

Chapter 1

Introduction

to tribology

MSE 485
Tribology

1. Definition
2. Relevance
3. Tribological contacts
4. Surfaces
5. Contact mechanics
6. Friction
7. Wear
8. Lubrication
9. Tribological system

- Etymology: from greek “tribos” (to rub) and “logy” (knowledge)
- « Tribology is the science and practice of interacting surfaces in relative motion and of the practices related thereto. »
P. Jost in: Lubrication (Tribology) education and research, A report on the present position and industrial needs, HMSO (1966)

- It hence studies the principles of **friction, wear, and lubrication**

Examples

- Some phenomena ruled by friction :

- Grasping objects



- Walking



- Writing



- Assembly strength (screw, nails, bolts)



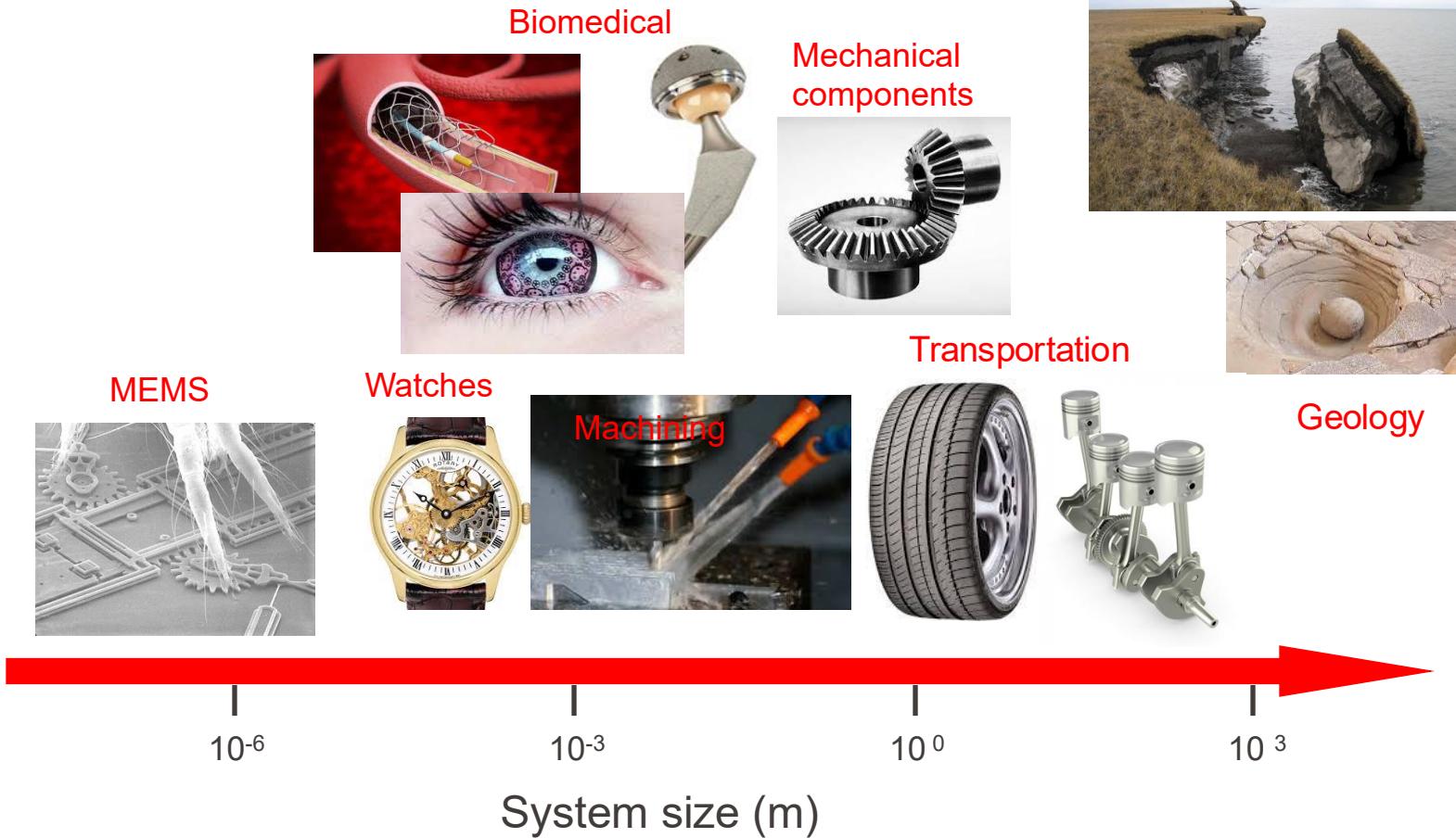
- Landslides



- Braking



Some tribology-related technologies



Some present challenges and opportunities for tribologists

- Hydrogen economy: Storage, generation, transportation, utilization
- Transportation (modern electric vehicles): Optimization of gears and dynamic seals (still 57% of the losses are due to friction) *Farfan-Cabrera Tribology International (2019)*
- Energy conversion: wind mills (low speed systems and high loads, marine environment, current generation)



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Country	Cost /year	Potential savings
UK	24 £billion	2 £billion
Canada	3.7 C\$ billion	0.83 C\$ billion
Spain	-	13.2 €billion (1.4% of the country's GDP)

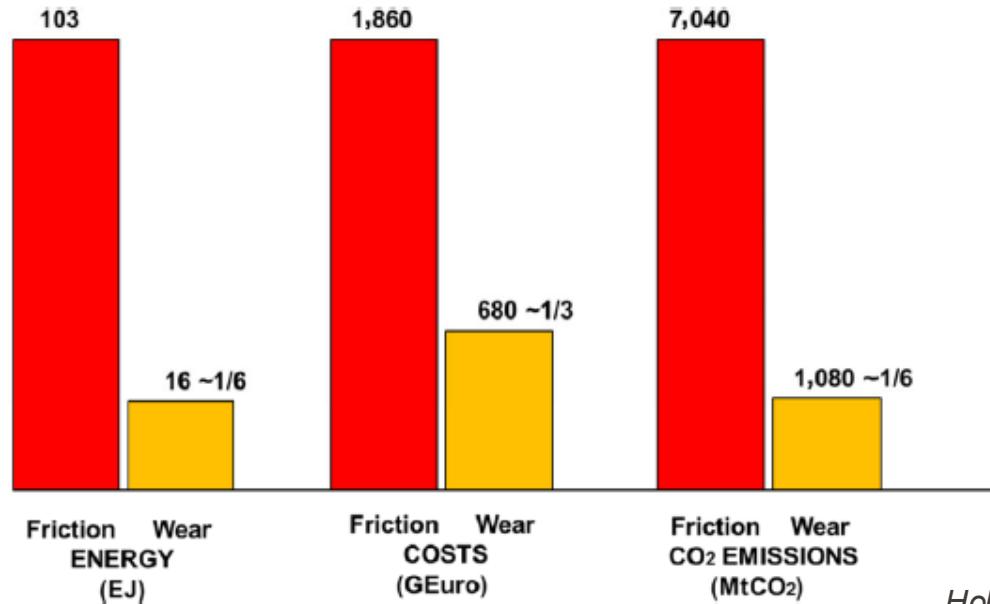
Good practice of tribology saves money....among others

