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Mechanics, Drionics

lectures 3-4

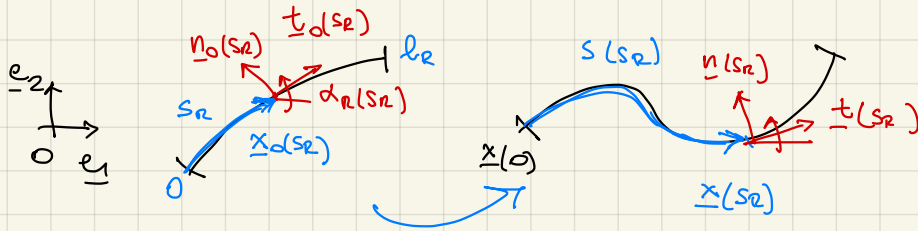
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# Lecture 4 Kinematics of deformable planar rods

Planar curves as models for 1D deformable structures (from elastic arms to elephant trunks)

Deformation: a map that transforms each point of a reference curve (reference configuration) into a point of a new, deformed curve (deformed configuration)



$$[0, l_R] \ni s_R \mapsto \underline{x}(s_R) \in \mathbb{R}^2$$

$$\text{s.t. } |\underline{x}'(s_R)| \neq 0 \quad \forall s_R \in [0, l_R]$$

Assume, for simplicity, that the ref. config. is parametrized by arc-length, but drop  $\sim$  notation.

$$[0, l_R] \ni s_R \mapsto \underline{x}_0(s_R) \quad \text{reference configuration,} \quad s_R \text{ ref. arc-length}$$

$$\underline{t}_0(s_R) = \underline{x}'_0(s_R) = (\cos \alpha_0(s_R), \sin \alpha_0(s_R))$$

$$c_0(s_R) = \alpha'_0(s_R) \quad \text{etc., ...}$$

$$[0, l_R] \ni s_R \mapsto \underline{x}(s_R) \quad \text{deformed configuration}$$

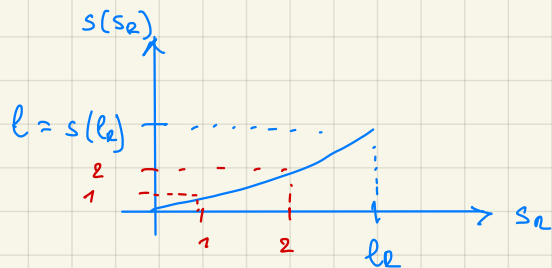
$\hookrightarrow$  ref. arc-length = marker of a point in the ref. config. (Lagrangian marker)

If the curve is stretched into deformation, arc-length  $s$  will NOT be  $s_R$ .

$$\underline{t}(s_R) = \frac{\underline{x}'(s_R)}{|\underline{x}'(s_R)|} = (\cos \alpha(s_R), \sin \alpha(s_R))$$

$$s(s_R) = \int_0^{s_R} |\underline{x}'(\sigma)| d\sigma$$

$$c(s_R) = \frac{\alpha'(s_R)}{|\underline{x}'(s_R)|}$$



From  $s'(s_R) = |\underline{x}'(s_R)| > 0 \quad \forall s_R$ ,  $s$  is monotone increasing:  $s_{R1} < s_{R2} \Rightarrow s(s_{R1}) < s(s_{R2})$

This has now a mechanical meaning. Two points along the curve cannot change their ordering when a deformation takes place.

$$s(l_R) = \int_0^{l_R} |\underline{x}'(s)| ds = l \quad \text{length of the deformed curve}$$

If  $|\underline{x}'(s_R)| = 1 \forall s_R$ , then  $s_R$  is arc-length also for  $\underline{x}(s_R)$ , and  $l = l_R$

If  $|\underline{x}'(s_R)| \neq 1$ , then  $s_R$  is NOT arc-length for  $\underline{x}(s_R)$ , and length of the curve changes with the deformation

Setting  $|\underline{x}'(s_R)| = 1 + \varepsilon(s_R) > 0$  we can define the extensional strain  $\varepsilon \in (-1, +\infty)$

$$\varepsilon(s_R) := \left| \frac{d}{ds_R} \underline{x}(s_R) \right| - 1 \quad \begin{cases} > 0 & \text{if the curve is locally lengthened} \\ < 0 & \text{shortened} \end{cases} \quad \begin{aligned} \frac{ds}{ds_R} &= |\underline{x}'(s_R)| > 1 \\ \frac{ds}{ds_R} &= |\underline{x}'(s_R)| < 1 \end{aligned}$$

