

$$m\ddot{x} + c\dot{x} + kx = F(t)$$

$$\Rightarrow x(t) = x_h(t) + x_p(t) \quad : \text{total response}$$

where  $x_h(t)$  is the solution of the homogeneous eq.

$$m\ddot{x} + c\dot{x} + kx = 0$$

represents transient response

free vibration dies out with time

$x_p(t)$  is the particular solution

represents steady-state response

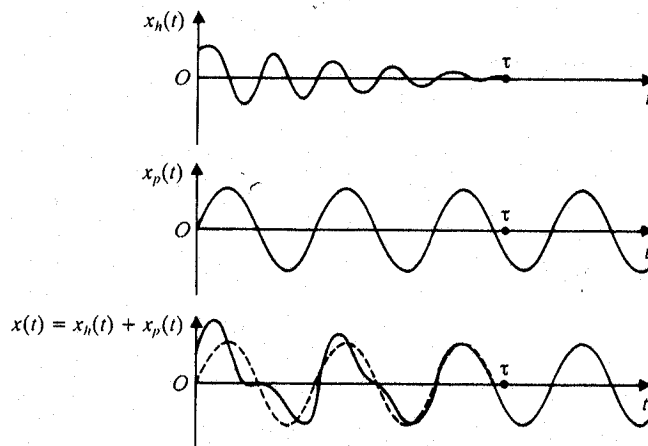
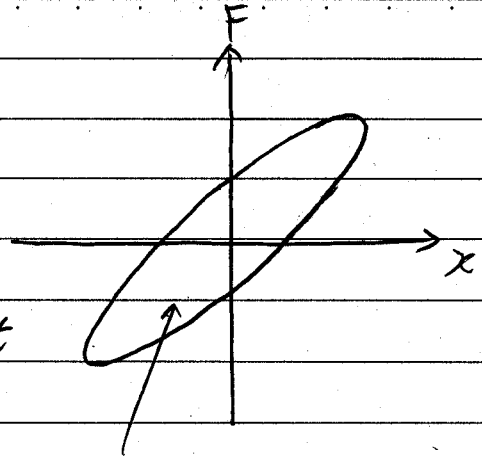


FIGURE 3.2 Homogenous, particular, and general solutions of Eq. (3.1) for an underdamped case.

- Energy Dissipated in Viscous Damping

$$E_d = \oint F dx = \int_0^{\frac{2\pi}{\omega}} F \dot{x} dt$$

$$= \int_0^{\frac{2\pi}{\omega}} F_0 \cos \omega t [-\omega X \sin(\omega t - \phi)] dt$$



$\Rightarrow$

$$E_d = \pi c \omega X^2$$

(30)

$E_d = \text{Area of Hysteresis Loop}$

## Impulse Response

$$\text{Impulse} = F \Delta t = m\dot{x}_2 - m\dot{x}_1 \quad (1)$$

$$\tilde{F} = \int_t^{t+\Delta t} F dt \quad (2)$$

### Unit impulse

$$\tilde{f} = \lim_{\Delta t \rightarrow 0} \int_t^{t+\Delta t} F dt = 1 \quad (3)$$

Consider

$$m\ddot{x} + c\dot{x} + kx = 0$$

for an underdamped system,

$$x(t) = e^{-\zeta \omega_n t} \left\{ x_0 \cos \omega_d t + \frac{v_0 + \zeta \omega_n x_0}{\omega_d} \sin \omega_d t \right\} \quad (4)$$

If the mass is at rest before the unit impulse is applied  
( $x = \dot{x} = 0$  for  $t < 0$  or at  $t = 0^-$ ) at  $t = 0$ ,

$$\begin{aligned} \text{Impulse} = \tilde{f} = 1 &= m\dot{x}(t=0) - m\dot{x}(t=0^-) \\ &= m\dot{x}(0) \end{aligned} \quad (5)$$

