

# Modular R.O.V for Sub-Sea Operations

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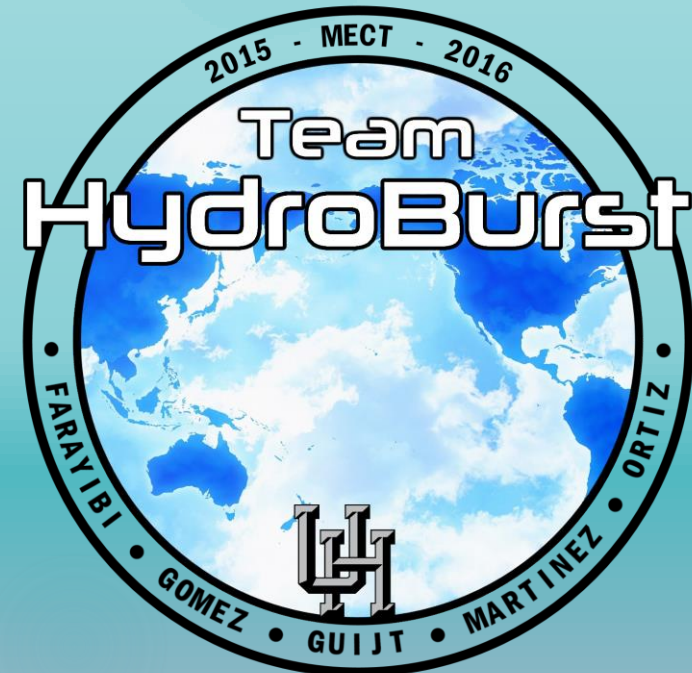
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VICTOR ORTIZ

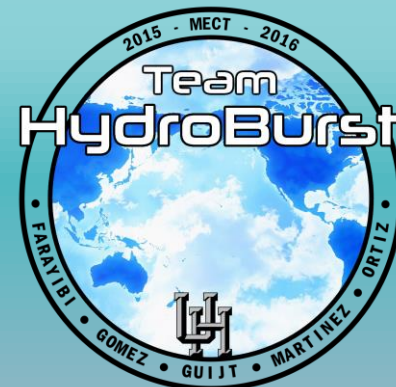
PROJECT UPDATE 1

10/19/2015



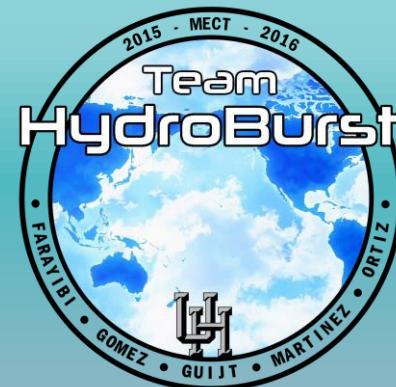
# Safety Moment

<b>LIFTING DO'S &amp; DON'TS</b>			
<b>DO LIFT AS A TEAM</b>  <p>Do lift bulky or heavy loads as a team. Doing so is smart and the safe way to work.</p>	<b>DO TURN WITH LEGS</b>  <p>Do move your legs and feet when turning or lowering the load. Avoid twisting at your waist.</p>	<b>DO USE YOUR LEGS</b>  <p>Do lift the load using your powerful leg and buttocks muscles. Your feet should be wide apart, head and back upright. Keep abdominal muscles tight and the load in close.</p>	<b>DO USE EQUIPMENT</b>  <p>Do use equipment like hand trucks, dolly's, or forklifts to do the heavy lifting. It's much less work and less risk of injury.</p>
<b>DON'T LIFT BULKY LOADS ALONE</b>  <p>Don't lift bulky or heavy loads alone. Doing so puts great stress on your low back muscles and spine.</p>	<b>DON'T TWIST WHEN LIFTING</b>  <p>Don't twist when lifting, lowering, or carrying any load as this increases your risk of back injury.</p>	<b>DON'T USE YOUR BACK</b>  <p>Don't lift the load with your rear end high and your head low. Use your leg muscles, not your weaker low back muscles.</p>	<b>DON'T LIFT HEAVY LOADS</b>  <p>Don't lift heavy loads when you can use equipment. It is less work and less stress on your low back.</p>



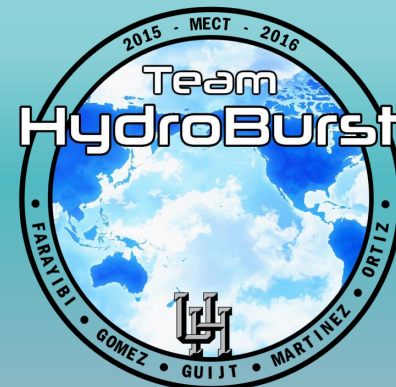
# ROV Presentation Outline

- ▶ Project Objectives / Timeline
- ▶ 2016 MATE Competition
- ▶ MATE Competition Specs
- ▶ Design Overview
- ▶ First Frame Design
- ▶ Second Frame Design
- ▶ Stress Analysis – Frame 1
- ▶ Stress Analysis – Frame 2
- ▶ Control System
- ▶ Propulsion System
- ▶ Thruster Positioning
- ▶ Arm Update
- ▶ Updated Project Budget



# Team Objective

- ▶ Team HydroBurst's primary objective is to design, test, and construct an underwater remotely operated vehicle (ROV) to compete in the 2016 National and International MATE Competitions.







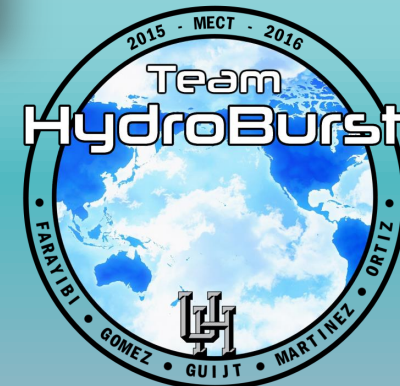
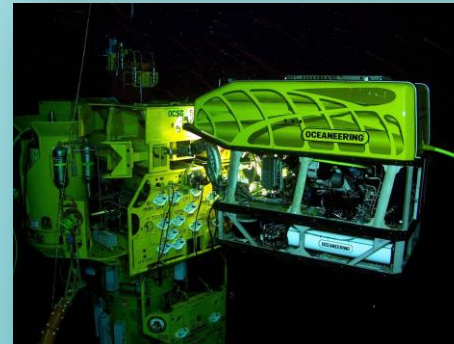
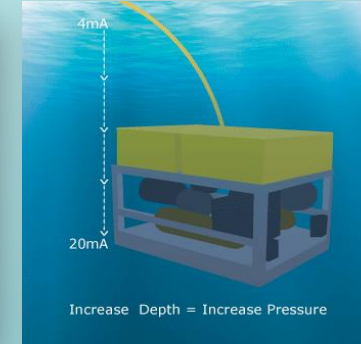
# 2016 MATE Competition

UNIVERSITY of HOUSTON

COLLEGE of TECHNOLOGY

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- ▶ Outer Space: Mission Europa
  - ▶ Measure thickness of ice
  - ▶ Measure total depth
  - ▶ Measure temp of thermal vents
  - ▶ Retrieve and connect ESP cable

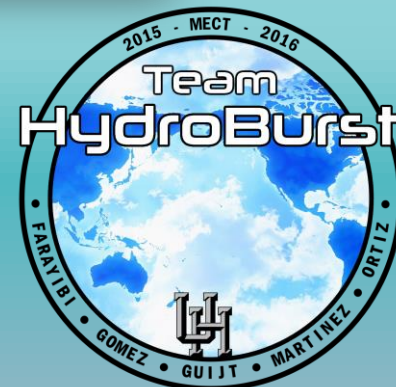
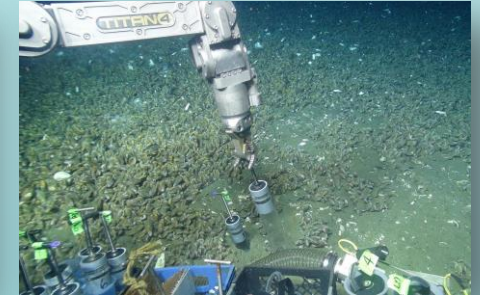
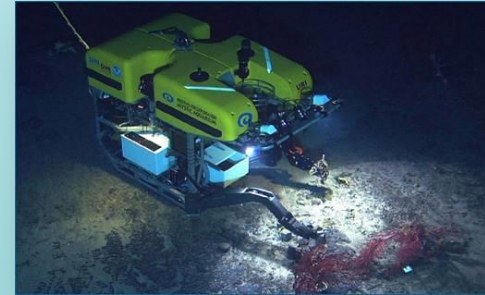




