} z_2$
- $\dot{z}_2 = z_1 - \frac{R}{L} z_2 + \frac{1}{L} V(t)$

State variables 1:

- $x_1 = \int_0^t I(t) dt$
 - $x_2 = I(t)$
1. $\dot{x}_1 = x_2$
 2. $\dot{x}_2 = -\frac{1}{LC}x_1 - \frac{R}{L}x_2 + \frac{1}{L}V(t)$

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{1}{LC} & -\frac{R}{L} \end{bmatrix}, B = \begin{bmatrix} 0 \\ \frac{1}{L} \end{bmatrix}$$

$$C = [0 \quad 1], D = 0$$

State variables 2:

- $z_1 = -\frac{1}{LC} \int_0^t I(t) dt$
 - $z_2 = I(t)$
1. $\dot{z}_1 = -\frac{1}{LC}z_2$
 2. $\dot{z}_2 = z_1 - \frac{R}{L}z_2 + \frac{1}{L}V(t)$

$$\bar{A} = \begin{bmatrix} 0 & -1/LC \\ 1 & -R/L \end{bmatrix}, \bar{B} = \begin{bmatrix} 0 \\ 1/L \end{bmatrix}$$

$$\bar{C} = [0 \quad 1], \bar{D} = 0$$

$$T = \begin{bmatrix} -\frac{1}{LC} & 0 \\ 0 & 1 \end{bmatrix}$$

$$G(s) = \frac{Cs}{1+RCs+LCs^2}$$

Different state space representations lead to the same transfer function, state-space is not **unique**!



Controller Canonical Form (CCF)

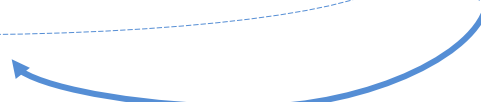
$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$

$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & \dots & -a_1 \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u$$

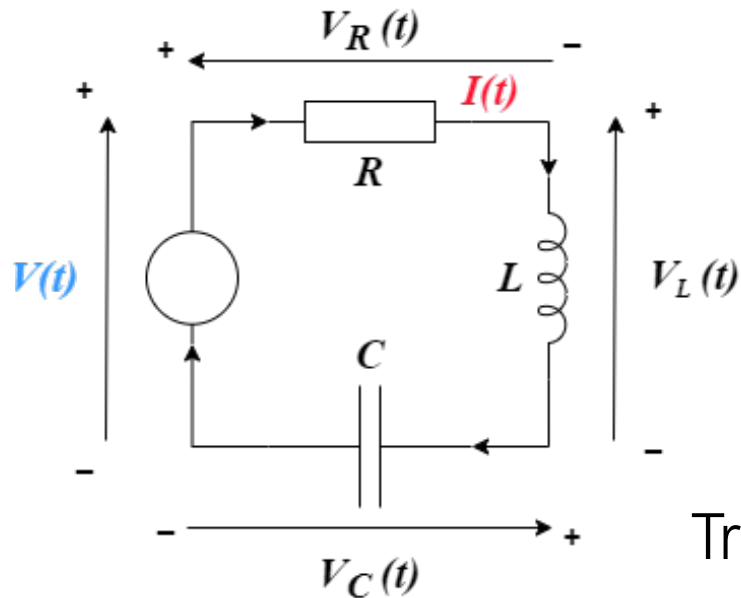
$$y = [b_n - a_n b_0 \quad b_{n-1} - a_{n-1} b_0 \quad \dots \quad b_2 - a_2 b_0 \quad b_1 - a_1 b_0] x + b_0 u$$

$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & \dots & -a_1 \end{bmatrix} \mathbf{x} + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u$$


- The control directly influences the characteristic equation as shown by the arrow
- The control can change the poles of your system, or in other words, the behavior of the system

