

# Disturbance Observer

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# Big picture

Disturbance and uncertainties in mechanical systems:

- ▶ system models are important in design: e.g., in ZPET, observer, and preview controls
- ▶ inevitable to have uncertainty in actual mechanical systems
- ▶ system is also subjected to disturbances

Related control design:

- ▶ robust control
- ▶ adaptive control

Disturbance observer is one example of robust control.

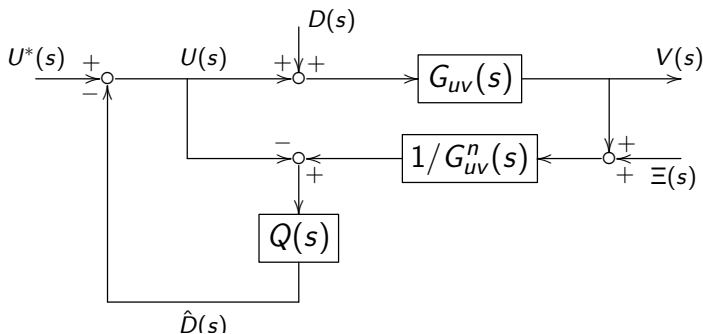
# Disturbance observer (DOB)

- introduced by Ohnishi (1987) and refined by Umeno and Hori (1991)

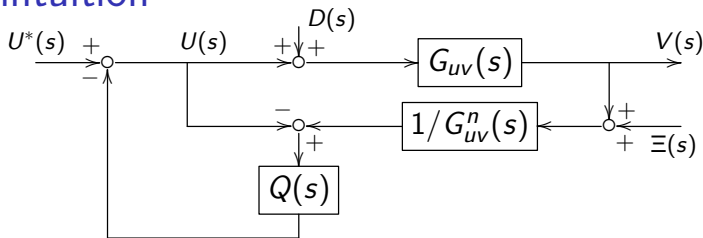
System:

$$V(s) = G_{uv}(s)[U(s) + D(s)]$$

$u(t)$ –input;  $d(t)$ –disturbance;  $v(t)$ –output;  $G_{uv}(s)$ –actual plant dynamics between  $u$  and  $v$ ;  $G_{nv}^n(s)$ –nominal model



# DOB intuition



if  $Q(s) = 1$ , then  $\hat{D}(s)$

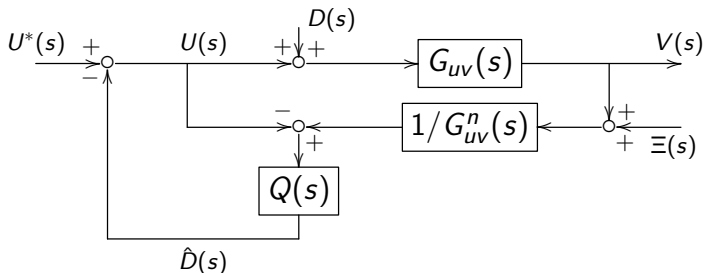
$$U(s) = U^*(s) - \left[ \frac{G_{uv}(s)}{G_{uv}^n(s)} (U(s) + D(s)) + \frac{1}{G_{uv}^n(s)} \Xi(s) - U(s) \right]$$

$$\Rightarrow U(s) = \frac{G_{uv}^n(s)}{G_{uv}(s)} U^*(s) - D(s) - \frac{1}{G_{uv}(s)} \Xi(s)$$

$$V(s) = G_{uv}^n(s) U^*(s) - \Xi(s)$$

i.e., dynamics between  $U^*(s)$  and  $V(s)$  follows the nominal model;  
and disturbance is rejected

# DOB intuition

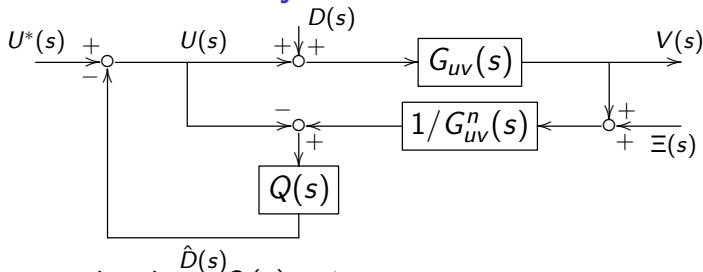


if  $Q(s) = 1$ , then

$$\begin{aligned}\hat{D}(s) &= \left( \frac{G_{uv}(s)}{G_{uv}^n(s)} - 1 \right) U(s) + \frac{1}{G_{uv}^n(s)} \Xi(s) + \frac{G_{uv}(s)}{G_{uv}^n(s)} D(s) \\ &\approx \frac{1}{G_{uv}(s)} \Xi(s) + D(s) \text{ if } G_{uv}(s) = G_{uv}^n(s)\end{aligned}$$

i.e., disturbance  $D(s)$  is estimated by  $\hat{D}(s)$ .

## DOB details: causality



It is impractical to have  $Q(s) = 1$ .

