

Linear Systems

Linear Quadratic Optimal Control

Xu Chen



Motivation

state feedback control:

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- ▶ performance is implicit: we assign eigenvalues to induce proper error convergence

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linear quadratic (LQ) optimal regulation control, aka, LQ regulator (or LQR):

- ▶ no need to specify closed-loop poles
- ▶ performance is explicit: a performance index is defined ahead of time

1. Problem formulation
2. Solution to the infinite-horizon/stationary LQ problem
3. Solution to the finite-horizon LQ problem
4. From finite-horizon LQ to stationary LQ

Goal

Consider an n -dimensional state-space system

$$\begin{aligned}\dot{x}(t) &= Ax(t) + Bu(t), \quad x(t_0) = x_0 \\ y(t) &= Cx(t)\end{aligned}\tag{1}$$

where $x \in \mathbb{R}^n$, $u \in \mathbb{R}^r$, and $y \in \mathbb{R}^m$.

LQ optimal control aims at minimizing the performance index

$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(x^T(t)Qx(t) + u^T(t)Ru(t) \right) dt$$

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- ▶ often, $Q = C^T C \Rightarrow x^T(t)Qx(t) = y(t)^T y(t)$
- ▶ $u^T(t)Ru(t)$ penalizes large control efforts

Observations

$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

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Solution concept: infinite-horizon/stationary LQ

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- we defined $V(t) = \frac{1}{2} x^T(t) P x(t)$, $P = P^T$, such that

$$\begin{aligned} \bar{J} + V(\infty) - V(0) &= \frac{1}{2} \int_0^{\infty} x^T(t) Q x(t) dt + \int_0^{\infty} \dot{V}(t) dt \\ &= \frac{1}{2} \int_0^{\infty} x^T(t) \left(Q + A^T P + P A \right) x(t) dt \end{aligned}$$

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- ▶ yielding $\bar{J}^0 = \frac{1}{2} x^T(0) P_+ x(0)$ where P_+ comes from $A^T P + P A + Q = 0$, when the origin is asymptotically stable.

Solution of the infinite-horizon LQ

It turns out that for

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with $\dot{x}(t) = Ax(t) + Bu(t)$, $x(t_0) = x_0$ and $R \succ 0$:

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- ▶ and the closed-loop system is **asymptotically stable**, with

$$J_{\min} = J^0 = \frac{1}{2} x(t_0)^T P_+ x(t_0)$$

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- ▶ under the optimal control, the closed loop is given by

$$\dot{x} = Ax - BR^{-1}B^T Px = \underbrace{(A - BR^{-1}B^T P)}_{A_c} x \text{ and } J =$$
$$\frac{1}{2} \int_{t_0}^{\infty} (x^T Q x + u^T R u) dt = \frac{1}{2} \int_{t_0}^{\infty} x^T \underbrace{(Q + PBR^{-1}B^T P)}_{Q_c} x dt$$

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- ▶ for the above closed-loop system, the Lyapunov Eq. is

$$\begin{aligned} A_c^T P + P A_c &= -Q_c \\ \Leftrightarrow (A - BR^{-1}B^T P)^T P + P (A - BR^{-1}B^T P) &= -Q - PBR^{-1}B^T P \\ \Leftrightarrow A^T P + PA - PBR^{-1}B^T P &= -Q \Leftarrow \text{the ARE!} \end{aligned}$$

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$$A_c^T P + PA_c = -Q_c$$

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$$\Leftrightarrow A^T P + PA - PBR^{-1}B^T P = -Q \Leftarrow \text{the ARE!}$$

- ▶ when the ARE solution P_+ is positive definite, $\frac{1}{2}x^T P_+ x$ is a Lyapunov function for the closed-loop system

Observations

► Lyapunov Eq. and the ARE:

Cost	$\bar{J} = \frac{1}{2} \int_0^\infty x^T Q x dt$	$J = \frac{1}{2} \int_{t_0}^\infty (x^T Q x + u^T R u) dt$ $\dot{x} = Ax + Bu$
Syst. dynamics	$\dot{x} = Ax$	(A, B) controllable/stabilizable (A, C) observable/detectable
Key Eq.	$A^T P + PA + Q = 0$	$A^T P + PA - PBR^{-1}B^T P + Q = 0$
Optimal control	N/A	$u(t) = -R^{-1}B^T P_+ x(t)$
Opt. cost	$\bar{J}^0 = \frac{1}{2} x^T(0) P_+ x(0)$	$J^0 = \frac{1}{2} x(t_0)^T P_+ x(t_0)$

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Opt. cost	$\bar{J}^0 = \frac{1}{2} x^T(0) P_+ x(0)$	$J^0 = \frac{1}{2} x(t_0)^T P_+ x(t_0)$

- the guaranteed closed-loop stability is an attractive feature
- more nice properties will show up later

