

# **Solution to Exam (2025-08-25)**

**MHA021 Finite Element Method**

## Problem 1

### 1.1 Task 1a

Multiply the differential equation with an (arbitrary) test function  $v(x)$  and integrate over the interval

$$-\int_0^L v \frac{d}{dx} \left[ EA \frac{du}{dx} \right] dx = \int_0^L v b dx$$

IBP. of the left-hand side gives (after moving the boundary terms to the right-hand side)

$$\int_0^L EA \frac{dv}{dx} \frac{du}{dx} dx = \int_0^L v b dx + v(L) \left[ EA \frac{du}{dx} \right]_{x=L} - v(0) \left[ EA \frac{du}{dx} \right]_{x=0}$$

At  $x = L$ , we can substitute the the boundary term given in the strong form ( $EA \frac{du}{dx}(0) = 0$ )  $\Rightarrow$  the weak form:

Find  $u$  such that

$$\int_0^L EA \frac{dv}{dx} \frac{du}{dx} dx = \int_0^L v b dx - v(0) \underbrace{\left[ EA \frac{du}{dx} \right]_{x=0}}_{\text{reaction force}=R} = \int_0^L v b dx - v(0) R$$

where  $u(0) = 0$ .

### 1.2 Task 1b

Approximate  $u$  using a linear combination of shape functions:  $u \approx u_h = \sum_i N_i(x) a_i = \mathbf{N} \mathbf{a}$ , where  $N = [N_1(x) \ N_2(x) \ \dots \ N_n(x)]$  is a row vector with the shape functions and  $\mathbf{a} = [a_1 \ a_2 \ \dots \ a_n]^T$  are the degrees of freedom. Use the same shape functions for the test function  $v(x) = \sum_i N_i(x) c_i = \mathbf{N} \mathbf{c}$ . Inserting into the weak form while using the notation  $\frac{d}{dx} \mathbf{N}(x) = \mathbf{B}$  gives

$$\mathbf{c}^T \int_0^L EA \mathbf{B}^T \mathbf{B} dx \mathbf{a} = \mathbf{c}^T \int_0^L \mathbf{N}^T b dx - \mathbf{c}^T \mathbf{N}^T(L) R$$

which should hold for arbitrary  $\mathbf{c} \Rightarrow \dots$

$$\int_0^L EA \mathbf{B}^T \mathbf{B} dx \mathbf{a} = \int_0^L \mathbf{N}^T b dx - \mathbf{N}^T(0) R$$

or simply  $\mathbf{K} \mathbf{a} = \mathbf{f}_l + \mathbf{f}_b$  with

$$\mathbf{K} = \int_0^L EA \mathbf{B}^T \mathbf{B} dx, \quad \mathbf{f}_l = \int_0^L \mathbf{N}^T b dx, \quad \mathbf{f}_b = -\mathbf{N}^T(0) R$$

### 1.3 Task 1c

Assembly of  $\mathbf{K}$  and  $\mathbf{f}_1$  (using the mid-point of each element to evaluate  $b$ ) gives the system of equations

$$\begin{bmatrix} 12000000. & -12000000. & 0. & 0. \\ -12000000. & 24000000. & -12000000. & 0. \\ 0. & -12000000. & 24000000. & -12000000. \\ 0. & 0. & -12000000. & 12000000. \end{bmatrix} \begin{bmatrix} 0 \\ a_2 \\ a_3 \\ a_4 \end{bmatrix} = \begin{bmatrix} 277.8 \\ 1111.1 \\ 2222.2 \\ 1388.9 \end{bmatrix} + \begin{bmatrix} -R \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

### 1.4 Task 1d

Displacement at the free end corresponds to the last element in  $\mathbf{a}$  which gives (after solving the system of equations above)  $u_h \approx 0.00081$  m which gives an error  $e_u = \frac{u(L)-u_h}{u} \approx -2.8\%$ .

