

Discretization: (Galerkin's Method)

Assume

$$w(x, t) = \sum_{i=1}^n \phi_i(x) q_i(t) \quad (22)$$

where $\phi_i(x)$ satisfies all B.C.s

Sub. eq. (22) into eq. (20),

$$\begin{aligned} & \rho_b A_b \sum_{i=1}^n \phi_i(x) \ddot{q}_i(t) + E_b I_b \sum_{i=1}^n \phi_i^{(4)}(x) q_i(t) \\ & + \left[\rho_p A_p \sum_{i=1}^n \phi_i(x) \ddot{q}_i(t) + E_p I_p \sum_{i=1}^n \phi_i^{(4)}(x) q_i(t) \right] [H(x - x_1) - H(x - x_2)] \\ & + 2E_p I_p \sum_{i=1}^n \phi_i^{(3)}(x) q_i(t) [H'(x - x_1) - H'(x - x_2)] + E_p I_p \sum_{i=1}^n \phi_i''(x) q_i(t) [H''(x - x_1) - H''(x - x_2)] \\ & + E_p d_{31} abv(t) [H''(x - x_1) - H''(x - x_2)] - f(x, t) = \varepsilon \end{aligned} \quad (23)$$

Min ε by $\langle \varepsilon, \phi_j \rangle = 0$

$$\Rightarrow \langle \varepsilon, \phi_j \rangle = \int_0^L \varepsilon(x, t) \phi_j(x) dx = 0 \quad j = 1, 2, \dots, n \quad (24)$$

$$\begin{aligned} \Rightarrow & \left[\rho_b A_b \left(\sum_{i=1}^n \int_0^L \phi_i(x) \phi_j(x) dx \right) + \rho_p A_p \left(\sum_{i=1}^n \int_0^L \phi_i(x) \phi_j(x) [H(x - x_1) - H(x - x_2)] dx \right) \right] \ddot{q}_i(t) \\ & + \left[E_b I_b \left(\sum_{i=1}^n \int_0^L \phi_i^{(4)}(x) \phi_j(x) dx \right) + E_p I_p \left(\sum_{i=1}^n \int_0^L \phi_i^{(4)}(x) \phi_j(x) [H(x - x_1) - H(x - x_2)] dx \right) \right] q_i(t) \\ & + \left[2E_p I_p \left(\sum_{i=1}^n \int_0^L \phi_i^{(3)}(x) \phi_j(x) [H'(x - x_1) - H'(x - x_2)] dx \right) \right] q_i(t) \\ & + \left[E_p I_p \left(\sum_{i=1}^n \int_0^L \phi_i''(x) \phi_j(x) [H''(x - x_1) - H''(x - x_2)] dx \right) \right] q_i(t) \\ & + E_p d_{31} abv(t) \left(\int_0^L \phi_j(x) [H''(x - x_1) - H''(x - x_2)] dx \right) - \int_0^L f(x, t) \phi_j(x) dx = 0 \end{aligned} \quad (25)$$

- $\int_0^L \phi_i^{(4)}(x) \phi_j(x) dx = \int_0^L \phi_i''(x) \phi_j''(x) dx$ (26)

- $\int_0^L \phi_i^{(4)}(x) \phi_j(x) [H(x-x_1) - H(x-x_2)] dx$ (27)

- $2 \int_0^L \phi_i^{(3)}(x) \phi_j(x) [H'(x-x_1) - H'(x-x_2)] dx$
 $= 2 \phi_i^{(3)}(x) \phi_j(x) [H(x-x_1) - H(x-x_2)] \Big|_0^L - 2 \int_0^L \phi_i^{(4)}(x) \phi_j(x) [H(x-x_1) - H(x-x_2)] dx$
 $- 2 \int_0^L \phi_i^{(3)}(x) \phi_j'(x) [H(x-x_1) - H(x-x_2)] dx$ (28)