Example

Item 4: From the algebraic Riccati equation, we have

$$\begin{aligned} A^T P_+ + P_+ A - P_+ B R^{-1} B^T P_+ + C^T C + \lambda P_+ - \lambda P_+ &= 0, \\ \Rightarrow (\lambda I_n + A^T) P_+ - P_+ (\lambda I_n + A) - P_+ B R^{-1} B^T P_+ + C^T C &= 0, \\ \Rightarrow -P_+ [\lambda_0 - (A - B R^{-1} B^T P_+)] + (\lambda I_n + A^T) P_+ + C^T C &= 0 \end{aligned}$$

Let f_i be the eigenvector of $A - B R^{-1} B^T P_+$ associated with the eigenvalue λ_i . Set $\lambda = \lambda_i$ in the last equality and multiply f_i from the right. Then, the first term vanishes after multiplication:

$$(\lambda_i I_n + A^T) P_+ f_i + C^T C f_i = 0.$$

Then,

$$H \begin{bmatrix} f_i \\ P_+ f_i \end{bmatrix} = \begin{bmatrix} A f_i - B R^{-1} B^T P_+ f_i \\ -C^T C f_i - A^T P_+ f_i \end{bmatrix} = \lambda_i \begin{bmatrix} f_i \\ P_+ f_i \end{bmatrix}.$$

This implies that $\begin{bmatrix} f_i \\ P_+ f_i \end{bmatrix}$ is the eigenvector of H associated with a stable eigenvalue λ_i . (iv) follows from this fact. \square

13.4.4 Example: Inverted Pendulum on a Cart

The inverted pendulum on a cart model is widely used and applied to many systems we see regularly. It is a classical problem in dynamics and is used extensively in control theory for designing controllers. Applications include rocket balancing, seaway and hoverboards, vertical robots, to name a few.

The system has two equations of motion:

$$(M + m)\ddot{x} + b\dot{x} + m l \ddot{\theta} \cos \theta = F, \quad (13.20)$$

$$(I + m l^2) \ddot{\theta} + m g l \sin \theta = -m l \ddot{x} \cos \theta, \quad (13.21)$$

where I is the moment of inertia of the pendulum, m is the mass of the pendulum, M is the mass of the cart, l is the length from the pendulum center of mass to the mounting joint, and b is the damping of the cart in the horizontal movement direction. Substituting for $\ddot{\theta}$ in 13.20 from 13.21 gives:

$$\ddot{x} = \frac{F(I + m l^2) - b \dot{x}(I + m l^2) - m^2 l^2 g \sin \theta \cos \theta + m l \dot{\theta}^2 \sin \theta (I + m l^2)}{(I + m l^2)(M + m) - m^2 l^2 \cos^2(\theta)}, \quad (13.22)$$

$$\ddot{\theta} = \frac{(M + m) m g l \sin \theta + m l b \dot{x} \cos \theta - m^2 l^2 \dot{\theta}^2 \cos \theta \sin \theta - m l F \cos \theta}{(M + m)(I + m l^2) - m^2 l^2 \cos^2 \theta}. \quad (13.23)$$

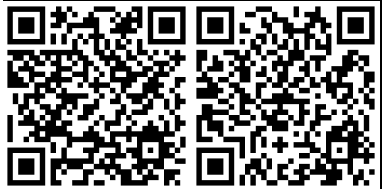
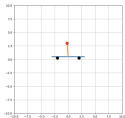
The system model has four states, which give the state vector:

$$X = \begin{bmatrix} x \\ \dot{x} \\ \theta \\ \dot{\theta} \end{bmatrix}.$$

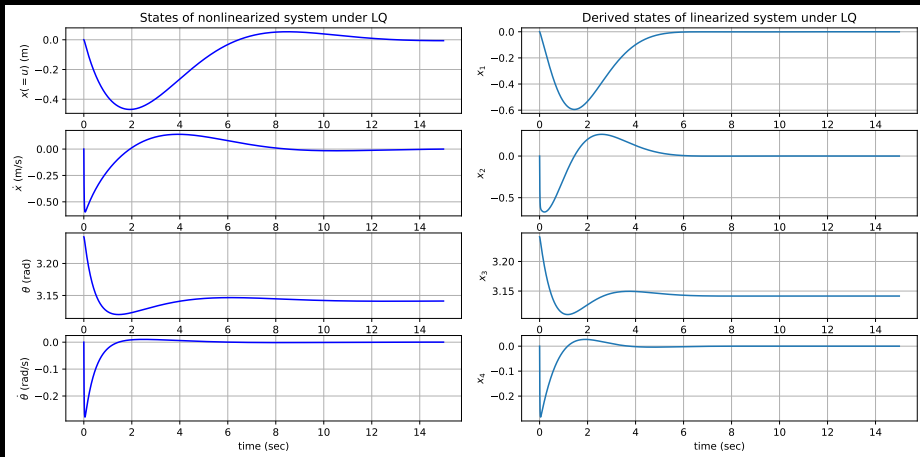
The derivation of the equations of motion is available at this link:

<https://ctms.engin.umich.edu/CTMS/index.php?example=InvertedPendulum§ion=SystemModeling>.

