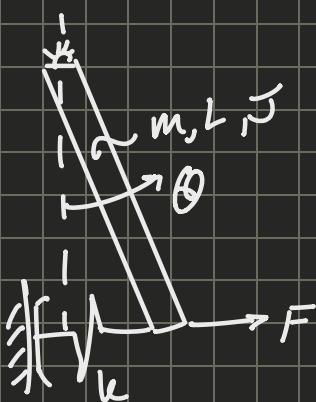


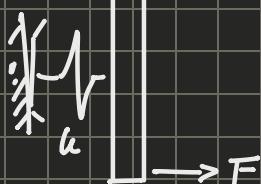
Lezione 2 -

Start of Systematic approach formulating the equation of motion

Starting with 1 d.o.f.:



Unloaded system
Reference system



1 d.o.f.

$\theta \rightarrow$ free variable

First step is usually counting the d.o.f and choosing the proper (simplest ones). In this case it's trivial.

We also need to place our system of reference and the convention of the motion.

For Lagrange we need to fix 3 quantities:

$E_C, V, \delta \mathcal{L}$, also q, \dot{q}, \ddot{q}

$$E_C = \frac{1}{2} m v_a^2 + \frac{1}{2} J w^2 \rightarrow \text{König Theorem} \rightarrow \text{general formula for each body.}$$

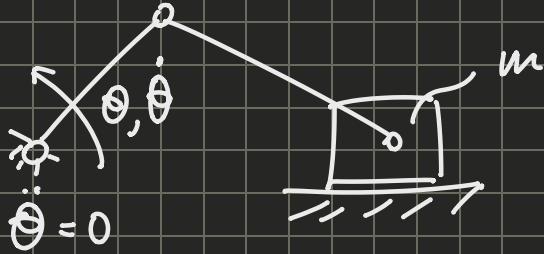
v_a and w are physical quantities, we need to figure out how to write them as function of our free variables.
 θ in this case

$$= \frac{1}{2} m^*(q) \dot{\theta}^2$$

Reduces

mass which
is a function
of the position

Derivative of the degree of freedom



$$E_C = \frac{1}{2} m(\theta) \dot{\theta}^2$$

$m^*(\theta)$ is the reduced mass as a function of θ

$m^*(q)$ is the reduced mass / or moment of inertia, related to the physical system. $\text{if } q = \theta$

$E_C = \frac{1}{2} m^*(\theta) \dot{\theta}^2$, general way of writing the kinetic energy.

In the original problem m is not a function of θ , but in the pivot it is as θ and $\dot{\theta}$ are related.

$$\underbrace{\frac{d}{dt} \left(\frac{\partial E_C}{\partial \dot{q}} \right)}_{\text{Left side}} - \frac{\partial E_C}{\partial q} + \frac{\partial V}{\partial q} = Q$$

$$\frac{\partial E_C}{\partial \dot{q}} = m^*(q) \dot{q}$$

$$\begin{aligned} \rightarrow \frac{d}{dt} \left(\frac{\partial E_C}{\partial \dot{q}} \right) &= m^*(q) \ddot{q} + \frac{\partial m^*(q)}{\partial q} \cdot \frac{dq}{dt} \dot{q} \\ &= m^*(q) \ddot{q} + \frac{\partial m^*(q)}{\partial q} \cdot \dot{q}^2 \end{aligned}$$

$$\frac{\partial E_C}{\partial q} = \frac{1}{2} \frac{\partial m^*(q)}{\partial q} \cdot \dot{q}^2$$

For each body we write this

$$\frac{d}{dt} \left(\frac{\partial E_C}{\partial \dot{q}} \right) - \frac{\partial E_C}{\partial q} = \overbrace{m^*(q) \ddot{q} + \frac{1}{2} \frac{\partial m^*(q)}{\partial q} \dot{q}^2}$$

\Rightarrow instead doing many derivatives, we only need to do one

If m is not dependent of the angle the first part is needed.

$m^*(q)$ can be whatever cor, sin, etc.

\rightarrow From this we are getting the inertial forces, we see that much of this is forces (or torques if the variables are Θ)

All these should be forces or torques, therefore from Lagrange we write dynamic equilibria.

The potential V are from gravitational forces and forces from springs.

We can split V in two components:

$$V = V_u + V_g = \underbrace{\frac{1}{2} k \Delta l^2}_{\text{Potential Energy of the spring}} + mg h_0 = \frac{1}{2} k \Delta l^2(q) + mg h_0(q)$$

Potential Energy of the spring

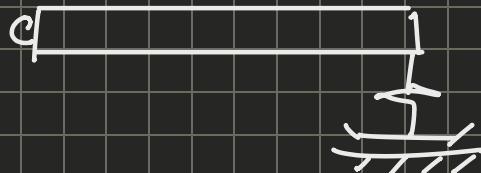
For each spring or mass we write these

We can split Δl in two parts:

$$V_u = \frac{1}{2} k \Delta l^2(q) = \frac{1}{2} k (\Delta l_0 + \Delta l_d(q))^2$$

Preload
of the spring

\hookrightarrow Difference in natural length and length in equilibrium position.