Example



$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0$$

State variables

- $x_1 = \int_0^t I(t) dt$
- $x_2 = I(t)$
- $\dot{x}_1 = \bar{x}_2$
- $\dot{x}_2 = -\frac{1}{LC} \bar{x}_1 - \frac{R}{L} \bar{x}_2 + \frac{1}{L} V(t),$

Transfer function: $G(s) = \frac{Cs}{1 + RCs + LCs^2}$

State-space representation in controllable form:

GENERAL FORM

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & \cdots & -a_1 \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -1/LC & -R/L \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -1/LC & -R/L \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

controller: $u = -[k_1 \quad k_2] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + R$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -1/LC - k_1 & -R/L - k_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} R$$

Transfer function:

$$G(s) = \frac{\frac{1}{L}s}{s^2 + \underbrace{\left(\frac{R}{L} + k_2\right)}_{\text{damping}} s + \underbrace{\left(\frac{1}{LC} + k_1\right)}_{\text{stiffness}}}$$

By **controller canonical form** you can change the **poles** of the system directly to achieve desired **damping** and **stiffness**.



Observer Canonical Form (OCF)



$$\frac{d^n y}{dt^n} + a_1 \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_{n-1} \frac{dy}{dt} + a_n y = b_0 \frac{d^n u}{dt^n} + b_1 \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_{n-1} \frac{du}{dt} + b_n u$$



$$G(s) = \frac{\hat{Y}(s)}{\hat{U}(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n}$$



$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} 0 & 0 & \dots & 0 & -a_n \\ 1 & 0 & \dots & 0 & -a_{n-1} \\ 0 & 1 & \dots & 0 & -a_{n-2} \\ \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & \dots & 1 & -a_1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} b_n - a_n b_0 \\ b_{n-1} - a_{n-1} b_0 \\ b_{n-2} - a_{n-2} b_0 \\ \vdots \\ b_1 - a_1 b_0 \end{bmatrix} u$$
$$y = [0 \quad 0 \quad 0 \quad \dots \quad 1] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + b_0 u$$

Advantage: Looking at just one state, namely, x_n , you have knowledge on all the other states

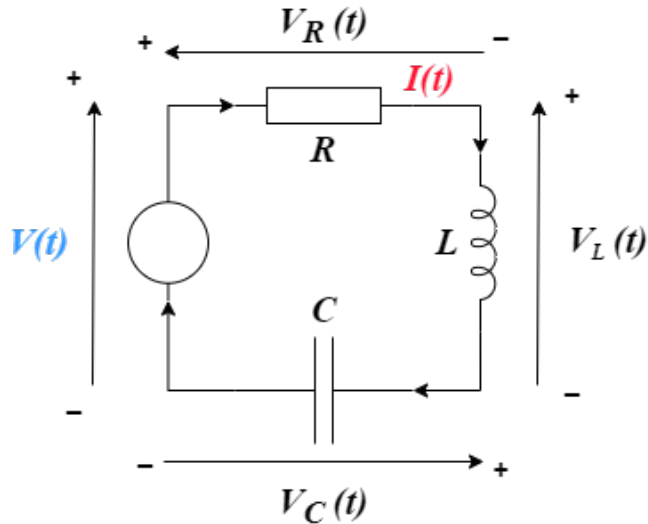


Remarks on Choosing State Variables



Example (Series RLC Circuit)

$$V(t) - V_R(t) - V_L(t) - V_C(t) = 0 \quad (1)$$



Capacitor:

$$\frac{CdV_C(t)}{dt} = I(t) \Rightarrow V_C(t) = \frac{1}{C} \int_0^t I(t) dt \quad (2)$$

Inductor:

$$V_L(t) = L \frac{dI(t)}{dt} \quad (3)$$

Resistor:

$$V_R(t) = RI(t) \quad (4)$$

We obtain an **integral-differential** equation by plugging equations (2) through (4) in (1),

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (5)$$

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (5)$$

We wish to convert (5) into a pure differential equation, therefore we take the derivative of (5):

$$\frac{dV(t)}{dt} - R \frac{dI(t)}{dt} - L \frac{d^2 I(t)}{dt^2} - \frac{1}{C} I(t) = 0 \quad (6)$$

