

# Physically Based Animation

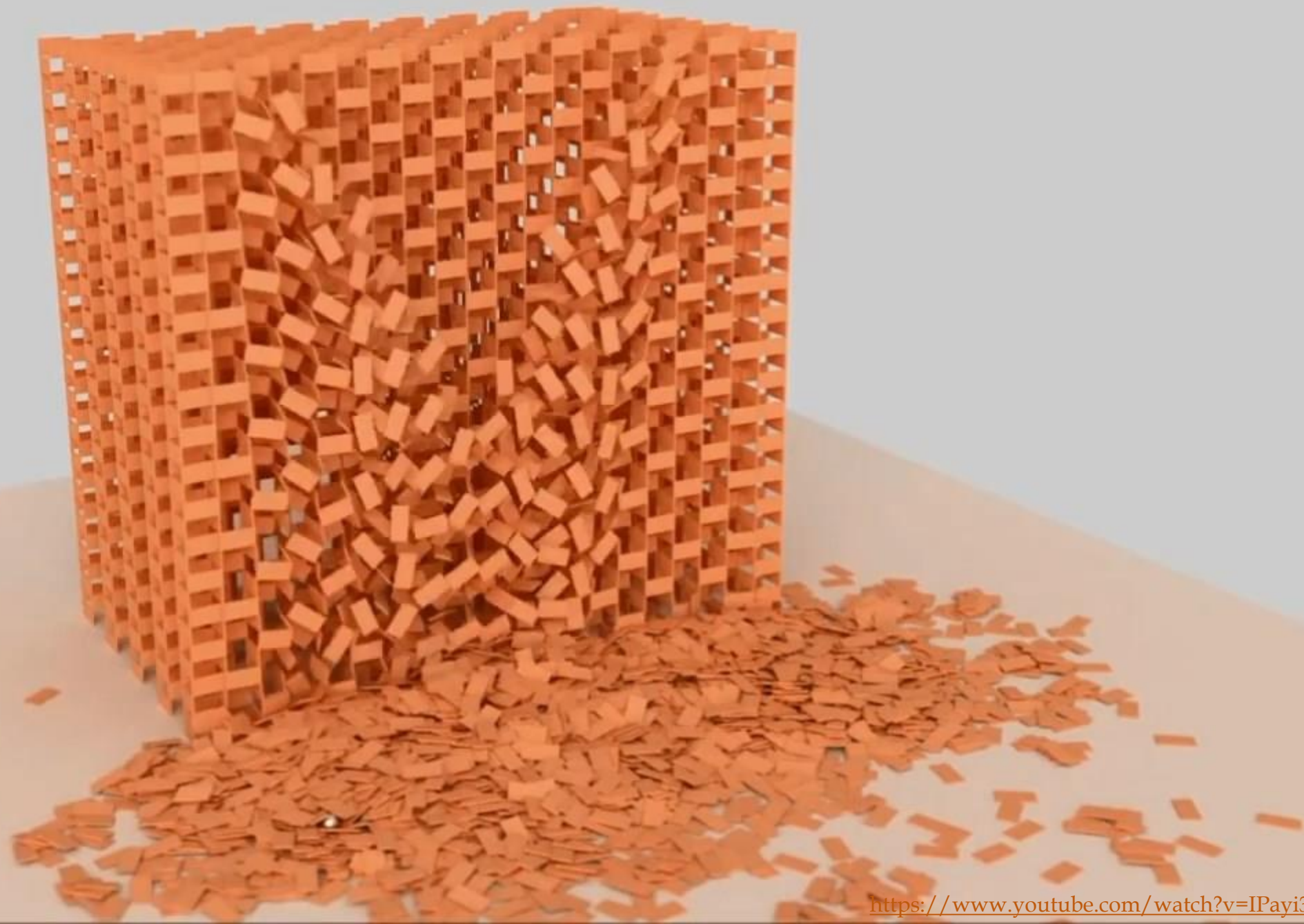
National Cheng Kung University, Fall 2017

Instructor: Shih-Chin Weng 翁士欽 (Style.Me)



# Motivations

- The goal of animation is bringing character to life
  - It is about *timing and spacing*
- Few animation principles (like secondary motion, following through, etc.) are inspired from physical effects
- But using keyframe animation for physical effects is quite hard to match the timing and movements of interacting objects
- Physically-based animation (or modeling) is about using the principles of physics to model dynamically evolving phenomena











[How Disney's 'Moana' created its amazing water effects](#)





# Foundations of Physically-Based Animation

## ■ Model

- A set of rules governing how something behaves (e.g. classical physics)

## ■ Simulation

- Predict the evolution over time
- Time integration
- Collision detection & response

# Dynamics & Numerical Integration





# Classical Physics

- The physics before the advent of quantum mechanics
  - The related physical laws are *deterministic*
- The job of classical physics is to *predict future behavior*
  - It also needs to be revertible (able to trace the history)
  - The *state* is everything we need to predict the future by using governing laws
- System: a collection of objects (i.e., particles, fields, waves, etc.)
- Dynamical system: a system changes with *time*

# Particle System

- Particles are objects

- have mass, position and velocity
- respond to force
- have no spatial extent

- Physical properties

- Position:  $\mathbf{x}(t)$
- Velocity:  $\mathbf{v}(t) = \frac{d\mathbf{x}}{dt} = \dot{\mathbf{x}}$
- Acceleration:  $\mathbf{a}(t) = \frac{d\mathbf{v}}{dt} = \dot{\mathbf{v}} = \frac{d^2\mathbf{x}}{dt^2} = \ddot{\mathbf{x}}$

- How do we update acceleration at any instant of time?

- Newton 2nd law:  $\mathbf{f} = m\mathbf{a}$

# Evolve the Particle System

```
for t in duration:
    # compute net force in another separated loop?
    for p in particles:
        f = compute_net_force(p)
        a = f / p.mass
        p.velocity += a * dt
        p.position += p.velocity * dt
```

■ *State* ( $x, v$ ): all the things we need to update the position

■  $v' = v + \Delta v = v + (a * \Delta t)$

■  $x' = x + \Delta x = x + (v' * \Delta t)$

■ The *state* of the  $n$  particles in  $\mathbb{R}^3$  is:

$$(\mathbf{x}_1, \mathbf{v}_1, \mathbf{x}_2, \mathbf{v}_2, \dots, \mathbf{x}_N, \mathbf{v}_N), \quad \mathbf{x}_i, \mathbf{v}_i \in \mathbb{R}^3$$

■ The space of state is called phase space sometimes



# Dynamical System

- Configuration:  $\mathbf{q}(t)$

- This is what we want to solve (e.g. the *state*) in generalized perspective

- Velocity (time derivative):  $\dot{\mathbf{q}} := \frac{d}{dt} \mathbf{q}$

- Mass:  $M$

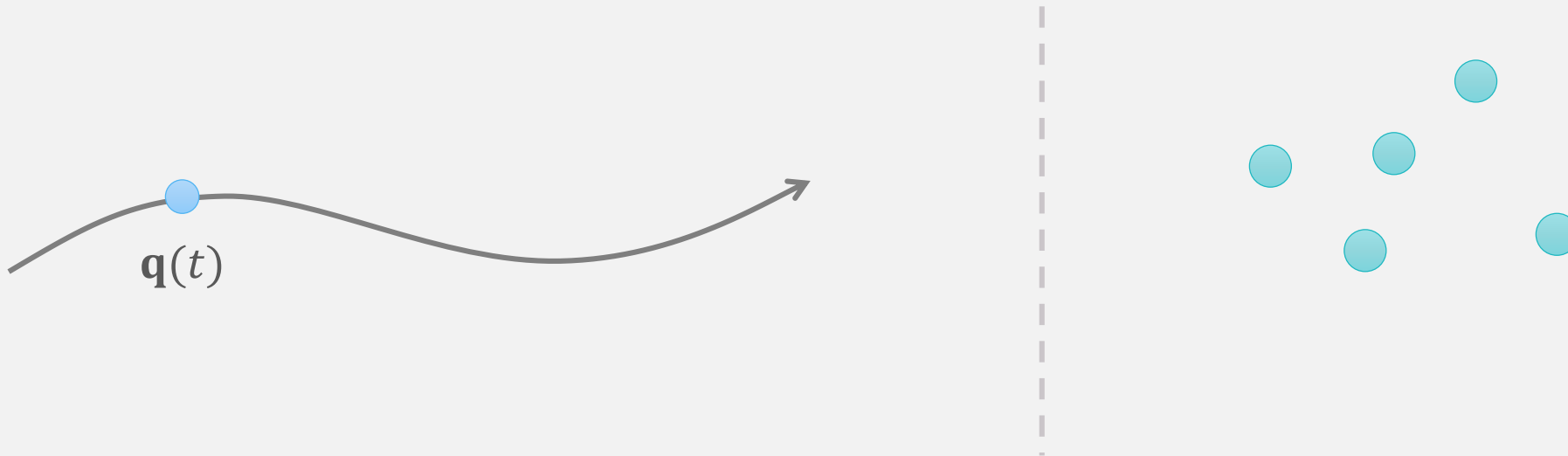
- Forces:  $F$

- Constraints:  $c(\mathbf{q}, \dot{\mathbf{q}}, t) = 0$



# Generalized Coordinates

- Often describing systems with many moving pieces
- The configuration (state vector) is a single vector with all states:  
 $\mathbf{q}(t) = (x_1, x_2, \dots, x_n) \in \mathbb{R}^n$
- The motion of the system is the trajectory of *state* vector

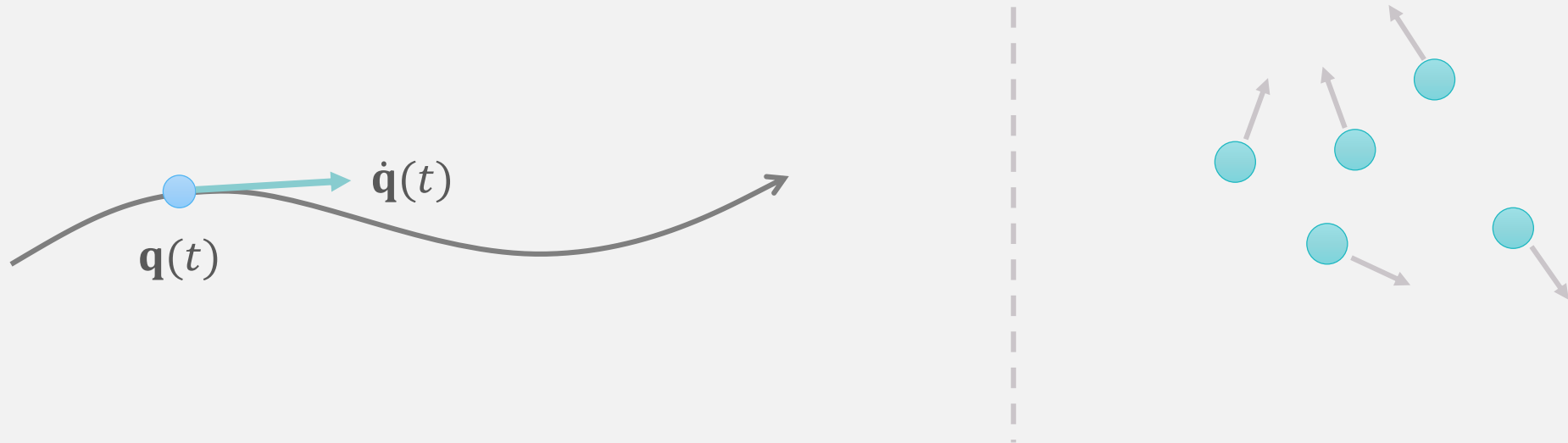


# Generalized Velocity

- The time derivative of generalized coordinates:

$$\dot{\mathbf{q}} = (\dot{x}_1, \dot{x}_2, \dots, \dot{x}_n)$$

- Tells us how the system evolves in small amount of time





# Why Using Numerical Integration?

$$\dot{q}(t) = \frac{d}{dt} q(t) = f(q(t))$$

$$q(t+h) = q(t) + \int_t^{t+h} \dot{q}(t) dt$$

$$= q(t) + \int_t^{t+h} f(q(t)) dt$$

*no analytic form usually!*

# Numerical Integration

- Key idea: replace derivatives with differences

$$\frac{d}{dt}q(t) = f(q(t))$$



*new configuration (unknown)*

*current configuration*

$$\frac{q_{k+1} - q_k}{h} = f(q(t_?))$$

*evaluate  $q$  at  $t_{k+1}$  or  $t_k$ ?*

*"time step", i.e., the time interval between  $q_{k+1} - q_k$*

# Ordinary Differential Equation (ODE)

- ‘*Ordinary*’ means that it only depends on time

$$\frac{d}{dt}u(t) = -au, \quad a > 0$$

*the exact solution is:*  $u(t) = e^{-at}$

it should decay: when  $t \rightarrow \infty$ ,  $u \rightarrow 0$



# Forward Euler Method

$$q_{k+1} - q_k = h * f(q_k)$$

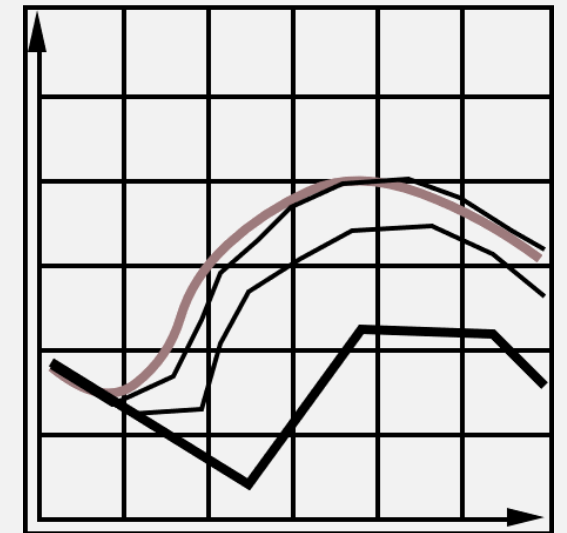
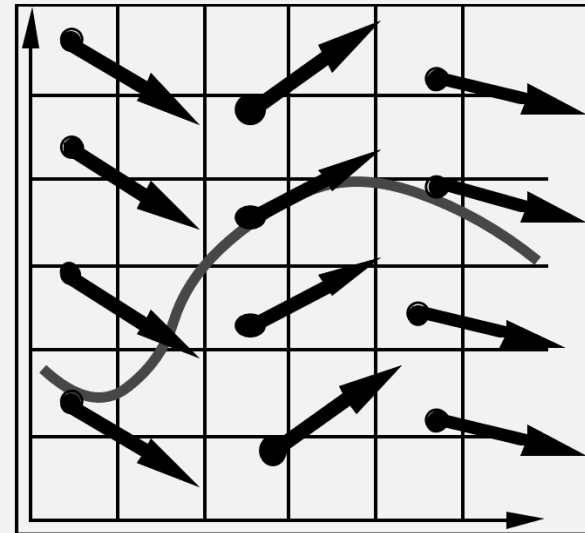
- Evaluate velocity at current configuration

