

January

3 mon

2005

Vibrations

The to and fro motion of a body about its mean position is called oscillations or vibrations.

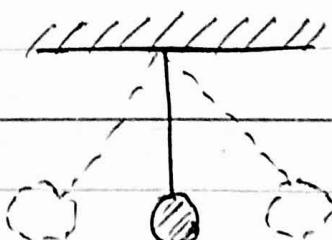
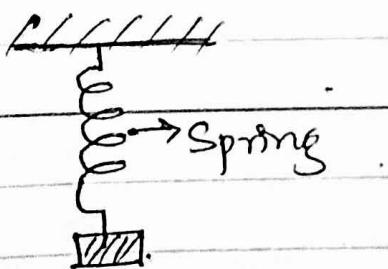
There are three parts of oscillations.

- 4 tue
- i) Free oscillation
 - ii) Damped "
 - iii) Forced. "

i) Free oscillation/vibration is defined as the body which vibrates with definite interval and with natural frequency.

Ex:- SHO. - Simple Harmonic Oscillation.

5 wed



Simple pendulum.

A particle is said to execute SHM when it vibrates periodically in such a way that at any instant, the restoring force acting on it is proportional to its displacement from mean position and oppositely directed.

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$$\therefore F_{\text{Restoring}} \propto -x \rightarrow \textcircled{1}$$

Where F is the restoring force acting on the oscillator when its displacement from equilibrium position is ' x '.

$$F_R = -kx. \rightarrow \textcircled{2}$$

7 Fri

Now according to Newton's 2nd law.

$$F = ma. \rightarrow \textcircled{3}$$

From eqn $\textcircled{2}$ and $\textcircled{3}$.

$$ma = -kx.$$

$$ma + kx = 0. \xrightarrow{\text{sum}} \textcircled{4}$$

We know that, $a = \frac{d^2x}{dt^2}$

$$\text{Eqn } \textcircled{4} \Rightarrow m \frac{d^2x}{dt^2} + kx = 0.$$

Divide by 'm' to above eqn.

$$\frac{d^2x}{dt^2} + \left(\frac{k}{m}\right)x = 0.$$

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10 min

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Let us substitute $\frac{k}{m} = \omega^2$ in above eqns.

$$\boxed{\frac{d^2x}{dt^2} + \omega^2 x = 0}$$

→ ⑤

$$\omega = \sqrt{\frac{k}{m}} \text{. Angular freq.}$$

Eqn ⑤ represents the differential of simple Harmonic oscillator/vibration.

11 min

Solution of differential equation can be given as

$$x = e^{\alpha t} \quad \text{where } x \text{ is the displacement or amplitude}$$

→ ⑥

Differentiating above 'x'

$$\frac{dx}{dt} = \alpha \cdot e^{\alpha t} \rightarrow ⑦ \quad \frac{d^2x}{dt^2} = \alpha^2 \cdot e^{\alpha t} \rightarrow ⑧$$

12 min

Substituting ⑦ and ⑧ in eqn ⑤.

$$\alpha^2 \cdot e^{\alpha t} + \omega^2 e^{\alpha t} = 0.$$

$$e^{\alpha t} (\alpha^2 + \omega^2) = 0.$$

$$e^{\alpha t} \neq 0; \therefore \alpha^2 + \omega^2 = 0.$$



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$$\alpha^2 = -\omega^2.$$

$$\therefore \alpha = \pm i\omega.$$

General solution of eqn ⑤ is given as

$$x = A e^{i\omega t} + B e^{-i\omega t} \rightarrow ⑥$$

where A and B are constants

14 Fri

$$\text{Now } e^{i\omega t} = \cos \omega t + i \sin \omega t.$$

$$e^{-i\omega t} = \cos \omega t - i \sin \omega t.$$

$$⑥ \Rightarrow x = A(\cos \omega t + i \sin \omega t) + B(\cos \omega t - i \sin \omega t).$$

$$\Rightarrow x = A \underline{\cos \omega t} + i \underline{A \sin \omega t} + B \underline{\cos \omega t} - B \underline{i \sin \omega t}.$$

15 Sat

$$x = \cos \omega t (A+B) + i \sin \omega t (A-B) \rightarrow ⑦$$

Replacing $(A+B)$ and $i(A-B)$ by ^{some} constants

$$A+B = R \sin \phi \quad \text{and} \quad i(A-B) = R \cos \phi.$$

$$⑦ \Rightarrow x = \cos \omega t \cdot R \sin \phi + \sin \omega t \cdot R \cos \phi.$$

$$x = R (\cos \omega t \cdot \sin \phi + \sin \omega t \cdot \cos \phi).$$

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$$x = R \sin(\omega t + \phi) \quad \rightarrow \textcircled{11}$$

Eqn $\textcircled{11}$ represents the final solⁿ of SHM.

where $R \rightarrow$ Amplitude of the oscillatory system.

$(\phi + \omega t) \rightarrow$ Phase of vibration.

$x = R \sin(\omega t + \phi) \rightarrow$ represents the displacements of SHM. (Amplitude)

18 tue

We now calculate the time period and frequency of SHM.

Time period is defined as the time required for one complete oscillation.

$$t = 2\pi/\omega \Rightarrow 2\pi/\sqrt{k/m} \quad \therefore \omega^2 = \frac{k}{m}$$

19 wed

$$\omega = \sqrt{k/m}$$

Frequency of SHM:

$$v = \frac{1}{T} = \frac{\omega}{2\pi} = \frac{\sqrt{k/m}}{2\pi}$$

Significance of ' ω ' in eqn $\textcircled{11}$.