Using linear shape functions:  $N_1^e = -\frac{1}{L}(x - x_{i+1})$  and  $N_2^e = \frac{1}{L}(x - x_i)$  gives

$$\mathbf{N}^e = [N_1^e, N_2^e] \Rightarrow \mathbf{B}^e = \frac{d}{dx} \mathbf{N}^e = \frac{1}{L^e} [-1, 1]$$

The normal force is computed as  $N_h^e = EA \mathbf{B}^e \mathbf{a}^e$  which gives for element 1:

$$N_h^1 = EA \mathbf{B}^1 \mathbf{a}^1 = EA \frac{1}{L^e} [-1, 1] \begin{bmatrix} 0 \\ a_2 \end{bmatrix} \approx 4722 \text{ N}$$

which gives an error  $e_N = \frac{N(0)-N_h^1}{N(0)} \approx -5.6\%$

### 1.5 Task 1e

The consistent element load vector becomes (see code for integration)

$$\mathbf{f}_1^e = \int_{x_1}^{x_2} (\mathbf{N}^e)^T b(x) dx = \dots = \begin{bmatrix} -3333x_1^2 + 1667x_1x_2 + 1667x_2^2 \\ -1667x_1^2 - 1667x_1x_2 + 3333x_2^2 \end{bmatrix}$$

Assembling using this load vector gives

$$\begin{bmatrix} 12000000. & -12000000. & 0. & 0. \\ -12000000. & 24000000. & -12000000. & 0. \\ 0. & -12000000. & 24000000. & -12000000. \\ 0. & 0. & -12000000. & 12000000. \end{bmatrix} \begin{bmatrix} 0 \\ a_2 \\ a_3 \\ a_4 \end{bmatrix} = \begin{bmatrix} 185.2 \\ 1111.1 \\ 2222.2 \\ 1481.5 \end{bmatrix} + \begin{bmatrix} -R \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

which gives the following errors instead:  $e_u = 0\%$ ,  $e_N \approx -3.7\%$

```
import numpy as np
import calvem.core as cfc

# c)
EA = 200e9*2e-5
P = 10e3
L = 1
b0 = P/L
K = np.zeros((4, 4))
f = np.zeros((4, 1))

def b(x):
    return b0*x/L

Le = L/3
Ke = EA/Le * np.array([[1, -1],[-1, 1]])
f11 = b(L/6)*Le/2 * np.array([[1],[1]])
f12 = b(3*L/6)*Le/2 * np.array([[1],[1]])
f13 = b(5*L/6)*Le/2 * np.array([[1],[1]])
cfc.assem(np.array([1, 2]), K, Ke, f, f11)
cfc.assem(np.array([2, 3]), K, Ke, f, f12)
cfc.assem(np.array([3, 4]), K, Ke, f, f13)

# Boundary conditions and solve the system of equations

bc_dofs = np.array([1]) # first dof prescribed
bc_vals = np.array([0]) # displacement is zero
a, r = cfc.solveq(K, f, bc_dofs, bc_vals)

# d)
# Postprocessing
u_FEM = a[-1, 0]
N_FEM = EA*(a[1,0] - a[0,0]) / Le

# e)
import sympy as sp
x, x1, x2 = sp.symbols("x x1 x2")

N = sp.Matrix([(x2-x)/(x2-x1), (x-x1)/(x2-x1)]) # shape functions
fle = sp.simplify(sp.integrate(N*b(x), (x, x1, x2)))
fle_ = sp.lambdify((x1, x2), fle, "numpy")

# c) Solve the problem again but with new load vector
```

```
K = np.zeros((4, 4))
f_new = np.zeros((4, 1))
f11 = fle_(0, Le)
f12 = fle_(Le, 2*Le)
f13 = fle_(2*Le, 3*Le)
cfc.assem(np.array([1, 2]), K, Ke, f_new, f11)
cfc.assem(np.array([2, 3]), K, Ke, f_new, f12)
cfc.assem(np.array([3, 4]), K, Ke, f_new, f13)

a_new, r_new = cfc.solveq(K, f_new, bc_dofs, bc_vals)
u_FEM_new = a_new[-1, 0]
N_FEM_new = EA*(a_new[1,0] - a_new[0,0]) / Le
```

## Problem 2: Heat equation

### 2.1 Task 2a

Since nothing is given on the top and bottom boundaries, we assume that these are insulated, i.e.  $q_n = 0$ . We then set up the boundary conditions formally as

$$T(\mathbf{x}) = 0^\circ\text{C}, \quad \mathbf{x} \text{ on } \Gamma_{\text{left}} \quad (2.1)$$

$$T(\mathbf{x}) = 0^\circ\text{C}, \quad \mathbf{x} \text{ on } \Gamma_{\text{right}} \quad (2.2)$$

$$\mathbf{q}^T(\mathbf{x})\mathbf{n} = 0 \text{ W/m}^2, \quad \mathbf{x} \text{ on } \Gamma_{\text{bottom}} \quad (2.3)$$

$$\mathbf{q}^T(\mathbf{x})\mathbf{n} = 0 \text{ W/m}^2, \quad \mathbf{x} \text{ on } \Gamma_{\text{top}} \quad (2.4)$$

$$\mathbf{q}^T(\mathbf{x})\mathbf{n} = 2 \text{ W/m}^2, \quad \mathbf{x} \text{ on } \Gamma_{\text{inner}} \quad (2.5)$$

