

Mechanics, Dynamics

Equilibrium of 2D rods

Lectures 3 and 4

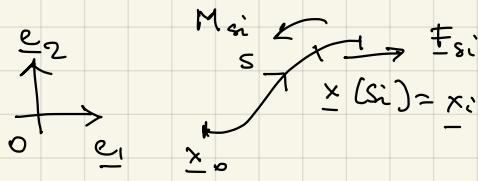
8-10 / 10 / 2025 

Lecture 3-4 Statics of flexor rods

8/10/2025

Start by describing all the loads that may act on the rods.

1. Concentrated forces and couples



$$\underline{F}_{si} = F_{si1} \underline{e}_1 + F_{si2} \underline{e}_2$$

$$\underline{M}_{si} = M_{si} \underline{e}_3 \quad \text{positive component along } \underline{e}_3 \leftarrow \text{counter-clockwise}$$

(thumb) (right hand)

$$\text{total force} \quad \underline{f} = \sum_i \underline{F}_{si}$$

$$\text{total moment with respect to } \underline{x}_0 \quad \underline{M}_{x_0} = \sum_i (x_i - x_0) \wedge \underline{F}_{si} + \sum_i \underline{M}_{si}$$

$$\begin{aligned} \underline{M}_{x_0} &= \underline{M}_{x_0} \cdot \underline{e}_3 = \underbrace{\underline{e}_3 \cdot \sum_i (x_i - x_0) \wedge \underline{F}_{si}}_{=} + \sum_i \underline{M}_{si} \\ &= \sum_i (x_i - x_0) \wedge \underline{F}_{si} \end{aligned}$$

2. Distributed forces : forces per unit length along the curve (linear densities)

$$s \mapsto \underline{f}(s) = (f_1(s), f_2(s)) = f_t(s) \underline{t}(s) + f_n(s) \underline{n}(s) \quad \text{applied at } \underline{x}(s)$$

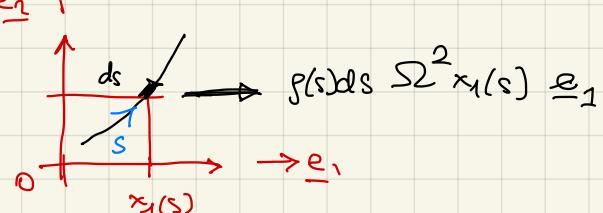
Examples

- weight $\underline{f}(s) = g(s) \underline{g}$ where typically $\underline{g} = -g \underline{e}_2$
 - $\approx 10 \text{ ms}^{-2}$ vertical, upwards
 - volumetric mass density
 - $g(s) = \text{mass } \rho, \text{unit length} = \rho_v(s) A(s)$
 - area of cross section

- centrifugal force $\underline{f}(s) = g(s) \Omega^2 x_1(s) \underline{e}_1$ $\Omega \approx$

