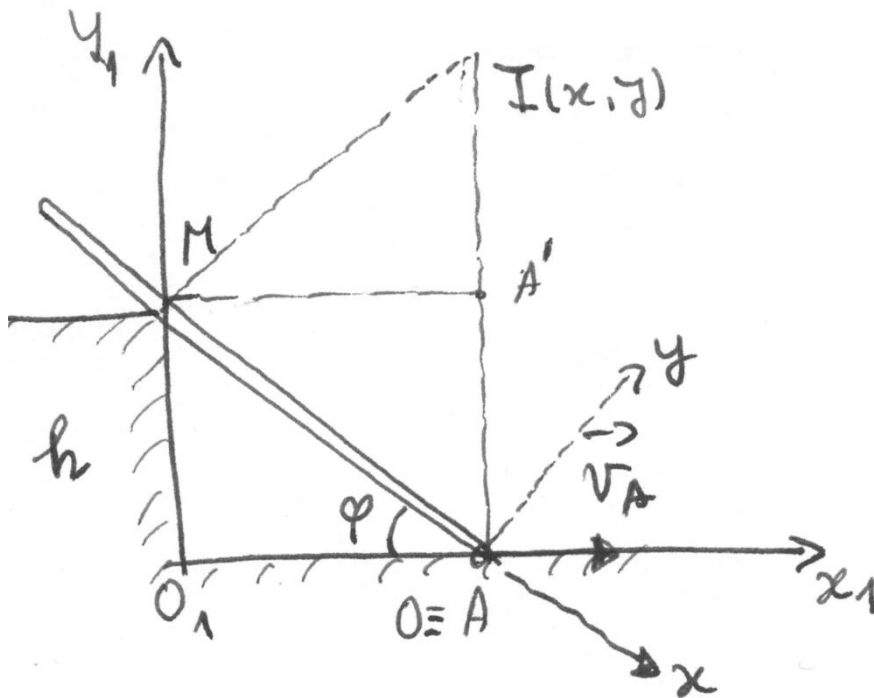


Theoretical Mechanics

THEORETICAL MECHANICS – KINEMATICS (midterm exam – 19.04.2019)

II. On a step of height h is leaning continuously a rigid bar AB . The extremity A of the bar is moving on the horizontal axis O_1x_1 with the velocity v_A (Fig.1). Find the space and body centre and the instantaneous angular velocity of the bar.



$$\begin{cases} x_{10} = h \operatorname{ctg} \varphi \\ y_{10} = 0 \end{cases}$$

Space centre:

$$\begin{cases} x_1 = x_{10} - \frac{dy_{10}}{d\varphi} \\ y_1 = y_{10} + \frac{dx_{10}}{d\varphi} \end{cases}$$

Theoretical Mechanics

$$\begin{cases} x_1 = h \operatorname{ctg} \varphi \\ y_1 = -h \cdot \frac{1}{\sin^2 \varphi} = -h \frac{\cos^2 \varphi + \sin^2 \varphi}{\sin^2 \varphi} = -h(\operatorname{ctg}^2 \varphi + 1) \end{cases}$$

$$\Rightarrow y_1 = -\frac{x_1^2}{h} + h - \text{parabola}$$

Body centre:

$$\begin{cases} x = \frac{dx_{10}}{d\varphi} \sin \varphi - \frac{dy_{10}}{d\varphi} \cos \varphi \\ y = \frac{dx_{10}}{d\varphi} \cos \varphi + \frac{dy_{10}}{d\varphi} \sin \varphi \end{cases}$$

$$\begin{cases} x = \frac{-h}{\sin^2 \varphi} \cdot \sin \varphi \\ y = \frac{-h}{\sin^2 \varphi} \cdot \cos \varphi \end{cases} \Rightarrow \begin{cases} x = -\frac{h}{\sin \varphi} \\ y = -h \frac{\operatorname{ctg} \varphi}{\sin \varphi} \end{cases} \Leftrightarrow \begin{cases} x^2 = h^2 \left(\frac{\cos^2 \varphi + \sin^2 \varphi}{\sin^2 \varphi} \right) \\ y^2 = \frac{h^2 \operatorname{ctg}^2 \varphi}{\sin^2 \varphi} \end{cases}$$