- $\int_0^L \phi_i''(x) \phi_j(x) [H''(x-x_1) - H''(x-x_2)] dx$
 $= \phi_i''(x) \phi_j(x) [H'(x-x_1) - H'(x-x_2)] \Big|_0^L - \int_0^L \phi_i^{(3)}(x) \phi_j(x) [H'(x-x_1) - H'(x-x_2)] dx$
 $- \int_0^L \phi_i''(x) \phi_j'(x) [H'(x-x_1) - H'(x-x_2)] dx$ (29)
 $= \int_0^L \phi_i^{(4)}(x) \phi_j(x) [H(x-x_1) - H(x-x_2)] dx + \int_0^L \phi_i^{(3)}(x) \phi_j'(x) [H(x-x_1) - H(x-x_2)] dx$
 $+ \int_0^L \phi_i^{(3)}(x) \phi_j'(x) [H(x-x_1) - H(x-x_2)] dx + \int_0^L \phi_i''(x) \phi_j''(x) [H(x-x_1) - H(x-x_2)] dx$

- $(27)+(28)+(29) = \int_0^L \phi_i''(x) \phi_j''(x) [H(x-x_1) - H(x-x_2)] dx$ (30)

- $\int_0^L \phi_j(x) [H''(x-x_1) - H''(x-x_2)] dx$
 $= \phi_j(x) [H'(x-x_1) - H'(x-x_2)] \Big|_0^L - \int_0^L \phi_j'(x) [H'(x-x_1) - H'(x-x_2)] dx$
 $= - \phi_j'(x) [H(x-x_1) - H(x-x_2)] \Big|_0^L + \int_0^L \phi_j''(x) [H(x-x_1) - H(x-x_2)] dx$
 $= \int_0^L \phi_j''(x) [H(x-x_1) - H(x-x_2)] dx = \phi_j'(x_2) - \phi_j'(x_1)$ (31)

Substituting (26), (30), (31) into (25),

$$\begin{aligned}
& \left[\rho_b A_b \left(\sum_{i=1}^n \int_0^L \phi_i(x) \phi_j(x) dx \right) + \rho_p A_p \left(\sum_{i=1}^n \int_0^L \phi_i(x) \phi_j(x) [H(x-x_1) - H(x-x_2)] dx \right) \right] \ddot{q}_i(t) \\
& + \left[E_b I_b \left(\sum_{i=1}^n \int_0^L \phi_i''(x) \phi_j''(x) dx \right) + E_p I_p \left(\sum_{i=1}^n \int_0^L \phi_i''(x) \phi_j''(x) [H(x-x_1) - H(x-x_2)] dx \right) \right] q_i(t) \\
& + E_p d_{31} abv(t) \left(\phi_j'(x_2) - \phi_j'(x_1) \right) = \int_0^L f(x,t) \phi_j(x) dx
\end{aligned} \tag{32}$$

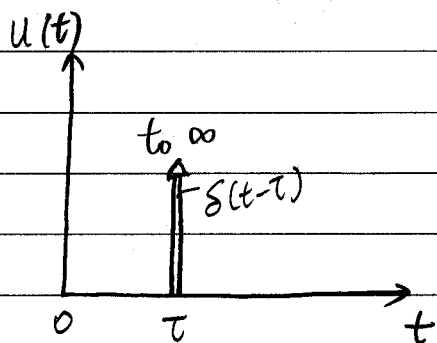
$$\sum_{i=1}^n m_{ij} \ddot{q}_i(t) + \sum_{i=1}^n k_{ij} q_i(t) = f_{c_j}(t) + f_{d_j}(t), \quad j = 1, 2, \dots, n$$

where $m_{ij} = \rho_b A_b \int_0^L \phi_i(x) \phi_j(x) dx + \rho_p A_p \int_{x_1}^{x_2} \phi_i(x) \phi_j(x) dx$

$$k_{ij} = E_b I_b \int_0^L \phi_i''(x) \phi_j''(x) dx + E_p I_p \int_{x_1}^{x_2} \phi_i''(x) \phi_j''(x) dx$$

$$f_{c_j} = E_p d_{31} abv(t) \left[\left(\phi_j'(x_1) - \phi_j'(x_2) \right) \right]$$

$$f_{d_j} = \int_0^L f(x,t) \phi_j(x) dx$$



unit impulse at $t = \tau$

$$u(t) = \delta(t - \tau)$$

where $\delta(t - \tau)$ is the Dirac delta function

$$\begin{cases} \delta(t - \tau) \rightarrow \infty, & t = \tau \\ \delta(t - \tau) = 0, & t \neq \tau \end{cases}$$