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If 't' is increased by $2\pi/\omega$

$$\text{i.e. } t = t + \frac{2\pi}{\omega}.$$

$$\therefore (1) \Rightarrow x = R \sin(\omega(t + \frac{2\pi}{\omega}) + \phi) \\ = R \sin(\omega t + 2\pi + \phi).$$

21 Fri

$$x = R \sin(2\pi + (\omega t + \phi))$$

$$x = R \sin(\omega t + \phi)$$

$$\therefore \sin(2\pi + \theta) = \sin \theta.$$

which means displacement is same after time
 $t = 2\pi/\omega$.

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So, the period $t = \frac{2\pi}{\omega}$ is the time period.

ii) Damped oscillation / vibration — For a free oscillation the energy remains constant and hence oscillator continues indefinitely.

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But in reality, the amplitude of the oscillatory system gradually decreases due to the damping forces like friction (air) and resistance of the media.

25 tue "The oscillator whose amplitude, in successive oscillations goes on decreasing due to the presence of resistive forces are called as damped oscillators and oscillations are called damped oscillations (DHO)."

The damping force always act in a opposite directions to the motion of oscillatory body and velocity dependent.

26 wed $F_{\text{damping}} \propto -V \Rightarrow F_{\text{damping}} = -\gamma V$.

$\therefore F_{\text{damping}} = -\gamma V$. \rightarrow Damping constant.
 \rightarrow ① $V \rightarrow$ Damping force velocity

Now $F_{\text{restoring}} = -kx$. \rightarrow ②

According to Newton's 2nd law of motion.

$$F = ma. \rightarrow ③$$

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Total Force, $F = \text{Restoring} + \text{Frapping}$.

$$\therefore ma = -kx - \tau \frac{dx}{dt}.$$

$$m \frac{d^2x}{dt^2} + \tau \frac{dx}{dt} + kx = 0.$$

Divide by m in the above eqn.

$$\frac{d^2x}{dt^2} + \frac{\tau}{m} \frac{dx}{dt} + \frac{k}{m} x = 0.$$

Substitute $\tau/m = \alpha k$ and $k/m = \omega^2$ in above eqn.

$$\therefore \boxed{\frac{d^2x}{dt^2} + 2\alpha k \frac{dx}{dt} + \omega^2 x = 0.} \rightarrow ④$$

where " αk " is the damping coefficient and " ω " is the natural freq. of oscillating body.

Eqn ④ represents the differential eqn of Damped Harmonic Oscillation.

This is second order linear differential Homogeneous equation in x .

The general solution of the above eqn can be written as.

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$$x = A \cdot e^{\alpha t} \quad x \text{ is the amplitude}$$

$$\frac{dx}{dt} = A \cdot \alpha \cdot e^{\alpha t} \Rightarrow \frac{d^2x}{dt^2} = A \cdot \alpha^2 e^{\alpha t}.$$

$$④ \Rightarrow A \cdot \alpha^2 \cdot e^{\alpha t} + 2k \cdot A \alpha e^{\alpha t} + w^2 A e^{\alpha t} = 0.$$

$$1 \text{ tue} \Rightarrow A e^{\alpha t} [\alpha^2 + 2k\alpha + w^2] = 0.$$

Since $A e^{\alpha t} \neq 0$ but $\alpha^2 + 2k\alpha + w^2 = 0$

Quadratic eqn.

The two roots of the quad. eqn. are.

$$\alpha = \frac{-2k \pm \sqrt{4k^2 - 4 \cdot 1 \cdot w^2}}{2 \cdot 1} = \frac{-2k \pm 2\sqrt{k^2 - w^2}}{2}$$

$$\alpha = -k \pm \sqrt{k^2 - w^2} \rightarrow ⑤.$$

2 wed $\therefore \alpha$ in eqn ⑤ has two values.

$$\alpha_1 = -k + \sqrt{k^2 - w^2} \quad \text{and} \quad \alpha_2 = -k - \sqrt{k^2 - w^2}$$

Therefore, general soln of differential eqn is.

$$x = A \cdot e^{\alpha_1 t} + B \cdot e^{\alpha_2 t}.$$

$$x = A \cdot e^{(-k + \sqrt{k^2 - w^2})t} + B \cdot e^{(-k - \sqrt{k^2 - w^2})t} \rightarrow ⑥$$

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Eqn ⑥ represents the amplitude of D.H.O.

The actual form of eqn ⑥ depends on whether

- (a) $k^2 > \omega^2$ +/cany damping
- (b) $k^2 < \omega^2$ Low "
- (c) $k^2 = \omega^2$ Critical "

Case-I $k^2 > \omega^2$ (+/cany damping)

If the damping is high, then $\sqrt{k^2 - \omega^2} = \beta$ is a real quantity and positive.

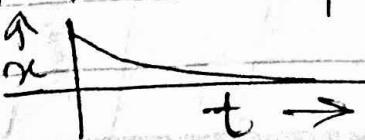
Then eqn ⑥ \Rightarrow

$$x = e^{-kt} [A \cdot e^{(\sqrt{k^2 - \omega^2})t} + B \cdot e^{(-\sqrt{k^2 - \omega^2})t}] \rightarrow \text{F}$$

$$x = e^{-kt} [A \cdot e^{\beta t} + B \cdot e^{-\beta t}]$$

$$x = A \cdot e^{-(k-\beta)t} + B \cdot e^{-(k+\beta)t}$$

Since both the exponents are negative then the displacement 'x' decreases continuously with the time. That is if the particle when once displaced return to its equilibrium position quite slowly.