- Easy to implement, it's straight forward

- Unstable with large step size  $h$

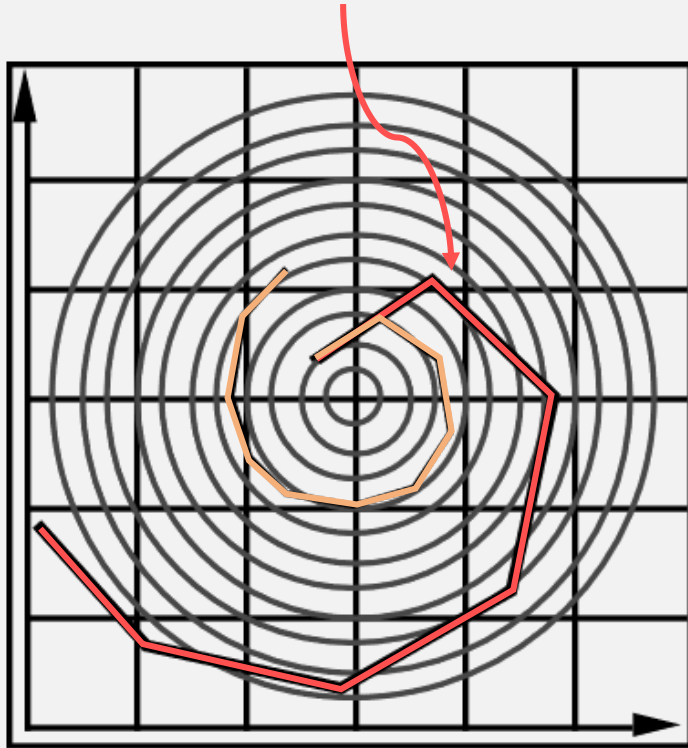
- $h \uparrow \Rightarrow error \uparrow$

- Overshooting or 'blowing up'



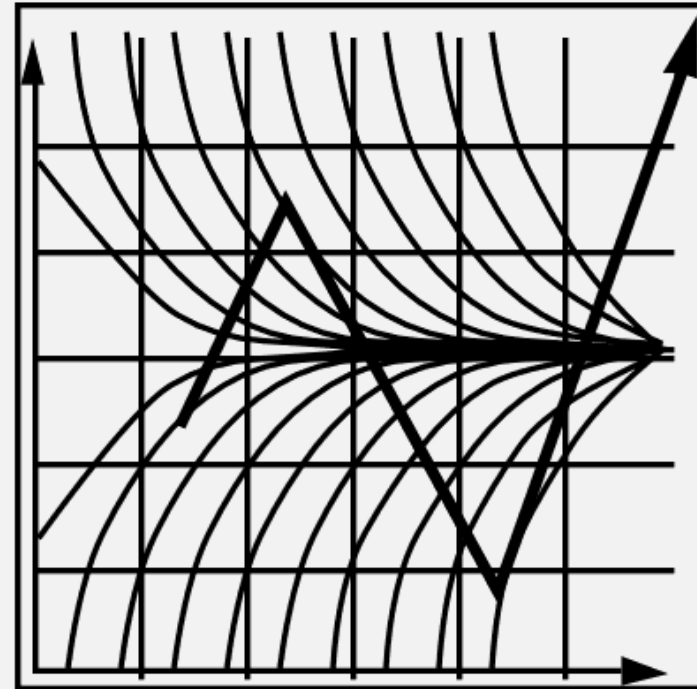
# Two Problems of Forward Euler

*Shrinking the step size will slow the drift, but **NEVER** eliminate it!*



Inaccuracy

$$h \uparrow \Rightarrow \text{error} \uparrow$$



Instability

# Example of Forward Euler Integration

$$\dot{u} = -au, \quad a > 0$$

$$u_{k+1} - u_k = h(-a u_k)$$

$$u_{k+1} = (1 - ha)u_k$$

*after n steps...*

$$u_n = (1 - ha)^n u_0$$

converges **ONLY** if  $|1 - ha| < 1$

if  $a$  is large,  $h$  needs to be very small!

# How Accurate is Forward Euler?

- The Taylor expansion of  $x(t)$  at  $t_0$  is

- $x(t_0 + h) = x(t_0) + \dot{x}(t_0)h + \ddot{x}(t_0)\frac{h^2}{2!} + \dots + \frac{\partial^n x(t_0)}{\partial t^n} \frac{h^n}{n!} + \dots$

- The *order* of integration schemes: how many levels of derivatives represent in the expansion

- Forward Euler is a *first-order* method and its error term is

- $\ddot{x}(t_0)\frac{h^2}{2!} + \dots + \frac{\partial^n x(t_0)}{\partial t^n} \frac{h^n}{n!} + \dots \approx O(h^2)$

- It's only correct if all derivatives beyond the first is zero!

- In general, an order  $n$  method will have  $O(h^{n+1})$  error

- Reduce the step size to  $h/2$ , the error becomes  $1/4$

- But the accumulated error is still linearly proportional to  $h$  (since we need double the accumulation steps)



# Runge-Kutta Methods

- Key idea: use the *slope* at more than one point to extrapolate the value at the future time step

$$\dot{q}(t) = \frac{d}{dt}q(t) = f(t, q(t))$$

$$q(t+h) = q(t) + \int_t^{t+h} \dot{q}(t) dt \approx q(t) + h \sum_i w_i f(t + v_i h, q(t + v_i h)) \quad v_i \in \mathbb{R}$$

*approximate by using quadrature*

# Second Order Runge-Kutta

- Use previous slope  $k_1$  to approximate the temporary  $q$  for  $k_2$

$$k_1 = f(t_n, q_n)$$

$$k_2 = f(t_n + \alpha h, q_n + \beta h k_1) \Rightarrow k_2 \approx f(t_n, q_n) + \alpha h \frac{\partial f}{\partial t}(t_n, q_n) + \beta h k_1 \frac{\partial f}{\partial q}(t_n, q_n) + O(h^2)$$

$$q_{n+1} = q_n + h(w_1 k_1 + w_2 k_2)$$



$$\begin{aligned} q_{n+1} &= q_n + h w_1 f(t_n, q_n) + h w_2 \left( f(t_n, q_n) + \alpha h \frac{\partial f}{\partial t}(t_n, q_n) + \beta h \frac{\partial f}{\partial q} \frac{\partial q}{\partial t}(t_n, q_n) \right) \\ &= q_n + h(w_1 + w_2) f(t_n, q_n) + h^2 w_2 (\alpha f_t + \beta f_q f) \end{aligned}$$

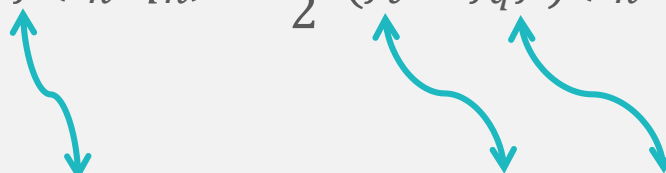
## Second Order Runge-Kutta (Cont.)

- Match the coefficients to the Taylor expansion:

$$q(t+h) = q(t) + h \frac{dq}{dt} + \frac{h^2}{2} \frac{d^2q}{dt^2} + O(h^3)$$

$\frac{d^2q}{dt^2} = \frac{df(t, q(t))}{dt} = \frac{\partial f}{\partial t} + \frac{\partial f}{\partial q} \frac{\partial q}{\partial t} = f_t + f_q f$

$$= q(t) + hf(t_n, q_n) + \frac{h^2}{2} (f_t + f_q f)(t_n, q_n) + O(h^3)$$


$$q_{n+1} = q_n + h(w_1 + w_2)f(t_n, q_n) + h^2 w_2 (\alpha f_t + \beta f_q f)(t_n, q_n)$$

- Get  $w_1 + w_2 = 1, \alpha w_2 = \frac{1}{2}, \beta w_2 = \frac{1}{2}$

- There are infinitely many choices!

- Classical 2<sup>nd</sup> order RK2:  $w_1, w_2 = \frac{1}{2}, \alpha, \beta = 1$

# Midpoint Method (RK2)

$$q_{n+1} = q_n + h(w_1 + w_2)f(t_n, q_n) + h^2w_2(\alpha f_t + \beta f_q f)(t_n, q_n) + O(h^3)$$

■ By choosing  $\alpha = \beta = \frac{1}{2}$ ,  $w_1 = 0$ ,  $w_2 = 1$

■ Then  $q_{n+1} = q_n + hf(t_n, q_n) + \frac{h^2}{2}(f_t + f_q f)(t_n, q_n)$

■ Integration steps:

a. Compute an Euler step

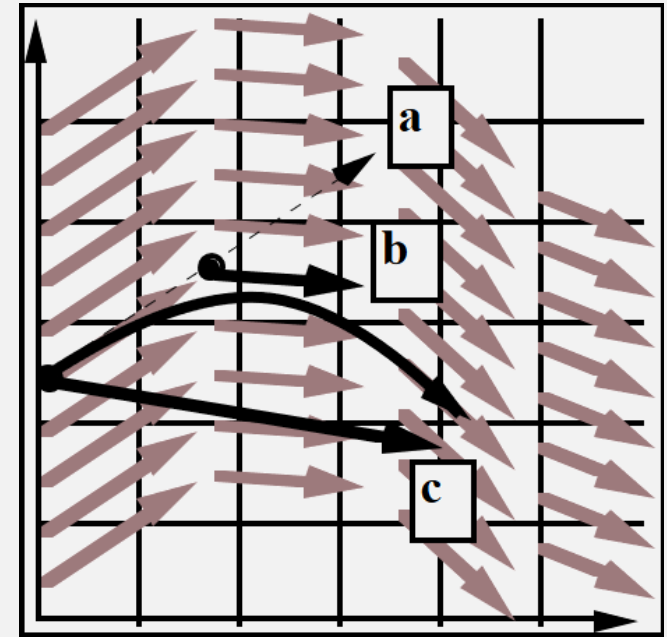
$$hk_1 = h * f(t_k, q_k)$$

b. Evaluate f at the midpoint

$$k_2 = f\left(t_k + \frac{h}{2}, q_k + \frac{hk_1}{2}\right)$$

c. Take full step with the slope at the midpoint

$$q_{k+1} = q_k + h * k_2$$



# Symplectic Euler

- Key ideas

- Update velocity using current configuration first
- Use that new velocity to update next configuration

- Used by many real-time physics engines like Bullet

- Also known as *semi-explicit Euler*, *semi-implicit Euler*, *Euler-Cromer* and *Newton-Stormer-Verlet* (so many names ...)

