#### Principles of Feedback Design

MIMO closed-loop analysis Robust stability MIMO feedback design

# Big picture

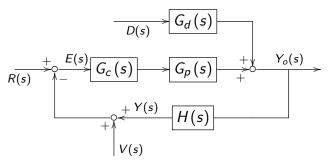
- we are pretty familiar with SISO feedback system design and analysis
- state-space designs (LQ, KF, LQG,...): time-domain; good mathematical formulation and solutions based on rigorous linear algebra
- frequency-domain and transfer-function analysis: builds intuition; good for properties such as stability robustness

#### MIMO closed-loop analysis

signals and transfer functions are vectors and matrices now:

- r (reference) and y (plant output): m-dimensional
- $ightharpoonup G_p(s)$ : p by m transfer function matrix

$$E(s) = R(s) - (H(s) Y_o(s) + V(s))$$
  
=  $R(s) - \{H(s) G_p(s) G_c(s) E(s) + H(s) G_d(s) D(s) + V(s)\}$  (1)



## MIMO closed-loop analysis

(1) gives

$$E(s) = (I_m + G_{\text{open}}(s))^{-1} R(s) - (I_m + G_{\text{open}}(s))^{-1} H(s) G_d(s) D(s) - (I_m + G_{\text{open}}(s))^{-1} V(s)$$

where the loop transfer function

$$G_{\text{open}}(s) = H(s) G_p(s) G_c(s)$$

We want to minimize  $E^*(s) \triangleq R(s) - Y(k) = E(s) + V(s)$ 

$$E^*(s) = \underline{(I_m + G_{\text{open}}(s))^{-1}}R(s)$$

$$-(I_m + G_{\text{open}}(s))^{-1} H(s) G_d(s) D(s) + \underline{(I_m + G_{\text{open}}(s))^{-1} G_{\text{open}}(s)} V(s)$$
Sonsitivity and complementary sonsitivity functions:

Sensitivity and complementary sensitivity functions:

$$S(s) \triangleq (I_m + G_{\text{open}}(s))^{-1}$$
$$T(s) \triangleq (I_m + G_{\text{open}}(s))^{-1} G_{\text{open}}(s)$$

Principles of Feedback Design

## Fundamental limitations in feedback design

$$E^*(s) = S(s)R(s) + T(s)V(s) - S(s)H(s)G_d(s)D(s)$$
  
 $Y(s) = R(s) - E^*(s) = T(s)R(s) + ...$ 

- $\triangleright$  sensitivity function S(s): explains disturbance-rejection ability
- ightharpoonup complementary sensitivity function T(s): explains reference tracking and sensor-noise rejection abilities
- fundamental constraint of feedback design:

$$S(s) + T(s) = I_m$$

equivalently

$$S(j\omega) + T(j\omega) = I_m$$

▶ cannot do well in all aspects: e.g., if  $S(j\omega) \approx 0$  (good disturbance rejection),  $T(j\omega)$  will be close to identity (bad sensor-noise rejection)

## Goals of SISO control design

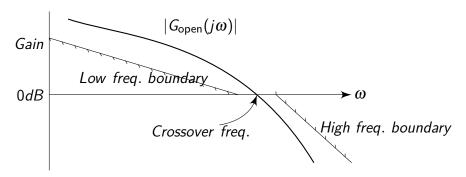
single-input single-output (SISO) control design:

$$S(j\omega) = \frac{1}{1 + G_{\mathsf{open}}(j\omega)}, \ \ T(j\omega) = \frac{G_{\mathsf{open}}(j\omega)}{1 + G_{\mathsf{open}}(j\omega)}$$

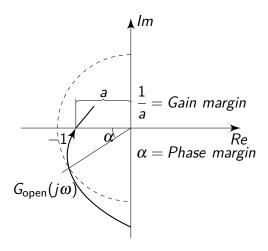
- goals:
  - 1. nominal stability
  - 2. stability robustness
  - 3. command following and disturbance rejection
  - 4. sensor-noise rejection
- feedback achieves: 1 (Nyquist theorem), 2 (sufficient (gain and phase) margins), and
  - ightharpoonup 3: small  $S(j\omega)$  at relevant frequencies (usually low frequency)
  - lacksquare 4: small  $T(j\omega)$  at relevant frequencies (usually high frequency)
- ▶ additional control design for meeting the performance goals: feedforward, predictive, preview controls, etc

