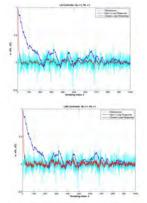
Linear-Quadratic-Gaussian Controllers

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Optimal Control and Estimation MAE 546
Princeton University, 2015

- LTI dynamic system
- Asymptotic stability of the constant-gain LQG regulator
- Coupling due to parameter uncertainty
- Robustness (loop transfer) recovery
- Stochastic robustness analysis and design



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The Problem: Control to Minimize Cost, Subject to Dynamic Constraint, Uncertain Disturbances, and Measurement Error

$$\dot{\mathbf{x}}(t) = \mathbf{F}\mathbf{x}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{L}\mathbf{w}(t), \quad \mathbf{x}(0) = \mathbf{x}_o$$
Dynamic System
$$\mathbf{z}(t) = \mathbf{H}\mathbf{x}(t) + \mathbf{n}(t)$$

$$\min_{\mathbf{u}} V(t_o) = \min_{\mathbf{u}} J(t_f)$$

Cost Function

$$= \frac{1}{2} \min_{\mathbf{u}} E \left\{ E \left[\mathbf{x}^{T}(t_{f}) \mathbf{S}(t_{f}) \mathbf{x}(t_{f}) | \mathfrak{I}_{D} \right] + E \left\{ \int_{0}^{t_{f}} \left[\mathbf{x}^{T}(t) \ \mathbf{u}^{T}(t) \right] \left[\begin{array}{cc} \mathbf{Q} & \mathbf{0} \\ \mathbf{0} & \mathbf{R} \end{array} \right] \left[\begin{array}{cc} \mathbf{x}(t) \\ \mathbf{u}(t) \end{array} \right] dt \middle| \mathfrak{I}_{D} \right\} \right\}$$

$$\mathfrak{I}_{D}(t) = \{\hat{\mathbf{x}}(t), \mathbf{P}(t), \mathbf{u}(t)\}$$

Initial Conditions and Dimensions

$$E[\mathbf{x}(0)] = \hat{\mathbf{x}}_{o}; \quad E\{[\mathbf{x}(0) - \hat{\mathbf{x}}(0)][\mathbf{x}(0) - \hat{\mathbf{x}}(0)]^{T}\} = \mathbf{P}(0)$$

$$E[\mathbf{w}(t)] = \mathbf{0}; \quad E[\mathbf{w}(t)\mathbf{w}^{T}(\tau)] = \mathbf{W}\delta(t - \tau)$$

$$E[\mathbf{n}(t)] = \mathbf{0}; \quad E[\mathbf{n}(t)\mathbf{n}^{T}(\tau)] = \mathbf{N}\delta(t - \tau)$$

$$E[\mathbf{w}(t)\mathbf{n}^{T}(\tau)] = 0$$
Statistics

$$\dim[\mathbf{x}(t)] = n \times 1$$

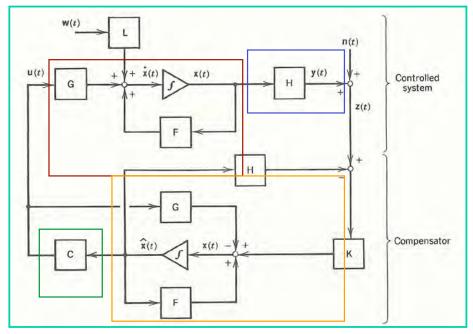
$$\dim[\mathbf{u}(t)] = m \times 1$$

$$\dim[\mathbf{w}(t)] = s \times 1$$

$$\dim[\mathbf{z}(t)] = \dim[\mathbf{n}(t)] = r \times 1$$

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Linear-Quadratic-Gaussian Control



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The Equations (Continuous-Time Model)

System State and Measurement

$$\dot{\mathbf{x}}(t) = \mathbf{F}(t)\mathbf{x}(t) + \mathbf{G}(t)\mathbf{u}(t) + \mathbf{L}(t)\mathbf{w}(t)$$
$$\mathbf{z}(t) = \mathbf{H}\mathbf{x}(t) + \mathbf{n}(t)$$