- Benefits: stable and preserving energy



# Symplectic Euler

- Key ideas

- Update velocity using current configuration first
- Use that new velocity to update next configuration

- Used by many real-time physics engines like Bullet

- Also known as *semi-explicit Euler*, *semi-implicit Euler*, *Newton-Störmer-Verlet* (so many names)

- Benefits: stable

**Read More**  
Integration Basics, Glenn Fiedler.

Newton-Störmer-

# Adaptive Step Sizes

- What is 'good' step size for explicit integrator?
  - The largest step size with acceptable integration error
- Vary step size  $h$  over the course of solving ODE
  - Compute two estimates of  $x(t_0 + h)$ 
    1.  $x_a$ : take one full step size
    2.  $x_b$ : take two steps with size  $h/2$
  - Measure the error  $e = |x_a - x_b|$
  - Scale the step size w.r.t. the error bound and acceptable error

## Adaptive Step Sizes (Cont.)

- Suppose the error bound is  $O(h^2)$  and the acceptable error is  $10^{-4}$  per step
- If current error is  $10^{-8}$ , we could increase step size to  $100h$

$$\left(\frac{10^{-4}}{10^{-8}}\right)^{\frac{1}{2}} h = 100h$$

- If current error is  $10^{-3}$ , we have to decrease the step size to  $0.316h$

$$\left(\frac{10^{-4}}{10^{-3}}\right)^{\frac{1}{2}} h \approx 0.316h$$

# Backward Euler Method

$$q_{k+1} - q_k = h * f(q_{k+1})$$

- Evaluate function  $f$  at *new* (i.e.,  $k+1$ ) configuration
- Need to solve a linear system for each time step
- Unconditionally stable, even if  $h$  is  $\infty$ 
  - But this does **NOT** mean it is accurate!
- Numerical dissipation
  - Converges to zero when  $h$  is large
  - Energy lost

# Example of Backward Euler Integration

$$\dot{u} = -au, \quad a > 0$$

$$u_{k+1} - u_k = h(-au_{k+1})$$

$$u_{k+1} = \frac{1}{1 + ah} u_k$$

*after n steps...*

$$u_n = \left( \frac{1}{1 + ah} \right)^n u_0$$

$a, h > 0 \Rightarrow 1 + ah > 1$ , always converges!

# 'Stiff' System

- Suppose we are trying to fix a particle movement in y-axis by

$$\dot{\mathbf{X}}(t) = \frac{d}{dt} \begin{pmatrix} \mathbf{x}(t) \\ \mathbf{y}(t) \end{pmatrix} = \begin{pmatrix} -\mathbf{x}(t) \\ -k\mathbf{y}(t) \end{pmatrix}$$

- $k$  is a large positive number  $\Rightarrow$  a quick decay along y-axis

$$\mathbf{X}_{\text{new}} = \mathbf{X}_0 + h\dot{\mathbf{X}}(t_0) = \begin{pmatrix} (1 - h)\mathbf{x}_0 \\ (1 - hk)\mathbf{y}_0 \end{pmatrix}$$

- It converges if  $|1 - hk| < 1 \Rightarrow hk < 2$ 
  - Where  $k$  is large, thus  $h$  has to be *small*
- *Stiff*: numerically unstable, unless the step size has to be small



# Numerical Integration of Particle System

- Suppose  $\ddot{\mathbf{x}}(t) = f(\mathbf{x}(t), \dot{\mathbf{x}}(t))$
- From Newton 2<sup>nd</sup> law of motion:

$$\frac{d}{dt} \begin{pmatrix} \mathbf{x} \\ \mathbf{v} \end{pmatrix} = \begin{pmatrix} \mathbf{v}(t) \\ f(\mathbf{x}(t), \mathbf{v}(t)) \end{pmatrix}$$

*which  $t$ ? at current or new configuration?*

# Forward Euler

$$\begin{pmatrix} \Delta x \\ \Delta v \end{pmatrix} = h \begin{pmatrix} v_n \\ f(x_n, v_n) \end{pmatrix} \quad \begin{aligned} x_{n+1} &= x + \Delta x = x + hv_n \\ v_{n+1} &= v_0 + hf(x_n, v_n) \end{aligned}$$

- Force function could be:
  - Unary forces like gravity and drag
  - n-ary forces like springs
  - Forces of spatial interaction
    - Attraction and repulsion between particles

# Backward Euler

$$\frac{d}{dt} \begin{pmatrix} x \\ v \end{pmatrix} = \begin{pmatrix} v(t) \\ f(x(t), v(t)) \end{pmatrix} \Rightarrow \begin{pmatrix} \Delta x \\ \Delta v \end{pmatrix} = h \begin{pmatrix} v_n + \Delta v \\ f(x_n + \Delta x, v_n + \Delta v) \end{pmatrix}$$

- Suppose  $\ddot{x}(t) = f(x(t), \dot{x}(t))$  and assume function  $f$  is smooth, then  $f$  could be approximated by Taylor expansion:

$$\begin{aligned} f(x_n + \Delta x, v_n + \Delta v) &= f(x_n, v_n) + \frac{\partial f}{\partial x} \Delta x + \frac{\partial f}{\partial v} \Delta v \\ &= f(x_n, v_n) + \frac{\partial f}{\partial x} h(v_n + \Delta v) + \frac{\partial f}{\partial v} \Delta v \end{aligned}$$

- Plug into the linear system and solve  $\Delta v$ :

$$\left( I - h^2 \frac{\partial f}{\partial x} - h \frac{\partial f}{\partial v} \right) \Delta v = f(x_n, v_n) + \frac{\partial f}{\partial x} h v_n \quad \text{then } x_{n+1} = x_n + \Delta x = x_n + h(v_n + \Delta v)$$

# Forward vs. Backward Euler

- Forward Euler is fast, simple, but not stable
  - Need to reduce the time step to avoid overshootings
- Backward Euler is unconditionally stable with prices
  - Need to solve system of equations in each time step
  - Temporal details disappear, due to numerical dissipation
    - Large time step would lose more energy?
- For 'stiff' system, backward Euler is more stable than forward Euler
- Total evaluation time is:

*number of time steps × the computation time in each step*

# Mass-Spring System



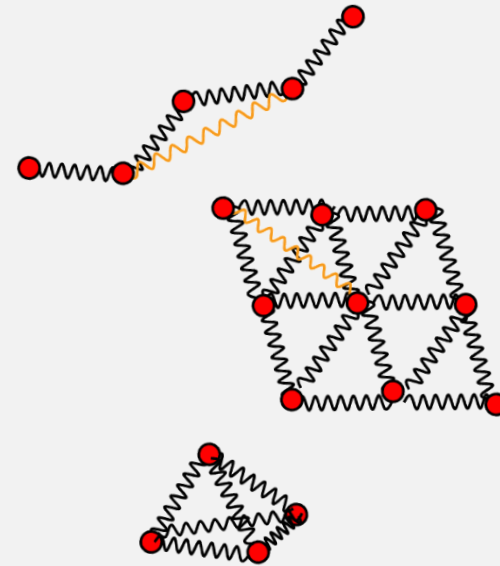
# Deformable Objects





# Mass-Spring Structure

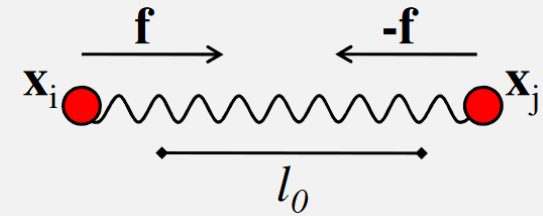
- A set of point masses connected by springs
- Each spring follows Hook's law and Newton's 3<sup>rd</sup> law of motion
- Structure for deformable objects:
  - Hair/rope: chain
    - Additional springs for bending and torsional resistance needed
  - Cloth: triangle mesh
    - Need extra springs to model bending
    - Low bending resistance but quite stiff w.r.t. stretching
  - Soft-body: tetrahedral mesh



# Physical Formulation

## ■ Spring force

$$\mathbf{f}_i^s = \mathbf{f}^s(\mathbf{x}_i, \mathbf{x}_j) = \overset{\text{stiffness}}{k_s} \frac{\mathbf{x}_j - \mathbf{x}_i}{|\mathbf{x}_j - \mathbf{x}_i|} (|\mathbf{x}_j - \mathbf{x}_i| - \overset{\text{rest length}}{l_0})$$



## ■ Damping force

$$\mathbf{f}_i^d = \mathbf{f}^d(\mathbf{x}_i, \mathbf{v}_i, \mathbf{x}_j, \mathbf{v}_j) = \overset{\text{damping}}{k_d} \left[ (\mathbf{v}_j - \mathbf{v}_i) \cdot \left( \frac{\mathbf{x}_j - \mathbf{x}_i}{|\mathbf{x}_j - \mathbf{x}_i|} \right) \right] \left( \frac{\mathbf{x}_j - \mathbf{x}_i}{|\mathbf{x}_j - \mathbf{x}_i|} \right)$$

## ■ Unified spring force: $\mathbf{f}_i = \mathbf{f}(\mathbf{x}_i, \mathbf{v}_i, \mathbf{x}_j, \mathbf{v}_j) = \mathbf{f}^s(\mathbf{x}_i, \mathbf{x}_j) + \mathbf{f}^d(\mathbf{x}_i, \mathbf{v}_i, \mathbf{x}_j, \mathbf{v}_j) = -\mathbf{f}_j$

# Implicit Integration

Mass matrix:  $\mathbf{M} \in \mathbb{R}^{3n \times 3n}$   
(diagonal matrix)

forces of n particles:  $\mathbf{f}: \mathbb{R}^{3n} \rightarrow \mathbb{R}^{3n}$

positions:  $\mathbf{x} \in \mathbb{R}^{3n}$

$$\mathbf{M}\mathbf{v}^{t+1} = \mathbf{M}\mathbf{v}^t + h * \mathbf{f}(\mathbf{x}^{t+1})$$

$$\Downarrow \quad \mathbf{x}^{t+1} = \mathbf{x}^t + h * \mathbf{v}^{t+1}$$

$$\mathbf{M}\mathbf{v}^{t+1} = \mathbf{M}\mathbf{v}^t + h * \mathbf{f}(\mathbf{x}^t + h * \mathbf{v}^{t+1})$$

$$\Downarrow \quad \textit{Taylor expansion}$$

$$\mathbf{M}\mathbf{v}^{t+1} = \mathbf{M}\mathbf{v}^t + h \left[ \mathbf{f}(\mathbf{x}^t) + \frac{\partial \mathbf{f}(\mathbf{x}^t)}{\partial \mathbf{x}} * (h * \mathbf{v}^{t+1}) \right]$$

## Implicit Integration (Cont.)

$$\begin{aligned}\mathbf{M}\mathbf{v}^{t+1} &= \mathbf{M}\mathbf{v}^t + h \left[ \mathbf{f}(\mathbf{x}^t) + \frac{\partial \mathbf{f}(\mathbf{x}^t)}{\partial \mathbf{x}} * (h * \mathbf{v}^{t+1}) \right] \\ &= \mathbf{M}\mathbf{v}^t + h\mathbf{f}(\mathbf{x}^t) + h^2 \mathbf{K}\mathbf{v}^{t+1}\end{aligned}$$

$$(\mathbf{M} - h^2 \mathbf{K}) \mathbf{v}^{t+1} = \mathbf{M}\mathbf{v}^t + h\mathbf{f}(\mathbf{x}^t)$$

 *How to construct matrix  $\mathbf{K}$ ?*

$$\mathbf{x}^{t+1} = \mathbf{x}^t + h * \mathbf{v}^{t+1}$$

# Jacobian Matrix K

- Contains the derivatives of all  $3n$  force components w.r.t. all  $3n$  position components of the particles
  - Also called *tangent stiffness* matrix
- A spring force between particles  $i$  and  $j$  constructs *four*  $3 \times 3$  sub-matrices:
  - Derivatives w.r.t. positions  $\mathbf{x}_i$  and  $\mathbf{x}_j$

$$\begin{aligned}\mathbf{K}_{i,i} &= \frac{\partial}{\partial \mathbf{x}_i} \mathbf{f}^s(\mathbf{x}_i, \mathbf{x}_j) = k_s \frac{\partial}{\partial \mathbf{x}_i} \left( (\mathbf{x}_j - \mathbf{x}_i) - l_0 \frac{\mathbf{x}_j - \mathbf{x}_i}{|\mathbf{x}_j - \mathbf{x}_i|} \right) \\ &= k_s \left( -\mathbf{I} + l_0 \left[ \frac{\mathbf{I} - (\mathbf{x}_j - \mathbf{x}_i)(\mathbf{x}_j - \mathbf{x}_i)^T}{|\mathbf{x}_j - \mathbf{x}_i|^3} \right] \right) \\ &= -\mathbf{K}_{i,j} = \mathbf{K}_{j,j} = -\mathbf{K}_{j,i}\end{aligned}$$

$$\left( \frac{f}{g} \right)' = \frac{f'g - fg'}{g^2}$$

$$\frac{\partial |\mathbf{x}|}{\partial \mathbf{x}} = \left( \frac{\mathbf{x}}{|\mathbf{x}|} \right)^T = \hat{\mathbf{x}}^T$$

$$\frac{\partial \hat{\mathbf{x}}}{\partial \mathbf{x}} = \frac{\mathbf{I} \cdot |\mathbf{x}| - \mathbf{x} \cdot \hat{\mathbf{x}}^T}{|\mathbf{x}|^2} = \frac{\mathbf{I} - \hat{\mathbf{x}} \cdot \hat{\mathbf{x}}^T}{|\mathbf{x}|}$$

Read More: [The Matrix Cookbook](#)

# Solve System Equations

$$(\mathbf{M} - h^2 \mathbf{K}) \mathbf{v}^{t+1} = \mathbf{M} \mathbf{v}^t + h \mathbf{f}(\mathbf{x}^t)$$

- For each time step

- K is reset to zero at first
- Then add each sub-matrix  $\mathbf{K}_{i,j}$  of each spring force to  $\mathbf{K}$

- $\mathbf{K}$  changes at each time step

- Using direct solver is not practical
- Solving the system with iterative methods like *Conjugate Gradients* is more common

# Three Phases of Collision Handling

## ■ *Collision detection*

- Detect whether there is a collision or not

## ■ *Collision determination*

- Find the collision point  $\mathbf{p}$  and other information like contact normal

## ■ *Collision response*

- Update simulation state of each entity such as position, velocity, etc.

