

# Determination of Dynamic Tensile Strength of Concrete Using a Split Hopkinson Pressure Bar

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## **Abstract**

## **1 Introduction**

Accurate predictions of material failure are critical to the design of structures and mechanical devices. In order to make useful predictions of failure, material test conditions must replicate real-world loading scenarios as closely as possible. Standard material tests use application-specific test fixtures and specimen geometries to approximate the expected stress state i.e. tension, compression, shear, or combined loading. An additional, and often overlooked, aspect of the loading condition is the rate at which loads are applied. Many structures and devices are designed to be impacted at high enough speeds that the constitutive response of materials is significantly different from the material's quasi-static behavior. In these applications it is important to quantify a material's failure properties at high strain rates in order to determine if the use of the material is appropriate.

In this report the high strain rate tensile behavior of concrete is investigated. Concrete is a very common material used in large structures such as buildings, bridges, and roadways, all of which may be subject to

high strain rate loading. The compressive and tensile strengths of concrete have been shown to increase at high strain rates. The relationship between strain rate and compressive strength is relatively well established, while only few studies have investigated how the tensile strength of concrete changes with loading rate [1]. A thorough understanding of this relationship may be able to further optimize structural design when high rate loading is expected. In order to load the brittle concrete samples in tension, a Flat Brazil Disc specimen is used.

A major challenge in dynamic material testing is accounting for the inertial effects of a rapidly moving loading apparatus. To overcome this challenge, Kolsky adapted a pressure bar technique originally used by Hopkinson to strike thin material specimens at high speeds[2]. The long and thin bars of a Split-Hopkinson Pressure Bar (SHPB), combined with a thin material specimen, allow strain measurements from the bars to be analyzed with one dimensional wave analysis. This simplification of three dimensional mechanics down to a single dimension allows for the inertia of the striker bar to be accounted for.

An unaltered SHPB generates trapezoidal pulses, with sharp leading and trailing edges. This results in high frequency content in the incident pressure wave that creates inaccuracies in strain measurements and may be undesirable for certain material types. The two most common ways of addressing this issue is the use of pulse shaping, as proposed by Frew et al., and dispersion correction, demonstrated by Follansbee [3] [4]. Both methods were used in this investigation. In this report, the experimental apparatus, test protocol, data reduction, and data analysis methods are described in the methods section. A summary of the measured and reduced data is listed in the results section. The relation of these findings to the current literature are then discussed, and final conclusions are drawn.

## 2 Methods

### 2.1 Experimental Techniques

#### 2.1.1 Split Hopkinson Pressure Bar

The SHPB system is built off of three main parts: the striker bar, the incident bar, and the transmitted bar as shown in Figure 1 [3] [4] [5]. A gas gun was used to accelerate the striker bar into the incident bar and create a one-dimensional compression wave that travels down the incident bar [5]. Upon reaching a test specimen placed at the end of the bar as in Figure 2, the specimen is deformed at a very high strain rate by the compression wave [6]. This deformation imparts a reaction force into the transmitted bar, creating a new compression wave. The remainder of the energy from the incident wave is reflected back along the incident bar as a tensile wave. The overall wavelength of the incident pulse is twice the length of the striker bar. Therefore, in order to fully measure both the incident and reflected pulses without interference between the two, the incident bar must be at least twice the length of the striker bar.

For a SHPB system to produce valid results, the stress wave must only propagate axially. For this assumption to be valid, several criteria must be met. The bars must be made from a homogeneous, isotropic material. Furthermore, the bars must be uniform in cross section, which is achieved through centerless grinding. Finally, initial impact must be kept low enough so that the elastic limit of the bar is not reached. It is also assumed that the stress distribution over the cross section of the bar must be uniform. This is not entirely accurate, but it has been proven that if the length of the bar is at least twenty times the diameter, the results are still acceptable.

