Subject: Noise Control Applications - Module 1 Assignement

Date: February 8, 2025 (Submitted)

Problem 1 - Cut-on Frequencies in Ducts and Pipes

The Matlab code for this problem is listed in Appendix 1.

Problem 1a

The lowest cut-on frequency for a rectangular duct with air flow is given by equation,

$$f_{\text{cut-on}} = 0.5 \cdot \frac{c}{L} \tag{1}$$

where c is the speed of sound in air, 343 $\frac{m}{s}$, and L is the largest side of the rectangular cross-section.

With cross-sectional dimensions of $L_x = 12$ cm and $L_y = 20$ cm, the lowest cut-on frequency for this rectangular duct is,

$$f_{cut-on} = 0.5 \cdot \frac{343 \frac{m}{s}}{0.20 m} = 857.5 \text{ Hz}$$

Problem 1b

The lowest cut-on frequency for a circular duct with air flow with the same cross-sectional area as the rectangular duct in part (a.) can be calculated using equation,

$$f_{\text{cut-on}} = 0.568 \cdot \frac{c}{d} \tag{2}$$

where c is the speed of sound in air, 343 $\frac{m}{s}$, and d is diameter of the circular duct.

The cross-sectional area of the rectangular duct is,

Area
$$_{\text{rectangular duct}} = 0.12 \text{ m} \cdot 0.20 \text{ m} = 0.024 \text{ m}$$

The corresponding diameter for this area is,

$$\mathrm{diameter} = \sqrt{\frac{0.24~\mathrm{m}^2}{\pi}} \cdot 2 = 0.17~\mathrm{m}$$

Using Eq. 2, the lowest cut-on frequency for this circular duct with air flow is,

$$f_{\text{cut-on}} = 0.568 \cdot \frac{1,500 \frac{\text{m}}{\text{s}}}{0.17 \text{ m}} = 1,114.5 \text{ Hz}$$

Problem 1c

The lowest cut-on frequency for this circular duct with water flow can be calculated using Eq. 2,

$$f_{\text{cut-on}} = 0.568 \cdot \frac{1,500 \frac{\text{m}}{\text{s}}}{0.17 \text{ m}} = 4,873.9 \text{ Hz}$$

The lowest cut-on frequency for water is considerable larger than it is for air flow.

Problem 1d

The speed of sound in air is calculated by,

$$c = \sqrt{\gamma \cdot R \cdot T_K} \tag{3}$$

where $\gamma=1.4$ is the ratio of specific heats, $R=287~\frac{J}{kg\cdot K}$ is the gas constant, and T_K is the absolute temperature in Kelvin.

Figure 1 illustrates how the lowest cut-on frequency changes as the air heats from 0° to 500° Celsius.

The square-root relationship between temperature and the speed of sound in air is apparent and governs the behaviour of the cut-on frequency.

Lowest Cut-on Frequency for a Circular Pipe with Air Flow Versus Air Temperature

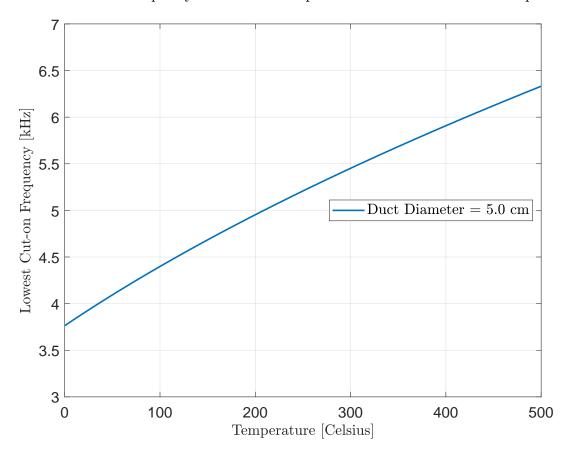


Figure 1: Lowest cut-on frequency for a circular 5 cm diameter duct versus air temperature.

Problem 1e

Question: Are cut-on frequencies higher for a circular or rectangular duct for a given cross-sectional area?

The lowest cut-on frequency is higher for a circular duct than for a rectangular duct for a given cross-sectional area.