$$\int_{-\infty}^{\infty} \delta(t - \tau) dt = 1$$

Consider discrete force at $x = x_d$,

$$f(x, t) = F_d(t) \delta(x - x_d)$$

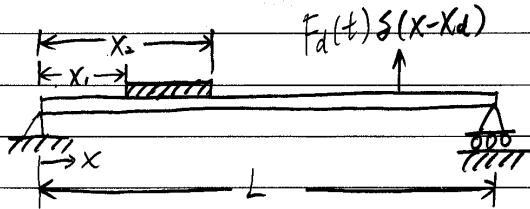
Sifting property: $\int_{-\infty}^{\infty} f(t) \delta(t - t_0) dt = f(t_0)$

$$\Rightarrow f_{ds} = \int_0^L f(x, t) \phi_s(x) dx$$

$$= \int_0^L F_d(t) \phi_s(x) \delta(x - x_d) dx$$

$$= F_d(t) \phi_s(x_d)$$

e.g. A Simply Supported Beam with a Piezoelectric Element



Choose $\phi_r(x) = \sin\left(\frac{r\pi x}{L}\right)$, $r=1, 2, \dots, N$

satisfy all B.C.s:

$$x=0: w=0, w''=0$$

$$x=L: w=0, w''=0$$

Assume

$$w(x,t) \simeq \sum_{r=1}^N \phi_r(x) \bar{q}_r(t)$$

\Rightarrow Discretized eq. of motion:

$$[M]\{\ddot{\bar{q}}\} + [C]\{\dot{\bar{q}}\} + [K]\{\bar{q}\} = \{f_c\} + \{f_d\}$$

where

$$m_{sr} = \int_0^L \rho_b A_b \phi_s(x) \phi_r(x) dx + \int_{x_1}^{x_2} \rho_p A_p \phi_s(x) \phi_r(x) dx$$

$$k_{sr} = \int_0^L E_b I_b \phi_s''(x) \phi_r''(x) dx + \int_{x_1}^{x_2} E_p I_p \phi_s''(x) \phi_r''(x) dx$$

$$f_{cs} = E_p d_{31} ab v(t) [\phi'_s(x_1) - \phi'_s(x_2)]$$

$$f_{ds} = F_d(t) \phi_s(x_d)$$

$$[c] = \alpha [M] + \beta [K]$$

for $s=r$:

$$m_{rr} = \frac{\rho_b A_b L}{2} + \rho_p A_p \left(\frac{x_2 - x_1}{2} \right) + \frac{\rho_p A_p L}{4\pi r} \left[\sin\left(\frac{2\pi r x_1}{L}\right) - \sin\left(\frac{2\pi r x_2}{L}\right) \right]$$

$$k_{rr} = \left(\frac{\pi r}{L} \right)^4 \left\{ \frac{E_b I_b L}{2} + E_p I_p \left(\frac{x_2 - x_1}{2} \right) + \frac{E_p I_p L}{4\pi r} \left[\sin\left(\frac{2\pi r x_1}{L}\right) - \sin\left(\frac{2\pi r x_2}{L}\right) \right] \right\}$$

for $s \neq r$:

$$m_{sr} = \frac{\rho_p A_p L}{\pi} \left[\frac{r \sin\left(\frac{s\pi x}{L}\right) \cos\left(\frac{r\pi x}{L}\right)}{(s^2 - r^2)} + \frac{s \cos\left(\frac{s\pi x}{L}\right) \sin\left(\frac{r\pi x}{L}\right)}{(r^2 - s^2)} \right]_{x_1}^{x_2}$$

$$k_{sr} = \frac{E_p I_p L}{\pi} \left(\frac{s r \pi^2}{L^2} \right)^2 \left[\frac{r \sin\left(\frac{s\pi x}{L}\right) \cos\left(\frac{r\pi x}{L}\right)}{(s^2 - r^2)} + \frac{s \cos\left(\frac{s\pi x}{L}\right) \sin\left(\frac{r\pi x}{L}\right)}{(r^2 - s^2)} \right]_{x_1}^{x_2}$$

$$M\ddot{z} + C\dot{z} + Kz = f_c + f_d \quad (a)$$

$$\text{Let state vector } x(t) = \begin{Bmatrix} z(t) \\ \dot{z}(t) \end{Bmatrix} \quad (b)$$