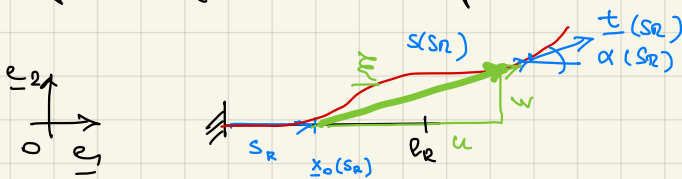
$$\underline{\xi}(s_R) = \underline{x}(s_R) - \underline{x}_R(s_R) \quad \text{displacement (vector) at } s_R$$

$$= \xi_1(s_R) \underline{e}_1 + \xi_2(s_R) \underline{e}_2$$

$$= u(s_R) \underline{t}_0(s_R) + w(s_R) \underline{n}_0(s_R)$$

$u, w$  axial and transversal components of the displacement at  $s_R$

If the reference configuration is straight line



$$\begin{aligned} \underline{\xi}(s_R) &= \underline{x}(s_R) - \underline{x}_R(s_R) \\ &= u(s_R) \underbrace{\underline{t}_0(s_R)}_{\underline{e}_1} + w(s_R) \underbrace{\underline{n}_0(s_R)}_{\underline{e}_2} \end{aligned}$$

$$\Rightarrow u(s_R) = \xi_1(s_R), \quad w(s_R) = \xi_2(s_R)$$

$$\underline{t}(s_R) = \frac{\underline{x}'(s_R)}{|\underline{x}'(s_R)|} = \frac{\underline{x}'(s_R)}{1 + \varepsilon(s_R)} = (\cos \alpha(s_R), \sin \alpha(s_R))$$

$$w(s_R) = \alpha(s_R) - \alpha_R(s_R) \quad \text{rotation at } s_R$$

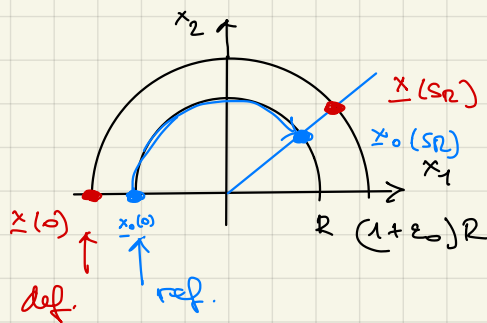
$$\begin{aligned} \kappa(s_R) &= w'(s_R) = \alpha'(s_R) - \alpha'_R(s_R) \quad \text{bending strain at } s_R \quad \kappa \in (-\infty, +\infty) \\ &= (1 + \varepsilon(s_R)) c(s_R) - c_R(s_R) \end{aligned}$$

Remark  $\kappa$  is NOT the difference of curvatures. It becomes the difference of curvatures only

when  $\varepsilon = 0$  (inextensible case), so that  $s_R$  is arc length and  $\kappa = \alpha'(s_R) - \alpha'_0(s_R) = c(s_R) - c_0(s_R)$

Extensional and bending strains  $\neq 0$  are the descriptors of the change of shape of a beam element.

## Example 1



$$s(s_R) = (1 + \epsilon_0) s_R$$

$$\alpha(s_R) = \alpha_0(s_R)$$

$$c_0(s_R) = -\frac{1}{R}, \quad c(s_R) = -\frac{1}{(1 + \epsilon_0)R}$$

With our definitions:  $\epsilon = \epsilon_0$ ,  $K = \alpha'(s_R) - \alpha'_0(s_R) = 0$ ,  $c(s_R) - c_0(s_R) = -\frac{1 + (1 + \epsilon_0)}{(1 + \epsilon_0)R} \neq 0$

so, in this example, ext. strain =  $\epsilon_0$  and no bending strain.

Remark: This is the configuration that would be reached due to a thermal dilation  $\epsilon_0 = \alpha \Delta t$  (purely external) of the reference configuration.