$\Delta l_0 \neq 0$, otherwise
we have motion.

$$\frac{\partial V_u}{\partial q} = k \Delta l_0 \underbrace{\frac{\partial \Delta l_d(q)}{\partial q}}_{\text{The term from } T_u \text{ one inertia}} + k \Delta l_d(q) \underbrace{\frac{\partial \Delta l_d(q)}{\partial q}}_{\text{These terms are the stiffness of the system.}}$$

\hookrightarrow Physically the variation of the length of the spring when we vary the d.o.f.

It is the relation between the length and the d.o.f.

We can walk on a tight rope,
even though they don't have flexion and stiffness, because
we preload the rope, allowing us to stay up.

$$k \Delta l_0 \frac{\partial \Delta l_d(q)}{\partial q} \rightarrow \text{tension in the rope}$$

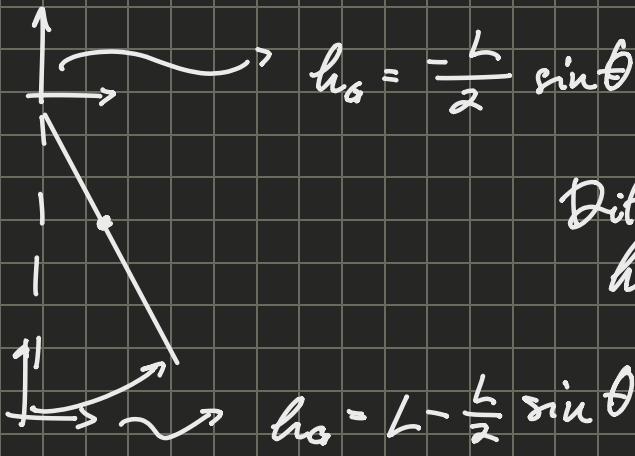
\hookrightarrow This is the kinematic relation, which

what is keeping us up because of the tension
we have pre-loaded into the system.

$$V_g = mg h_g(q)$$

$$\frac{\partial V_g}{\partial q} = mg \underbrace{\frac{\partial h_g(q)}{\partial q}}$$

Kinematic relation between vertical position of center of gravity and the d.o.f.



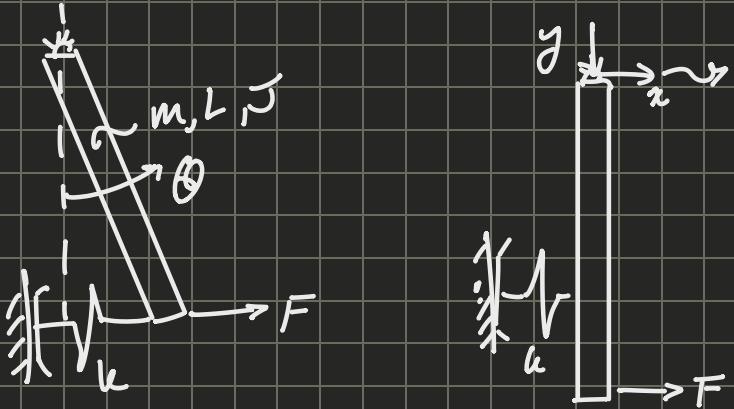
Different system have different h_g , but the derivative is the same.

Since the proportional difference doesn't change anything.

With only 3 derivatives we get the equation of motion.

We get a non-linear differential equation.

This is very abstract but once we apply it it becomes easier.



$$E_C = \frac{1}{2} m \underline{V_G}^2 + \frac{1}{2} \underline{\omega}^2$$

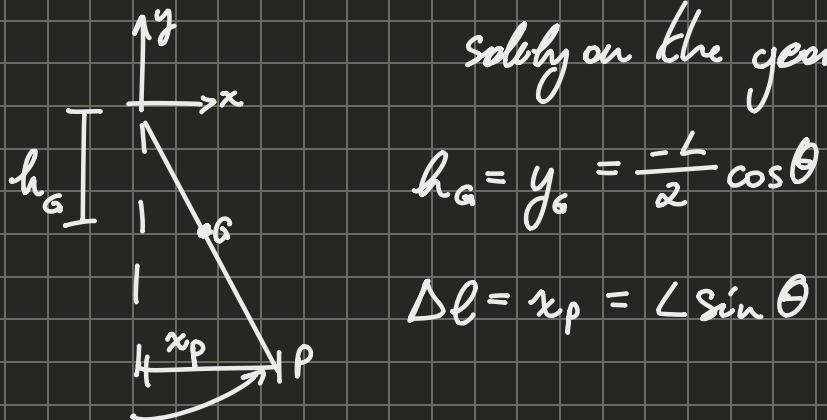
$$V = V_u + V_g = \frac{1}{2} k \underline{\Delta\ell}^2 + m g \underline{h}_g$$