For the dimensions given in class, the rectangular duct is not square. This produces a larger dimension and thus a smaller, lowest cut-on frequency. If the rectangular duct is square dimensions on the order of the circular duct diameter with the same cross-sectional area, the cut-on frequencies are approximately equal.

Question: What about in air versus water?

The lowest cut-on frequency is larger for water than for air. The cut-on frequency is proportional to the speed of sound and the speed of sound in water is greater than the speed of sound in air.

Question: What about cold versus hot air?

The lowest cut-on frequency is higher for warm air than it is for cold air.

Problem 2 - Muffler Design Comparison

The Matlab code for this problem is listed in Appendix 2.

For these muffler comparisons, the following assumptions were made:

- There is no flow.
- There are no resistive terms.
- The load impedance was not included because the transmission loss does not require them.
- For Parts b, c, and d, the side branch length offset, L₀, was set to zero.

Problems 2a, 2b, and 2c

Figure 2 shows the transmission loss profiles for a simple expansion chamber, a double-tuned expansion chamber, and a cascaded double-tuned expansion chamber muffler.

The peaks for the simple expansion chamber (red, dashed line) are approximately 22 dB and occur at frequencies with a wavelength that is a quarter of the length of the expansion chamber. Minimal loss occurs at half wavelength multiples.

The addition of the extension tube inside the muffler produces a quarter wavelength resonator. The side branch of Ji (2005; Slide 11, Lecture 3 notes) was used to calculate L_o . For the cascaded double-tuned expansion chamber, the extension tubes produce a secondary quarter wavelength resonator.

As noted in the office hours session, there is no damping which produces artificially high resonances.

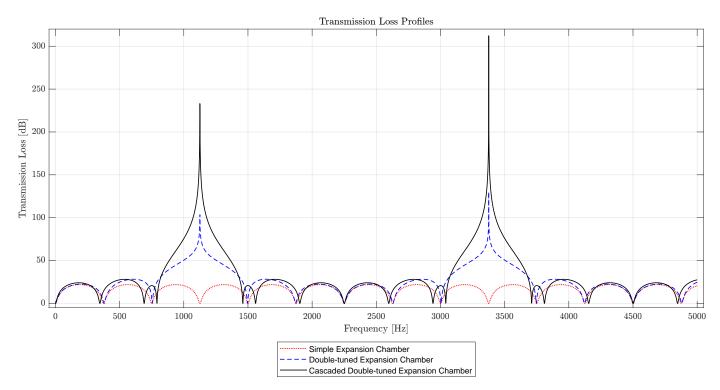


Figure 2: Transmission loss profiles for a simple expansion chamber, a double-tuned expansion chamber, and a cascaded double-tuned expansion chamber mufflers.

Problem 2d

Figure 3 shows the transmission loss profiles for a cascaded double-tuned expansion chamber, and a modified cascaded double-tuned expansion chamber muffler.

Two modifications were made to the original system:

- 1. The left 3" extension tube in the left chamber was shortened to 2" inches, making the respective muffler section 1" longer.
- 2. The left 3" extension tube in the right chamber was lengthened to 4", making the respective muffler section 1" shorter.

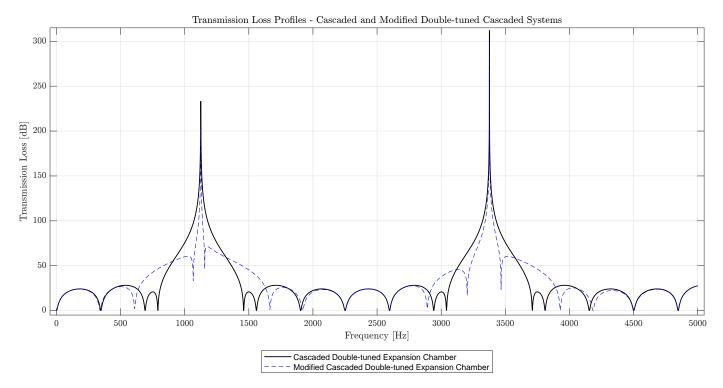


Figure 3: Transmission loss profiles for a cascaded double-tuned expansion chamber muffler and a modified version of this muffler.

These modifications change the symmetry of the cascaded system, and allow the resonate frequencies to be independently changed.

Problem 3 - Bugle Recorder

Diameters of holes should be smaller than a wavelength.