MATE

# 2016 MATE Competition

- ▶ Inner Space: Mission-critical equipment recovery
  - ▶ Survey the seafloor
  - ▶ Collect equipment located on sea floor
- ▶ Inner Space: Forensic Fingerprinting
  - ▶ Collect oil sample
  - ▶ Analyze a gas chromatograph

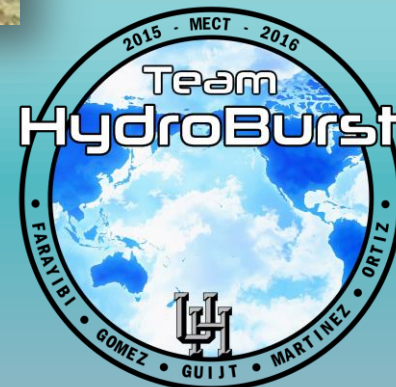




MATE

# 2016 MATE Competition

- ▶ Inner Space: Deepwater Coral Study
  - ▶ Photograph corals and compare with last years images
  - ▶ Collect coral samples for analysis
- ▶ Inner Space: Rigs to reefs
  - ▶ Attach a flange to top of wellhead
  - ▶ Secure flange with two bolts
  - ▶ Install cap on flange
  - ▶ Secure the cap with four bolts

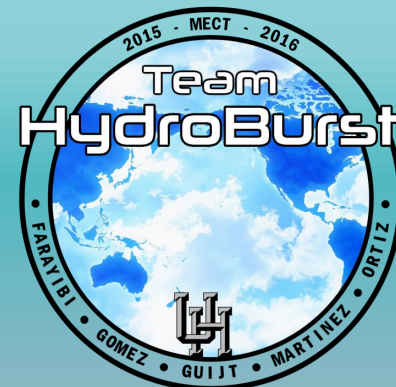






# MATE Competition Specs

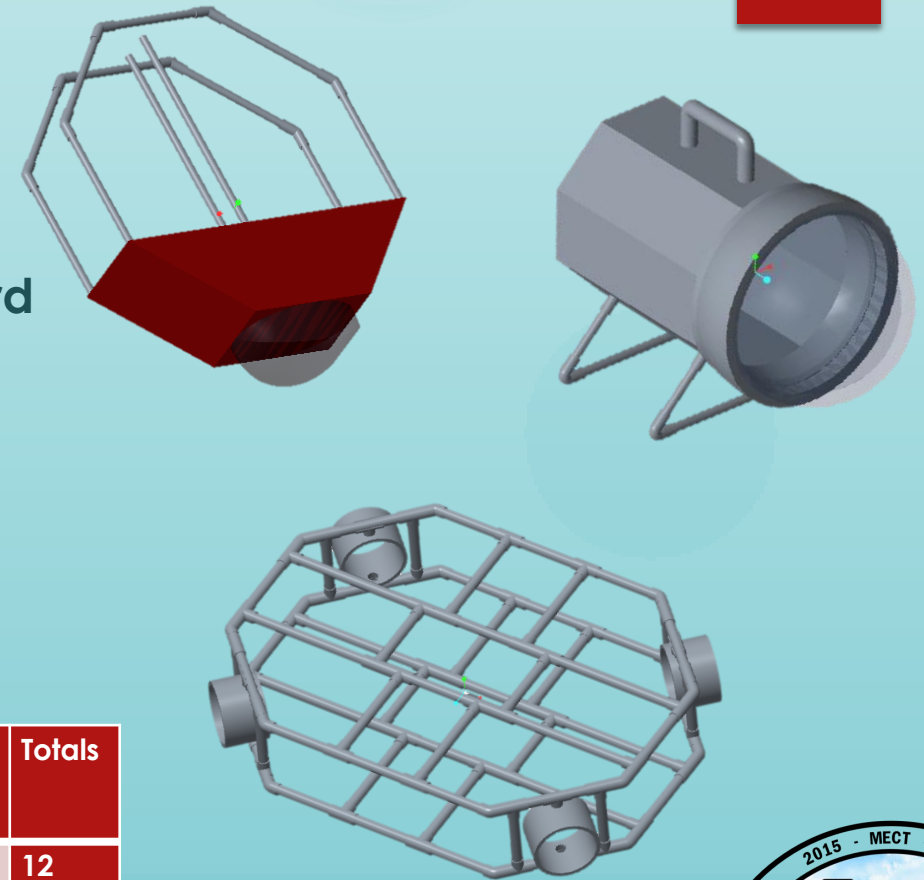
- ▶ **Size Constraints (released September 30<sup>th</sup>):**
  - ▶ **Maximum Diameter of 85cm**
    - ▶ +5 Points for diameter between 64.1cm and 70cm
    - ▶ +10 points for diameter between 58.1cm and 64cm
    - ▶ +20 points for diameter less than 58cm
  - ▶ **Mass**
    - ▶ +5 Points for mass between 19.01kg and 22kg
    - ▶ +10 points for mass between 17.01kg and 19kg
    - ▶ +20 points for mass less than 17kg
- ▶ **Our Design Goals based on Constraints:**
  - ▶ Diameter less than 58cm (22.8346 inches).
  - ▶ Mass less than 17kg (37.4786 lbf)



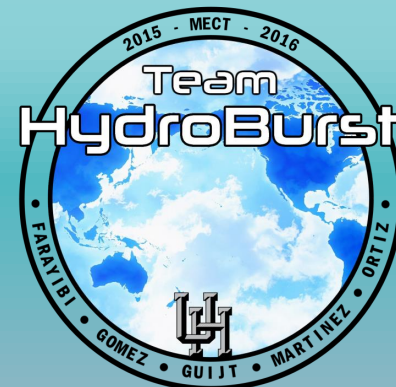


# Design Overview

- ▶ As covered during the last presentation:
  - ▶ we weighed the pros and cons of various standard ROV frame designs.
  - ▶ Decided upon the Octagonal model as the most effective model.

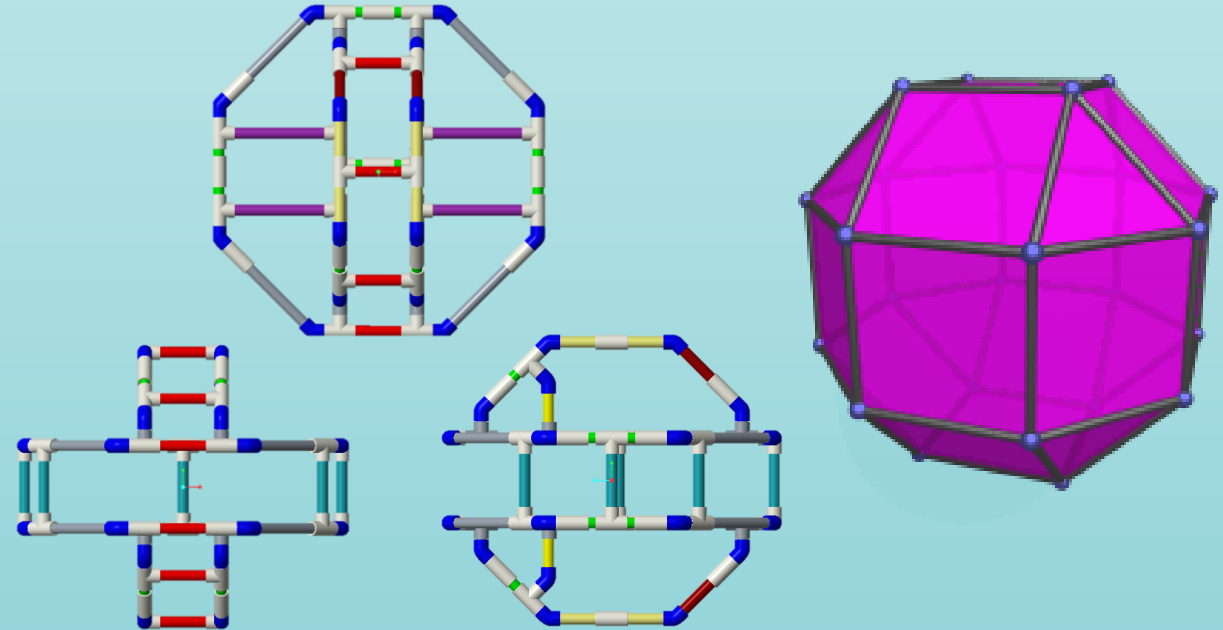


	Maneuverability	Hydro-dynamics	Internal Space Utilization	Modular Optimization	Corrosion Resistance	Totals
Rectangular	1	1	2	3	5	12
Cylindrical	3	4	3	1	5	16
Octagonal	4	3	2	3	5	17
Flat	2	3	3	2	5	16

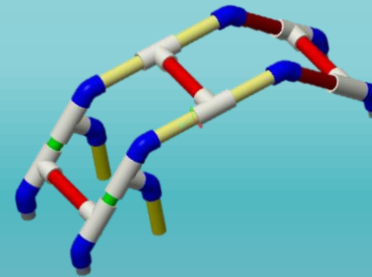


# First Frame Design

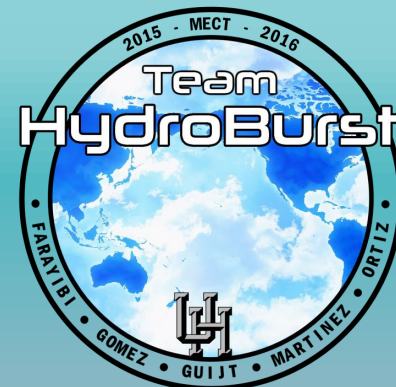
- ▶ Shape: Rhombicuboctahedron
  - ▶ To make it as spherical as possible using existing fittings.
- ▶ Material: CPVC
  - ▶ Light weight, designed for water usage, and can handle high pressures.
- ▶ Maximum diameter: 21 inches
  - ▶ Minimum of 1.00 inch clearance on all sides.
- ▶ Frame weight: 4.5 lbs.



