

Analysis of Mode I and Mixed Mode Stress Intensity Factors with Digital Image Correlation

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Abstract

Failure in a material is often dependent on the presence of flaws within the part. In this experiment, Digital Image Correlation was used to analyze the displacement of a thin sheet of PMMA with an induced edge crack within a symmetric three point bend configuration. This test was then repeated using an asymmetric three point bend configuration to produce a set of data under mixed mode loading. The Westergaard stress function solution for an edge crack was used to compute the stress intensity factors for each data set, and the Mode 1 data was compared to an analytical solution calculated with strength of materials equations.

1 Introduction

When a material fails, the theoretical stress required to break the molecular bonds is less than the ultimate stress. This occurs because of geometric features, such as cracks, which cause stress concentrations that magnify the effect of applied loads. Classical strengths of materials equations are useful for producing general solutions to problems, but in order to analyze specific cases in detail, Linear Elastic Fracture Mechanics (LEFM) are used instead. LEFM assumes that cracks are inherently present in a material, that a crack is a free surface within the surrounding stress field, the elastic solution surrounding this crack

is based on a stress intensity factor K , and that the stress on the crack tip can be used to predict the crack propagation. Furthermore, the area surrounding a crack consists of three major parts. The first part, consisting of the crack tip, is a region of plastic deformation at which it is not valid to use LEFM due to the plastic deformation there. The second region is the linear elastic region, where LEFM is applied in describing the stress state of the material. The third section is the far field where the stress state is no longer affected by the presence of the crack. By analyzing the second region, information on the stress intensity factor can be determined for a variety of loads.

In previous works, such as that of Dally and Sanford (citation), strain gauges were used to capture displacement around crack tips. While strain gauges are useful in providing displacement data, they also only are valid over the region directly underneath the gauge, limiting the information provided. Optical techniques, however, can be used to provide the full-field displacement data surrounding a crack. This data is useful because through analyzing the contours present in full-field data, it can be determined what mode in which the crack is loaded as well as provides a more complete data set for analysis. Digital image correlation is attractive method because it is applicable on most materials, does not require any particularly complicated equipment, and is easy to set up and use.

This paper describes the use of digital image correlation (DIC) to capture displacement data around a crack tip when it undergoes pure Mode 1 loading and mixed Mode 1 and Mode 2 loadings. Using this information, the stress intensity factors are calculated using the Westergaard stress function solution for both Mode 1 and mixed mode loads. (Dally and Sanford). These calculated stress intensity factors from LEFM are then compared to the analytical solution for the given loading scenario.

2 Methods

2.1 Experimental Techniques

2.1.1 Digital Image Correlation

DIC is a commonly used optical technique in experimental mechanics to accurately measure full-field displacements, and rotations by capturing a sequence of images of the surface in question during testing. Using two cameras these measurements can be taken in 3D. If only 2D measurements are needed, as in this experiment, only one camera is needed for imaging the surface. For correlation to take place an area of interest with in the images is selected and then divided into square sections known as subsets. Each subset is made up of the same number of pixels. The subsets are matched from the deformed images to the reference image through a correlation function within the DIC algorithm. By corresponding each pixel to an actual unit of length deformations and rotations can then be tracked using a DIC algorithm [1].

For the DIC algorithm to be effective each subset must contain enough unique features. These features are related to contrast or pixel values within each subset. A high-contrast uniform granular surface is desired to create such usable features. In practice this is known as a speckle pattern. A usable speckle pattern consists of uniformly dispersed speckles of random shapes and varying sizes. The size of the speckles and number of subsets influence the accuracy of measurements thus it is important to verify a proper speckle pattern is used [?].

2.1.2 Experimental Determination of Displacement Fields

Mode I displacement fields can be obtained by symmetrically loading a SENB specimen. The geometry of the SENB specimen is given in Figure 7. Alignment of the roller directly above the crack tip creates pure bending around the crack, which induces a predominantly Mode I crack opening, and therefore Mode I displacement fields. Mixed mode displacement fields can be obtained by inducing eccentricity in the SENB specimen. This can be done by moving one of the lower support rollers, creating a combination of bending (Mode I) and shear (Mode II) around the crack. This configuration is given in Figure 8.

2.1.3 Calculation of Stress Intensity Factors

Theoretical Mode I stress intensity factors (K_I) can be calculated using the closed-form equation:

$$K_I = Y\sigma\sqrt{\pi a} \quad (1)$$

where σ is the farfield stress, a is the crack length and Y is a geometric factor specific to the loading configuration and specimen geometry. Using the solution developed in *The Stress Analysis of Cracks Handbook*, the stress intensity factor for the single edge notched bend specimen being examined is [2]

$$K_I = \frac{P}{B\sqrt{w}} \left(\frac{\frac{3S}{w}\sqrt{\frac{a}{w}}}{2(1 + \frac{2a}{w})(1 - \frac{a}{w})^{1.5}} \right) \left[1.99 - \frac{a}{w} \left(1 - \frac{a}{w} \right) \left[2.15 - 3.93 \left(\frac{a}{w} \right) + 2.7 \left(\frac{a}{w} \right)^2 \right] \right] \quad (2)$$

where P is the applied load, S is the support span, w is the specimen width, B is the specimen thickness and a is the crack length.