Problem 3a

Problem 3b

Table 1 summarizes the

Note	Frequency [Hz]	Distance from End of Pipe [mm]
C5	523	0
F5	698	0
A5	880	0
С6	1,046	0

Table 1: Measured $L_{Slow,A}$ sound pressure levels. Distance between the SLM and sound source was approximately 0.3 meters. The sound level of the quiet room was approximately 30.0 dB SPL A.

Problem 4 - Intake Duct

Problem 4a

Problem 4b

Problem 4c

Problem 4d

Problem 5 - Intake Duct Silencer

Problem 5a
Problem 5b
Problem 5c
Problem 5d
Problem 5e

```
%% Synopsis
3
4
    % Question 1 - Cut-on Frequencies in Ducts and Pipes
9
   %% Environment
10
    close all; clear; clc;
12 % restored efaultpath;
   \% \ addpath ( \ genpath ( \ '' \ ) \, , \ '-begin ' \ ) \, ; addpath ( genpath ( \ '../40 \ Assignments/00 \ Support ' \ ) , \ '-begin ' \ );
14
16
   \% \ set (\ 0\ ,\ 'DefaultFigurePosition',\ [\ 400\ 400\ 900\ 400\ ]\ );\ \%\ [\ left\ bottom\ width\ height\ ] \ set (\ 0\ ,\ 'DefaultFigurePaperPositionMode',\ 'manual'\ );
18
   set (0, 'DefaultFigureWindowStyle', 'normal');
   set( 0, 'DefaultLineLineWidth', 1.5 );
set( 0, 'DefaultTextInterpreter', 'Latex' );
   format ShortG;
24
   pause ( 1 );
26
    PRINT FIGURES = 0;
28
29
   %% Define Constants and Anonymous Functions
   c\_air = 343; % The speed of sound in air (meters per second).
    c water = 1500; % The speed of sound in water (meters per second).
    gamma \ = \ 1.4\,; \quad \% \ The \ ratio \ of \ specific \ heats \ [unitless].
    R = 287; % The gas constant [Joules per ( kilogram * Kelvin)].
38
40
    h_f_{cut}_{on}_{rectangular}_{duct} = @(c, L) 0.5 .* c./ L;
41
    \% c - The speed of sound.
    \% L - The largest cross-section dimension of the rectangular duct.
45
    h f cut on circular duct = @(c, d) 0.568 .* c ./ d;
46
47
48 \quad \% \quad c \quad - \quad The \quad speec \quad of \quad sound.
49
    \% L - The diameter of the circular duct.
    h speed of sound in air = @(gamma, R, temperature Kelvin) sqrt(gamma .* R .*
        temperature Kelvin );
56 %% Problem 1a
58
   \% The cross-sectional dimensions for the rectangular duct are: Lx = 12 cm and Ly = 20 cm.
59
60 % The largest dimension is Ly = 20 cm or 0.2 m.
61
   % The cut-on frequency is,
62
63
    h_f_cut_on_rectangular_duct( c_air, 0.2 ); % 857.5 Hz (shown in class 858 Hz)
         fprintf( 1, '\n Problem 1a: The lowest cut-on frequency for the rectangular pipe with air is
64
          \%3.1f Hz.\n', h_f_cut_on_rectangular_duct( c_air, 0.2 ));
68
   %% Problem 1b
70
    \% The cross-sectional dimensions for the rectangular duct are: Lx = 12 cm and Ly = 20 cm.
   \% The cross-sectional area of the rectangular duct is 12 cm * 20 cm = 240 cm^2 or 0.024 m^2.
72
    rectangular duct cross sectional area = 0.12 * 0.20; % 0.024 m^2
```

```
74
75 % The diameter of the circulat pipe is,
    circular_duct_diameter = sqrt ( 0.024 / pi ) * 2; % 0.17481 meters
78
   % Check:
79
        \% pi * ( circular duct diameter / 2 ) ^2 CHECKED
80
81
82
    % The cut-on frequency for the circular duct is,
    83
84
        ) with air is \%3.1f Hz.\n', h_f_cut_on_circular_duct( c_air, circular_duct_diameter ) );
85
86
87
   %% Problem 1c
88
89
    \% The cut-on frequency for the circular duct with water is,
    h_f_cut_on_circular_duct( c_water, circular_duct_diameter ); % 4,873.9 Hz
        fprintf( 1, '\n Problem 1c: The lowest cut—on frequency for the circular pipe (of equal area
        ) with water is \%3.1f\ Hz.\n', h_f\_cut\_on\_circular\_duct(c\_water, circular\_duct\_diameter));
    \% The cut-on frequency should be higher because it is proportional to the