This is DEAD BEAT

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Notes Non oscillatory or aperiodic motion.

Ex:- Pendulum in viscous medium.

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(b) Case - II $k^2 < \omega^2$ — Under damping / low damping.
Here $\sqrt{k^2 - \omega^2}$ is imaginary.

So. eqn ⑥ $\sqrt{k^2 - \omega^2} = \sqrt{-(\omega^2 - k^2)} = i\sqrt{\omega^2 - k^2} = iw$,
where $\sqrt{-1} = i$ and $w = \sqrt{\omega^2 - k^2}$.

\therefore Eqn ⑥ $\Rightarrow x = e^{-kt} [A e^{i\omega t} + B e^{-i\omega t}]$

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$$x = e^{-kt} [A(\cos \omega t + i \sin \omega t) + B(\cos \omega t - i \sin \omega t)]$$

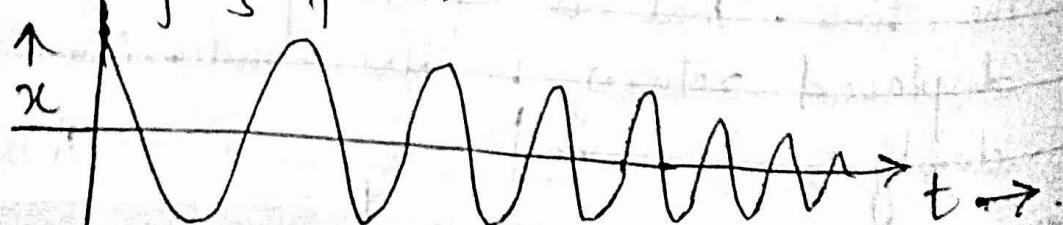
$$x = e^{-kt} [(A+B) \cos \omega t + i(A-B) \sin \omega t]$$

$$\Rightarrow A+B = a_0 \sin \phi ; i(A-B) = a_0 \cos \phi .$$

$$\therefore x = e^{-kt} [a_0 \sin \phi \cos \omega t + a_0 \cos \phi \sin \omega t].$$

9 wed $x = a_0 e^{-kt} [\sin \phi \cos \omega t + \cos \phi \sin \omega t]$
 $\boxed{x = a_0 e^{-kt} \sin(\omega t + \phi)} \rightarrow ⑧$

$a_0 e^{-kt}$ — represents amplitude of oscillation and it decays exponentially with time due to e^{-kt} .
 e^{-kt} — Damping factor.



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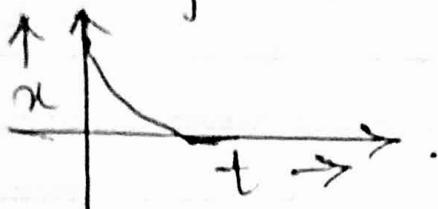
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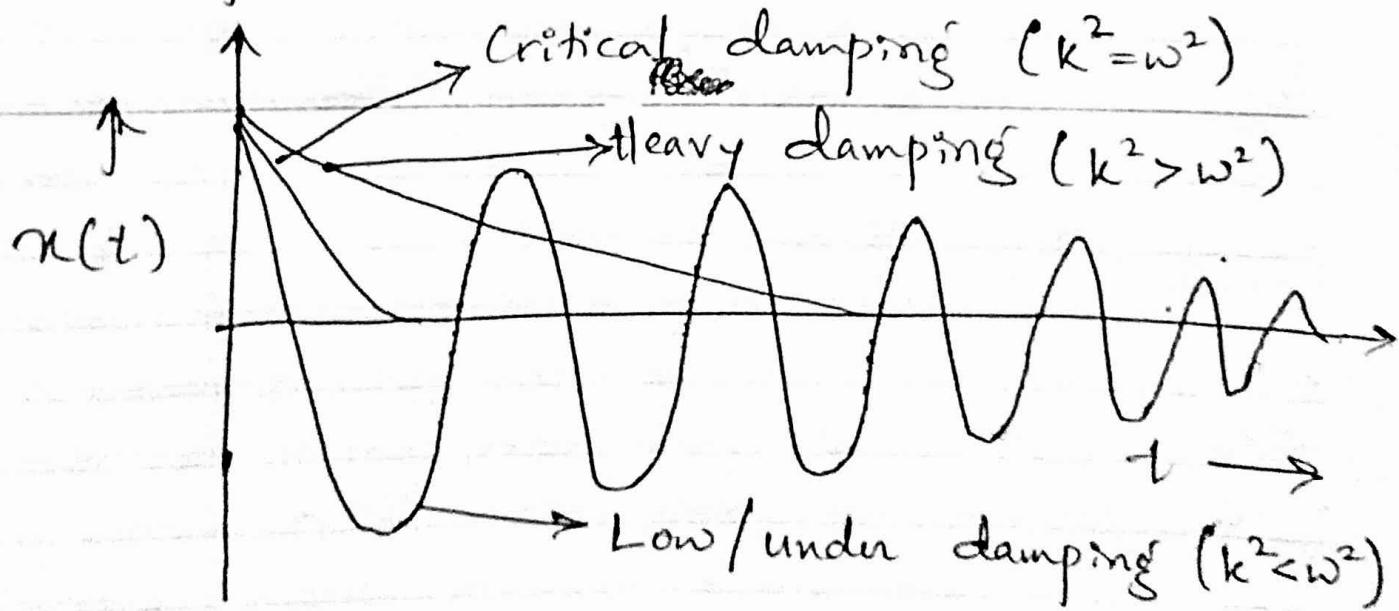
⑥ Case III $k^2 = \omega^2$ (critical damping)
 Eqn ⑥ will be like $x = (A+B)e^{-kt}$

Therefore, the displacement approaches to zero asymptotically for given value of initial position.

