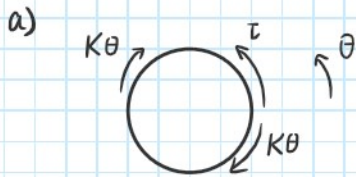
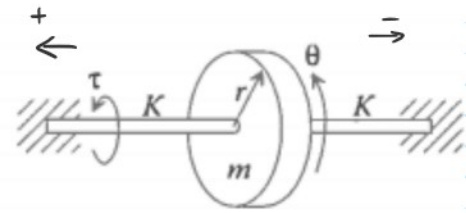


Homework 10

1. Consider the system shown by the figure on the right.
 - (a) Draw the free-body diagram. (5 points)
 - (b) Derive the equation of motion. (10 points)
 - (c) Determine the transfer function (assuming all initial conditions are zero). (5 points)



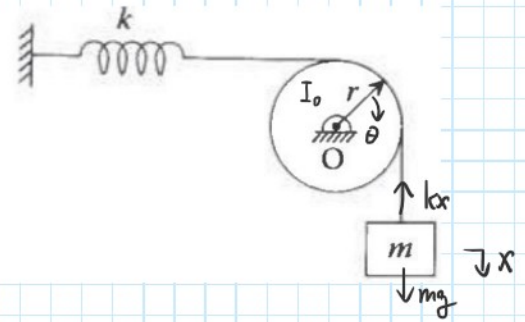
b).

$$I\ddot{\theta} = \tau - 2K\theta$$

c). $\ddot{\theta} + \frac{2K}{I}\theta = \frac{\tau}{I} \Rightarrow (I s^2 + 2K) X(s) = U(s)$

$$\therefore G(s) = \frac{X(s)}{U(s)} = \frac{1}{I s^2 + 2K}$$

2. Consider the pulley system as shown on the right. A block of mass m is connected to a translational spring of stiffness k through a cable, which passes by a pulley. The pulley rotates about a fixed mass center O . The moment of inertia of the pulley about its mass center is I_o . Determine the equation of motion using the energy method. (20 points)



$$T: \frac{1}{2}m\dot{x}^2 + \frac{1}{2}I_o\dot{\theta}^2 \quad r\theta = x$$

$$r\dot{\theta} = \dot{x}$$

$$V: -mgx + \frac{1}{2}kx^2 \quad r\ddot{\theta} = \ddot{x}$$

$$\therefore \text{Total energy } E = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}I_o\dot{\theta}^2 - mgx + \frac{1}{2}kx^2$$

$$\therefore \frac{dE}{dt} = m\dot{x}\ddot{x} + I_o\dot{\theta}\ddot{\theta} - mg\dot{x} + kx\dot{x} = 0$$

$$m\dot{x}\ddot{x} + \frac{I_o}{r^2}\dot{x}\ddot{x} - mg\dot{x} + kx\dot{x} = 0$$

$$(m + \frac{I_o}{r^2})\ddot{x} - mg + kx = 0$$

3. The double pulley system below has an inner radius of r_1 and an outer radius of r_2 . The mass moment of inertia of the pulley about the point O is I_o . A translational spring of stiffness k and a block of mass m are suspended by cables wrapped around the pulley as shown. Determine the equation of motion using the energy method. (30 points)

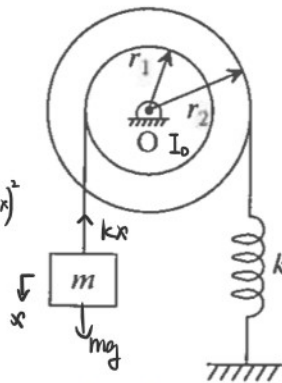
$$T = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}I_o\dot{\theta}^2 = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}\frac{I_o}{r_1^2}\dot{x}^2$$

