

Rigid Body Dynamics

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Problem Statement

Exploring rigid body dynamics and its use in Advitiy -

- Studying rotational transformation, time derivative with respect to rotating frame
- Understanding Moment of Inertia tensor
- Understanding Angular momentum and torque
- Euler's equation
- Stability of torque free rigid bodies
- Runge Kutta order 4 solver and its implementation for simulating constant torque rigid bodies

Rotational Transformation between Frames

$$\begin{aligned}\mathbf{u}_{x'} &= c_{x'x}\mathbf{u}_x + c_{x'y}\mathbf{u}_y + c_{x'z}\mathbf{u}_z \\ \mathbf{u}_{y'} &= c_{y'x}\mathbf{u}_x + c_{y'y}\mathbf{u}_y + c_{y'z}\mathbf{u}_z \\ \mathbf{u}_{z'} &= c_{z'x}\mathbf{u}_x + c_{z'y}\mathbf{u}_y + c_{z'z}\mathbf{u}_z\end{aligned}\tag{1}$$

$$\mathbf{C} = \begin{bmatrix} c_{x'x} & c_{x'y} & c_{x'z} \\ c_{y'x} & c_{y'y} & c_{y'z} \\ c_{z'x} & c_{z'y} & c_{z'z} \end{bmatrix}\tag{2}$$

$$[C][C]^t = [C]^t[C] = [1]\tag{3}$$

$$\mathbf{r} = x\mathbf{u}_x + y\mathbf{u}_y + z\mathbf{u}_z = x'\mathbf{u}_{x'} + y'\mathbf{u}_{y'} + z'\mathbf{u}_{z'}\tag{4}$$

$$\begin{Bmatrix} x' \\ y' \\ z' \end{Bmatrix} = [C] \begin{Bmatrix} x \\ y \\ z \end{Bmatrix}\tag{5}$$

Transport Theorem

The following equation, known as the Transport Theorem, is a formula for the derivative of any vector \mathbf{r} frame of reference O

$$\left(\frac{d\mathbf{r}}{dt}\right)_{/O} = \left(\frac{d\mathbf{r}}{dt}\right)_{/P} + \mathbf{w}_{P/O} \times \mathbf{r} \quad (6)$$

Moment of Inertia tensor

$$\mathbf{I} = \begin{bmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{bmatrix} \quad (7)$$

$$I_{xx} = \sum_{i=1}^N m_i (r_i^2 - x_i^2) = \sum_{i=1}^N m_i (y_i^2 + z_i^2) \quad (8)$$

$$I_{xy} = - \sum_{i=1}^N m_i (x_i y_i) = I_{yx} \quad (9)$$

Following is applicable only to points for which O is stationary in inertial frame or when O coincides with Center of Mass

$$H_x = I_{xx} \omega_x + I_{xy} \omega_y + I_{xz} \omega_z \quad (10)$$

$$\{H\}_O = [I]_O \{\omega\} \quad (11)$$

Parallel Axis Theorem

When in the shifted frame, $A = (a, b, c)$

$$[I]_B = [I]_A + M \begin{pmatrix} 2(by_c + cz_c) & -(bx_c + ay_c) & -(cx_c + az_c) \\ -(bx_c + ay_c) & 2(cz_c + ax_c) & -(cy_c + bz_c) \\ -(cx_c + az_c) & -(cy_c + bz_c) & 2(ax_c + by_c) \end{pmatrix} + K \quad (12)$$

$$K = M \begin{pmatrix} b^2 + c^2 & -ab & -ac \\ -ab & c^2 + a^2 & -bc \\ -ac & -bc & a^2 + b^2 \end{pmatrix} \quad (13)$$

$$[I]_B = [I]_C + K \quad (14)$$

Rotational transformation of Inertia tensor components-

$$[I'] = [C][I][C]^t \quad (15)$$

Principal Directions

Those exceptional directions such that, when the body rotates about an axis parallel to principal direction, the angular momentum vector has the same direction.

$$I = \begin{bmatrix} I_1 & 0 & 0 \\ 0 & I_2 & 0 \\ 0 & 0 & I_3 \end{bmatrix} \quad (16)$$

To convert a general frame into a principal direction frame, we need to solve the following

$$[I]\{u\} = \lambda\{u\} \quad (17)$$

Directions obtained from solutions of u are the principal directions where,

$$\lambda = H/\omega \quad (18)$$

Angular momentum of a rigid body

$$\mathbf{H}_B = \sum_{i=1}^N \mathbf{b}_i \times m_i \mathbf{v}_i = \sum_{i=1}^N \mathbf{b}_c \times m_i \mathbf{v}_i + \sum_{i=1}^N \mathbf{c}_i \times m_i \mathbf{v}_i \quad (19)$$

$$\mathbf{H}_C = \sum_{i=1}^N \mathbf{c}_i \times m_i \mathbf{v}_i. \quad (20)$$

$$\mathbf{H}_B = \mathbf{b}_c \times \mathbf{P} + \mathbf{H}_c \quad (21)$$

When the moment center is taken as origin of body coordinate system and with zero velocity

$$\mathbf{H}_o = \sum_{i=1}^N m_i \mathbf{r}_i \times (\mathbf{w} \times \mathbf{r}_i) \quad (22)$$