Here, the critically damped oscillator approaches to the equilibrium position more rapidly than compared to heavy damped.



Curves of all the three cases:



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iii) Forced/Driven Oscillators (FHO):

When oscillator is subjected to external periodic force and it oscillates with the frequency of external periodic force, it is called as Forced Harmonic Oscillator (FHO).

15 tue

$$F = F_{\text{Restoring}} + F_{\text{Damping}} + F_{\text{ext-periodic force}}$$

$$= -kx - \gamma v + F_0 \sin pt.$$

$$ma = -kx - \gamma \frac{dx}{dt} + F_0 \sin pt.$$

$$m \frac{d^2x}{dt^2} + \gamma \frac{dx}{dt} + kx = F_0 \sin pt.$$

16 wed Divide the above eqn by m .

$$\frac{d^2x}{dt^2} + \frac{\gamma}{m} \frac{dx}{dt} + \left(\frac{k}{m}\right)x = \frac{F_0}{m} \sin pt.$$

$$\boxed{\frac{d^2x}{dt^2} + 2K \frac{dx}{dt} + \omega^2 x = f_0 \sin pt} \rightarrow ①$$

Eqn ① is the differential eqn of FHO.

$$\text{where } \frac{\gamma}{m} = 2K ; \frac{k}{m} = \omega^2 ; \frac{F_0}{m} = f_0.$$

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Let us assume the solution of eqn ① as.

$$x = A \sin(pt - \theta)$$

where p is the driving frequency.

$$\frac{dx}{dt} = Ap \cos(pt - \theta)$$

$$\frac{d^2x}{dt^2} = -Ap^2 \sin(pt - \theta).$$

Substituting above values in eqn ①.

$$-Ap^2 \sin(pt - \theta) + 2KAp \cos(pt - \theta) + \omega^2 A \sin(pt - \theta) \\ = f_0 \sin((pt - \theta) + \theta). \quad \rightarrow \sin(A+B)$$

$$\Rightarrow (\sin pt - \theta)(\omega^2 A - Ap^2) + 2KAp \cos(pt - \theta) = \\ f_0 \sin(pt - \theta) \cos \theta + f_0 \cos(pt - \theta) \sin \theta.$$

$$\Rightarrow A(\omega^2 - p^2) \sin(pt - \theta) + \cancel{2KAp \cos(pt - \theta)} = \\ f_0 \sin(pt - \theta) \cos \theta + f_0 \sin \theta \cos(pt - \theta). \quad \rightarrow ②$$

Now case-1 If $(pt - \theta) = 0$ then $\therefore \sin 0 = 0$
 eqn ② $\Rightarrow 2KAp = f_0 \sin \theta \rightarrow ③ \quad \cos \theta = 1$

case-2 If $(pt - \theta) = \pi/2$ then $\therefore \sin \pi/2 = 1$
 eqn ② $\Rightarrow (\omega^2 - p^2) A = f_0 \cos \theta \rightarrow ④ \quad \cos \pi/2 = 0.$

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Squaring and adding eqn ③ and ④ we get

$$4K^2 p^2 A^2 + A^2 (\omega^2 - p^2)^2 = f_0^2 (\sin^2 \theta + \cos^2 \theta)$$

$$A^2 [4K^2 p^2 + (\omega^2 - p^2)^2] = f_0^2$$

22 tue

$$A = \frac{f_0}{\sqrt{(\omega^2 - p^2)^2 + 4K^2 p^2}} \rightarrow ⑤$$

Now let us substitute eqn ⑤ in $x = A \sin(pt - \theta)$

$$x = \frac{f_0}{\sqrt{(\omega^2 - p^2)^2 + 4K^2 p^2}} \sin(pt - \theta) \rightarrow ⑥$$

23 wed

Eqn ⑥ is the General solution eqn of FHO.

② Case - I when $\omega > p$ and damping is low

i.e., Driving frequency is very less than the natural frequency.

$$\text{Eqn } ⑤ \Rightarrow A = \frac{f_0}{\omega^2} = \frac{F_0/m}{k/m} = \frac{F_0}{k}$$

