

## [EnergyPlus\_Support] Zone Level Airflows

6 messages

Celia King-Scott <celiakingscott@yahoo.com>

Reply-To: EnergyPlus\_Support@yahoogroups.com

To: "EnergyPlus\_Support@yahoogroups.com" <EnergyPlus\_Support@yahoogroups.com>

Wed, Jun 26, 2013 at 11:17 PM

Hi,

I was wondering if any can help me with an issue I am experiencing in EnergyPlus.

Initially when we looked at the User Design Airflow column under Zone Cooling Loads in the output I saw that the supply air-to-room air temperature was much less than 20F even though the room setpoint was at 75 and supply temperature was 55F. (Following ASHRAE 90.1 2007 Appendix G requirements for Baseline model). We corrected this by adjusting the design outside air at the zone level sizing parameter however now the undercooled hours are about 1000 (much more than allowed). This tells that without taking the outside air into account for zone sizing, there isn't enough airflow to cool the space during the year. The outside air zone sizing parameter isn't used for calculating the cooling capacity, only the airflow, which is what I think is causing difference in temperature differences in the zone. However how do we meet the supply air-to-room air temperature delta T requirement and have less than 300 unmet load hours as also required?

I have attached the Baseline idf and output for your review.

Thank you for your time on this. Any help is greatly appreciated.

Celia

\_\_\_-,\_\_-,\_\_-

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Jim Dirkes <jim@buildingperformanceteam.com>

Reply-To: EnergyPlus\_Support@yahoogroups.com

To: "EnergyPlus\_Support@yahoogroups.com" <EnergyPlus\_Support@yahoogroups.com>

Dear Celia,

The "Central Cooling Design Supply Temperature" must be 11.1C below the zone cooling setpoint in order to achieve your goal. Since you supply the air at 13.3C and serve 20 zones, ALL of the zone setpoints must be ~24.4 — and they are not.

Thu, Jun 27, 2013 at 12:53 AM

Similarly, the zone heating setpoints must be 11.1C lower than, in this case, the Zone Heating Design Supply Air Temperature (35C) in Sizing:Zone – and they are not.

For an ASHRAE 90.1 energy model, each zone <u>on a particular system</u> must use a uniform setpoint for heating or cooling! The fact that the actual design may use somewhat different zone setpoints is not relevant, although I suppose that you are allowed to specify a delta T <u>larger</u> than 11.1C. The purpose of the Baseline – Proposed model is ... the comparison between a "fictitious" Baseline system and an almost-true Proposed system. In order to make the comparison consistent and understandable, some variables (like setpoints and schedules) must be regulated.

I looked very quickly at your IDF and may have missed some things, but I think that you will see an improvement if you change these setpoints.

ОБј15	ОБј16	ОБј17	Оьј18	Оьј19
SuperMarket_HTG SETP_SCH_24/7	SuperMarket_CLG SETP_SCH_24/7	UPC/Pharmacy_CL GSETP_SCH_24/7		Tenant_CLGSETP _SCH_6-10
Temperature	Temperature	Temperature	Temperature	Temperature
Through: 12/31	Through: 12/31	Through: 12/31	Through: 12/31	Through: 12/31
For SummerDesignDay	For WinterDesignDay	For WinterDesignDay	For SummerDesignDay	For WinterDesignDay
Until: 24:00	Until: 24:00	Until: 24:00	Until: 24:00	Until: 24:00
19.44	24.4	22.78	15.56	25.56
For: AllOtherDays	For: AllOtherDays	For: AllOtherDays	For: AllOtherDays	For: AllOtherDays
Until: 6:00	Until: 6:00	Until: 6:00	Until: 6:00	Until: 6:00
19.44	24.4	22.78	15.56	25.56
Until: 22:00	Until: 22:00	Until: 22:00	Until: 22:00	Until: 22:00
19.44	24.4	22.78	19.44	24.4
Until: 24:00	Until: 24:00	Until: 24:00	Until: 24:00	Until: 24:00
19.44	24.4	22.78	15.56	25.56

