

# Mathematics for Rocket Simulation

January 31, 2026

This document describes the mathematical model used for rigid body dynamics in the rocket simulation.

## 1 Rigid body

### Definitions:

- $M$  is the mass of the rigid body.
- $i, j$  are indices representing particles in the rigid body.
- $\mathbf{r}_i$  is the position vector of particle  $i$  in the rigid body.
- $m_i$  is the mass of particle  $i$  in the rigid body.
- $O_M$  is the center of mass of the rigid body. (1)
- $I$  is the inertia tensor of the rigid body about its center of mass. (6)
- $T$  is the kinetic energy of the rigid body. (12)
- $\omega$  is the angular velocity of the rigid body. (22)
- $L$  is the angular momentum of the rigid body. (24)
- $\tau$  is the torque applied to the rigid body.
- $\rho(\mathbf{r})$  is the mass density at position  $\mathbf{r}$ .
- $V$  is the volume occupied by the rigid body.
- $a, b$  are indices representing the axes  $x, y, z$ .

A rigid body is an idealization of a solid body in which deformation is neglected. The distance between any two given points on a rigid body remains constant in time regardless of external forces or moments exerted on it.

### 1.1 Center of mass

The center of mass  $O_M$  of a system of particles is given by:

$$O_M = \frac{1}{M} \sum_i m_i \mathbf{r}_i \quad (1)$$

There may also be use for the infinite integral form:

$$O_M = \frac{1}{M} \int_V \mathbf{r} \rho(\mathbf{r}) dV \quad (2)$$

If the rigid body is made up of both point masses and continuous mass distributions, the total center of mass is given by:

$$O_M = O_{M,\text{discrete}} + O_{M,\text{continuous}} \quad (3)$$

## 1.2 Inertia tensor

The index notation form of the inertia tensor  $I$  of a rigid body about its center of mass is given by:

$$I_{ab} \equiv \sum_i m_i (r_i^2 \delta_{ab} - r_{i,a} r_{i,b}) \quad (4)$$

There may also be use for the infinite integral form:

$$I_{ab} \equiv \int_V \rho(\mathbf{r}) (r^2 \delta_{ab} - r_a r_b) dV \quad (5)$$

Where:

$\delta_{ab}$  is the Kronecker delta defined as:

$$\delta_{ab} = \begin{cases} 1 & \text{if } a = b \\ 0 & \text{if } a \neq b \end{cases}$$

This gives simply the full matrix form as:

$$I = \sum_i m_i \begin{bmatrix} y_i^2 + z_i^2 & -x_i y_i & -x_i z_i \\ -y_i x_i & x_i^2 + z_i^2 & -y_i z_i \\ -z_i x_i & -z_i y_i & x_i^2 + y_i^2 \end{bmatrix} \quad (6)$$

Where:

$x_i, y_i, z_i$  are the coordinates of particle  $i$

Or for the continuous case:

$$I = \int_V \rho(\mathbf{r}) \begin{bmatrix} y^2 + z^2 & -xy & -xz \\ -yx & x^2 + z^2 & -yz \\ -zx & -zy & x^2 + y^2 \end{bmatrix} dV \quad (7)$$

Note that  $I$  is a symmetric, i.e.,  $I_{ab} = I_{ba}$ .

For a rigid body that is made up of point and continuous mass distributions, the total inertia tensor is simply the sum of the two:

$$I = I_{\text{discrete}} + I_{\text{continuous}} \quad (8)$$

To get the inertia tensor around an arbitrary point  $P$ , we can use the parallel axis theorem:

$$I_{ab}^P = I_{ab}^{O_M} + M (d^2 \delta_{ab} - d_a d_b) \quad (9)$$

Where:

$I_{ab}^P$  is the inertia tensor about point  $P$ .  
 $I_{ab}^{O_M}$  is the inertia tensor about the center of mass  $O_M$ .  
 $\mathbf{d} = \overrightarrow{O_M P}$  is the displacement vector from the center of mass to point  $P$ .

to get the inertia tensor in world coordinates, we can use the rotation matrix  $R$  derived from the orientation quaternion  $q$ :

$$I_{\text{world}} = R I R^T \quad (10)$$

if the inertia tensor is diagonalized, one can use a quaternion  $Q$  to rotate it to world coordinates as:

$$I_{\text{world}} = Q I Q^{-1} \quad (11)$$

### 1.3 Kinetic energy

The rotational kinetic energy  $T_R$  of a rigid body is given by:

$$T_R = \frac{1}{2} \sum_{a,b} I_{ab} \omega_a \omega_b = \frac{1}{2} \omega L \quad (12)$$

### 1.4 Torque and angular acceleration

Torque  $\tau$  can be calculated from a force  $\mathbf{F}$  applied at point  $P$  as:

$$\tau = \overrightarrow{O_M P} \times \mathbf{F} \quad (13)$$

Where:

$\overrightarrow{O_M P}$  is the position vector from the center of mass  $O_M$  to point  $P$ .