Complete Frame Assembly

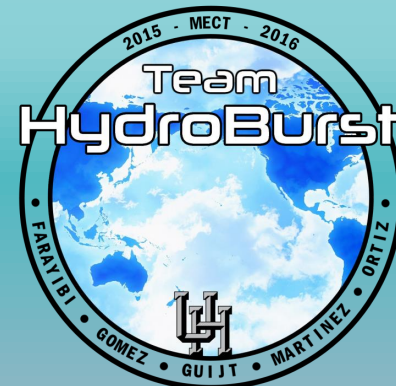
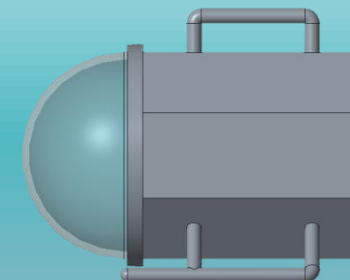
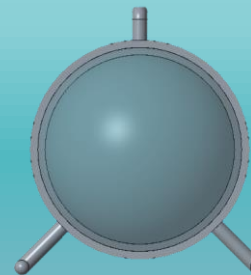
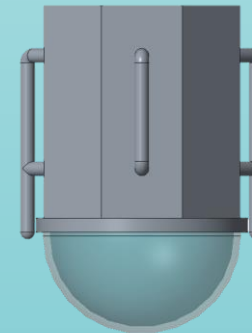
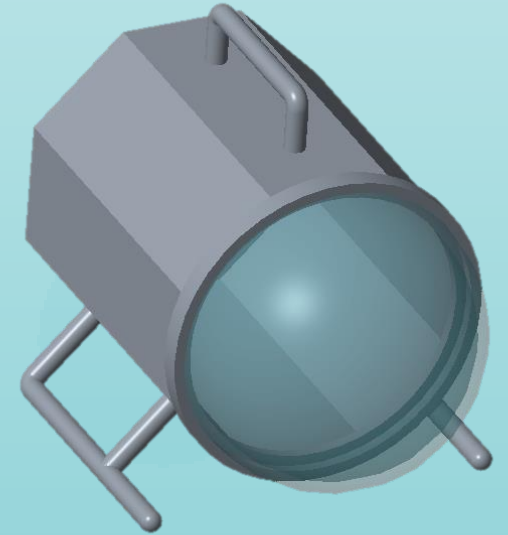
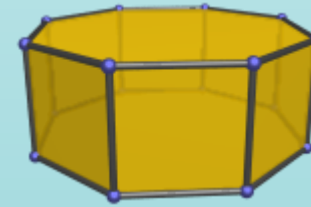


Module (Arms, Sensors, etc...)



# Second Frame Design

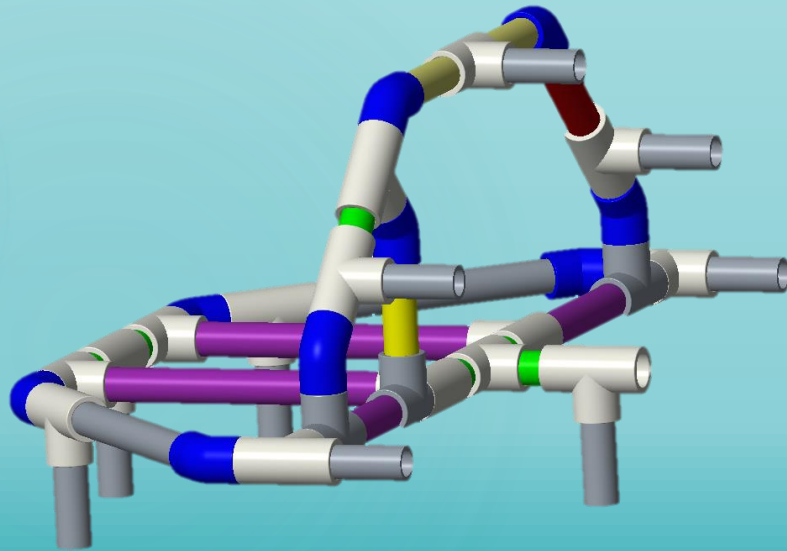
- ▶ Shape: Octagonal Prism.
  - ▶ Solid Frame.
- ▶ Material: Plastic or Acrylic.
  - ▶ Light weight and strong.
- ▶ Maximum diameter: 21 inches.
  - ▶ Minimum of 1.00 inch clearance on all sides.
- ▶ Clear Dome on the front for access and Camera.



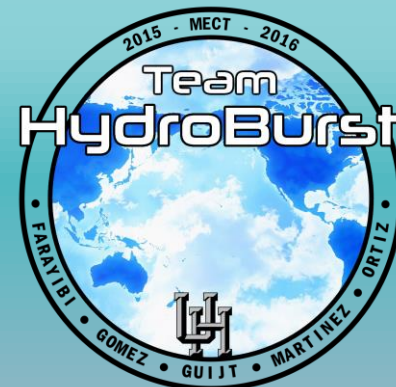
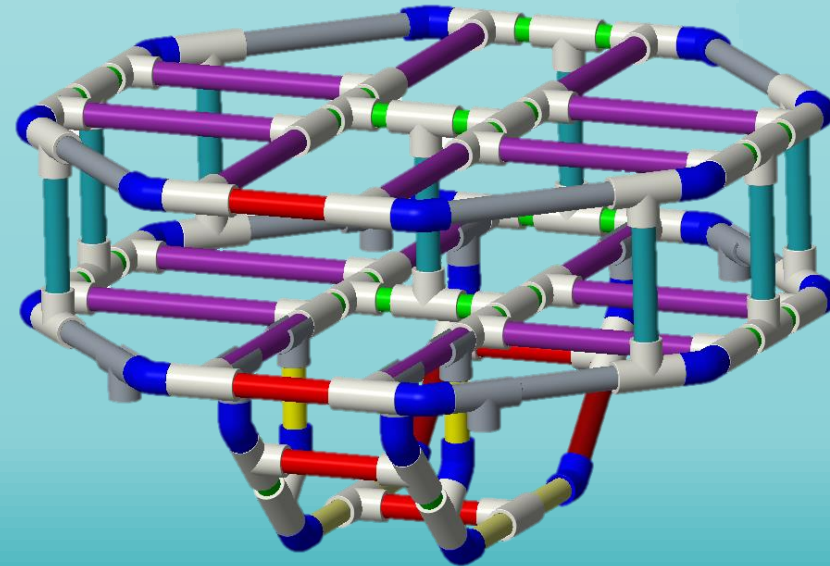