Experimental Mode I stress intensity factors can be found using experimental displacement fields. From the Westergaard solution, the displacements around the crack can be described by the following equations [3]

$$u_x = \frac{K_I}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa - 1) \cos\left(\frac{\theta}{2}\right) - \cos\left(\frac{3\theta}{2}\right) \right] \quad (3)$$

$$u_y = \frac{K_I}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa + 1) \sin\left(\frac{\theta}{2}\right) - \sin\left(\frac{3\theta}{2}\right) \right] \quad (4)$$

where (r,θ) are the polar coordinates of the point, μ is the shear modulus, $\kappa = \frac{3-\nu}{1+\nu}$, and ν is Poisson's ratio for the plane stress state of the experiment.

Mixed mode stress intensity factors can also be found using the experimental mixed mode displacement fields. The displacement field for a combined Mode I and Mode II loading situation is the sum of the

displacements for each individual loading condition, as given by the following equations [3]

$$u_x = \frac{K_I}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa - 1) \cos\left(\frac{\theta}{2}\right) - \cos\left(\frac{3\theta}{2}\right) \right] + \frac{K_{II}}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa + 3) \sin\left(\frac{\theta}{2}\right) + \sin\left(\frac{3\theta}{2}\right) \right] \quad (5)$$

$$u_y = \frac{K_I}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa + 1) \sin\left(\frac{\theta}{2}\right) - \sin\left(\frac{3\theta}{2}\right) \right] - \frac{K_{II}}{8\mu\pi} \sqrt{2\pi r} \left[(2\kappa - 3) \cos\left(\frac{\theta}{2}\right) + \cos\left(\frac{3\theta}{2}\right) \right] \quad (6)$$

Thus, individual K_I and K_{II} values can be solved by solving the system of equations with the known displacement values at any given point.

2.2 Procedure

The experiment was conducted with a thin rectangular PMAA specimen . To simulate a crack the specimen was cut with a bandsaw in the center to a length of 25.4 mm. This method was chosen to allow for a blunt crack tip so that premature failure of the specimen would not occur while testing. One side of the plate was painted completely white and then speckled using a black aerosol paint. The can of aerosol paint was lightly dispensed in the direction of the specimen with a small electric fan in front. Here, the fan aided in random dispersion of speckles while also maintaining a descent speckle size. A close up of the resulting speckle pattern can be seen in figure ??

To extract Mode I data the specimen was placed symmetrically in a three-point bend fixture within an electronic screw-driven universal testing machine developed by Instron . A light load was used to hold the specimen in place while the DIC camera was adjusted. The DIC system developed by Correlated Solutions consisted of one camera with a 35 mm lens mounted to a tripod and two adjustable green LED lights with flexible attachments. The camera and lights were set and adjusted following the manufacturers recommendations [?]. The test setup with the DIC system in place is shown in figure 6.

Several images were taken to obtain an initial reference and to get an estimate of unwanted noise. The load

was then manually incremented while subsequently taking an image at each increment. Care was taken to record the load as close to the point in time in which the image was captured since the specimen would relax with time. Once the threshold of 1000 N was reached the specimen was unloaded. For mixed mode loading the bottom span in the three-point fixture was adjusted to create an asymmetric load. This was achieved by simply moving one support towards the center while the other remained in place. This lead to a gap of four inches from one support to the center on one side and two inches on the other as can be seen in figure ???. Then the same steps for Mode I, as mentioned above, were repeated. Table 1 shows the sequence for each load and the correlated image number for both Mode I and mixed mode testing. These images were finaly analyzed for displacements using an available commercial software Vic-2D from correlated solutions.

2.3 Error and Uncertainties

To minimize noise and induced error best practices were fallowed. For verification an error reported by Vic-2D for each image analyzed with in the correlation was relied on. This reported error is an average measure of confidence margin for the data set [4] and is related to speckle pattern, subset size and quantity, focus, contrast, glare, and F-stop [?]. The software reported an error of 0.006 for all image results in this experiment. Typically an error in the range of 0.02 - 0.05 is considered acceptable and since the above error is much lower the results from the DIC analysis can be considered reliable. Other source of error could arise from time between the pictures taken and time at which the load was recorded. The specimen tended to relax under a constant displacement resulting in a dropping load. To best avoid this type of error one individual was tasked with taking an image and verbally notifying another whom would then read the resulting load to a third individual for recording the image number and load.

3 Results

In total, eight images were recorded and analyzed for each loading scenario. DIC was used to calculate u_x and u_y for each of the eight load increments. Rectangular sub-regions surrounding the crack were selected

for analysis. Signal to Noise Ratio tended to decrease with increased load so analysis was focused on the 1010 N load increment. Equation 3 and Equation 4 were inverted to solve for K_I and experimental displacement fields were used to create contour plots of K_I , shown in Figure 1 and Figure 2. Inverting these equations created singularities as the denominator trended towards 0 at $\theta = \pm\pi$, so values exceeding 10 were eliminated to increase contrast.