## Special cases

1. Inextensible curve,  $\epsilon \equiv 0$

$$|x'| = 1 + \epsilon = 1 \Rightarrow s'(s_R) \equiv 1 \Rightarrow s(s_R) = s_R$$

$s_R$  is arc-length on a deformed curve as well  
[drop  $R$  from  $s_R$ ]

$$c(s_R) = \alpha'(s_R)$$

$$K = \alpha'(s_R) - \alpha'_0(s_R) = c(s_R) - c_0(s_R)$$

If, in addition, the ref. config. is straight ( $\alpha'_0(s_R) \equiv 0$ ), then  $K(s_R) = \alpha'(s_R) = c(s_R)$

In this last case, change of shape = change of curvature from zero to  $\alpha'(s_R) = c(s_R)$

2. If  $\epsilon \equiv 0$  and  $K \equiv 0$ , inextensible and inflexible rod  $\Rightarrow$  the rod is rigid

$\epsilon = 0 \Rightarrow s_R$  is arc-length on  $x$  as well and, in particular,  $x$  and  $x_0$  have the same length

$$K = 0 \Rightarrow \underbrace{\alpha'(s_R)}_{= c(s_R)} = \underbrace{\alpha'_0(s_R)}_{= c_0(s_R)} \quad \text{so that } x \text{ and } x_0 \text{ have the same curvature}$$

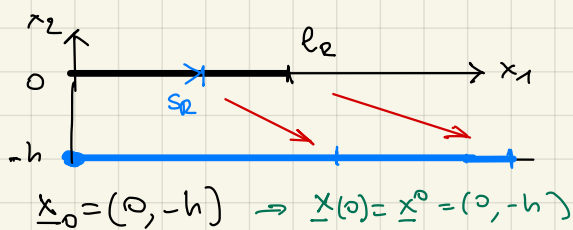
The two curves  $x$  and  $x_0$  can only differ by the integration constants  $\alpha(0), \alpha_0(0); x(0), x_0(0)$

in the curvature  $\rightarrow$  curve integration

$\Rightarrow$  The reference and deformed curve can differ at most by a rigid motion.

## Example 2 (pure stretching of a straight rod)

[change notation:  $\underline{x}_R(s_R) \rightarrow \underline{x}_0(s_R)$ ]



$$\begin{aligned}\underline{x}_0(s_R) &= 0 + s_R \underline{e}_1 \\ \underline{x}_R(s_R) &= 0 + s_R \underline{e}_1, \quad \underline{t}_R = \underline{x}'_R = \underline{e}_1 = \cos \alpha_R \underline{e}_1 + \sin \alpha_R \underline{e}_2 \\ \text{ref. arc-length} &\leftarrow \\ \Rightarrow \alpha_R &\equiv 0, \quad c_R(s_R) = \frac{ds_R}{ds_R} = 1 \\ \alpha_0 &\equiv 0, \quad c_0(s_R) = \frac{ds_0}{ds_R} = 1 \quad \text{etc.}\end{aligned}$$

$$\underline{x}(s_R) = \underline{x}_0 + x_1(s_R) \underline{e}_1$$

$$\underline{x}'(s_R) = x'_1(s_R) \underline{e}_1 + 0 \underline{e}_2$$

$$|\underline{x}'(s_R)| = |x'_1(s_R)| = 1 + \varepsilon(s_R) > 0$$

$$s'(s_R) = 1 + \varepsilon(s_R) \equiv \varepsilon_0 \in (-1, +\infty)$$

$$s(s_R) = \int_0^{s_R} (1 + \varepsilon(\sigma)) d\sigma$$

$$= \int_0^{s_R} (1 + \varepsilon_0) d\sigma = (1 + \varepsilon_0) s_R$$

Special case: affine deformation (drawing not enough!)

$$x'_1(s_R) \equiv \lambda_0 = 1 + \varepsilon_0 \quad \text{constant}$$

$$\Rightarrow x_1(s_R) - \underbrace{x_1(0)}_{=0} = \lambda_0 s_R = (1 + \varepsilon_0) s_R$$

$$l = s(l_R) = (1 + \varepsilon_0) l_R = \lambda_0 l_R \Rightarrow$$

$$\lambda_0 = \frac{l}{l_R} \quad \text{stretching ratio}$$

$$l = l_R + \varepsilon_0 l_R \Rightarrow \varepsilon_0 = \frac{l - l_R}{l_R} \quad \text{change of length per unit reference length}$$