# Stress Analysis – Frame 1

Quarter Frame

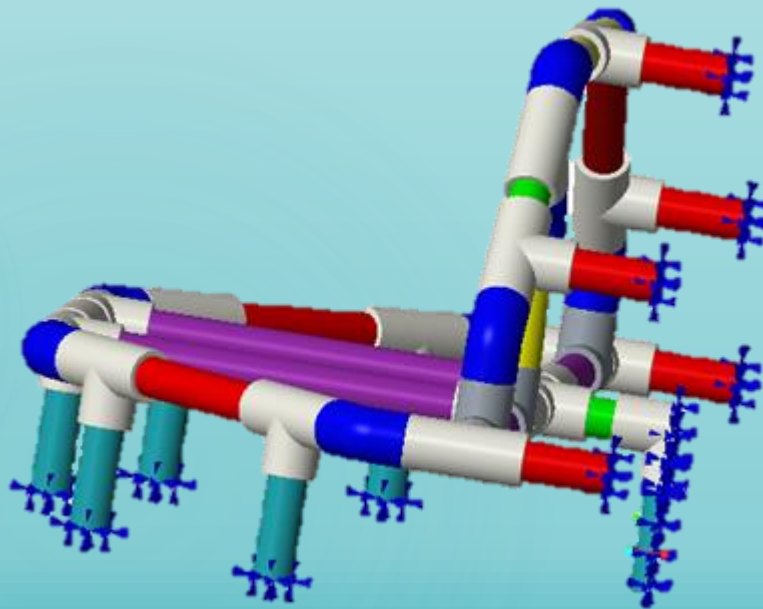


Full Frame



# Stress Analysis – Frame 1

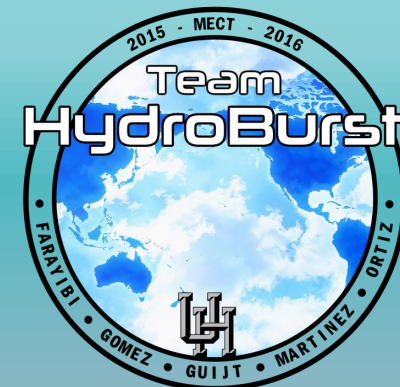
## Constraints



## CPVC Material Definition

Density	0.0559973	lbm/in^3
<b>Structural</b> Thermal Miscellaneous Appearance User Defined		
Symmetry	Isotropic	
Stress-Strain Response	Linear	
Poisson's Ratio	0.3	
Young's Modulus	370000	psi
Coeff. of Thermal Expansion	3.9e-05	/K
Mechanisms Damping		sec/in
<b>Material Limits</b>		
Tensile Yield Stress	7600	psi
Tensile Ultimate Stress	12500	psi
Compressive Ultimate Stress		lbm/(in sec^2)

<b>Structural</b> Thermal Miscellaneous Appearance User Defined		
Symmetry	Isotropic	
<b>Properties</b>		
Specific Heat Capacity	775002	in^2/(sec^2 F)
Thermal Conductivity	7.71521	in lbm/(sec^3 F)



# Stress Analysis – Frame 1

## Pressure

Pressure calculations

$$P = \rho \cdot g \cdot h$$

$P$  = Pressure  
 $\rho$  = Density  
 $g$  = Gravity  
 $h$  = Underwater distance

1. 40 feet underwater	2. 100 feet underwater
$h := 40 \text{ ft}$	$h_2 := 100 \text{ ft}$
$P := \rho_w \cdot g \cdot h = 17.352 \text{ psi}$	$P_2 := \rho_w \cdot g \cdot h_2 = 43.381 \text{ psi}$

## CPVC Material Assignment

**Pressure Load**

Name: Pressure\_40ft

Member of Set: LoadSet1

References:

Surfaces: ☒ Individual ☐ Boundary ☐ Intent

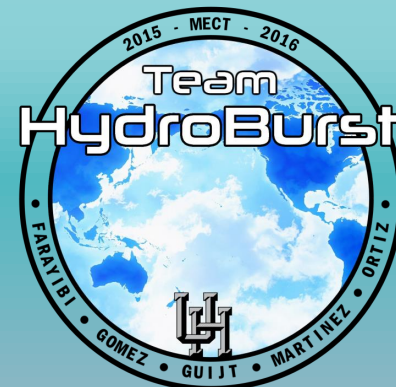
Surface : HALF-INCH-PIPE\_3-65-INCH.PRT  
 Surface : HALF-INCH-PIPE\_3-65-INCH.PRT  
 Surface : HALF-INCH-COUPLING-T.PRT  
 Surface : HALF-INCH-COUPLING-T.PRT

Surface Sets...

Pressure: Advanced >>

Value: 17.352 psi

Preview OK Cancel

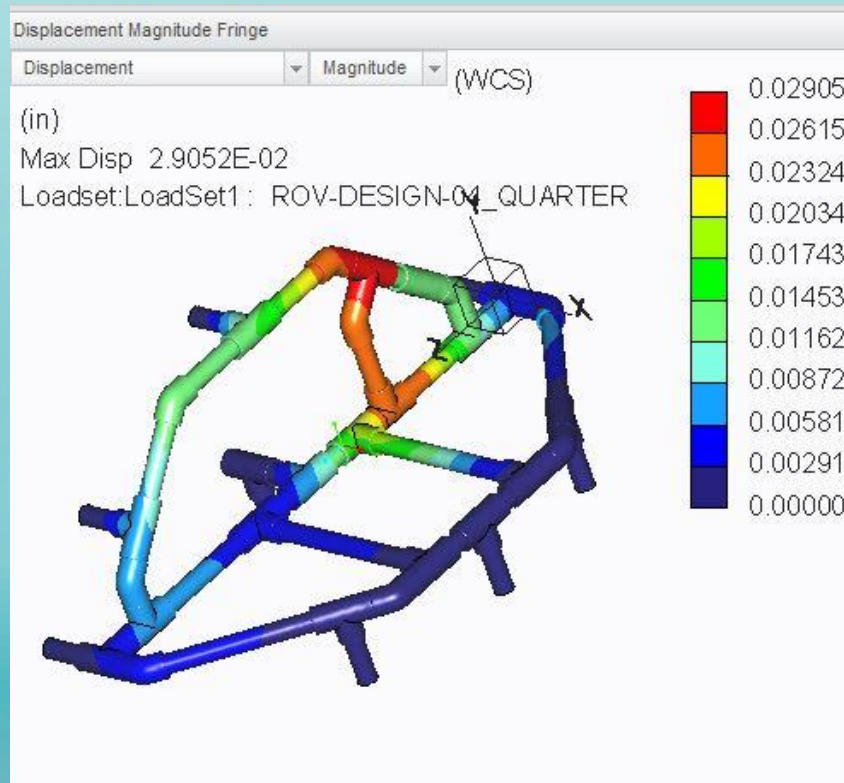




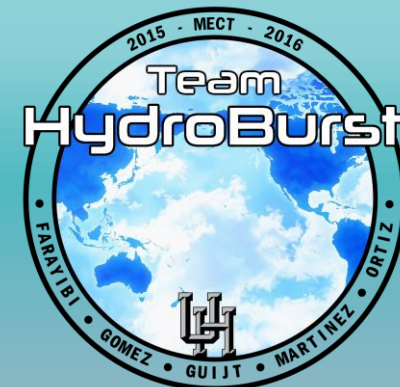
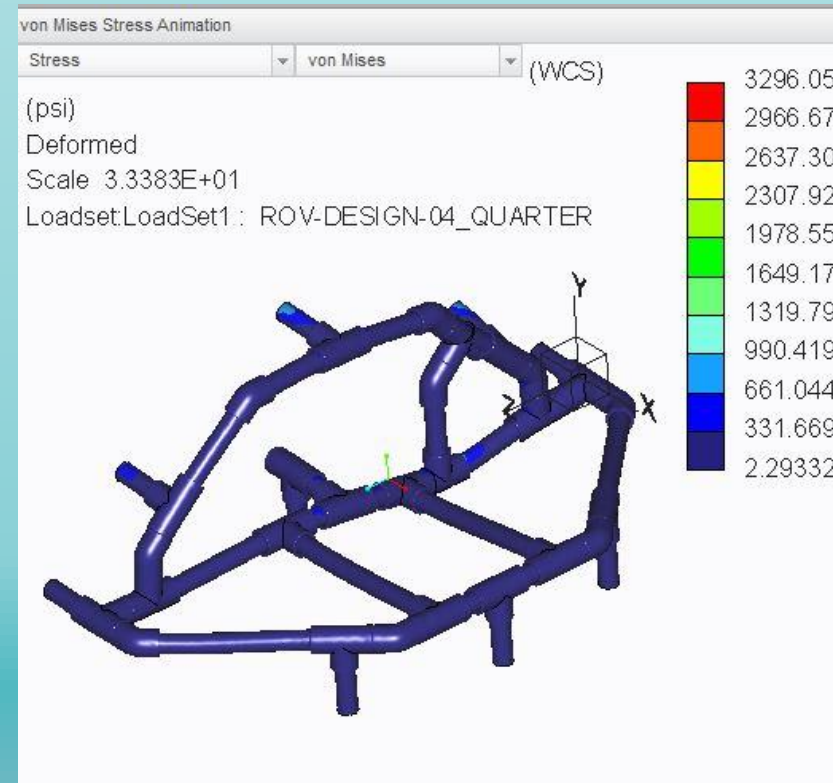
# Stress Analysis – Frame 1