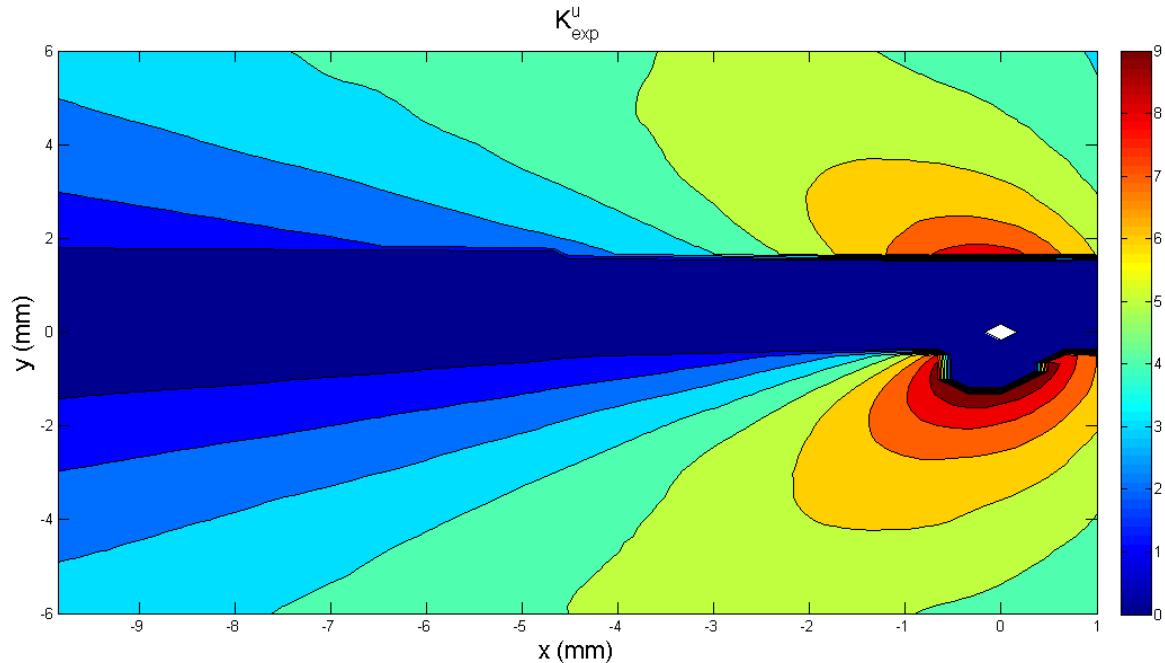


Figure 1: Contour plot of experimental values K_I^U . Values above 10 have been set to 0 for clarity

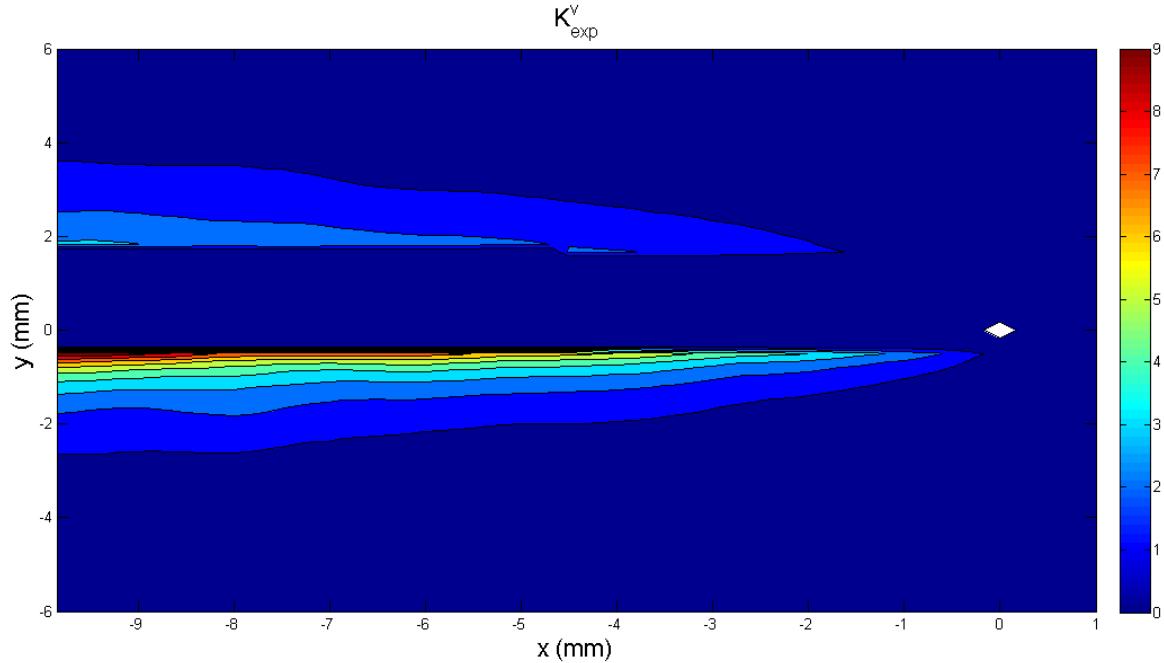


Figure 2: Contour plot of experimental values K_1^V . Values above 10 have been set to 0 for clarity

Experimental K_I values were observed to be highly dependent on location. K_I became less accurate as distance from the crack increased for values calculated from u_x data. The opposite relation was observed for u_y data. To analyze how well theoretical K_I fit the experimental data, two points along the x-axis on the $K_I^{u_x}$ plot were chosen for further analysis. K_I was then calculated for each of the eight load increments. Results are shown in Figure 3. K_I at both locations closely matched predicted values with $R^2 > 0.98$ in both cases.