One of the limitations of a SHPB system is that with finite bars, dispersion effects must be considered due to variations in the speed different frequencies travel within a medium. Two methods are commonly used to compensate for dispersion. The first is the use of pulse shaping. Pulse shaping uses a sacrificial material placed between the striker bar and the incident bar. An ideal pulse shaper reduces the initial slope of the incident pulse so that it more closely resembles the specimen material response. By

doing this, the high frequency content of the system is decreased and it can be assumed that only a few frequencies dominate the pulse. The second method of compensating for dispersion is through the use of post processing dispersion correction. Measurements are taken by strain gauges in the center of each bar to ensure that the incident and reflected waves are separately recorded. To then march these waveforms forward or backward in time to represent the conditions upon specimen failure, the Fourier series

$$F(n\Delta T) = \frac{A_0}{2} + \sum_{k=1,2,\dots}^N [A_k \cos(k\omega_0 n\Delta T - \phi_{TS}) + B_k \sin(k\omega_0 n\Delta T - \phi_{TS})] \quad (1)$$

is used, where

$$\begin{aligned} \phi_{TS} &= k\omega_0(\Delta x / C_k) \\ A_0 &= \frac{2}{T} \int_0^T f(x) dt \\ A_k &= \frac{2}{T} \int_0^T f(t) \cos(k\omega_0 t) dt \\ B_k &= \frac{2}{T} \int_0^T f(t) \sin(k\omega_0 t) dt \end{aligned}$$

Here,  $T$  is the period of the wave,  $\omega_0$  is the fundamental frequency  $\frac{2\pi}{T}$ ,  $n = 1, 2, \dots, 2N$ , where  $N$  = half the number of recorded data points,  $C_k$  is the speed of a wave at the  $k$ th frequency,  $\Delta T$  is the time step between recording intervals, and  $\Delta x$  is the distance from the strain gauge to the specimen[7]. Once the waves have been propagated so they are at the same point in time, the strain gauge readings can be translated into strain and force values can be determined. Force on the left side of the bar is found through the equation

$$F_1 = A_{Bar} E_{Bar} (\varepsilon_I + \varepsilon_R) \quad (2)$$

and similarly, force on the right side of the bar is

$$F_2 = A_{Bar} E_{Bar} (\varepsilon_T) \quad (3)$$

where  $A_{Bar}$  is the area of the SHPB,  $E_{Bar}$  is the elastic modulus of the SHPB,  $\varepsilon_I$ ,  $\varepsilon_R$ , and  $\varepsilon_T$  are the incident, reflected and transmitted strains respectively. From equilibrium both of these force values should be equal. Finally, from ASTM D3957-08 (cite standard) the tensile strength of the Brazil Disc specimens is found from

$$\sigma_t = \frac{2P}{\pi Dt} \quad (4)$$

where  $P$  is the force applied by one side of the bar,  $t$  is the specimen thickness, and  $D$  is the specimen diameter. Through the application of forces on the specimen geometry, the compression wave from the SHPB will produce a tensile loading to fail the specimen.

### 2.1.2 High Strain Rate Data Acquisition

Data for the SHPB is recorded through strain gauges adhered to the center of each bar. These gauges are arranged in a full bridge configuration which only measures deformations in the axial direction. This further helps satisfy the theoretical requirement for axial wave propagation because due to the difficulty in perfectly aligning the bars, other minute deformations may exist.

In order to avoid aliasing a sample frequency equal to or greater than the Nyquist frequency ( $f_N$ ) must be used [8]. The Nyquist frequency is simply double the highest frequency component of the measured signal. Since high strain rates, as seen in the SHPB, result in high frequency components a very high sample rate is required.

### 2.1.3 Statistical Analysis

Central tendency and dispersion are two common ways to quantify the distribution of a data set [9]. Central tendency is quantified using the measures of mean and median. The mean of a data set is given by

$$\bar{x} = \sum_{i=1}^n \frac{x_i}{n} \quad (5)$$

where  $\bar{x}$  is the mean,  $n$  is the number of data points and  $x_i$  is the  $i$ th data point. The median is the central value of an ordered set of the data. Dispersion represents the distribution of data around the central tendency, usually the mean. Dispersion is measured using standard deviation and variance, given by

$$S_x = \left[ \sum_{i=1}^n \frac{(x_i - \bar{x})^2}{n-1} \right]^{\frac{1}{2}} \quad (6)$$

$$S_x^2 = \sum_{i=1}^n \frac{(x_i - \bar{x})^2}{n-1} \quad (7)$$

where  $S_x$  is the standard deviation and  $S_x^2$  is the variance.