Control Law

$$\mathbf{u}(t) = -\mathbf{C}(t)\hat{\mathbf{x}}(t) + \mathbf{C}_F(t)\mathbf{y}_C(t)$$

State Estimate
$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}(t)\hat{\mathbf{x}}(t) + \mathbf{G}(t)\mathbf{u}(t) + \mathbf{K}(t)[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t)]$$

= $[\mathbf{F}(t) - \mathbf{G}(t)\mathbf{C}(t) - \mathbf{K}(t)\mathbf{H}]\hat{\mathbf{x}}(t) + \mathbf{G}(t)\mathbf{C}_F(t)\mathbf{y}_C(t) + \mathbf{K}(t)\mathbf{z}(t)$

Estimator Gain and State Covariance Estimate

$$\mathbf{K}(t) = \mathbf{P}(t)\mathbf{H}^T \mathbf{N}^{-1}(t)$$

$$\dot{\mathbf{P}}(t) = \mathbf{F}(t)\mathbf{P}(t) + \mathbf{P}(t)\mathbf{F}^{T}(t) + \mathbf{L}(t)\mathbf{W}(t)\mathbf{L}^{T}(t) - \mathbf{P}(t)\mathbf{H}^{T}\mathbf{N}^{-1}(t)\mathbf{H}\mathbf{P}(t)$$

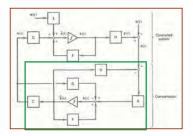
Control Gain and Adjoint Covariance Estimate

$$\mathbf{C}(t) = \mathbf{R}^{-1}(t)\mathbf{G}^{T}(t)\mathbf{S}(t)$$

$$\dot{\mathbf{S}}(t) = -\mathbf{Q}(t) - \mathbf{F}(t)^{T} \mathbf{S}(t) - \mathbf{S}(t) \mathbf{F}(t) + \mathbf{S}(t) \mathbf{G}(t) \mathbf{R}^{-1}(t) \mathbf{G}^{T}(t) \mathbf{S}(t)$$

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Estimator in the Feedback Loop



Linear-Gaussian (LG) state estimator adds dynamics to the feedback signal

$$\mathbf{u}(t) = -\mathbf{C}(t)\hat{\mathbf{x}}(t)$$

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t)[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t)]$$

Thus, state estimator can be viewed as a "compensator" Bandwidth of the compensation is dependent on the multivariable signal/noise ratio, PH⁷N⁻¹

$$\mathbf{K}(t) = \mathbf{P}(t)\mathbf{H}^{T}\mathbf{N}^{-1}$$

$$\dot{\mathbf{P}}(t) = \mathbf{F}\mathbf{P}(t) + \mathbf{P}(t)\mathbf{F}^{T} + \mathbf{L}\mathbf{W}\mathbf{L}^{T} - \mathbf{P}(t)\mathbf{H}^{T}\mathbf{N}^{-1}\mathbf{H}\mathbf{P}(t)$$

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Scalar LTI Example of Estimator Compensation

Dynamic system and measurement

$$\dot{x} = x + w; \quad z = Hx + n$$

Estimator differential equation

$$\hat{x} = \hat{x} + K(z - H\hat{x}) = (1 - KH)\hat{x} + Kz$$

Laplace transform of estimator

$$[s - (1 - KH)]\hat{x}(s) = Kz(s)$$

Estimator transfer function

Low-pass filter

$$\hat{x}(s) = \frac{K}{\left[s - (1 - KH)\right]} z(s)$$

$$\frac{\hat{x}(s)}{z(s)} = \frac{K}{\left[s - (1 - KH)\right]}$$

Steady-State Scalar Filter Gain

Signal "Power" = State Estimate Variance = P
Noise "Power" = Measurement Error Variance = N

H = Projection from Noise Space to Signal Space

Constant, scalar filter gain

$$K = \frac{PH}{N}$$

Algebraic Riccati equation

$$0 = 2P + W - \frac{P^2 H^2}{N}; \quad P^2 - \frac{2N}{H^2} P - \frac{WN}{H^2} = 0$$

$$P = \frac{N}{H^{2}} \pm \sqrt{\left(\frac{N}{H^{2}}\right)^{2} + \frac{WN}{H^{2}}} = \frac{N}{H^{2}} \left[1 \pm \sqrt{1 + \frac{WH^{2}}{N}} \right]$$

