

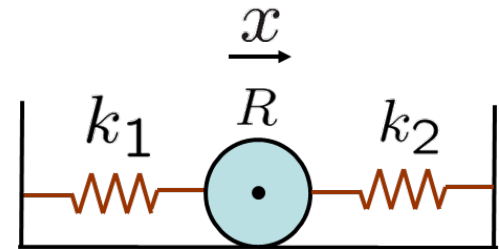
ME 340 Dynamics of Mechanical Systems

Lagrangian Dynamics Part 2

X Energy vs Lagrangian methods:

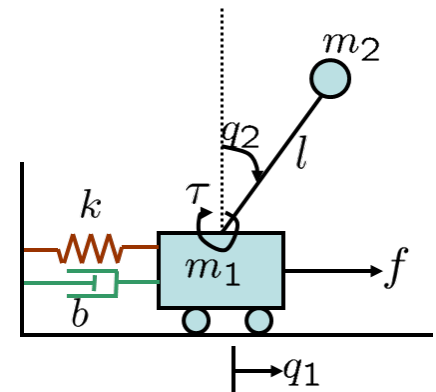
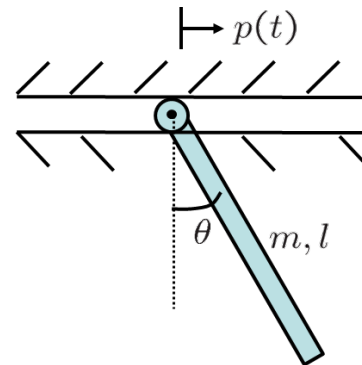
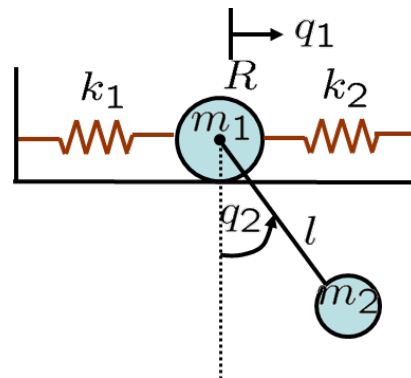
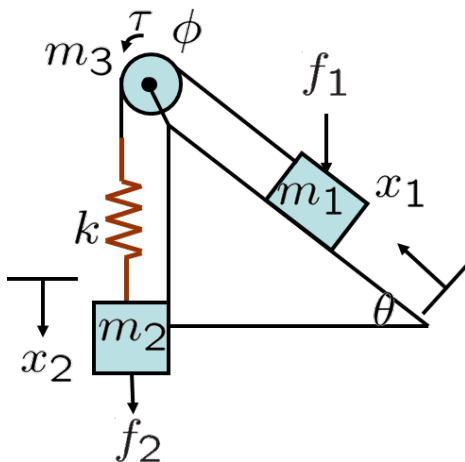
- Energy approach:

- All components are energy storing
- No work done on the system
- We only get one equation – only good for 1 DOF



- Lagrangian dynamics:

- Based on both Energy and Work, e.g., work done by dampers
- Many generalized coordinates – one equation for each DOF
- Simpler in some cases over Free Body Diagram method





Lagrangian dynamics

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

- q_i : generalized coordinates, $1 \leq i \leq N$
 - \dot{q}_i : generalized velocities
 - T is the total Kinetic Energy in the system
 - V is the total Potential Energy
 - Q_i : generalized non-conservative forces
- $L = T - V$ is called the Lagrangian of the system.
Equivalently,

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = Q_i$$

X Lagrangian dynamics

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

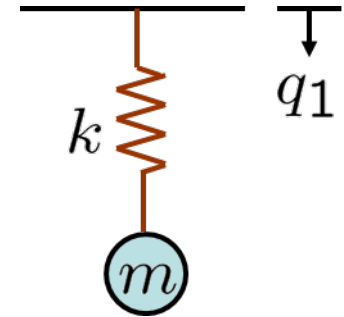
General procedure

- Step 1: determine DOF and generalized coordinates;
- Step 2: write out the potential and kinetic energy;
- Step 3: calculate derivatives;
- Step 4: determine non-conservative generalized forces;
- Step 5: derive Lagrangian equations.