First Step - transform the underlined to a function of θ

$$\underline{V_G}^2 = \frac{L^2}{4} \dot{\theta}^2$$

$$\underline{\omega}^2 = \dot{\theta}^2$$

For $\Delta\ell$ and h we can use something like this to focus solely on the geometry



$$h_g = y_g = -\frac{L}{2} \cos \theta$$

$$\Delta\ell = x_p = L \sin \theta$$

$$E_C = \frac{1}{2} \left(\frac{L}{4} m + J \right) \dot{\theta}^2$$

$$\frac{\partial E_C}{\partial q}$$

$$V_u = \frac{1}{2} k (L \sin \theta)^2$$

$$\frac{\partial V_u}{\partial q}$$

$$V_g = -m g \frac{L}{2} \cos \theta$$

$$\frac{\partial V_g}{\partial q}$$

As we said we can use the derivatives and plug them in as

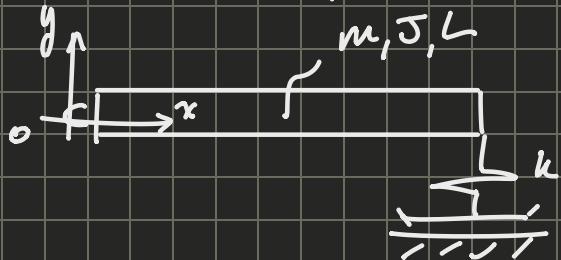
everything else will be the same

$$\left(\frac{L}{4}m + J\right)\ddot{\theta} + kL^2 \sin\theta \cos\theta + mg \frac{L}{2} \sin\theta = 0$$

m Inertial force
l Inertial moment of inertia
J Torsional moment of inertia
k Spring Tension Force

$\frac{\partial V_u}{\partial q}$
 $\frac{\partial V_g}{\partial q}$
Weight force

Other example:



$$E_k = \frac{1}{2} \left(m \frac{L^2}{4} + J \right) \dot{\theta} \rightarrow \text{The system is the same. and kinetic energy is independent of position, for the same system.}$$

$$V = V_u + V_g$$

$$= \frac{1}{2} h \left(\Delta l_0 + \Delta l_{\alpha}(\theta) \right)^2 + mg h_0(\theta)$$



$$h_0 = y_G = \frac{L}{2} \sin\theta \rightarrow \text{The same}$$

$$\Delta l_{\alpha}(\theta) = x_P(\theta) = L \sin\theta$$

\rightarrow Not the same

$$\frac{\partial h}{\partial q} = \frac{L}{2} \cos\theta$$

$$\hat{J} = m \frac{L^2}{4} + J$$

Before $\Delta l_0 = 0$
 \uparrow now no

$$\frac{\partial \Delta l_{\alpha}}{\partial q} = L \cos\theta$$

$$\hat{J} \ddot{\theta} + kL^2 \sin\theta \cos\theta + h \Delta l_0 L \cos\theta + mg \frac{L}{2} \cos\theta = 0$$

New term which was not present before.

dayrage does not give us a pure equation of motion, due to the presence of Δb , from the initial pre-load.

Since we don't know Θ and Δb , we have find one of the two.

One approach to find Δb is static equilibrium:



$$mg \frac{d}{2} - k \Delta b = 0 \rightarrow \frac{mg}{2k} = \Delta b.$$

This approach doesn't allow us to find the sign without knowing the direction.

With this approach we find compression as positive since we are finding compression as positive whenever we said traction was positive.

This actually changes the result if we keep the positive sign. This means that we go from cancelling out to adding.

The approach we can use is knowing that the derivative of the potential energy at the equilibrium is 0

discharge $\rightarrow E_c \rightarrow$ dynamic forces
 $\hookrightarrow V \rightarrow$ static forces

$$\frac{\partial V}{\partial \theta} = 0$$

$$V(\theta) = kL^2 \cancel{\sin \theta \cos \theta} + k\Delta l_0 L \cos \theta + mg \frac{L}{2} \cos \theta$$
$$= k\Delta l_0 L + mg \frac{L}{2}$$

$$\Rightarrow \Delta l_0 = -\frac{mg}{2k}$$

→ We can evaluate V in the equilibrium position to find Δl_0 .

We find therefore that the general equation of motion is:

$$\boxed{\hat{J}\ddot{\theta} + kL^2 \sin \theta \cos \theta = 0}$$

→ This is non-linear, since we are studying vibrations, we can linearize the equation so we can study small oscillations.

It is possible to write the linear equation directly