### 2.2 Task 2b

We multiply the Partial Differential Equation (PDE) by an arbitrary scalar test function,  $v(\mathbf{x})$ , and integrate over the domain,  $\Omega$ , resulting in

$$\int_{\Omega} v \nabla^T \mathbf{q} \, d\Omega = 0 \quad (2.6)$$

Applying Green-Gauss theorem to the left hand side, results in

$$\int_{\Gamma} v \mathbf{n}^T \mathbf{q} \, d\Gamma - \int_{\Omega} [\nabla v]^T \mathbf{q} \, d\Omega = 0 \quad (2.7)$$

Finally, we split the boundary terms into the Dirichlet ( $\Gamma_g = \Gamma_{\text{left}} \cup \Gamma_{\text{right}}$ ) and Neumann ( $\Gamma_h = \Gamma_{\text{top}} \cup \Gamma_{\text{bottom}}$  and  $\Gamma_{\text{inner}}$ ) parts, insert the constitutive law,  $\mathbf{q} = -k\nabla T$ , and re-arrange to get

$$\int_{\Omega} [\nabla v]^T [k \nabla T] \, d\Omega = - \int_{\Gamma_h} v q_n \, d\Gamma - \int_{\Gamma_{\text{inner}}} v q_n \, d\Gamma - \int_{\Gamma_g} v \mathbf{n}^T \mathbf{q} \, d\Gamma \quad (2.8)$$

Still subject to the constraint,  $T(\mathbf{x}) = 0^\circ\text{C}$  for  $x$  on  $\Gamma_g$ . Inserting that  $q_n = 0$  on  $\Gamma_h$ , we get

$$\int_{\Omega} [\nabla v]^T [k \nabla T] \, d\Omega = - \int_{\Gamma_{\text{inner}}} v q_n \, d\Gamma - \int_{\Gamma_g} v \mathbf{n}^T \mathbf{q} \, d\Gamma \quad (2.9)$$

### 2.3 Task 2c

To get the FE form, we introduce the approximation of the test function,  $v(\mathbf{x}) \approx v_h(\mathbf{x}) = \sum_{i=1}^{N_d} N_i(\mathbf{x})c_i = \mathbf{N} \mathbf{c}$ , where  $c_i$  are arbitrary weights, that are coordinate independent. This results

in

$$\sum_{i=1}^{N_d} c_i \underbrace{\left[ \int_{\Omega} [\nabla N_i(\mathbf{x})]^T [k \nabla T] d\Omega - \left[ - \int_{\Gamma_{\text{inner}}} N_i(\mathbf{x}) q_n d\Gamma - \int_{\Gamma_g} N_i(\mathbf{x}) \mathbf{n}^T \mathbf{q} d\Gamma \right] \right]}_{r_i} = 0 \quad (2.10)$$

Since  $c_i$  are arbitrary coefficients, and this has to hold for any  $c_i$ , this implies that  $r_i = 0$  has to hold. We can see this by considering the case that  $c_\alpha = 1$ , and  $c_i = 0$  if  $i \neq \alpha$ . Then  $r_\alpha = 0$  follows, and this has to hold for any  $1 \leq \alpha \leq N_d$ . We thus obtain the FE form,

$$\int_{\Omega} [\nabla N_i(\mathbf{x})]^T [k \nabla T] d\Omega = - \int_{\Gamma_{\text{inner}}} N_i(\mathbf{x}) q_n d\Gamma - \int_{\Gamma_g} N_i(\mathbf{x}) \mathbf{n}^T \mathbf{q} d\Gamma \quad (2.11)$$

and can finally insert the FE-approximation for the temperature,  $T(\mathbf{x}) \approx T_h(\mathbf{x}) = \sum_{i=1}^{N_d} N_i(\mathbf{x}) a_i = \mathbf{N}\mathbf{a}$ , to obtain,

