

Gradient-based shape optimization

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1 Problem placement

Given is a *Boundary Value Problem* (BVP) which governs a primal unknown field $\mathbf{u} \in \mathbf{V}$ and which in residual form writes,

$$\mathbf{R}(\mathbf{u}) = \mathbf{0} . \quad (1)$$

Exemplarily, in case of linear elasticity the aforementioned residual form is,

$$\mathbf{R}(\mathbf{u}) = \nabla \cdot \boldsymbol{\sigma} + \mathbf{b} \quad \text{in } \Omega , \quad (2a)$$

$$\boldsymbol{\sigma} = \mathbf{C} : \boldsymbol{\varepsilon} \quad \text{in } \Omega , \quad (2b)$$

$$\boldsymbol{\varepsilon} = \frac{1}{2} \left(\nabla \mathbf{u} + \nabla^T \mathbf{u} \right) \quad \text{in } \Omega , \quad (2c)$$

$$\mathbf{u} = \bar{\mathbf{u}} \quad \text{on } \Gamma_d , \quad (2d)$$

$$\boldsymbol{\sigma} \cdot \mathbf{n} = \bar{\mathbf{t}} \quad \text{on } \Gamma_n . \quad (2e)$$

Moreover, given is an objective function $\mathcal{J}(\mathbf{s}, \mathbf{u})$ where $\mathbf{s} \in \mathcal{S}$ represents a continuous design variable and \mathbf{u} the primal solution of problem in Eq. (2). The task is to minimize \mathcal{J} with respect to the design variables \mathbf{s} subject to the residual equation in Eq. (1), namely,

$$\min_{\bar{\mathbf{s}} \in \mathcal{S}} \mathcal{J}(\bar{\mathbf{s}}, \mathbf{u}) \text{ subject to } \mathbf{R}(\mathbf{s}, \mathbf{u}) = \mathbf{0} \text{ in } \Omega(\mathbf{s}) . \quad (3)$$

Problem in Eq. (3) can be formulated as the minimization of the following augmented by Lagrange Multipliers potential,

$$\mathcal{L}(\mathbf{s}, \mathbf{u}, \boldsymbol{\lambda}) = \mathcal{J}(\mathbf{s}, \mathbf{u}) + \int_{\Omega(\mathbf{s})} \boldsymbol{\lambda} \cdot \mathbf{R}(\mathbf{s}, \mathbf{u}) \, d\Omega . \quad (4)$$

The stationary point of potential in Eq. (4) is sought for which,

$$\delta \mathcal{L} = \delta \mathcal{J} + \int_{\Omega(\mathbf{s})} \delta \boldsymbol{\lambda} \cdot \mathbf{R} + \boldsymbol{\lambda} \cdot \frac{\partial \mathbf{R}}{\partial \mathbf{s}} \cdot \delta \mathbf{s} + \frac{\partial \mathbf{R}}{\partial \mathbf{u}} \cdot \delta \mathbf{u} \, d\Omega . \quad (5)$$

Taking advantage of the fact that $\mathbf{R}(\mathbf{s}, \mathbf{u}) = \mathbf{0}$ in $\Omega(\mathbf{s})$, Eq. (5) simplifies to,

$$\delta \mathcal{L} = \delta \mathcal{J} + \int_{\Omega(\mathbf{s})} \boldsymbol{\lambda} \cdot \frac{\partial \mathbf{R}}{\partial \mathbf{s}} \cdot \delta \mathbf{s} + \frac{\partial \mathbf{R}}{\partial \mathbf{u}} \cdot \delta \mathbf{u} \, d\Omega . \quad (6)$$

2 Shape optimization in linear elasticity

Herein assumed is that the residual \mathbf{R} is given by the discrete problem meaning,

$$\mathbf{R}(\hat{\mathbf{s}}, \hat{\mathbf{u}}) = \mathbf{K}(\hat{\mathbf{s}})\hat{\mathbf{u}} - \mathbf{F} , \quad (7)$$

where $\hat{\mathbf{s}}$ and $\hat{\mathbf{u}}$ stand for the discrete design variables $\hat{\mathbf{s}} = [\hat{s}_0 \cdots \hat{s}_i \cdots \hat{s}_n]$ and the discrete primal solution field $\hat{\mathbf{u}} = [\hat{u}_0 \cdots \hat{u}_i \cdots \hat{u}_n]$, n being the number of *Degrees of Freedom* (DOFs). Moreover concerning the variables appearing in Eq. (7),

$$\mathbf{K}(\hat{\mathbf{s}}) = \int_{\Omega_h(\hat{\mathbf{s}})} \mathbf{B}(\hat{\mathbf{s}})^T \mathbf{C} \mathbf{B}(\hat{\mathbf{s}}) d\Omega , \quad (8a)$$

