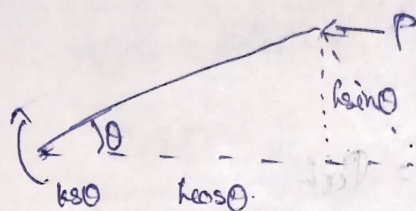
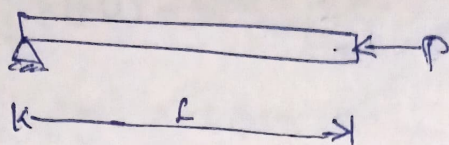


Tutorial-1

2. a)



$$\sum M_A = 0$$

$$-k\theta + P L \sin \theta = 0$$

$$P = \frac{k\theta}{L \sin \theta}$$

For small deflection

$$P_{cr} = \frac{k\theta}{L\theta} = \underline{\underline{\frac{k}{L}}}$$

$$P = \frac{k\theta}{L \sin \theta}$$

Energy Approach

$$\Pi = U + V$$

$$U = \frac{1}{2} (k\theta) \theta = \frac{1}{2} k\theta^2$$

$$V = PL(1 - \cos \theta)$$

$$\Pi = \frac{1}{2} k\theta^2 - PL(1 - \cos \theta)$$

For eqn TPE

$$\frac{d\Pi}{d\theta} = 0$$

$$k\theta - PL \sin \theta = 0$$

$$P_{cr} = \frac{k\theta}{L \sin \theta}$$

To get Post buckling nature of Equilibrium -

$$\frac{d^2 \Pi}{d\theta^2} = k - Pl \cos \theta$$

In small deflection

$$\frac{d^2 \Pi}{d\theta^2} = k - Pl$$

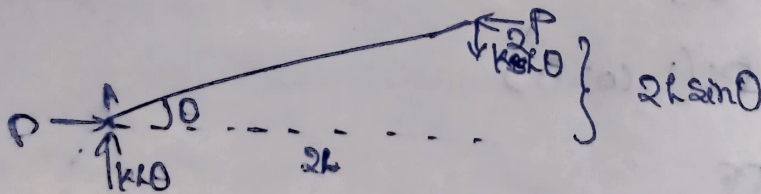
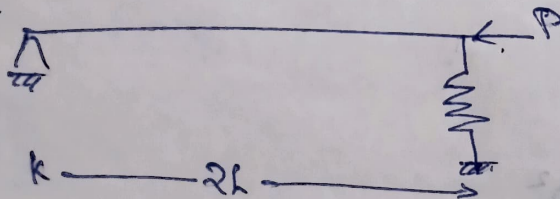
w.k.t $P_{cr} = k/l$ (or) $k = P_{cr} l$

If $P < P_{cr} \Rightarrow \frac{d^2 \Pi}{d\theta^2} \Rightarrow (+) \rightarrow$ stable.

$P > P_{cr} \Rightarrow \frac{d^2 \Pi}{d\theta^2} \Rightarrow (-) \rightarrow$ unstable.

$P = P_{cr} \Rightarrow \frac{d^2 \Pi}{d\theta^2} = 0 \rightarrow$ No conclusion.

b)



$\sum M_A = 0$

$$-k\theta \times 2L + P \times 2L \sin \theta = 0$$

$$P \times 2L \sin \theta = 2kL\theta$$

$$P = 2k\theta$$

$$P \sin \theta = k\theta$$

$$P = 2k\theta$$

$P_{cr} = 2KL$ — bifurcation approach

$$\Pi = \frac{1}{2} k(\Delta\theta)^2 - 2PL(1 - \cos\theta)$$

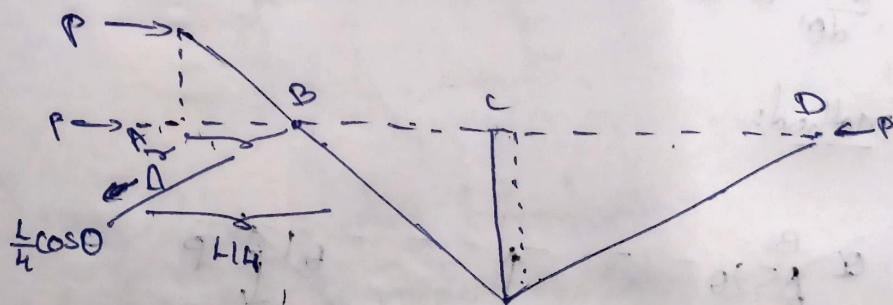
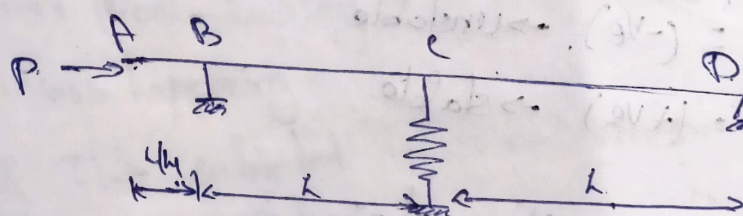
$$\frac{d\Pi}{d\theta} = 4k\Delta\theta - 2PL\sin\theta = 0$$