James V Dirkes II, PE, BEMP, LEED AP

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Energy Analysis, Commissioning & Training Services
1631 Acacia Drive, Grand Rapids, MI 49504 USA
616 450 8653

From: EnergyPlus\_Support@yahoogroups.com [mailto:EnergyPlus\_Support@yahoogroups.com] On Behalf Of Celia King-Scott **Sent:** Wednesday, June 26, 2013 5:17 PM To: EnergyPlus\_Support@yahoogroups.com Subject: [EnergyPlus\_Support] Zone Level Airflows Hi, I was wondering if any can help me with an issue I am experiencing in EnergyPlus. Initially when we looked at the User Design Airflow column under Zone Cooling Loads in the output I saw that the supply air-to-room air temperature was much less than 20F even though the room setpoint was at 75 and supply temperature was 55F. (Following ASHRAE 90.1 2007 Appendix G requirements for Baseline model). We corrected this by adjusting the design outside air at the zone level sizing parameter however now the undercooled hours are about 1000 (much more than allowed). This tells that without taking the outside air into account for zone sizing, there isn't enough airflow to cool the space during the year. The outside air zone sizing parameter isn't used for calculating the cooling capacity, only the airflow, which is what I think is causing difference in temperature differences in the zone. However how do we meet the supply air-to-room air temperature delta T requirement and have less than 300 unmet load hours as also required? I have attached the Baseline idf and output for your review. Thank you for your time on this. Any help is greatly appreciated. Celia

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matthewrlarson@eaton.com <matthewrlarson@eaton.com> Reply-To: EnergyPlus\_Support@yahoogroups.com To: EnergyPlus\_Support@yahoogroups.com Fri, Jun 28, 2013 at 7:31 PM

I've worked on this model some as well and wanted to expand on this discussion. In order to zone the capacities at the zone level, the sizing:zone object was used with the TemperatureDifference Zone Cooling Design Supply Air Temperature Method using 20F dT shown below. You can see this from looking at the HVAC Sizing Summary Report for Zone Cooling under the Calculated dT determined from the Calculated Design Load and Air Flow. The User Design Load is then 1.15 times the Calculated which makes sense since I have a 1.15 sizing factor. The interesting part comes in when looking at the User Design Air Flow. This was increased past the 1.15 sizing factor for a number of the zones which resulted in a lower User Design dT shown in the below table. I determined this was from specifying design outdoor air conditions per person and per square foot. When I removed the design OA conditions, the User Design dT matched the Calculated dT. The problem now is that the total airflow reduced significantly so the undercooled hours is now about 1,000, which tells me that without taking the outside air into account for zone sizing, there isn't enough airflow to cool the space during the year. My takeaway is that the outside air zone sizing parameter isn't used for calculating the cooling capacity, only the airflow, which is what I think is causing difference in temperature differences between the zones. My thinking is that to satisfy the baseline requirements for 90.1, the temperature difference should be specified at 20F which will properly size the cooling capacity and then the airflow can be increased if required based on the OA design specifications. ASHRAE 90.1 App G Section G3.1.2.8 specifies the airflow be based on a 20F dT but I see no other way to do this than let the airflow be adjusted based on design OA even though it may reduce the design temperature difference. I should also note that the required