We can rewrite eq. (a) in state space form:

$$\dot{x} = Ax + Bu + \hat{B}u_d$$

$$y = C_0 x + Du$$

where

$$A = \begin{bmatrix} 0 & I \\ -M^{-1}K & -M^{-1}C \end{bmatrix}$$

$$B = \begin{bmatrix} 0 \\ M^{-1} \end{bmatrix} \varepsilon_p d_{31} ab \begin{bmatrix} \phi'_1(x_1) - \phi'_1(x_2) \\ \vdots \\ \phi'_N(x_1) - \phi'_N(x_2) \end{bmatrix} \quad \text{for } u = v(t)$$

$$\hat{B} = \begin{bmatrix} 0 \\ M^{-1} \end{bmatrix} \begin{bmatrix} \phi_1(x_d) \\ \vdots \\ \phi_N(x_d) \end{bmatrix} \quad \text{for } u_d = F_d(t)$$

$$\text{if } y = w(x_0, t) = \sum_{r=1}^N \phi_r(x_0) \phi_r(t)$$

$$\Rightarrow C_0 = [\phi_1(x_0) \ \phi_2(x_0) \ \dots \ \phi_N(x_0) \ 0 \ 0 \ \dots \ 0]$$

$$D = 0$$

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% Simply supported beam with a PZT element

clear

% initialize

b=1.27e-2;
L=0.3;
x1=0.1; x2=0.16;
xd=0.18;

Eb=7.1e10;
pb=2700;  tb=2.286e-3;

Ec=6.49e10;
pc=7600;  tc=0.762e-3;
d31=-175e-12;

Ac=b*tc;  Ab=b*tb;
Ib=b*tb^3/12;  Ic=b*tc^3/12;
a=(tb+tc)/2;

% stiffness and mass matrices

N=5; % no. of expansion terms

K=zeros(N);
M=zeros(N);
C=zeros(N);
Fc=zeros(N,1); Fd=zeros(N,1);

for r=1:N;
    for s=1:N;
        if r == s
            K(r,s)=(pi*r/L)^4*(Eb*Ib*L/2+Ec*Ic*(x2-x1)/2+...
                Ec*Ic*L/(4*pi*r)*(sin(2*pi*r*x1/L)-sin(2*pi*r*x2/L)));
            M(r,s)=pb*Ab*L/2+pc*Ac*(x2-x1)/2+...
                pc*Ac*L/(4*pi*r)*(sin(2*pi*r*x1/L)-sin(2*pi*r*x2/L));
        else
            K(r,s)=Ec*Ic*L/pi*(pi^2*r*s/L^2)^2*...
                ((r*sin(s*pi*x2/L)*cos(r*pi*x2/L))/(s^2-r^2)+...
                (s*sin(r*pi*x2/L)*cos(s*pi*x2/L))/(r^2-s^2)-...
                ((r*sin(s*pi*x1/L)*cos(r*pi*x1/L))/(s^2-r^2)+...
                (s*sin(r*pi*x1/L)*cos(s*pi*x1/L))/(r^2-s^2)));
            M(r,s)=pc*Ac*L/pi*((r*sin(s*pi*x2/L)*cos(r*pi*x2/L))/(s^2-r^2)+...
                (s*sin(r*pi*x2/L)*cos(s*pi*x2/L))/(r^2-s^2)-...
                ((r*sin(s*pi*x1/L)*cos(r*pi*x1/L))/(s^2-r^2)+...
                (s*sin(r*pi*x1/L)*cos(s*pi*x1/L))/(r^2-s^2)));
        end;
    end;
end;

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    % due to voltage input
    Fc(r)=-a*Ec*d3l*b*(pi*r/L)*(cos(r*pi*x2/L)-cos(r*pi*x1/L));
    % due to discrete force with magnitude 1/100
    Fd(r)=1/100*sin(r*pi*xd/L);
end;

% add internal damping

C=0.64*M+1.2e-6*K;

% state-space model

AL=-inv(M)*K;
AR=-inv(M)*C;
A=[zeros(N) eye(N);...
   AL AR];
BL1=inv(M)*Fc; BL2=inv(M)*Fd;
B1=[zeros(N,1);BL1];
B2=[zeros(N,1);BL2];
for r=1:N;
    CCw(1,r)=sin(r*pi/2); % displacement w at midpoint (x=L/2)
end;
CC=[CCw zeros(1,N)];
D=[0];