$$4k\Delta\theta = 2PL\sin\theta$$

$$2k\Delta\theta = PL$$

$$P_{cr} = 2KL$$

If $P < P_{cr}$, $\frac{d^2\Pi}{d\theta^2} \rightarrow (+ve)$ stable
 If $P > P_{cr}$, $\frac{d^2\Pi}{d\theta^2} \rightarrow (-ve)$ unstable
 If $P = P_{cr}$, $\frac{d^2\Pi}{d\theta^2} = 0 \rightarrow$ No conclusion



$$AB = \frac{PL}{4}(1 - \cos\theta)$$

$$BC = PL(1 - \cos\theta)$$

$$CD = PL(1 - \cos\theta)$$

$$U = \frac{1}{2} k(\Delta\theta)^2$$

$$V = -\frac{PL}{4}(1 - \cos\theta) - PL(1 - \cos\theta)$$

$$\Pi = \frac{1}{2} k(\Delta\theta)^2 - \frac{PL}{4}(1 - \cos\theta) - 2PL(1 - \cos\theta)$$

$$\frac{d\Pi}{d\theta} = k\Delta\theta - \frac{PL}{4}\sin\theta - 2PL\sin\theta = 0$$

For small deflection.

$$k\Delta\theta = \frac{PL}{4} + 2PL$$

$$KL = \frac{qP}{H}$$

$$\frac{HPL}{q} = P$$

$$\boxed{P_{cr} = \frac{4KL}{q}}$$

$$\frac{d^2 \Pi}{d\theta^2} = KL^2 - \frac{PL}{H} \cos \theta - 2PL \cos \theta$$

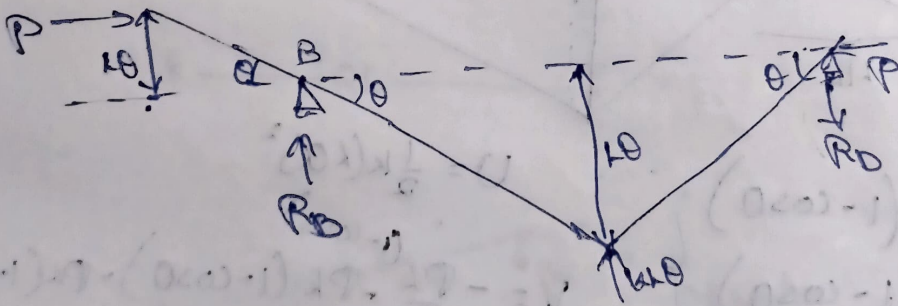
$$\frac{d^2 \Pi}{d\theta^2} = KL^2 - \frac{4LP}{q}$$

$$P > P_{cr} \quad \frac{d^2 \Pi}{d\theta^2} = (-ve) \rightarrow \text{unstable}$$

$$P < P_{cr} \quad \frac{d^2 \Pi}{d\theta^2} = (+ve) \rightarrow \text{stable}$$

$$P = P_{cr} \quad \frac{d^2 \Pi}{d\theta^2} = 0 \rightarrow \text{No conclusion}$$

Bifurcation Method:



$$\Sigma M_B = 0$$

$$P\left(\frac{L\theta}{H}\right) - (KL\theta)L + R_D(2L) = 0$$

$$\frac{P\theta}{4} - KL\theta = -2R_D$$

$$\frac{KL\theta}{2} - \frac{P\theta}{8} = R_D$$

$$R_D = \frac{4KL\theta - P\theta}{8}$$

$$\sum M_c = 0$$

$$R_{DL} - P_{DL} = 0$$

$$R_D = P_D$$

$$4KL D - P_D = 8P_D$$

$$4KL D = 9P_D$$

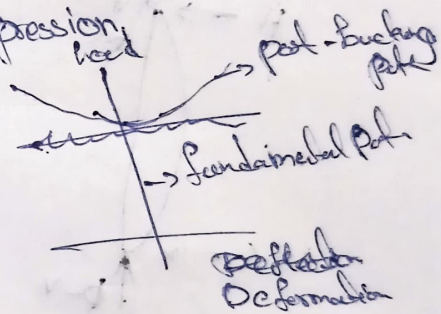
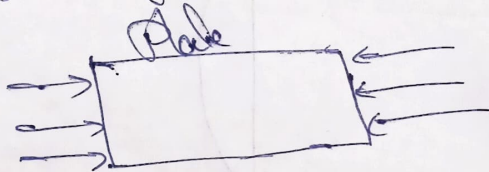
$$\boxed{P = \frac{4KL}{9}}$$

1.) Types of Bifurcation instability:-

a) Stable symmetric bifurcation:-

Load required to maintain equilibrium increases after post-buckling (after P_{cr}) implies there is a hardening process happening.