These forces contribute to total force \underline{f}

through integrals : $\int_0^l \underline{f}(s) ds$



3. Distributed couples : couples per unit length along the curve (linear densities)

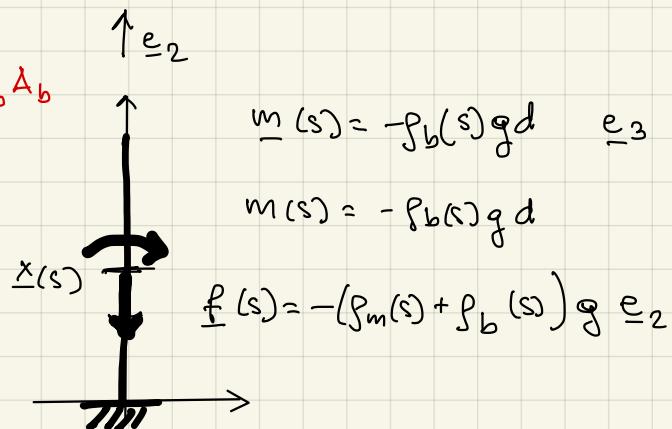
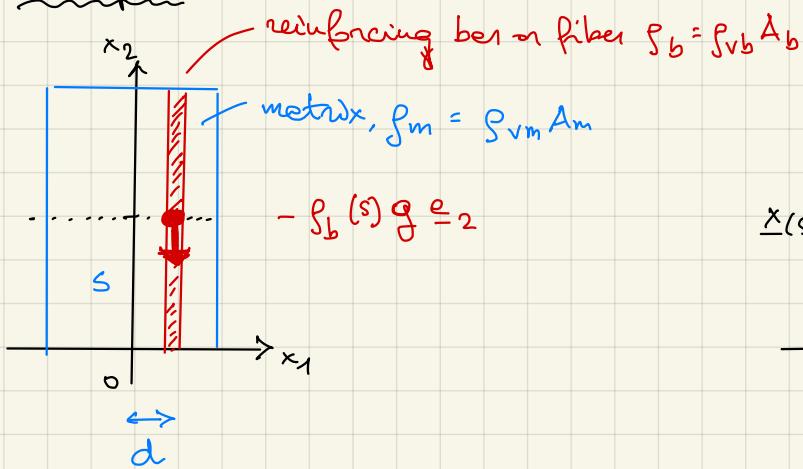
$$s \mapsto \underline{m}(s) = m(s) \underline{e}_3 \quad \text{applied at } \underline{x}(s)$$

These couples do not contribute to total force \underline{f} and contribute to total moment M_{x_0} .

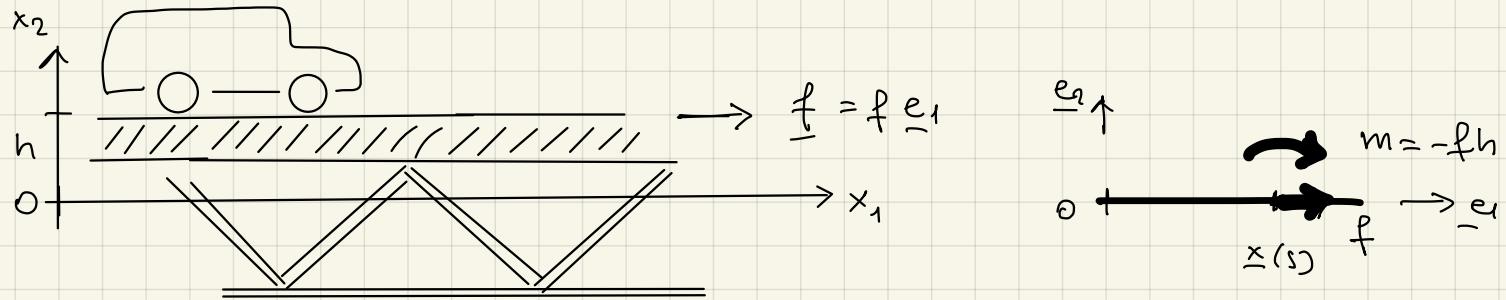
through integrals : $\int_0^l m(s) ds$

Distributed couples typically arise from distributed forces applied with an offset (eccentrically) with respect to $\underline{x}(s)$ (mid-axis of the rod)