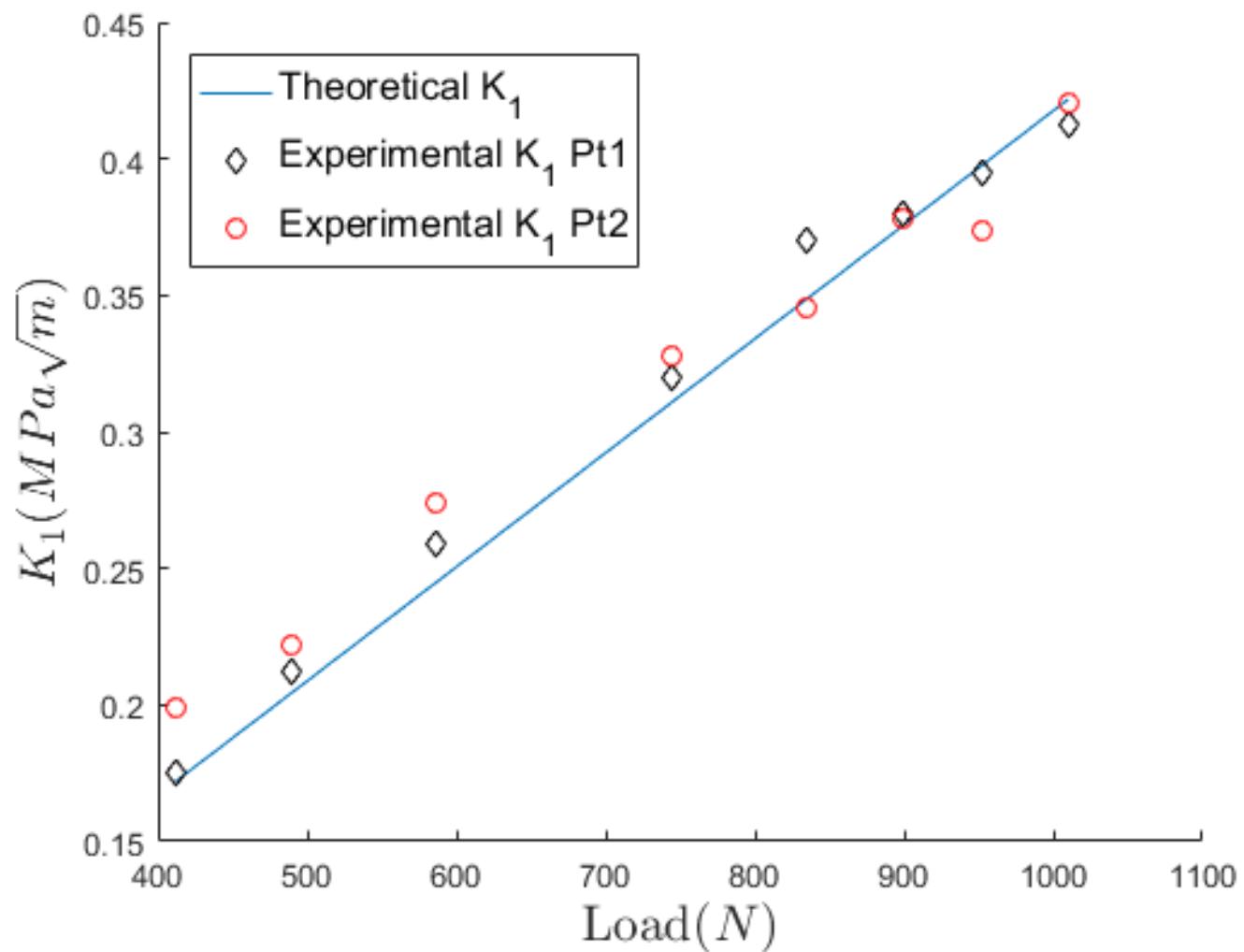


Figure 3: Theoretical and experimental mode I stress intensity factor at two points along the x-axis.

Contour plots of raw u_x and u_y displacement data were drawn and shown in Figure ?? and Figure 5.