Energy impact

- ~23% (119 EJ) of the world's total energy consumption originates from tribological contacts.
 - 20% (103 EJ) to overcome friction
 - 3% (16 EJ) to remanufacture worn parts and spare equipment due to wear
- Potential reduction 40% in the long term (15 years) by:
 - New surfaces
 - Materials
 - Lubrication technologies for friction reduction and wear protection

CO₂ emission by friction accounts for 1/5th of global CO₂ emission (35.000 Mt CO₂)

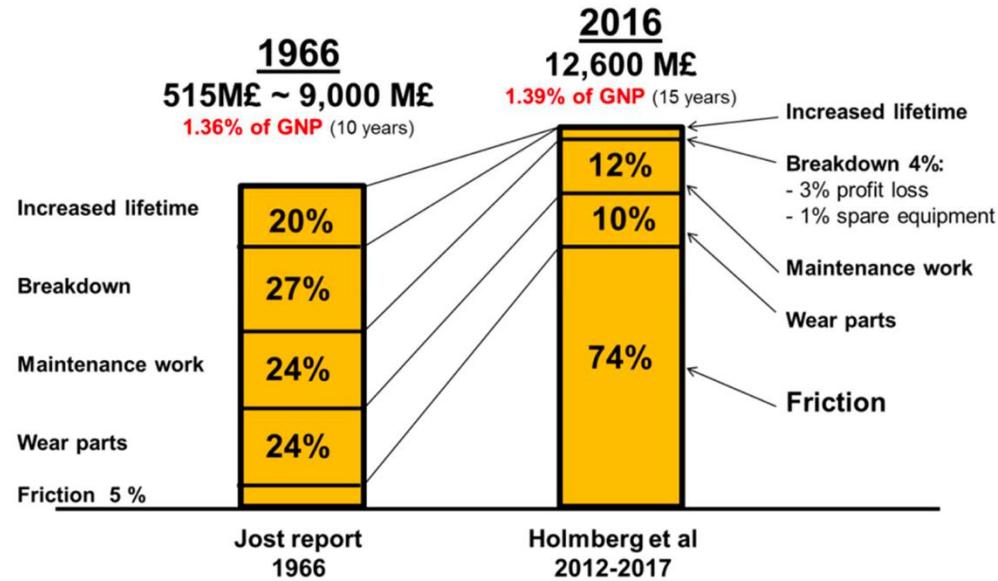
- Energy consumption, costs and CO₂ emissions due to friction and wear



- Tyre wear particles:
 - EU: 1.300.000 tons/year
 - Average size 25 µm (4-265 µm)
- Wear of implants:
 - Hip joints: 100.000 particles/step in the body (MoP)



- Possible annual savings in Great Britain (1966 and 2016) by using modern tribological solutions.

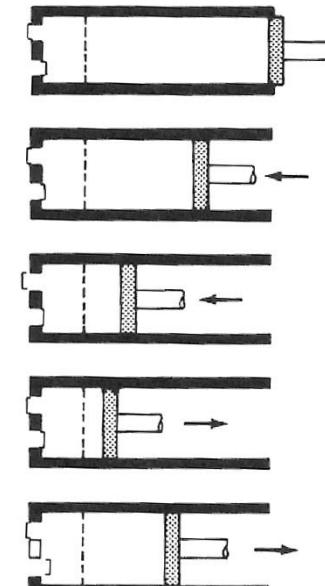
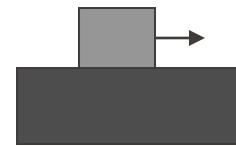


1. Concept
2. Relevance

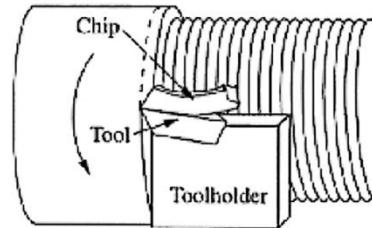
3. Tribological contacts

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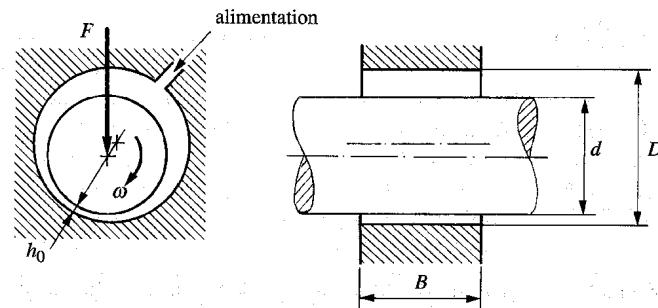
Sliding contacts (*sliding wear*)



Piston/cylinder



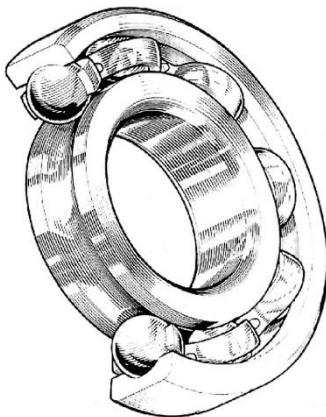
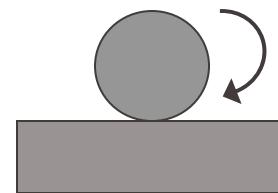
Tool/workpiece during turning



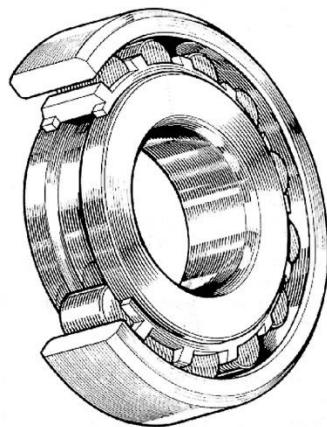
Axle/cushion in a bearing

Rolling contacts (*rolling wear*)

- A round body rolling on a counter body

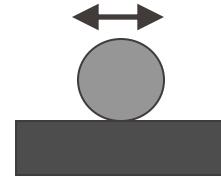


Ball bearing

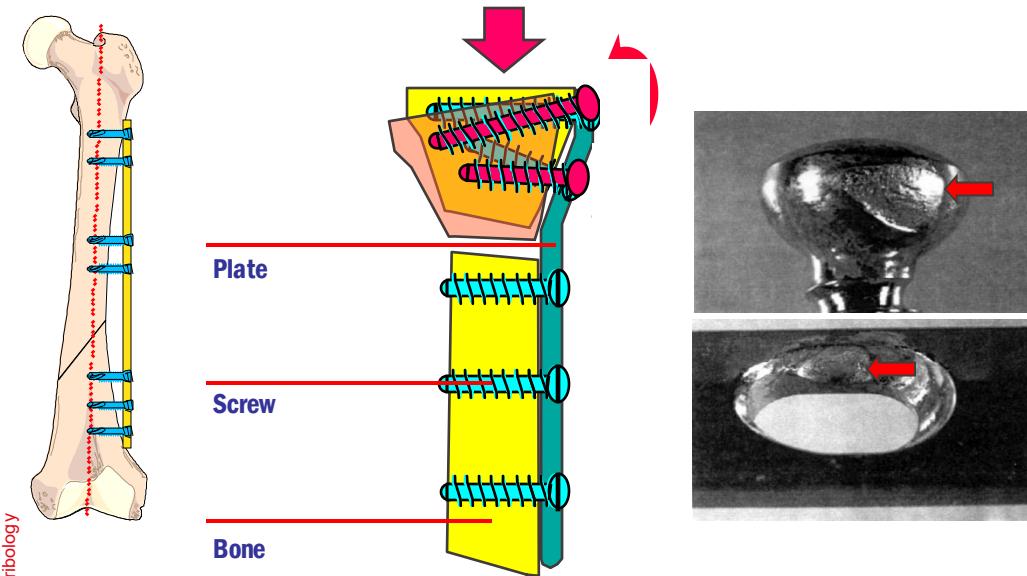


Roller bearing

Fretting (*fretting wear*)