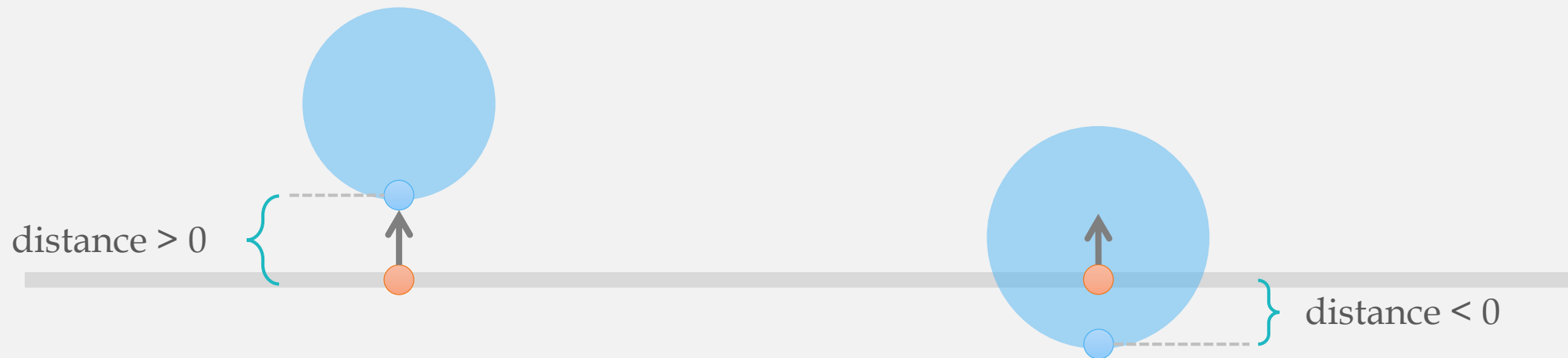
# Collision Detection

- Broad-phase

- Using bound volumes to determine if there are any potential intersections

- Narrow-phase

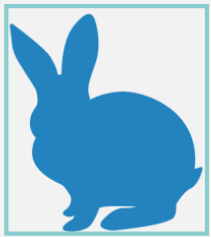
- Compute closest point, normal and distance/penetration





Recap

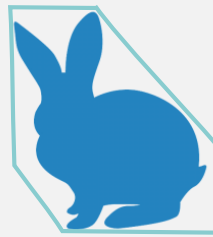
# Types of Bounding Volumes



AABB



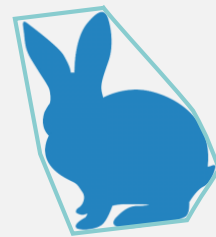
sphere



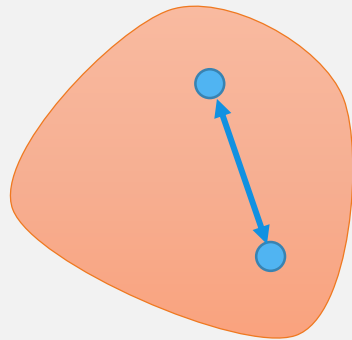
k-DOP



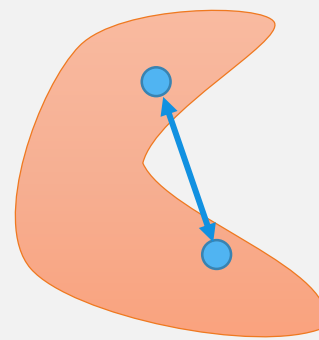
OBB



convex hull

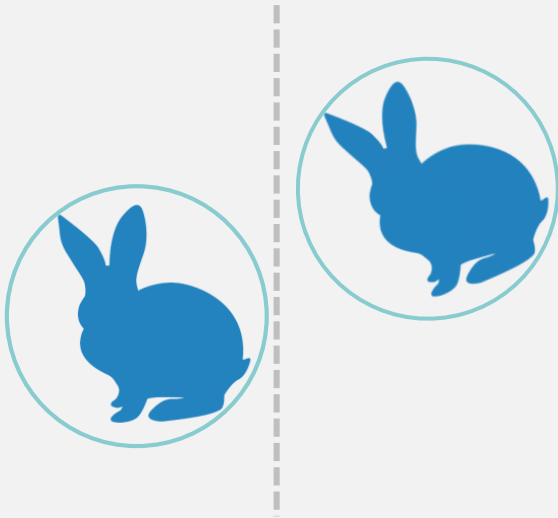


convex

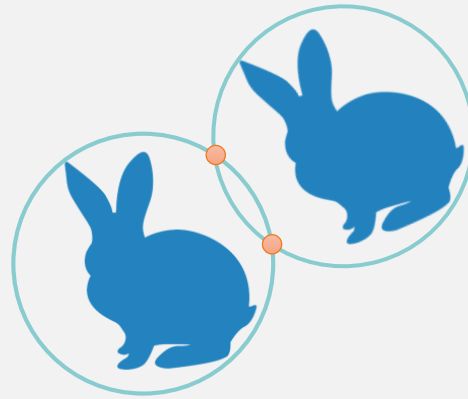


concave

# Collision Detection (Cont.)



Board-Phase: not collide



Board-Phase: collide  
Narrow-Phase: no intersection



Board-Phase: collide  
Narrow-Phase: intersection

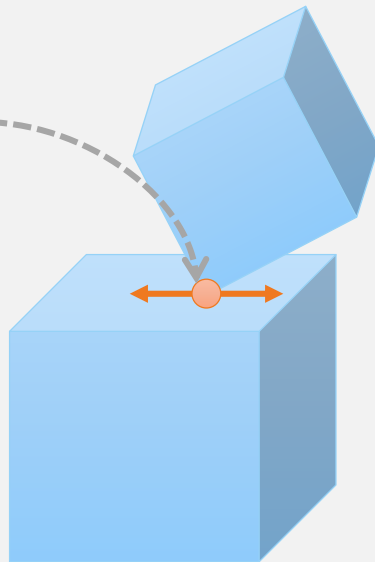
# Type of Collisions

- *Genericity*: objects in 'general position'

- A tiny perturbation does not change the nature of its interaction with the stuff around it
- *Degenerate*: the objects are **NOT** in general position

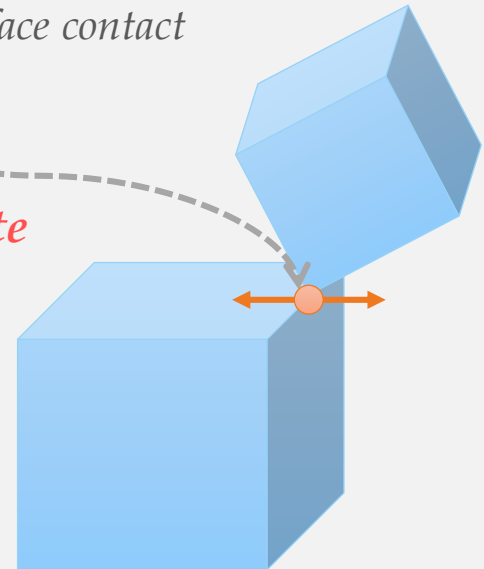
*when moving each object in the contact plane,  
it's still a vertex-face contact*

*generic*



*small movement would make such  
vertex-edge contact become a vertex-face contact*

*degenerate*



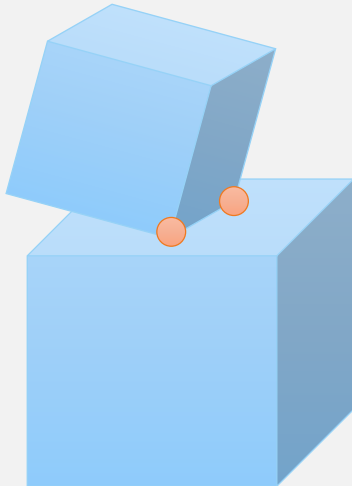
## Type of Collisions (Cont.)

- There are 9 combinations of collision tests between geometry primitives (e.g. vertex, edge and face)
- But there are only two *generic* types of collisions:
  - Vertex-face
  - Edge-edge
- All other types of collisions are all *degenerate*, which means a very small perturbation will remove them as possibilities

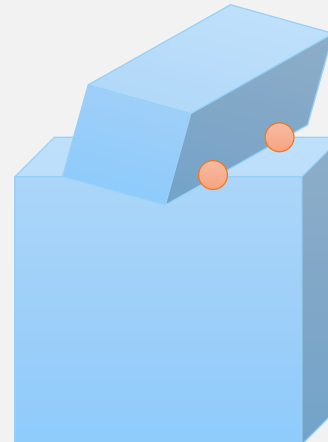
# Type of Collisions (Cont.)

- Even degenerate collisions have generic collisions occurring with them
  - Thus it's critical to handle generic collisions
- An edge-face collision occurs when there are

two vertex-face contacts



two edge-edge contacts



# Vertex-Face Test

- Suppose all faces have been triangularized
- Find the closest point  $x_p$  on the triangle to point  $x_4$

$$\mathbf{x}_p = w_1 \mathbf{x}_1 + w_2 \mathbf{x}_2 + w_3 \mathbf{x}_3 = \mathbf{x}_3 + w_1 \vec{\mathbf{x}}_{13} + w_2 \vec{\mathbf{x}}_{23}$$

where  $w_1 + w_2 + w_3 = 1$ ,  $\vec{\mathbf{x}}_{ij} = \mathbf{x}_j - \mathbf{x}_i$

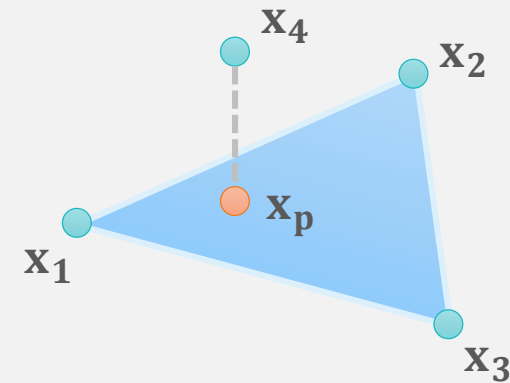
$$\vec{\mathbf{x}}_{4p} = \vec{\mathbf{x}}_{43} - w_1 \vec{\mathbf{x}}_{13} - w_2 \vec{\mathbf{x}}_{23} = \mathbf{0}$$

$$[\vec{\mathbf{x}}_{13} \quad \vec{\mathbf{x}}_{23}] \begin{bmatrix} w_1 \\ w_2 \end{bmatrix} = [\vec{\mathbf{x}}_{43}]$$

Underdetermined! Use normal equation  $A^T A x = A^T b \Rightarrow$

$$\begin{bmatrix} \mathbf{x}_{13} \\ \mathbf{x}_{23} \end{bmatrix} [\vec{\mathbf{x}}_{13} \quad \vec{\mathbf{x}}_{23}] \begin{bmatrix} w_1 \\ w_2 \end{bmatrix} = \begin{bmatrix} \vec{\mathbf{x}}_{13} \\ \vec{\mathbf{x}}_{23} \end{bmatrix} [\vec{\mathbf{x}}_{43}]$$

$$\begin{bmatrix} \vec{\mathbf{x}}_{13} \cdot \vec{\mathbf{x}}_{13} & \vec{\mathbf{x}}_{13} \cdot \vec{\mathbf{x}}_{23} \\ \vec{\mathbf{x}}_{23} \cdot \vec{\mathbf{x}}_{13} & \vec{\mathbf{x}}_{23} \cdot \vec{\mathbf{x}}_{23} \end{bmatrix} \begin{bmatrix} w_1 \\ w_2 \end{bmatrix} = \begin{bmatrix} \vec{\mathbf{x}}_{13} \cdot \vec{\mathbf{x}}_{43} \\ \vec{\mathbf{x}}_{23} \cdot \vec{\mathbf{x}}_{43} \end{bmatrix}$$



# Edge-Edge Test

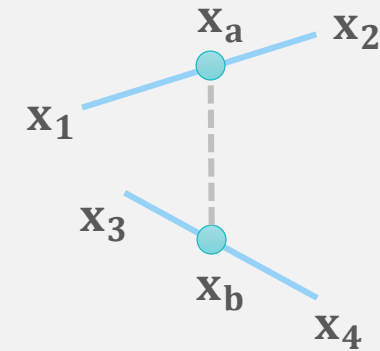
$$\mathbf{x}_a = \mathbf{x}_1 + a\vec{\mathbf{x}}_{21}, \quad \mathbf{x}_b = \mathbf{x}_3 + b\vec{\mathbf{x}}_{43}, \quad a, b \in [0,1]$$

$$\mathbf{x}_b - \mathbf{x}_a = \vec{\mathbf{x}}_{31} + b\vec{\mathbf{x}}_{43} - a\vec{\mathbf{x}}_{21} \Rightarrow [\vec{\mathbf{x}}_{21} \quad -\vec{\mathbf{x}}_{43}] \begin{bmatrix} a \\ b \end{bmatrix} = [\vec{\mathbf{x}}_{31}]$$