A trivial example: spring-mass system

例题

- ★ q_1 : position of the mass, only 1 DOF
- ★ \dot{q}_1 : velocity of the mass
- ★ $T = \frac{1}{2}m\dot{q}_1^2$ is the total Kinetic Energy
- ★ $V = \frac{1}{2}kq_1^2 - mgq_1$ is the total Potential Energy
- ★ $Q_1 = 0$ no non-conservative forces



$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_1} \right) - \frac{\partial T}{\partial q_1} + \frac{\partial V}{\partial q_1} = Q_1$$

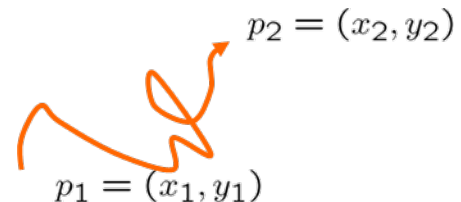
$$\Rightarrow \frac{d}{dt} \left(\frac{\partial}{\partial \dot{q}_1} \left(\frac{1}{2}m\dot{q}_1^2 \right) \right) - \frac{\partial}{\partial q_1} \left(\frac{1}{2}m\dot{q}_1^2 \right) + \frac{\partial}{\partial q_1} \left(\frac{1}{2}kq_1^2 - mgq_1 \right) = 0$$

$$\Rightarrow \frac{d}{dt} (m\dot{q}_1) - \frac{\partial}{\partial q_1} \left(\frac{1}{2}m\dot{q}_1^2 \right) + \frac{\partial}{\partial q_1} \left(\frac{1}{2}kq_1^2 - mgq_1 \right) = 0$$

$$\Rightarrow m\ddot{q}_1 + 0 + kq_1 - mg = 0$$

X Kinetic and potential energy

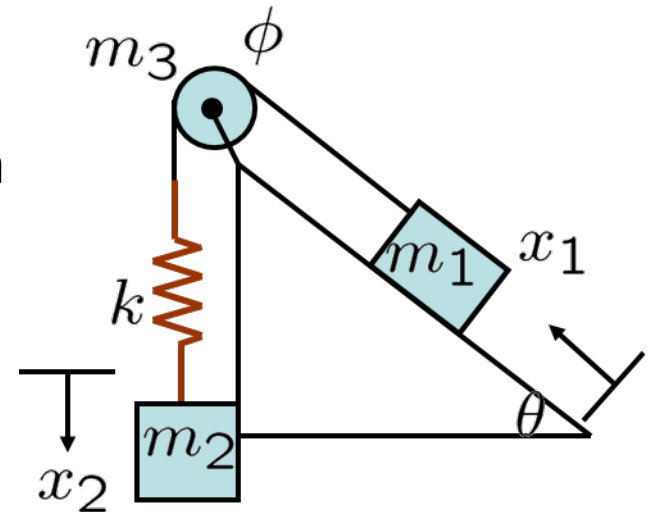
- The *kinetic energy* of an object is the energy that it possesses due to its motion.
 - For example, $\frac{1}{2}m\dot{x}^2$, $\frac{1}{2}I_C\dot{\theta}^2$
- *Potential energy*: $V = V_{\text{elastic}} + V_{\text{gravity}}$, “stored” energy
 - V_{elastic} : stored in springs
 - For example, $\frac{1}{2}kx^2$ (linear springs), $\frac{1}{2}K\theta^2$ (torsional springs)
 - V_{gravity} : stored in mass with potential field
 - mgh , where h is taken w.r.t. some fixed point (datum)
 - V_{gravity} is “extra” potential energy from the gravity datum
 - Change in potential energy by taking the object from point A to point B is $V_B - V_A$
 - Does not depend on the path
 - Work done by conservative forces
 - Independent of the path
 - Depends only on the end points



Wedge example

例题 2

- Three masses
- Smooth wedge surface \rightarrow no friction
- No external forces/torques



- DOF: 2
- Generalized coordinates

$$q_1 = x_1, q_2 = x_2, x_1 = R\phi \Rightarrow \dot{x}_1 = R\dot{\phi}$$

- Kinetic energy

$$T = \frac{1}{2}m_1\dot{x}_1^2 + \frac{1}{2}m_2\dot{x}_2^2 + \frac{1}{2} \overbrace{\left(\frac{1}{2}m_3R^2\right)}^{\text{Solid cylinder}} \left(\frac{\dot{x}_1}{R}\right)^2$$

- Potential energy

$$V = m_1gx_1 \sin \theta - m_2gx_2 + \frac{1}{2}k(x_1 - x_2)^2$$

- Generalized forces

$$Q_1 = 0, Q_2 = 0$$

Wedge example

- Three masses
- Smooth wedge surface \rightarrow no friction
- No external forces/torques