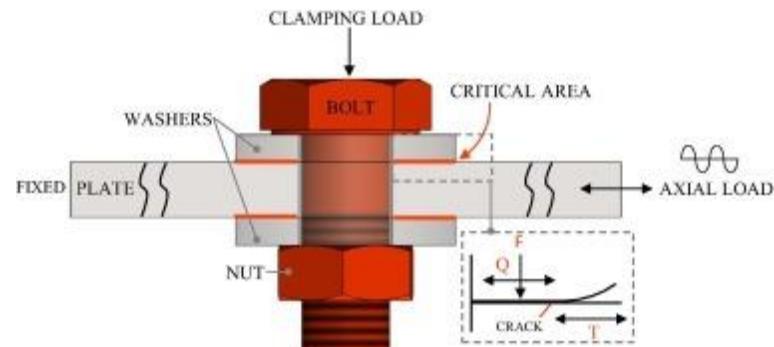


- Low amplitude relative motion (vibration) of two interacting bodies
Wear due to small relative displacements



Tribology

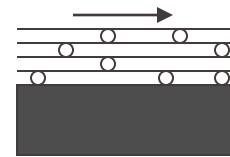
- Screw/plate for orthopaedic fixation : micromovements



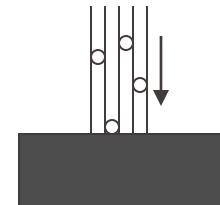
Bolt unions

Other tribological situations

- Particles carried by a fluid sliding over a body
→ Erosion



- Particles impacting on a body
→ Impact wear



- Gas particles imploding in turbulent fluids
→ Cavitation (*cavitational wear*)



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Surfaces: the elements through which solids contact

- 2 dimensional (planar) defect with certain thickness



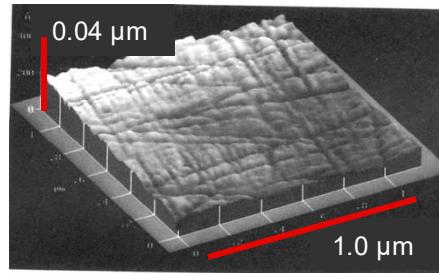
Silicon wafer: atomically flat, uniform chemistry



Steel pipe: rough, partially rusted

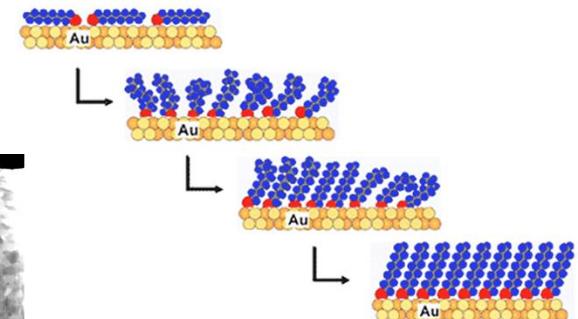
Surfaces: not simple, neither flat

- Topographical features: roughness...
 - Contact area, contact stresses, wetting



Mirror polished steel surface: AFM image

- Chemical features: adsorbed molecules, oxides...
 - Influences friction



- Microstructural features:
 - Influences wear



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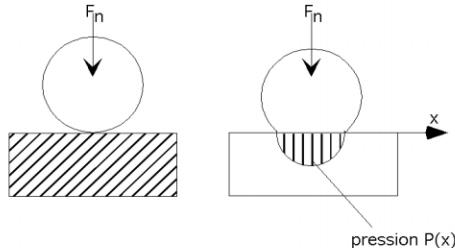
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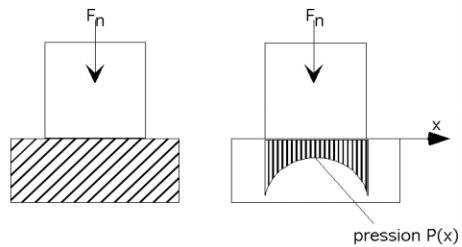
Study of the amplitude and distribution of mechanical stresses in a contact.

- Conformity of the contact

Non-conformal contact :

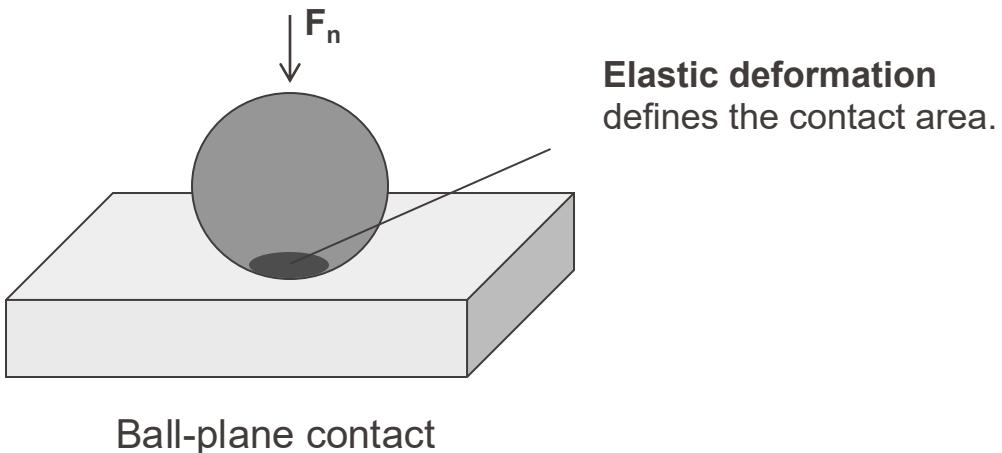


Conformal contact :



Analysis of elastic stress fields

- Hertz mechanics for non-conformal contacts:
 - Calculation of elastic strain and stress in terms of load, geometrical parameters and materials.