$$V = -mgx + \frac{1}{2}k\left(\frac{r_2}{r_1}x\right)^2$$

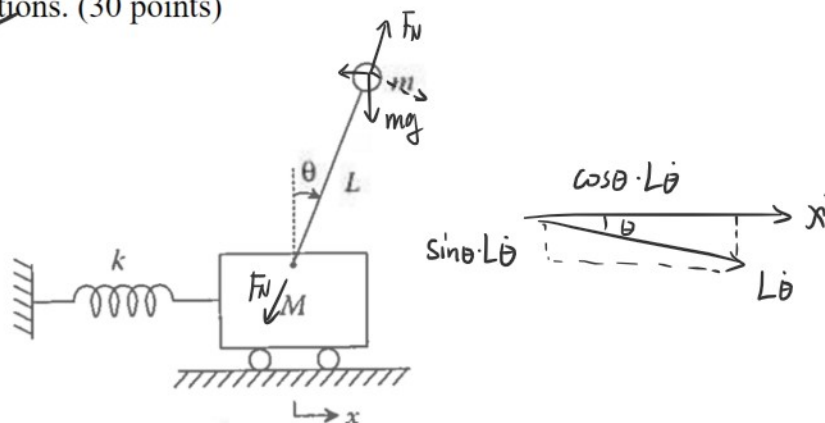
$$E = T + V = \frac{1}{2}\left(m + \frac{I_o}{r_1^2}\right)\dot{x}^2 - mgx + \frac{1}{2}k\left(\frac{r_2}{r_1}\right)^2x^2$$

$$\therefore \frac{dE}{dt} = \left(m + \frac{I_o}{r_1^2}\right)\dot{x}\ddot{x} - mg\dot{x} + \frac{r_2^2}{r_1^2}kx\dot{x} = 0$$

$$\therefore \left(m + \frac{I_o}{r_1^2}\right)\ddot{x} - mg + \frac{r_2^2}{r_1^2}kx = 0$$



4. Consider the following mechanical system, where a simple pendulum is pivoted on a cart of mass M . The pendulum consists of a point mass m concentrated at the tip of a massless rod of length L . The cart is connected to a translational spring of stiffness k . Denote the displacement of the cart as x and the angular displacement of the pendulum as θ . Derive the equations of motion using Lagrange's equations. (30 points)



$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} + \frac{\partial V}{\partial q_i} = Q_i$$

$$q_1 = x \quad q_2 = \theta \quad Q_1 = Q_2 = 0$$

$$T = \left(\frac{1}{2} M \dot{x}^2 \right) + \left[\frac{1}{2} m (\cos \theta L \dot{\theta} + \dot{x})^2 + \frac{1}{2} m (\sin \theta L \dot{\theta})^2 \right] = \frac{1}{2} (M+m) \dot{x}^2 + \frac{1}{2} m L^2 \dot{\theta}^2 + m \cos \theta L \dot{\theta} \dot{x}$$

$$V = \frac{1}{2} k x^2 - mg \cdot L (1 - \cos \theta) = \frac{1}{2} k x^2 + mg L \cos \theta - mg L$$

$$\text{For } q_1: \begin{cases} \frac{\partial T}{\partial x} = (M+m) \dot{x} + m L \cos \theta \cdot \dot{\theta} \Rightarrow \frac{d}{dt} \left(\frac{\partial T}{\partial \dot{x}} \right) = (M+m) \ddot{x} + [-m L \sin \theta \cdot \dot{\theta}^2 + m L \cos \theta \cdot \ddot{\theta}] \\ \frac{\partial T}{\partial x} = 0 \\ \frac{\partial V}{\partial x} = kx \end{cases}$$

$$\therefore (M+m) \ddot{x} - m L \sin \theta \cdot \dot{\theta}^2 + m L \cos \theta \cdot \ddot{\theta} + kx = 0$$

$$\text{For } q_2: \begin{cases} \frac{\partial T}{\partial \theta} = m L^2 \ddot{\theta} + m L \cos \theta \cdot \ddot{x} \Rightarrow \frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}} \right) = m L^2 \ddot{\theta} - m L \sin \theta \cdot \dot{\theta} \dot{x} + m L \cos \theta \cdot \ddot{x} \\ \frac{\partial T}{\partial \theta} = 0 \\ \frac{\partial V}{\partial \theta} = -mg \sin \theta \end{cases}$$

$$\therefore m L^2 \ddot{\theta} - m L \sin \theta \cdot \dot{\theta} \dot{x} + m L \cos \theta \cdot \ddot{x} - mg \sin \theta = 0$$