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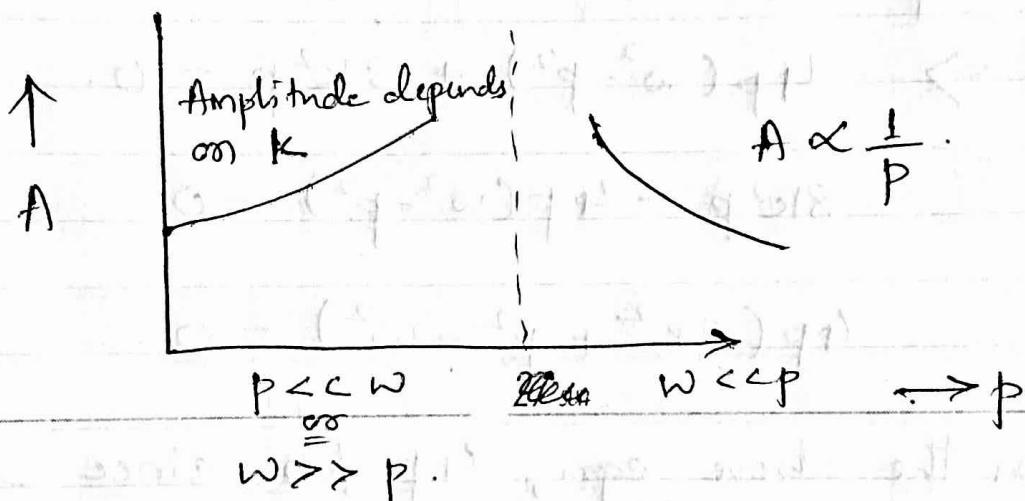
In this case amplitude depends on 'k' force constant only but not on the mass, damping and frequency of driving force.

(b) Case - II When $\omega \ll p$ and damping is low.

then Eqn (5) $\Rightarrow A \approx \frac{f_0}{p^2}$

25 fri

\Rightarrow If 'p' increases then amplitude 'A' will decrease



26 sat

(c) Case - III when $\omega = p$ (Resonance case):

In this case the amplitude will be maximum.

Eqn (5) $\Rightarrow A = \frac{f_0}{\sqrt{4k^2p^2}} \Rightarrow A = \frac{f_0}{2kp}$

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but in order to get 'A' maximum then we should get denominator of eqn ⑤ be minimum.

Consider $V = (w^2 - p^2)^2 + 4k^2p^2$ should be minimum.

also $\frac{du}{dp}$ should be minimum.

1 tue

$$\therefore \frac{dV}{dp} = 2(w^2 - p^2)(-2p) + 8k^2p = 0.$$

$$\Rightarrow -4p(w^2 - p^2) + 8k^2p = 0.$$

$$8k^2p - 4p(w^2 - p^2) = 0.$$

$$4p(2k^2 + p^2 - w^2) = 0.$$

2 wed

In the above eqn, $4p \neq 0$ since driving frequency can't be zero.

$$\therefore p^2 - w^2 + 2k^2 = 0$$

$$p^2 = w^2 - 2k^2$$

$$p = \sqrt{w^2 - 2k^2} = p_r \rightarrow \text{Resonant frequency}$$

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At this frequency ' ω_r ' the amplitude will be maximum.

Substitute ' ω_r ' in the above expression for amplitude

$$A = \frac{\omega_0}{\sqrt{(\omega^2 - \omega_r^2)^2 + 4K^2\omega^2}}$$

Replace $\omega_r \rightarrow \omega_r = \sqrt{\omega^2 - 2K^2}$

$$\therefore A = \frac{\omega_0}{\sqrt{(\omega^2 - (\sqrt{\omega^2 - 2K^2})^2)^2 + 4K^2(\sqrt{\omega^2 - 2K^2})^2}}$$

$$A_{max} = \frac{\omega_0}{\sqrt{(2K^2)^2 + 4K^2(\omega^2 - 2K^2)}}$$

$$A_{max} = \frac{\omega_0}{2K\sqrt{\omega^2 - K^2}}$$

For a certain value of driving frequency the amplitude will be maximum which is called as amplitude maximum Resonance.

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Velocity Resonance :

The displacement of a driven oscillation is given by

$$x = A \sin(pt - \theta) \rightarrow ①.$$

Substitute for A in eqn ①.

8 Tue

$$x = \frac{f_0}{\sqrt{(w^2 - p^2)^2 + 4k^2p^2}} \sin(pt - \theta)$$

∴ The velocity of the driven oscillator at that instant is

$$v = \frac{dx}{dt} = pA \cos(pt - \theta) \rightarrow ②.$$

9 wed

$$\therefore v = \frac{f_0}{\sqrt{(w^2 - p^2)^2 + 4k^2p^2}} p \cos(pt - \theta)$$

$$\langle v = v_0 \sin(pt - \theta + \pi/2) \rangle \rightarrow ③.$$

$$\sin(\pi/2 + \theta) = \cos \theta.$$

∴ \sin is positive
in 2nd quadrant.

Where $v_0 = \frac{f_0 p}{\sqrt{(w^2 - p^2)^2 + 4k^2p^2}}$ and $\theta = \tan^{-1} \left(\frac{2kp}{w^2 - p^2} \right)$

WKT Eqn ③ $\Rightarrow \omega k A p = f_0 \sin \theta.$

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Eqn ④ $\Rightarrow (\omega^2 - p^2) A = f_0 \cos \theta.$

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Dividing eqn ③ by ④.

$$\frac{f_0 \sin \theta}{f_0 \cos \theta} = \frac{2k_f p}{(\omega^2 - p^2)}$$

$$\tan \theta = \frac{2kp}{(\omega^2 - p^2)} \Rightarrow \boxed{\theta = \tan^{-1} \left(\frac{2kp}{\omega^2 - p^2} \right)}$$

11th

V_0 is known as the velocity amplitude. Thus the velocity amplitude V_0 varies with p .

When $p=0$, $\theta=0$. and maximum when $p=\omega$. Hence at the frequency $p=\omega$ (driving frequency is equal to the natural frequency). of the impressed force, the velocity has the maximum value and we call it ~~180°~~ as velocity resonance.