Examples

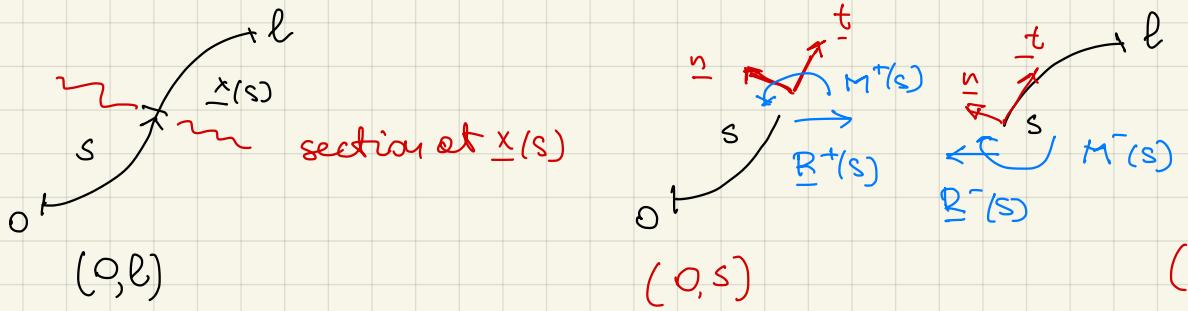


Another example is the load transmitted by a car breaking on a bridge



4. Internal "forces" (stresses in 3D): "forces" transmitted through internal points

Take $s \in (0, l)$ and assume that no concentrated "forces" (forces or couples) act near $\underline{x}(s)$



The action of + part (s, l)

on the - part $(0, l)$

can be represented as

a force $R^+(s)$ applied at $\underline{x}(s)$ (total force)

a couple $M^+(s) e_3$, or $M^+(s)$ (total moment)

applied at $\underline{x}(s)$ with respect to $\underline{x}(s)$

The action of - part $(0, s)$

on the + part (s, l)

can be represented as

a force $R^-(s)$ applied at $\underline{x}(s)$

a couple $M^-(s) e_3$, or $M^-(s)$

applied at $\underline{x}(s)$

similarly

represented means "with the same total force and moment"



We can write:

$$\underline{R}^+(s) = N^+(s) \underline{t}(s) + T^+(s) \underline{n}(s)$$

↳ axial force ↳ shear force
(spazio normale) (spazio di taglio)

$N^+(s)$ bending moment

Remark "action and reaction" principle would suggest $\underline{R}^-(s) = -\underline{R}^+(s)$ and $M^-(s) = -M^+(s)$.

We will see that this is indeed true.

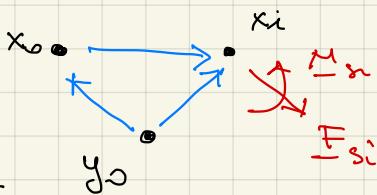
Equilibrium

For a rod to be in equilibrium we'll require that it's in equilibrium as a whole, and also that every one of its parts is in equilibrium. Equilibrium means:

$$\begin{cases} \underline{\underline{F}} = \underline{\underline{0}} & \text{total force} = \underline{\underline{0}} \\ M_{x_0} = \underline{\underline{0}} & \text{total moment w/r to an arbitrary chosen point } x_0 = \underline{\underline{0}} \end{cases}$$

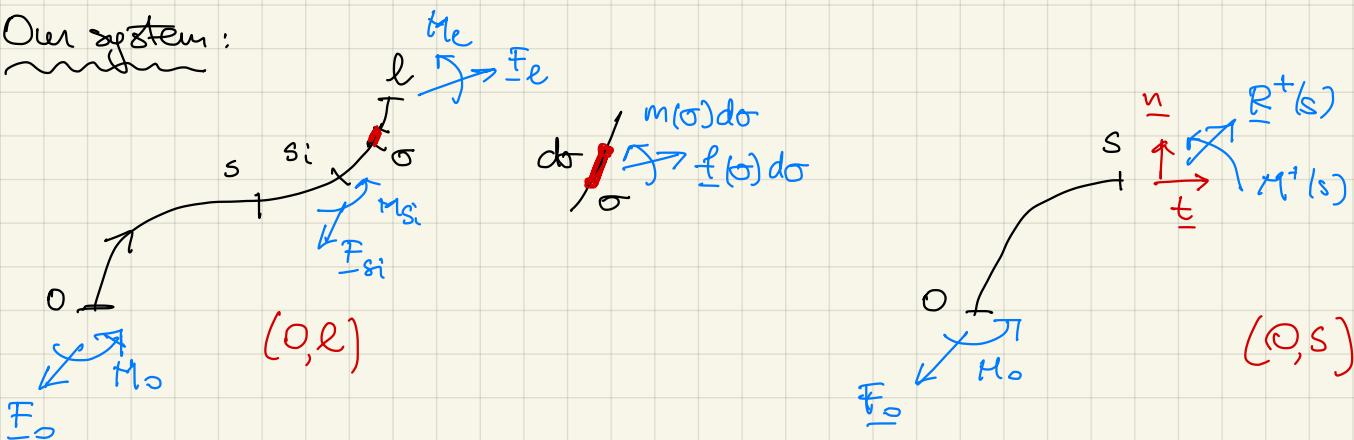
Remark the reason the choice of x_0 is immaterial is that, if the equations above hold,