For experiments involving strength of materials due to brittle fracture, a Weibull distribution function can be applied to show the probability of failure at a given strength value [9]. The Weibull Distribution is given by

$$p(x) = 1 - e^{-\left[\frac{(x-x_o)}{b}\right]^m} \text{ for } x > x_o \quad (8)$$

$$p(x) = 0 \text{ for } x < x_o \quad (9)$$

where  $p(x)$  is the probability of failure occurring at  $x$ ,  $x_o$  is the zero strength value of the distribution,  $b$  is scale parameter and  $m$  is the Weibull slope parameter. The values of distribution parameters  $x_o$ ,  $b$  and  $m$  can be determined iteratively or by use of a commercial software such as MATLAB. MATLAB has a built in function, *wblfit*, that generates the Weibull parameters and probability distribution function with a 95% confidence interval [10].

## 2.2 Procedure

Concrete Brazil Disc specimens with a diameter of 19.05 mm and thickness of 6.35 mm were loaded in a SHPB. The SHPB utilized 19.05 mm diameter aluminum 7075-T6 bars for the incident and transmitted bars. The incident bar length was 2.438 m and the transmitted bar length was 1.930 m. Aluminum platens with a matching acoustic impedance were attached to the loading end of the incident and transmitted bars

to prevent damage to the SHPB from the concrete specimens. A 1.058 mm thick, 9.525 mm diameter lead pulse shaper was placed on the non-loading end of the incident bar for each test. The striker bar was propelled using a custom gas gun. The experimental setup can be seen in Figure 2.

Initial calibration was conducted to determine a bar wave speed using a gas gun pressure of 10.2 psi. Specimens were tested at varying gas gun pressures, ranging from 8 to 12.5 psi. Strain gauges were located 1.219 m from the loading point on the incident bar and 0.965 m from the loading point on the transmitted bar. Voltage outputs from the strain gauges were collected using a Tektronix DPO 2004B oscilloscope.

### 2.3 Error and Uncertainties

The main source of uncertainty in the experiment was in the data analysis. It has been shown that the data is very sensitive to the exact value of  $C_0$  as well as in shifting the pulse through time [7]. In our data analysis, the wave speed was determined from a calibration experiment. However, force equilibrium was not satisfied, likely due to errors in shifting the incident and/or reflected waves.

## 3 Results and Discussion

The calculation of dynamic tensile strength assumed force homogenization, therefore only the transmitted wave strain was used. Thirty five concrete Brazil Disc specimens produced a mean dynamic tensile strength of 15.77 MPa, with a median of 15.18 MPa and standard deviation of 3.60 MPa. Additional data was excluded based on incomplete wave forms captured during the experiment.

The dynamic tensile strength was found to fit previous experimental findings and was significantly larger than the quasi-static tensile strength [1]. Published values for the quasi-static tensile strength of concrete range from 2.2 to 4.2 MPa [11]. This matches previous findings that the tensile strength of concrete increases with increased strain rate [1] [12]. Additionally, the data was also found to fit a Weibull distribution with  $x_o = 7.84$  MPa,  $b = 17.18$  MPa and  $m = 4.9951$ . A Weibull Probability Distribution Function plot was

created to verify the Weibull distribution parameters and is presented in Figure 3. A comparison of the gas gun pressure and dynamic tensile strength of concrete showed no relationship, which is visualized in Figure 4.

The pulse shaper as mentioned previously was chosen based on the lower rise of the generated pulse when compared to pulses created using other pulse shapers as shown in Figure 5. By using a pulse shaper a value of 50 was used for  $k$  instead of  $f_N$  as suggested by Li and Lambros [13] to aid in computation time with out loss of accuracy.

## 4 Conclusion

For this report, dynamic evaluation of concrete tensile strength was performed. The previously reported positive correlation between strain rate and ultimate tensile strength was confirmed. However, little correlation between SHPB gas gun pressure was observed. Future examinations of the dynamic behavior of material may wish to additionally measure striker bar velocity in order to further quantify the effects of gas gun pressure.