Apply normal equation  $A^T A x = A^T b$  to find the closest point

$$\begin{bmatrix} \vec{\mathbf{x}}_{21} \\ -\vec{\mathbf{x}}_{43} \end{bmatrix} [\vec{\mathbf{x}}_{21} \quad -\vec{\mathbf{x}}_{43}] \begin{bmatrix} a \\ b \end{bmatrix} = \begin{bmatrix} \vec{\mathbf{x}}_{21} \\ -\vec{\mathbf{x}}_{43} \end{bmatrix} [\vec{\mathbf{x}}_{31}]$$

$$\begin{bmatrix} \vec{\mathbf{x}}_{21} \cdot \vec{\mathbf{x}}_{21} & -\vec{\mathbf{x}}_{21} \cdot \vec{\mathbf{x}}_{43} \\ -\vec{\mathbf{x}}_{43} \cdot \vec{\mathbf{x}}_{21} & \vec{\mathbf{x}}_{43} \cdot \vec{\mathbf{x}}_{43} \end{bmatrix} \begin{bmatrix} a \\ b \end{bmatrix} = \begin{bmatrix} \vec{\mathbf{x}}_{21} \cdot \vec{\mathbf{x}}_{31} \\ -\vec{\mathbf{x}}_{43} \cdot \vec{\mathbf{x}}_{31} \end{bmatrix}$$



# Continuous Intersection Tests

- Assume all vertices move with constant velocity over the time step

- Suppose contact occurs at time  $t$
- $\mathbf{x}_i(t)$  is the vertex  $\mathbf{x}_i$  at time  $t$ ,  $\mathbf{v}_i$  is the velocity of  $\mathbf{x}_i$
- $\vec{\mathbf{x}}_{ij}(t) = \mathbf{x}_{ij} + t\vec{\mathbf{v}}_{ij}$

- For Vertex-Face contact, vertices  $\mathbf{x}_1, \mathbf{x}_2, \mathbf{x}_3, \mathbf{x}_4$  are coplanar

- $\mathbf{x}_4$  is contained in the triangle:  $\mathbf{x}_4(t) = u\vec{\mathbf{x}}_{31}(t) + v\vec{\mathbf{x}}_{21}(t)$

- This is a non-linear system and hard to solve

- But we could apply use orthogonality:

$$\begin{aligned}\vec{\mathbf{x}}_{41}(t) \cdot \vec{\mathbf{N}} &= \vec{\mathbf{x}}_{41}(t) \cdot (\vec{\mathbf{x}}_{21}(t) \times \vec{\mathbf{x}}_{31}(t)) \\ &= (\vec{\mathbf{x}}_{41} + t\vec{\mathbf{v}}_{41}) \cdot ((\vec{\mathbf{x}}_{21} + t\vec{\mathbf{v}}_{21}) \times (\vec{\mathbf{x}}_{31} + t\vec{\mathbf{v}}_{31})) = 0\end{aligned}$$

a *cubic* equation for  $t$ ,  
solve it and plug it back to solve  $u, v$

- For Edge-Edge contact, 4 points are coplanar

$$(\vec{\mathbf{x}}_{21}(t) \times \vec{\mathbf{x}}_{43}(t)) \cdot \vec{\mathbf{x}}_{13}(t) = 0$$



# Collision Response

- Given a surface particle  $\mathbf{q}$  of object A is inside a tetrahedron  $\mathbf{t}$  of an object B
- In the case of self-collision, two objects are identical
  - It's far from trivial to find a stable way to resolve such collision
- Decide the future position of the penetrated vertex  $\mathbf{q}$ 
  - Move  $\mathbf{q}$  to the closest surface point of object B
    - Not recommended due to stability issue
  - Move  $\mathbf{q}$  back to where it penetrated object B
    - Backward ray casting along the surface normal at  $\mathbf{q}$
- Then the collision response force is defined as  $\mathbf{f}_{coll} = k(\mathbf{q}' - \mathbf{q})$ 
  - $k$  is stiffness coefficient

# Practical Issues

---

- The object behavior depends on the spring network
- Hard to get desired behavior by tweaking the stiffness of relevant springs
- Simulate inextensibility would make system '*stiff*'
- Can't capture volumetric effects directly

# Mass-Spring Model for Hair Simulation

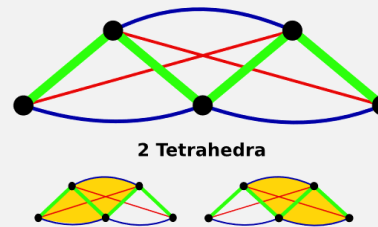
## ■ Hair is properties

- Bending
- Inextensibility
- Torsion

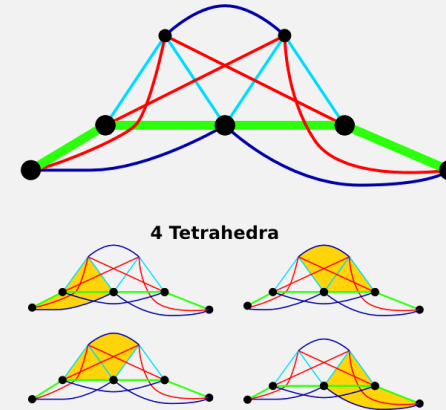


[Iben et al., SCA'13]

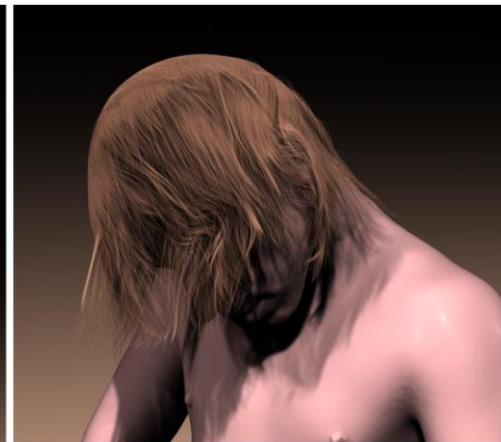
(a) Curly Hair Springs



(b) Straight Hair Springs



- Edge Springs (desired hair curve)
- Extra Edge Springs (form triangles)
- Bending Springs (prevent bend)
- Torsion Springs (prevent twist)
- Tetrahedral Altitude Springs (prevent collapse)



[Selle et al., SIG'08]

# Hair Troubles in Disney's *Tangled*



<https://www.youtube.com/watch?v=9K-Gv4XVb10>

# Rigid Body Dynamics

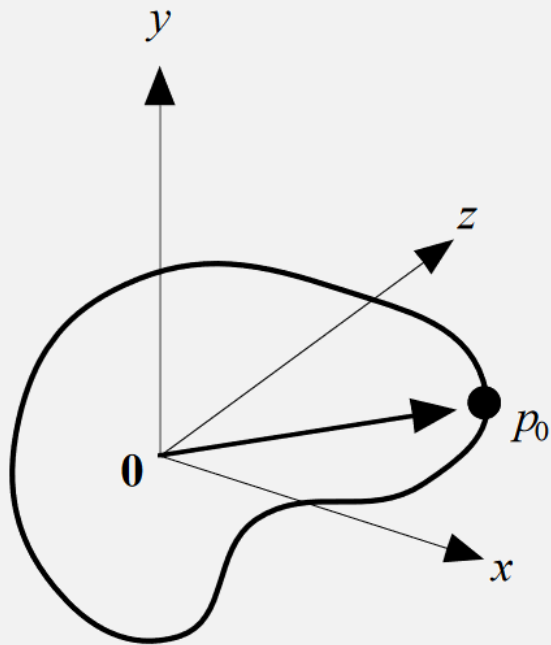


# Rigid Body

- Collection of particles
- No deformation: the distance between any two particles is constant
- Rigid motion: rotation + translation  $\Rightarrow$  6 DOFs

# Orientation

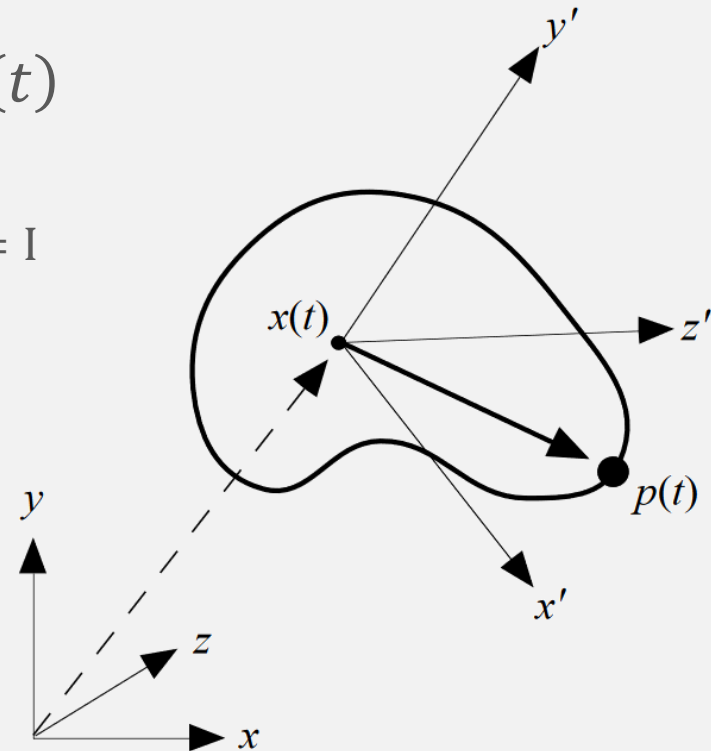
body space



$$p(t) = R(t)p_0 + x(t)$$

$$R(t)^T R(t) = R(t) R(t)^T = I$$

world space



# Center of Mass and Body Velocity

- The coordinate of body's geometric center is

- $(0, 0, 0)$  in body space

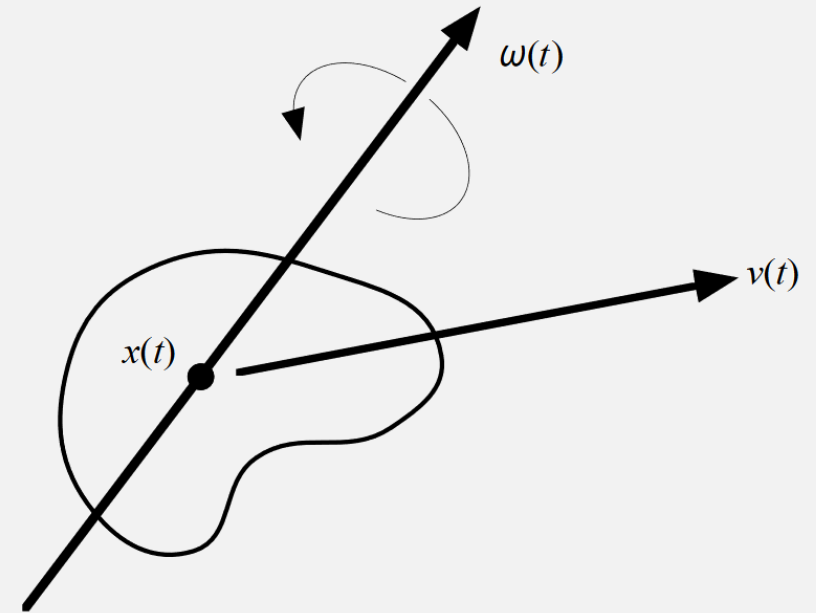
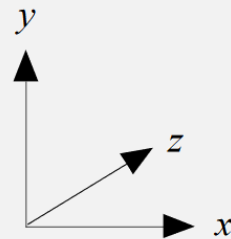
$$\sum m_i \mathbf{p}_i(0) = \mathbf{0}$$

- $\mathbf{x}(t)$  in world space

$$\mathbf{x}(t) = \frac{\sum m_i \mathbf{p}_i(t)}{\sum m_i}$$

- The body velocity in world space

$$\mathbf{v}(t) = \frac{d\mathbf{x}(t)}{dt} = \dot{\mathbf{x}}(t)$$





# Orientation

- The columns of  $R(t)$ :

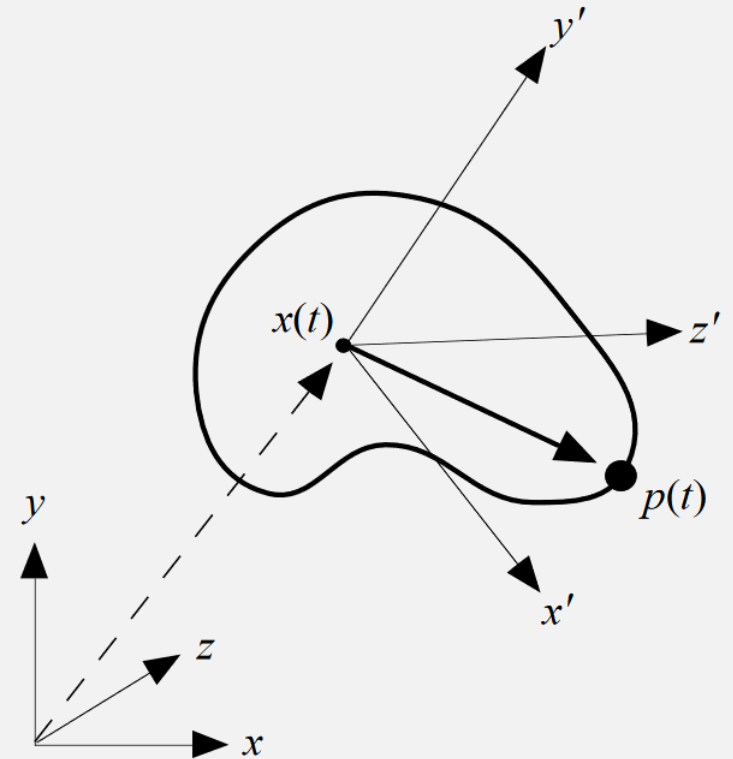
- The directions of the transformed  $x, y, z$  axes in body space at time  $t$

$$p(t) = R(t)p(t) + x(t)$$

$$R(t) \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix} = \begin{pmatrix} r_{xx} \\ r_{xy} \\ r_{xz} \end{pmatrix} = x'$$

$\Downarrow$

$$R(t) = \begin{pmatrix} r_{xx} & r_{yx} & r_{zx} \\ r_{xy} & r_{yy} & r_{zy} \\ r_{xz} & r_{yz} & r_{zz} \end{pmatrix}$$



# Angular Velocity

■  $\omega(t)$  is a vector represents the axis of spin

■ The magnitude  $|\omega(t)|$  is the speed of spin

■ How are  $R(t)$  and  $\omega(t)$  related?