Hertz Contact Mechanics Formalism: example for a ball-plane contact

- Radius of contact area (circle)

$$a = \frac{1.5F_n R}{E'}^{\frac{1}{3}}$$

- Maximum contact pressure

$$p_0 = \frac{3F_n}{2\pi a^2}$$

- Average contact pressure

$$p_m = \frac{F_n}{\pi a^2}$$

$$\frac{1}{E'} = 0.5 \left[\frac{1 - \nu_1^2}{E_1} + \frac{1 - \nu_2^2}{E_2} \right]$$

- Maximum deflection

$$w = 1.31 \frac{F_n^2}{E'^2 R}^{\frac{1}{3}}$$

E = Young's modulus
v = Poisson's ratio

- Maximum shear stress

$$\tau_{\max} = \frac{p_0}{3}$$

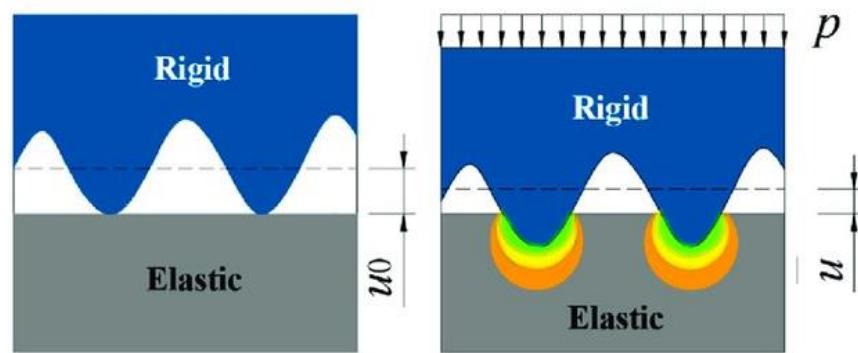
- Depth of maximum shear strength

$$z = 0.638 a$$

But real surfaces have a certain roughness

“Putting two solids together is rather like turning Switzerland upside down and standing it on Austria – the area of intimate contact will be small”

F.P. Bowden



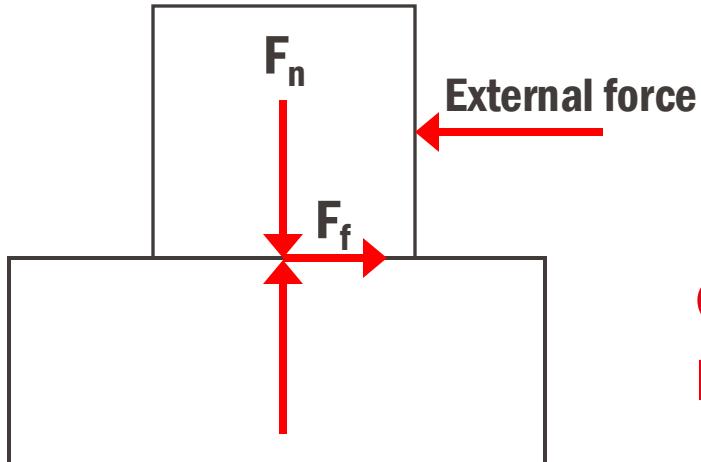
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What is friction?

- Tangential force (F_f) at the surface between two bodies preventing (static friction) or opposing to (dynamic friction) the relative motion of the two bodies caused by an external force.

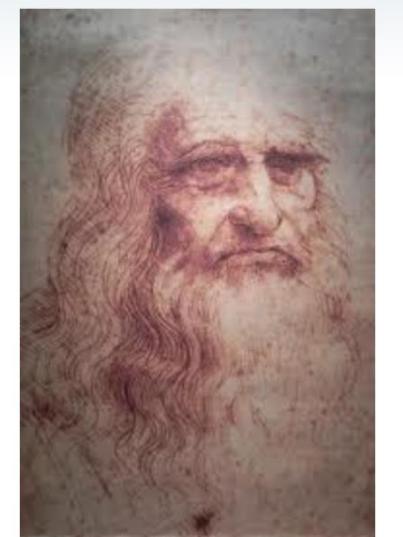


Involved forces

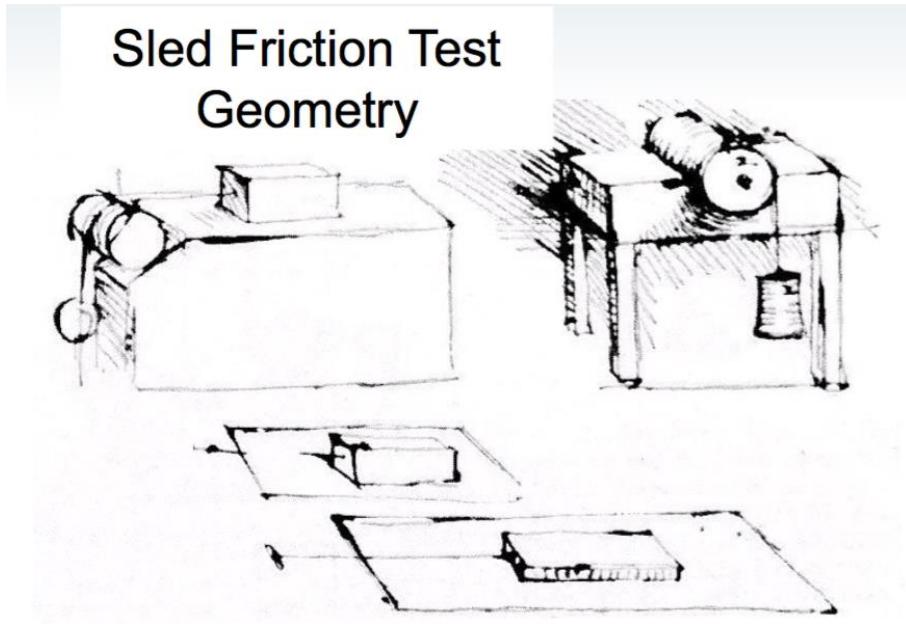
Coefficient of friction:
 $\mu = F_f / F_n$

Motivation to study friction: machine conception

- The friction force is proportional with the applied normal force
- The friction force is independent of the nominal/apparent area of contact



Leonardo Da Vinci



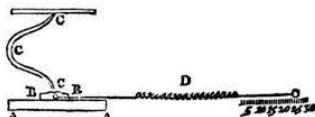
- Amonton's laws (1699) – actually already proposed by Da Vinci (1500):
 1. The friction force is proportional with the applied normal force: $F_t = \mu F_n$
 2. The friction force is independent of the nominal/apparent area of contact
 3. The friction force is independent of sliding speed (Coulomb's law of friction, 1785)

Friction history

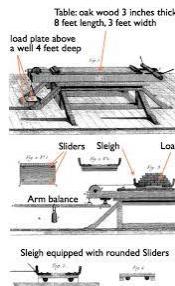
DaVinci (1500)



Amontons (1699)



Coulomb (1785)



Industrial polymers
(1920-1940)

Polymers

Metals

Ceramics

Wood

1500

1600

1700

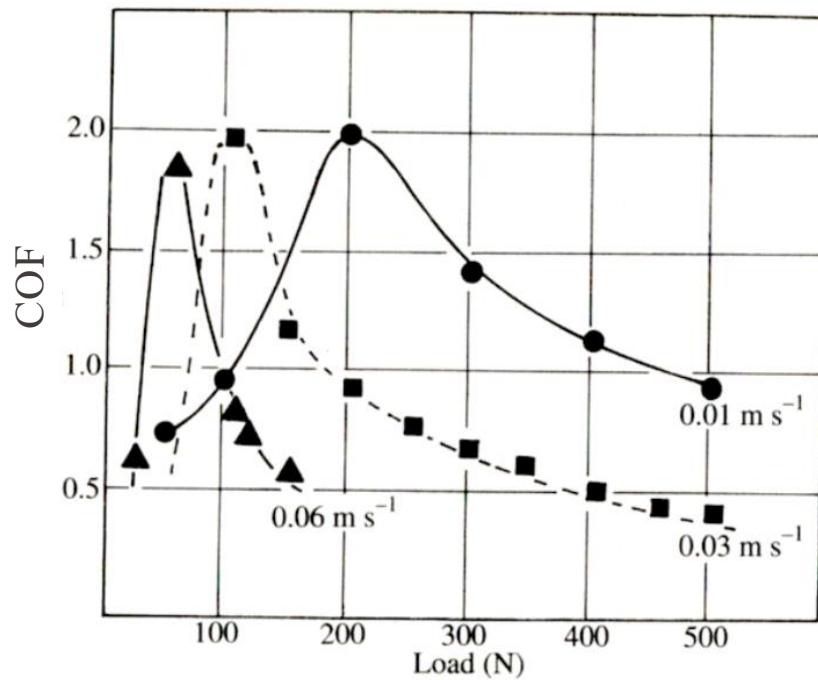
1800

1900

2000

What about polymers??