## 5 Figures

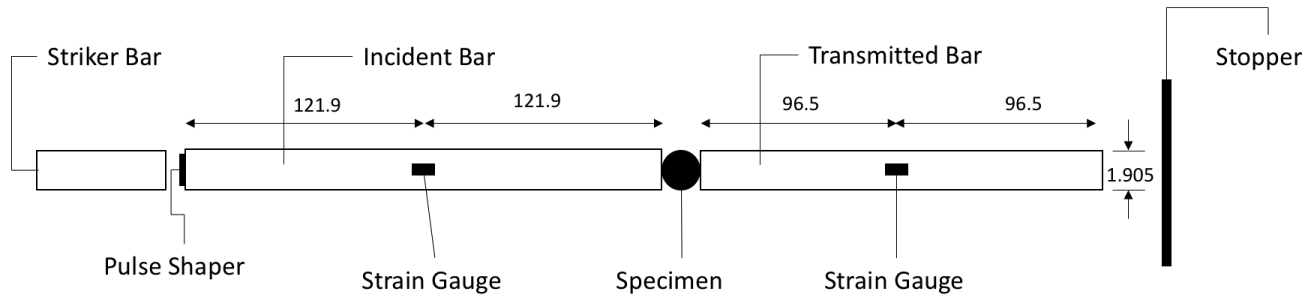


Figure 1: Schematic of SHPB used for this experiment. The strain gauge locations are shown. All dimensions are in centimeters.