- Two equations

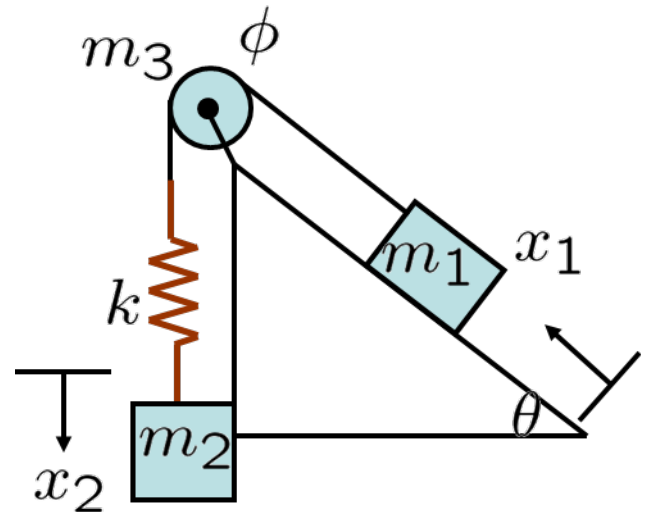
$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{x}_1} \right) - \frac{\partial T}{\partial x_1} + \frac{\partial V}{\partial x_1} = 0$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{x}_2} \right) - \frac{\partial T}{\partial x_2} + \frac{\partial V}{\partial x_2} = 0$$

- Equations of motion

$$\left(m_1 + \frac{m_3}{2} \right) \ddot{x}_1 + m_1 g \sin \theta + k(x_1 - x_2) = 0$$

$$m_2 \ddot{x}_2 - m_2 g + k(x_2 - x_1) = 0$$

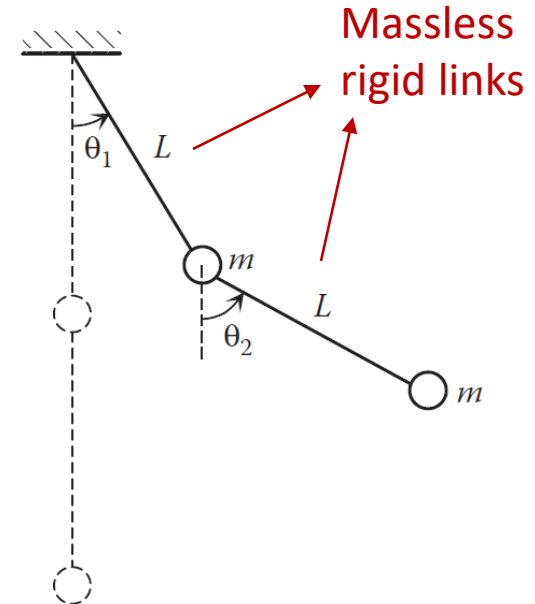


Example: double-pendulum system

双摆例题

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

- Two **point** masses
- DOF
- Generalized coordinates
- Kinetic energy
- Potential energy
- Generalized forces
- Equations of motion



Example: double-pendulum system

$$T = \frac{1}{2}mv_1^2 + \frac{1}{2}mv_2^2$$

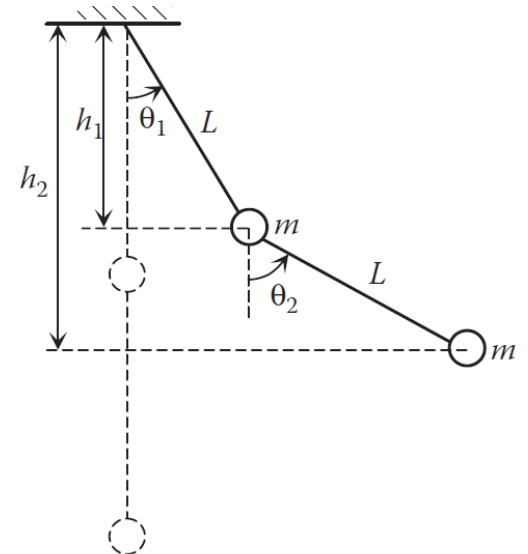
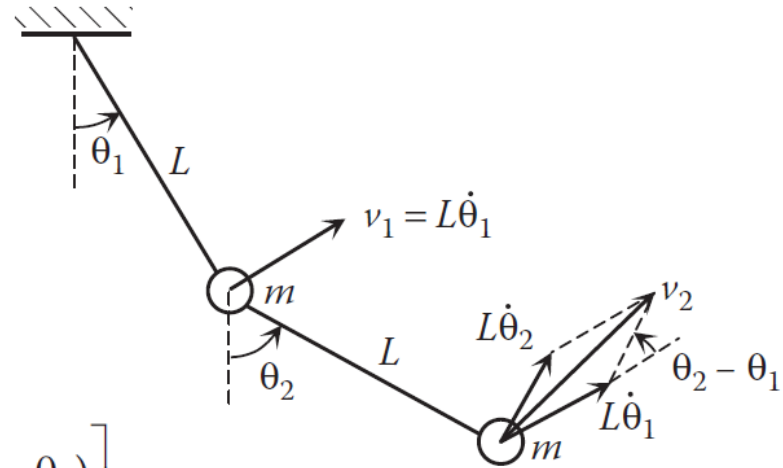
$$v_1 = L\dot{\theta}_1$$