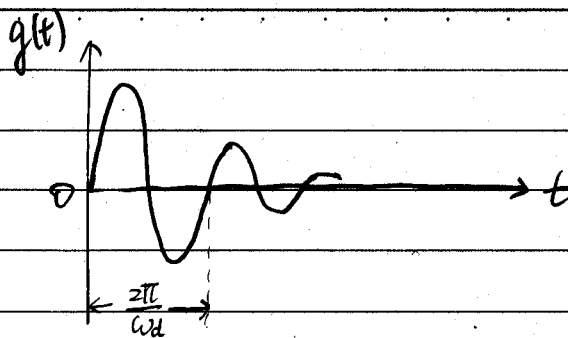
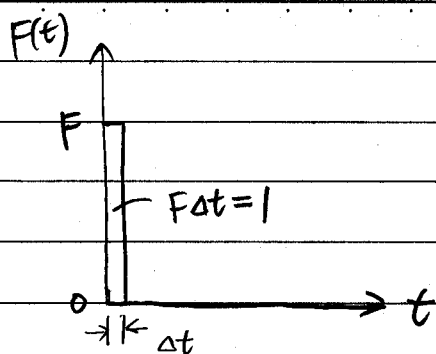
ICs:

$$\begin{aligned} x(0) &= x_0 = 0 \\ \dot{x}(0) &= v_0 = \frac{1}{m} \end{aligned} \quad (6)$$

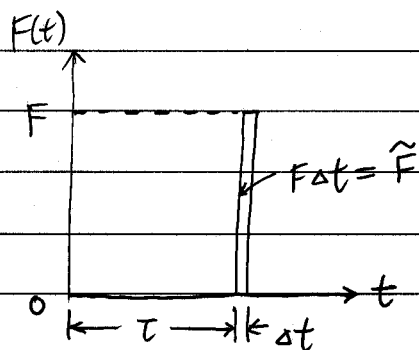
Sub. into eq. (4),

$$x(t) = g(t) = \frac{1}{m\omega_d} e^{-\zeta \omega_n t} \sin \omega_d t \quad (7)$$

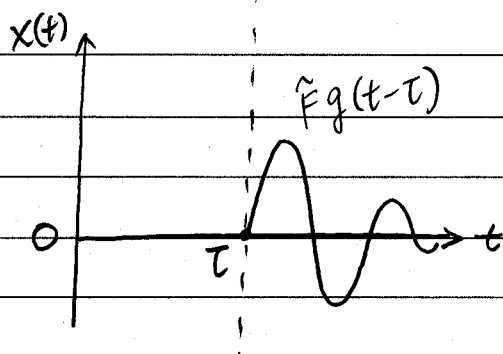
: impulse response function



If the impulse  $\tilde{F}$  is applied at an arbitrary time  $t = \tau$ ,

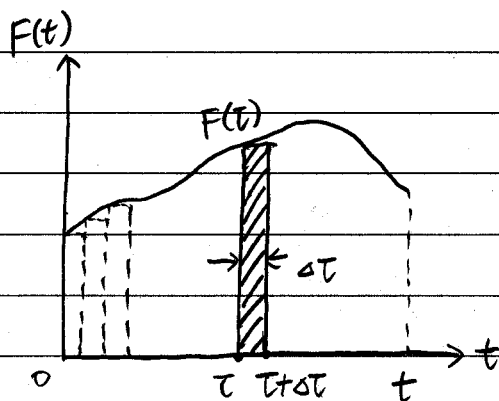


the displacement  $x$  at a later time  $t$  due to that impulse at time  $\tau$ , is given by



$$x(t) = \tilde{F} g(t - \tau) \quad (8)$$

### Convolution Integral



For an arbitrary forcing function  $F(t)$ ,

the response at  $t$  due to the impulse  $\tilde{F} = F(\tau) \Delta \tau$ :

$$\Delta x(t) = F(\tau) \Delta \tau g(t - \tau) \quad (9)$$

The total response at time  $t$  as the result of all prior impulses applied,

$$X(t) \simeq \sum F(\tau) g(t-\tau) \Delta \tau \quad (10)$$

Letting  $\Delta \tau \rightarrow 0$ ,

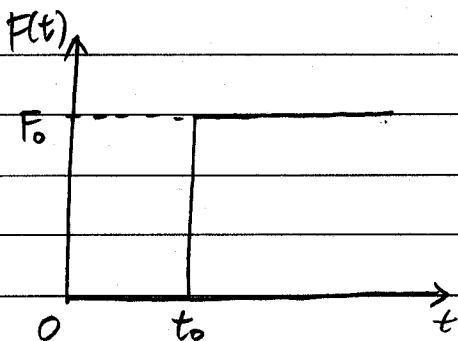
$$X(t) = \int_0^t F(\tau) g(t-\tau) d\tau \quad (11)$$