then also $M_{y_0} = \underline{\underline{0}}$ & $\underline{\underline{F}}_{y_0}$. Indeed



$$M_{y_0} = \sum_i (x_i - y_0) \wedge \underline{\underline{F}}_{si} + \sum_i M_{si} = M_{x_0} + (x_0 - y_0) \wedge \sum_i \underline{\underline{F}}_{si}$$

Our system:



Global equilibrium $(0, l)$

$$\underline{\underline{0}} = \underline{\underline{F}}_O + \int_0^l \underline{\underline{f}}(\sigma) d\sigma + \sum_{0 \leq s_i \leq l} \underline{\underline{F}}_{si} + \underline{\underline{F}}_e$$

$$0 = M_{x(O)} = M_O + \int_0^l m(\sigma) d\sigma + \sum_{0 \leq s_i \leq l} M_{si} + M_l$$

$$+ \int_0^l (\underline{x}(\sigma) - \underline{x}(0)) \wedge \underline{\underline{f}}(\sigma) d\sigma + \sum_{0 \leq s_i \leq l} (\underline{x}(s_i) - \underline{x}(0)) \wedge \underline{\underline{F}}_{si} + (\underline{x}(l) - \underline{x}(0)) \wedge \underline{\underline{F}}_e$$

no external concentrated loads in a neighbourhood of s

Equilibrium of part $(0, s)$

$$\underline{\underline{0}} = \underline{\underline{F}}_O + \int_0^s \underline{\underline{f}}(\sigma) d\sigma + \sum_{0 \leq s_i \leq s} \underline{\underline{F}}_{si} + \underline{\underline{R}}^+(s)$$

$$0 = M_{x(O)} = M_O + \int_0^s m(\sigma) d\sigma + \sum_{0 \leq s_i \leq s} M_{si} + M^+(s)$$

$$+ \int_0^s (\underline{x}(\sigma) - \underline{x}(0)) \wedge \underline{\underline{f}}(\sigma) d\sigma + \sum_{0 \leq s_i \leq s} (\underline{x}(s_i) - \underline{x}(0)) \wedge \underline{\underline{F}}_{si} + (\underline{x}(s) - \underline{x}(0)) \wedge \underline{\underline{R}}^+(s)$$

$$\underline{0} = \underline{F}_0 + \int_0^s \underline{f}(s) ds + \sum_{0 < s_i < s} \underline{F}_{s_i} + \underline{R}^+(s)$$

$$0 = M_{x(s)} = M_0 + \int_0^s m(s) ds + \sum_{0 < s_i < s} M_{s_i} + M^+(s)$$

$$+ \int_s^s (\underline{x}(s) - \underline{x}(s)) \wedge \underline{f}(s) ds + \sum_{0 < s_i < s} (\underline{x}(s_i) - \underline{x}(s)) \wedge \underline{F}_{s_i} + (\underline{x}(s) - \underline{x}(s)) \wedge \underline{R}^+(s)$$

These formulas show that $\underline{R}^+(s)$ and $M^+(s)$ are continuous away from points s_i where concentrated loads are applied. (actually: differentiable, see later)