At velocity Resonance

$$V_0 = \frac{f_0 \omega_0}{\sqrt{(\omega^2 - \omega_0^2) + 4k^2 \omega^2}}$$

$$V_0 = \frac{f_0}{2K} = f_0 \tau$$

$$\text{where } \tau = \frac{1}{2K}$$

$$\theta = \tan^{-1} \frac{2kp}{(\omega^2 - \omega^2)} = \tan^{-1} (\theta)$$

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Notes

$$= \frac{\theta}{2}$$

$$\therefore \theta = \frac{\pi}{2}$$

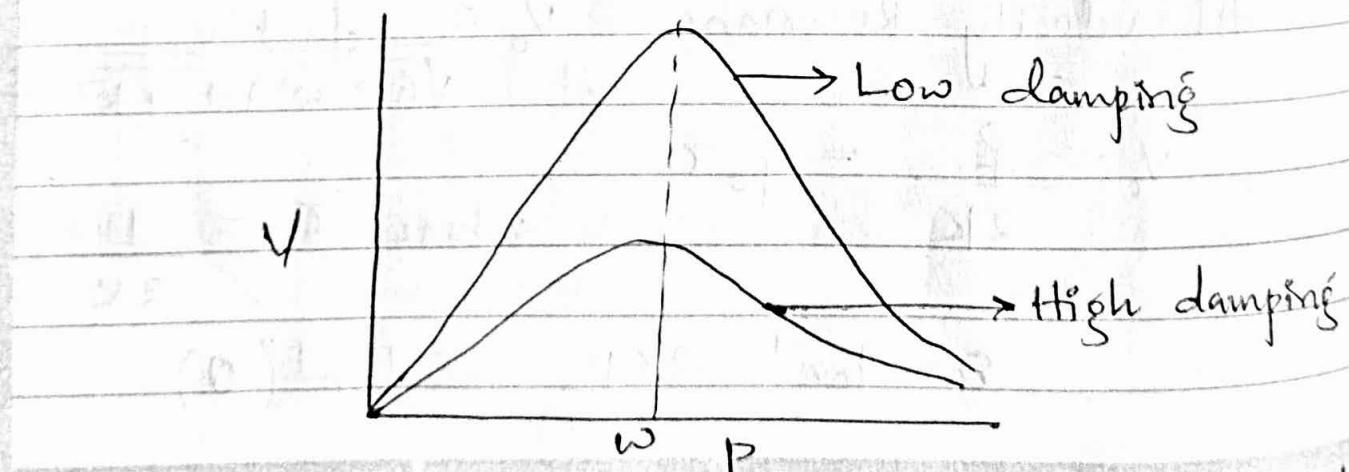
14 mon

Consider eqn ③ $V = V_0 \sin (\omega t - \theta + \frac{\pi}{2})$

$$\therefore \text{Velocity phase constant} = -\theta + \frac{\pi}{2} = -\frac{\pi}{2} + \frac{\pi}{2} = 0.$$

Now, as we know that the displacement at resonance lags behind in phase by $\pi/2$, behind the driving force and as we have just seen
velocity then leads the displacement in phase by $\pi/2$.

Therefore, at resonance, the velocity of driven oscillation is in phase with the driving force. This is the most favourable situation for transfer of energy from the applied force to the oscillator, because the rate of work done on the oscillator by the impressed force is Fv which is always positive for F and v in phase.



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When $P \geq \omega$, the velocity amplitude is smaller than $P = \omega$.

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17 thu

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Sharpness of Resonance :

At resonance, the amplitude of the oscillating system becomes maximum. It decreases from the maximum value with change in frequency of imposed force.

18 fri
The term sharpness of resonance refers to rate of fall of amplitude with change in forced frequency on either side of Resonant frequency.

The rate at which the amplitude changes corresponding to small change in frequency of applied external force.

$$\text{change in Amp} = A_{\max} - A.$$

$$= \frac{f_0}{2kw} - \frac{f_0}{2kp}$$

$$= \frac{f_0}{2kw} - \frac{f_0}{2kp} \quad \therefore w^2 - p^2 = 0 \text{ and}$$

$$= \frac{f_0 p - f_0 w}{2kw p} \quad \text{since } w=p. \text{ Small change in applied freq.}$$

$$= \frac{f_0 (p-w)}{2kw p}$$

$$= \frac{f_0 (p-w)}{2kw p}$$

$$\text{Sharpness} = \frac{\text{change in Amp}}{\text{change in freq of Applied force.}}$$