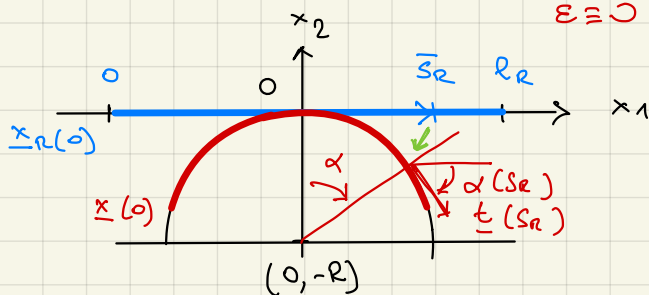
If deformation is not affine, same story but at the local/inferential level

$$\lambda(s_R) = |\underline{x}'(s_R)| = \frac{ds(s_R)}{ds_R}, \quad \varepsilon(s_R) = \frac{ds(s_R)}{ds_R} - 1 = \frac{ds(s_R) - ds_R}{ds_R}$$

## Example 3 (pure bending of an inextensible, initially straight curve)

$$\varepsilon \equiv 0$$

$$c_R(s_R) \equiv 0$$



$$\underline{x}_R(s_R) = \left(-\frac{l_R}{2}, 0\right) + s_R \underline{e}_1, \quad s_R \in (0, l_R)$$

$$\underline{x}(s_R) \text{ maps the circle, } \underline{x}\left(\frac{l_R}{2}\right) = \underline{x}_R\left(\frac{l_R}{2}\right) = (0, 0)$$

$\hookrightarrow$  arc-length or def. curve as well, since  $\varepsilon \equiv 0$

Impose that  $s(\bar{s}_R) = \bar{s}_R$ :

$$-\underbrace{\alpha(\bar{s}_R)}_{<0} R = \bar{s}_R - \frac{l_R}{2} \Rightarrow \alpha(s_R) = -\frac{1}{R} \left(s_R - \frac{l_R}{2}\right)$$

[Check sign:  $\alpha > 0$  for  $s < \frac{l_R}{2}$ ,  $\alpha < 0$  for  $s > \frac{l_R}{2}$ ]

$$\text{Notice: angular coverage } \alpha(l_R) = -\frac{1}{R} \left(l_R - \frac{l_R}{2}\right) = -\frac{l_R}{2R} = -\frac{\pi}{2} \text{ if } l_R = \pi R$$

$$\text{Also: } c(s_R) = \alpha'(s_R) = -\frac{1}{R}, \quad \kappa(s_R) = \alpha'(s_R) = -\frac{1}{R}$$

arc-length  $\uparrow$

$$= c(s_R) = c_R(s_R) = 0$$

[Ex: How long should the ref. curve be to cover 20 times a disk of radius R?]

$$A: 20 \times 2\pi R$$

From (\*) giving  $s_R \mapsto \alpha(s_R)$  can reconstruct the curve by integrating

$$\underline{x}'(s_R) = \cos \alpha(s_R) \underline{e}_1 + \sin \alpha(s_R) \underline{e}_2$$

$$\underline{x}\left(\frac{l_R}{2}\right) = (0, 0)$$

$\rightarrow$  exercise : write the parametrization  $s \mapsto \underline{x}(s)$  explicitly

#### Example 4 (later)

Roll a ribbon around a spool.

a) circular disk, ghost ribbon (ribbon thickness = 0)

b) logarithmic spiral, to take into account the thickness of the ribbon.

## Relation between strains $\varepsilon$ , $\kappa$ and displacements

$$\left[ \begin{array}{l} \text{Change notation } \underline{x}_R(s_R) \rightarrow \underline{x}_0(s_0) \\ \underline{t}_R(s_R) \rightarrow \underline{t}_0(s_0) \end{array} \right]$$

Assume  $s_R$  is arc-length on  $x_R$  and drop  $\sim$

$$\underline{x}(s_R) = \underline{x}_0(s_R) + \underline{\xi}(s_R)$$

$$\begin{aligned} \underline{x}'(s_R) &= \underline{x}'_0(s_R) + \underline{\xi}'(s_R) \\ &= \underline{t}_0(s_0) + \underline{\xi}'(s_R) \end{aligned}$$