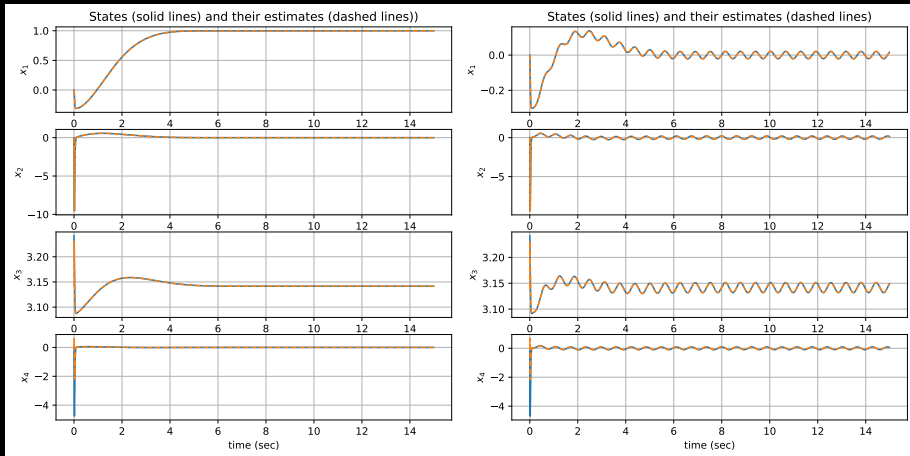
An animated version of the example in Python is provided at <https://github.com/macis-lab/Python-Controls-Visualization/tree/main>



Example



LQ with State Feedback



Example: Stationary LQR of a pure inertia system

► Consider

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u, \quad J = \frac{1}{2} \int_0^\infty \left(x^T \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} x + Ru^2 \right) dt, \quad R > 0$$

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- the ARE is

$$0 = \begin{bmatrix} 0 & 0 \\ 1 & 0 \end{bmatrix} P + P \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} - P \begin{bmatrix} 0 \\ 1 \end{bmatrix} \frac{1}{R} \begin{bmatrix} 0 & 1 \end{bmatrix} P \Rightarrow P_+ = \begin{bmatrix} \sqrt{2}R^{1/4} & R^{1/2} \\ R^{1/2} & \sqrt{2}R^{3/4} \end{bmatrix}$$

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- the closed-loop A matrix can be computed to be

$$A_c = A - BR^{-1}B^T P_+ = \begin{bmatrix} 0 & 1 \\ -R^{-1/2} & -\sqrt{2}R^{-1/4} \end{bmatrix}$$

- \Rightarrow closed-loop eigenvalues:

$$\lambda_{1,2} = -\frac{1}{\sqrt{2}R^{1/4}} \pm \frac{1}{\sqrt{2}R^{1/4}}j$$

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u, \quad J = \frac{1}{2} \int_0^\infty \left(x^T \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} x + Ru^2 \right) dt$$



Figure: Eigenvalue $\lambda_{1,2} = -\frac{1}{\sqrt{2}R^{1/4}} \pm \frac{1}{\sqrt{2}R^{1/4}}j$ evolution (root locus)

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Figure: Eigenvalue $\lambda_{1,2} = -\frac{1}{\sqrt{2}R^{1/4}} \pm \frac{1}{\sqrt{2}R^{1/4}}j$ evolution (root locus)

- $R \uparrow$ (more penalty on the control input) $\Rightarrow \lambda_{1,2}$ move closer to the origin \Rightarrow slower state convergence to zero

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Figure: Eigenvalue $\lambda_{1,2} = -\frac{1}{\sqrt{2}R^{1/4}} \pm \frac{1}{\sqrt{2}R^{1/4}}j$ evolution (root locus)

- $R \uparrow$ (more penalty on the control input) $\Rightarrow \lambda_{1,2}$ move closer to the origin \Rightarrow slower state convergence to zero
- $R \downarrow$ (allow for large control efforts) $\Rightarrow \lambda_{1,2}$ move further to the left of the complex plane \Rightarrow faster speed of closed-loop dynamics

MATLAB commands

- *care*: solves the ARE for a continuous-time system:

$$[P, \Lambda, K] = \text{care}(A, B, C^T C, R)$$

where $K = R^{-1}B^T P$ and Λ is a diagonal matrix with the closed-loop eigenvalues, i.e., the eigenvalues of $A - BK$, in the diagonal entries.

MATLAB commands

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where $K = R^{-1}B^T P$ and Λ is a diagonal matrix with the closed-loop eigenvalues, i.e., the eigenvalues of $A - BK$, in the diagonal entries.

- *lqr* and *lqry*: provide the LQ regulator with

$$[K, P, \Lambda] = \text{lqr}(A, B, C^T C, R)$$

$$[K, P, \Lambda] = \text{lqry}(\text{sys}, Q_y, R)$$

where *sys* is defined by $\dot{x} = Ax + Bu$, $y = Cx + Du$, and

$$J = \frac{1}{2} \int_0^\infty (y^T Q_y y + u^T R u) dt$$

1. Problem formulation
2. Solution to the infinite-horizon/stationary LQ problem
3. Solution to the finite-horizon LQ problem
4. From finite-horizon LQ to stationary LQ

Solution to the finite-horizon LQ

Consider the performance index

$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

with $\dot{x} = Ax + Bu$, $x(t_0) = x_0$, $S \succeq 0$, $R \succ 0$, and $Q = C^T C$.