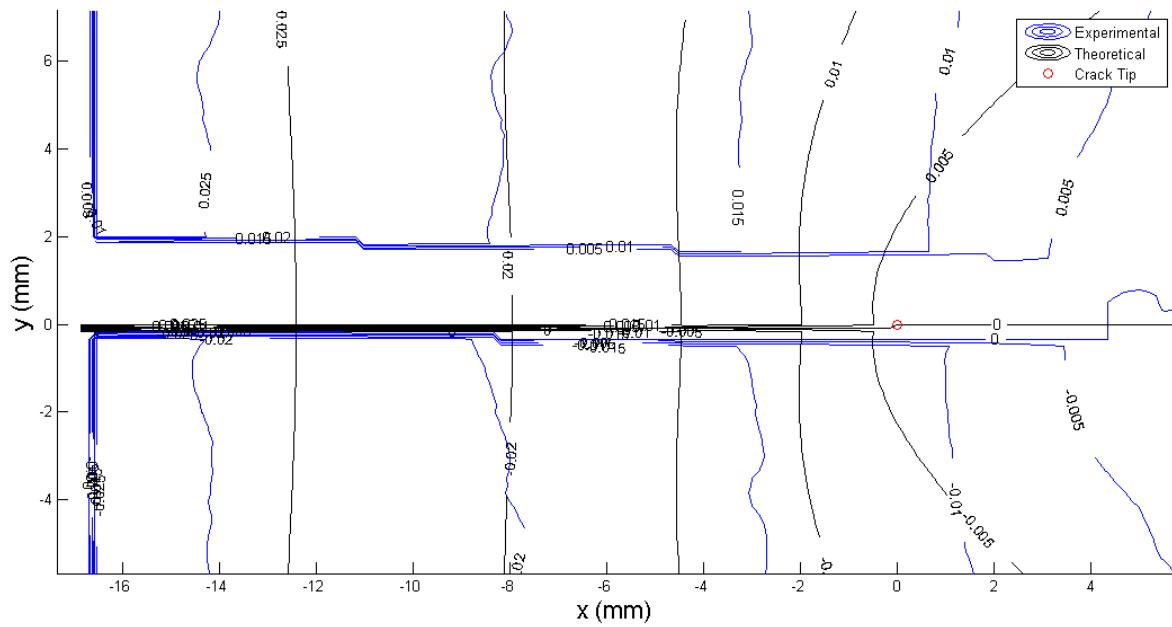


Figure 4: Comparison between theoretical and experimental U displacements

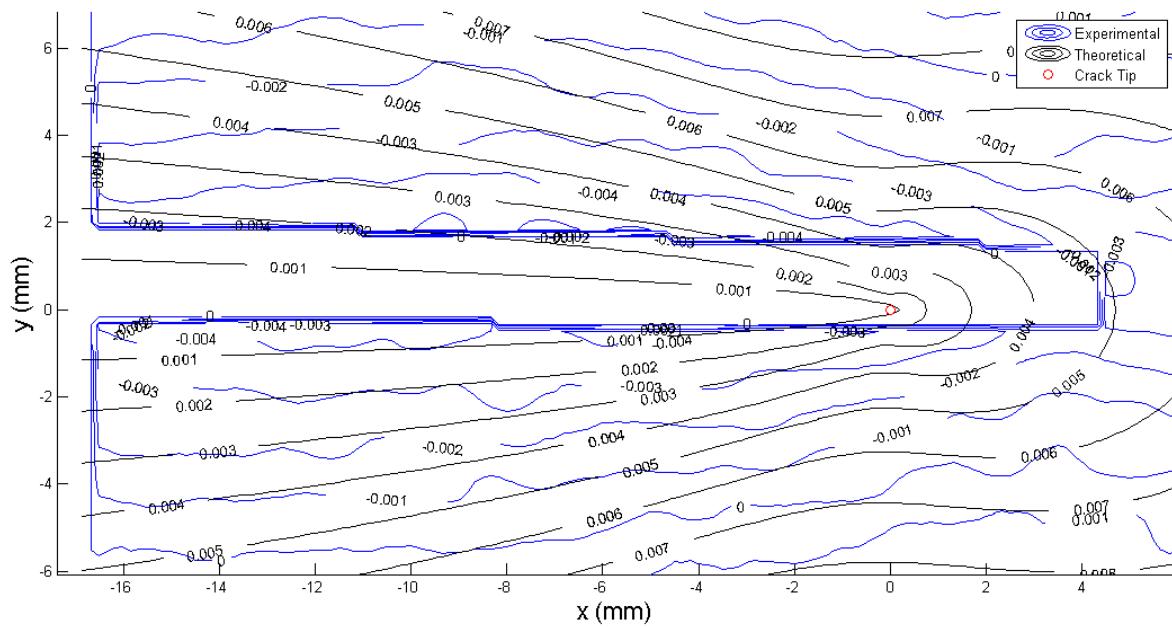


Figure 5: Comparison between theoretical and experimental V displacements

4 Discussion

The general shape of both u_x and u_y fields in Mode I tests matched well with predictions. Field magnitudes also agreed closely, but spacing between contour lines did not always match perfectly. The roughness in experimental contours is likely due to errors inherent in DIC methods such as finite pixel/speckle size and subset selection. There are several possible explanations for disagreements in contour spacing. First, theoretical calculations depend on bulk modulus, μ , which was not measured, but derived from Young's Modulus and Poisson's ratio, taken from on-line sources. Additionally, theoretical calculations assume an infinitely thin crack with a sharp tip. The test specimen crack was cut with a band saw and therefore has a finite width and a blunted tip.