Torque on a system of particles

Total torque about B is defined as -

$$\tau_B = \sum_{i=1}^N \mathbf{r}_i \times \mathbf{F}_i \quad (23)$$

$$\mathbf{H}_B = \sum_{i=1}^N \mathbf{r}_i \times m_i \mathbf{v}_i. \quad (24)$$

$$\tau = \frac{d\mathbf{H}_B}{dt} + \mathbf{v}_B \times \mathbf{P} \quad (25)$$

Thus, torque is rate of change of angular momentum only when moment center is stationary or its velocity is parallel to velocity of the body. A special case of the second condition appears when moment center is COM.

Euler's Equations

For O fixed in body and in inertial space or when $O = \text{COM}$

$$\tau_O = \frac{d\mathbf{H}_O}{dt} \quad (26)$$

$$\tau_O = \left(\frac{d\mathbf{H}_O}{dt}\right)_{rel} + \mathbf{w} \times \mathbf{H}_O \quad (27)$$

$$\tau_O = (I\dot{\mathbf{w}}) + \mathbf{w} \times (I\mathbf{w}) \quad (28)$$

$$\tau_1 = (I_1\dot{w}_1) + (I_3 - I_2)w_2w_3 \quad (29)$$

Torque-free motion

Assuming two of the principal moments of inertia to be same

$$I_1 = I_2 \quad (30)$$

$$I_1 \dot{w}_1 = (I_1 - I_3)w_3w_2 \quad (31)$$

$$I_1 \dot{w}_2 = -(I_1 - I_3)w_3w_1 \quad (32)$$

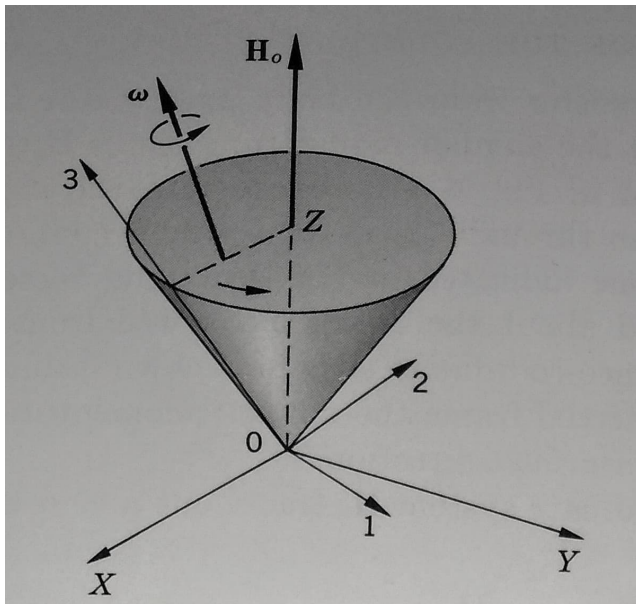
$$I_3 \dot{w}_3 = 0 \quad (33)$$

$$w_1 = w_0 \cos(\gamma - st) \quad (34)$$

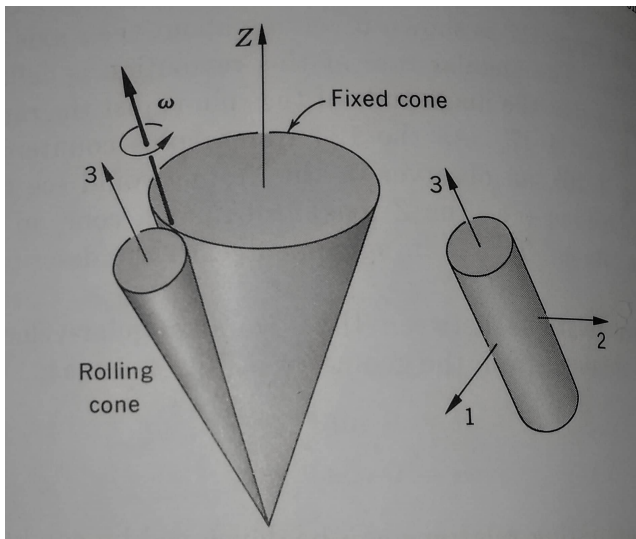
$$w_2 = w_0 \sin(\gamma - st) \quad (35)$$