$$= \frac{f_0 (p-w)}{2kw p} \quad (p-w)$$

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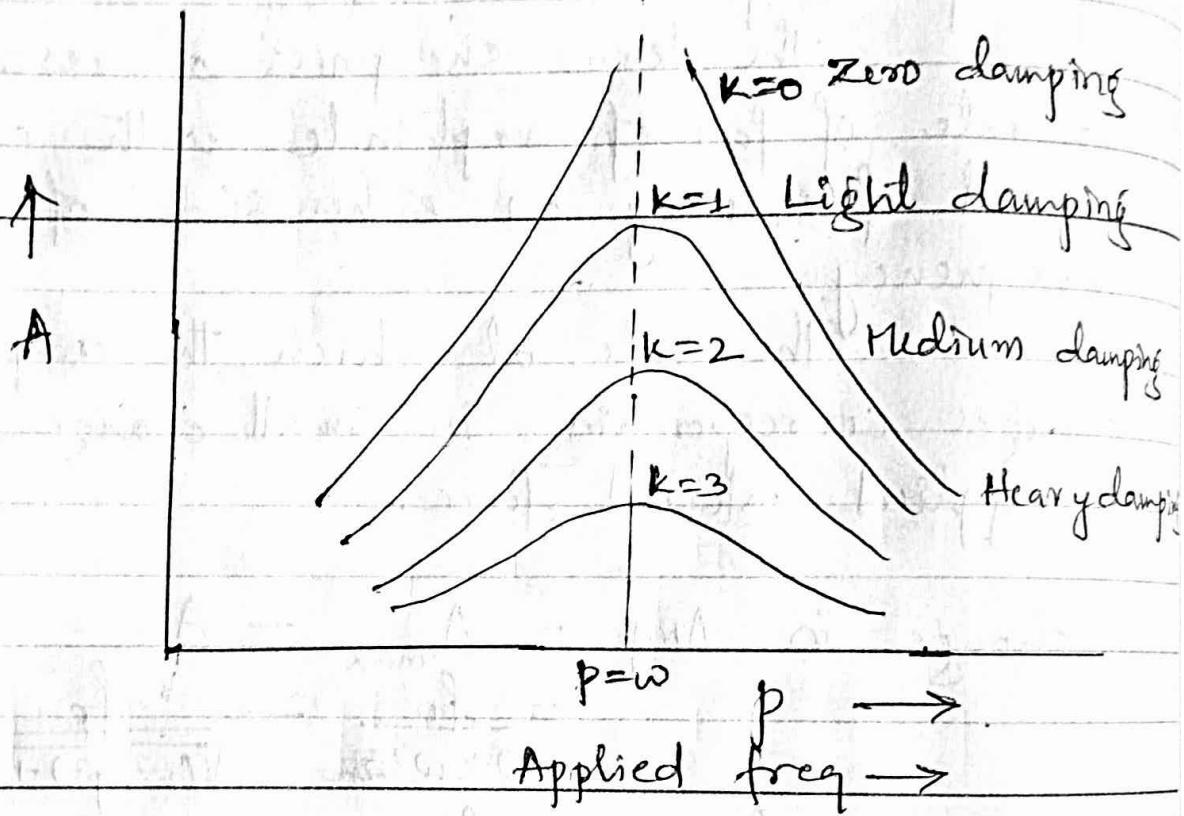
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$$\text{Sharpness} = \frac{\omega}{2k\omega p}$$

Sharpness is inversely proportional to k .

22 tue



23 wed

When the k is large, curves are flat = Resonance is flat

If k is small \rightarrow curves are sharp peak.

If k is zero $\rightarrow A$ is infinity.

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When the damping is low, the amplitude falls very rapidly on either side of Resonant frequency for which the resonance is sharp.

On the other hand, for high damping the amplitude falls off very slowly on either side of resonance.

Resonance is either flat or sharp depending on the damping for the oscillating system is large or small.

Example for flat resonance is observed in the resonance of an air column of a aspirator bottle with tuning fork. Due to its large damping the air column responds with tuning fork over a wide range in the neighbourhood of resonance. Thus in this case, it is actually difficult to predict the exact point ~~of~~ of resonance. and hence the resonance is said to flat.

While in the case of sonometer wire, the damping is small and responds only with one particular frequency i.e its own natural frequency, hence the resonance is sharp.

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Quality factor

Q factor is a measure of sharpness of resonance, which explains how fast energy decays in an oscillatory systems. It's a dimensionless quantity.

The mathematical expression is

$$29 \text{ tue} \quad Q = \frac{\text{Energy stored in oscillator}}{\text{Energy lost per cycles.}} \rightarrow (1).$$

$$\begin{aligned} \text{Energy stored} &= KE + PE \quad \left| \begin{array}{l} x = A \sin(pt - \theta) \\ \frac{dx}{dt} = Ap \cos(pt - \theta) \end{array} \right. \\ &= \frac{1}{2}mv^2 + \frac{1}{2}kx^2 \end{aligned}$$

$$30 \text{ wed} \quad = \frac{1}{2}m [A^2 p^2 \cos^2(pt - \theta)] + \frac{1}{2}k [A^2 \sin^2(pt - \theta)]$$

$$= \frac{1}{2}m \cdot A^2 p^2 \cos^2(pt - \theta) + \frac{1}{2}m \omega^2 A^2 \sin^2(pt - \theta) \rightarrow (2)$$

$$\text{where } \omega = \sqrt{\frac{k}{m}} \Rightarrow k = \omega^2 m.$$

Energy loss per cycle = damping force \times velocity.

$$= T \sqrt{V} \times V.$$

$$= T \sqrt{\frac{dx}{dt} \times \frac{dx}{dt}}.$$

$$= T \sqrt{[A^2 p^2 \cos^2(pt - \theta)]}$$

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$$\text{Where } T = \frac{2\pi}{\omega} = \frac{2\pi}{P}$$

$$2k = \frac{\gamma}{m} \quad \text{and} \quad \tau = \frac{1}{2k} \Rightarrow \gamma = 2km = m \times \frac{1}{\tau}$$

Damping coeff.