Theoretical Mechanics

$$\Leftrightarrow \begin{cases} x^2 = h^2 (\operatorname{ctg}^2 \varphi + 1) \\ y^2 = h^2 \frac{\operatorname{ctg}^2 \varphi}{\sin^2 \varphi} = h^2 \cdot \frac{\operatorname{ctg}^2 \varphi}{\frac{h^2}{x^2}} = x^2 \operatorname{ctg}^2 \varphi \Rightarrow y^2 = x^2 \left(\frac{x^2}{h^2} - 1 \right) \\ \sin \dot{\varphi} = -\frac{h}{x} \end{cases} \quad \text{(body centre)}$$

$$\vec{v}_A = \vec{\omega} \times \vec{IA} \Rightarrow v_A = |\vec{\omega} \times \vec{IA}| = \omega \cdot IA \Rightarrow \omega = \frac{v_A}{IA} = \frac{v_A \sin^2 \varphi}{h}$$

$$MA^2 = AA' \cdot IA \Rightarrow IA = \frac{MA^2}{AA'} = \frac{h^2}{\sin^2 \varphi}$$

$$\frac{\sin \dot{\varphi}}{2} = \frac{MA}{IA} \Rightarrow MA = \frac{h}{\sin \dot{\varphi}}$$

Theoretical Mechanics

III. Consider a rigid body. In the mobile frame $Oxyz$ three points of the body $O(0,0,0)$, $A(1,1,0)$ and $B(1,1,1)$ have the velocities $\vec{v}_O(2,1,-3)$, $\vec{v}_A(0,3,-1)$ and $\vec{v}_B(-1,2,-1)$. Find the equations of the instantaneous helical axis, the translation velocity \vec{v}_{tr} and the angular velocity $\vec{\omega}$.

Let be $\vec{\omega}(p, q, r)$ $\vec{v}_A = \vec{v}_O + \vec{\omega} \times \vec{r}_A$

$$(0, 3, -1) = (2, 1, -3) + \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ p & q & r \\ 1 & 1 & 0 \end{vmatrix} = (2, 1, -3) + (-r, r, p-q) \Rightarrow \begin{cases} r=2 \\ p-q=2 \end{cases}$$

$$\vec{v}_B = \vec{v}_O + \vec{\omega} \times \vec{r}_B$$

$$(-1, 2, -1) = (2, 1, -3) + \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ p & q & r \\ 1 & 1 & 1 \end{vmatrix} = (2, 1, -3) + (q-r, r-p, p-q) \Rightarrow \begin{cases} q-r=-3 \\ r-p=1 \\ p-q=2 \end{cases}$$

Theoretical Mechanics

Thus $p=1, r=2, q=-1 \Rightarrow \vec{\omega} (1, -1, 2) \Rightarrow \omega = \sqrt{6}$

instantaneous helical axis:

$$\frac{v_{0x} + qz + ry}{p} = \frac{v_{0y} + rx - pz}{q} = \frac{v_{0z} + py - qx}{r}$$

