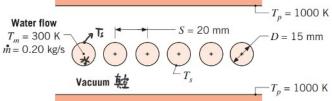
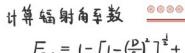
## ME 320 Homework 9

13.37 Water flowing through a large number of long, circular, thin-walled tubes is heated by means of hot parallel plates above and below the tube array. The space between the plates is evacuated, and the plate and tube surfaces may be approximated as blackbodies.



(a) Neglecting axial variations, determine the tube surface temperature,  $T_s$ , if water flows through each tube at a mass rate of  $\dot{m} = 0.20 \,\mathrm{kg/s}$  and a mean temperature of  $T_m = 300 \text{ K}$ .

根据 Table A-6, 假下= Tm=300K时 1/4 = 805 x 10 Ns/m2 k\* = 0.613 W/m·K  $P_r = 5.83$ 



$$F_{p-s} = 1 - \left[1 - \left(\frac{D}{S}\right)^{2}\right]^{\frac{1}{2}} + \left(\frac{D}{S}\right) \cdot \tan^{-1}\left[\sqrt{\frac{s^{2} - D^{2}}{D^{2}}}\right]$$

$$= 1 - \left[1 - \left(\frac{3}{4}\right)^{2}\right]^{\frac{1}{2}} + \frac{3}{4} \cdot \tan^{-1}\left[\sqrt{\frac{7}{9}}\right]$$

$$= 0.88$$

计算雷诺数 
$$R_e = \frac{U \cdot D}{\mu} = \frac{4 \dot{m}}{\pi D \mu} = 19855.58$$

Hint: Need to consider internal flow in each pipe. Thin wall means no need to

consider the conduction resistance of the pipe wall. Need to find the view factor

from the table. The final equation involving  $T_s$  cannot be solved easily, need to

trial-and-error a value of Ts and repeat until the difference in temperature

before the current and previous run is less than a 1 K.

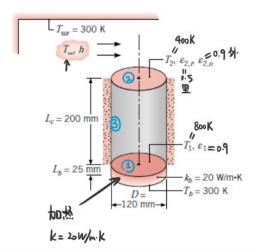
单根水管的能量守恒

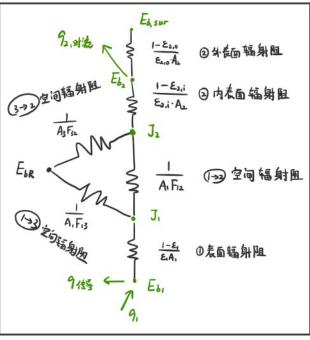
$$A_{p} \cdot F_{p-s} \cdot \sigma \cdot (T_{p}^{4} - T_{s}^{4}) = h \cdot A_{s} \cdot (T_{s} - T_{m})$$



13.96 Coated metallic disks are cured by placing them at the top of a cylindrical furnace whose bottom surface is electrically heated and whose sidewall may be approximated as a reradiating surface. Curing is accomplished by maintaining a disk at  $T_2 = 400$  K for an extended period. The electrically heated surface is maintained at  $T_1 = 800 \text{ K}$  and is mounted on a ceramic base material of thermal conductivity  $k = 20 \text{ W/m} \cdot \text{K}$ . The bottom of the base material, as well as the ambient air and large surroundings above the disk, are maintained at a temperature of 300 K. Emissivities of the heater and the disk inner and outer surfaces are  $\varepsilon_1 = 0.9$ ,  $\varepsilon_{2,1} = 0.5$ , and  $\varepsilon_{2,o} = 0.9$ , respectively.

Assuming steady-state operation and neglecting convection within the cylindrical cavity, determine the electric power that must be supplied to the heater\_and the convection coefficient h that must be maintained at the outer surface of the disk in order to satisfy the prescribed conditions.





Note: There is heat generation and heat conduction at the bottom of the cylinder. There is only radiation insider the cylinder. There is convection and radiation on the top of the cylinder.

根据热阻分析

$$9_{1} = \frac{E_{b_{1}} - E_{b_{2}}}{\frac{1 - E_{1}}{E_{1}A_{1}} + \frac{1 - E_{1,j}}{E_{2,j}A_{2}}} + \left[\left(\frac{1}{A_{1}F_{13}}\right)^{-1} + \left(\frac{1}{A_{1}F_{3}} + \overline{A_{2}F_{2}}\right)^{-1}\right]^{-1} + k \cdot A \cdot \frac{\Delta T}{\Delta X}$$

## 根据角系数公式



$$F_{12} = \frac{1}{2} \left( S - \sqrt{S^2 - 4(\frac{r_2}{r_1})^2} \right)$$

$$= \frac{1}{2} \left( 13.1 - \sqrt{13.1^2 - 4} \right) = 0.0768$$

×	F21 = F12 =0.0718	Fa
F12=0-0788	<b>X</b>	F3.2
F13=1-F12. = 0.923	F23=1-E1 = 0.923	F33

$$F_{13} = 1 - F_{12} = 0.923$$

$$F_{23} = 1 - F_{21} = 1 - F_{12} = 0.923$$

$$9 編集 = 9 新 + \frac{E_{b_2} - E_{sur}}{\frac{1 - E_{2,0}}{E_{1,0} \cdot A_2}}$$

$$82.9 W = h_0 \cdot A_{2} \cdot (T_{2} - T_{\infty}) + \mathcal{E}_{2,0} \cdot A_{2} \cdot \sigma \cdot (T_{2}^{4} - T_{SUr}^{4})$$

$$\downarrow \qquad \qquad \downarrow \qquad \qquad$$

$$= 64.37 \text{ W/m}^2.\text{K}$$