5 Conclusion

test

6 Figures

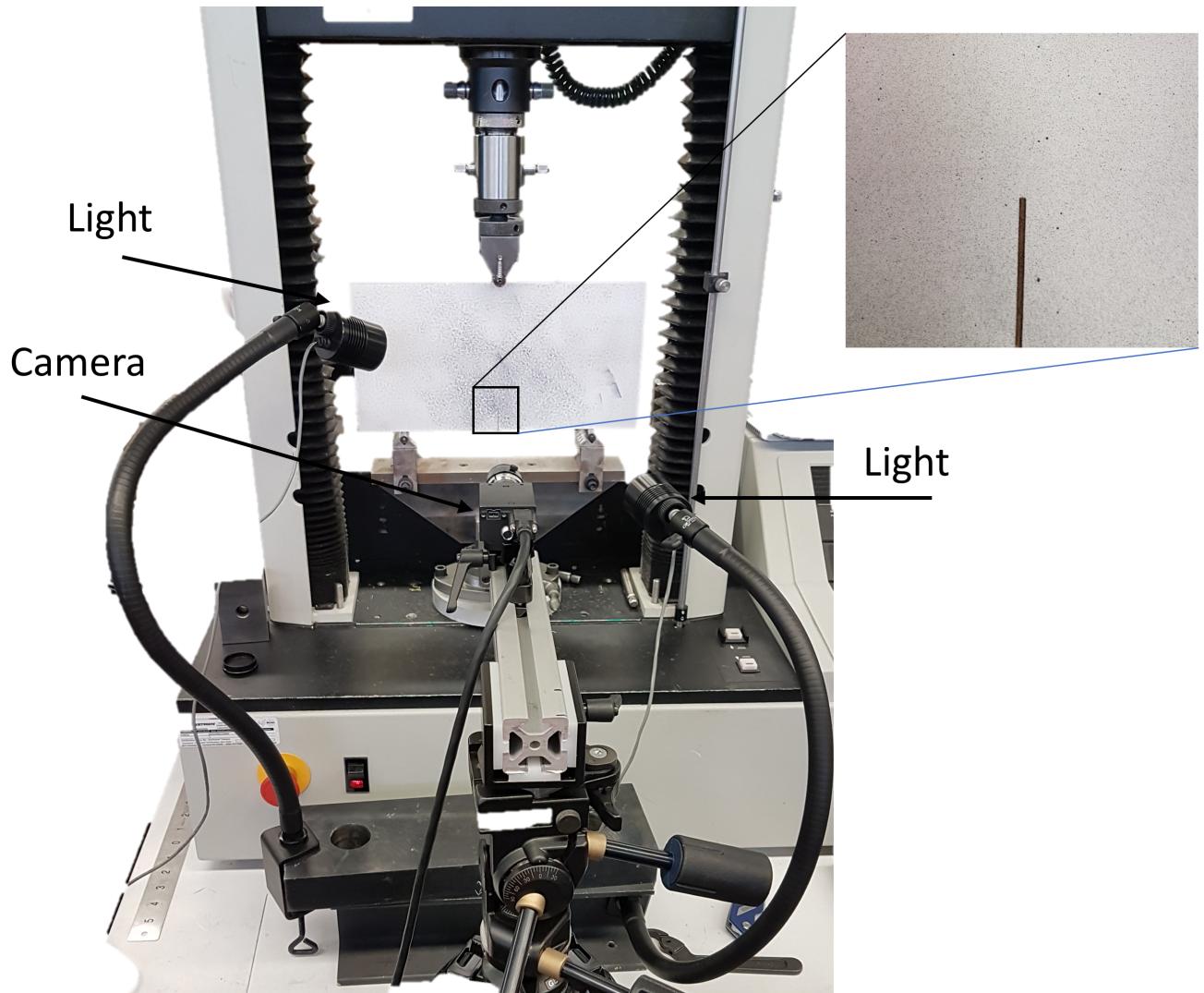


Figure 6: The specimen is mounted to the fixture with the DIC system in place, here the camera is fixed to a tripod and two adjustable lights are seen on either side. A close up of the speckle pattern surrounding the crack is also shown

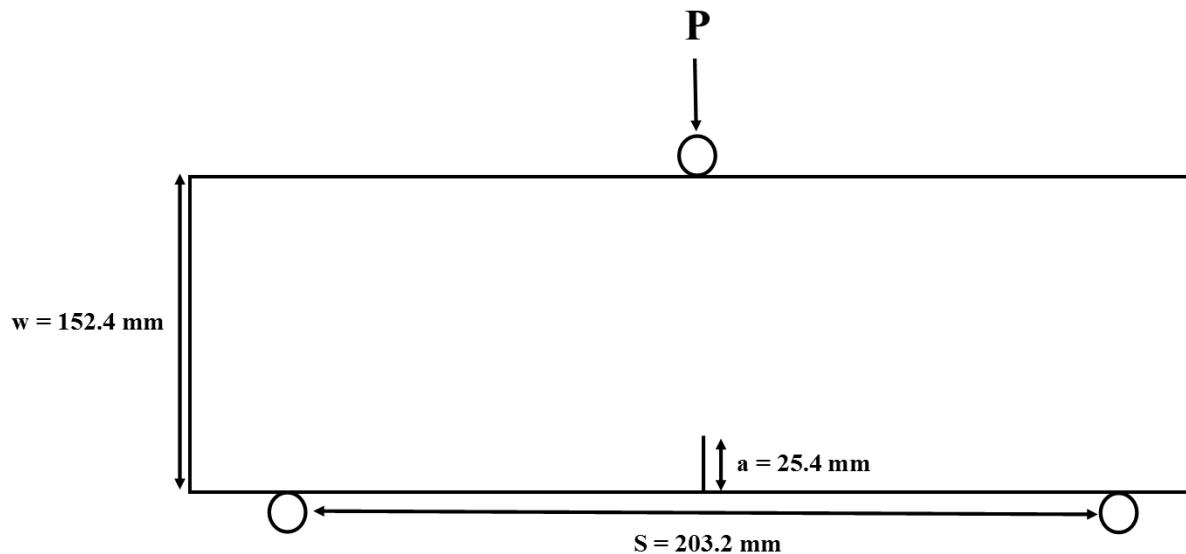


Figure 7: Single Edged Notch Bend specimen, where thickness $B = 8.75 \text{ mm}$. Applied load, P , is applied to the top roller