Steady-State Filter Gain

$$K = \frac{\left\{\frac{N}{H^2}\left[1 + \sqrt{1 + \frac{WH^2}{N}}\right]\right\}H}{N} = \left\{\frac{1}{H}\left[1 + \sqrt{1 + \frac{WH^2}{N}}\right]\right\}$$

$$K \xrightarrow{W>>N} \sqrt{\frac{W}{N}}$$

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Dynamic Constraint on the Certainty-Equivalent Cost

P(t) is independent of u(t); therefore

$$\min_{\mathbf{u}} J = \min_{\mathbf{u}} J_{CE} + J_{S}$$

 J_{CE} is

Identical in form to the deterministic cost function

Minimized subject to dynamic constraint based on the

state estimate

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t)[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t)]$$

Kalman-Bucy Filter Provides Estimate of the State Mean Value

Filter residual is a Gaussian process

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t) \left[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t) \right]
\triangleq \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t)\mathbf{\varepsilon}(t)$$

Filter equation is analogous to deterministic dynamic constraint on deterministic cost function

$$\dot{\mathbf{x}}(t) = \mathbf{F}\mathbf{x}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{L}\mathbf{w}(t)$$

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Control That Minimizes the Certainty-Equivalent Cost

Optimizing control history is generated by a time-varying feedback control law

$$\mathbf{u}(t) = -\mathbf{C}(t)\hat{\mathbf{x}}(t)$$

The control gain is the same as the deterministic gain

$$\mathbf{C}(t) = \mathbf{R}^{-1}\mathbf{G}^{T}\mathbf{S}(t)$$

$$\dot{\mathbf{S}}(t) = -\mathbf{Q} - \mathbf{F}^{T}\mathbf{S}(t) - \mathbf{S}(t)\mathbf{F} + \mathbf{S}(t)\mathbf{G}\mathbf{R}^{-1}\mathbf{G}^{T}\mathbf{S}(t)$$

$$\mathbf{S}(t_{f}) \text{ given}$$

Optimal Cost for the Continuous-Time LQG Controller

Certainty-equivalent cost

$$J_{CE} = \frac{1}{2} \operatorname{Tr} \left[\mathbf{S}(0) E \left[\hat{\mathbf{x}}(0) \hat{\mathbf{x}}^{T}(0) \right] + \int_{0}^{t_{f}} \mathbf{S}(t) \mathbf{K}(t) \mathbf{N} \mathbf{K}^{T}(t) dt \right]$$

Total cost

$$J = J_{CE} + J_{S}$$

$$= \frac{1}{2} \operatorname{Tr} \left\{ \mathbf{S}(0) E \left[\hat{\mathbf{x}}(0) \hat{\mathbf{x}}^{T}(0) \right] + \int_{0}^{t_{f}} \mathbf{S}(t) \mathbf{K}(t) \mathbf{N} \mathbf{K}^{T}(t) dt \right\}$$

$$+ \frac{1}{2} \operatorname{Tr} \left[\mathbf{S}(t_{f}) \mathbf{P}(t_{f}) + \int_{0}^{t_{f}} \mathbf{Q} \mathbf{P}(t) dt \right]$$

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Discrete-Time LQG Controller

Kalman filter produces state estimate

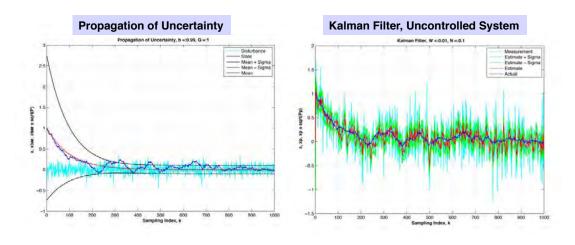
$$\hat{\mathbf{x}}_{k}(-) = \mathbf{\Phi}\hat{\mathbf{x}}_{k-1}(+) + \mathbf{\Gamma}\mathbf{C}_{k-1}\hat{\mathbf{x}}_{k-1}(+)$$

$$\left|\hat{\mathbf{x}}_{k}(+) = \hat{\mathbf{x}}_{k}(-) + \mathbf{K}_{k}\left[\mathbf{z}_{k} - \mathbf{H}\hat{\mathbf{x}}_{k}(-)\right]\right|$$