$$\frac{2 - z - 2y}{1} = \frac{1 + 2x - z}{-1} = \frac{-3 + y + z}{2}$$

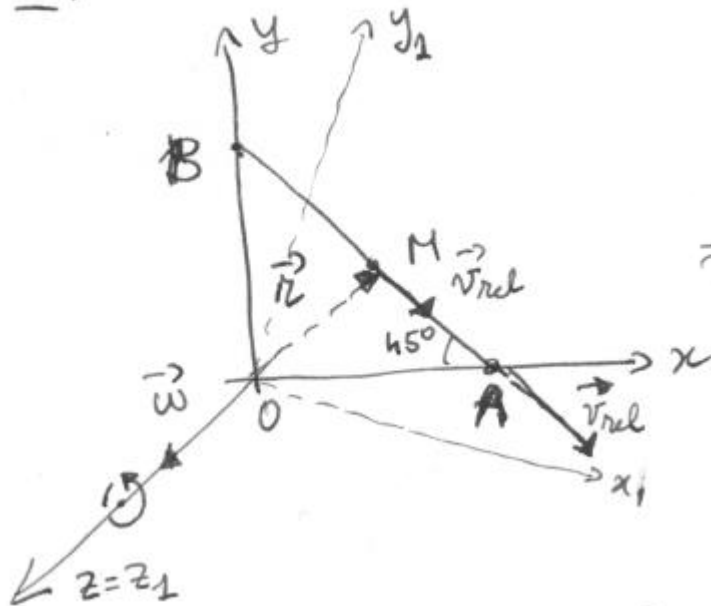
Translation velocity :

$$v_{tr} = \vec{v}_0 \cdot \frac{\vec{\omega}}{\omega} = (2, 1, -3) \left(\frac{1}{\sqrt{6}}, -\frac{1}{\sqrt{6}}, \frac{2}{\sqrt{6}} \right) = -\frac{5}{\sqrt{6}}$$

Theoretical Mechanics

IV. An isosceles right triangle OAB , $m(\hat{O}) = \frac{\pi}{2}$, rotates in his plane about the fixed point O with the angular velocity $\omega = \text{const}$. A material point M moves uniformly along the side $AB = c$ (from B to A) with the speed $v_M = \frac{c\omega}{2\pi}$. Find the absolute velocity and acceleration of M when it reaches the point A .

IV.



$$v_{rel} = \frac{c \cdot \omega}{2\pi}$$

$$\vec{v}_{rel} = \frac{c\omega}{2\pi} \cdot \frac{\sqrt{2}}{2} \cdot \vec{x} - \frac{c\omega\sqrt{2}}{2\pi} \cdot \vec{y}$$

$$\vec{v}_{transp} = \vec{\omega} \times \vec{OA} = \omega \cdot \vec{k} \times \frac{c\sqrt{2}}{2} \vec{x} = c\omega \cdot \frac{\sqrt{2}}{2} \cdot \vec{y}$$

$$\vec{v}_M|_{M=A} = \vec{v}_{transp} + \vec{v}_{rel}$$

$$= \frac{c\omega}{2\pi} \frac{\sqrt{2}}{2} \cdot \vec{x} + c\omega \cdot \frac{\sqrt{2}}{2} \left(1 - \frac{1}{2\pi}\right) \cdot \vec{y}$$

$$\vec{a}_{transp} = \vec{\omega} \times (\vec{\omega} \times \vec{OA}) = -\omega^2 \cdot \vec{OA} = -\omega^2 c \frac{\sqrt{2}}{2} \cdot \vec{x}$$

Theoretical Mechanics

$$\vec{a}_{rel} = \frac{\partial \vec{v}_{rel}}{\partial t} = 0$$

$$\vec{a}_{Coriolis} = 2 \vec{\omega} \times \vec{v}_{rel} = \cancel{2} \omega \cdot \frac{c\omega}{\cancel{2}\pi} \frac{\sqrt{2}}{2} \cdot \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 0 & 0 & 1 \\ 1 & -1 & 0 \end{vmatrix} = \frac{c\omega^2}{\pi} \cdot \frac{\sqrt{2}}{2} (\vec{i} + \vec{j})$$

$$\vec{a}_M = \vec{a}_{transport} + \vec{a}_{rel} + \vec{a}_{Coriolis} = \frac{c\omega^2\sqrt{2}}{2} \left(\frac{1}{\pi} - 1 \right) \cdot \vec{i} + \frac{c\omega^2}{\pi} \cdot \frac{\sqrt{2}}{2} \cdot \vec{j}$$