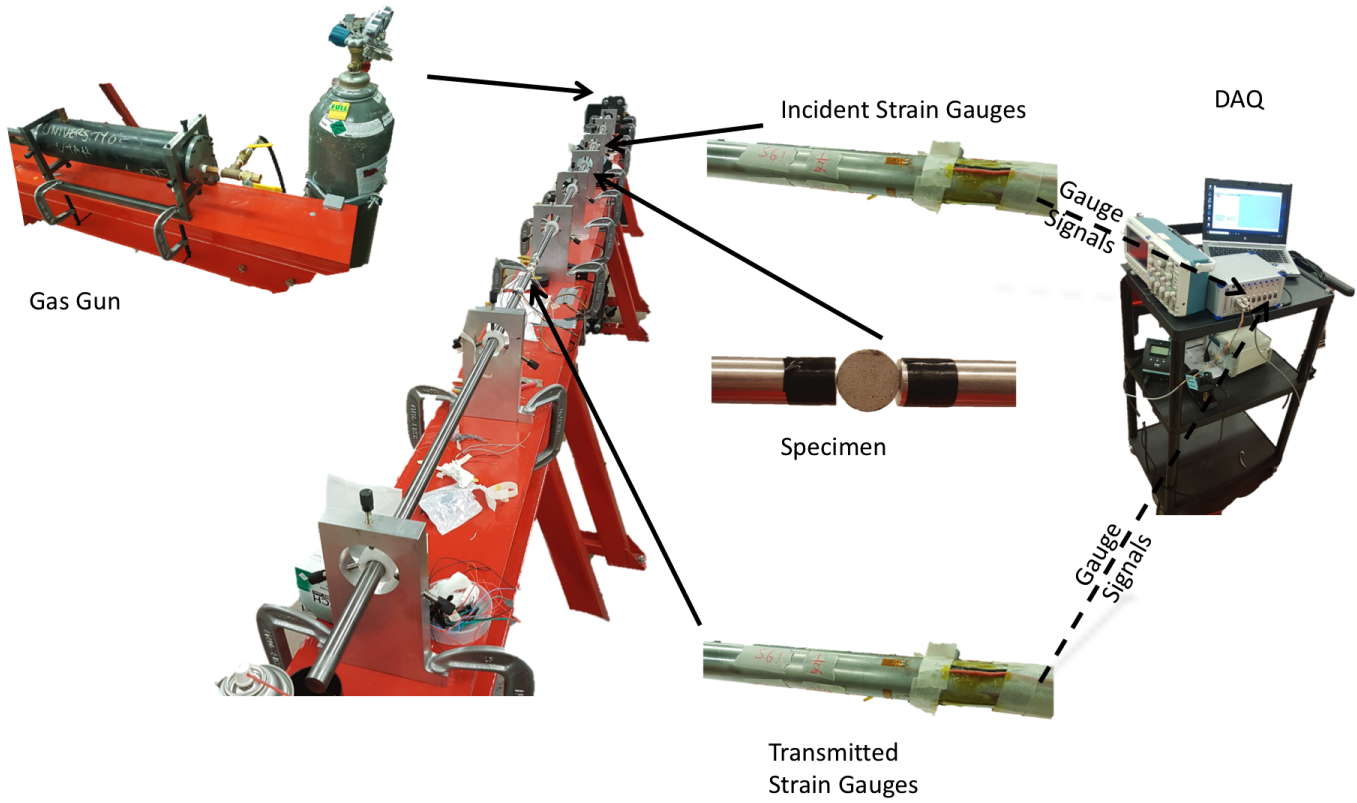


Figure 2: Experimental setup of the Split Hopkinson Pressure Bar

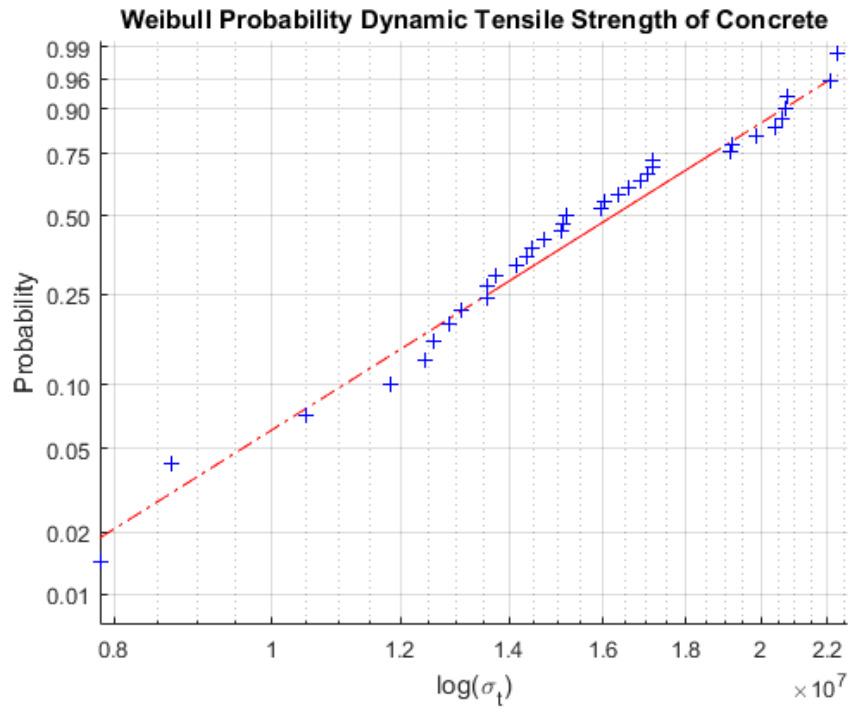


Figure 3: Weibull Probability Distribution Function plot of concrete Brazil Disc specimens