## 40 ft. below water

### Displacement (in)



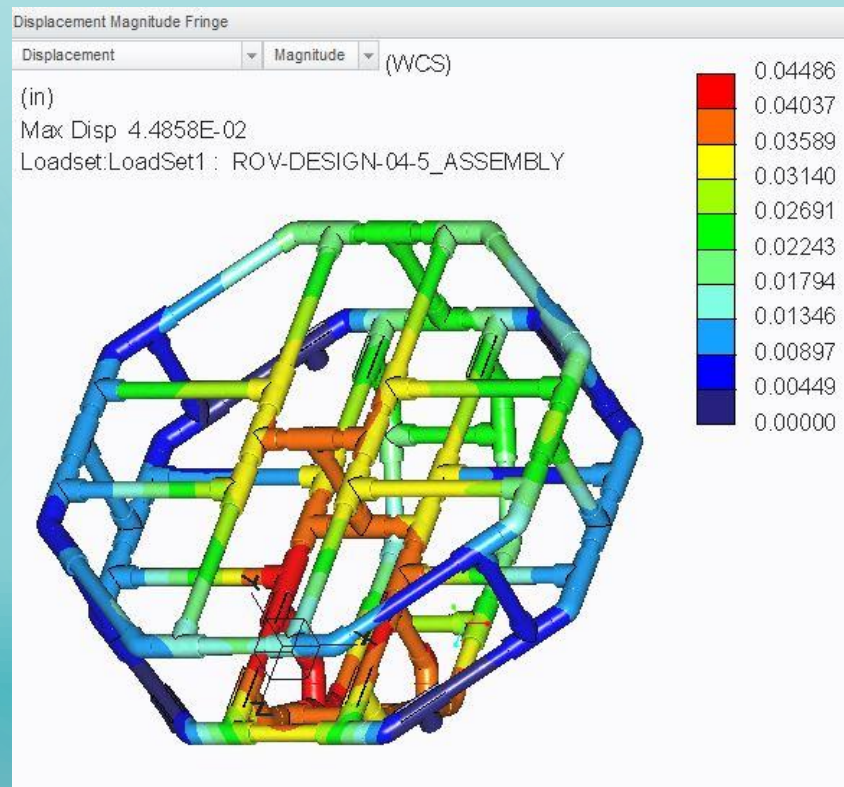
### Von Mises Stress (psi)



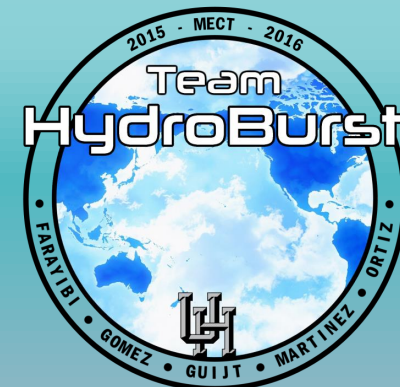
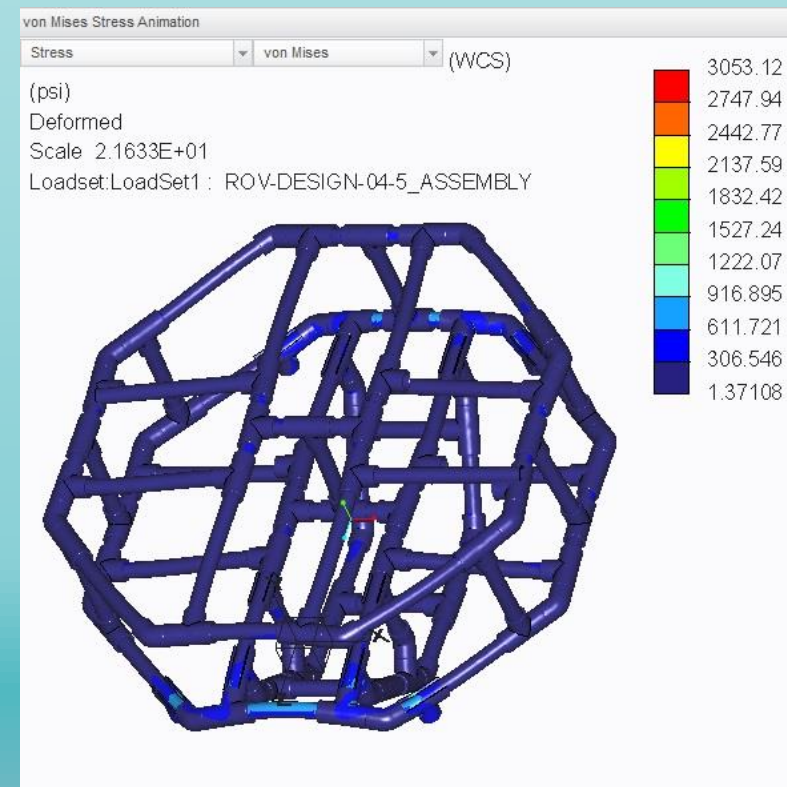
# Stress Analysis – Frame 1

## 40 ft. below water

### Displacement (in)



### Von Mises Stress (psi)

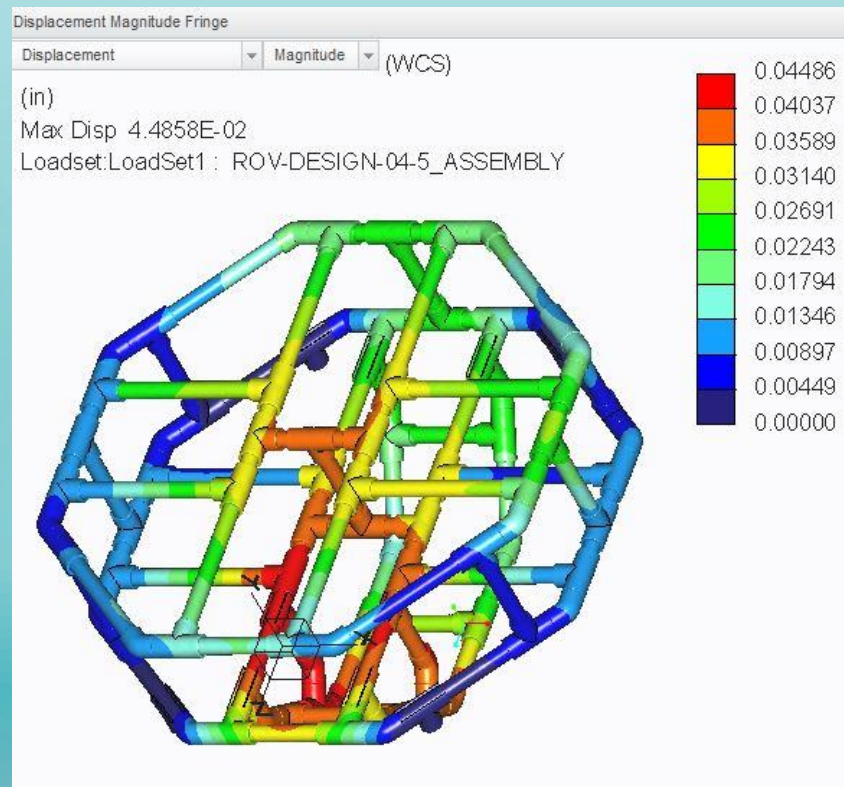




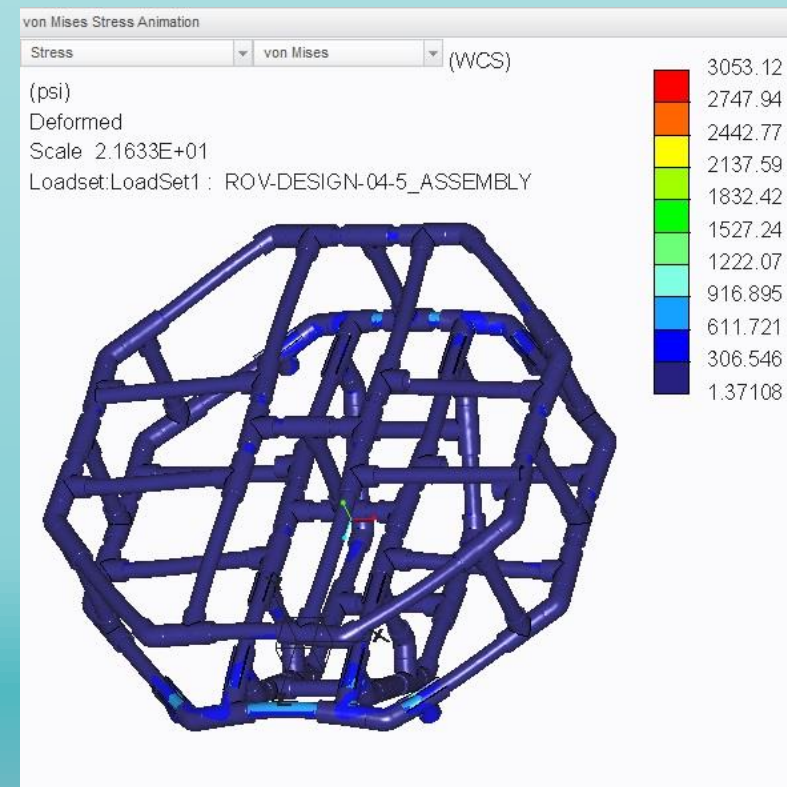
# Stress Analysis – Frame 1

## 100 ft. below water

### Displacement (in)



### Von Mises Stress (psi)



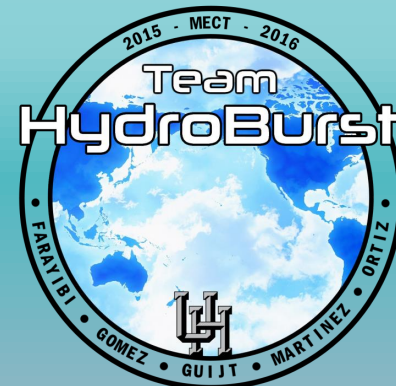


# Stress Analysis – Frame 1

Model	Max. Von Mises Stress		Max. Displacement	
	40 ft. depth	100 ft. depth	40 ft. depth	100 ft. depth
Quarter Frame	3296 psi	9914 psi	0.029 inches	0.0734 inches
Full Frame	3053 psi	7633 psi	0.045 inches	0.112 inches