The angular acceleration  $\alpha$  of the rigid body is given by:

$$\alpha = I^{-1}(\tau - \omega \times (I\omega)) \quad (14)$$

Integrating using angular momentum  $L$  is often more stable. where:

$$\dot{L} = \tau \quad (15)$$

### 1.5 Conserved quantities

In the absence of external forces and torques, the following quantities are conserved for a rigid body:

- Linear momentum  $\mathbf{p} = M\mathbf{v}$
- Angular momentum  $\mathbf{L} = I_{\text{world}}\omega$
- Kinetic energy  $T = T_T + T_R = \frac{1}{2}Mv^2 + \frac{1}{2}\omega L$

- phase space volume
- Center of mass position  $O_M$
- Mass  $M$
- Inertia tensor in body space  $I$
- Volume  $V$
- Density distribution  $\rho(\mathbf{r})$
- Total momentum  $\mathbf{P} = \sum_i \mathbf{p}_i$
- Total angular momentum  $\mathbf{L} = \sum_i \mathbf{r}_i \times \mathbf{p}_i$

## 1.6 Integration of position

The position of a rigid body is simply integrated as a point mass, using the center of mass  $O_M$  as the reference point.

$$\mathbf{O}_M(t) = \mathbf{O}_M(0) + \mathbf{v}(0)t + \int_0^t (t - t')\mathbf{a}(t')dt' \quad (16)$$

or in the finite difference form:

$$\mathbf{O}_M(t + \Delta t) = \mathbf{O}_M(t) + \mathbf{v}(t)\Delta t + \frac{1}{2}\mathbf{a}(t)(\Delta t)^2 \quad (17)$$

and the velocity as:

$$\mathbf{v}(t) = \mathbf{v}(0) + \int_0^t \mathbf{a}(t')dt' \quad (18)$$

or in the finite difference form:

$$\mathbf{v}(t + \Delta t) = \mathbf{v}(t) + \mathbf{a}(t)\Delta t \quad (19)$$

where:

- $\mathbf{v}(t)$  is the velocity of the center of mass at time  $t$ .
- $\mathbf{a}(t)$  is the acceleration of the center of mass at time  $t$ .

note that this only uses second order integration for position, and can be extended using the Taylor series if higher order accuracy is needed.

## 1.7 Integration of orientation

The orientation of the rigid body is represented using a quaternion  $q$ . The quaternion is updated using the angular velocity  $\omega$  as follows:

$$\dot{q}(t) = \frac{1}{2}\Omega(\omega(t))q(t) \quad (20)$$

$$q(t + \Delta t) = q(t) + \dot{q}(t)\Delta t \quad (21)$$

where:

$\Omega(\omega(t))$  is a pure quaternion  $(0, \omega)$

Note that the quaternion should be normalized often to prevent drift.

The angular velocity  $\omega$  is updated using the angular acceleration  $\alpha$  as follows:

$$\omega(t) = \omega(0) + \int_0^t \alpha(t')dt' \quad (22)$$

or in the finite difference form:

$$\omega(t + \Delta t) = \omega(t) + \alpha(t)\Delta t \quad (23)$$

where:

$\alpha(t)$  is the angular acceleration of the rigid body at time  $t$ , see (14)

But instead of integrating the angular velocity directly, it is often more stable to integrate the angular momentum  $L$  and then compute the angular velocity from it:

$$L(t) = L(0) + \int_0^t \tau(t')dt' \quad (24)$$

or in the finite difference form:

$$L(t + \Delta t) = L(t) + \tau(t)\Delta t \quad (25)$$

then compute the angular velocity as:

$$\omega(t) = I_{\text{world}}^{-1}L(t) \quad (26)$$

## 2 References

[https://ocw.mit.edu/courses/8-09-classical-mechanics-iii-fall-2014/6fe39e8d5ce4ce746ca256dfea665eda\\_MIT8\\_09F14\\_Chapter\\_2.pdf](https://ocw.mit.edu/courses/8-09-classical-mechanics-iii-fall-2014/6fe39e8d5ce4ce746ca256dfea665eda_MIT8_09F14_Chapter_2.pdf)