# SISO loop shaping

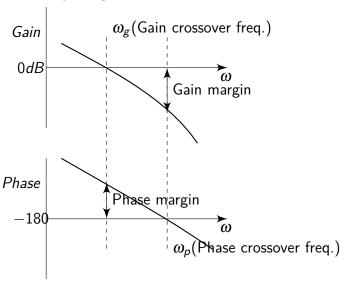
typical loop shape (magnitude response of  $G_{open}$ ):



the idea of stability margins:



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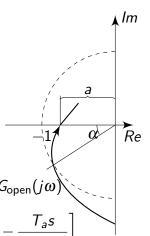
 $G_{\mathrm{open}}(j\omega)$  should be sufficiently far away from (-1,0) for robust stability. Commonly there are uncertainties and the actual case is

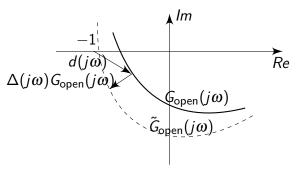
$$ilde{G}_{ ext{open}}\left(s
ight) = G_{ ext{open}}\left(s
ight)\left[1+\Delta\left(s
ight)
ight]$$

e.g. ignored actuator dynamics in a positioning system:

$$ilde{G}_{\mathsf{open}}\left(s
ight) = G_{\mathsf{open}}\left(s
ight) rac{1}{T_{\mathsf{a}}s+1} = G_{\mathsf{open}}\left(s
ight) \left[1 - rac{T_{\mathsf{a}}s}{T_{\mathsf{a}}s+1}
ight]$$

$$\Delta(j\omega) = -\frac{T_a j\omega}{T_a j\omega + 1}$$





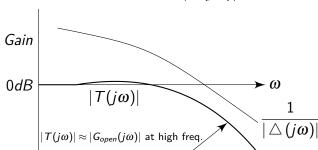
if nominal stability holds, robust stability needs 
$$|\Delta(j\omega) \, G_{open}(j\omega)| = \left| \tilde{G}_{open}(j\omega) - G_{open}(j\omega) \right| < \overline{|1 + G_{open}(j\omega)|}$$
 
$$\Leftrightarrow \left| \left| \Delta(j\omega) \frac{G_{open}(j\omega)}{1 + G_{open}(j\omega)} \right| < 1 \Leftrightarrow |\Delta(j\omega) \, T(j\omega)| < 1, \ \forall \omega \right|$$

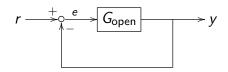
if  $|G_{open}(j\omega)| \ll 1$  then

$$\left| \Delta \left( j\omega 
ight) rac{G_{open} \left( j\omega 
ight)}{1 + G_{open} \left( j\omega 
ight)} 
ight| < 1$$

approximately means

$$|G_{open}(j\omega)| < \frac{1}{|\Delta(j\omega)|}$$





▶ assume  $G_{\text{open}}$  is  $m \times m$  and realized by

$$\frac{\mathrm{d}x(t)}{\mathrm{d}t} = Ax(t) + Be(t), \ x \in \mathbb{R}^{m \times 1}$$
$$y(t) = Cx(t)$$

the closed-loop dynamics is

$$\begin{cases} \frac{\mathrm{d}x(t)}{\mathrm{d}t} &= (A - BC)x(t) + Br(t) \\ y(t) &= Cx(t) \end{cases}$$
 (2)

(2) gives the closed-loop transfer function

$$G_{\text{closed}}(s) = C(sI - A + BC)^{-1}B$$

▶ closed-loop stability depends on the eigenvalues eig(A - BC), which come from

$$\phi_{ ext{closed}}(s) = \det(sI - A + BC) = \det\left\{ (sI - A) \left[ I + (sI - A)^{-1} BC \right] \right\}$$

$$= \det(sI - A) \det\left( I + C (sI - A)^{-1} B \right)$$

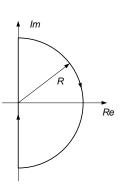
$$= \underbrace{\det(sI - A)}_{ ext{open loop } \phi_{ ext{open}}(s)} \det(I + G_{ ext{open}}(s))$$

hence

$$rac{\phi_{ ext{closed}}\left(s
ight)}{\phi_{ ext{open}}\left(s
ight)} = \det\left(I + G_{ ext{open}}\left(s
ight)
ight)$$

$$\frac{\phi_{\mathsf{closed}}\left(s\right)}{\phi_{\mathsf{open}}\left(s\right)} = \det\left(I + G_{\mathsf{open}}\left(s\right)\right) = \frac{\prod_{j=1}^{n_1}\left(s - p_{\mathsf{cl}}\right)}{\prod_{i=1}^{n_2}\left(s - p_{\mathsf{ol}}\right)}$$