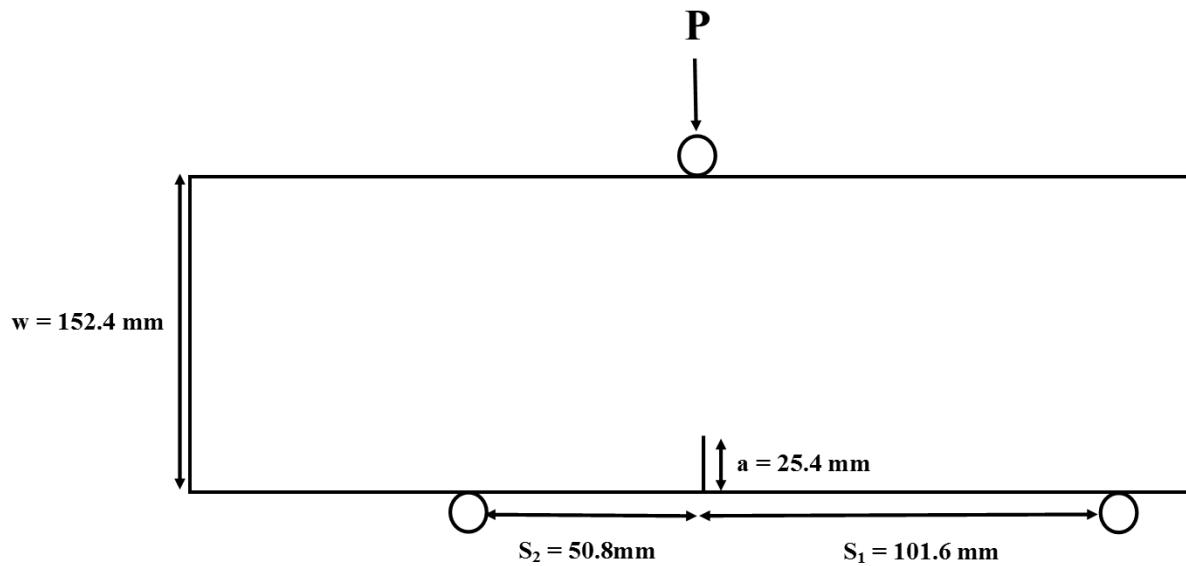


Figure 8: Mixed mode Single Edged Notch Bend specimen, where thickness $B = 8.75 \text{ mm}$. Applied load, P , is applied to the top roller

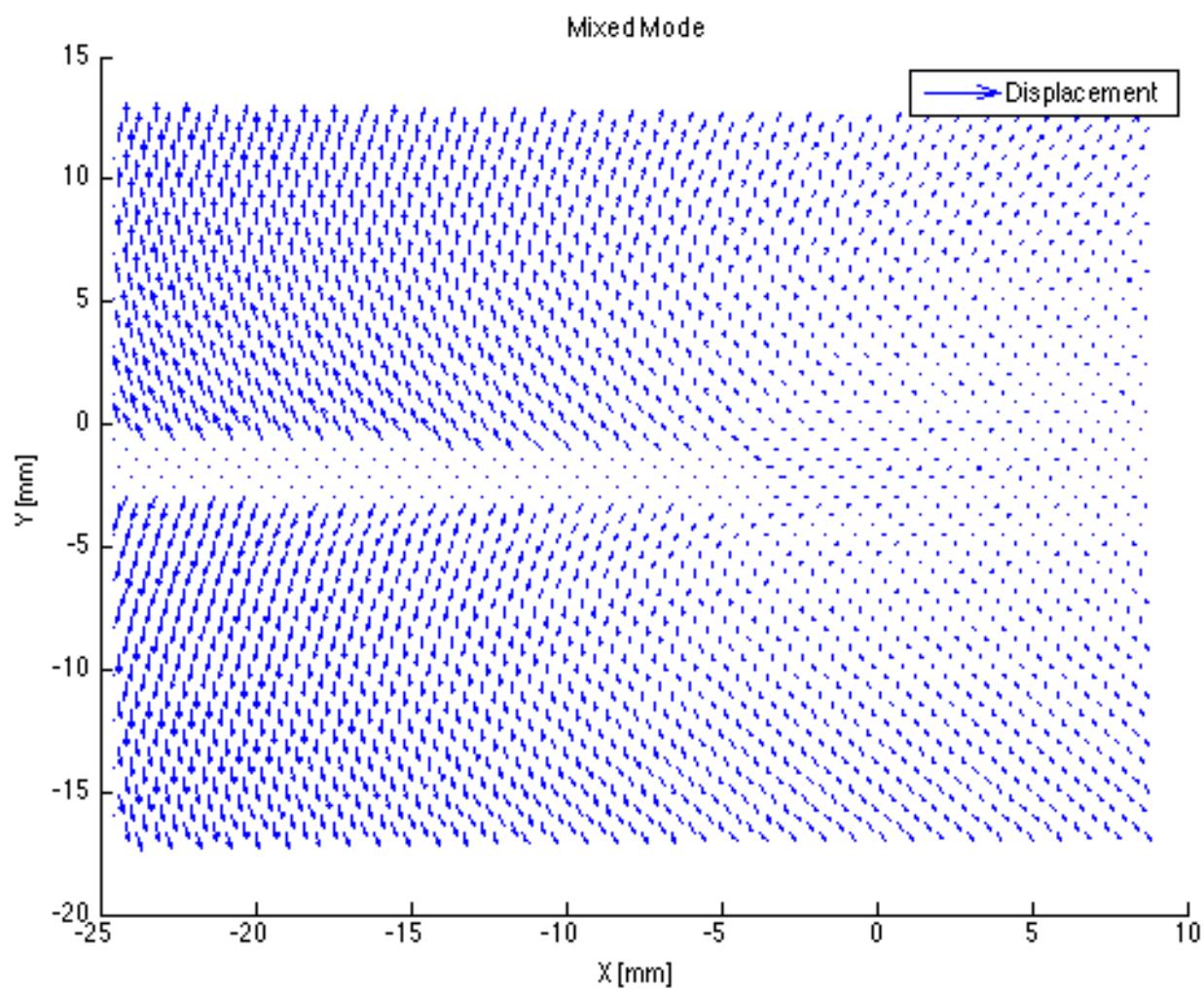


Figure 9: Displacements vectors around crack for an asymmetric load 587 N.