Solution to the finite-horizon LQ

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with $\dot{x} = Ax + Bu$, $x(t_0) = x_0$, $S \succeq 0$, $R \succ 0$, and $Q = C^T C$.

► do a similar Lyapunov construction: $V(t) \triangleq \frac{1}{2}x^T(t)P(t)x(t)$

Solution to the finite-horizon LQ

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$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

with $\dot{x} = Ax + Bu$, $x(t_0) = x_0$, $S \succeq 0$, $R \succ 0$, and $Q = C^T C$.

- ▶ do a similar Lyapunov construction: $V(t) \triangleq \frac{1}{2}x^T(t)P(t)x(t)$
- ▶ then

$$\begin{aligned} \frac{d}{dt}V(t) &= \frac{1}{2}\dot{x}^T(t)P(t)x(t) + \frac{1}{2}x^T(t)\dot{P}(t)x(t) + \frac{1}{2}x^T(t)P(t)\dot{x}(t) \\ &= \frac{1}{2}(Ax + Bu)^T P x + \frac{1}{2}x^T \frac{dP}{dt} x + \frac{1}{2}x^T P (Ax + Bu) \\ &= \frac{1}{2} \left\{ x^T(t) \left(A^T P + \frac{dP}{dt} + PA \right) x(t) + u^T B^T P x + x^T P B u \right\} \end{aligned}$$

Solution to the finite-horizon LQ

with $\frac{d}{dt}V(t)$ from the last slide, we have

$$\begin{aligned} V(t_f) - V(t_0) &= \int_{t_0}^{t_f} \dot{V} dt \\ &= \frac{1}{2} \int_{t_0}^{t_f} \left(x^T \left(A^T P + PA + \frac{dP}{dt} \right) x + u^T B^T P x + x^T P B u \right) dt \end{aligned}$$

Solution to the finite-horizon LQ

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► adding

$$J = \frac{1}{2} x^T(t_f) S x(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(x^T(t) Q x(t) + u^T(t) R u(t) \right) dt$$

to both sides yields

$$\begin{aligned} J + V(t_f) - V(t_0) &= \frac{1}{2} x^T(t_f) S x(t_f) + \\ &\frac{1}{2} \int_{t_0}^{t_f} \left(x^T \left(A^T P + PA + Q + \frac{dP}{dt} \right) x + \underbrace{u^T B^T P x + x^T P B u}_{\text{products of } x \text{ and } u} + \underbrace{u^T R u}_{\text{quadratic}} \right) dt \end{aligned}$$

Solution to the finite-horizon LQ

- “complete the squares” in $\underbrace{u^T B^T P x + x^T P B u}_{\text{products of } x \text{ and } u} + \underbrace{u^T R u}_{\text{quadratic}}$:

$$\begin{aligned} & u^T B^T P x + x^T P B u + u^T R u \stackrel{\text{scalar case}}{=} R u^2 + 2 u B P x \\ &= R u^2 + 2 \left(x P B R^{-1/2} \right) \underbrace{R^{1/2} u}_{\sqrt{R u^2}} + \left(R^{-1/2} B P x \right)^2 - \left(R^{-1/2} B P x \right)^2 \\ &= \left(R^{1/2} u + R^{-1/2} B P x \right)^2 - \left(R^{-1/2} B P x \right)^2 \end{aligned}$$

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- extending the concept to the general vector case:

$$u^T B^T P x + x^T P B u + u^T R u = \left\| R^{\frac{1}{2}} u + R^{-\frac{1}{2}} B^T P x \right\|_2^2 - x^T P B R^{-1} B^T P x$$

Solution to the finite-horizon LQ

$$J + V(t_f) - V(t_0) = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(x^T \left(A^T P + PA + Q + \frac{dP}{dt} \right) x + u^T B^T P x + x^T P B u + u^T R u \right) dt$$

⇓ “completing the squares”

$$J + \frac{1}{2}x^T(t_f)P(t_f)x(t_f) - \frac{1}{2}x^T(t_0)P(t_0)x(t_0) = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(\underline{\underline{x^T \left(\frac{dP}{dt} + A^T P + PA + Q - PBR^{-1}B^T P \right) x}} + \underline{\underline{\|R^{\frac{1}{2}}u + R^{-\frac{1}{2}}B^T P x\|_2^2}} \right) dt$$

Solution to the finite-horizon LQ

$$J + V(t_f) - V(t_0) = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(x^T \left(A^T P + PA + Q + \frac{dP}{dt} \right) x + u^T B^T P x + x^T P B u + u^T R u \right) dt$$

⇓ “completing the squares”

$$J + \frac{1}{2}x^T(t_f)P(t_f)x(t_f) - \frac{1}{2}x^T(t_0)P(t_0)x(t_0) = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} \left(x^T \left(\frac{dP}{dt} + A^T P + PA + Q - PBR^{-1}B^T P \right) x + \underbrace{\|R^{\frac{1}{2}}u + R^{-\frac{1}{2}}B^T P x\|_2^2}_{\geq 0} \right) dt$$