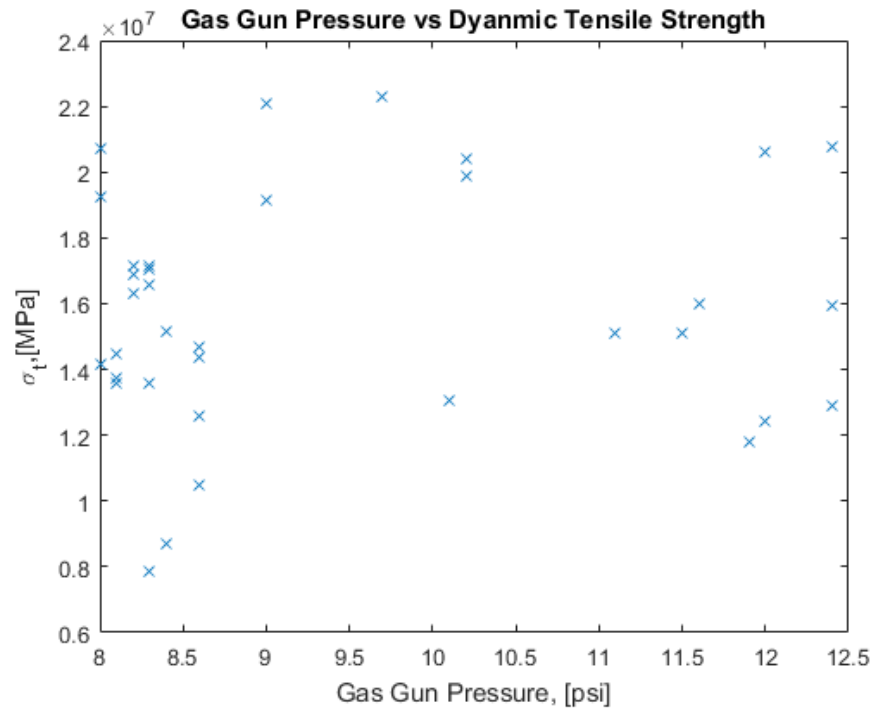


Figure 4: Comparison of Gas Gun Pressure and Calculated Dynamic Tensile Strength of Concrete

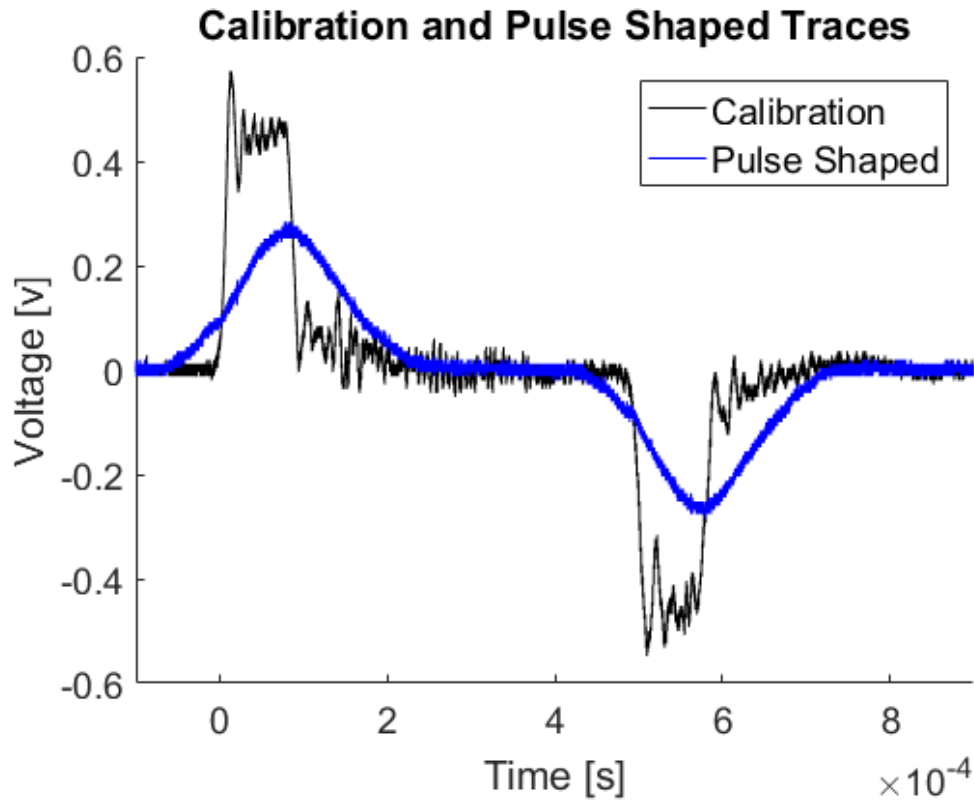


Figure 5: Calibration wave form and comparison of the waveforms generated by use of different pulse shapers with the bars apart.

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