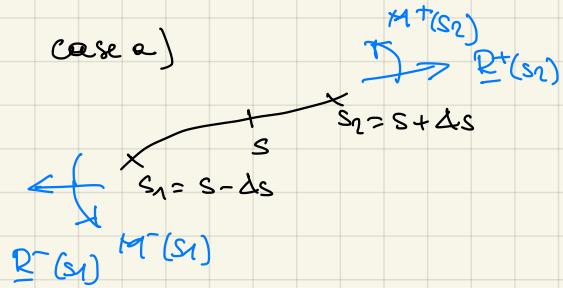
At s_i where \underline{F}_{s_i} (resp. M_{s_i}) is applied, $\underline{R}^+(s)$ (resp., $M^+(s)$) has a jump discontinuity.
(see later for jump relations)

Equilibrium of part (s, l)

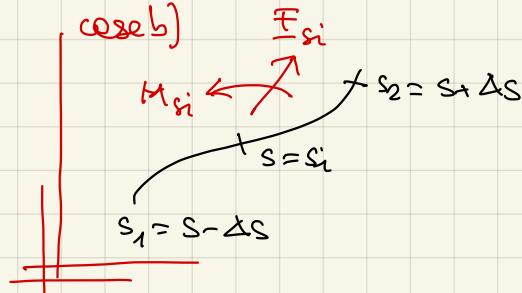
Analogous to case for $(0, s)$, and it shows that $\underline{R}^-(s)$ (respectively, $M^-(s)$) is differentiable away from points s_i where concentrated forces (resp., concentrated couples) are applied

Equilibrium of part $(0 < s_1 = s - \Delta s, s_2 = s + \Delta s < l)$

case a)



case b)



$$\underline{0} = \underline{R}^-(s_1) + \int_{s_1}^{s_2} \underline{f}(s) ds + \sum_{s_1 < s < s_2} \underline{F}_{s_i} + \underline{R}^+(s_2)$$

$$0 = M_{x(s)} = M^-(s_1) + \int_{s_1}^{s_2} m(s) ds + \sum_{s_1 < s_i < s_2} M_{s_i} + M^+(s_2)$$

$$+ (\underline{x}(s_1) - \underline{x}(s)) \wedge \underline{R}^-(s_1) + \int_{s_1}^{s_2} (\underline{x}(s) - \underline{x}(s)) \wedge \underline{f}(s) ds + \sum_{s_1 < s_i < s_2} (\underline{x}(s_i) - \underline{x}(s)) \wedge \underline{F}_{s_i} + (\underline{x}(s_2) - \underline{x}(s)) \wedge \underline{R}^+(s_2)$$

..
(*)

$$\underline{O} = \underline{R}^-(s_1) + \int_{s_1}^{s_2} \underline{f}(\sigma) d\sigma + \sum_{s_1 < s_i < s_2} \underline{F}_{s_i} + \underline{R}^+(s_2)$$

$$\underline{O} = \underline{M}_{x(s)} = \underline{M}^-(s_1) + \int_{s_1}^{s_2} \underline{m}(\sigma) d\sigma + \sum_{s_1 < s_i < s_2} \underline{M}_{s_i} + \underline{M}^+(s_2)$$

$$+ (\underline{x}(s_i) - \underline{x}(s)) \wedge \underline{R}^-(s_1) + \int_{s_1}^{s_2} (\underline{x}(\sigma) - \underline{x}(s)) \wedge \underline{f}(\sigma) d\sigma + \sum_{s_1 < s_i < s_2} (\underline{x}(s_i) - \underline{x}(s)) \wedge \underline{F}_{s_i} + (\underline{x}(s_2) - \underline{x}(s)) \wedge \underline{R}^+(s_2)$$

a) Case of no concentrated loads at s_i , hence or ($s_1 = s - \Delta s$, $s_2 = s + \Delta s$) for Δs small enough.