% control gain
Kc=1.0e+004*[-1.4035 -0.1594 0.5005 0.2457 -0.1712...
             -0.0289 -0.0033 0.0076 0.0045 -0.0030];
Ac=A-B1*Kc;

% impulse response

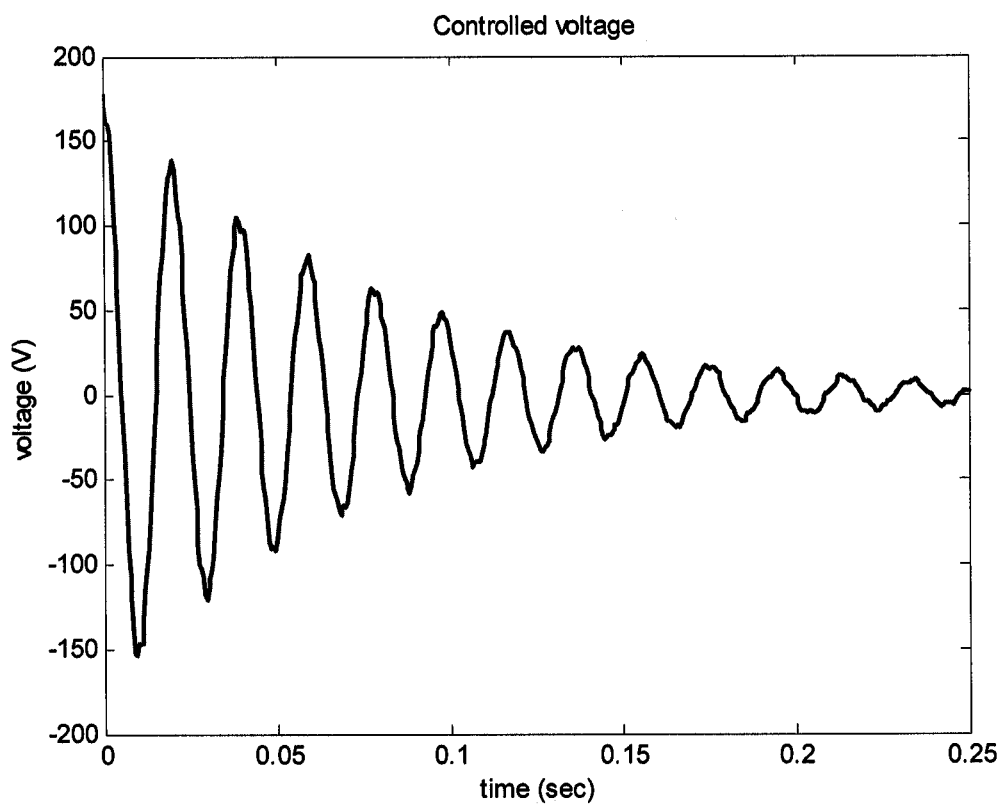
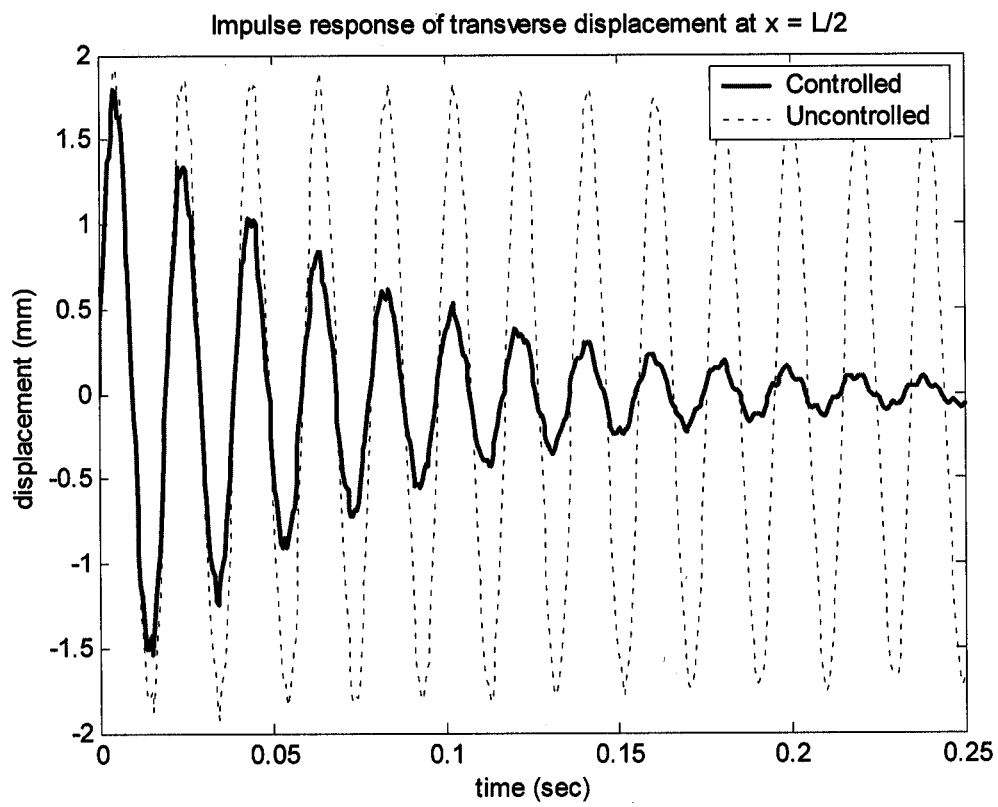
t=0:0.0005:0.25;
IU=1;
[y,x,t]=impz(A,B2,CC,D,IU,t); % uncontrolled response
[yc,x,t]=impz(Ac,B2,CC,D,IU,t); % controlled response
u=-Kc*x'; % controlled voltage