Now compute the various quantities ....

$$|\underline{x}'|^2 = 1 + 2 \underline{t}_0 \cdot \underline{\xi}' + \underline{\xi}' \cdot \underline{\xi}'$$

$$|\underline{x}'| = 1 + \varepsilon = (1 + 2 \underline{t}_0 \cdot \underline{\xi}' + |\underline{\xi}'|^2)^{1/2} \Rightarrow \varepsilon = (1 + 2 \underline{t}_0 \cdot \underline{\xi}' + |\underline{\xi}'|^2)^{1/2} - 1$$

$$\underline{\xi} = u \underline{t}_0 + w \underline{n}_0$$

$$\underline{t}'_0 = c_0 \underline{n}_0$$

$$\underline{n}'_0 = -c_0 \underline{t}_0$$

$$\begin{aligned} \underline{\xi}' &= u' \underline{t}_0 + u \underline{t}'_0 + w' \underline{n}_0 + w \underline{n}'_0 \\ &= (u' - c_0 w) \underline{t}_0 + (w' + c_0 u) \underline{n}_0 \end{aligned}$$

$$\Rightarrow \underline{\xi}' \cdot \underline{t}_0 = (u' - c_0 w)$$

$$|\underline{\xi}'|^2 = (u' - c_0 w)^2 + (w' + c_0 u)^2$$

$$\begin{aligned} \varepsilon &= (1 + 2(u' - c_0 w) + (u' - c_0 w)^2 + (w' + c_0 u)^2)^{1/2} - 1 \\ &= \sqrt{(1 + u' - c_0 w)^2 + (w' + c_0 u)^2} - 1 \end{aligned}$$

Special case :  $c_0 \equiv 0$  (straight ref. config.)

$$\varepsilon = \sqrt{(1 + u')^2 + w'^2} - 1$$

(1)

Similar calculations for  $\sin w$ ,  $\cos w$ ,  $\kappa$  as functions of  $u$ ,  $w$ .

General formulas in LNs, here special case  $c_R \equiv 0$

$$\cos w = \frac{1 + u'}{((1 + u')^2 + w'^2)^{1/2}}, \quad \sin w = \frac{w'}{((1 + u')^2 + w'^2)^{1/2}}, \quad \kappa = \omega' = \frac{w''(1 + u') - w'u''}{((1 + u')^2 + w'^2)} \quad (\text{Ex})$$

(2)

Complements Proof of  $K = \omega' = \frac{w''(1+u') - w'u''}{(1+u')^2 + w'^2}$  in the case  $C_0 \equiv 0$ .

We have  $C_0 = \alpha'_0 \equiv 0 \Rightarrow \alpha_0(s_R) \equiv \alpha_0^0 = \text{const.}$  Assume  $\alpha_0^0 = 0$ , <sup>for simplicity</sup> so that  $w = \alpha - \alpha_0 = \alpha$   ~~$\alpha_0 = 0$~~

$$\underline{x}(s_R) = \underline{x}_0(s_R) + \underline{\xi}(s_R) = s_R \underline{e}_1 + u(s_R) \underline{t}_0 + w(s_R) \underline{n}_0$$

$\underline{t}_0 = \underline{e}_1$        $\underline{n}_0 = \underline{e}_2$

$$\underline{t} = \cos \alpha \underline{e}_1 + \sin \alpha \underline{e}_2 = \frac{1}{|\underline{x}'|} \left( \underline{t}_0 + \underline{\xi}'(s_R) \right) = \frac{1}{|\underline{x}'|} \left( (1+u') \underline{e}_1 + w' \underline{e}_2 \right)$$

$$\cos \alpha = \cos w = \frac{1}{\sqrt{(1+u')^2 + w'^2}} (1+u')$$

$$\sin \alpha = \sin w = \frac{w'}{\sqrt{(1+u')^2 + w'^2}}$$

Re:  $|\underline{x}'| = 1 + \varepsilon =$   
 $= \sqrt{(1+u')^2 + w'^2}$

To compute  $K = \omega'$ , observe that

$$\frac{d}{ds_R} \cos w = -w' \sin w =$$

$$= \frac{1}{(1+u')^2 + w'^2} \left[ u'' \left( (1+u')^2 + w'^2 \right)^{1/2} - (1+u') \frac{\cancel{2} (1+u') u'' + \cancel{2} w' w''}{\left( (1+u')^2 + w'^2 \right)^{1/2}} \right]$$