■  $r(t) = a + b$ , is a vector fixed to the body

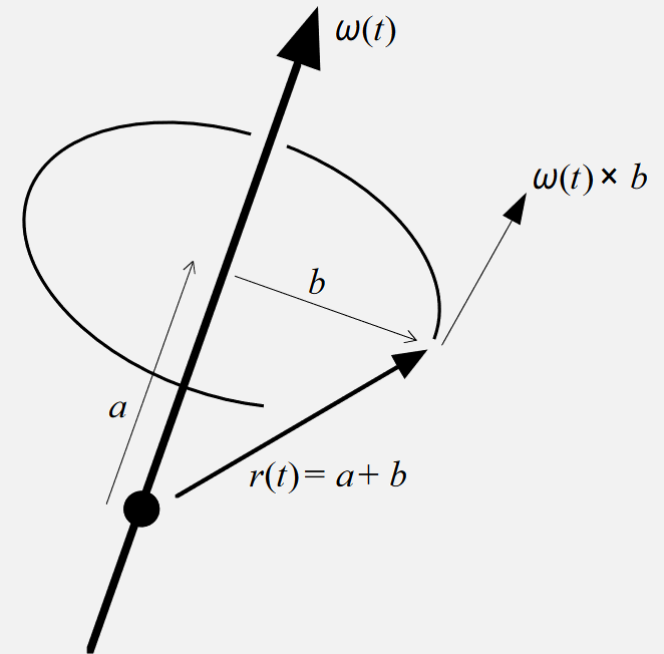
□ translation-invariant

• Its time derivative is only related to rotation

□ moving in a circle of radius  $b$

■  $\dot{r}(t) = \omega(t) \times b = \omega(t) \times b + \boxed{\omega(t) \times a} \stackrel{=0}{=}$

$$= \omega(t) \times (a + b) = \omega(t) \times r(t)$$



# Skew Matrix

$$\forall \mathbf{a}, \mathbf{b} \in \mathbb{R}^3$$

$$\mathbf{a} \times \mathbf{b} = \begin{pmatrix} a_y b_z - b_y a_z \\ -a_x b_z + b_x a_z \\ a_x b_y - b_x a_y \end{pmatrix} = \begin{pmatrix} 0 & -a_z & a_y \\ a_z & 0 & -a_x \\ -a_y & a_x & 0 \end{pmatrix} \begin{pmatrix} b_x \\ b_y \\ b_z \end{pmatrix} = \mathbf{a}^* \mathbf{b}$$

# Change of Orientation

■ The columns of  $\dot{R}(t)$

■ The velocity with which the x, y, z axes are being transformed

■ Given  $\dot{r}(t) = \omega(t) \times r(t)$ , and apply it to x, y, z-axis:

$$\begin{aligned}\dot{R}(t) &= \left( \omega(t) \times \begin{pmatrix} r_{xx} \\ r_{xy} \\ r_{xz} \end{pmatrix} \quad \omega(t) \times \begin{pmatrix} r_{yx} \\ r_{yy} \\ r_{yz} \end{pmatrix} \quad \omega(t) \times \begin{pmatrix} r_{zx} \\ r_{zy} \\ r_{zz} \end{pmatrix} \right) \\ &= \omega^*(t) \left( \begin{pmatrix} r_{xx} \\ r_{xy} \\ r_{xz} \end{pmatrix} \quad \begin{pmatrix} r_{yx} \\ r_{yy} \\ r_{yz} \end{pmatrix} \quad \begin{pmatrix} r_{zx} \\ r_{zy} \\ r_{zz} \end{pmatrix} \right) = \omega^*(t) R(t)\end{aligned}$$

$$\dot{R}(t) = \omega(t)^* R(t)$$

# Velocity of a Particle

$$r_i(t) = R(t)r_{0i} + x(t)$$

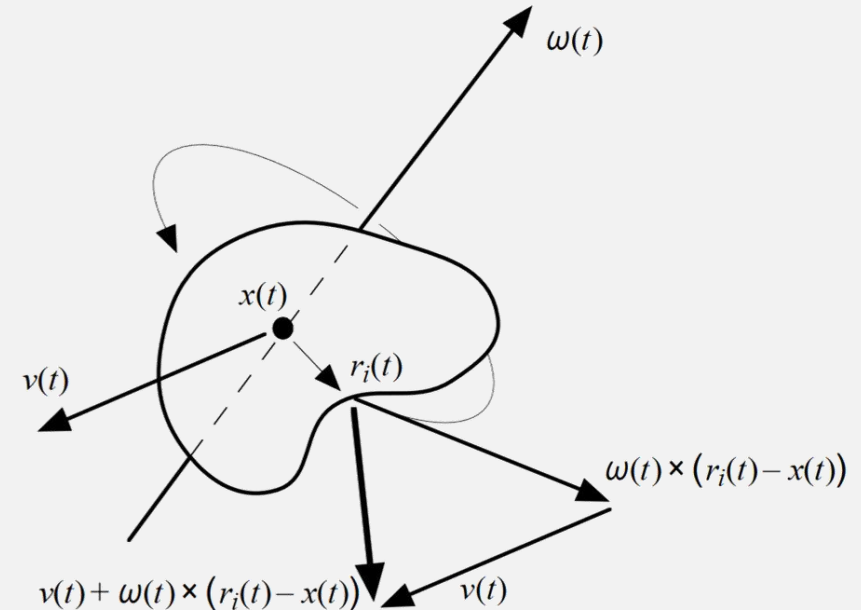
$\Downarrow$  *time derivative*

$$\dot{r}_i(t) = \dot{R}(t)r_{0i} + \dot{x}(t) = \omega(t) \times R(t)r_{0i} + v(t)$$

$$= \omega(t) \times (R(t)r_{0i} + \cancel{x(t)} - \cancel{x(t)}) + v(t)$$

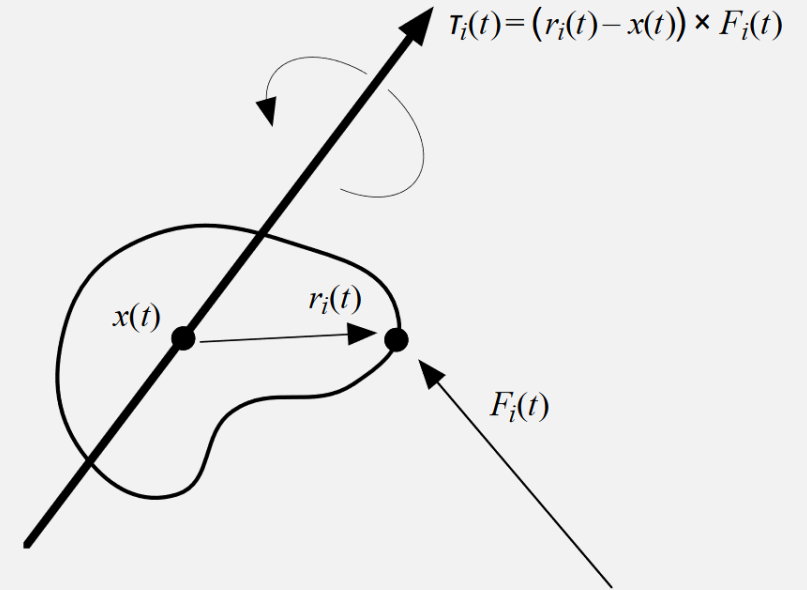
$$= \omega(t) \times (r_i(t) - x(t)) + v(t)$$

$$= \text{angular component} + \text{linear component}$$



# Force and Torque

- Conceptual model: external force acting on a particular *particle* of the body
- The total external force:  $F(t) = \sum F_i(t)$ 
  - Convey no information about where the various forces acting on
- The external torque acting on the  $i$ th particle
  - $\tau_i(t) = (r_i(t) - x(t)) \times F_i(t)$
- Total external torque provide some information about the distribution of the forces  $F_i(t)$  over the body
  - $\tau(t) = \sum \tau_i(t) = \sum (r_i(t) - x(t)) \times F_i(t)$



# Linear Momentum

- For single particle  $p$ , its linear momentum is  $p_i = m_i v_i$
- While the total linear momentum of a rigid body at time  $t$  is

$$\begin{aligned} P(t) &= \sum_i m_i \dot{r}_i(t) \\ &= \sum_i \left( m_i v(t) + m_i \omega_i(t) \times (r_i(t) - x(t)) \right) \\ &= \sum_i m_i v(t) + \omega_i(t) \times \boxed{\sum_i m_i (r_i(t) - x(t))} = 0 \\ &= M v(t) \end{aligned}$$

- The change in linear momentum is the total force exerting on the body

$$\dot{v}(t) = \frac{\dot{P}(t)}{M} \Rightarrow \dot{P}(t) = F(t)$$

# Angular Momentum

- Angular momentum of a single particle is  $L_i = r_i \times (m_i v_i) = r_i \times p_i$
- All particles within the body has the same angular velocity  $\omega$ , thus their linearly velocity  $v_i = \omega \times r_i$
- The total angular momentum is

$$\begin{aligned} L(t) &= \sum_i r_i(t) \times m_i v_i(t) \\ &= \sum_i r_i(t) \times m_i (\omega \times r_i(t)) \\ &= \sum_i -m_i r_i(t) \times (r_i(t) \times \omega(t)) \\ &= -m_i r_i^*(t) r_i^*(t) \omega = I(t) \omega(t) \end{aligned}$$



# The Inertia Tensor

Let  $r'_i = r_i(t) - x(t)$

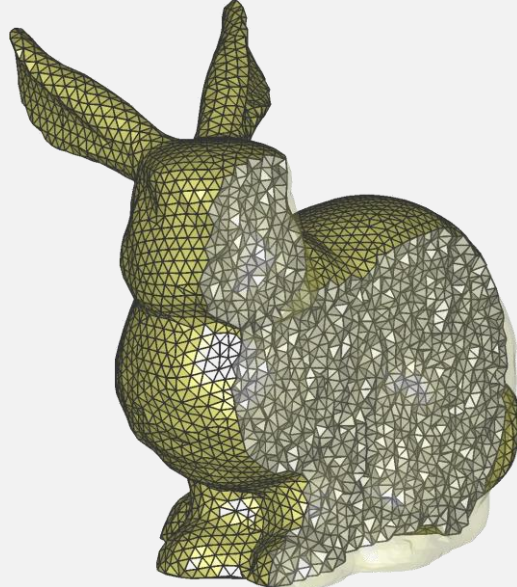
$$\begin{aligned}
 I(t) &= \sum \begin{pmatrix} m_i(r_{iy}'^2 + r_{iz}'^2) & -m_i r'_{ix} r'_{iy} & -m_i r'_{ix} r'_{iz} \\ -m_i r'_{iy} r'_{ix} & m_i(r_{ix}'^2 + r_{iz}'^2) & -m_i r'_{iy} r'_{iz} \\ -m_i r'_{iz} r'_{ix} & -m_i r'_{iz} r'_{iy} & m_i(r_{ix}'^2 + r_{iy}'^2) \end{pmatrix} \\
 &= \sum m_i r_i'^T r_i' \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix} - \begin{pmatrix} m_i r_{ix}'^2 & m_i r'_{ix} r'_{iy} & m_i r'_{ix} r'_{iz} \\ m_i r'_{iy} r'_{ix} & m_i r_{iy}'^2 & m_i r'_{iy} r'_{iz} \\ m_i r'_{iz} r'_{ix} & m_i r'_{iz} r'_{iy} & m_i r_{iz}'^2 \end{pmatrix} \left( \begin{matrix} r'_{ix} \\ r'_{iy} \\ r'_{iz} \end{matrix} \right) (r'_{ix} \quad r'_{iy} \quad r'_{iz}) = r' r'^T \\
 &= \sum m_i ((r_i'^T r_i') \mathbf{1} - r' r'^T) \\
 &\Downarrow \\
 I(t) &= R(t) \left( \sum m_i ((r_{0i}^T r_{0i}) \mathbf{1} - r_{0i} r_{0i}^T) \right) R(t)^T = R(t) I_{body} R(t)^T
 \end{aligned}$$

$r'_i = R(t) r_{0i}, \quad R(t) R(t)^T = \mathbf{1}$

precompute it!