Using the notation:

- $\dot{V}(t) = \frac{dV(t)}{dt}$
- $\dot{I}(t) = \frac{dI(t)}{dt}$
- $\ddot{I}(t) = \frac{d^2 I(t)}{dt^2}$, we have from (6) that

$$\dot{V}(t) - R\dot{I}(t) - L\ddot{I}(t) - \frac{1}{C}I(t) = 0 \quad (7)$$

Remark

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (*)$$

$$\dot{V}(t) - R \dot{I}(t) - L \ddot{I}(t) - \frac{1}{C} I(t) = 0 \quad (**)$$

State-space representation of (**):

$$\begin{aligned} \bullet \quad x_1(t) &= I(t) \\ \bullet \quad x_2(t) &= \dot{I}(t) \end{aligned} \quad \longrightarrow \quad \begin{aligned} \bullet \quad \dot{x}_1(t) &= \dot{I}(t) = x_2(t) \\ \bullet \quad \dot{x}_2(t) &= \ddot{I}(t) = \frac{1}{L} \dot{V}(t) - \frac{R}{L} \dot{I}(t) - \frac{1}{LC} I(t) \end{aligned}$$

$$\dot{x}_1(t) = x_2(t) \quad (8)$$

$$\dot{x}_2(t) = -\frac{1}{LC} x_1(t) - \frac{R}{L} x_2(t) + \frac{1}{L} \dot{V}(t), \quad (9)$$

where the last equation (9) is derived using (**)

$$\dot{V}(t) - R \dot{I}(t) - L \ddot{I}(t) - \frac{1}{C} I(t) = 0 \quad (**)$$

So, the **state-space representation** of (**) is given by

$$\dot{x}_1(t) = x_2(t)$$

$$\dot{x}_2(t) = -\frac{1}{LC} x_1(t) - \frac{R}{L} x_2(t) + \frac{1}{L} \dot{V}(t)$$

The state-space representation contains the derivative of the input $V(t)$, which is undesirable. This happened because of an unsuitable choice of state variables.

Remark

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (*)$$

$$\dot{V}(t) - R \dot{I}(t) - L \ddot{I}(t) - \frac{1}{C} I(t) = 0 \quad (**)$$

To avoid this issue, let us define a **different pair** of state-space variables, and use the **integral-differential equation**:

State-space representation of (*):

$$\begin{aligned} \bullet \quad \bar{x}_1(t) &= \int_0^t I(t) dt & \longrightarrow & \bullet \quad \dot{\bar{x}}_1(t) = I(t) = \bar{x}_2(t) \\ \bullet \quad \bar{x}_2(t) &= I(t) & & \bullet \quad \dot{\bar{x}}_2(t) = \frac{1}{L} V(t) - \frac{R}{L} I(t) - \frac{1}{LC} \int_0^t I(t) dt \end{aligned}$$

$$\dot{\bar{x}}_1(t) = \bar{x}_2(t) \quad (10)$$

$$\dot{\bar{x}}_2(t) = -\frac{1}{LC} \bar{x}_1(t) - \frac{R}{L} \bar{x}_2(t) + \frac{1}{L} V(t), \quad (11)$$

where the last equation (11) is derived using (*)

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (*)$$

$$\dot{V}(t) - R \dot{I}(t) - L \ddot{I}(t) - \frac{1}{C} I(t) = 0 \quad (**)$$

State-Space Representation of (**)

$$\begin{aligned} \dot{x}_1(t) &= x_2(t) \\ \dot{x}_2(t) &= -\frac{1}{LC} x_1(t) - \frac{R}{L} x_2(t) + \frac{1}{L} \dot{V}(t) \end{aligned}$$

with variables 1:

- $x_1(t) = I(t)$
- $x_2(t) = \dot{I}(t)$

State-Space Representation of (*)

$$\begin{aligned} \dot{\bar{x}}_1(t) &= \bar{x}_2(t) \\ \dot{\bar{x}}_2(t) &= -\frac{1}{LC} \bar{x}_1(t) - \frac{R}{L} \bar{x}_2(t) + \frac{1}{L} V(t) \end{aligned}$$

with variables 2:

- $\bar{x}_1(t) = \int_0^t I(t) dt$
- $\bar{x}_2(t) = I(t)$

$$V(t) - RI(t) - L \frac{dI(t)}{dt} - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (*)$$

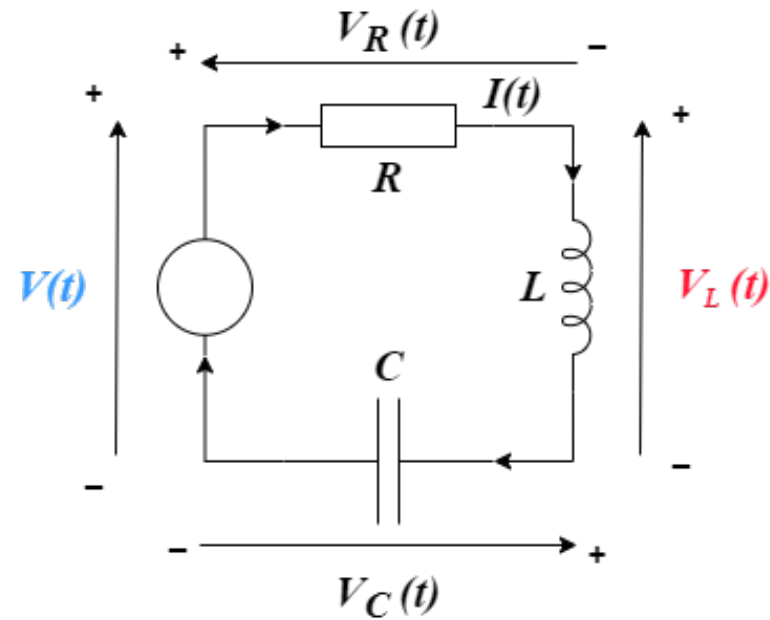
$$\dot{V}(t) - R \dot{I}(t) - L \ddot{I}(t) - \frac{1}{C} I(t) = 0 \quad (**)$$

The lesson to take home is:

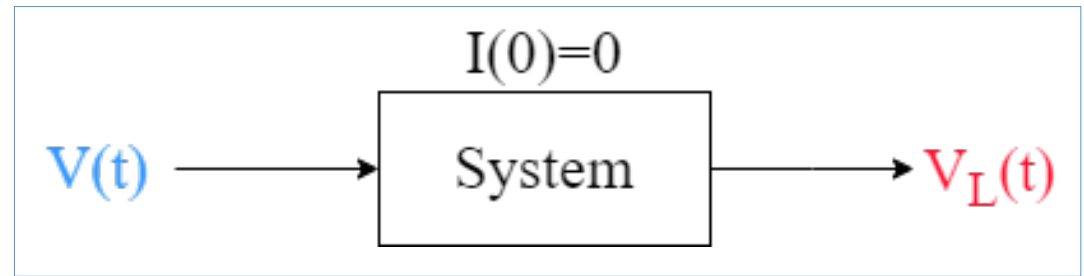
- Don't convert the integral-differential equation (*) into a pure differential one (**)
- Consider the integral-differential equation and choose the state-variables in a clever way to avoid appearance of **input derivatives** in the state-space representation



Another Example (Series RLC Circuit with Output as Voltage across Inductor)



Let us analyze the RLC Circuit using a different output, namely $V_L(t)$, instead of $I(t)$.