Closed-loop system uses state estimate for feedback control

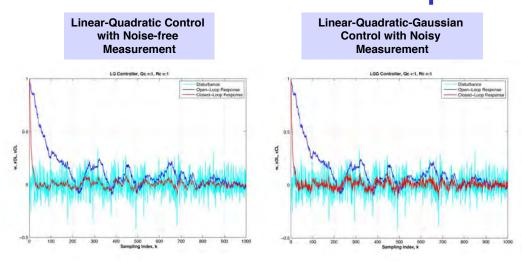
$$\mathbf{x}_{k+1} = \mathbf{\Phi}\mathbf{x}_k - \mathbf{\Gamma}\mathbf{C}_k\hat{\mathbf{x}}_k(+)$$

Response of Discrete-Time 1st-Order System to Disturbance, and Kalman Filter Estimate from Noisy Measurement



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Comparison of 1st-Order Discrete-Time LQ and LQG Control Response



Asymptotic Stability of the LQG Regulator

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System Equations with LQG Control

With perfect knowledge of the system

$$\dot{\mathbf{x}}(t) = \mathbf{F}\mathbf{x}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{L}\mathbf{w}(t)$$
$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t)[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t)]$$

State estimate error

$$\mathbf{\varepsilon}(t) = \mathbf{x}(t) - \hat{\mathbf{x}}(t)$$

State estimate error dynamics

$$\dot{\mathbf{\varepsilon}}(t) = (\mathbf{F} - \mathbf{K}\mathbf{H})\mathbf{\varepsilon}(t) + \mathbf{L}\mathbf{w}(t) - \mathbf{K}\mathbf{n}(t)$$

Control-Loop and Estimator Eigenvalues are Uncoupled

$$\begin{bmatrix} \dot{\mathbf{x}}(t) \\ \dot{\boldsymbol{\varepsilon}}(t) \end{bmatrix} = \begin{bmatrix} (\mathbf{F} - \mathbf{G}\mathbf{C}) & \mathbf{G}\mathbf{C} \\ \mathbf{0} & (\mathbf{F} - \mathbf{K}\mathbf{H}) \end{bmatrix} \begin{bmatrix} \mathbf{x}(t) \\ \boldsymbol{\varepsilon}(t) \end{bmatrix} + \begin{bmatrix} \mathbf{L} & \mathbf{0} \\ \mathbf{L} & -\mathbf{K} \end{bmatrix} \begin{bmatrix} \mathbf{w}(t) \\ \mathbf{n}(t) \end{bmatrix}$$

Upper-block-triangular stability matrix LQG system is stable because

(F – GC) is stable (F – KH) is stable

Estimate error affects state response

$$\dot{\mathbf{x}}(t) = (\mathbf{F} - \mathbf{GC})\mathbf{x}(t) + \mathbf{GC}\boldsymbol{\varepsilon}(t) + \mathbf{Lw}(t)$$

Actual state does not affect error response Disturbance affects both equally

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Parameter Uncertainty Introduces Coupling

Coupling Due To Parameter Uncertainty

 $\left\{ \mathbf{F}_{\!\scriptscriptstyle{A}},\!\mathbf{G}_{\!\scriptscriptstyle{A}},\!\mathbf{H}_{\!\scriptscriptstyle{A}} \right\}$ Actual System:

Assumed System: $\{F,G,H\}$

$$\dot{\mathbf{x}}(t) = \mathbf{F}_{A}\mathbf{x}(t) + \mathbf{G}_{A}\mathbf{u}(t) + \mathbf{L}\mathbf{w}(t)$$

$$\dot{\hat{\mathbf{x}}}(t) = \mathbf{F}\hat{\mathbf{x}}(t) + \mathbf{G}\mathbf{u}(t) + \mathbf{K}(t)[\mathbf{z}(t) - \mathbf{H}\hat{\mathbf{x}}(t)]$$

$$\mathbf{z}(t) = \mathbf{H}_{A}\mathbf{x}(t) + \mathbf{n}(t)$$

$$\mathbf{u}(t) = -\mathbf{C}(t)\hat{\mathbf{x}}(t)$$

$$\mathbf{z}(t) = \mathbf{H}_{A}\mathbf{x}(t) + \mathbf{n}(t)$$
$$\mathbf{u}(t) = -\mathbf{C}(t)\hat{\mathbf{x}}(t)$$

$$\begin{bmatrix} \dot{\mathbf{x}}(t) \\ \dot{\boldsymbol{\epsilon}}(t) \end{bmatrix} = \begin{bmatrix} (\mathbf{F}_A - \mathbf{G}_A \mathbf{C}) & \mathbf{G}_A \mathbf{C} \\ [(\mathbf{F}_A - \mathbf{F}) - (\mathbf{G}_A - \mathbf{G})\mathbf{C} - \mathbf{K}(\mathbf{H}_A - \mathbf{H})] & [\mathbf{F} + (\mathbf{G}_A - \mathbf{G})\mathbf{C} - \mathbf{K}\mathbf{H}] \end{bmatrix} \begin{bmatrix} \mathbf{x}(t) \\ \boldsymbol{\epsilon}(t) \end{bmatrix} + \cdots$$

Closed-loop control and estimator responses are coupled

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Effects of Parameter Uncertainty on Closed-Loop Stability

$$|s\mathbf{I}_{2n} - \mathbf{F}_{CL}| = \begin{vmatrix} \mathbf{S}\mathbf{I}_n - (\mathbf{F}_A - \mathbf{G}_A \mathbf{C}) \\ -[(\mathbf{F}_A - \mathbf{F}) - (\mathbf{G}_A - \mathbf{G})\mathbf{C} - \mathbf{K}(\mathbf{H}_A - \mathbf{H}) \end{bmatrix} \quad \left\{ s\mathbf{I}_n - [\mathbf{F} + (\mathbf{G}_A - \mathbf{G})\mathbf{C} - \mathbf{K}\mathbf{H}] \right\}$$
$$= \Delta_{CL}(s) = 0$$

- Uncertain parameters affect closed-loop eigenvalues
- Coupling can lead to instability for numerous reasons
 - Improper control gain
 - Control effect on estimator block
 - Redistribution of damping

Doyle's Counter-Example of LQG Robustness (1978)

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u + \begin{bmatrix} 1 \\ 1 \end{bmatrix} w$$
Unstable Plant
$$z = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + n$$

Design Matrices
$$\mathbf{Q} = Q \begin{bmatrix} 1 & 1 \\ 1 & 1 \end{bmatrix}; \quad \mathbf{R} = 1; \quad \mathbf{W} = W \begin{bmatrix} 1 & 1 \\ 1 & 1 \end{bmatrix}; \quad \mathbf{N} = 1$$

Control and Estimator Gains

$$\mathbf{C} = (2 + \sqrt{4 + Q}) \begin{bmatrix} 1 & 1 \end{bmatrix} = \begin{bmatrix} c & c \end{bmatrix}$$
$$\mathbf{K} = (2 + \sqrt{4 + W}) \begin{bmatrix} 1 \\ 1 \end{bmatrix} = \begin{bmatrix} k \\ k \end{bmatrix}$$

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Uncertainty in the Control Effect

System Matrices

$$\mathbf{F}_{A} = \mathbf{F}; \quad \mathbf{G}_{A} = \begin{bmatrix} 0 \\ \mu \end{bmatrix}; \quad \mathbf{H}_{A} = \mathbf{H}$$

