## ME 340 Dynamics of Mechanical Systems

# Lagrangian Dynamics Part 1

#### Recap

- To analyze the behavior of a mechanical system, we need to derive the equation(s) of motion.
- So far, our tools include

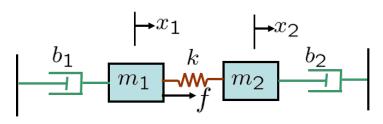
片=定律● Newton's 2<sup>nd</sup> law + free body diagram: planar systems

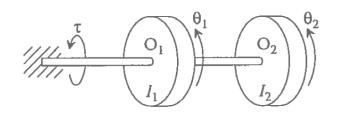
$$\sum F = m\ddot{x}$$
  $\sum \tau = I\ddot{\theta}$ 

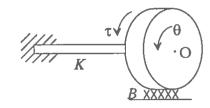
能量的 • Energy method: 1DoF conservative systems

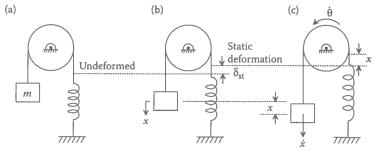
$$T + V = \text{constant} \Leftrightarrow \frac{d}{dt}(T + V) = 0$$

What we have seen...





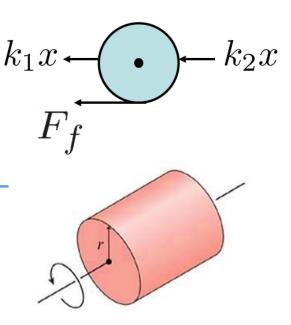




- The system:
  - A cylinder is connected with two springs.
  - Friction exists between the cylinder and ground.
- $k_1$   $k_2$   $k_2$

- Rolling without slipping.
- Free-body diagram
  - Translational:  $m\ddot{x} = -F_f k_1x k_2x$
  - Rotational: <u>I ö = R. F</u>
- Geometric constraint: x = R·P
- Equation of motion

$$\left(\frac{3m}{2}\right)\ddot{x} + (k_1 + k_2)x = 0$$



Disk or solid cylinder about its axis  $I = \frac{1}{2}MR^2$ 

Is energy method applicable?

Translation:  $m\ddot{x} = -F_f - k_1 x - k_2 x$ 

 $Rotation: I\ddot{\theta} = RF_f$ 

Geometric constraint:  $x = R\theta$ 

$$\frac{mR^2}{2}\ddot{\theta} = RF_f : F_f = \frac{m}{2}\ddot{x}$$

$$m\ddot{x} = -\frac{m}{2}\ddot{x} - k_1 x - k_2 x$$

$$\frac{3}{2}m\ddot{x} + (k_1 + k_2)x = 0$$

$$T + V = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}I\dot{\theta}^2 + \frac{1}{2}k_1x^2 + \frac{1}{2}k_2x^2$$

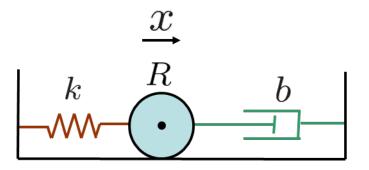
$$T + V = \frac{1}{2}m\dot{x}^2 + \frac{mR^2}{4}\left(\frac{\dot{x}}{R}\right)^2 + \frac{1}{2}(k_1 + k_2)x^2 = \frac{3}{4}m\dot{x}^2 + \frac{1}{2}(k_1 + k_2)x^2$$

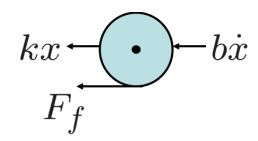
$$\frac{d}{dt}(T + V) = 0 = \frac{3}{2}m\dot{x}\ddot{x} + (k_1 + k_2)x\dot{x}$$

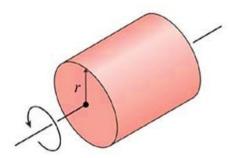
$$\frac{3}{2}m\ddot{x} + (k_1 + k_2)x = 0$$

- The system:
  - A cylinder is connected with a spring and a damper.
  - Friction exists between the cylinder and ground.
  - · Rolling without slipping.
- Free body diagram
  - Translational:  $m\ddot{x} = -F_f kx bx$
  - Rotational: <u>I = R F</u>
- Geometric constraint: χ= Rθ
- Equation of motion

$$\left(\frac{3m}{2}\right)\ddot{x} + b\dot{x} + kx = 0$$







Disk or solid cylinder about its axis  $I = \frac{1}{2}MR^2$ 

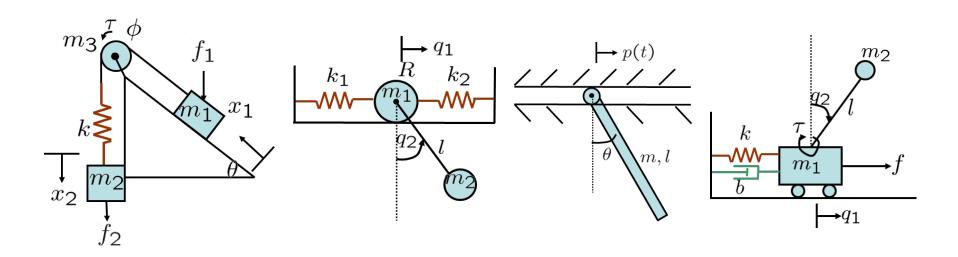
Translation:  $m\ddot{x} = -F_f - kx - bx$ 

 $Rotation: I\ddot{\theta} = RF_f$ 

Geometric constraint:  $x = R\theta$ 

$$\frac{3}{2}m\ddot{x} + b\dot{x} + kx = 0$$

#### What if we have the systems below...



- When dealing with complex mechanical systems with high number of degrees of freedom, 高自由度
  - Using Newton's laws requires free body diagrams for all elements
  - Energy method only applies to scenarios where the system is 1DoF and conservative
- We need a more powerful tool to analyze such systems.