    % speed of sound in a given medium.
97
9.8
99
    %% Problem 1d
    fprintf( 1, '\n Problem 1d: See the figure.\n' );
    \texttt{temperature\_range\_celsius} \ = \ 0:0.1:500\,; \quad \% \ \textit{Celsius}
        temperature range kelvin = temperature range celsius + 273.15; % Kelvin
    FONT SIZE = 14;
    figure(); ...
        temperature_range_kelvin ), 0.05 ) ./ 1e3 ); grid on; legend( 'Duct Diameter = 5.0 cm', 'Location', 'East', 'FontSize', FONT_SIZE, 'Interpreter
          'Latex');
           set ( gca, 'FontSize', FONT SIZE );
        xlabel( 'Temperature [Celsius]', 'FontSize', FONT_SIZE );  
% xl = get(gca, 'xlabel'); pxl = get(xl, 'position'); pxl(2) = 1.1 * pxl(2); 
% set(xl, 'position', pxl);
116
        \label( \ \ \ \ \ Cut-on \ \ Frequency \ [kHz]', \ \ \ \ \ FontSize', \ FONT\_SIZE \ );
118
           caption = sprintf( 'Lowest Cut-on Frequency for a Circular Pipe with Air Flow Versus Air
        Temperature\n');
           title ( caption , 'FontSize', FONT SIZE );
        ylim([3 7]);
128
129
   %% Problem 1e
    fprintf( 1, '\n Problem 1e: See Section Problem 1e of the Matlab script for the answers.\n\n');
    % Question: Are cut-on frequencies higher for a circular or rectangular duct for a given cross-
        sectional area?
    \% The lowest cut-on frequency is higher for a circular duct than for a
    \% rectangular duct for a given cross-sectional area.
138
    % For the dimensions given in class, the rectangular duct is not square.
    % This produces a larger dimension and thus a smaller, lowest cut-on
    % frequency.
    \% If the rectangular duct is square dimensions on the order of the circular
    \% duct diameter with the same cross-sectional area, the the cut-on
    % frequencies are approximately equal.
```

```
146
148
    % Question: What about in air versus water?
149
    \% The lowest cut-on frequency is larger with water than air. This due to
     % the fact that the cut-on frequency is proportional to the speed of sound
     % and the speed of sound in water is greater than it is in air.
154
155 % Question: What about cold versus hot air?
156
     % For a circular pipe, the cut-on frequency is higher in warm air than cold
158
     \% air.
159
162 \quad \% \quad Clean-up
     if ( ~isempty( findobj( 'Type', 'figure')))
    monitors = get( 0, 'MonitorPositions');
164
165
              if ( size( monitors, 1 ) == 1 )
    autoArrangeFigures( 2, 2, 1 );
elseif ( 1 < size( monitors, 1 ) )
167
168
169
                   autoArrangeFigures(2,2,1);
170
     end
172
173
     if (PRINT FIGURES == 1)
          saveas \overline{(\ gcf\,,\ 'Cut-on\ Frequency\ Versus\ Temperature-Sunday\,,\ January\ 19\,,\ 2025.pdf\,'\ )\,;}
174
176
     fprintf(1, '\n\n*** Processing Complete ***\n\n');
```

```
%% Synopsis
4
    % Question 2 - Muffler Design Comparison
9
10 \quad \overline{\%\%} \quad Environment
    close all; clear; clc;
   \% restored efault path;
    \% \ \ addpath \left( \ \ genpath \left( \ \ ' \ ' \ \right), \ \ '-begin \ ' \ );
    addpath ( genpath ( '../00 Support'), '-begin');
16
   % set(0, 'DefaultFigurePosition', [ 400 400 900 400 ]); % [ left bottom width height ] set(0, 'DefaultFigurePaperPositionMode', 'manual'); set(0, 'DefaultFigureWindowStyle', 'normal'); set(0, 'DefaultLineLineWidth', 1.5); set(0, 'DefaultTextInterpreter', 'Latex');
18
19
24
    format ShortG;
26
    pause ( 1 );
28
    PRINT FIGURES = 0;
29
   % Constants
    rho0 = 1.21; % Air density (kg per m^3).
    c = 343; % Speed of sound in air (meters per second).
    frequency set = 0:1:5e3; % Hertz
38
40
    %% Dimensions
41
42
43