Characteristic Equation

$$\begin{vmatrix} (s-1) & -1 & 0 & 0 \\ 0 & (s-1) & \mu c & \mu c \\ -k & 0 & (s-1+k) & -1 \\ -k & 0 & (c+k) & (s-1+c) \end{vmatrix} = 0$$

$$\left| s^4 + a_3 s^3 + a_2 s^2 + a_1 s + a_0 \right| = \Delta_{CL}(s) = 0$$

Stability Effect of Parameter Variation

Routh's Stability Criterion (necessary condition)

- All coefficients of $\Delta(s)$ must be positive for stability
 - μ is nominally equal to 1
 - μ can force a_0 and a_1 to change sign
 - Result is dependent on magnitude of *ck*

$$a_1 = k + c - 4 + 2(\mu - 1)ck$$

 $a_0 = 1 + (1 - \mu)ck$

- Arbitrarily small uncertainty, $\mu = 1 + \varepsilon$, could cause unstability
- Not surprising: uncertainty is in the control effect

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The Counter-Example Raises a Flag

Solution

Choose **Q** and **W** to be small, increasing allowable range of **µ**

- However, The counter-example is irrelevant because it does not satisfy the requirements for LQ and LG stability
 - The open-loop system is unstable, so it requires feedback control to restore stability
 - To guarantee stability, Q and W must be positive definite, but

$$\mathbf{Q} = Q \begin{bmatrix} 1 & 1 \\ 1 & 1 \end{bmatrix}; \quad hence, |\mathbf{Q}| = 0$$

$$\mathbf{W} = W \begin{bmatrix} 1 & 1 \\ 1 & 1 \end{bmatrix}; \quad hence, |\mathbf{W}| = 0$$

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Robustness (Loop Transfer Recovery)

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Loop-Transfer Recovery

(Doyle and Stein, 1979)

 Proposition: LQG and LQ robustness would be the same if the control vector had the same effect on the state and its estimate

$$\mathbf{C}\mathbf{x}(t)$$
 and $\mathbf{C}\hat{\mathbf{x}}(t)$ produce same expected value of control, $E[\mathbf{u}(t)]$ but not the same
$$E\{[\mathbf{u}_{LQ}(t) - \mathbf{u}_{LQG}(t)][\mathbf{u}_{LQ}(t) - \mathbf{u}_{LQG}(t)]^T\}$$
 as $\hat{\mathbf{x}}(t)$ contains measurement errors but $\mathbf{x}(t)$ does not

- Therefore, restoring the <u>correct mean value</u> from z(t) restores closed-loop robustness
- Solution: Increase the assumed "process noise" for estimator design as follows (see text for details)

$$\mathbf{W} = \mathbf{W}_o + k^2 \mathbf{G} \mathbf{G}^T$$

Stochastic Robustness Analysis and Design

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Expression of Uncertainty in the System Model

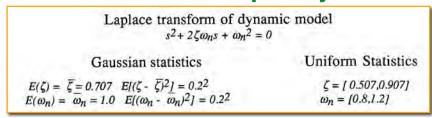
System uncertainty may be expressed as

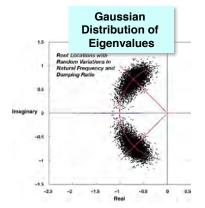
- Elements of **F**
- Coefficients of $\Delta(s)$
- Eigenvalues, λ
- Frequency response/singular values/time response,

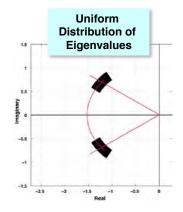
 $A(j\omega), \sigma(j\omega), \mathbf{x}(t)$

- Variation may be
 - Deterministic, e.g.,
 - Upper/lower bounds ("worst-case")
 - Probabilistic, e.g.,
 - · Gaussian distribution
- Bounded variation is equivalent to probabilistic variation with uniform distribution

Stochastic Root Locus: Uncertain Damping Ratio and Natural Frequency

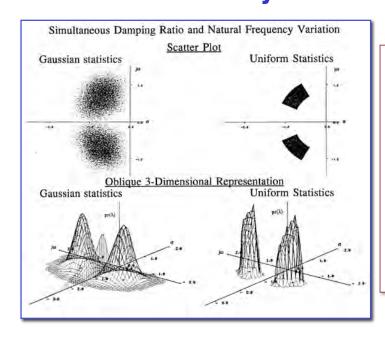






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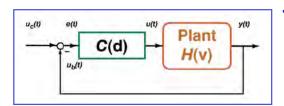
Probability of Instability