$$\mathbf{F} = \mathbf{N}(\mathbf{X}_0)^T \mathbf{f} . \quad (8b)$$

$\Omega_h(\hat{\mathbf{s}})$ being the discretized domain of the problem. In the discrete case, $\hat{\mathbf{s}}$ stands for the coordinates of the nodes in the provided mesh $\Omega_h(\hat{\mathbf{s}})$. In Eq. (8), $\mathbf{B}(\hat{\mathbf{s}})$ stands for the B-operator matrix of the problem in Eq. (2), $\mathbf{N}^T(\mathbf{X}_0)$ for the matrix of the basis functions corresponding to the discretization evaluated at \mathbf{X}_0 and \mathbf{f} the constant load which is applied at \mathbf{X}_0 . Accordingly, the objective function is defined by the strain-energy meaning that,

$$\mathcal{J}(\hat{\mathbf{s}}, \hat{\mathbf{u}}) = -\frac{1}{2} \hat{\mathbf{u}}^T \mathbf{K}(\hat{\mathbf{s}}) \hat{\mathbf{u}} = -\frac{1}{2} \hat{\mathbf{u}}^T \mathbf{F} , \quad (9)$$

which has to be minimized so that the strain-energy is maximized. In this way, the augmented potential in Eq. (4) becomes,

$$\mathcal{L}(\hat{\mathbf{s}}, \hat{\mathbf{u}}, \bar{\boldsymbol{\lambda}}) = \mathcal{J}(\hat{\mathbf{s}}, \hat{\mathbf{u}}) + \bar{\boldsymbol{\lambda}}^T \mathbf{R}(\hat{\mathbf{s}}, \hat{\mathbf{u}}) , \quad (10)$$

where $\bar{\boldsymbol{\lambda}}$ stands for the vector of the discrete ajoint variables (Lagrange Multipliers) in this case. The first variation of the augmented potential in Eq. (6) becomes,

$$\delta \mathcal{L} = \left(\frac{\partial \mathcal{J}}{\partial \hat{\mathbf{s}}} + \bar{\boldsymbol{\lambda}}^T \frac{\partial \mathbf{R}}{\partial \hat{\mathbf{s}}} \right) \delta \hat{\mathbf{s}} + \left(\frac{\partial \mathcal{J}}{\partial \hat{\mathbf{u}}} + \bar{\boldsymbol{\lambda}}^T \frac{\partial \mathbf{R}}{\partial \hat{\mathbf{u}}} \right) \delta \hat{\mathbf{u}} . \quad (11)$$

To simplify the problem, the discrete vector of the Lagrange Multipliers $\bar{\boldsymbol{\lambda}}$ is chosen such that the following condition holds,

$$\frac{\partial \mathcal{J}}{\partial \hat{\mathbf{u}}} + \bar{\boldsymbol{\lambda}}^T \frac{\partial \mathbf{R}}{\partial \hat{\mathbf{u}}} = \mathbf{0} , \quad (12)$$

since the vector of the Lagrange Multipliers $\bar{\boldsymbol{\lambda}}$ can be freely selected. Eq. (12) can be alternatively written as,

$$\left(\frac{\partial \mathbf{R}}{\partial \hat{\mathbf{u}}} \right)^T \hat{\boldsymbol{\lambda}} = - \left(\frac{\partial \mathcal{J}}{\partial \hat{\mathbf{u}}} \right)^T . \quad (13)$$

For linear elastic problems, relation in Eq. (7) results into,

$$\left(\frac{\partial \mathbf{R}}{\partial \hat{\mathbf{u}}} \right)^T = \mathbf{K}(\hat{\mathbf{s}})^T = \mathbf{K}(\hat{\mathbf{s}}) , \quad (14)$$

and by substituting Eq. (14) into Eq. (13) given that $\mathbf{F} = \mathbf{K}(\hat{\mathbf{s}})\hat{\mathbf{u}}$ from Eq. (7) and given that $\partial\mathcal{J}/\partial\hat{\mathbf{u}} = -1/2\mathbf{F}$ from Eq. (9), Eq. (13) results in,

$$\bar{\boldsymbol{\lambda}} = \frac{1}{2}\hat{\mathbf{u}} , \quad (15)$$

concerning the computation of the discrete adjoint variables $\bar{\boldsymbol{\lambda}}$. Substituting Eq. (15) into Eq. (11) it results in,

$$\delta\mathcal{L} = \left(\frac{\partial\mathcal{J}}{\partial\hat{\mathbf{s}}} + \frac{1}{2}\hat{\mathbf{u}}^T \frac{\partial\mathbf{R}}{\partial\hat{\mathbf{s}}} \right) \delta\hat{\mathbf{s}} . \quad (16)$$