    convert.inches to meters = 0.0254;
    convert.foot\_to\_meters = 0.3048;
45
46
    dimensions.inlet diameter meters = 2 * convert.inches to meters; % 0.0508 meters
    dimensions.inlet length meters = 6 * convert.foot to meters; % 1.82 meters
47
48
    {\tt dimensions.muffler\_diameter\_meters} = 10 \ * \ {\tt convert.inches\_to\_meters}; \quad \% \ \textit{0.254} \ \textit{meters}
    dimensions.muffler\_length\_meters = 18 * convert.inches\_to\_meters; ~\% ~0.4572 ~meters
    {\tt dimensions.outlet\_diameter\_meters} \ = \ 2 \ * \ {\tt convert.inches\_to\_meters}; \quad \% \ \textit{0.0508} \ \textit{meters}
    dimensions.outlet length meters = 1 * convert.foot to meters; % 0.3048 meters
54
    outlet\_flanged = false;
56
    dimensions.overhang = 3 *convert.inches_to_meters; % 0.0762 meters
58
    segment\_diameters = [ ...
59
         dimensions.outlet_diameter_meters, ...
61
          dimensions.muffler diameter meters, ...
62
          dimensions.inlet diameter meters, ...
64
    h_area_from_diameter = @(d) pi .* d.^2 ./ 4;
67
    segment areas = h area from diameter (segment diameters);
68
69
    segment lengths = [ ...
70
         dimensions.outlet length meters, ...
          {\tt dimensions.muffler\_length\_meters}\;,\;\; \dots
         dimensions.inlet_length_meters, ...
         {\tt dimensions.overhang}\;,\;\; \dots
74
```

```
78
    %% Part a - Simple Expansion Chamber
79
    nFreq = length ( frequency set );

TL = zeros ( nFreq , 1 );
80
81
82
83
    for frequency_index = 1:1:nFreq
84
         f \ = \ frequency\_set \left( \ frequency\_index \ \right);
85
86
        T \text{ total} = [1 0; 0 1];
87
88
89
        T1 = duct\_segment\_transfer\_matrix (~f~,~rhoo~,~c~,~0.3048~,~0.0020268~)~;~~\%~Duct~-~Outlet~
        90
91
92
        97
                                                                                             + (rho0*c)*
99
             % The transmission loss calculation does not require a load impedance.
    end
    TL parta = TL; % The maximum peak value should be about 22 (21.952) dB.
104
    % Expected behaviour:
         1.) 0 dB at 0 Hz.
108
         2.) The transmission loss of a straight duct section is zero; energy out equals energy in.
    max( TL parta ); % 22 dB
114 7\% Part b - Double-tuned Expansion Chamber
    annulus_area_squared_meters = pi/4 * (0.254^2 - 0.0508^2);
116
    L\ o=0; % Assume that the Lo extension is neglible.
118
119
120
    \begin{array}{rll} nFreq &=& length \, ( & frequency \underline{\hspace{0.5cm}} set & ) \, ; \\ TL &=& zeros \, ( & nFreq \, , & 1 \, \end{array} ) \, ; \end{array}
    for frequency index = 1:1:nFreq
126
         f = frequency set(frequency index);
128
        T_{total} = [1 0; 0 1];
        T1 = \; duct\_segment\_transfer\_matrix\left( \;\; f \;, \;\; rho0 \;, \;\; c \;, \;\; \left(0.3048 \;+\; 0.0762\right) \;, \;\; 0.0020268 \;\; \right);
        134
        k = 2 * pi * f / c;
             {\rm Z\_A} \, = \, -1\,{\rm j} * {\rm rh} \, {\rm o} 0 * {\rm c} / \, {\rm annulus\_area\_squared\_meters} * \, {\rm cot} \, \left( \, k \; * \; \left( \; \; 0.0762 \; + \; L\_o \; \right) \; \right) \, ;
                 T2 = [ 1 0; 1/Z_A 1 ];
                     T4 = T2;
139
        T \text{ net} = T5 * T4 * T3 * T2 * T1 * T \text{ total};
        T11 = T \text{ net}(1, 1); \quad T12 = T \text{ net}(1, 2); \quad T21 = T \text{ net}(2, 1); \quad T22 = T \text{ net}(2, 2);
        144
             \% \ \ The \ transmission \ loss \ calculation \ does \ not \ require \ a \ load \ impedance.
    end
148
149
    TL partb = TL;
    % Expected behaviour:
```