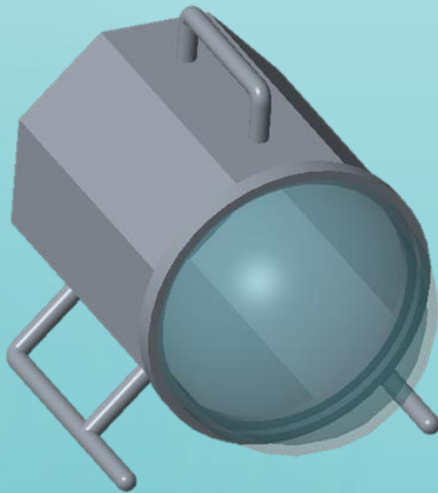
► CPVC Material Properties:

- Tensile Strength (Yield Strength) 7,600 psi
- Allowable Stress =  $\frac{2}{3} \sigma_y$  5066.67 psi



# Stress Analysis – Frame 2

## ROV Frame



## Acrylic Material Definition

Density 0.68 g/cm<sup>3</sup>

Structural Thermal Miscellaneous Appearance User Defined

Symmetry Isotropic

Stress-Strain Response Linear

Poisson's Ratio 0.3

Young's Modulus 425000 psi

Coeff. of Thermal Expansion 3.9e-05 /F

Mechanisms Damping sec/in

Material Limits

Tensile Yield Stress 7500 psi

Tensile Ultimate Stress 12100 psi

Compressive Ultimate Stress lbm/(in sec<sup>2</sup>)

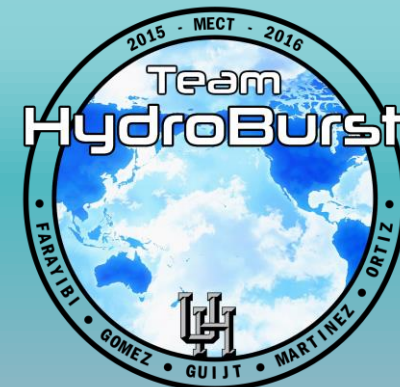
Structural Thermal Miscellaneous Appearance User Defined

Symmetry Isotropic

Properties

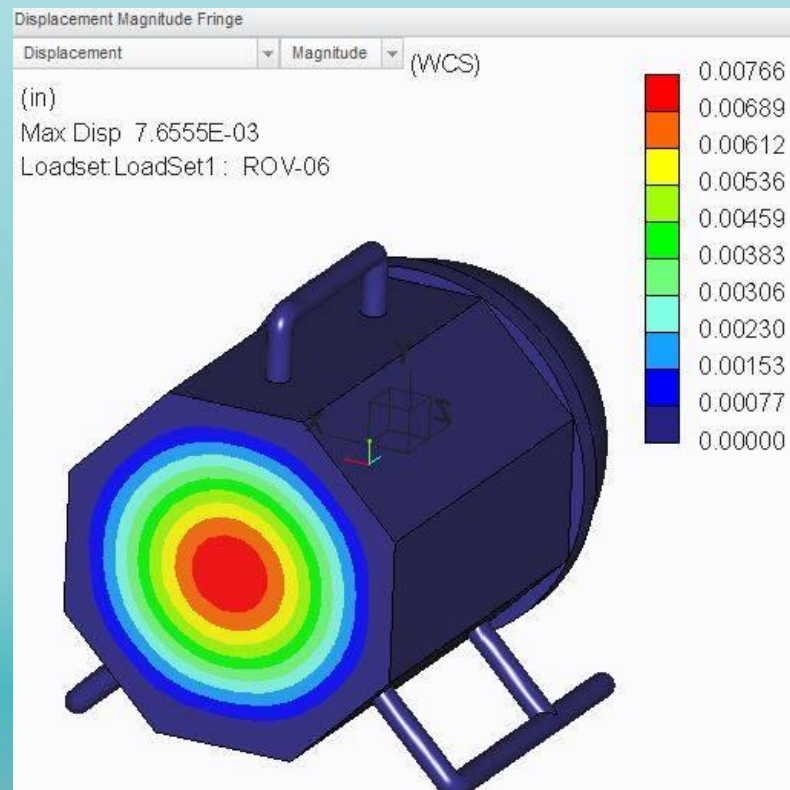
Specific Heat Capacity 1779.39 joule/(kg K)

Thermal Conductivity 0.202 W/(m K)

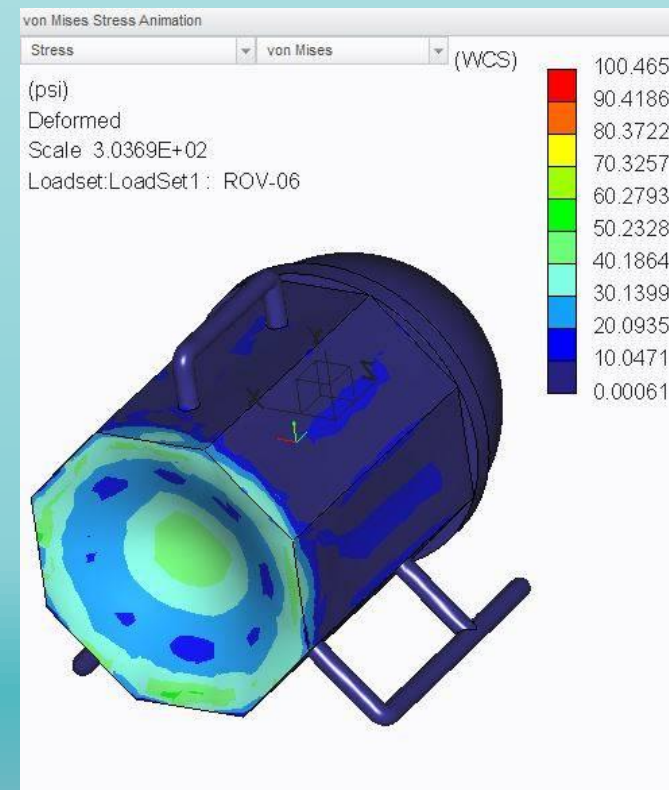


# Stress Analysis – Frame 2

## Displacement (in)



## Von Mises Stress (psi)

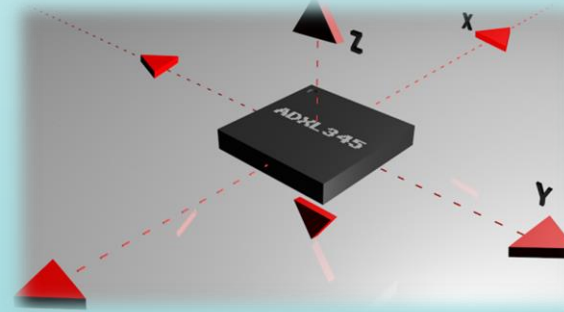




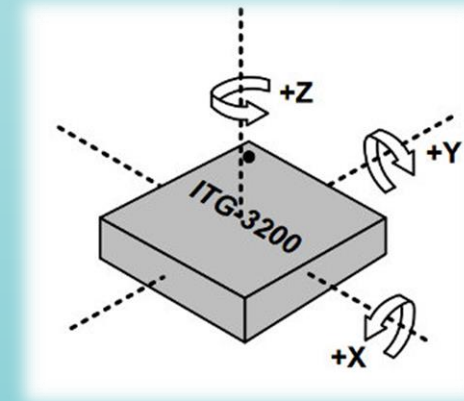
# Control System

## ► Feedback Sensors:

► Accelerometer.



► Gyroscope.