$$w_3 = \text{constant} \quad (36)$$

Torque-free motion



Torque-free motion



Stability of Torque-free motion

Initially

$$\mathbf{w} = w_0 \hat{\mathbf{k}} \quad (37)$$

Now let's disturb the motion slightly

$$w_x = \delta w_x, w_y = \delta w_y, w_z = w_0 + \delta w_z \quad (38)$$

$$A\delta\dot{w}_x + (C - B)w_0\delta w_y = 0 \quad (39)$$

$$A\delta\ddot{w}_x + (C - B)w_0\delta\dot{w}_y = 0 \quad (40)$$

$$\delta\ddot{w} + k\delta w = 0 \quad (41)$$

if k is positive,

$$\delta w \propto e^{\pm i\sqrt{k}t} \quad (42)$$

Runge Kutta Method

$$\dot{y} = f(y, t) \quad (43)$$

$$y(t_0) = y_0 \quad (44)$$

$$y_{n+1} = y_n + 1/6(k_1 + 2k_2 + 2k_3 + k_4) \quad (45)$$

$$t_{n+1} = t_n + h \quad (46)$$

$$k_1 = hf(t_n, y_n) \quad (47)$$

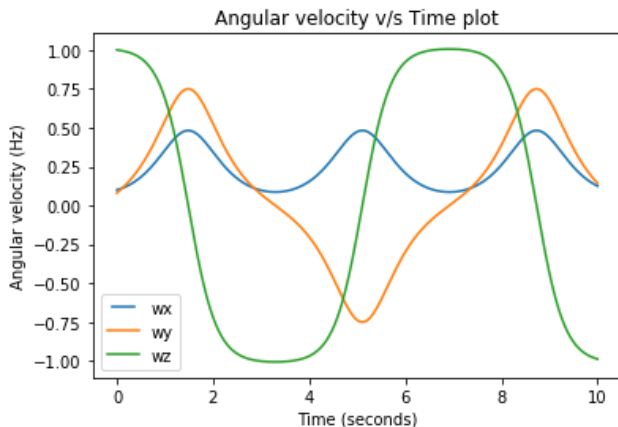
$$k_2 = hf(t_n + h/2, y_n + k_1/2) \quad (48)$$

$$k_3 = hf(t_n + h/2, y_n + k_2/2) \quad (49)$$

$$k_4 = hf(t_n + h, y_n + k) \quad (50)$$

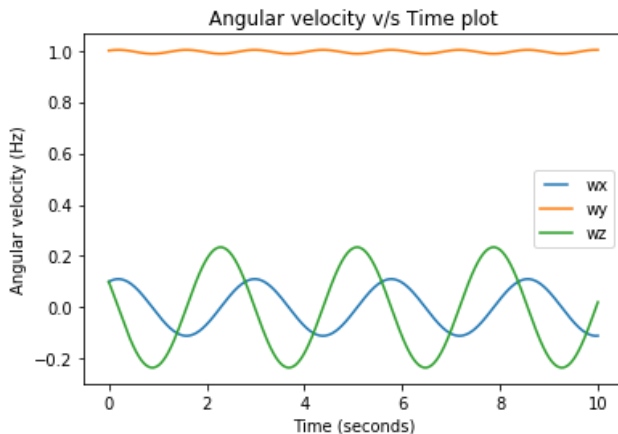
Simulations - intermediate axis

$$\mathbf{I} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 5 & 0 \\ 0 & 0 & 3 \end{bmatrix}, w_0 = \begin{pmatrix} 0.1 \\ 0.08 \\ 1 \end{pmatrix}, M = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \quad (51)$$



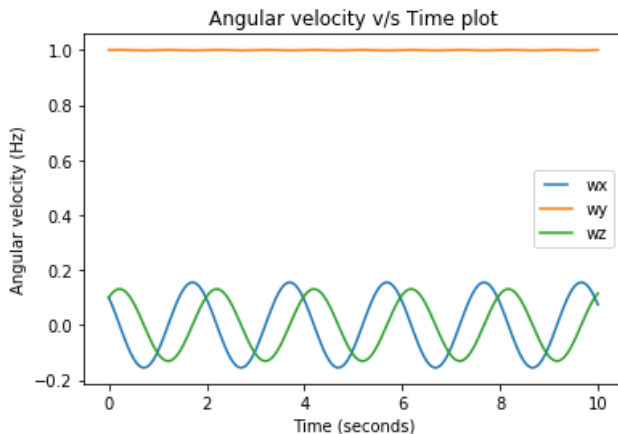
Simulations - major axis

$$\mathbf{I} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 5 & 0 \\ 0 & 0 & 3 \end{bmatrix}, w_0 = \begin{pmatrix} 0.1 \\ 1 \\ 0.1 \end{pmatrix}, M = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \quad (52)$$



Simulations - minor axis

$$\mathbf{I} = \begin{bmatrix} 3 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 4 \end{bmatrix}, w0 = \begin{pmatrix} 0.1 \\ 1 \\ 0.1 \end{pmatrix}, M = \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \quad (53)$$



Thank you