$$= \frac{1}{\left( (1+u')^2 + w'^2 \right)^{1/2}} \left[ u'' - (1+u') \frac{(1+u') u'' + w' w''}{(1+u')^2 + w'^2} \right]$$

$$= - \frac{1}{\sqrt{(1+u')^2 + w'^2}} \left[ \cancel{(1+u')^2 u''} + (1+u') w' w'' - \cancel{u'' (1+u')^2} - u'' w'^2 \right]$$

$$= - \frac{1}{\sqrt{(1+u')^2 + w'^2}} \left[ \frac{(1+u') w'' - u'' w'}{(1+u')^2 + w'^2} \right] w' = - \frac{(1+u') w'' - u'' w'}{(1+u')^2 + w'^2} \underbrace{\frac{w'}{\sqrt{(1+u')^2 + w'^2}}}_{\sin w}$$

From this identity we recognize that

$$\omega' = K = \frac{(1+u') w'' - u'' w'}{(1+u')^2 + w'^2}$$



Case of "small deformations":  $\underline{x}(s_R) = \underline{x}_0(s_R) + \gamma \bar{u} \underline{t}_0 + \gamma \bar{w} \underline{n}_0$   
 $\underline{L} = \underline{e}_1$   $\underline{L} = \underline{e}_2$  because  $\underline{c}_0 \equiv 0$

where  $u = ||u|| \frac{u}{||u||} = \gamma \bar{u}$ ,  $\gamma$  small:  $\gamma \ll 1$ ,  $\gamma^2 \ll \gamma$ : neglect  $\gamma^2$  w/r to  $\gamma$   
 eg.,  $\sup |u|$ ,  $(\int_0^{l_R} |u(\sigma)|^2 d\sigma)^{1/2}$ , etc., ....

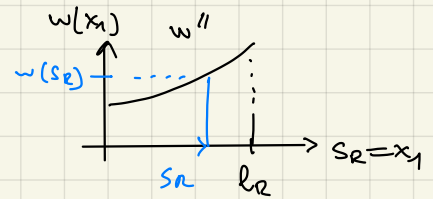
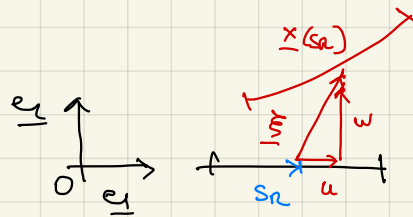
$$\varepsilon \stackrel{(1)}{=} (1 + 2\gamma \bar{u}' + \gamma^2 \bar{u}'^2 + \gamma^2 \bar{w}'^2)^{1/2} - 1$$

$$\simeq (1 + 2\gamma \bar{u}')^{1/2} - 1 \simeq \cancel{1} + \frac{1}{2} 2\gamma \bar{u}' - \cancel{1} = \gamma \bar{u}' = u'$$

$$(1+t)^{1/2} = 1 + \frac{1}{2}(1+t)^{-1/2} \Big|_{t=0} t + \dots \simeq 1 + \frac{1}{2} t + \dots$$

$$\kappa \stackrel{(2)}{=} \frac{\gamma \bar{w}'' (1 + \gamma \bar{u}') - \gamma^2 \bar{w}' \bar{u}''}{1 + 2\gamma \bar{u}' + \gamma^2 \bar{u}'^2 + \gamma^2 \bar{w}'^2}$$

$$\simeq \frac{\gamma \bar{w}''}{1} = w''$$



In other words,  $\boxed{\varepsilon \simeq u'}$   $\boxed{\kappa \simeq w''}$  as in *Scienza delle Costruzioni*.  
 $(1)$   $(2)$

### Remark

In the solution of elasticity pb's in large deformations it is often more convenient to work directly with the geometric variables  $\varepsilon(s_R) = s'(s_R) - 1$ ,  $\kappa(s_R) = w'(s_R)$  rather than with displacement components  $u, w$   
 possibly  $= c(s_R) - c_0(s_R)$