► the best that the control can do in minimizing the cost is to have

$$\underline{\underline{u(t) = -K(t)x(t) = -R^{-1}B^T P(t)x(t)}} \\ -\frac{dP}{dt} = \underline{\underline{A^T P + PA - PBR^{-1}B^T P + Q}}, \quad \underline{P(t_f) = S}$$

to yield the optimal cost $J^0 = \frac{1}{2}x_0^T P(t_0)x_0$

Observations

$u(t) = -K(t)x(t) = -R^{-1}B^T P(t)x(t)$ optimal state feedback control

$-\frac{dP}{dt} = A^T P + PA - PBR^{-1}B^T P + Q, P(t_f) = S$ the Riccati differential equation

- boundary condition of the Riccati equation is given at the final time $t_f \Rightarrow$ the equation must be integrated backward in time

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- ▶ boundary condition of the Riccati equation is given at the final time $t_f \Rightarrow$ the equation must be integrated backward in time
- ▶ *backward* integration of

$$-\frac{dP}{dt} = A^T P + PA + Q - PBR^{-1}B^T P, P(t_f) = S$$

is equivalent to the *forward* integration of

$$\frac{dP^*}{dt} = A^T P^* + P^* A + Q - P^* B R^{-1} B^T P^*, P^*(0) = S \quad (2)$$

by letting $P(t) = P^*(t_f - t)$

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by letting $P(t) = P^*(t_f - t)$

- ▶ Eq. (2) can be solved by numerical integration

Observations

$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

$$J^0 = \frac{1}{2}x_0^T P(t_0)x_0$$

- the minimum value J^0 is a function of the initial state $x(t_0)$

Observations

$$J = \frac{1}{2}x^T(t_f)Sx(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t)Qx(t) + u^T(t)Ru(t)) dt$$

$$J^0 = \frac{1}{2}x_0^T P(t_0)x_0$$

- ▶ the minimum value J^0 is a function of the initial state $x(t_0)$
- ▶ J (and hence J^0) is nonnegative $\Rightarrow P(t_0)$ is at least positive semidefinite

Observations

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- ▶ t_0 can be taken anywhere in $(0, t_f) \Rightarrow P(t)$ is at least positive semidefinite for any t

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$$J^0 = \frac{1}{2}x_0^T P(t_0)x_0$$

- ▶ the minimum value J^0 is a function of the initial state $x(t_0)$
- ▶ J (and hence J^0) is nonnegative $\Rightarrow P(t_0)$ is at least positive semidefinite
- ▶ t_0 can be taken anywhere in $(0, t_f) \Rightarrow P(t)$ is at least positive semidefinite for any t
- ▶ the state feedback law is time varying because of $P(t)$

Example: LQR of a pure inertia system

Consider

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u, \quad J = \frac{1}{2} x^T(t_f) S x(t_f) + \frac{1}{2} \int_0^{t_f} (x^T Q x + R u^2) dt$$

$$\text{where } S = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}, \quad Q = \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix}, \quad R > 0$$

Example: LQR of a pure inertia system

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$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u, \quad J = \frac{1}{2} x^T(t_f) S x(t_f) + \frac{1}{2} \int_0^{t_f} (x^T Q x + R u^2) dt$$

where $S = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, $Q = \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix}$, $R > 0$

► we let $P(t) = P^*(t_f - t)$ and solve

$$\frac{dP^*}{dt} = A^T P^* + P^* A + Q - P^* B R^{-1} B^T P^*, \quad P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$\Leftrightarrow \frac{dP^*}{dt} = \begin{bmatrix} 0 & 0 \\ 1 & 0 \end{bmatrix} P^* + P^* \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} - P^* \begin{bmatrix} 0 \\ 1 \end{bmatrix} \frac{1}{R} \begin{bmatrix} 0 & 1 \end{bmatrix} P^*$$

Example: LQR of a pure inertia system

Consider

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u, \quad J = \frac{1}{2} x^T(t_f) S x(t_f) + \frac{1}{2} \int_0^{t_f} (x^T Q x + R u^2) dt$$

where $S = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, $Q = \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix}$, $R > 0$

► we let $P(t) = P^*(t_f - t)$ and solve

$$\frac{dP^*}{dt} = A^T P^* + P^* A + Q - P^* B R^{-1} B^T P^*, \quad P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$\Leftrightarrow \frac{dP^*}{dt} = \begin{bmatrix} 0 & 0 \\ 1 & 0 \end{bmatrix} P^* + P^* \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} - P^* \begin{bmatrix} 0 \\ 1 \end{bmatrix} \frac{1}{R} \begin{bmatrix} 0 & 1 \end{bmatrix} P^*$$

► letting

$$P^* = \begin{bmatrix} p_{11}^* & p_{12}^* \\ p_{12}^* & p_{22}^* \end{bmatrix} \Rightarrow \begin{cases} \frac{d}{dt} p_{11}^* = 1 - \frac{1}{R} (p_{12}^*)^2 & p_{11}^*(0) = 1 \\ \frac{d}{dt} p_{12}^* = p_{11}^* - \frac{1}{R} p_{12}^* p_{22}^* & p_{12}^*(0) = 0 \\ \frac{d}{dt} p_{22}^* = 2p_{12}^* - \frac{1}{R} (p_{22}^*)^2 & p_{22}^*(0) = 1 \end{cases}$$