ventilation air is less than the User Design Air Flow for all the zones. The OA is just being accounted for after the cooling capacity is calculated instead of before. If I am missing anything or if anyone has any thoughts on this, I'd love to get some feedback. Below are some screenshots showing the sizing:zone object and the HVAC Sizing Summary Report for Zone Cooling.

```
Sizing:Zone,
   Sales.
                       !- Zone Name
   TemperatureDifference.
                            !- Zone Cooling Design Supply Air Temperature Method
                         !- Zone Cooling Design Supply Air Temperature {C}
   11.1.
                            !- Zone Cooling Design Supply Air Temperature Difference {C}
   SupplyAirTemperature,
                           !- Zone Heating Design Supply Air Temperature Method
   35.0.
                            !- Zone Heating Design Supply Air Temperature {C}
                     !- Zone Heating Design Supply Air Temperature Difference {C}
                          !- Zone Cooling Design Supply Air Humidity Ratio {kg-H20/kg-air}
   0.0085.
   0.0080.
                          !- Zone Heating Design Supply Air Humidity Ratio {kg-H20/kg-air}
   Sales OA Design,
                          !- Design Specification Outdoor Air Object Name
                          !- Zone Heating Sizing Factor
                          !- Zone Cooling Sizing Factor
                          !- Cooling Design Air Flow Method
   DesignDay,
                          !- Cooling Design Air Flow Rate {m3/s}
                          !- Cooling Minimum Air Flow per Zone Floor Area {m3/s-m2}
                          !- Cooling Minimum Air Flow {m3/s}
                          !- Cooling Minimum Air Flow Fraction
   DesignDay.
                          !- Heating Design Air Flow Method
                          !- Heating Design Air Flow Rate {m3/s}
                          !- Heating Maximum Air Flow per Zone Floor Area {m3/s-m2}
                          !- Heating Maximum Air Flow {m3/s}
                          !- Heating Maximum Air Flow Fraction
                          !- Design Specification Zone Air Distribution Object Name
DesignSpecification:OutdoorAir,
                          !- Name
   Sales OA Design,
                    !- Outdoor Air Method
   Sum.
   0.47194744E-02,
                                        !- Outdoor Air Flow per Person {m3/s-person}
                        !- Outdoor Air Flow per Zone Floor Area {m3/s-m2}
   0.91440004E-03,
                          !- Outdoor Air Flow per Zone {m3/s}
```

REPORT:	HVAC Sizing Summary						
FOR:	Entire Facility						
Zone Coo	ling						
		Calculated		Calculated			
		Design Load	Usor Dosign	Design Air Flow	User Design Air		
		_	User Design Load [Btu/h]	_	_	Calculated dT	Hear Dasign dT
	OPTICALO	[Btu/h]		[ft3/min]	Flow [ft3/min]		User Design dT
	OPTICALS			128.063			14.5104234
	BREAKRO			900.68			14.96978704
	UPC	1574.43		74.586			19.54533403
	BAKERY	34648.97	39846.31	1641.436	1887.651	19.54531254	19.54531398
	FRONTOF	1901.2	2186.38	90.066	133.126	19.54533753	15.20684108
	CUSTOME	2430.31	2794.86	115.132	165.399	19.54527878	15.646003
	STOCKRO	30692.66	35296.56	1454.012	3255.522	19.54531987	10.0389430
	GRVESTIB	15858.72	18237.53	751.28	863.972	19.54530934	19.5453114
	FOODTEN	11897.65	13682.3	563.631	648.176	19.54530995	19.5453029
	HALLWAY	1871.6	2152.34	88.664	199.224	19.5452829	10.00335004
	OPTICALE	2434.15	2799.27	115.314	175.485	19.54526417	14.7700183
	SALES	0	0	0	28158.07	#DIV/0!	(
	FOODTEN	15005.29	17256.09	710.85	817.478	19.54531482	19.5453102
	PHARMAC	3154.97	3628.22	149.462	171.881	19.54522567	19.54528402
	APOFFICE	2964.86	3409.58	140.455	368.575	19.54534008	8.56547112
	GMVESTIE	15079.74	17341.7	714.377	821.533	19.54531322	19.5453251
	MEATUTIL	. 0	0	0	371.052	#DIV/0!	(
	TRAINING	5509.4	6335.81	260.999	300.148	19.54527142	19.5453267
	DELI	51298.15	58992.87	2430.162	2794.687	19.54531716	19.5453114
	BANKTEN	2767.94	3183.13	131.126	150.795	19.54537931	19.5453602

Thanks,

Matthew Larson, PE, CEM, BEMP, LEED AP BD+C Project Engineer

Eaton Energy Solutions, Inc.