- Friction of polymers may depend on load and on sliding velocity
- Example: COF for nylon sliding against steel



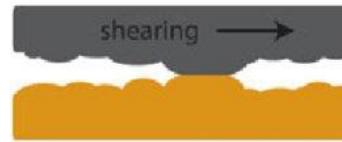
Source: Williams "Engineering Tribology" (1994)

Origin of friction

"Interfacial friction is caused by the ploughing of asperities in the mating surface and adhesion forces between the interacting asperity summits"

F.P Bowden and D. Tabor (1942)

- **Adhesion:** due to the shear resistance between contacting surfaces.

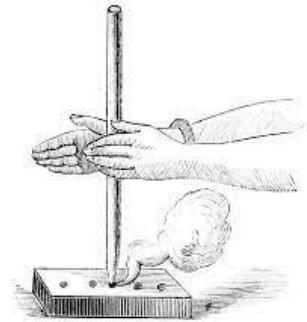


- **Ploughing:** due to resistance of surface asperities ploughing the contacting surface.

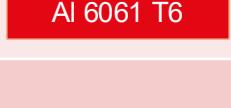


Consequences of friction

- Energy dissipation: heating
- Surface traction: shearing, failure, wear



Friction is a system parameter – not a material parameter !

Influence of	Diagram	Parameter	COF
Sliding partner (X)		Al6061 T6	0.38
		Copper	0.28
		Steel 1032	0.23
		Teflon	0.07
Contact configuration		Al 6061 T6	0.38
		Ti6Al4V	0.29
Environment		Vacuum	> 4 (seizure)
		10^{-3} mbar O ₂	1.50
		1 mbar O ₂	0.40
		Oil film	< 0.10
Roughness		R _q 390 nm	0.31
		R _q 220 nm	0.20
		R _q 120 nm	0.09
		R _q 68 nm	0.09

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Definitions with very different implications

- *Deterioration* throughout prolonged use, due to friction
- **Progressive loss of material** from the surface of a solid body due to mechanical interactions occurring during contact and relative motion with a solid, liquid or gaseous counter body.
- These two notions are not necessarily related :
 - Durability** of a system functionality
 - Loss** of material

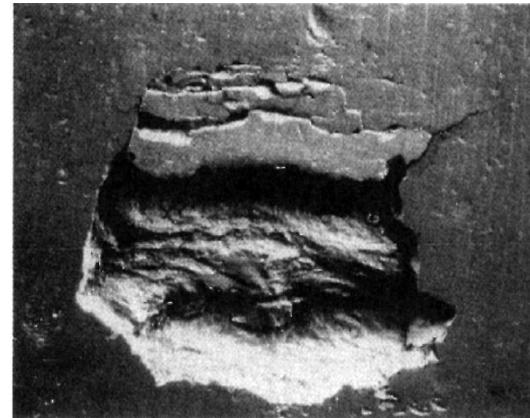
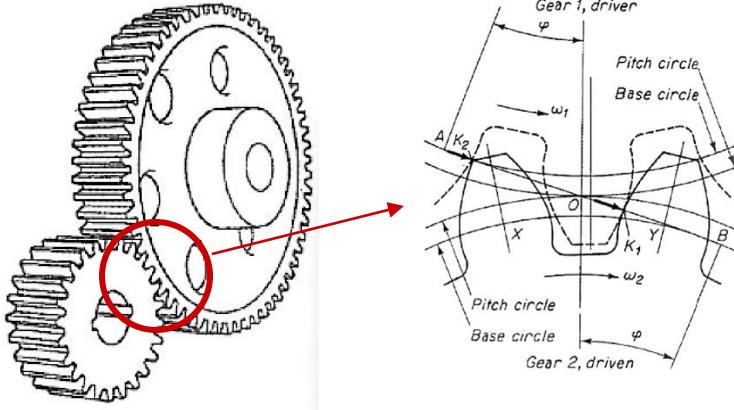
Example of progressive material loss

- Tyres : loss of functionality due to the progressive material removal.



Example of sudden loss of function by wear

- Gears : loss of functionality due to the sudden removal of a single tiny particle after long operational periods without any significative loss of material.

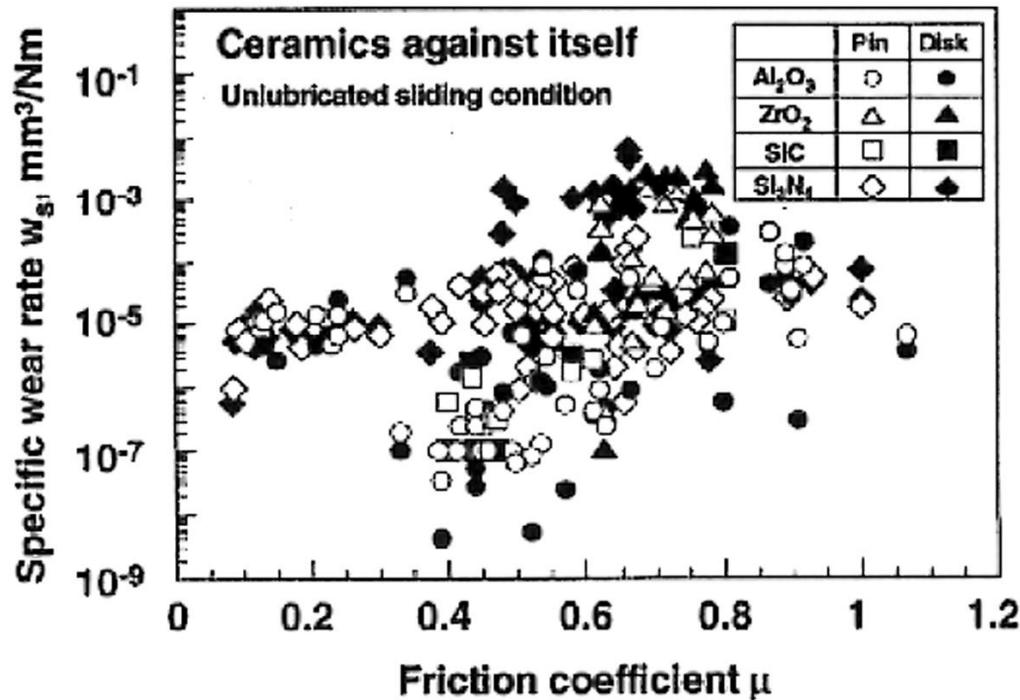


Fatigue failure of a bearing steel component.

