

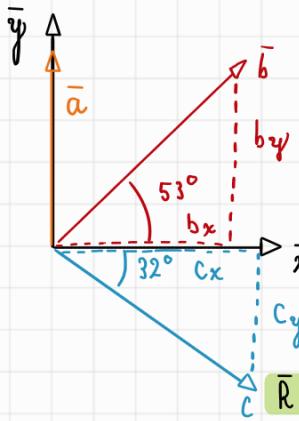
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DINÁMICA DE SISTEMAS MECÁNICOS

PROBLEM SETS

$$\text{ej 2} \quad \|\bar{a}\| = 10, \|\bar{b}\| = 20, \|\bar{c}\| = 35$$



J. Componentes x-y de cada vector

$$\bar{a} = 0\hat{x} + 10\hat{y}$$

$$\bar{b} = 12.04\hat{x} + 15.97\hat{y}, \quad 20 \sin(53^\circ) = b_y = 15.97$$

$$\bar{c} = 29.68\hat{x} - 18.55\hat{y} \quad 20 \cos(53^\circ) = b_x = 12.04$$

$$35 \sin(-32^\circ) = b_y = -18.55$$

$$35 \cos(-32^\circ) = b_x = 29.68$$

$$\bar{R} = \bar{a} + \bar{b} + \bar{c} = 41.72\hat{x} + 7.42\hat{y}$$

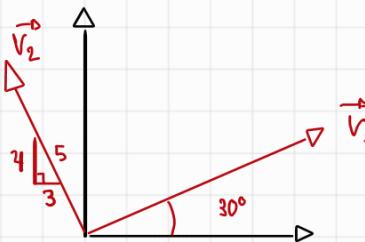
$$\|\bar{R}\| = \sqrt{(41.72)^2 + (7.42)^2} = 42.374 \quad \theta_R = \tan^{-1}(7.42 / 41.72) = 10.08^\circ$$

$$\text{proj}_{\bar{R}}(\bar{a}) = \frac{(0 * 41.72) + (10 * 7.42)}{42.374} = 1.751$$

$$\text{proj}_{\bar{R}}(\bar{b}) = \frac{(12.04 * 41.72) + (15.97 * 7.42)}{42.374} = 14.65$$

$$\text{proj}_{\bar{R}}(\bar{c}) = \frac{(29.68 * 41.72) + (-18.55 * 7.42)}{42.374} = 25.97$$

ej 3 \bar{V}_1, \bar{V}_2 de mag V_1, V_2 , det $\bar{V}_1 + \bar{V}_2, \bar{V}_1 - \bar{V}_2, \bar{V}_1 \times \bar{V}_2, \bar{V}_1 \cdot \bar{V}_2$



$$V_{1x} = V_1 \cos(30^\circ) = \sqrt{3}/2 V_1$$

$$V_{1y} = V_1 \sin(30^\circ) = 1/2 V_1$$

$$V_{2x} = 3/5 * V_2 = 3/5 V_2$$

$$V_{2y} = 4/5 * V_2 = 4/5 V_2$$

$$\bar{V}_1 + \bar{V}_2 = (\sqrt{3}/2 V_1 + 3/5 V_2) \hat{x} + (1/2 V_1 + 4/5 V_2) \hat{y}$$

$$\bar{V}_1 - \bar{V}_2 = (\sqrt{3}/2 V_1 - 3/5 V_2) \hat{x} + (1/2 V_1 - 4/5 V_2) \hat{y}$$

$$\bar{V}_1 \times \bar{V}_2 = (V_{1x} - V_{2y})(V_{2x} * V_{1y}) = (\sqrt{3}/2 V_1 * 4/5 V_2) - (3/5 V_2 * 1/2 V_1) = (V_1 V_2)((-3 + 4\sqrt{3})/10)$$

$$\bar{V}_1 \cdot \bar{V}_2 = (\sqrt{3}/2 V_1 * 3/5 V_2) + (1/2 V_1 * 4/5 V_2) = V_1 V_2 \cdot 3\sqrt{3}/10 + V_2 V_1 \cdot 2/5 = V_1 V_2 ((4 + 3\sqrt{3})/10)$$

ej 4 Tomamos un punto de referencia en O y el marco $e_r - e_\theta$ que sigue al punto B

$$r = l \hat{e}_r$$

$\dot{r} = l \hat{e}_r + l \dot{\hat{e}}_r = l \hat{e}_r + l \dot{\theta} \hat{e}_\theta$

$\ddot{r} = l \ddot{\hat{e}}_r + l \dot{\hat{e}}_r + l \dot{\theta} \dot{\hat{e}}_\theta + (\ddot{\theta} \hat{e}_\theta + \dot{\theta} \dot{\hat{e}}_\theta) l$

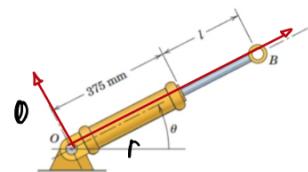
$$\text{de nuevo } \hat{e}_r = \dot{\theta} \hat{e}_\theta \text{ y } \dot{\hat{e}}_\theta = -\dot{\theta} \hat{e}_r$$

$$\ddot{r} = l \ddot{\hat{e}}_r + l \dot{\theta} \hat{e}_\theta + l \dot{\theta} \dot{\hat{e}}_\theta + l \ddot{\theta} \hat{e}_\theta - \dot{\theta}^2 l$$

$$\ddot{r} = \hat{e}_r (l - \dot{\theta}^2 l) + \hat{e}_\theta (l \ddot{\theta} + 2 \dot{\theta} \dot{\theta})$$



A medida que el cilindro hidráulico gira alrededor de O, la longitud expuesta l del vástago del pistón P es controlada por la acción de la presión del aceite del cilindro. Si el cilindro gira a una velocidad constante de $\dot{\theta}=60 \text{ deg/s}$ y l esta decreciendo a una razón constante de 150 mm/s, calcule la velocidad v de la punta del cilindro B y su aceleración.



Usamos coordenadas polares

$$\dot{\theta} = 60 \text{ deg/s}, \ddot{\theta} = 0$$

pero la magnitud de \dot{r} depende de que tan rápido gira, que es $\dot{\theta}$ $\dot{r} = 125 \text{ mm}, \dot{l} = -150 \text{ mm/s} = \dot{r}, \ddot{l} = 0 = \ddot{r}$ $r = 375 \text{ mm} + l$

$$\ddot{r} = -\dot{\theta}^2 l \hat{e}_r + 2\dot{\theta}\dot{\phi} \hat{e}_\theta$$

Finalmente reemplazamos con los datos del problema

$$r = -1.150 \text{ m/s} \hat{e}_r + (0.375 + l) \frac{\pi}{3} \hat{e}_\theta$$

$$a = -\frac{\pi^2}{9} (0.375 + l) \hat{e}_r - 0.1\pi \hat{e}_\theta$$

Las magnitudes son entonces

$$\|v\| = \sqrt{(-0.15)^2 + [(0.375 + l) \frac{\pi}{3}]^2} \quad [\text{mm/s}]$$

$$\|a\| = \sqrt{[-\frac{\pi^2}{9} (0.375 + l)]^2 - (0.1\pi)^2} \quad [\text{mm/s}]$$