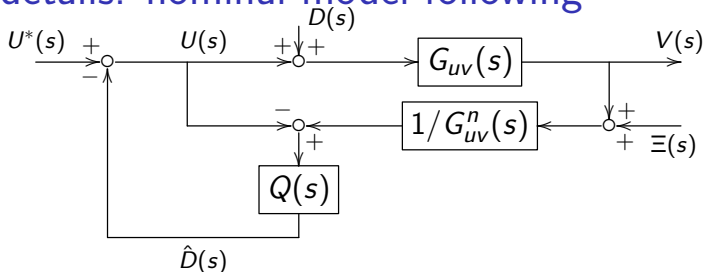
e.g., if  $G_{uv}(s) = 1/s^2$ , then  $1/G_{uv}^n(s) = s^2$  (not realizable)

$Q(s)$  should be designed such that  $Q(s)/G_{uv}^n(s)$  is causal. e.g. (low-pass filter)

$$Q(s) = \frac{1 + \sum_{k=1}^{N-r} a_k (\tau s)^k}{1 + \sum_{k=1}^N a_k (\tau s)^k}, \quad Q(s) = \frac{3\tau s + 1}{(\tau s + 1)^3}, \quad Q(s) = \frac{6(\tau s)^2 + 4\tau s + 1}{(\tau s + 1)^4}$$

where  $\tau$  determines the filter bandwidth

## DOB details: nominal model following



Block diagram analysis gives

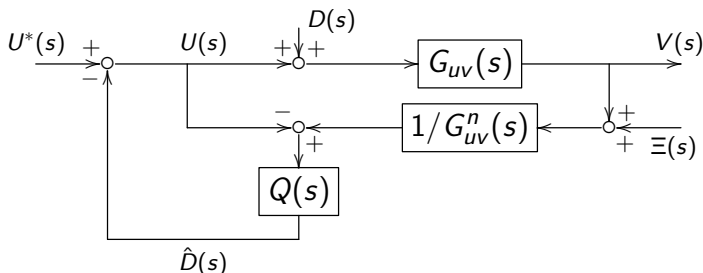
$$V(s) = G_{uv}^o(s) U^*(s) + G_{dv}^o(s) D(s) + G_{\xi v}^o(s) \Xi(s)$$

where

$$G_{uv}^o = \frac{G_{uv} G_{uv}^n}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}, \quad G_{dv}^o = \frac{G_{uv} G_{uv}^n (1 - Q)}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}$$

$$G_{\xi v}^o = -\frac{G_{uv} Q}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}$$

# DOB details: nominal model following



if  $Q(s) \approx 1$ , we have disturbance rejection and nominal model following:

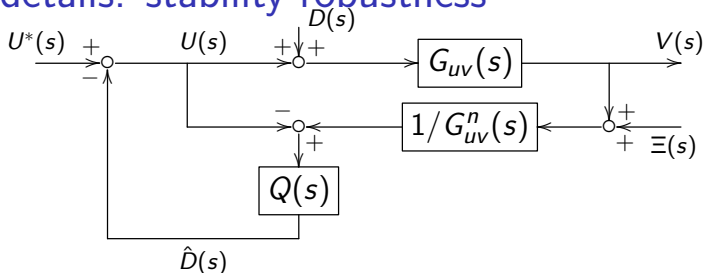
$$G_{uv}^o \approx G_{uv}^n, \quad G_{dv}^o \approx 0, \quad G_{\xi v}^o = -1$$

if  $Q(s) \approx 0$ , DOB is cut off:

$$G_{uv}^o \approx G_{uv}, \quad G_{dv}^o \approx G_{uv}, \quad G_{\xi v}^o \approx 0$$



# DOB details: stability robustness



$$G_{uv}^o = \frac{G_{uv} G_{uv}^n}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}, \quad G_{dv}^o = \frac{G_{uv} G_{uv}^n (1 - Q)}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}, \quad G_{\xi v}^o = -\frac{G_{uv} Q}{G_{uv}^n + (G_{uv} - G_{uv}^n) Q}$$

closed-loop characteristic equation:

$$G_{uv}^n(s) + (G_{uv}(s) - G_{uv}^n(s)) Q(s) = 0$$

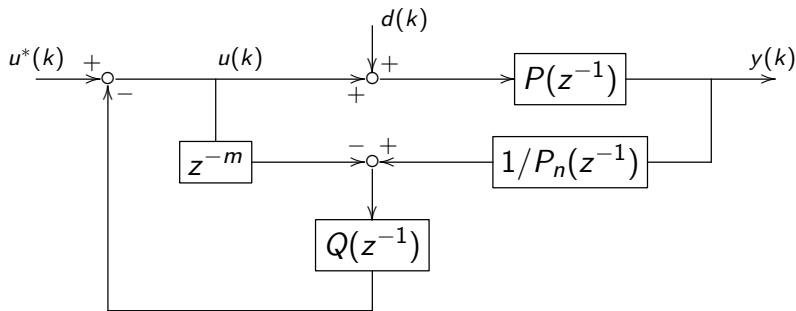
$$\Leftrightarrow G_{uv}^n(s) (1 + \Delta(s) Q(s)) = 0, \text{ if } G_{uv}(s) = G_{uv}^n(s) (1 + \Delta(s))$$

**robust stability condition:** stable zeros for  $G_{uv}^n(s)$ , plus

$$\|\Delta(j\omega) Q(j\omega)\| < 1, \quad \forall \omega$$

# Application example

## Discrete-time case



where  $P(z^{-1}) \approx z^{-m}P_n(z^{-1})$

see more details in, e.g., X. Chen and M. Tomizuka, "Optimal Plant Shaping for High Bandwidth Disturbance Rejection in Discrete Disturbance Observers," in Proceedings of the 2010 American Control Conference, Baltimore, MD, Jun. 30-Jul. 02, 2010, pp. 2641-2646