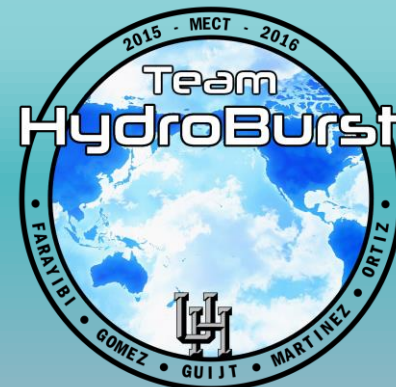
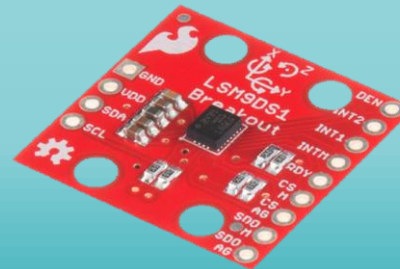


## ► 3-in-1 Sensor: 9 Degrees of Freedom IMU (\$24.95)

► 3-axis accelerometer

► 3-axis gyroscope

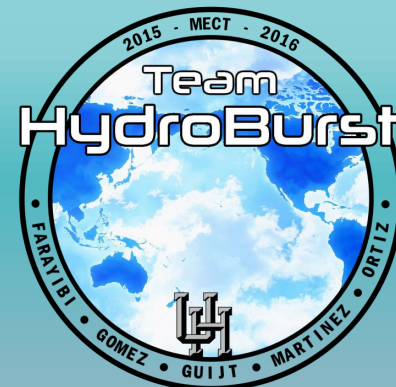
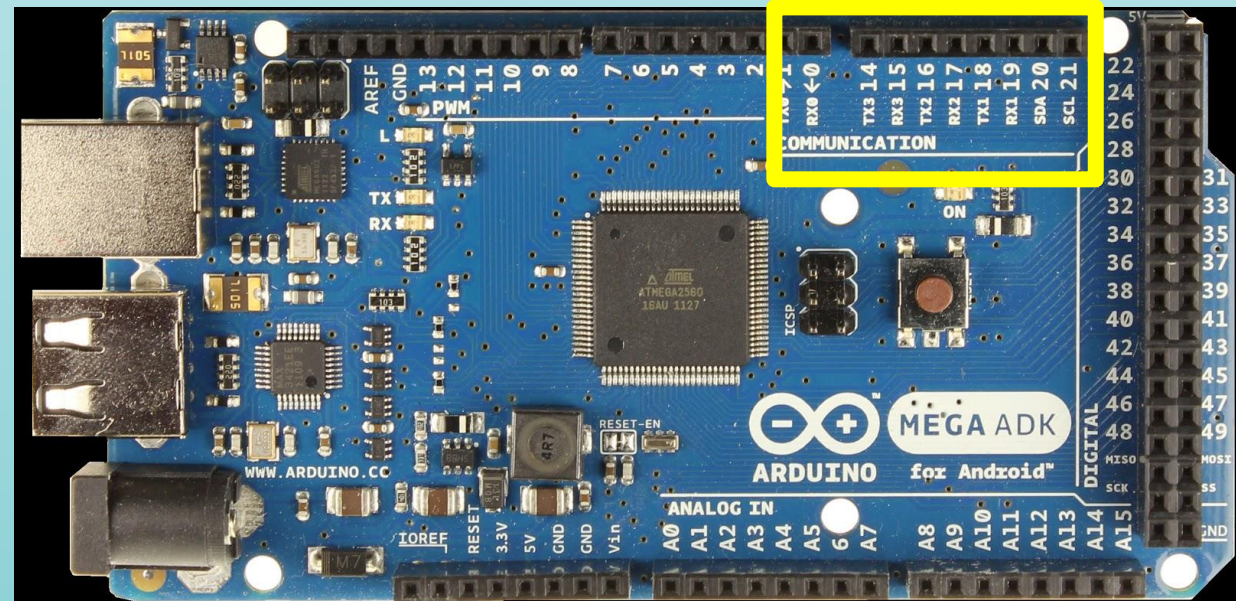
► 3-axis magnetometer



# Control System

## ► Communications:

- UART.
- I2C
- USB Hub



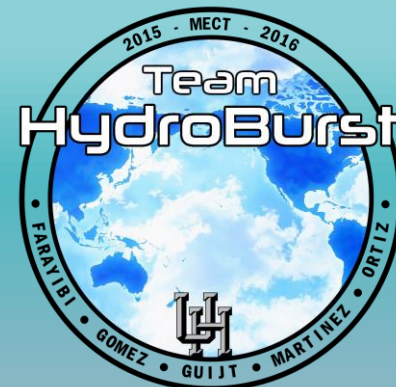
# Propulsion System

## ► Blue Robotics T100 Thruster

- Price- \$134.00 ea.
- Motor Type- High efficiency brushless
- Weight in air- 0.93 lb.
- Max Power- 130W
- Operating Voltage- 12 volts
- Max Thrust 5.2:4 Forward (5.2 lbf) Reverse (4 lbf)
- Min Thrust (0.03 lbf)

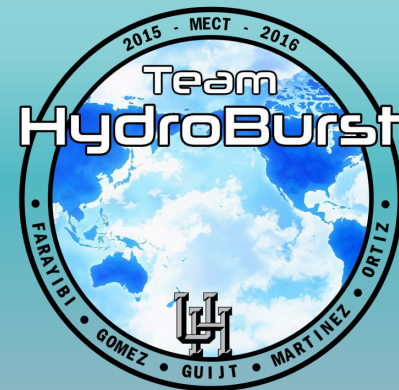
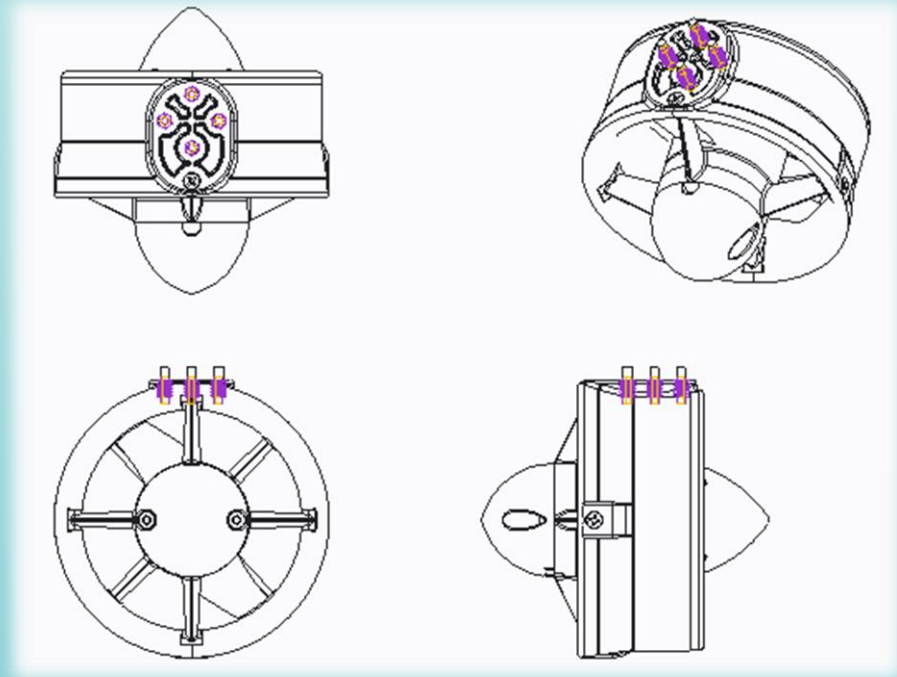
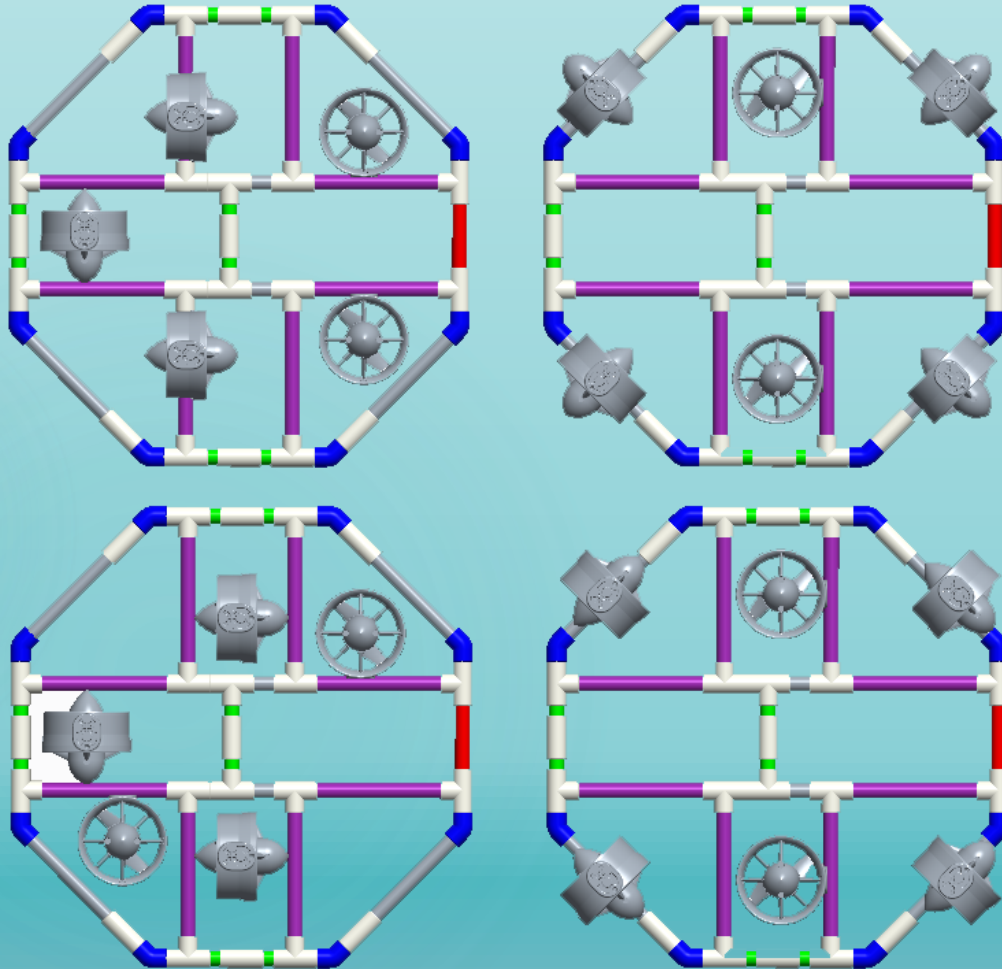


