

Combined beam checks

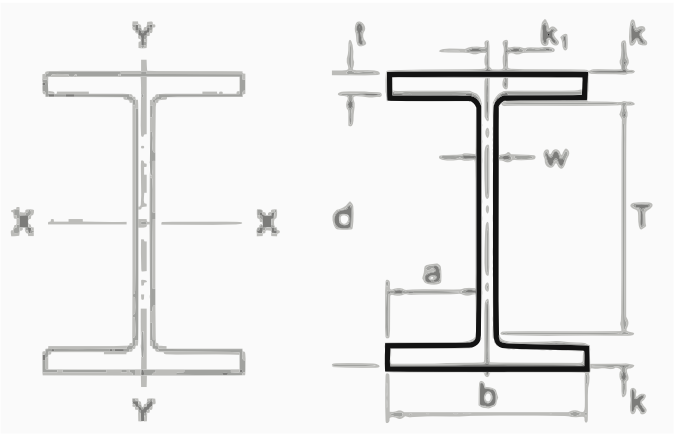
Process a comprehensive design check for a W-section beam.

Calculation ID: Blank calculation id

Verification status: Work in progress

Assumptions:

- Calc is applicable for Class 1, 2 and 3 W sections only
- Tension resistance is checked based on yielding only
- Compression force is taken as a positive value



W-Beam Axis

Inputs

Compression force in member	$C_f = 50\text{ kN}$
Tension force in member	$T_f = 50\text{ kN}$
Bending moment on member about x-axis	$M_{fx} = 50\text{ kN}$
Bending moment on member about y-axis	$M_{fy} = 50\text{ kN}$
Shear force on member in x-axis direction	$V_{fx} = 50\text{ kN}$
Shear force on member in y-axis direction	$V_{fy} = 50\text{ kN}$
Overall depth of section	$d = 257\text{ mm}$
Overall flange width	$b = 204\text{ mm}$
Web thickness	$w = 8.9\text{ mm}$
Flange thickness	$t = 15.7\text{ mm}$
Full unreduced cross-sectional area of member	$A = 8.58 \times 10^3\text{ mm}^2$
Radius of gyration about centroidal principal x axis	$r_x = 110\text{ mm}$
Radius of gyration about centroidal principal y axis	$r_y = 51\text{ mm}$
Elastic section modulus about x-axis	$S_x = 8.06 \times 10^5\text{ mm}^3$
Elastic section modulus about y-axis	$S_y = 2.18 \times 10^5\text{ mm}^3$
Moment of inertia major axis	$I_x = 1.04 \times 10^8\text{ mm}^4$
Moment of inertia minor axis	$I_y = 2.22 \times 10^7\text{ mm}^4$
Plastic section modulus about x-axis	$Z_x = 9.01 \times 10^5\text{ mm}^3$
Plastic section modulus about y-axis	$Z_y = 3.32 \times 10^5\text{ mm}^3$
Saint-Venant torsion constant of cross-section	$J = 6.25 \times 10^5\text{ mm}^4$
Torsional warping constant of cross-section	$C_w = 3.24 \times 10^{11}\text{ mm}^6$

Yield strength	$F_y = 350\text{ MPa}$
Tensile strength	$F_u = 410\text{ MPa}$
Modulus of elasticity of steel	$E = 2 \times 10^5\text{ MPa}$
Shear modulus of steel	$G = 7.7 \times 10^4\text{ MPa}$
Effective length factor for buckling about x-axis	$K_x = 1$
Unbraced length or span of member about principal x-axis direction	$L_x = 2 \times 10^3\text{ mm}$
Effective length factor for buckling about y-axis	$K_y = 1$
Unbraced length or span of member about principal y-axis direction	$L_y = 2 \times 10^3\text{ mm}$
Unbraced length or span of member for torsional buckling	$L_z = 2 \times 10^3\text{ mm}$
Distance from centroid of shear center in principal x-axis direction	$x_0 = 0\text{ mm}$
Distance from centroid of shear center in principal y-axis direction	$y_0 = 0\text{ mm}$
Lateral support condition for a member	$Condition = Laterally_Unsupported$
Moment resistance coefficient accounting for moment gradient	$\omega_2 = 1$
Coefficient to determine equivalent uniform bending effect in beam-columns for the principal x-axis	$\omega_{1x} = 1$
Coefficient to determine equivalent uniform bending effect in beam-columns for the principal y-axis	$\omega_{1y} = 1$