$$\underbrace{\int_{\Omega} [\nabla N_i(\mathbf{x})]^T [k \nabla N_j] d\Omega}_{K_{ij}} a_j = - \underbrace{\int_{\Gamma_{\text{inner}}} N_i(\mathbf{x}) q_n d\Gamma - \int_{\Gamma_g} N_i(\mathbf{x}) \mathbf{n}^T \mathbf{q} d\Gamma}_{f_i} \quad (2.12)$$

$$T_h(\mathbf{x}) = 0^\circ\text{C} \text{ on } \Gamma_{\text{left}} \quad (2.13)$$

$$T_h(\mathbf{x}) = 0^\circ\text{C} \text{ on } \Gamma_{\text{right}} \quad (2.14)$$

where the reaction flux,  $q_n(\mathbf{x})$  is unknown on  $\Gamma_g$  and  $q_n(\mathbf{x}) = 2 \text{ W/m}^2$  on  $\Gamma_{\text{inner}}$ .

## 2.4 Task 2d & 2e

### # Problem 2

#### # General packages

```
import numpy as np
import scipy.io
import matplotlib.pyplot as plt
import matplotlib.tri as tri

# CALFEM packages
import calfem.core as cfc
import calfem.vis_mpl as cfv
```

#### # Load mesh data

```
mesh = scipy.io.loadmat('mesh_task2.mat')
```

```
Coord = mesh['Coord'] # [x, y] coords for each node
```

```
Dofs = mesh['Dofs']
```

```
Edof = mesh['Edof']                                # [element number, dof1, dof2, dof3]
Ex = mesh['Ex']
Ey = mesh['Ey']
right_dofs = mesh['right_dofs']
left_dofs = mesh['left_dofs']
inner_edgedofs = mesh['inner_edgedofs']
inner_ex = mesh['inner_ex']
inner_ey = mesh['inner_ey']

# Plot the mesh
plotpar = np.array([1, 1, 2]) # parameters for line style, color, marker
# cfv.eldraw2(Ex, Ey)
# plt.xlabel("x [m]")
# plt.ylabel("y [m]")
# plt.show()

nel=np.shape(Edof)[0]
ndofs=np.max(Edof[:,1:])
dimension = 2

T1 = 0 # Prescribed temperature on right boundary [C]
T2 = 0 # Prescribed temperature on left boundary [C]
t = 0.01 # thickness [m]
qn = -2.0 # [W/m^2]
k = 1.0 # Heat conductivity [W/mC]

D = np.eye(dimension) * k
K = np.zeros([ndofs,ndofs])
f = np.zeros([ndofs, 1])

for el in range(nel):
    Ke = cfc.flw2te(Ex[el,:], Ey[el,:], [t], D)
    K = cfc.assem(Edof[el,1:], K, Ke)

# Assemble Neumann boundary conditions
num_edges = inner_edgedofs.shape[0]
for edge in range(num_edges):
    ex = inner_ex[edge, :]
    ey = inner_ey[edge, :]
    edof = inner_edgedofs[edge, :]-1
    Le = np.sqrt( (ex[1]-ex[0])**2 + (ey[1] - ey[0])**2 )
    fe = -(t * Le * qn / 2) * np.array([[1], [1]])
    f[edof] += fe
bcpresc=np.vstack([right_dofs,left_dofs]).flatten()
```



```
bcVal1=np.ones_like(right_dofs)*T2
bcVal2=np.ones_like(left_dofs)*T1
bcVal=np.vstack([bcVal1,bcVal2]).flatten()
a, q = cfc.solveq(K,f,bcpresc,bcVal)

print(a)
np.linalg.norm(a)

# Heat flux vector in element 10, 20, 30

ed = cfc.extract_ed(Edof[:, 1:], a)
el = 10
es10, et10 = cfc.flw2ts(Ex[el-1, :], Ey[el-1, :], D, ed[el-1, :])
el = 20
es20, et20 = cfc.flw2ts(Ex[el-1, :], Ey[el-1, :], D, ed[el-1, :])
el = 30
es30, et30 = cfc.flw2ts(Ex[el-1, :], Ey[el-1, :], D, ed[el-1, :])
display(es10, es20, es30) # heat flux vectors
```

## Problem 3

See matlab file at the bottom for full calculations.