Wear rate and friction

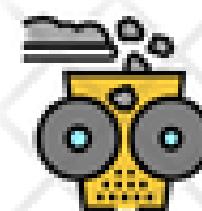
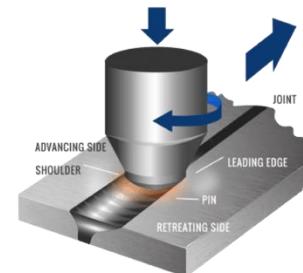
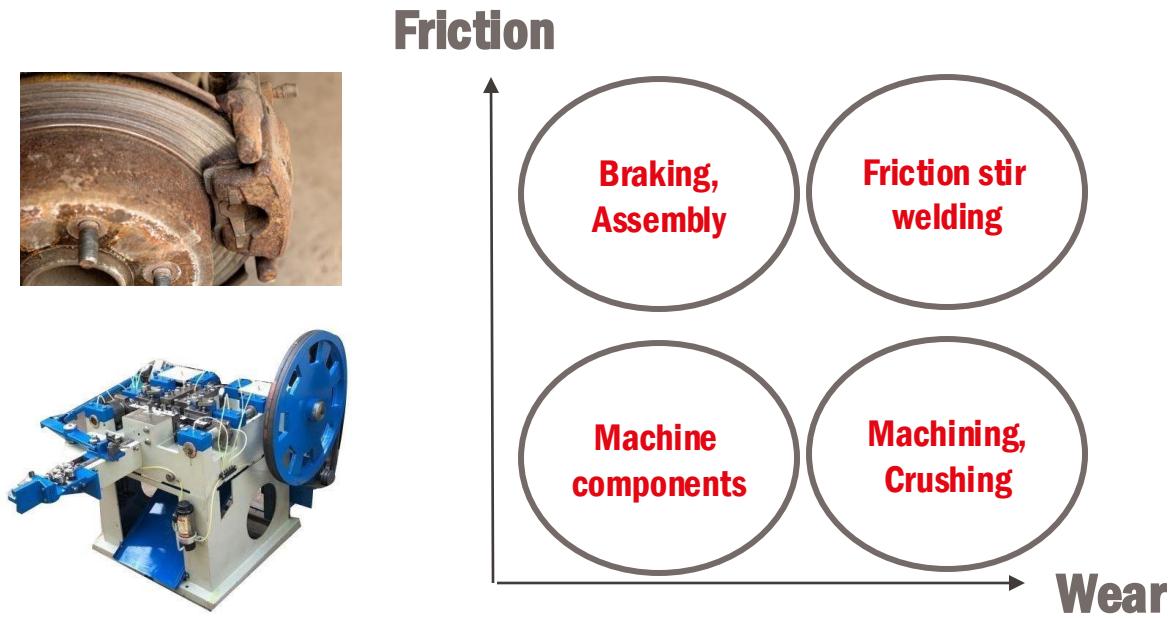
(Kato 2001)

- No obvious correlation between these two parameters.



Technological and economic aspects

- Wear and friction are not necessarily negative phenomena to be absolutely avoided!



First wear study: gold coins and material loss

Experiments and Observations on the various Alloys, on the specific Gravity, and on the comparative Wear of Gold. Being the Substance of a Report made to the Right Honourable the Lords of the Committee of Privy Council, appointed to take into Consideration the State of the Coins of this Kingdom, and the present Establishment and Constitution of His Majesty's Mint. By Charles Hatchett, Esq. F.R.S. Read January 13, 1803. [Phil. Trans. 1803, p. 43.]

- Experimental conditions (Charles Hatchett 1803):
 - Material: Type of gold (ductile or hard)
 - Topography: coins with flat, smooth, and broad surfaces and coins with protuberant parts
 - Mechanical variables: sliding speed, pressure, type of contact and contact geometry
- Quantification of wear: coin weight loss

Wear formalism

Outcome of two centuries of scientific effort to quantify wear:

- Numerous equations available for wear.

Meng and Ludema (Wear 181-183(2) (1995) 443-457) identified:

182 equations for wear published between 1955 and 1995.

625 involved variables, either as **numerator or denominator**

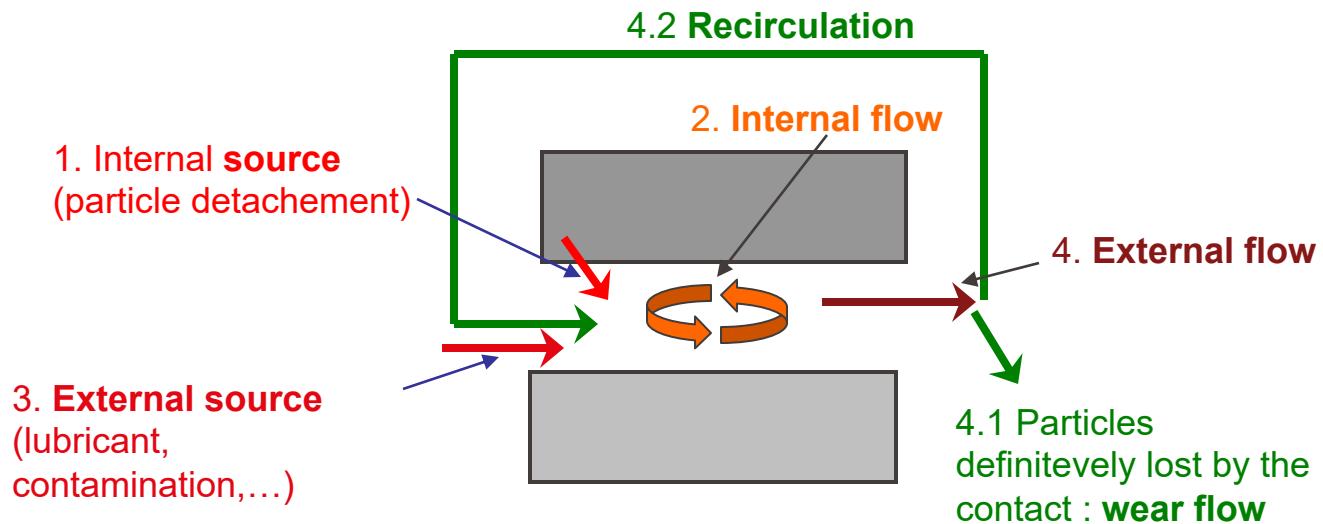
- No single predictive model/equation of wear exists per today.
- Wear involves chemical and physical interactions with the mechanical components – difficult to model.

No universal formalism !

Existing laws apply to very specific cases only !

More than a mass loss: Third body concept and material flow

- Wear can be described as a flow of particles:



Wear is a system response
Wear resistance is not a material
property

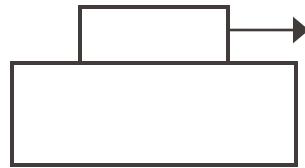
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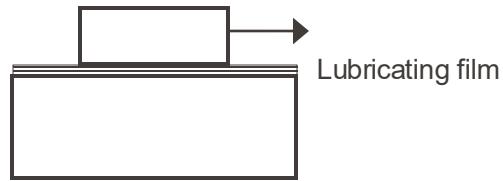
9. Tribological system

What is lubrication?