76

```
%
                                   0 dB at 0 Hz.
                       1.)
            %
154
                       2.)
                                   0 dB at same locations as a simple expansion chamber.
                        3.) Peaks at 1,125 Hz and 3,376 Hz;
155 %
156
            \% Frequency at which the quarter-wavelength is 0.0762 meters.
            \% 343 / ( 4 * 0.0762 ); \hat{\%} 1,125 Hz.
158
            \% Also work at three-quarter-wavelength.
            % 3 * 1125; % 3,375 Hz
           %% Part c - Cascaded, Double-tuned Expansion Chamber
166
            annulus_area_squared_meters = pi/4 * (0.254^2 - 0.0508^2);
168
169
           Lo = 0; % Assume that the Lo extension is neglible.
172
            nFreq = length ( frequency \_set );
173
                       TL = zeros(nFreq, 1);
174
            for frequency_index = 1:1:nFreq
176
                       f \ = \ frequency\_set \left( \ frequency\_index \ \right);
                                  k = 2 * pi * \overline{f} / c;
178
179
                                            Z A = -1j*rho0*c/annulus area squared meters*cot(k * ( 0.0762 + L o ) );
181
                       T_{total} = [1 0; 0 1];
182
                      T1 = duct\_segment\_transfer\_matrix( f, rho0, c, (0.3048 + 0.0762), 0.0020268 ); % Duct - (0.3048 + 0.0762), 0.0020268 
                      184
185
186
                       T5 = \ duct\_segment\_transfer\_matrix (\ f\ ,\ rho0\ ,\ c\ ,\ 2*0.0762\ ,\ 0.050671\ )\ ; \quad \%\ \mathit{Duct}
187
                      T6 = T2;
                                                  \% Straight Side Branch
                       T7 = T3; \% Duct
                      T8 = T2; % Straight Side Branch
                      T9 = duct segment transfer matrix (f, rho0, c, (1.8288 + 0.0762), 0.0020268); % Duct -
                       Inlet
                      T \quad net \ = \ T9 \ * \ T8 \ * \ T7 \ * \ T6 \ * \ T5 \ * \ T4 \ * \ T3 \ * \ T2 \ * \ T1 \ * \ T \ \ total;
                      T11 = T_net(1, 1); T12 = T_net(1, 2); T21 = T_net(2, 1); T22 = T_net(2, 2);
                      198
            end
            TL partc = TL;
           %% Part d - Cascaded, Double-tuned Expansion Chamber
            nFreq = length ( frequency set );
                      TL = zeros(nFreq, 1);
208
            for frequency_index = 1:1:nFreq
210
                       f = frequency\_set (frequency\_index);

k = 2*pi*f/c;
213
                                            Z A = -1j*rho0*c/annulus area squared meters*cot(k * ( 0.0762 + L o ));
214
215
                       T_{total} = [1 0; 0 1];
216
                      T2 = [1 0; 1/Z A 1]; \% Straight Side Branch
                      T3 = duct\_segment\_transfer\_matrix (\ f \,,\ rho0 \,,\ c \,,\ 0.0076 \,,\ 0.050671\ ) \,; \quad \% \ \mathit{Duct}
                                                  % Straight Side Branch
                       T5 = \ duct \_segment \_transfer \_matrix (\ f , \ rho0 \, , \ c \, , \ 2*0.0762 \, , \ 0.050671 \ ) \, ; \quad \% \ \mathit{Duct}
                      T6 = T2; % Straight Side Branch
                      T9 = duct\_segment\_transfer\_matrix( f, rho0, c, (1.8288 + 0.0762), 0.0020268 ); % Duct - (1.8288 + 0.0762), 0.0020268
```

```