Example: LQR of a pure inertia system

Example 12.2.1 (Motor Control) To see an example performance of the observer design, we apply the algorithm to a motor system and program Equation 12.10. Here, the three states are the current in the motor electronics, the angular position and angular velocity of the motor. Only the angular position is directly measured on the output side. We compute first the plant eigenvalues, then check observability and place the observer eigenvalues.

```
% observer/motorsbs.m
% State observer design for motion control in MATLAB
%% Continuous-time system model
% motor parameters
L = 1e-3; R = 1; J = 5e-5; B = 1e-4; K = 0.1;

% state-space model
A = [-R/L, 0, -K/L; 0, 0, 1; K/J, 0, -B/J];
B = [1/L; 0; 0];
C = [0, 1, 0];
D = [0];

% check original eigenvalues
eig(A)

%% Observer design
% check observability
O = obsv(A,C);
rank(O)

% desired poles for the observer
pole_des = [-500-250j, -500+250j, -1000];

% design observer by placing poles of A-LC
lt = place(A, C', pole_des);
L = lt';

% check poles of estimator-error dynamics
est_poles = eig(A - L*C)

%% Simulation
% define augmented system to run the simulation
Aaug = [A, zeros(3,3); L*C, A-L*C];
Baug = [B; 0];
Caug = [C, zeros(1,3)];
Daug = 0;
sys = ss(Aaug, Baug, Caug, Daug);

% define initial conditions
x0 = [10, 2, 10]'; xhat0 = [0, 0, 0]'; x0 = [x0; xhat0];

% define simulink parameters
Tend = 0.03; % simulation end time
amplitude = 10; % sin wave input amplitude
initpha = 0; % initial phase
freq = 600; % sin wave freq (rad/s)
t = 0:1e-4:Tend;

u = amplitude*sin(freq*t+initpha);

[Y,T,X] = lsim(sys,u,t,x0);
```

```
x0 = sp_array([10, 2, 10]); xhat0 = sp_array([0, 0, 0]); x0 =
    sp_array(x0, xhat0); reshape(6, 3)

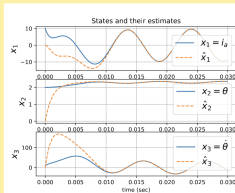
Tend = 0.03; amplitude = 10; initpha = 0; freq = 600
t = sp.arange(0, Tend, 1e-4)

u = amplitude * sp.sin(freq * t + initpha)

[Y, T, X] = ct.lsim(sys, u, t, x0)

plt.figure()
plt.subplot(3, 1, 1)
plt.plot(t, X[1, 0], t, X[1, 3], '--', linewidth=1.5)
plt.xlabel('time (sec)')
plt.legend(['$x_1 = i_a$', '$\hat{x}_1$'], fontsize=16)
plt.grid()
plt.ylabel('$x_1, \hat{x}_1$', fontsize=16)
plt.title('States and their estimates')
plt.subplot(3, 1, 2)
plt.plot(t, X[1, 1], t, X[1, 4], '--', linewidth=1.5)
plt.xlabel('time (sec)')
plt.legend(['$x_2 = \theta$', '$\hat{x}_2$'], fontsize=16)
plt.grid()
plt.ylabel('$x_2, \hat{x}_2$', fontsize=16)
plt.subplot(3, 1, 3)
plt.plot(t, X[1, 2], t, X[1, 5], '--', linewidth=1.5)
plt.xlabel('time (sec)')
plt.legend(['$x_3 = \dot{\theta}$', '$\hat{x}_3$'], fontsize=16)
plt.grid()
plt.ylabel('$x_3, \hat{x}_3$', fontsize=16)
plt.show()
```

From the generated result below, we see that despite the initial error between the true states and the estimated states, the estimation errors quickly converge to zero for all the three states after about 0.01 second. Try to modify the observer eigenvalues and see how they affect the convergence.



Example: LQR of a pure inertia system: analysis

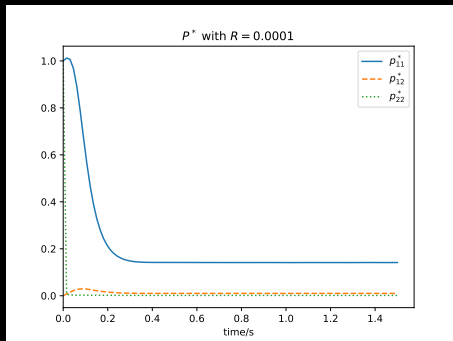


Figure: LQ example: $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, $P(t) = P^*(t_f - t)$