% plot results

figure(1),plot(t,yc*1000,t,y*1000,':') % unit (mm)
title('Impulse response of transverse displacement at x = L/2')
xlabel('time (sec)')
ylabel('displacement (mm)')

figure(2),plot(t,u)
title('Controlled voltage')
xlabel('time (sec)')
ylabel('voltage (V)')

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Smart Materials and Structures

□ Overview

- Smart materials
- Smart structures

□ Characteristics of Smart Materials

- Piezoelectric materials
 - Constitutive equations
 - Polarization field, coercive field, Young's modulus, Curie temperature
 - PZT vs. PVDF
- Electrostrictive materials
- Magnetostrictive materials
- Shape memory alloys
 - Nitinol
 - Martensite, austenite
 - Pseudoelastic, shape memory effect
- Electro-rheological (ER) fluids
 - Bingham plastic
- Magneto-rheological (MR) fluids
 - Three basic modes of operation
- Optic fibers

□ Vibration

- SDOF system
 - Damping
 - Natural frequency
 - Harmonic excitation
 - Resonance
 - Quality factor
 - Transient, steady-state responses
 - Energy dissipated
 - Impulse responses, convolution integral
 - Base excitation
 - Transfer function, FRF
- MDOF system
 - Equations of motion
 - Natural frequencies
 - Mode shapes

□ Structural Control

- Passive, active, active-passive, semi-active
- State space model

□ Transducers

- Sensors, actuators
- Smart sensors & actuators
 - Solid-state
 - Smart fluids
- Piezoelectric actuators/sensors
 - Motor, generator
 - Performance (blocked force, free deflection)
 - Design (stack, unimorph, bimorph, Moonie)
 - Piezoelectric accelerometer
- MR damper
 - Accumulator, offset
 - Model
 - On-off control
 - Applications (suspension systems, buildings)

□ Structural Dynamics

- Hamilton's principle
- Lagrange's equation
- Generalized coordinates
- Constraints
- Equation of motion

□ Integrated Systems

- Suspension systems with MR dampers
- Beam with piezoelectric elements
- Galerkin's method
 - Discretization
 - Comparison functions
- Discrete forces
- Rayleigh damping

□ ACE 3220/5120 Final Exam:

- **December 9, 2006 (Saturday), 3:00 – 5:00 pm**
- **ERB 407**
- **Closed-books/closed-notes**
- **One A4 hand-written sheet (double-sided)**
- **Calculator**
- **Cover concepts, theory, and applications**
- **Review lecture & tutorial notes, handouts, examples, and homework problems**