## Lagrangian dynamics

## (田) 能量学恒法

- To apply energy approach:
  - All components have to be energy storing
  - No work done on the system
  - We only get one equation only good for 1 degree-of-freedom (DOF) systems
  - Extremely limiting

## (新) 拉格盷日动力学

- Lagrangian dynamics:
  - Based on both Energy and Work, e.g., work done by dampers
  - Can handle many generalized coordinates one equation for each DOF
  - Simpler in some cases over Free Body Diagram method

## Lagrangian dynamics

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

- $q_i$ : generalized coordinates,  $1 \leq i \leq N$
- $\dot{q}_i$ : generalized velocities
- T is the total Kinetic Energy in the system
- V is the total Potential Energy
- $Q_i$ : generalized non-conservative forces

 $q_i$  describe the configuration of the system relative to some reference configuration (position, angle, etc)

 $Q_i$  are forces (torque) that inject or remove energy

• L = T - V is called the Lagrangian of the system. Equivalently,

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = Q_i$$

## Review: partial derivative 偏微分复习

 A partial derivative of a function of multiple variables is its derivative with respect to one of those variables, with the others held constant.

(1) 
$$z = f(x,y) = x^2 + xy + 2y^2$$

(2) 
$$z = f(t) = x^2 + xy + 2y^2$$
  $\frac{\partial f}{\partial x}$ ?  $\frac{\partial f}{\partial y}$ ?  $\frac{\partial f}{\partial t}$ ?

(3) 
$$g(y,\theta) = mgx \sin \theta - mgy + \frac{1}{2}k(x+y)^2$$

(4) 
$$T(x, \dot{x}) = \frac{1}{2}m\dot{x}^2 + mgx$$
  $\frac{\partial T}{\partial \dot{x}}$ ?  $\frac{\partial T}{\partial x}$ ?  $\frac{\partial T}{\partial t}$ ?

## A simple example: spring-mass system



- $\star$   $q_1$ : position of the mass, only 1 DOF
- $\star$   $\dot{q}_1$ : velocity of the mass
- $\star T = \frac{1}{2}m\dot{q}_1^2$  is the total Kinetic Energy
- \*  $V = \frac{1}{2}kq_1^2 mgq_1$  is the total Potential Energy
- $\star Q_1 = 0$  no non-conservative forces

$$L = T - V = \frac{1}{2}m\dot{q}_1^2 - \frac{1}{2}kq_1^2 + mgq_1$$

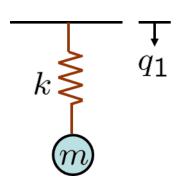
$$\frac{\partial L}{\partial q_1} = -kq_1 + mg$$

$$\frac{\partial L}{\partial \dot{q}_1} = m\dot{q}_1$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_1} \right) = m \ddot{q}_1$$

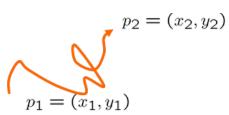
$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_1} \right) - \frac{\partial L}{\partial q_1} = Q_i = 0$$

$$m\ddot{q}_1 + kq_1 - mg = 0$$



## Kinetic and potential energy

- The kinetic energy of an object is the energy associated with its translational or rotational motion.
  - For example,  $T_{\text{trans}} = \frac{1}{2}m\dot{x}^2$ ,  $T_{\text{rot}} = \frac{1}{2}I_C\dot{\theta}^2$
- Potential energy:  $V = V_{\text{elastic}} + V_{\text{gravity}}$ , is the "stored" energy
  - *V*<sub>elastic</sub>: stored in springs
    - For example,  $V_{\rm elastic} = \frac{1}{2}kx^2$  (linear springs),  $V_{\rm elastic} = \frac{1}{2}K\theta^2$  (torsional springs)
  - $V_{\text{gravity}}$ : stored in mass due to gravitational field
    - $V_{gravity} = mgy$ , where y is taken w.r.t. some fixed point (datum)
    - ullet  $V_{
      m gravity}$  is "extra" potential energy from the gravity datum
  - Change in potential energy by taking the object from point A to point B is  $V_B-V_A$ 
    - Does not depend on the path
  - Work done by conservative forces
    - Independent of the path
    - Depends only on the end points



## Lagrangian dynamics

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

#### **General procedure for Lagrangian method:**

- Step 1: determine DOF, geometric constraints, and generalized coordinates  $q_i$ ;
- Step 2: write out the potential and kinetic energies;
- Step 3: calculate partial and full derivatives;
- Step 4: determine non-conservative generalized forces  $Q_i$ ;
- Step 5: assemble equations of motion.