Calculation

Classification of W section for axial compression

	Width of stiffened or unstiffened compression elements	$\begin{aligned} b_{el} &= \frac{b}{2} \\ &= \frac{204\text{ mm}}{2} \\ &= 102\text{ mm} \end{aligned}$
S16-14 11.2 Table 2	Width to thickness ratio	$\begin{aligned} \frac{b_{el}}{t} &= \frac{b_{el}}{t} \\ &= \frac{102\text{ mm}}{15.7\text{ mm}} \\ &= 6.5 \end{aligned}$
		$\frac{b_{el}}{t} = 6.5$
S16-14 11.2 Table 1	Width to thickness limit for elements in axial compression	$\begin{aligned} Limit &= \frac{200}{\sqrt{F_y}} \\ &= \frac{200}{\sqrt{350\text{ MPa}}} \\ &= 10.7 \end{aligned}$
	Check for maximum width to thickness ratio for flange	
	$\begin{aligned} \frac{b_{el}}{t} &\leq \frac{200}{\sqrt{F_y}} \\ 6.5 &\leq \frac{200}{\sqrt{350\text{ MPa}}} \\ 6.5 &\leq 10.7 \end{aligned}$	
	Flange does not exceed maximum permissible width to thickness ratio.	

Check = Flange is not Class 4

2/24/2021	Report Preview		
	Clear depth of web between flanges	$h = d - 2 \times t$ $= 257\text{ mm} - 2 \times 15.7\text{ mm}$ $= 226\text{ mm}$	
S16-14 11.3.2 b)	Width to thickness ratio	$\frac{h}{w} = \frac{h}{w}$ $= \frac{226\text{ mm}}{8.9\text{ mm}}$ $= 25.3$	
			$\frac{h}{w} = 25.3$
S16-14 11.2 Table 1	Maximum width to thickness ratio for webs supported on both edges	$Limit = \frac{670}{\sqrt{F_y}}$ $= \frac{670}{\sqrt{350\text{ MPa}}}$ $= 35.8$	
	Check for maximum width to thickness ratio for web		
		$\frac{h}{w} \leq \frac{670}{\sqrt{F_y}}$ $25.3 \leq \frac{670}{\sqrt{350\text{ MPa}}}$ $25.3 \leq 35.8$	
	Web does not exceed maximum permissible width to thickness ratio!		
Check = Web is not Class 4			

Classification of W section for flexural compression

	Width of stiffened or unstiffened compression elements	$b_{el} = \frac{b}{2}$ $= \frac{204\text{ mm}}{2}$ $= 102\text{ mm}$	
S16-14 11.2 Table 2	Width to thickness ratio	$\frac{b_{el}}{t} = \frac{b_{el}}{t}$ $= \frac{102\text{ mm}}{15.7\text{ mm}}$ $= 6.5$	
			$\frac{b_{el}}{t} = 6.5$
S16-14 11.2 Table 2	Section classification limit for Class 1 sections	$Class\ 1\ Limit = \frac{145}{\sqrt{F_y}}$ $= \frac{145}{\sqrt{350\text{ MPa}}}$ $= 7.75$	
S16-14 11.2 Table 2	Section classification limit for Class 2 sections	$Class\ 2\ Limit = \frac{170}{\sqrt{F_y}}$ $= \frac{170}{\sqrt{350\text{ MPa}}}$ $= 9.09$	
S16-14 11.2 Table 2	Section classification limit for Class 3 sections	$Class\ 3\ Limit = \frac{200}{\sqrt{F_y}}$ $= \frac{200}{\sqrt{350\text{ MPa}}}$ $= 10.7$	
			Class = Class_1
	Clear depth of web between flanges	$h = d - 2 \times t$	