### 3.1 Task 3a

The local coordinate of point A is,  $\xi_A = [0, 1]^T$ , which gives with the shape functions in the formula sheet for a 4-noded quadrilateral,

$$\mathbf{x}_A = \sum_{\alpha=1}^4 \mathbf{x}_{\alpha} \hat{N}_{\alpha}(\xi_A) = \begin{bmatrix} 0.0150 \\ 0.0170 \end{bmatrix} \text{ m} \quad (3.1)$$

### 3.2 Task 3b

Given the displacement vector, we transform this into the dof vector as  $\mathbf{a}^e = [u_{x,1}^e, u_{y,1}^e, u_{x,2}^e, \dots]^T$ , and then we have the vectorized shape functions,

$$\mathbf{N}^e = \begin{bmatrix} M_1 & 0 & M_2 & \dots \\ 0 & M_1 & 0 & \dots \end{bmatrix} \quad (3.2)$$

and the displacement at the point  $\mathbf{x}_A$  is then given as

$$\mathbf{u}_A = \mathbf{N}^e \mathbf{a}^e = \begin{bmatrix} 0.0375 \\ 0.0160 \end{bmatrix} \text{ mm} \quad (3.3)$$

To calculate the strains, we need the spatial gradients of the shape functions, i.e. the B-matrix. So we need the isoparametric mapping, and from the formula sheet we use

$$\mathbf{J} = \frac{\partial \mathbf{x}}{\partial \xi} = \sum_{\alpha=1}^{N_{\text{nodes}}} \mathbf{x}_{\alpha}^e \left[ \frac{\partial \hat{N}_{\alpha}}{\partial \xi} \right]^T, \quad \frac{\partial N_i^e}{\partial \mathbf{x}} = \mathbf{J}^{-T} \frac{\partial \hat{N}_i^e}{\partial \xi} \quad (3.4)$$

$$\mathbf{B} = \begin{bmatrix} \frac{\partial N_1^e}{\partial x_1} & 0 & \frac{\partial N_2^e}{\partial x_1} & 0 & \dots \\ 0 & \frac{\partial N_1^e}{\partial x_2} & 0 & \frac{\partial N_2^e}{\partial x_2} & \dots \\ \frac{\partial N_1^e}{\partial x_2} & \frac{\partial N_1^e}{\partial x_1} & \frac{\partial N_2^e}{\partial x_2} & \frac{\partial N_2^e}{\partial x_1} & \dots \end{bmatrix} \quad (3.5)$$

The strain is then given as

$$\epsilon = \mathbf{B}^e \mathbf{a}^e = \begin{bmatrix} 0.017 \\ 0.046 \\ 0.017 \end{bmatrix} \% \quad (3.6)$$

and the stress as

$$\sigma = \mathbf{D} \epsilon = \begin{bmatrix} 71.1 \\ 117.9 \\ 13.7 \end{bmatrix} \text{ MPa} \quad (3.7)$$

### 3.3 Task 3c

Calculating the integral implies that we have

$$\int_{\Omega^e} d\Omega \approx \sum_{\alpha=1}^4 w_{\alpha} \det(\mathbf{J}(\xi_{\alpha})) = 50 \text{ mm}^2 \quad (3.8)$$