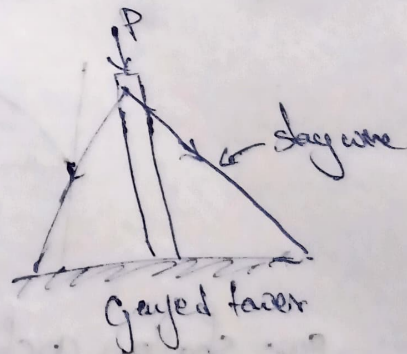
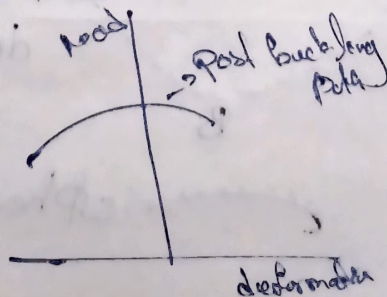
Eg:- Plate subjected to uniaxial compression



b) Unstable symmetric bifurcation:-

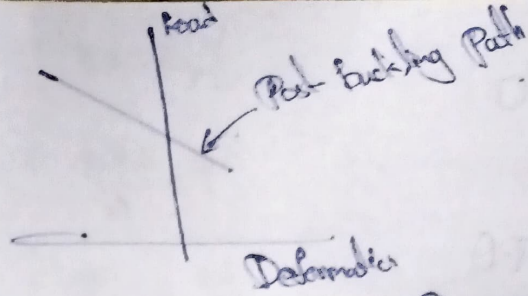
* Load required to maintain equilibrium reduces post-buckling.
* Implies there is a softening process.

Eg: guyed tower.

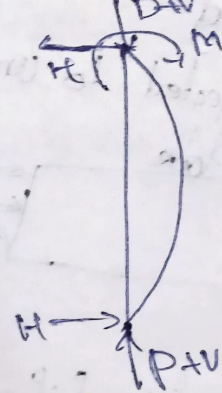
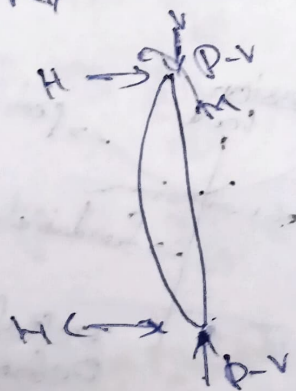
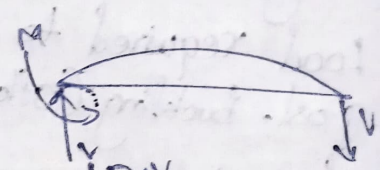
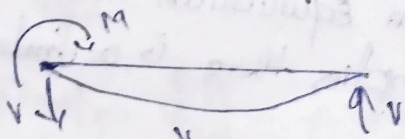
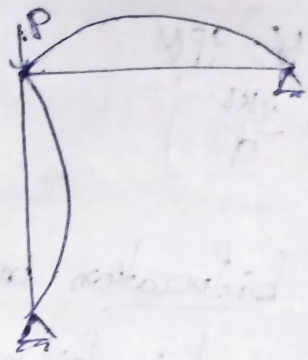
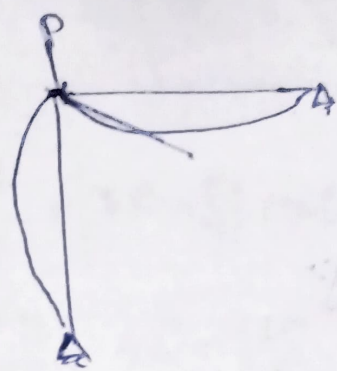


c) Asymmetric bifurcation:-

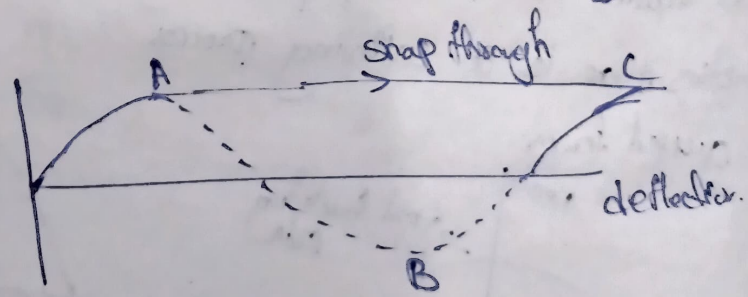
Buckling may increase (or) decrease the load required to maintain equilibrium post-buckling.



Eg:



d) Limit Load instability (or) snap through buckling



eg:- shallow arches:

spherical caps

Van mises Truss:



* When load increases, there should exist a horizontal force for this member.

* There should be a case where $\alpha = 0$ i.e., member becomes horizontal then, the member came to another side opposite (-ve side) called snap through buckling.

c) Shell Buckling (∞) Finite disturbance buckling:-

(This has feature of Bifurcation and snap through.