$$\underline{O} = \underline{R}^-(s - \Delta s) + \int_{s - \Delta s}^{s + \Delta s} \underline{f}(\sigma) d\sigma + \boxed{\underline{O}} + \underline{R}^+(s + \Delta s)$$

$$\lim_{\Delta s \rightarrow 0} \underline{O} = \underline{R}^-(s) + \underline{O} + \boxed{\underline{O}} + \underline{R}^+(s)$$

$$\Rightarrow \boxed{\underline{R}^-(s) = - \underline{R}^+(s)} \quad (\text{action/reaction 1})$$

Take lim and recall that $\underline{R}^\pm(s)$, $\underline{M}^\pm(s)$
 $\Delta s \rightarrow 0$

are continuous away from s_i

$$\cancel{\boxed{\underline{R}^-(s) + \underline{F}_{s_i}}}$$

$$\underline{O} = \underline{M}_{x(s)} = \underline{M}^-(s - \Delta s) + \int_{s - \Delta s}^{s + \Delta s} \underline{m}(\sigma) d\sigma + \boxed{\underline{O}} + \underline{M}^+(s + \Delta s) + (\underline{x}(s - \Delta s) - \underline{x}(s)) \wedge \underline{R}^-(s - \Delta s)$$

$$\downarrow \quad \downarrow \quad \downarrow \quad \downarrow \quad \downarrow \quad \downarrow \quad \downarrow \quad \downarrow$$

$$+ (\underline{x}(\sigma) - \underline{x}(s)) \wedge \underline{f}(\sigma) d\sigma$$

$$+ \boxed{\underline{O}} + (\underline{x}(s + \Delta s) - \underline{x}(s)) \wedge \underline{R}^+(s + \Delta s)$$

$$\lim_{\Delta s \rightarrow 0} \underline{O} = \underline{M}(s) + \underline{O} + \boxed{\underline{O}} + \underline{M}^+(s) + \underline{O} + \underline{O} + \boxed{\underline{O}} + \underline{O}$$

$$\Rightarrow \boxed{\underline{M}^-(s) = - \underline{M}^+(s)} \quad (\text{action/reaction 2})$$

$$\cancel{\boxed{\underline{M}^-(s) + \underline{M}_{s_i} + (\underline{x}(s_i) - \underline{x}(s_i)) \wedge \underline{F}_{s_i}}} = 0$$

Notation write $\underline{R}^+(s) = \underline{R}(s)$ and $\underline{R}^-(s) = - \underline{R}(s)$
 $\underline{M}^+(s) = \underline{M}(s)$ $\underline{M}^-(s) = - \underline{M}(s)$

These are well defined continuous functions away from $s = s_i$. In particular,

$$\underline{R}(s) = \underline{N}(s) \underline{t}(s) + \underline{T}(s) \underline{n}(s)$$

continuous

these are also continuous if α is continuous

b) Case $s = s_i$. Need to add \underline{F}_{s_i} and \underline{M}_{s_i} to the balance equations above, see $\cancel{\boxed{\quad}}$

(case b) continued

Know from balances on $(0, s)$ that $\underline{R}(s)$, $M(s)$ are not defined at $s=s_i$ because they jump.

But they have well defined limits from right (s_+) and left (s_-), and jumps at $s=s_i$

$$0 = \underline{R}(s_i-) + \underline{F}_{s_i} + \underline{R}^+(s_i+) \Rightarrow \underline{R}(s_i+) - \underline{R}(s_i-) + \underline{F}_{s_i} = 0 \Rightarrow [\underline{R}]_{(s_i)} + \underline{F}_{s_i} = 0$$

$$0 = M^-(s_i-) + M_{s_i} + M^+(s_i+) \Rightarrow M(s_i+) - M(s_i-) + M_{s_i} = 0 \Rightarrow [M]_{(s_i)} + M_{s_i} = 0$$

Remark [skip]

Assume $s \neq s_i$. Consider the partial equilibria of $(0, s)$ and (s, l) to obtain global $(0, l)$ equil.

by summing the two partial balances. Or consider global $(0, l)$ and partial $(0, s)$ equil.