### 3.4 Python code

```
# Solution Problem 3, Re-Exam MHA021, 2025-08-25
# This code is a AI-translation from the Matlab solution written by Knut Andreas Meyer
import numpy as np
```

```
def shape_values(xi):
    """
    xi: array-like of length 2 (xi1, xi2)
    returns N as a 1D array of length 4
    """
    xi = np.asarray(xi).reshape(2,)
    N = np.array([
        (1 - xi[0]) * (1 - xi[1]),
        (1 + xi[0]) * (1 - xi[1]),
        (1 + xi[0]) * (1 + xi[1]),
        (1 - xi[0]) * (1 + xi[1]),
    ]) / 4.0
    return N
```

```
def shape_gradients(xi):
    """
    xi: array-like of length 2
    returns dN/dxi as a (2 x 4) array
    Row 0 = dN/dxi1, Row 1 = dN/dxi2
    """
    xi = np.asarray(xi).reshape(2,)
    dNdx1 = np.array([
```

```
        [-(1 - xi[1]), -(1 - xi[0])], # node 1
        [(1 - xi[1]), -(1 + xi[0])], # node 2
        [(1 + xi[1]), +(1 + xi[0])], # node 3
        [-(1 + xi[1]), +(1 - xi[0])], # node 4
    ]).T / 4.0 # transpose to get (2 x 4)
    return dNdx_i

def element_jacobian(xi, x):
    """
    xi: array-like of length 2
    x: (2 x 4) nodal coordinates array (each column is a node: [x; y])
    returns J as a (2 x 2) array
    """
    dNdx_i = shape_gradients(xi) # (2 x 4)
    J = x @ dNdx_i.T # (2 x 4) @ (4 x 2) = (2 x 2)
    return J

E = 210e9
nu = 0.3
t = 0.025 # thickness

# Plane-stress constitutive matrix D
D = (E / (1 - nu**2)) * np.array([
    [1.0, nu, 0.0],
    [nu, 1.0, 0.0],
    [0.0, 0.0, (1 - nu) / 2.0]
], dtype=float)

# Nodal coordinates (2 x 4): each column is a node [x; y]
x = np.array([
    [0.014, 0.021, 0.018, 0.012],
    [0.010, 0.009, 0.018, 0.016],
], dtype=float)

# Element dofs (8 x 1): [u1 v1 u2 v2 u3 v3 u4 v4]^T
ae = (1e-5) * np.array([4.1, 1.0, 3.5, 1.6, 3.8, 1.7, 3.7, 1.5], dtype=float)

xi_A = np.array([0.0, 1.0], dtype=float)

# Shape values and point in physical coordinates
Ns = shape_values(xi_A) # (4,)
xA = x @ Ns # (2,) equivalent to (Ns * x')' in MATLAB
```

```
# Interpolation matrix Nv and displacement at A
Nv = np.array([
    [Ns[0], 0.0,  Ns[1], 0.0,  Ns[2], 0.0,  Ns[3], 0.0],
    [0.0,  Ns[0], 0.0,  Ns[1], 0.0,  Ns[2], 0.0,  Ns[3]],
], dtype=float)
uA = Nv @ ae          # (2,)

# Gradients in parent coordinates
dNdx_i = shape_gradients(xi_A)      # (2 x 4)

# Jacobian (direct)
J = x @ dNdx_i.T                # (2 x 2)

# Alternative Jacobian (constructed in a loop)
J1 = np.zeros((2, 2))
for i in range(x.shape[1]):      # i = 0..3
    J1 += np.outer(x[:, i], dNdx_i[:, i])

# Map gradients to physical coordinates: dN/dx = inv(J') * dN/dxi
dNdx = np.linalg.inv(J.T) @ dNdx_i # (2 x 4)

# Strain-displacement matrix B (3 x 8)
B = np.zeros((3, 8))
B[0, 0::2] = dNdx[0, :] # epsilon_xx terms
B[1, 1::2] = dNdx[1, :] # epsilon_yy terms
B[2, 0::2] = dNdx[1, :] # gamma_xy terms (shear)
B[2, 1::2] = dNdx[0, :]

# Strain and stress at xi_A
strain = B @ ae          # (3,)
stress = D @ strain      # (3,)

# 2x2 Gauss quadrature for area integration
qr_points_1d = np.array([-1.0 / np.sqrt(3.0), 1.0 / np.sqrt(3.0)])
qr_weights_1d = np.array([1.0, 1.0])

# Build 2D rule
qr_points_2d = np.zeros((2, qr_points_1d.size ** 2)) # (2 x 4)
qr_weights_2d = np.zeros(qr_points_1d.size ** 2)     # (4,)
k = 0
for i in range(qr_points_1d.size):
    for j in range(qr_points_1d.size):
        qr_points_2d[:, k] = [qr_points_1d[i], qr_points_1d[j]]
```

```
qr_weights_2d[k] = qr_weights_1d[i] * qr_weights_1d[j]
k += 1

# Integrate area A = \int det(J) dxi over the parent domain
A = 0.0
for i in range(qr_weights_2d.size):
    xi = qr_points_2d[:, i]
    Ji = element_jacobian(xi, x)
    A += qr_weights_2d[i] * np.linalg.det(Ji)

# --- Output ---
np.set_printoptions(precision=6, suppress=True)
print("Ns =", Ns)
print("xA =", xA)
print("uA =", uA)
print("J =\n", J)
print("J1 (loop) =\n", J1)
print("dNdx =\n", dNdx)
print("B =\n", B)
print("strain =", strain)
print("stress =", stress)
print("A =", A)
```