# Approximating $I_{body}$

- Goal: discretize the mesh volume and accumulate inertia tensor
- Bounding boxes, spheres
  - Simple but may not fit to geometry properly
- Convex decomposition
  - Tetrahedralized mesh



# Forward Euler Integration

$$\frac{d}{dt}\mathbf{X}(t) = \frac{d}{dt}\begin{pmatrix} x(t) \\ R(t) \\ P(t) \\ L(t) \end{pmatrix} = \begin{pmatrix} v(t) \\ \omega(t)^* R(t) \\ F(t) \\ \tau(t) \end{pmatrix}$$

$$v(t) = P(t)/M$$

$$I(t) = R(t)I_{body}R(t)^T$$

$$\omega(t) = I(t)^{-1}L(t)$$

## initialization

$$M \leftarrow \sum_i m_i$$

$$\bar{x}_0 \leftarrow \frac{\sum_i x_{0i}}{M}$$

$$r_{0i} \leftarrow x_{0i} - \bar{x}_0$$

$$I_{body}^{-1} \leftarrow \left( -\sum_i m_i r_{0i}^* r_{0i}^* \right)^{-1}$$

initlaize  $\mathbf{x}, \mathbf{v}, \mathbf{R}, \mathbf{L}$

$$I^{-1} \leftarrow R I_{body}^{-1} R^T$$

$$\omega \leftarrow I^{-1}L$$

## simulation loop

$$F \leftarrow \sum_i f_i$$

$$\tau \leftarrow \sum_i r_i \times f_i$$

$$x \leftarrow x + v\Delta t$$

$$v \leftarrow v + \frac{F}{M}\Delta t$$

$$R \leftarrow R + \omega^* R\Delta t$$

$$L \leftarrow L + \tau\Delta t$$

$$I^{-1} \leftarrow R I_0^{-1} R^T$$

$$\omega \leftarrow I^{-1}L$$

$$r_i \leftarrow R r_{0i}$$

$$x_i \leftarrow x + r_i$$

$$v_i \leftarrow v + \omega \times r_i$$

# Rotation Matrix vs. Unit Quaternion

- Most important reason to avoid rotation matrix is the *numerical drift* in

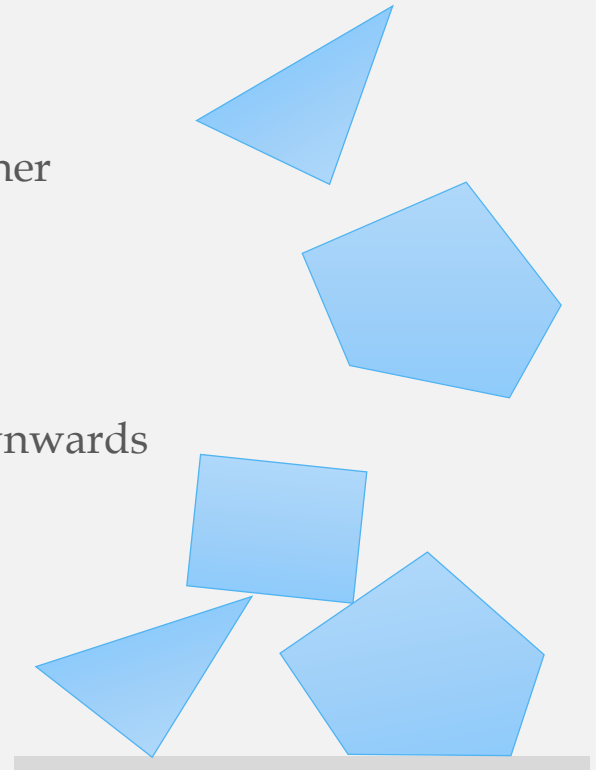
$$\dot{R}(t) = \omega^*(t)R(t)$$

- The numerical errors in the 9 coefficients in  $R(t)$
- Unit quaternion has far less drift because it only has one extra variable
  - The drift could be resolved by renormalization

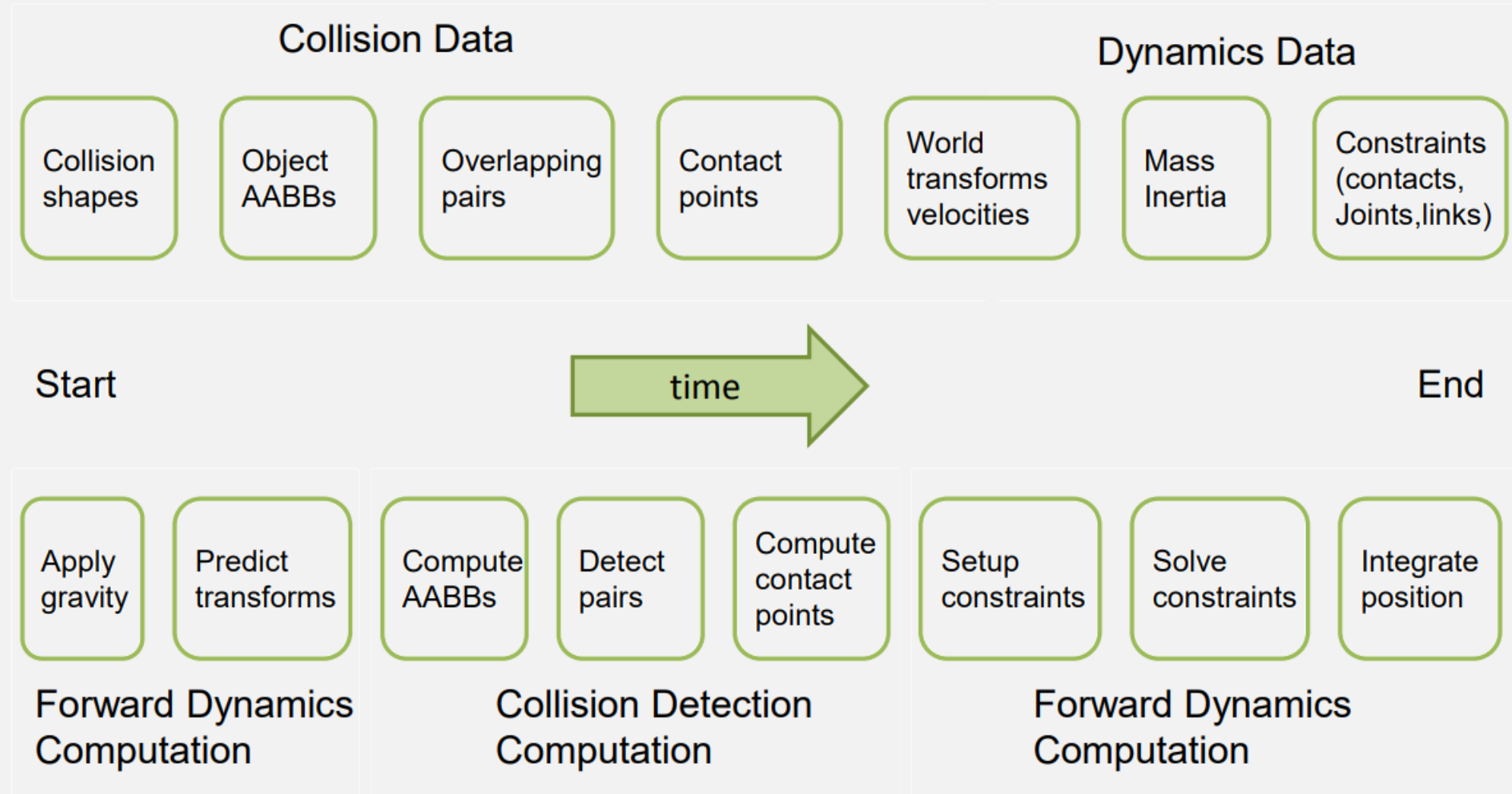
$$\dot{q}(t) = \frac{1}{2}(0, \omega(t)) * q(t)$$

# Types of Rigid Body Contact

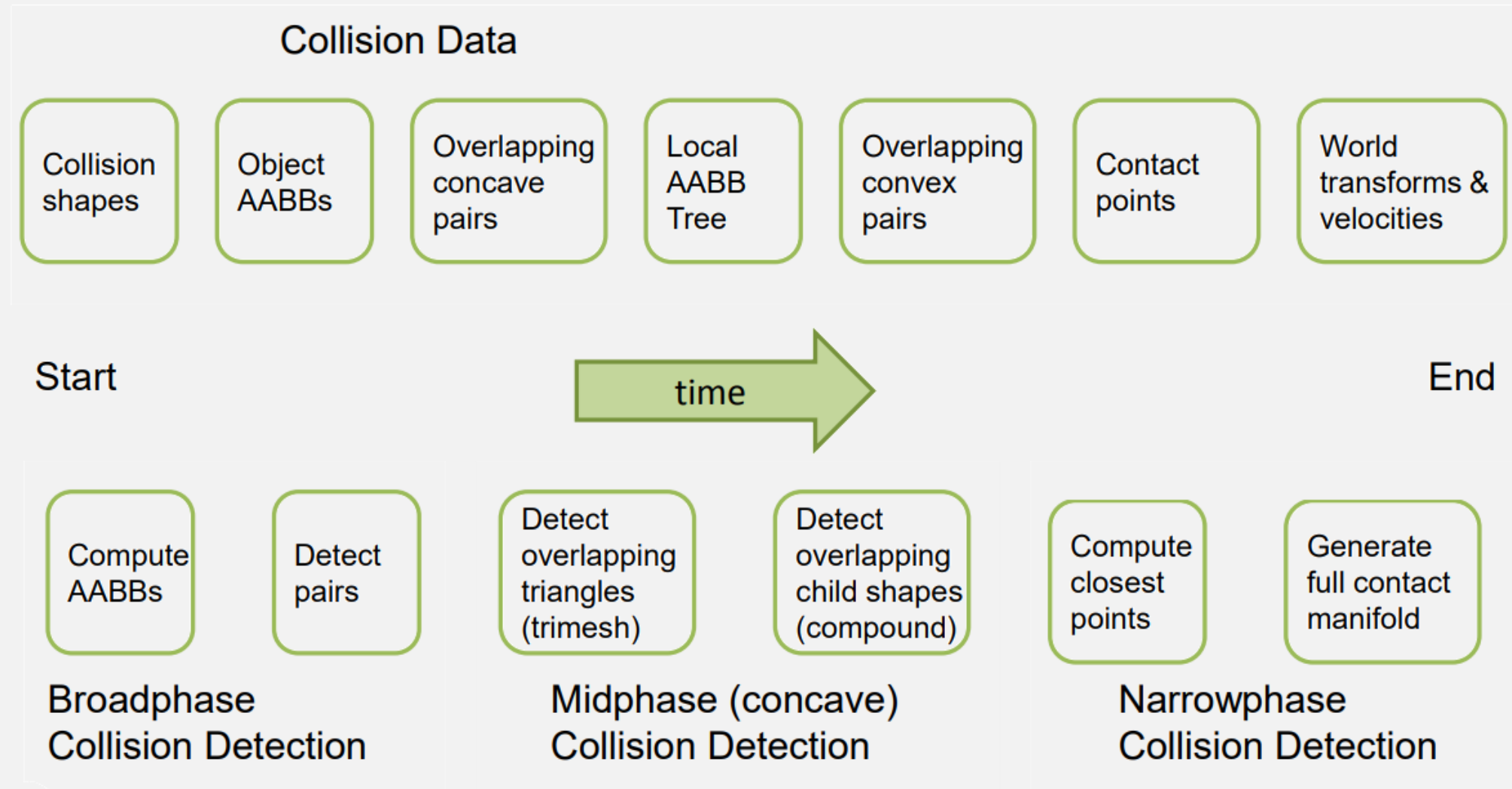
- Rigid bodies are non-flexible  $\Rightarrow$  no inter-penetration
- At the instant of contact, particles have to change the velocity abruptly
- There are two types of collision contacts
  - *Colliding contact*
    - Two bodies are in contact at point  $p$ , and their velocity towards each other
    - Requires an instantaneous change in velocity
  - *Resting contact*
    - Bodies are resting on one another at some point  $p$
    - Compute a *contact force* that prevents the particle from accelerating downwards