- ▶ evaluate  $\det(I + G_{\text{open}}(s))$  along the D contour  $(R \to \infty)$
- ► Z closed-loop "unstable" eigen values in  $\prod_{j=1}^{n_1} (s p_{cl})$  contribute to  $2\pi Z$  net increase in phase
- ▶ P open-loop "unstable" eigen values in  $\prod_{j=1}^{n_2} (s-p_{\text{ol}})$  contribute to  $-2\pi P$  net increase in phase
- stable eigen values do not contribute to net phase change



the number of counter clockwise encirclements of the origin by  $\det(I + G_{\text{open}}(s))$  is:

$$N(0, det(I + G_{open}(s)), D) = P - Z$$

stability condition: Z = 0

#### Theorem (Multivariable Nyquist Stability Criterion)

the closed-loop system is asymptotically stable if and only if

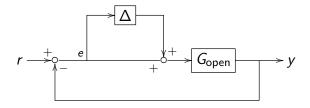
$$N(0, det(I + G_{open}(s)), D) = P$$

i.e., the number of counterclockwise encirclements of the origin by  $det(I + G_{open}(s))$  along the D contour equals the number of open-loop unstable eigen values (of the A matrix).

Given the nominal model  $G_{\text{open}}$ , let the actual open loop be perturbed to

$$\tilde{G}_{\mathsf{open}} \; (j\omega) = G_{\mathsf{open}}(j\omega) [I + \Delta(j\omega)]$$

where  $\Delta(j\omega)$  is the uncertainty (bounded by  $\sigma(\Delta(j\omega)) \leq \bar{\sigma}$ )



what properties should the nominal system possess in order to have robust stability?

obviously need a stable nominal system to start with:

$$N(0, det(I + G_{open}(s)), D) = P$$

for robust stability, we need

$$N(0,\det(I+G_{\mathsf{open}}(s)(1+\Delta(s))),\mathsf{D})=P$$
 for all possible  $\Delta$ 

under nominal stability, we need the boundary condition

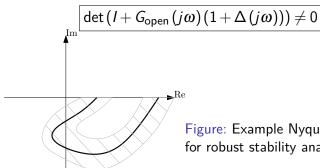


Figure: Example Nyquist plot for robust stability analysis

note the determinant equivalence:

$$\det(I + G_{\text{open}}(j\omega)(1 + \Delta(j\omega))) = \det(I + G_{\text{open}}(j\omega))$$

$$\times \det\left[I + (I + G_{\text{open}}(j\omega))^{-1}G_{\text{open}}(j\omega)\Delta(j\omega)\right]$$

as the system is open-loop asymptotically stable, no poles are on the imaginary, i.e.,

$$\det(I + G_{\text{open}}(j\omega)) \neq 0$$

▶ hence  $\det(I + G_{\text{open}}(j\omega)(1 + \Delta(j\omega))) \neq 0 \iff$ 

$$\det \left[ I + \underbrace{\left( I + G_{\text{open}}(j\omega) \right)^{-1} G_{\text{open}}(j\omega)}_{T(j\omega)} \Delta(j\omega) \right] \neq 0 \qquad (3)$$

- ▶ intuitively, (3) means  $T(j\omega)\Delta(j\omega)$  should be "smaller than" I
- ▶ mathematically, (3) will be violated if  $\exists x \neq 0$  that achieves

$$[I + T(j\omega)\Delta(j\omega)]x = 0$$
  

$$\Leftrightarrow T(j\omega)\Delta(j\omega)x = -x$$
(4)

which will make the singular value

$$\sigma_{\mathsf{max}}[T(j\omega)\Delta(j\omega)] = \max_{v \neq 0} \frac{||T(j\omega)\Delta(j\omega)v||_2}{||v||_2} \ge \frac{||T(j\omega)\Delta(j\omega)x||_2}{||x||_2}$$

as this cannot happen, we must have

$$\sigma_{\mathsf{max}}[T(j\omega)\Delta(j\omega)] < 1$$

It turns out this is both necessary and sufficient if  $\Delta(j\omega)$  is unstructured (can 'attack' from any directions). Message: we can design  $G_{\text{open}}$  such that  $\sigma_{\text{max}}[\Delta(j\omega)] < \sigma_{\text{min}} \left[ T^{-1}(j\omega) \right]$ .

#### Summary

- 1. Big picture
- 2. MIMO closed-loop analysis
- 3. Loop shaping SISO case
- 4. MIMO stability and robust stability MIMO Nyquist criterion MIMO robust stability