Example: LQR of a pure inertia system: analysis

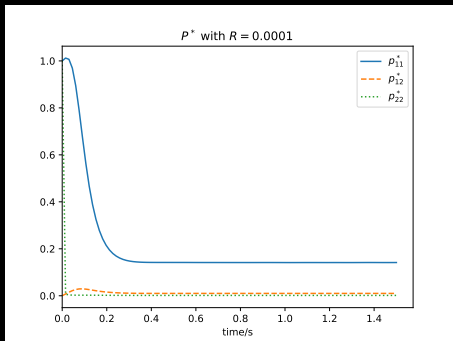


Figure: LQ example: $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, $P(t) = P^*(t_f - t)$

- if the final time t_f is large, $P^*(t)$ forward converges to a stationary value

Example: LQR of a pure inertia system: analysis

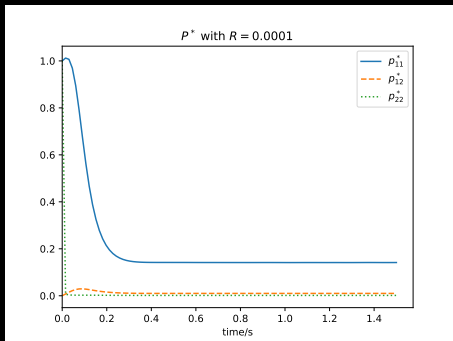


Figure: LQ example: $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$, $P(t) = P^*(t_f - t)$

- ▶ if the final time t_f is large, $P^*(t)$ forward converges to a stationary value
- ▶ i.e., $P(t)$ backward converges to a stationary value at $P(0)$

Example: LQR of a pure inertia system: analysis

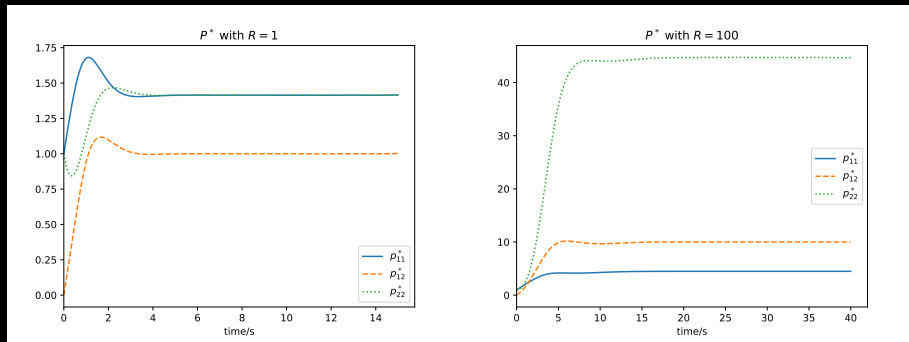


Figure: LQ example with different penalties on control. $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$

Example: LQR of a pure inertia system: analysis

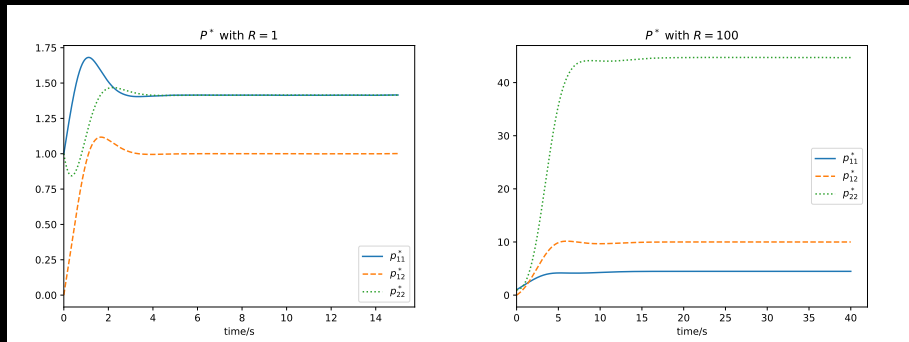


Figure: LQ example with different penalties on control. $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$

► a larger R results in a longer transient

Example: LQR of a pure inertia system: analysis

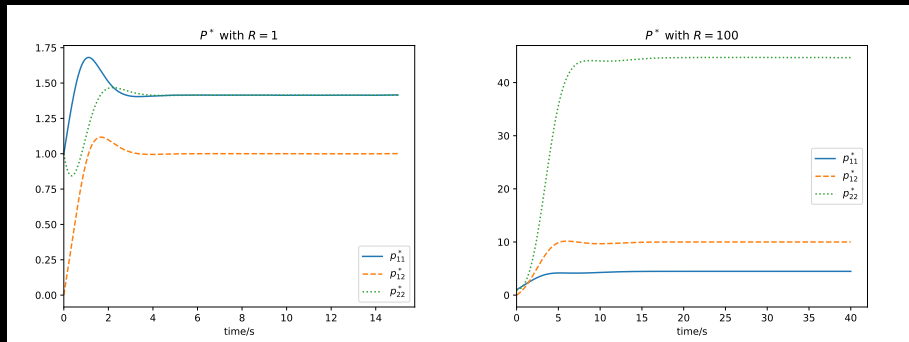
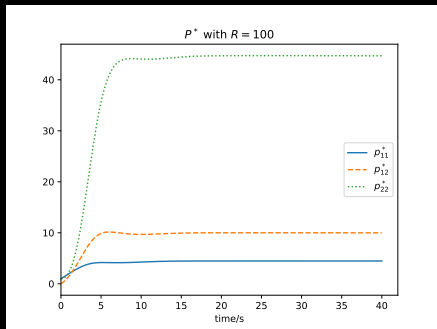