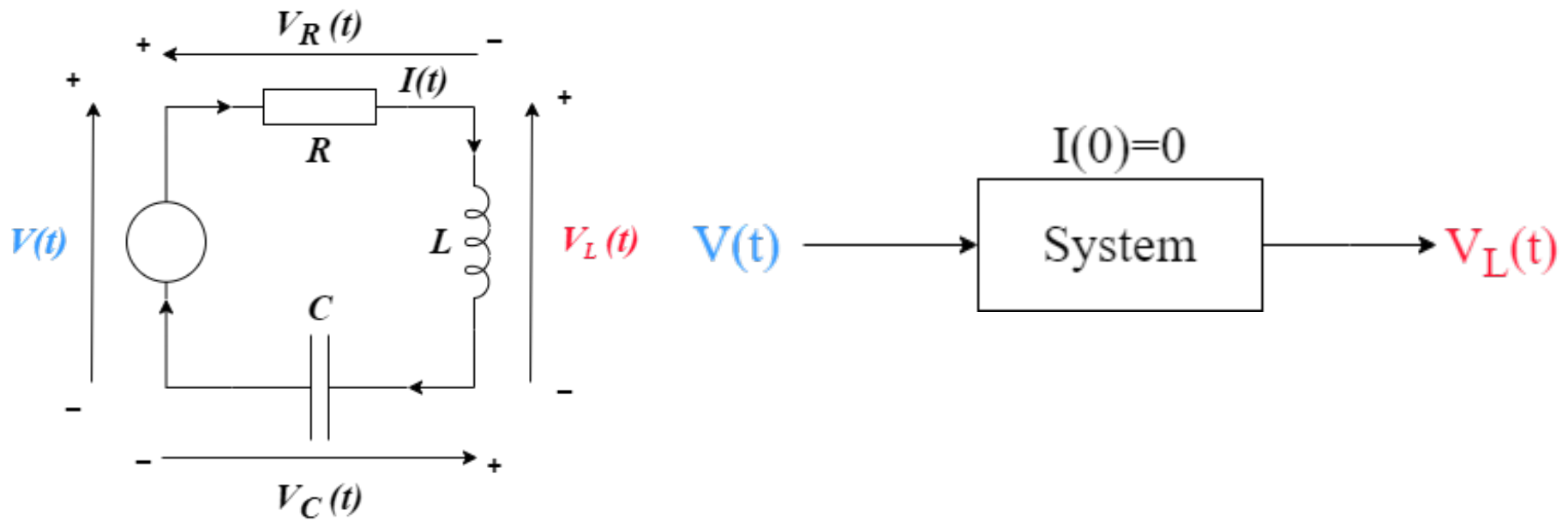


Once again, by KVL, i.e., [Kirchoff's voltage law](#), we have:

$$V(t) - RI(t) - V_L(t) - \frac{1}{C} \int_0^t I(t) dt = 0 \quad (12)$$

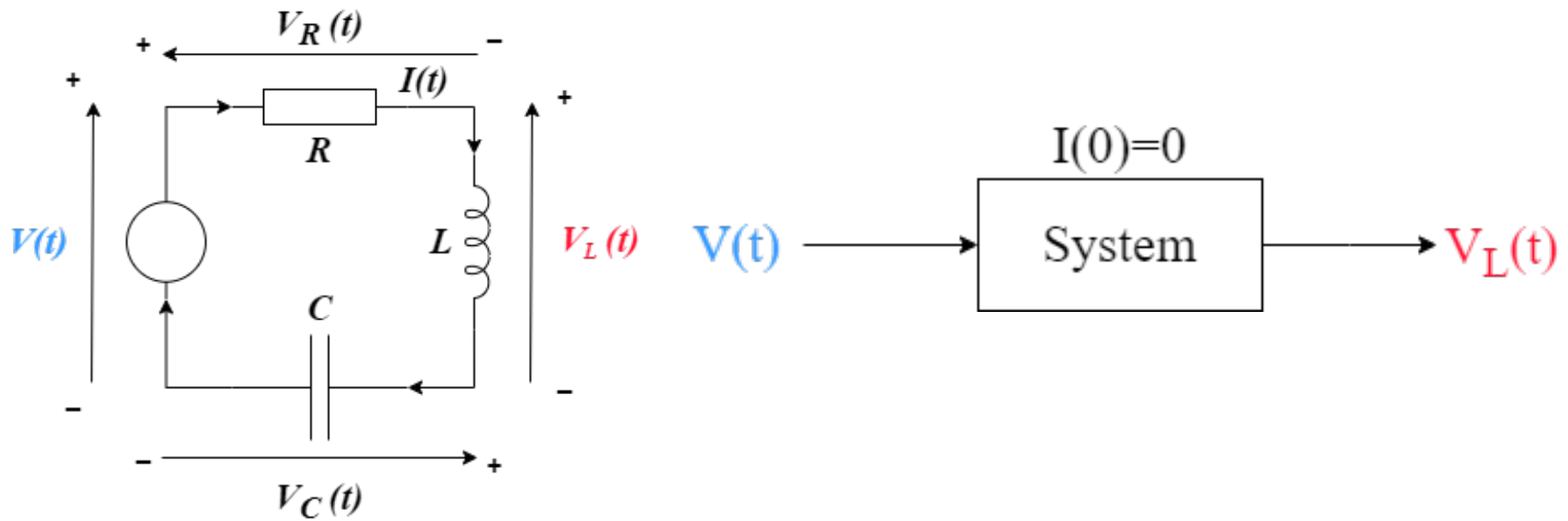
Replacing $V_L(t) = L \frac{dI(t)}{dt}$ or $I(t) = \frac{1}{L} \int_0^t V_L(t) dt$ in equation (12) to eliminate current $I(t)$:

$$V(t) - \frac{R}{L} \int_0^t V_L(t) dt - V_L(t) - \frac{1}{C} \int_0^t \int_0^t V_L(t) dt dt = 0. \quad (13)$$



$$V(t) - \frac{R}{L} \int_0^t V_L(t) dt - V_L(t) - \frac{1}{C} \int_0^t \int_0^t V_L(t) dt dt = 0. \quad (13)$$

How to avoid the appearance of the input derivatives in the state-space representation?



$$V(t) - \frac{R}{L} \int_0^t V_L(t) dt - V_L(t) - \frac{1}{C} \int_0^t \int_0^t V_L(t) dt dt = 0. \quad (13)$$

- Use the integral-differential equation without converting it to a pure differential equation
- Proper choice of the state-space variables

$$V(t) - \frac{R}{L} \int_0^t V_L(t) dt - V_L(t) - \frac{1}{C} \int_0^t \int_0^t V_L(t) dt dt = 0. \quad (13)$$

State-space representation of (13):

$$\begin{aligned} \bullet \quad x_1(t) &= \int_0^t \int_0^t V_L(t) dt dt \\ \bullet \quad x_2(t) &= \int_0^t V_L(t) dt \end{aligned} \quad \longrightarrow \quad \begin{aligned} \bullet \quad \dot{x}_1(t) &= \int_0^t V_L(t) dt = x_2(t) \\ \bullet \quad \dot{x}_2(t) &= V_L(t) = -\frac{R}{L} x_2(t) - \frac{1}{LC} x_1(t) + V(t) \end{aligned}$$

$$\dot{x}_1(t) = x_2(t) \quad (14)$$

$$\dot{x}_2(t) = -\frac{R}{L} x_2(t) - \frac{1}{LC} x_1(t) + V(t) \quad (15)$$

FINAL REMARK

- Traditionally, to get state-space representation for second-order differential equations, we usually assign the first state to be $V_L(t)$ (or $I(t)$) and second state to be $\dot{V}_L(t)$ (or $\dot{I}(t)$)
- However, when there are **zeros** in the system, using the described method leads to the appearance of the **derivatives of input** in state-space representation, which is undesirable
- Therefore, choose state-space variables cleverly!



Next lecture:

State-space representation and A , T , and
 D -type variables