- Reduction in friction and/or wear by interposing a separating film of lubricant between two interacting bodies in relative motion.



Non lubricated contact



Lubricated contact

- Lubricants : liquid, gas, solid, semi-solid, powder

Technology started very early... 2400 BC

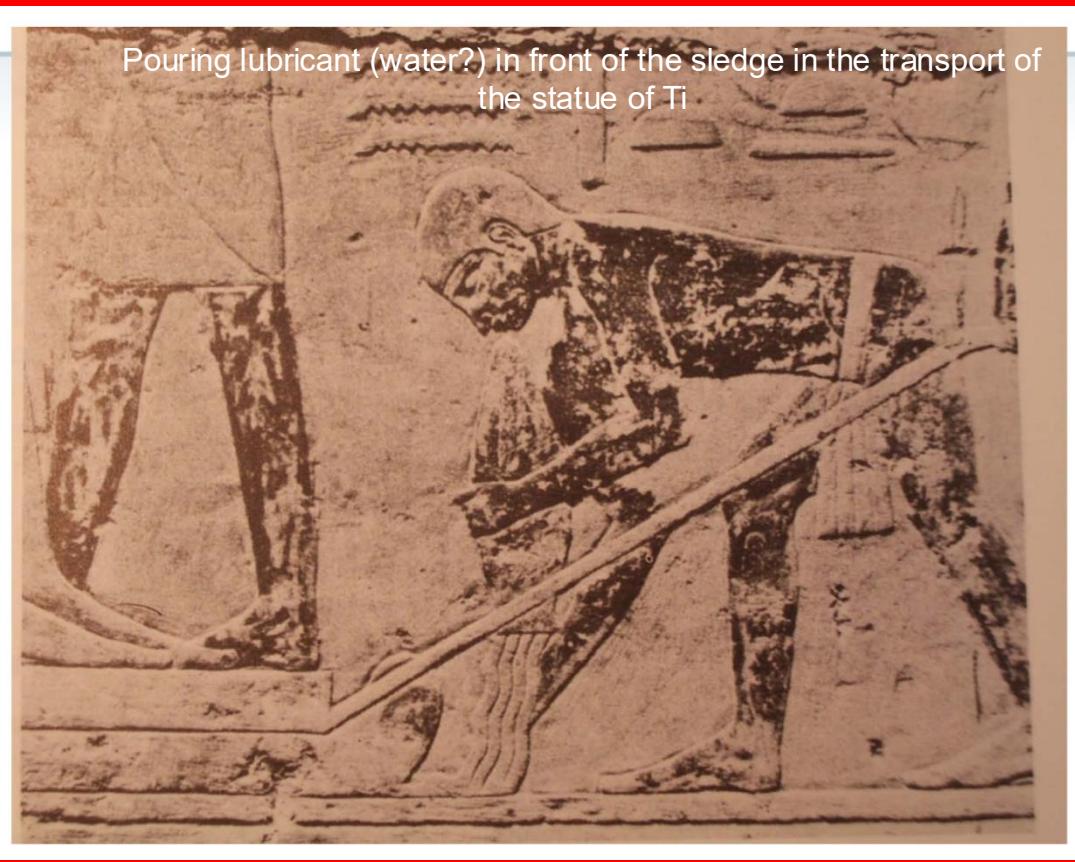
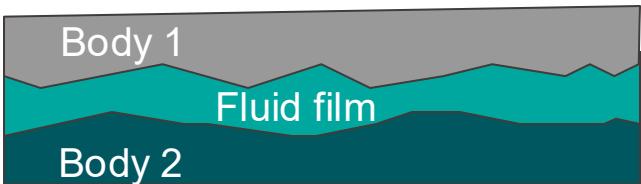


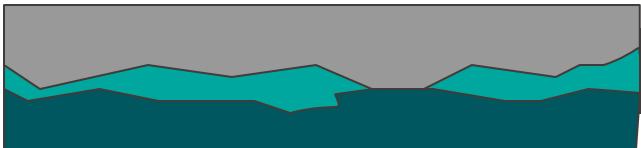
Figure taken from "History of Tribology" by Duncan Dowson (1993)

Regimes of fluid lubrication

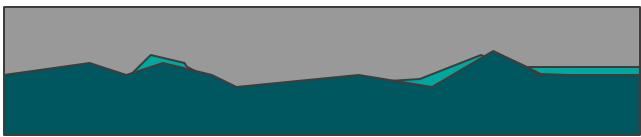


Hydrodynamic regime

The film is thick enough to entirely separate the two surfaces.