Using Eq. (7) it can be seen that,

$$\frac{\partial\mathbf{R}}{\partial\hat{\mathbf{s}}} = \frac{\partial\mathbf{K}}{\partial\hat{\mathbf{s}}}\hat{\mathbf{u}} , \quad (17)$$

provided that the load vector \mathbf{F} does not depend on the design variables. Substituting Eq. (17) into Eq. (16) results in,

$$\delta\mathcal{L} = \overbrace{\left(\frac{\partial\mathcal{J}}{\partial\hat{\mathbf{s}}} + \frac{1}{2}\hat{\mathbf{u}}^T \frac{\partial\mathbf{K}}{\partial\hat{\mathbf{s}}}\hat{\mathbf{u}} \right)}^{\text{sensitivity field } \mathbf{S}} \delta\hat{\mathbf{s}} . \quad (18)$$

Since the objective function can be defined either way in Eq. (9) using the last equality one has that $\partial\mathcal{J}/\partial\hat{\mathbf{s}} = -\hat{\mathbf{u}}^T\partial\mathbf{F}/\partial\hat{\mathbf{s}} = \mathbf{0}$ and thus the sensitivity vector is given by,

$$\mathbf{S} = \frac{1}{2}\hat{\mathbf{u}}^T \frac{\partial\mathbf{K}}{\partial\hat{\mathbf{s}}}\hat{\mathbf{u}} . \quad (19)$$

3 Sensitivity analysis

Concerning the computation of the sensitivities defined in Eq. (19) one can either compute them analytically using relation in Eq. (8a) or evaluate them using a finite differencing scheme. Herein the second option is preferred since it is more general and does not require lengthy derivations. Within the finite differencing approach the derivative $\partial\mathbf{K}/\partial\hat{\mathbf{s}}$ in Eq. (19) is approximated as,

$$\frac{\partial\mathbf{K}}{\partial\hat{s}_i} = \frac{1}{d_i} (\mathbf{K}(s_i + d_i) - \mathbf{K}(s_i)) , \quad (20)$$

for each design variable s_i in \mathbf{s} where d_i stands for the perturbation of design variable s_i . Note that $d_i \neq d_j$ for $i \neq j$ with $i, j = 1, \dots, n$ and those perturbation are chosen based on the corresponding element size. However, in this work the perturbations are all chosen the same, that is $d_1 = \dots = d_n = d$ and this value may be scaled with a small number $\epsilon \approx 1e-4$.

4 Gradient-based shape optimization

Within the gradient-based shape optimization a set of optimization iterations is formulated within which we are moving in the negative gradient direction (which is in this case is the sensitivity vector) in the design space \mathcal{S} . In this was, at each optimization iteration the design variables are updated as,

$$\mathbf{s}^{\hat{i}} = \mathbf{s}^{\hat{i}} - \alpha^{\hat{i}} \mathbf{S}^{\hat{i}}, \quad (21)$$

where $\alpha^{\hat{i}}$ is a small number in the neighbourhood of $\epsilon = 1e-3$ which weights the sensitivities $\mathbf{S}^{\hat{i}}$ at the \hat{i} -th optimization iteration. Typically $\alpha^{\hat{i}}$ adapts within the optimization iterations but herein we chose it constant, that is $\alpha^{\hat{i}} = \alpha$ for all \hat{i} . The algorithm for the aforementioned gradient-based shape optimization writes,

Algorithm 1: Gradient-based shape optimization

Data: Define perturbation d , sensitivity weight α , maximum number of iterations n_{\max} and a tolerance ϵ

Result: Shape optimal design

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1 Initialization  $\hat{i} = 1$ ;
2 while  $\hat{i} \leq n_{\max}$  do
3   Compute the unperturbed stiffness matrix  $\mathbf{K}(\hat{\mathbf{s}}^{\hat{i}})$  using Eq.(8a);
4   Solve for the primal field  $\mathbf{u}^{\hat{i}}$  using Eq. (7);
5   Compute the ojective function  $\mathcal{J}$  using Eq. (9);
6   if  $|\mathcal{J}| \leq \epsilon$  then
7     | break; // Convergence has been achieved
8   end
9   for all design variables  $\hat{s}_i^{\hat{i}}$  do
10    Compute the perturbed stiffness matrix  $\mathbf{K}(\hat{\mathbf{s}}^{\hat{i}} + d_i^{\hat{i}})$  with Eq. (8a);
11    Compute the derivative  $\partial \mathbf{K} / \partial \hat{s}_i^{\hat{i}}$  using Eq. (20);
12    Compute the corresponding sensitivity  $S_i$  using Eq. (19);
13    Recover the unperturbed stiffness matrix  $\mathbf{K}(\hat{\mathbf{s}}^{\hat{i}})$ ;
14  end
15  Update the design variables using Eq. (21);
16   $\hat{i} = \hat{i} + 1$ ;
17 end
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