- Nonlinear mapping from probability density functions (pdf) of uncertain parameters to pdf of roots
- Finite probability of instability with Gaussian (unbounded) distribution
- Zero probability of instability for some uniform distributions

Probabilistic Control Design

- Design constant-parameter controller (CPC) for satisfactory stability and performance in an uncertain environment
- Monte Carlo Evaluation of simulated system response with
 - competing CPC designs [Design parameters = d]
 - given statistical model of uncertainty in the plant [Uncertain plant parameters = v]

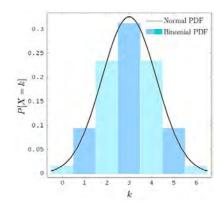


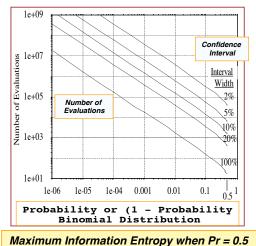
- Search for best CPC
 - Exhaustive search
 - Random search
 - Multivariate line search
 - Genetic algorithm
 - Simulated annealing

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Design Outcome Follows Binomial Distribution

- Binomial distribution: Satisfactory/Unsatisfactory
- · Confidence intervals of probability estimate are functions of
 - Actual probability
 - Number of trials





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Example: Probability of Stable Control of an Unstable Plant



Longitudinal dynamics for a Forward-Swept-Wing Airplane

$$\mathbf{F} = \begin{bmatrix} -2gf_{11}/V & \rho V^2 f_{12}/2 & \rho V f_{13} & -g \\ -45/V^2 & \rho V f_{22}/2 & 1 & 0 \\ 0 & \rho V^2 f_{32}/2 & \rho V f_{33} & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix}$$

$$\mathbf{G} = \begin{bmatrix} g_{11} & g_{12} \\ 0 & 0 \\ g_{31} & g_{32} \\ 0 & 0 \end{bmatrix}; \quad \mathbf{x} = \begin{bmatrix} V, \text{ Airspeed} \\ \alpha, \text{ Angle of attack} \\ q, \text{ Pitch rate} \\ \theta, \text{ Pitch angle} \end{bmatrix}$$

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Example: Probability of Stable Control of an Unstable Plant

Nominal eigenvalues (one unstable)

$$\lambda_{1-4} = -0.1 \pm 0.057 j$$
, -5.15 , 3.35

Air density and airspeed, ρ and V, have uniform distributions(±30%)

10 coefficients have Gaussian distributions ($\sigma = 30\%$)

$$\mathbf{p} = \begin{bmatrix} \rho & V & f_{11} & f_{12} & f_{13} & f_{22} & f_{32} & f_{33} & g_{11} & g_{12} & g_{31} & g_{32} \end{bmatrix}^T$$

Environment

Uncontrolled Dynamics

Control Effect

LQ Regulators for the Example

Three stabilizing feedback control laws

Case a) LQR with low control weighting

$$\mathbf{Q} = diag(1,1,1,0); \quad \mathbf{R} = (1,1); \quad \lambda_{1-4_{nominal}} = -35, -5.1, -3.3, -.02$$

$$\mathbf{C} = \begin{bmatrix} 0.17 & 130 & 33 & 0.36 \\ 0.98 & -11 & -3 & -1.1 \end{bmatrix}$$

Case b) LQR with high control weighting

$$\mathbf{Q} = diag(1,1,1,0); \quad \mathbf{R} = (1000,1000); \quad \lambda_{1-4_{nominal}} = -5.2, -3.4, -1.1, -.02$$

$$\mathbf{C} = \begin{bmatrix} 0.03 & 83 & 21 & -0.06 \\ 0.01 & -63 & -16 & -1.9 \end{bmatrix}$$