2/24/2021		Report Preview	
		$= 257\text{ mm} - 2 \times 15.7\text{ mm}$ $= 226\text{ mm}$	
S16-14 11.3.2 b)	Width to thickness ratio	$\frac{h}{w} = \frac{h}{w}$ $= \frac{226\text{ mm}}{8.9\text{ mm}}$ $= 25.3$	
		$\frac{h}{w} = 25.3$	
		Axial compressive load at yield stress $C_y = F_y \times A \times 10^{-3}$ $= 350\text{ MPa} \times 8.58 \times 10^3\text{ mm}^2 \times 10^{-3}$ $= 3 \times 10^3\text{ kN}$	
S16-14 11.2 Table 2	Section classification limit for Class 1 sections	$Class\ 1\ Limit = \frac{1.1 \times 10^3}{\sqrt{F_y}} \times \left(1 - 0.39 \times \frac{C_f}{\phi \times C_y}\right)$ $= \frac{1.1 \times 10^3}{\sqrt{350\text{ MPa}}} \times \left(1 - 0.39 \times \frac{50\text{ kN}}{0.9 \times 3 \times 10^3\text{ kN}}\right)$ $= 58.4$	
S16-14 11.2 Table 2	Section classification limit for Class 2 sections	$Class\ 2\ Limit = \frac{1.7 \times 10^3}{\sqrt{F_y}} \times \left(1 - 0.61 \times \frac{C_f}{\phi \times C_y}\right)$ $= \frac{1.7 \times 10^3}{\sqrt{350\text{ MPa}}} \times \left(1 - 0.61 \times \frac{50\text{ kN}}{0.9 \times 3 \times 10^3\text{ kN}}\right)$ $= 89.8$	
S16-14 11.2 Table 2	Section classification limit for Class 3 sections	$Class\ 3\ Limit = \frac{1.9 \times 10^3}{\sqrt{F_y}} \times \left(1 - 0.65 \times \frac{C_f}{\phi \times C_y}\right)$ $= \frac{1.9 \times 10^3}{\sqrt{350\text{ MPa}}} \times \left(1 - 0.65 \times \frac{50\text{ kN}}{0.9 \times 3 \times 10^3\text{ kN}}\right)$ $= 100$	
		Class = Class_1	

Shear capacity

Shear in the x-axis direction

		Clear depth of web between flanges $h = d - 2 \times t$ $= 257\text{ mm} - 2 \times 15.7\text{ mm}$ $= 226\text{ mm}$	
		Shear area $A_w = d \times w$ $= 257\text{ mm} \times 8.9\text{ mm}$ $= 2.29 \times 10^3\text{ mm}^2$	
		$A_w = 2.29 \times 10^3\text{ mm}^2$	
		Ultimate shear stress $F_s = 0.66 \times F_y$ $= 0.66 \times 350\text{ MPa}$ $= 231\text{ MPa}$	
		$F_s = 231\text{ MPa}$	
S16-14 13.4.1.1 a) i)	Factored shear resistance of a member or component	$V_r = \phi \times A_w \times F_s \times 10^{-3}$ $= 0.9 \times 2.29 \times 10^3\text{ mm}^2 \times 231\text{ MPa} \times 10^{-3}$ $= 476\text{ kN}$	
		$V_r = 476\text{ kN}$	

$V_{rx} = 476\text{ kN}$

Shear in the y-axis direction

Overall depth of section for weak axis shear (depth of 2 flanges)

$$\begin{aligned}d &= 2 \times b \\&= 2 \times 204\text{ mm} \\&= 408\text{ mm}\end{aligned}$$