By substituting Eq. (7) into Eq. (11),

Convolution  
(Duhamel)  
Integral

$$X(t) = \frac{1}{m\omega_d} \int_0^t F(\tau) e^{-\zeta\omega_n(t-\tau)} \sin \omega_d(t-\tau) d\tau \quad (12)$$

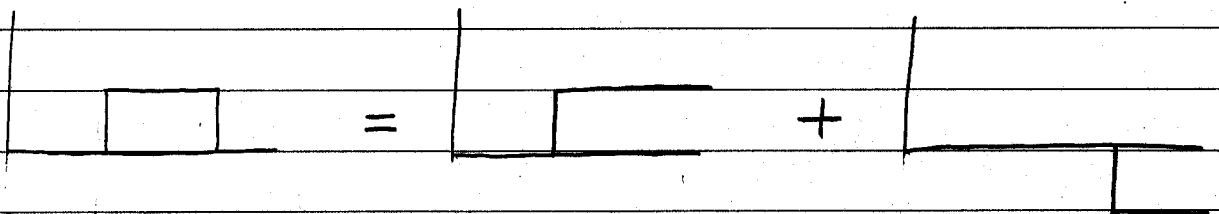
### • Step Response

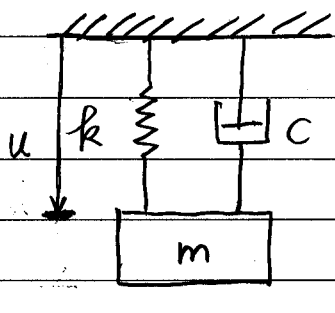


$$\begin{aligned} X(t) &= \frac{F_0}{m\omega_d} \int_{t_0}^t e^{-\zeta\omega_n(t-\tau)} \sin \omega_d(t-\tau) d\tau \\ &= \frac{F_0}{k} \left[ 1 - \frac{e^{-\zeta\omega_n(t-t_0)}}{\sqrt{1-\zeta^2}} \cos\{\omega_d(t-t_0) - \phi\} \right] \end{aligned}$$

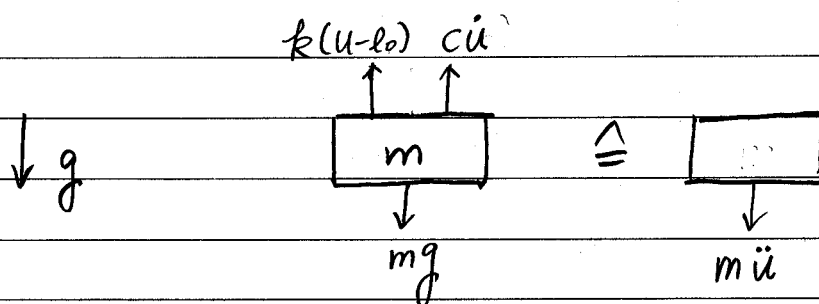
where  $\phi = \tan^{-1}\left(\frac{\zeta}{\sqrt{1-\zeta^2}}\right)$

### • Pulse





$l_0$ : unstretched length of spring



$$m\ddot{u} = mg - k(u - l_0) - c\dot{u}$$

$$\Rightarrow m\ddot{u} + c\dot{u} + ku = mg + kl_0$$

find static equilibrium ( $\dot{u} = \ddot{u} = 0$ )

$$u_{eq} = \frac{mg + kl_0}{k} = \frac{mg}{k} + l_0 > l_0$$

define  $x = u - u_{eq}$ ,  $\dot{x} = \dot{u}$ ,  $\ddot{x} = \ddot{u}$

$$\Rightarrow m\ddot{x} + c\dot{x} + k(x + u_{eq}) = mg + kl_0$$

or  $m\ddot{x} + c\dot{x} + kx + k\left(\frac{mg + kl_0}{k}\right) = mg + kl_0$

$$\therefore m\ddot{x} + c\dot{x} + kx = 0$$