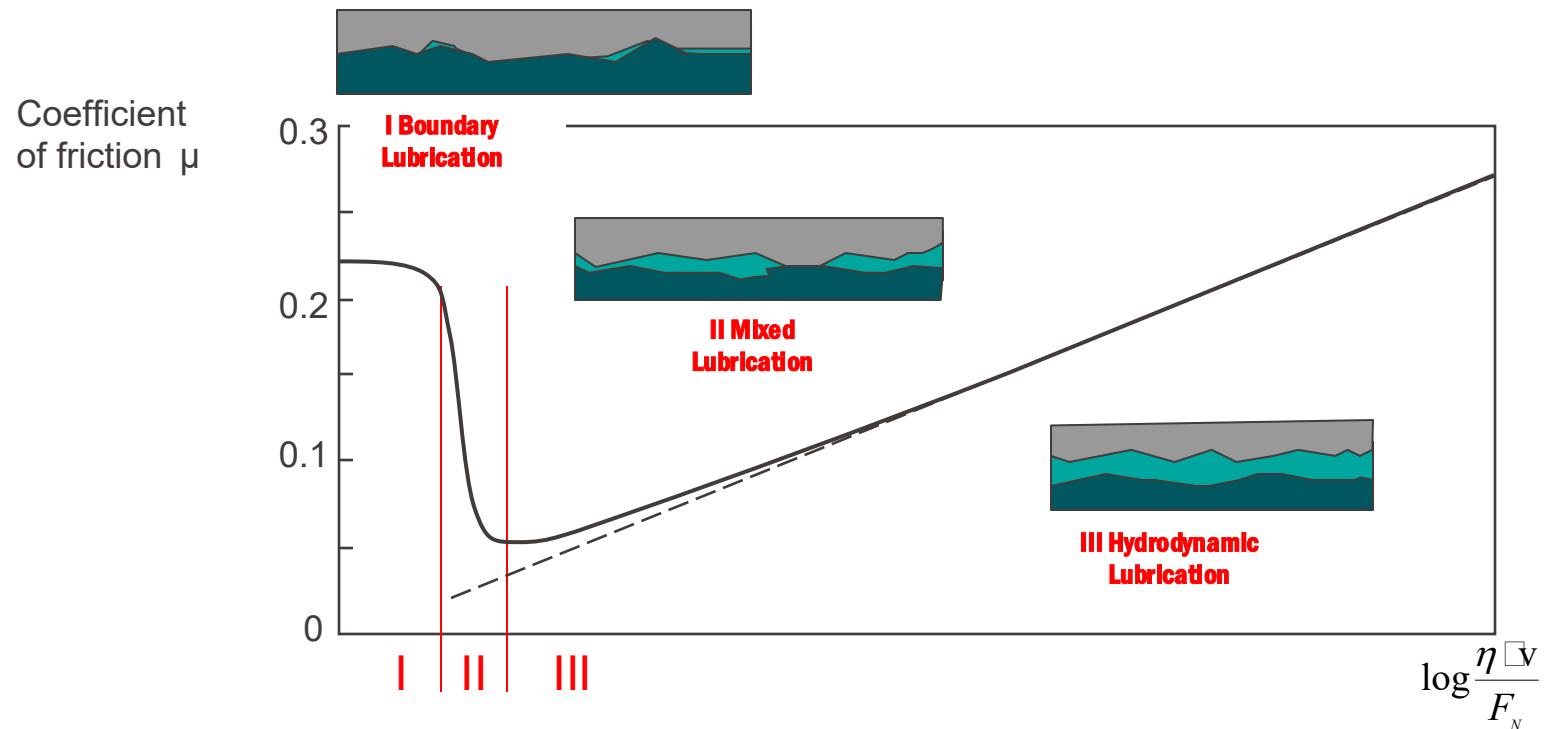
Mixed regime



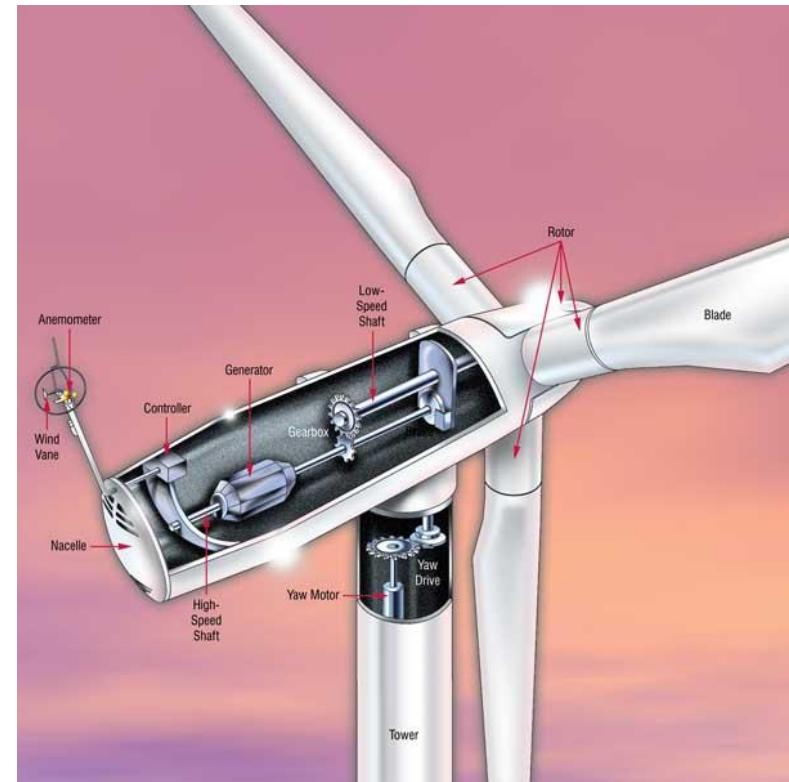
Boundary regime

The film is not thick enough to separate the two surfaces. The friction is determined by the contacts between asperities.

Fluid lubrication regimes: Stribeck curve



Back to low speed systems and high loads...



Source: Luo et al. *Friction* 8(4): 643–665 (2020)



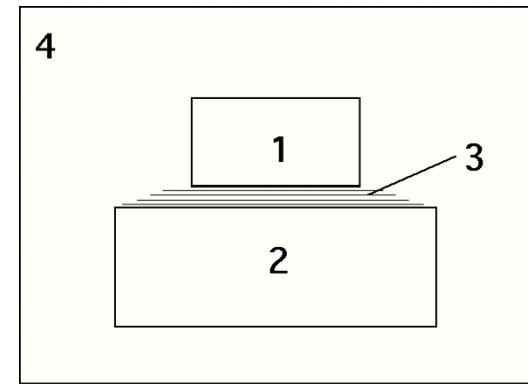
Fig. 17 Wear of gears in a wind turbine.

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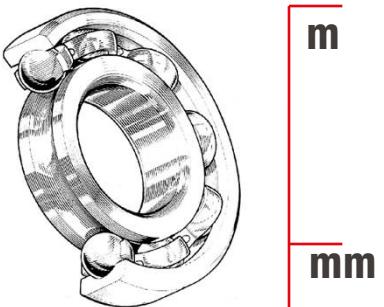
A tribology system is characterized by

- **Loading:**
 - Type of motion, normal force, speed, temperature...
- **System structure :**
 - Elements : body and counter body 1 et 2, lubricant 3, environment 4
 - Properties: geometry, materials, surfaces
- **Interactions** between the elements generate friction and wear and therefore may modify the structure of the system. For example:
 - Wear can change the geometry of elements, or
 - Heating due to friction can reduce the strength of a material in contact.



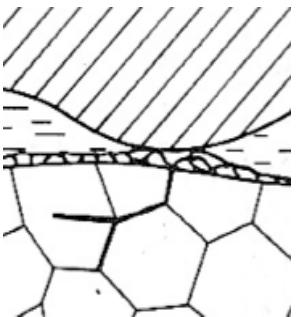
System approach to tribology: multi-scale, multi disciplinary

- System



- Macroscopic contact

- Microscopic contact



Mechanics

- Geometry,
- Loads, vibrations
- Motion
- Heat, mass transport
- Lubrication

Material science

- Elastic deformation
- Roughness
- Asperity deformation
- Structural transformation
- Cracking

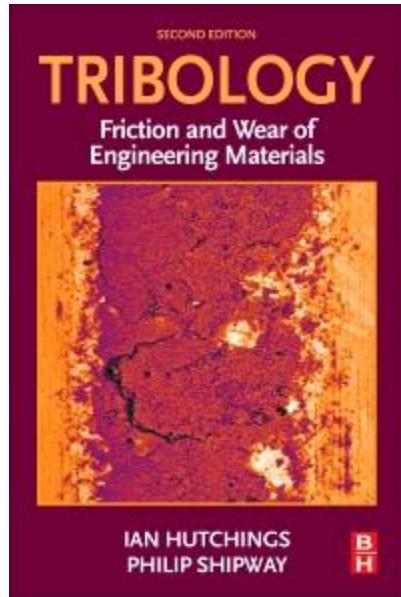
Surface chemistry and physics

- Surface reactions
- Third bodies
- Electrostatic repulsion

La tribologie apporte la richesse de sa complexité

Y. Berthier

Tribology Friction and Wear of Engineering Materials (2nd Edition)
Authors: Ian Hutchings Philip Shipway
ISBN: 9780081009512



- **Engineering Tribology**
J.A. Williams
Oxford University Press (1994) ISBN 0-19-856503-8 G.W.
- **Engineering Tribology**
Stachowiak et A.W. Batchelor
Elsevier (1993) ISBN 0-444-89235-4
- **Physical Analysis for Tribology**
T.F.J Quinn
Cambridge University Press (1991) ISBN 0-521-32602-8
- **Contact Mechanics**
K. L. Johnson
Cambridge University press, (1985) ISBN 0-521-34796-3