# Multi Physics Pipeline in Bullet



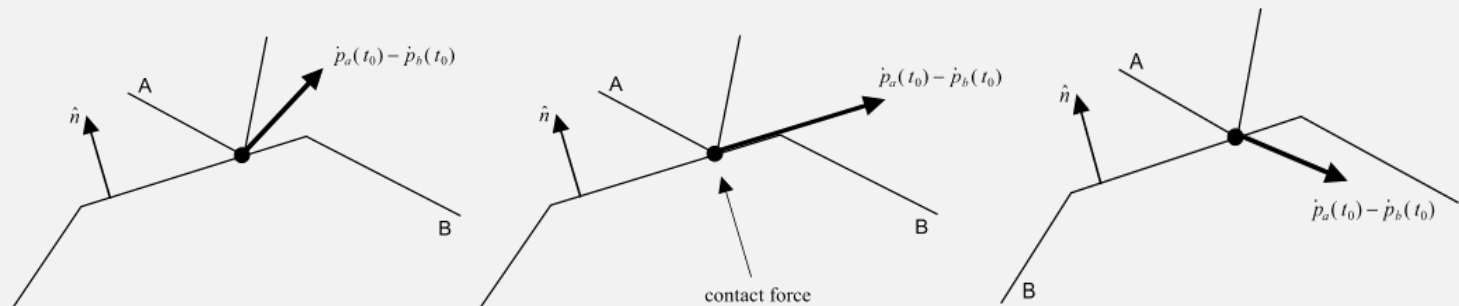
# Collision Detection Pipeline in Bullet



# Handling Colliding Contact

- Goal: find the changes of velocities of two bodies
  - Only consider two generic cases: vertex-face and edge-edge contacts
- Contact normal  $\hat{\mathbf{n}}$ 
  - vertex-face: the outwards pointing unit normal of the contact face
  - edge-edge: the cross product of two edges
- Relative velocity:  $v_{rel} = \hat{\mathbf{n}}(t_0) \cdot (\dot{\mathbf{p}}_a(t_0) - \dot{\mathbf{p}}_b(t_0))$ 
  - $\dot{\mathbf{p}}_i(t_0) = \mathbf{v}_i(t_0) + \boldsymbol{\omega}_i(t_0) \times (\mathbf{p}_i(t_0) - \mathbf{x}_i(t_0))$

$$v_{rel} \begin{cases} > 0 & \text{moving apart} \\ = 0 & \text{resting contact} \\ < 0 & \text{colliding} \end{cases}$$





# Collision Impulse J

## ■ Impulse $\mathbf{J} = \mathbf{F}\Delta t$

- Concept: apply very large force in a very short amount of time (like delta function)
- Result in a finite change in momentum  $M\Delta\mathbf{v}$
- Impulses are always exchanged along contact normal  $\hat{\mathbf{n}}(t_0)$ ,  $\mathbf{J} = j\hat{\mathbf{n}}$

## ■ Impulse could change both linear and angular momentum of the object

$$\begin{array}{l} \Delta\mathbf{P} = \mathbf{J} \\ \Delta\mathbf{L} = \mathbf{r}_a \times \mathbf{J} = j(\mathbf{r}_a \times \hat{\mathbf{n}}) \end{array} \Rightarrow \begin{array}{l} \Delta\mathbf{v}_a = \frac{\Delta\mathbf{P}}{m_a} = \frac{j\hat{\mathbf{n}}}{m_a} \\ \Delta\boldsymbol{\omega}_a = \mathbf{I}^{-1}\Delta\mathbf{L} = j\mathbf{I}^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}) \end{array}$$

$$\begin{aligned} \dot{\mathbf{p}}_a^+ &= (\mathbf{v}_a + \Delta\mathbf{v}_a) + (\boldsymbol{\omega}_a + \Delta\boldsymbol{\omega}_a) \times \mathbf{r}_a \\ &= \mathbf{v}_a + \boldsymbol{\omega}_a \times \mathbf{r}_a + \Delta\mathbf{v}_a + \Delta\boldsymbol{\omega}_a \times \mathbf{r}_a \\ &= \dot{\mathbf{p}}_a + \Delta\mathbf{v}_a + \Delta\boldsymbol{\omega}_a \times \mathbf{r}_a = \dot{\mathbf{p}}_a + j \left( \frac{\hat{\mathbf{n}}}{m_a} + \mathbf{I}_a^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}) \times \mathbf{r}_a \right) \end{aligned}$$

# Handling Colliding Contact (Cont.)

- For colliding contacts, the relative velocity after collision  $v_{rel}^+$  has to be

$$v_{rel}^+ = \hat{\mathbf{n}} \cdot (\dot{\mathbf{p}}_a^+ - \dot{\mathbf{p}}_b^+) \geq 0, \quad v_{rel}^+ = -\varepsilon v_{rel}^-, \quad \varepsilon \in \mathbb{R}$$

- $\varepsilon$  is the restitution coefficient:  $\varepsilon = 1$  for fully elastic, while  $\varepsilon = 0$  for fully inelastic

$$\begin{aligned} v_{rel}^+ &= \hat{\mathbf{n}} \cdot \left( \dot{\mathbf{p}}_a + j \left( \frac{\hat{\mathbf{n}}}{m_a} + \mathbf{I}_a^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}) \times \mathbf{r}_a \right) - \dot{\mathbf{p}}_b - j \left( \frac{\hat{\mathbf{n}}}{m_b} + \mathbf{I}_b^{-1}(\mathbf{r}_b \times \hat{\mathbf{n}}) \times \mathbf{r}_b \right) \right) \\ &= v_{rel}^- + \left( j \left( \frac{1}{m_a} + \mathbf{I}_a^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}) \times \mathbf{r}_a \cdot \hat{\mathbf{n}} \right) - j \left( \frac{1}{m_b} + \mathbf{I}_b^{-1}(\mathbf{r}_b \times \hat{\mathbf{n}}) \times \mathbf{r}_b \cdot \hat{\mathbf{n}} \right) \right) = -\varepsilon v_{rel}^- \end{aligned}$$

*solve*

$$j = \frac{-(1 + \varepsilon)v_{rel}^-}{\frac{1}{m_a} + \mathbf{I}_a^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}) \times \mathbf{r}_a \cdot \hat{\mathbf{n}} + \frac{1}{m_b} + \mathbf{I}_b^{-1}(\mathbf{r}_b \times \hat{\mathbf{n}}) \times \mathbf{r}_b \cdot \hat{\mathbf{n}}}$$

*update*

$$\mathbf{v}_a^+ = \mathbf{v}_a + \frac{j\hat{\mathbf{n}}}{m_a}, \quad \mathbf{v}_b^+ = \mathbf{v}_b + \frac{j\hat{\mathbf{n}}}{m_b}$$

$$\boldsymbol{\omega}_a^+ = \boldsymbol{\omega}_a + j\mathbf{I}_a^{-1}(\mathbf{r}_a \times \hat{\mathbf{n}}), \quad \boldsymbol{\omega}_b^+ = \boldsymbol{\omega}_b + j\mathbf{I}_b^{-1}(\mathbf{r}_b \times \hat{\mathbf{n}})$$

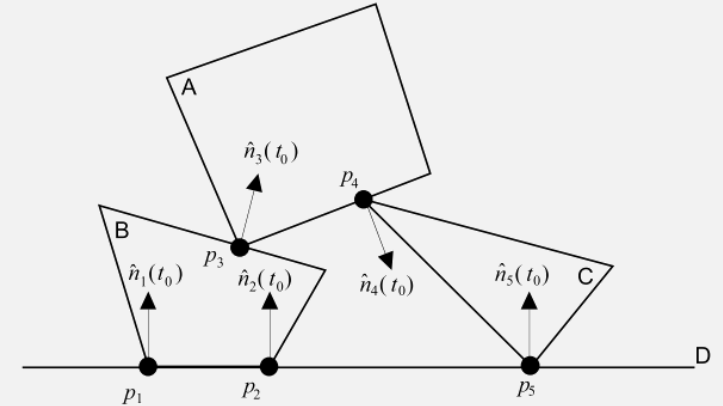
# Handling Resting Contacts

■ Goal: determine all *contact force* at the same time

■ Contact force is along contact normal:  $\mathbf{f}_i = f_i \hat{\mathbf{n}}_i(t_0)$

■ Requirements for contact forces

1. Must prevent inter-penetration
2. Repulsive: push bodies apart, never act like glue
3. Become zero if the bodies begin to separate



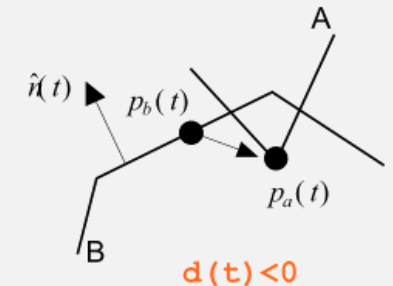
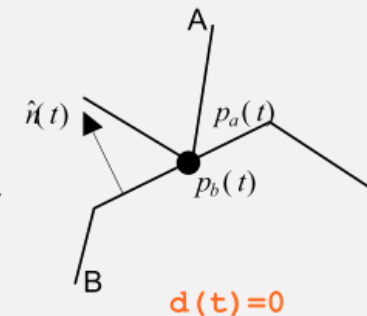
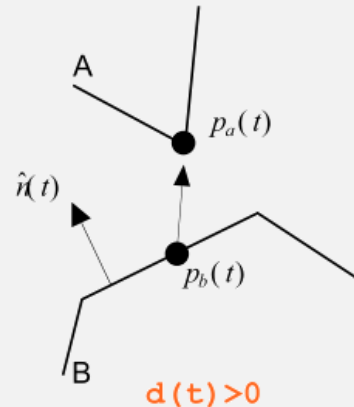
■ Separation distance  $d_i(t) = \hat{\mathbf{n}}_i(t) \cdot (\mathbf{p}_a(t) - \mathbf{p}_b(t))$

■ The separation between A and B near  $\mathbf{p}_a(t)$

□  $d_i(t) = 0$ : at  $i$ -th contact point

□  $d_i(t) > 0$ : lost contact

□  $d_i(t) < 0$ : inter-penetrate



# Handling Resting Contacts (Cont.)

- At the contact moment  $d_i(t_0) = 0$ , we want to keep it from decreasing at time  $t_0$  (e.g.  $\dot{d}_i(t_0) \geq 0$ )

- The separation velocity at time  $t$

$$\dot{d}_i(t) = \hat{\mathbf{n}}_i(t) \cdot (\mathbf{p}_a(t) - \mathbf{p}_b(t)) + \hat{\mathbf{n}}_i(t) \cdot (\dot{\mathbf{p}}_a(t) - \dot{\mathbf{p}}_b(t))$$

- At time  $t_0$  when contact occurs

$$\square \mathbf{p}_a(t_0) = \mathbf{p}_b(t_0) \Rightarrow \dot{d}_i(t_0) = \hat{\mathbf{n}}_i(t_0) \cdot (\dot{\mathbf{p}}_a(t_0) - \dot{\mathbf{p}}_b(t_0))$$

- For resting contact  $d_i(t_0) = \dot{d}_i(t_0) = 0$ , and the second time derivative is:

$$\begin{aligned} \ddot{d}_i(t_0) &= \ddot{\mathbf{n}}_i(t_0) \cdot (\mathbf{p}_a(t_0) - \mathbf{p}_b(t_0)) \stackrel{=0}{=} + 2\dot{\mathbf{n}}_i(t_0) \cdot (\dot{\mathbf{p}}_a(t_0) - \dot{\mathbf{p}}_b(t_0)) + \hat{\mathbf{n}}_i(t_0) \cdot (\ddot{\mathbf{p}}_a(t_0) - \ddot{\mathbf{p}}_b(t_0)) \\ &= 2\dot{\mathbf{n}}_i(t_0) \cdot (\dot{\mathbf{p}}_a(t_0) - \dot{\mathbf{p}}_b(t_0)) + \hat{\mathbf{n}}_i(t_0) \cdot (\ddot{\mathbf{p}}_a(t_0) - \ddot{\mathbf{p}}_b(t_0)) \end{aligned}$$

# Solve Contact Forces

- $\ddot{d}_i(t_0)$  measures how the two bodies are accelerating towards each other
- Formulate the requirements of contact forces
  1.  $\ddot{d}_i(t_0) \geq 0$ : prevent inter-penetration
  2.  $f_i \geq 0$ : be repulsive
  3.  $f_i \ddot{d}_i(t_0) = 0$ : become zero if the bodies begin to separate
- To find the contact forces satisfying above requirements, we express each  $\ddot{d}_i(t_0)$  as a function of the unknown  $f_i$ 's

$$\ddot{d}_i(t_0) = a_{i1}f_1 + a_{i2}f_2 + \cdots + a_{in}f_n + b_i$$

$$\begin{pmatrix} \ddot{d}_1(t_0) \\ \vdots \\ \ddot{d}_n(t_0) \end{pmatrix} = A \begin{pmatrix} f_1 \\ \vdots \\ f_n \end{pmatrix} + \begin{pmatrix} b_1 \\ \vdots \\ b_n \end{pmatrix}$$

# WEE GAFFES

SOMETIMES THE COMPUTER  
MAKES MISTAKES.

IT'S NOT OUR FAULT.





# GUARDIANS OF THE GALAXY

## VOL. 2



# References

---

- Real-Time Physics, SIGGRAPH Course Notes, 2008.
- David Baraff and Andrew Witkin, Physically Based Modeling, SIGGRAPH Course Notes, 2001.
- Interactive Simulation of Rigid Body Dynamics in Computer Graphics, STAR, EUROGRAPHICS, 2012.
- Donald House, John C. Keyser, Foundations of Physically Based Modeling and Animation, CRC Press, 2017.