Clear depth of web between flanges for weak axis shear (width of unstiffened compression elements)

$$\begin{aligned}h &= \frac{b}{2} \\&= \frac{204\text{ mm}}{2} \\&= 102\text{ mm}\end{aligned}$$

Web thickness for weak axis shear (flange thickness)

$$w = 15.7\text{ mm}$$

Shear area

$$\begin{aligned}A_w &= d \times w \\&= 408\text{ mm} \times 15.7\text{ mm} \\&= 6.41 \times 10^3\text{ mm}^2\end{aligned}$$

$$A_w = 6.41 \times 10^3\text{ mm}^2$$

Ultimate shear stress

$$\begin{aligned}F_s &= 0.66 \times F_y \\&= 0.66 \times 350\text{ MPa} \\&= 231\text{ MPa}\end{aligned}$$

$$F_s = 231\text{ MPa}$$

SI6-14 13.4.1.1 a) i)

Factored shear resistance of a member or component

$$\begin{aligned}V_r &= \phi \times A_w \times F_s \times 10^{-3} \\&= 0.9 \times 6.41 \times 10^3\text{ mm}^2 \times 231\text{ MPa} \times 10^{-3} \\&= 1.33 \times 10^3\text{ kN}\end{aligned}$$

$$V_r = 1.33 \times 10^3\text{ kN}$$

$$V_{ry} = 1.33 \times 10^3\text{ kN}$$

Slenderness ratio check

Slenderness ratio about x-axis

$$\begin{aligned}\frac{K_x L_x}{r_x} &= \frac{K_x \times L_x}{r_x} \\&= \frac{1 \times 2 \times 10^3\text{ mm}}{110\text{ mm}} \\&= 18.2\end{aligned}$$

Slenderness ratio about y-axis

$$\begin{aligned}\frac{K_y L_y}{r_y} &= \frac{K_y \times L_y}{r_y} \\&= \frac{1 \times 2 \times 10^3\text{ mm}}{51\text{ mm}} \\&= 39.2\end{aligned}$$

SI6-14 10.4.1

Slenderness ratio of member

$$\begin{aligned}\frac{KL}{r} &= \max\left(\frac{K_x L_x}{r_x}, \frac{K_y L_y}{r_y}\right) \\&= \max(18.2, 39.2) \\&= 39.2\end{aligned}$$

$$\frac{KL}{r} = 39.2$$

$$\begin{aligned}\frac{KL}{r} &\leq 200 \\39.2 &\leq 200 \\39.2 &\leq 200\end{aligned}$$

Slenderness ratio does not exceed maximum permissible slenderness ratio.

Check = Slenderness ratio ok!

Axial compression capacity

Factored axial compressive resistance based on flexural buckling (about x-axis and y-axis)

S16-14 13.3.1	Euler buckling stress about x-axis	$F_{ex} = \frac{\pi^2 \times E}{\left(\frac{K_x \times L_x}{r_x}\right)^2}$ $= \frac{(\pi)^2 \times 2 \times 10^5 \text{ MPa}}{\left(\frac{1 \times 2 \times 10^3 \text{ mm}}{110 \text{ mm}}\right)^2}$ $= 5.97 \times 10^3 \text{ MPa}$
S16-14 13.3.1	Euler buckling stress about y-axis	$F_{ey} = \frac{\pi^2 \times E}{\left(\frac{K_y \times L_y}{r_y}\right)^2}$ $= \frac{(\pi)^2 \times 2 \times 10^5 \text{ MPa}}{\left(\frac{1 \times 2 \times 10^3 \text{ mm}}{51 \text{ mm}}\right)^2}$ $= 1.28 \times 10^3 \text{ MPa}$
	Euler buckling stress	$F_e = \min(F_{ex}, F_{ey})$ $= \min(5.97 \times 10^3 \text{ MPa}, 1.28 \times 10^3 \text{ MPa})$ $= 1.28 \times 10^3 \text{ MPa}$
S16-14 13.3.1	Non-dimensional slenderness parameter	$\lambda = \sqrt{\frac{F_y}{F_e}}$ $= \sqrt{\frac{350 \text{ MPa}}{1.28 \times 10^3 \text{ MPa}}}$ $= 0.522$
		$\lambda = 0.522$
S16-14 13.3.1	Factored compressive resistance of a member or component	$C_r = \frac{\phi \times A \times F_y}{(1 + \lambda^{2 \times n})^{\frac{1}{n}}} \times 10^{-3}$ $= \frac{0.9 \times 8.58 \times 10^3 \text{ mm}^2 \times 350 \text{ MPa}}{(1 + 0.522^{2 \times 1.34})^{\frac{1}{1.34}}} \times 10^{-3}$ $= 2.4 \times 10^3 \text{ kN}$
		$C_r = 2.4 \times 10^3 \text{ kN}$