Case c) Case b with gains multiplied by 5 for bandwidth (loop-transfer) recovery

$$\lambda_{1-4_{nominal}} = -32, -5.2, -3.4, -0.01$$

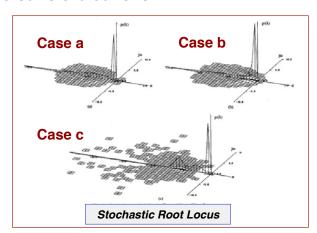
$$\mathbf{C} = \begin{bmatrix}
0.13 & 413 & 105 & -0.32 \\
0.05 & -313 & -81 & -1.1 - 9.5
\end{bmatrix}$$

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Stochastic Robustness

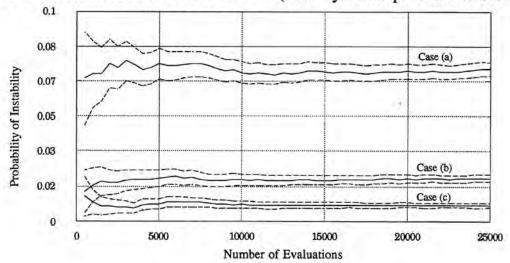
(Ray, Stengel, 1991)

- Distribution of closed-loop roots with
 - Gaussian uncertainty in 10 parameters
 - Uniform uncertainty in velocity and air density
- 25,000 Monte Carlo evaluations
- Probability of instability
- a) Pr = 0.072
- b) Pr = 0.021
- c) Pr = 0.0076



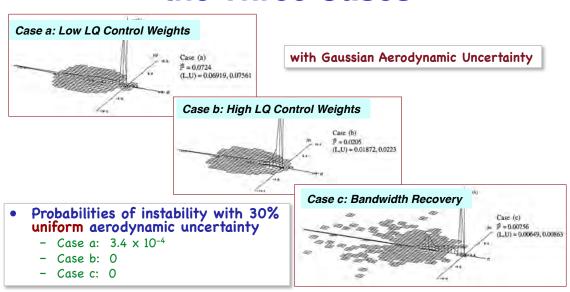
Probabilities of Instability for the Three Cases

95% CONFIDENCE INTERVALS (with dynamic pressure effects)



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Stochastic Root Loci for the Three Cases



ACC Benchmark Control Problem, 1991

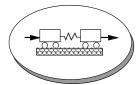
- Parameters of 4th-order mass-spring system
 - Uniform probability density functions for
 - $0.5 < m_1, m_2 < 1.5$
 - 0.5 < k < 2
- Probability of Instability, P_i
 - $-m_i = 1$ (unstable) or 0 (stable)
- Probability of Settling Time Exceedance, P_{ts}
 - $-m_{ts} = 1$ (exceeded) or 0 (not exceeded)
- Probability of Control Limit Exceedance, P_n
 - $-m_u = 1$ (exceeded) or 0 (not exceeded)
- Design Cost Function
- 10 controllers designed for the competition

$$J = aP_i^2 + bP_{ts}^2 + cP_u^2$$

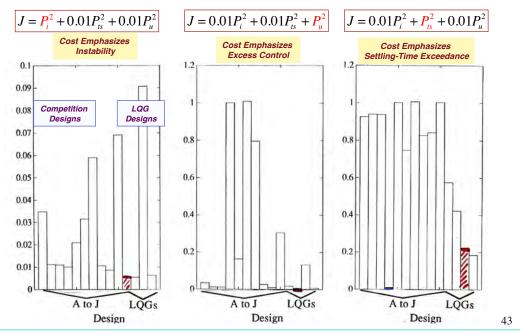
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Stochastic LQG Design for Benchmark Control Problem

- SISO Linear-Quadratic-Gaussian Regulators (Marrison)
 - Implicit model following with control-rate weighting and scalar output (5th order)
 - Kalman filter with single measurement (4th order)
 - Design parameters
 - Control cost function weights
 - · Springs and masses in ideal model
 - · Estimator weights
 - Search
 - · Multivariate line search
 - · Genetic algorithm



Comparison of Design Costs for Benchmark Control Problem



Stochastic LQG controller more robust in 39 of 40 benchmark controller comparisons

Estimation of Minimum Design Cost Using Jackknife/Bootstrapping Evaluation