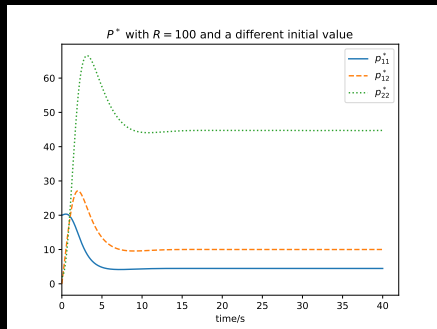
Figure: LQ example with different penalties on control. $P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$

- ▶ a larger R results in a longer transient
- ▶ i.e., a larger penalty on the control input yields a longer time to settle

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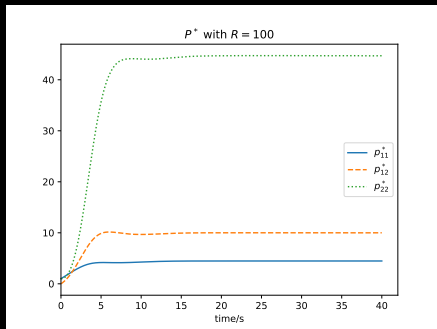
$$(a) P^*(0) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$



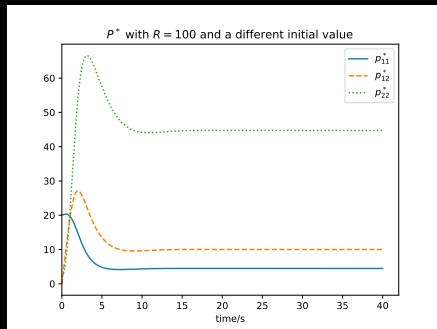
$$(b) P^*(0) = \begin{bmatrix} 20 & 0 \\ 0 & 2 \end{bmatrix}$$

Figure: LQ with different boundary values in Riccati difference Eq.

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► for the same R , the initial value $P(t_f) = S$ becomes irrelevant

1. Problem formulation
2. Solution to the infinite-horizon/stationary LQ problem
3. Solution to the finite-horizon LQ problem
4. From finite-horizon LQ to stationary LQ

From LQ to stationary LQ

► the ARE and the Riccati differential Eq.:

Cost	$J = \frac{1}{2} \int_{t_0}^{\infty} (x^T Q x + u^T R u) dt$	$J = \frac{1}{2} x^T(t_f) S x(t_f) + \frac{1}{2} \int_{t_0}^{t_f} (x^T(t) Q x(t) + u^T(t) R u(t)) dt$
	$\dot{x} = Ax + Bu$	$\dot{x} = Ax + Bu$
Syst.	(A, B) controllable/stabilizable (A, C) observable/detectable	
Key Eq.	$A^T P + PA - PBR^{-1}B^T P + Q = 0$	$-\frac{dP}{dt} = A^T P + PA - PBR^{-1}B^T P + Q$ $P(t_f) = S$
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- in the example, we see that P in the Riccati differential Eq. converges to a stationary value given sufficient time
- when $t_f \rightarrow \infty$, the Riccati differential Eq. converges to ARE and the LQ becomes the stationary LQ, under two conditions that we now discuss in details:
 - (A, B) is controllable/stabilizable
 - (A, C) is observable/detectable

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if (A, B) is controllable or stabilizable, then $P(t)$ is guaranteed to converge to a bounded and stationary value

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 - ▶ in this case, the Riccati equation is

$$-\frac{dP}{dt} = P + P + 1 = 2P + 1 \Leftrightarrow \frac{dP^*}{dt} = 2P^* + 1$$

forward integration of P^* (backward integration of P), will drive $P^*(\infty)$ and $P(0)$ to infinity

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- ▶ *intuition*: if the system is observable, $y = Cx$ will relate to all states \Rightarrow regulating $x^T Q x = x^T C^T C x$ will regulate all states
- ▶ *formally*: if (A, C) is observable (detectable), the solution of the Riccati equation will converge to a positive (semi)definite value P_+ (proof in course notes)

Additional excellent properties of stationary LQ

- ▶ we know stationary LQR yields guaranteed closed-loop stability for controllable (stabilizable) and observable (detectable) systems

It turns out that LQ regulators with full state feedback has excellent additional properties of:

- ▶ at least a 60 degree phase margin
- ▶ infinite gain margin
- ▶ stability is guaranteed up to a 50% reduction in the gain

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- ▶ if there is not a good idea for the structure for Q and R , start with diagonal matrices;
- ▶ gain an idea of the magnitude of each state variable and input variable
- ▶ call them $x_{i,\max}$ ($i = 1, \dots, n$) and $u_{i,\max}$ ($i = 1, \dots, r$)
- ▶ make the diagonal elements of Q and R inversely proportional to $||x_{i,\max}||^2$ and $||u_{i,\max}||^2$, respectively.