Eaton's Electrical Services & Systems 143 Union Blvd, Suite 350 Lakewood, CO 80228

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MatthewRLarson@Eaton.com www.eaton.com/energysolutions



From: EnergyPlus\_Support@yahoogroups.com [mailto:EnergyPlus\_Support@yahoogroups.com] On Behalf Of Jim Dirkes

**Sent:** Wednesday, June 26, 2013 4:54 PM **To:** EnergyPlus\_Support@yahoogroups.com

Subject: RE: [EnergyPlus\_Support] Zone Level Airflows

[Quoted text hidden]

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jeannieboef@gmail.com <jeannieboef@gmail.com>

Reply-To: EnergyPlus\_Support@yahoogroups.com

To: "EnergyPlus\_Support@yahoogroups.com" <EnergyPlus\_Support@yahoogroups.com>

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Sizing:zone object you specify the air temperature and humidity with which the air should enter the zone (it doesn't matter where the air comes from or what amount of it is ODA). This air's energy (enthalpy and moisture) is added to the zones energy and moisture balance equations. In effect the volume airflow to meet the load at specified supply temp is calculated...sizing factor is included.

This is how the load is deturmined for the zone. The equipment is then sized (sizing:system object) to meet this load (air flow at specified temp). It may be specified that the equipment must supply a minimum ODA amount (and the equipment looks up this value from the sizing:zone object). If the value of the minimum ODA flowrate exceeds the total needed for cooling, the ODA rate will be used (100% ODA) and conditioned to the setpoints. The setpoint manager assigned air

Fri, Jun 28, 2013 at 8:27 PM

setpoint temps should be setup to deliver at least the air at the same setpoints as the design air setpoints that you put in the sizing:zone object.

>

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Jim Dirkes <jim@buildingperformanceteam.com>

Reply-To: EnergyPlus\_Support@yahoogroups.com

To: "EnergyPlus\_Support@yahoogroups.com" <EnergyPlus\_Support@yahoogroups.com>

Fri, Jun 28, 2013 at 8:37 PM

In addition to Jean's comments.....

I have not looked into it in detail, but every so often an auto-sized zone will not meet all of the loads (i.e., there will be lot of unmet load hours).

ASHRAE 90.1 anticipates this and allows the modeler to increase the sizing factor as needed to get the unmet loads under control.

If this situation occurs, I check my inputs for the problem zone(s) and if they look OK, I try raising the zone sizing factor. That normally helps.

## James V Dirkes II, PE, BEMP, LEED AP

www.buildingperformanceteam.com

Energy Analysis, Commissioning & Training Services 1631 Acacia Drive, Grand Rapids, MI 49504 USA 616 450 8653

From: EnergyPlus\_Support@yahoogroups.com [mailto:EnergyPlus\_Support@yahoogroups.com] On Behalf Of jeannieboef@gmail.com

Sent: Friday, June 28, 2013 2:28 PM

**To:** EnergyPlus\_Support@yahoogroups.com **Cc:** <EnergyPlus\_Support@yahoogroups.com>

Subject: Re: [EnergyPlus\_Support] Zone Level Airflows

Sizing:zone object you specify the air temperature and humidity with which the air should enter the zone (it doesn't matter where the air comes from or what amount of it is ODA). This air's energy (enthalpy and moisture) is added to the zones energy and moisture balance equations. In effect the volume airflow to meet the load at specified supply temp is calculated...sizing factor is included.

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>

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Jim Dirkes <jim@buildingperformanceteam.com>

Reply-To: EnergyPlus\_Support@yahoogroups.com

To: "EnergyPlus\_Support@yahoogroups.com" <EnergyPlus\_Support@yahoogroups.com>

Fri, Jun 28, 2013 at 9:22 PM

p.s., Just in case it escaped your notice, the "SystemSummary" report provides a very nice summary, zone by zone, of unmet load hours for each month.

> James V Dirkes II, PE, BEMP, LEED AP www.buildingperformanceteam.com **Energy Analysis, Commissioning & Training Services** 1631 Acacia Drive, Grand Rapids, MI 49504 USA 616 450 8653

From: EnergyPlus\_Support@yahoogroups.com [mailto:EnergyPlus\_Support@yahoogroups.com] On Behalf Of Jim Dirkes

Sent: Friday, June 28, 2013 2:38 PM

To: EnergyPlus\_Support@yahoogroups.com

Subject: RE: [EnergyPlus\_Support] Zone Level Airflows

## In addition to Jean's comments.....

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