 **BlueRobotics**  
T100 Thruster





# Thruster Positioning





# Arms Update

## Servo Motors vs. Stepper Motors

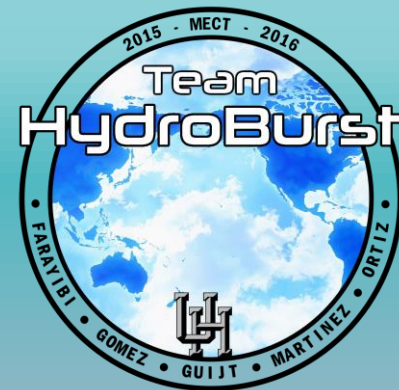
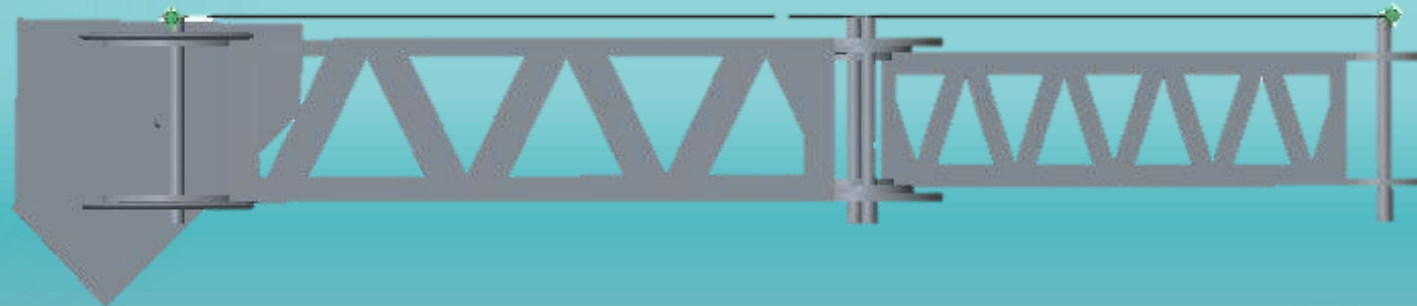
### Servo Motor

- ▶ Fast
- ▶ High Torque
- ▶ Accurate Rotation With Limited Angle ( $\approx 180^\circ$ )
- ▶ More complicated setup with PWM turning

### Stepper Motor

- ▶ Slow
- ▶ Precise Rotation
- ▶ Requires External Control Circuit or Microcontroller
- ▶ Easy Set-Up & Control

# Arms Update



# Updated Project Budget

## ► Current Cash Flow:

### ► Expenditures:

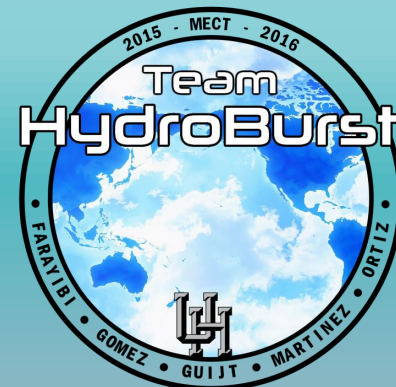
- CPVC components (\$86.35)

### ► Income:

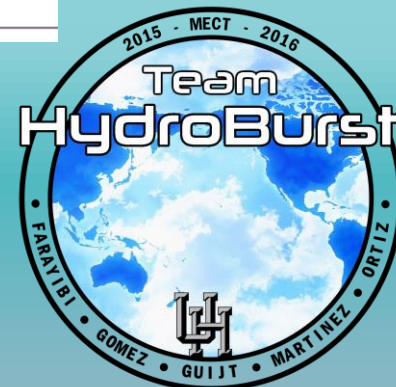
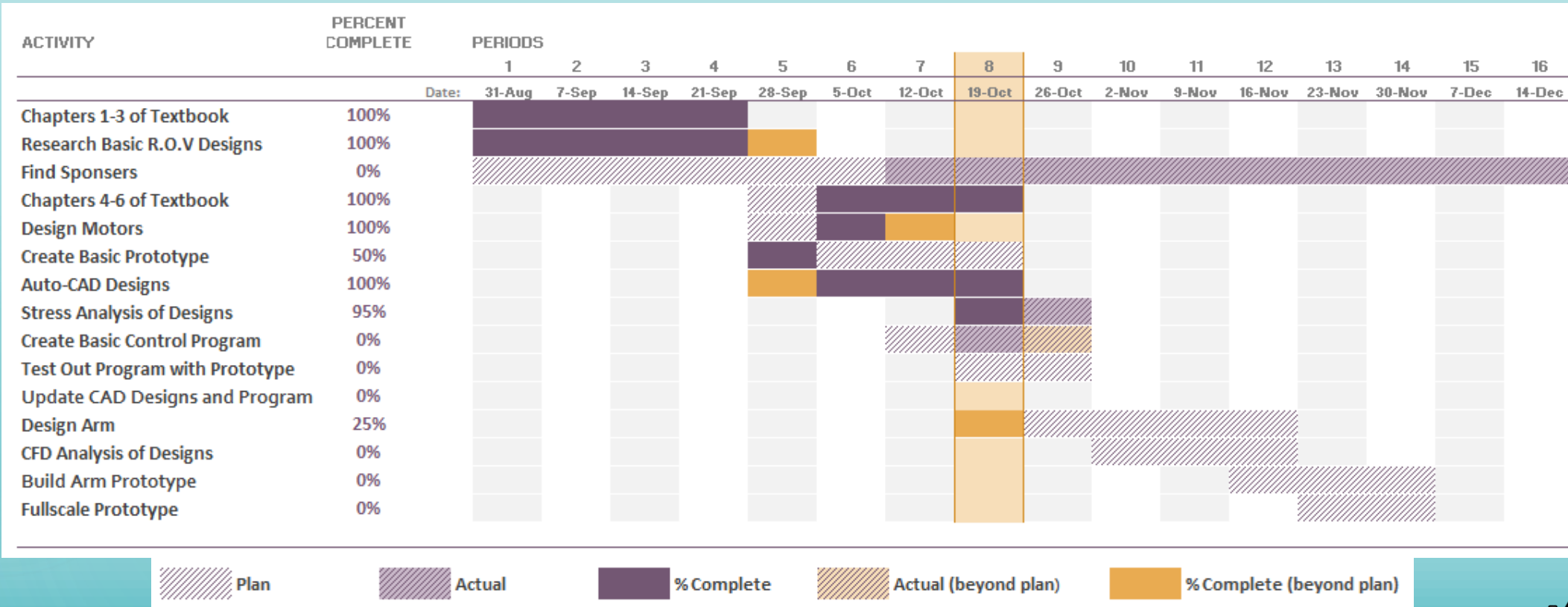
- No Sponsors as of 10/19

## ► Planned Expenditures:

► Final Frame	-	\$100
► Propulsion System	-	\$800
► Control System	-	\$200
► Arm Module	-	\$700
► Tether System	-	\$200
► Total	-	\$2,000



# Updated Gantt Chart





# Thank You for Listening!

## Any Questions, Comments, or Concerns?