$$v_2^2 = (L\dot{\theta}_1)^2 + (L\dot{\theta}_2)^2 + 2L^2\dot{\theta}_1\dot{\theta}_2 \cos(\theta_2 - \theta_1)$$

$$T = \frac{1}{2}m(L\dot{\theta}_1)^2 + \frac{1}{2}m[(L\dot{\theta}_1)^2 + (L\dot{\theta}_2)^2 + 2L^2\dot{\theta}_1\dot{\theta}_2 \cos(\theta_2 - \theta_1)]$$

$$V_g = -mgh_1 - mgh_2$$

$$V = -2mgL \cos \theta_1 - mgL \cos \theta_2$$



Example: double-pendulum system

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = 0, \quad i = 1, 2$$

$$\frac{\partial T}{\partial \dot{\theta}_1} = 2mL^2\dot{\theta}_1 + mL^2\dot{\theta}_2 \cos(\theta_2 - \theta_1)$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}_1} \right) = 2mL^2\ddot{\theta}_1 + mL^2\ddot{\theta}_2 \cos(\theta_2 - \theta_1) - mL^2\dot{\theta}_2 \sin(\theta_2 - \theta_1)(\dot{\theta}_2 - \dot{\theta}_1)$$

$$\frac{\partial T}{\partial \theta_1} = -mL^2\dot{\theta}_1\dot{\theta}_2 \sin(\theta_2 - \theta_1)(-1)$$

$$\frac{\partial V}{\partial \theta_1} = 2mgL \sin \theta_1$$

$$\frac{\partial T}{\partial \dot{\theta}_2} = mL^2\dot{\theta}_2 + mL^2\dot{\theta}_1 \cos(\theta_2 - \theta_1)$$

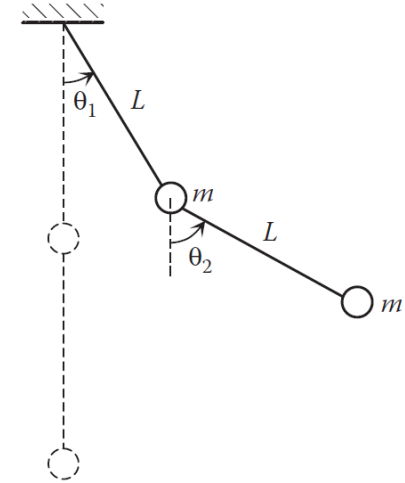
$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}_2} \right) = mL^2\ddot{\theta}_2 + mL^2\ddot{\theta}_1 \cos(\theta_2 - \theta_1) - mL^2\dot{\theta}_1 \sin(\theta_2 - \theta_1)(\dot{\theta}_2 - \dot{\theta}_1)$$

$$\frac{\partial T}{\partial \theta_2} = -mL^2\dot{\theta}_1\dot{\theta}_2 \sin(\theta_2 - \theta_1)$$

$$\frac{\partial V}{\partial \theta_2} = mgL \sin \theta_2$$

Example: double-pendulum system

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = 0, \quad i = 1, 2$$



$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}_1} \right) - \frac{\partial T}{\partial \theta_1} + \frac{\partial V}{\partial \theta_1} = 2mL^2\ddot{\theta}_1 + mL^2\ddot{\theta}_2 \cos(\theta_2 - \theta_1) - mL^2\dot{\theta}_2^2 \sin(\theta_2 - \theta_1) + 2mgL \sin \theta_1 = 0$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}_2} \right) - \frac{\partial T}{\partial \theta_2} + \frac{\partial V}{\partial \theta_2} = mL^2\ddot{\theta}_2 + mL^2\ddot{\theta}_1 \cos(\theta_2 - \theta_1) + mL^2\dot{\theta}_1^2 \sin(\theta_2 - \theta_1) + mgL \sin \theta_2 = 0$$