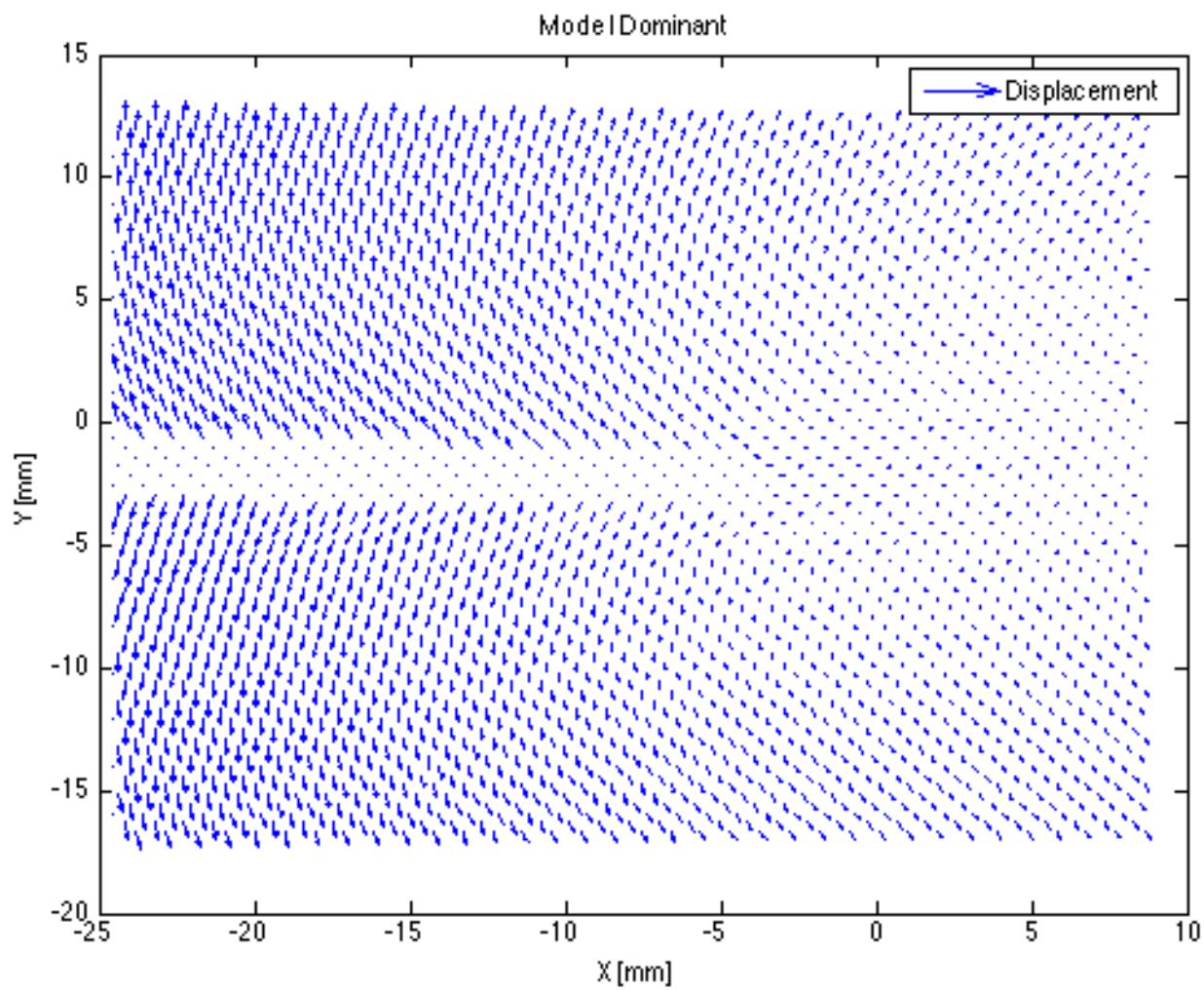


Figure 10: Displacements vectors around crack tip for a symmetric load of 534.7 N

7 Tables

Recoded Experimental Data:

Mode I		Mixed Mode	
Load [N]	Image Number	Load [N]	Image Number
0-5	7.784	0-4	25.58
6	7.784	5	27.58
7	93.77	6	86
8	215.6	7	191.7
9	295	8	324
10	412	9	431
11	489.6	10	486
12	587	11	534.7
13	745	12	629.1
14	834	13	761.7
15	899	14	805.3
16	952	15	849.6
17	1010	16	896
-	-	17	1000

Table 1: Loads and associated image number, first few images were used as reference images.

8 Appendix

8.1 Code

References

- [1] H. W. S. Michael A. Sutton, Jean-Jose Orteu, *Image Correlation for Shape, Motion and Deformation Measurements*. Springer, 2009.
- [2] G. I. H. Tada, P.C. Paris, *The Stress Analysis of Cracks Handbook*, 2nd ed. Paris Productions, 1985.
- [3] T. Anderson, *Fracture Mechanics Fundamentals and Applications*, 3rd ed. CRC Press, 2005.
- [4] *Vic-2D Reference Manual*, Correlated Solutions.