

Lab 4 Pre-Lab

- Express the signal $x(t) = 3e^{-t} \sin 2t$ as an exponentially decaying harmonic signal.

$$x(t) = 3e^{-t} \sin 2t = 3e^{-t} \cos\left(2t - \frac{\pi}{2}\right)$$

$$x(t) = Ae^{-\gamma t} \cos(\omega t - \theta)$$

- Find all local extrema of the exponentially decaying harmonic signal $x(t) = 3e^{-t/2} \cos(3t - 1)$ and determine the elapsed time between consecutive minima.

$$x(t) = 3e^{-\frac{t}{2}} \cos(3t - 1)$$

$$\begin{aligned} \dot{x}(t) &= -\frac{3}{2}e^{-\frac{t}{2}} \cos(3t - 1) - 9e^{-\frac{t}{2}} \sin(3t - 1) = -3e^{-\frac{t}{2}} \left[\frac{1}{2} \cos(3t - 1) + 3 \sin(3t - 1) \right] \\ &= -3e^{-\frac{t}{2}} \cdot \cos(3t - 1 + \tan^{-1}(-6)) \cdot \frac{\sqrt{37}}{2} \end{aligned}$$

$$\ddot{x}(t) = \frac{37}{4} \cdot 3e^{-\frac{t}{2}} \cdot \cos(3t - 1 + 2\tan^{-1}(-6))$$

$$\text{Let } \dot{x}(t) = 0 \Rightarrow t = \frac{1 + \tan^{-1}(6)}{3} + \frac{2k\pi}{3} \quad (\text{local minimum})$$

$$t = \frac{1 + \tan^{-1}(6)}{3} + \frac{\pi + 2l\pi}{3} \quad (\text{local maximum})$$

$$\therefore \text{elapsed time} = \frac{2\pi}{3}$$

- Determine the logarithmic decrement for the exponentially decaying harmonic signal $x(t) = e^{-3t} \cos(t + \pi/2)$ and relate this to the ratio between successive minima.

$$\begin{aligned} \dot{x}(t) &= -3e^{-3t} \cos(t + \frac{\pi}{2}) - e^{-3t} \sin(t + \frac{\pi}{2}) \\ &= -e^{-3t} [3 \cos(t + \frac{\pi}{2}) + \sin(t + \frac{\pi}{2})] = -e^{-3t} [\cos(t + \frac{\pi}{2} - \tan^{-1}(\frac{1}{3}))] \end{aligned}$$

$$\therefore t_n = \tan^{-1} \frac{1}{3} - \frac{\pi}{2} + k\pi$$

$$\therefore \frac{x(t_{n+1})}{x(t_n)} = \frac{e^{-3 \times \pi}}{e^{-3 \times 0}} = -e^{-3\pi}$$

4. Measurements on a mechanical system result in a signal that is dominated by the solution to the differential equation

$$2 \frac{d^2 x}{dt^2}(t) + 24 \frac{dx}{dt}(t) + 18x(t) = 2.$$

Determine whether the system is overdamped or underdamped.

$$2 \ddot{x}(t) + 24 \dot{x}(t) + 18x(t) = 2$$

$$\ddot{x}(t) + 12 \dot{x}(t) + 9x(t) = 1$$

$$\therefore \begin{cases} \omega_n^2 = 9 \\ 2\xi\omega_n = 12 \end{cases} \Rightarrow \begin{cases} \omega_n = 3 \\ \xi = 2 > 1 \end{cases} \therefore \text{Overdamped}$$

5. For what value of k is the mass-spring-damper system, governed by the differential equation

$$m \frac{d^2 x}{dt^2}(t) + c \frac{dx}{dt}(t) + kx(t) = f(t),$$

critically damped?

$$\ddot{x}(t) + \frac{c}{m} \dot{x}(t) + \frac{k}{m} x(t) = \frac{1}{m} f(t)$$

$$\begin{cases} 2\xi\omega_n = \frac{c}{m} \\ \omega_n^2 = \frac{k}{m} \end{cases} \Rightarrow \xi = \frac{c}{2\sqrt{mk}} = 1 \therefore k = \frac{c^2}{4m}$$

6. Graph the unit step response of a linear, second-order, time-invariant system with natural frequency $\omega_n = 1/2$ and damping ratio $\zeta = 1/\sqrt{2}$.

$$\ddot{x} + \frac{1}{\sqrt{2}} \dot{x} + \frac{1}{4} x = 0$$

For unit step response

$$x(t) = \frac{A}{\omega_n^2} \left[1 - e^{\frac{\zeta \omega_n t}{2}} \left(\cos \omega_d t + \frac{\zeta \omega_n}{\omega_d} \sin \omega_d t \right) \right]$$

$$= 4 \left[1 - e^{\frac{t}{2\sqrt{2}}} \left(\cos \frac{1}{2\sqrt{2}} t + \sin \frac{1}{2\sqrt{2}} t \right) \right]$$