Factored axial compressive resistance based on torsional buckling

	Effective length factor for torsional buckling	$K_z = 1$
S16-14 13.3.2	Polar radius of gyration about shear centre	$\bar{r}_o^2 = x_o^2 + y_o^2 + r_x^2 + r_y^2$ $= (0 \text{ mm})^2 + (0 \text{ mm})^2 + (110 \text{ mm})^2 + (51 \text{ mm})^2$ $= 1.47 \times 10^4 \text{ mm}^2$
S16-14 13.3.1	Euler buckling stress	$F_e = \left(\frac{\pi^2 \times E \times C_w}{(K_z \times L)^2} + G \times J\right) \times \frac{1}{A \times \bar{r}_o^2}$ $= \left(\frac{(\pi)^2 \times 2 \times 10^5 \text{ MPa} \times 3.24 \times 10^{11} \text{ mm}^6}{(1 \times 2 \times 10^3 \text{ mm})^2} + 7.7 \times 10^4 \text{ MPa} \times 6.25 \times 10^5 \text{ mm}^4\right) \times \frac{1}{8.58 \times 10^3 \text{ mm}^2 \times 1.47 \times 10^4 \text{ mm}^2}$ $= 1.65 \times 10^3 \text{ MPa}$
S16-14 13.3.1	Non-dimensional slenderness parameter	$\lambda = \sqrt{\frac{F_y}{F_e}}$ $= \sqrt{\frac{350 \text{ MPa}}{1.65 \times 10^3 \text{ MPa}}}$ $= 0.461$

	Parameter for compressive resistance	$n = 1.34$	
<i>S16-14</i> 13.3.1	Factored compressive resistance of a member or component	$C_r = \frac{\phi \times A \times F_y}{(1 + \lambda^{2 \times n})^{\frac{1}{n}}} \times 10^{-3}$ $= \frac{0.9 \times 8.58 \times 10^3 \text{ mm}^2 \times 350 \text{ MPa}}{(1 + 0.461^{2 \times 1.34})^{\frac{1}{1.34}}} \times 10^{-3}$ $= 2.47 \times 10^3 \text{ kN}$	$C_r = 2.47 \times 10^3 \text{ kN}$

Overall axial compressive resistance

	Non-dimensional slenderness parameter	$\lambda = 0.522$	
			$\lambda = 0.522$
	Factored compressive resistance of member based on flexural buckling	$C_{r,flexural} = 2.4 \times 10^3 \text{ kN}$	
	Factored compressive resistance of member based on torsional buckling	$C_{r,torsional} = 2.47 \times 10^3 \text{ kN}$	
<i>S16-14</i> 13.3	Factored compressive resistance of a member or component	$C_r = \min(C_{r,flexural}, C_{r,torsional})$ $= \min(2.4 \times 10^3 \text{ kN}, 2.47 \times 10^3 \text{ kN})$ $= 2.4 \times 10^3 \text{ kN}$	$C_r = 2.4 \times 10^3 \text{ kN}$