Substitute $T = \frac{2\pi}{P}$ and $\gamma = \frac{m}{\tau}$

1 fm

$$\text{Energy loss} = T \gamma [A^2 p^2 \cos^2(pt - \theta)].$$

$$= \frac{2\pi}{P} \cdot \frac{m}{\tau} [A^2 p^2 \cos^2(pt - \theta)] \rightarrow \textcircled{3}.$$

Substitute eqn \textcircled{2} and \textcircled{3} in eqn \textcircled{1}.

$$Q_{\text{sat}} = 2\pi \cdot \frac{1}{2} m A^2 p^2 \cos^2(pt - \theta) + \frac{1}{2} m \omega^2 A^2 \sin^2(pt - \theta)$$

$$\frac{2\pi}{P} \cdot \frac{m}{\tau} [A^2 p^2 \cos^2(pt - \theta)]$$

$$\text{WKT } \sin^2 \theta + \cos^2 \theta = 1; \text{ Avg value of } \cos^2 \theta \text{ or } \sin^2 \theta = \frac{1}{2}$$

$$\therefore Q = 2\pi \cdot \frac{1}{2} m A^2 p^2 \cos^2(pt - \theta) + \frac{1}{2} m \omega^2 A^2 \sin^2(pt - \theta)$$

$$\frac{2\pi}{P} \cdot \frac{m}{\tau} A^2 p^2 \cos^2(pt - \theta)$$

$$\begin{cases} \cos^2(pt - \theta) = \frac{1}{2} \\ \sin^2(pt - \theta) = \frac{1}{2} \end{cases}$$

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$$\begin{aligned}
 Q &= 2\pi \cdot \frac{\frac{1}{4}m\lambda^2 p^2 + \frac{1}{4}m\omega^2 A^2}{\frac{2\pi}{P} \cdot \frac{m}{2} \lambda^2 p^2 \cdot \frac{1}{2}} \\
 &= \frac{\frac{1}{4}m\lambda^2 (p^2 + \omega^2)}{m\lambda^2 p^2} \cdot \frac{2\pi}{P} \\
 &= \frac{\frac{1}{2}(p^2 + \omega^2)}{P} \cdot \frac{2\pi}{\lambda}
 \end{aligned}$$

5 tue

At resonance $\omega = \phi$;

$$\therefore Q = \frac{1}{2} \left(\frac{2\omega^2}{\lambda} \right) \frac{2\pi}{\lambda}$$

$$Q = \omega z$$

6 wed

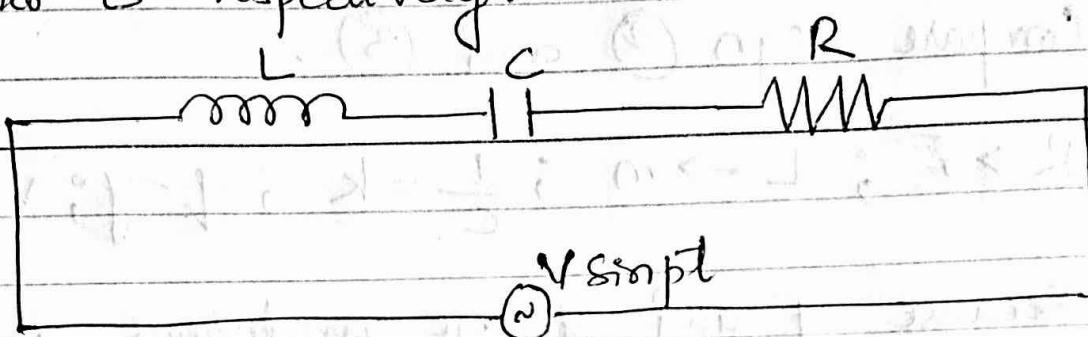
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7 thuLCR Resonance

Consider a series LCR circuit in which V_{sinpt} is the applied AC voltage. The voltage drop across the inductor L , Resistor R and capacitor C respectively.



$$V_L = L \frac{di}{dt} \quad V_R = iR \quad V_C = \frac{q}{C}$$

Since the applied voltage must be equal to the sum of the voltage drop across each of the elements.

$$V_L + V_R + V_C = V_{\text{sinpt}} \rightarrow (1)$$

$$L \frac{di}{dt} + iR + \frac{1}{C} q = V_{\text{sinpt}}$$

$$\text{But } i = \frac{dq}{dt}$$

$$\therefore L \frac{d^2q}{dt^2} + R \frac{dq}{dt} + \frac{1}{C} q = V_{\text{sinpt}}$$

Divide above eqn by L .

$$\frac{d^2q}{dt^2} + \left(\frac{R}{L} \frac{dq}{dt} \right) + \left(\frac{1}{LC} \right) q = \frac{V_{\text{sinpt}}}{L}$$

$\rightarrow (2)$

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The above eqn resembles the eqn of motion of the vibrating system given by.

$$\frac{d^2y}{dt^2} + \left(\frac{r}{m}\right) \frac{dy}{dt} + \left(\frac{k}{m}\right)y = \frac{F}{m} \text{ simply} \rightarrow \textcircled{3}$$

Compare eqn \textcircled{2} and \textcircled{3}.

12 tue

$$R \rightarrow r ; L \rightarrow m ; \frac{1}{C} = k ; q - y ; V \rightarrow F ;$$

Because q and y are analogous quantity.

$$\text{Thus } \frac{1}{LC} = \frac{k}{m} = \omega^2 \Rightarrow \omega = \sqrt{\frac{k}{m}}$$

13 wed Resonance condition, $\phi = \omega = \sqrt{\frac{k}{m}}$

$$\boxed{\frac{1}{LC} = \omega^2}$$

\therefore In LCR circuit at Resonance;

$$\boxed{\omega^2 = \frac{1}{LC}}$$

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