

flint neill	Client / Project	Job No.	Sub. Ref.	No. 1/2
SUBJECT Frequency response from state-space eqns.		Date 03/11/16	Prep. by LHY	Check. by

FREQUENCY RESPONSE (FROM STATE-SPACE EQNS)

State-space equations:

$$\begin{bmatrix} \dot{\underline{y}} \\ \underline{y} \end{bmatrix} = \begin{bmatrix} 0 & \underline{I} \\ -\underline{M}^{-1}\underline{K} & -\underline{M}^{-1}\underline{C} \end{bmatrix} \begin{bmatrix} \underline{y} \\ \dot{\underline{y}} \end{bmatrix} + \begin{bmatrix} 0 \\ -\underline{M}^{-1} \end{bmatrix} \underline{F}(t)$$

$$\text{ie } \underset{\substack{\uparrow \\ (2N \times 1)}}{\dot{\underline{x}}(t)} = \underset{\substack{\uparrow \\ (2N \times 2N)}}{[\underline{A}]} \underset{\substack{\uparrow \\ (2N \times 1)}}{\underline{x}(t)} + \underset{\substack{\uparrow \\ (2N \times N)}}{[\underline{B}]} \underset{\substack{\uparrow \\ (N \times 1)}}{\underline{F}(t)}$$

These are
"state-space
equations"
that define
the problem

$$\text{Also: } \underset{\substack{\uparrow \\ (N_0 \times 1)}}{\underline{y}_0(t)} = \underset{\substack{\uparrow \\ (N_0 \times 2N)}}{[\underline{C}]} \underset{\substack{\uparrow \\ (2N \times 1)}}{\underline{x}(t)} + \underset{\substack{\uparrow \\ (N_0 \times N)}}{[\underline{D}]} \underset{\substack{\uparrow \\ (N \times 1)}}{\underline{F}(t)}$$

Note $\underline{y}_0 \neq \underline{y}$. This is
the output vector
($N_0 \times 1$)

Frequency response is defined as follows:

$$\underset{\substack{\uparrow \\ \text{Laplace} \\ \text{(or Fourier)} \\ \text{transform} \\ \text{of } \underline{y}_0(t)}}{\underline{y}_0(s)} = [\underline{G}(s)] \underset{\substack{\uparrow \\ \text{Laplace (or Fourier)} \\ \text{transform of } \underline{F}(t)}}{\underline{F}(s)}$$

Laplace Fourier
 $s \Leftrightarrow j\omega$

$$\text{Dimensions: } (N_0 \times 1) = (N_0 \times N) \cdot (N \times 1).$$

ie $[\underline{G}(s)]$ is a $(N_0 \times N)$ matrix, mapping inputs (force)
to outputs.

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Properties of Laplace transform:

$$\mathcal{L}(\underline{\dot{y}}(t)) = s \underline{y}(s)$$

$$\mathcal{L}(\underline{\ddot{y}}(t)) = s \underline{\dot{y}}(s) \quad (= s^2 \underline{y}(s))$$

Laplace transform of state equations:

$$\left. \begin{aligned} \mathcal{L}(\underline{\dot{x}}(t)) &= [A_c] \underline{x}(s) + [B] \underline{F}(s) \\ \mathcal{L}(\underline{y}_o(t)) &= [C] \underline{x}(s) + [D] \underline{F}(s) \end{aligned} \right\}$$

$$\left. \begin{aligned} s \underline{x}(s) &= [A_c] \underline{x}(s) + [B] \underline{F}(s) \\ \underline{y}_o(s) &= [C] \underline{x}(s) + [D] \underline{F}(s) \end{aligned} \right\}$$

$$\Rightarrow (s \overset{\text{Identity matrix}}{[I]} - [A_c]) \underline{x}(s) = [B] \underline{F}(s)$$

$$\Rightarrow \underline{x}(s) = (s [I] - [A_c])^{-1} [B] \underline{F}(s)$$

$$\Rightarrow \underline{y}_o(s) = ([C] (s [I] - [A_c])^{-1} [B] + [D]) \underline{F}(s)$$

$$\Rightarrow [G(s)] = [C] (s [I] - [A_c])^{-1} [B] + [D]$$

Defines $[G(s)]$ in terms of state-space matrices A, B, C, D .

Let $s = j\omega$ ^{\nwarrow \underline{F}} ; this gives frequency response $[G(\omega)]$ (which is complex)

$$[G(\omega)] = [C] (j\omega [I] - [A_c])^{-1} [B] + [D]$$

Frequency response - system with constraints

$$\text{Let } \left. \begin{aligned} M' &= Z^T M Z \\ K' &= Z^T K Z \\ C' &= Z^T C Z \end{aligned} \right\} \quad \text{where } Z = \text{Null}(J).$$

$$\Rightarrow A' = \begin{bmatrix} 0 & I \\ -M'^{-1}K' & -M'^{-1}C' \end{bmatrix} \quad \text{is state matrix of unconstrained problem.}$$

It can be shown that

$$\dot{x}' = A' x' + B' f$$

$$L: s x'(s) = A' x'(s) + B' f(s)$$

$$\Rightarrow (sI - A') x'(s) = B' f(s)$$

$$\Rightarrow x'(s) = [sI - A']^{-1} B' f(s)$$

$$\text{Recall } x = Z_2 x' \quad \text{where } Z_2 = \begin{bmatrix} Z & 0 \\ 0 & Z \end{bmatrix}$$

$$\Rightarrow x(s) = Z_2 x'(s)$$

$$\Rightarrow x(s) = \underbrace{Z_2 [sI - A']^{-1} B'}_{G(s)} f(s)$$

$G(s)$

↑

Transfer matrix

mapping applied loads
to state variables