$C_r = 2.4 \times 10^3 \text{ kN}$

Moment capacity (laterally unsupported member)

Strong axis bending

	Plastic moment resistance	$M_p = Z \times F_y \times 10^{-6}$ $= 9.01 \times 10^5 \text{ mm}^3 \times 350 \text{ MPa} \times 10^{-6}$ $= 315 \text{ kNm}$	$M_p = 315 \text{ kNm}$
	Critical elastic moment of a laterally unbraced beam	$M_u = \frac{\omega_2 \times \pi}{L} \times \sqrt{E \times I_y \times G \times J + \left(\frac{\pi \times E}{L}\right)^2 \times I_y \times C_w} \times 10^{-6}$ $= \frac{1 \times \pi}{2 \times 10^3 \text{ mm}} \times \sqrt{2 \times 10^5 \text{ MPa} \times 2.22 \times 10^7 \text{ mm}^4 \times 7.7 \times 10^4 \text{ MPa} \times 6.25 \times 10^5 \text{ mm}^4 + \left(\frac{\pi \times 2 \times 10^5 \text{ MPa}}{2 \times 10^3 \text{ mm}}\right)^2 \times 2.22 \times 10^7 \text{ mm}^4 \times 3.24 \times 10^{11} \text{ mm}^6} \times 10^{-6}$ $= 1.51 \times 10^3 \text{ kNm}$	$M_u = 1.51 \times 10^3 \text{ kNm}$
<i>S16-14</i> 13.6 a) i)	Factored moment resistance of a member or component	$M_r = \min\left(1.15 \times \phi \times M_p \times \left(1 - \frac{0.28 \times M_p}{M_u}\right), \phi \times M_p\right)$ $= \min\left(1.15 \times 0.9 \times 315 \text{ kNm} \times \left(1 - \frac{0.28 \times 315 \text{ kNm}}{1.51 \times 10^3 \text{ kNm}}\right), 0.9 \times 315 \text{ kNm}\right)$ $= 284 \text{ kNm}$	$M_r = 284 \text{ kNm}$

Weak axis bending

S16-14 13.5 a)

Factored moment resistance of a member or component

$$\begin{aligned} M_r &= \phi \times F_y \times Z \times 10^{-6} \\ &= 0.9 \times 350\text{ MPa} \times 3.32 \times 10^5\text{ mm}^3 \times 10^{-6} \\ &= 105\text{ kNm} \end{aligned}$$

$M_r = 105\text{ kNm}$

$M_{ry} = 105\text{ kN}$

Axial tension capacity

S16-14 13.2 a) i)

Factored tensile resistance of a member or component

$$\begin{aligned} T_r &= \phi \times \frac{A_g \times F_y}{1 \times 10^3} \\ &= 0.9 \times \frac{8.58 \times 10^3\text{ mm}^2 \times 350\text{ MPa}}{1 \times 10^3} \\ &= 2.7 \times 10^3\text{ kN} \end{aligned}$$

$T_r = 2.7 \times 10^3\text{ kN}$

$T_r = 2.7 \times 10^3\text{ kN}$

Capacity check summary

Summary table of demands, capacities and utilisation ratios for axial forces, moments and shear forces:

Summary	Compression, Cf/Cr	Tension, Tf/Tr	Strong Axis Moment, Mfx/Mrx	Weak Axis Moment, Mfy/Mry	Shear in X-Axis Direction, Vfx/Vrx	Shear in Y-Axis Direction, Vfy/Vry
Demand (kN)	50	50	50	50	50	50
Capacity (kN)	2395.786	2702.7	283.815	104.58	475.53	1331.724
Utilisation	0.021	0.019	0.176	0.478	0.105	0.038

Bibliography

S16-14 CSA S16 (2014) Canadian Standards Association: Design of steel structures. Canada.