to obtain equil. of (s, l) by subtraction

$$\underline{f}(0, l) = \underline{f}(0, s) + \underline{f}(s, l) \quad \underline{R}^+(s) \text{ and } \underline{R}^-(s) \text{ cancel out}$$

$$M_{x(0)}(0, l) = M_{x(0)}(0, s) + M_{x(0)}(s, l) \quad M^+(s) \text{ and } M^-(s) \quad " \quad "$$

(by action and reaction)

Differential equations of equilibrium

Take $s \neq s_i$, s.t. there are no concentrated loads in $s - \Delta s, s + \Delta s$

Consider force and $M_{x(0)}$ balance for $(0, s)$ and take $\frac{d}{ds}$

$$0 = \underline{\theta}(0, s) = \underline{F}_0 + \int_0^s \underline{f}(\sigma) d\sigma + \sum_{s_i < s} \underline{F}_{si} + \underline{R}(s)$$

$$\Rightarrow \boxed{\underline{\theta} = \underline{f}(s) + \underline{R}'(s)} \quad (*)$$

$$0 = M_{x(0)}(0, s) = M_0 + \int_0^s m(\sigma) d\sigma + \sum_{s_i < s} M_{si} + M(s)$$

$$+ \int_0^s (\underline{x}(\sigma) - \underline{x}(0)) \wedge \underline{f}(\sigma) d\sigma + \sum_{s_i < s} (\underline{x}(s_i) - \underline{x}(0)) \wedge \underline{F}_{si} + (\underline{x}(s) - \underline{x}(0)) \wedge \underline{R}(s)$$

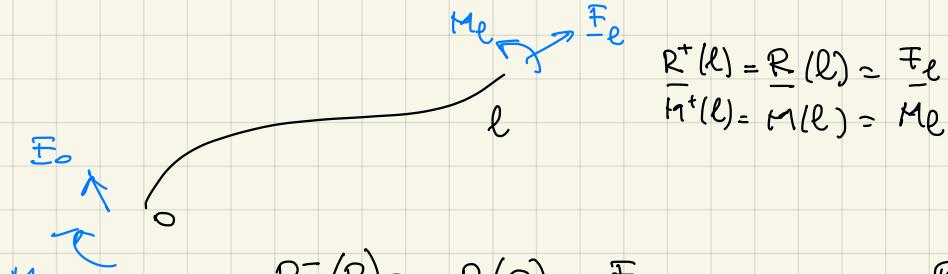
$$\Rightarrow 0 = m(s) + m'(s) + (\underline{x}(s) - \underline{x}(0)) \wedge \underline{f}(s) + \underline{t}(s) \wedge \underline{R}(s) + (\underline{x}(s) - \underline{x}(0)) \wedge \underline{R}'(s)$$

sum to zero by (*)

$$\underbrace{N(s) \underline{t}(s) + T(s) \underline{n}(s)}_{e_3 \cdot \underline{t} \wedge \underline{t} = 0} \quad \underbrace{e_3 \cdot \underline{t} \wedge \underline{n} = 1}$$

$$\Rightarrow \boxed{\dot{m}'(s) + T(s) + m(s) = 0}$$

BC's for the differential equations of equilibrium



$$\underline{R}^+(l) = \underline{R}(l) = \underline{F}_l$$

$$\underline{M}^+(l) = \underline{M}(l) = \underline{M}_l$$

$$R^-(0) = -R(0) = \underline{F}_0$$

$$M^-(0) = -M(0) = \underline{M}_0$$

$$\underline{R}(0) = -\underline{F}_0$$

$$M(0) = -\underline{M}_0$$

Re: In this case, R and M are differentiable at s

Summary of Statics of 2d rods

$$\lim_{s \rightarrow 0} \underline{R}(s) = -\underline{F}_o \quad \lim_{s \rightarrow 0} \underline{M}(s) = -M_o$$

$$\frac{d}{ds} \underline{R}(s) + \underline{f}(s) = 0 \quad (*)$$

$$\frac{d}{ds} \underline{M}(s) + T(s) + M(s) = 0$$

$$[\underline{R}]_{(s_0)} + \underline{F}_{s_0} = 0$$

$$[\underline{M}]_{(s_i)} + M_{s_i} = 0$$

$$\lim_{s \rightarrow l} \underline{R}(s) = \underline{F}_l \quad \lim_{s \rightarrow l} \underline{M}(s) = M_l$$

(*) in components, relative to the local frame $\{\underline{x}(s), \underline{t}(s), \underline{n}(s)\}$

$$0 = \underbrace{\frac{d}{ds} (N(s)\underline{t}(s) + T(s)\underline{n}(s))}_{N'\underline{t} + N\underline{t}' + T'\underline{n} + Tn'} + f_t \underline{t}(s) + f_n \underline{n}(s)$$

$$(N' - cT) \underline{t} + (T' + cN) \underline{n}$$

$$\Rightarrow \begin{cases} N' - cT + f_t = 0 \\ T' + cN + f_n = 0 \end{cases}$$

Special case straight rod, $c=0$:

$$\begin{cases} \frac{d}{ds} N(s) + f_t(s) = 0 \\ \frac{d}{ds} T(s) + f_n(s) = 0 \end{cases}$$

- Summary of equilibrium equations on a straight rod

$$M' + T + m = 0$$

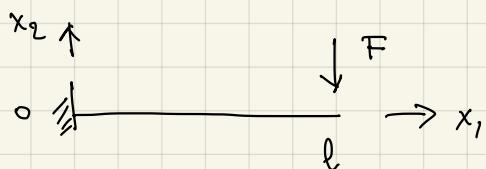
$$T' + f_u = 0$$

$$N' + f_t = 0$$

- Boundary conditions (at $s=0$; as an exercise, work out BC's at $s=l$)

clamp	hinge	roller // t	free	unknown
$u(0) = w(0) = 0$	$u(0) = w(0) = 0$	$u(0) \text{ } u \quad w(0) = 0$	$u(0), w(0)$	
$\alpha(0) = 0$	$\alpha(0) \text{ } u$	$\alpha(0) \text{ } u$	$\alpha(0)$	
$\text{Rot} = -N(0) \text{ } u$	$\text{Rot} = -N(0) \text{ } u$	$\text{Rot} = -N(0) = 0$	$F_{t0} = -N(0)$	
$R_{u0} = -T(0) \text{ } u$	$R_{u0} = -T(0) \text{ } u$	$R_{u0} = -T(0) \text{ } u$	$F_{u0} = -T(0)$	
$M(0) \text{ } u$	$M(0) = 0$	$M(0) = 0$	$M_0 = -M(0)$	

- Diagrams of the components of internal forces



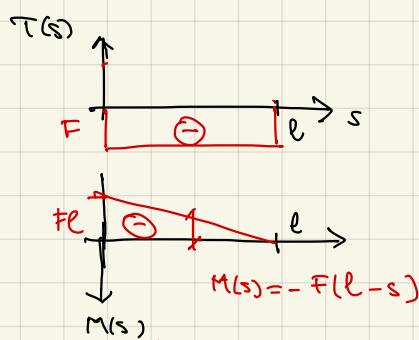
Free Body Diagram (FBD): replace constraints with the unknown reactions they can exert

	$R_{u1} = 0$	$= -N(0)$
	$R_{t1} - F = 0 \Rightarrow R_{t1} = F$	$= -T(0)$
	$M_0 - Fl = 0 \Rightarrow M_0 = Fl$	$= -M(0)$

$$N' = 0, N(0) = 0 \Rightarrow N(s) \equiv 0$$

$$T' = 0, T(l) = -F \Rightarrow T(s) = -F$$

$$M' = -T, M(0) = -Fl \Rightarrow M(s) = -F(l-s)$$



Remark: the M graph is on the side of stretched fibers. Positive M stretches bottom fibers