Inlet
          T_{net} = T9 * T8 * T7 * T6 * T5 * T4 * T3 * T2 * T1 * <math>T_{total};
          T11 = T_{net}(1, 1); \quad T12 = T_{net}(1, 2); \quad T21 = T_{net}(2, 1); \quad T22 = T_{net}(2, 2);
          end
     TL partd = TL;
238
    %% Plot Transmission Loss Profiles
    Y LIMITS = \begin{bmatrix} -5 & 320 \end{bmatrix};
     h_figure_1 = figure(); ...
          plot (frequency_set, TL_parta, 'LineWidth', 1.0, 'LineStyle', ':', 'Color', 'r'); hold on; plot (frequency_set, TL_partb, 'LineWidth', 0.9, 'LineStyle', '--', 'Color', 'b'); plot (frequency_set, TL_partc, 'LineWidth', 0.9, 'LineStyle', '-', 'Color', 'k'); grid on;
246
              legend ( ...
                    'Simple Expansion Chamber', ...
                    'Double-tuned Expansion Chamber', ...
                    'Cascaded Double-tuned Expansion Chamber', ...
249
          \label{location} \begin{tabular}{ll} $$ 'Location', "SouthOutside" ); \\ xlabel( "Frequency [Hz]" ); ylabel( "Transmission Loss [dB]" ); \\ \end{tabular}
          title ( 'Transmission Loss Profiles');
253
          %
254
          Ax \ = \ g\,c\,a\ ;
              Ax. XAxis. TickLabelInterpreter = 'latex';
256
               Ax. YAxis. TickLabelInterpreter = 'latex';
          axis ( \begin{bmatrix} -50 & 5e3+50 & Y & LIMITS \end{bmatrix} );
259
261
     Y_LIMITS = [ -5 315 ];
262
     h_figure_2 = figure(); ...
          plot (frequency_set, TL_partc, 'LineWidth', 1.0, 'LineStyle', '-', 'Color', 'k'); hold on; plot (frequency_set, TL_partd, 'LineWidth', 0.6, 'LineStyle', '--', 'Color', 'b'); grid on;
264
               legend ( ...
                    'Cascaded Double-tuned Expansion Chamber', ...
                    'Modified Cascaded Double-tuned Expansion Chamber', ...
                    'Location', 'SouthOutside');
          272
          %
          Ax \ = \ g\,c\,a\ ;
              Ax.XAxis.TickLabelInterpreter = 'latex';
               Ax.YAxis.TickLabelInterpreter = 'latex';
          axis ( \begin{bmatrix} -50 & 5e3+50 & Y_LIMITS \end{bmatrix} );
279
280
281
     % Clean-up
     if ( ~isempty( findobj( 'Type', 'figure')))
    monitors = get( 0, 'MonitorPositions');
283
284
               if (size(monitors, 1) == 1)
               autoArrangeFigures(2, 2, 1);
elseif (1 < size(monitors, 1))</pre>
287
288
                    autoArrangeFigures(2,2,1);
               \mathbf{end}
289
     end
     if ( PRINT FIGURES == 1 )
               exportgraphics (h_figure_1, 'Assignment 1 - Question 2 Figure All TL Profiles.pdf', '
          Append', true );
               exportgraphics (h_figure_2, 'Assignment 1 - Question 2 Figure Comparison TL Plot For
          Cascaded Systems.pdf', 'Append', true );
     end
```

```
2
   %% Synopsis
4
   % Question 3 - Bugle Recorder
9
10 \(\overline{\pi_\infty}\) Note(s)
   \% For the lowest frequency, use 1 duct sgement with an open-ended
   % impedance (see the example of the horn in class).
14
16
   %% Environment
18
19
   close all; clear; clc;
   \% restored efault path;
   % addpath( genpath( '' ), '-begin ' );
addpath( genpath( '../00 Support ' ), '-begin ' );
24
   % set(0, 'DefaultFigurePosition', [400 400 900 400 ]); % [left bottom width height]
   set (0, 'DefaultFigurePaperPositionMode', 'manual');
set (0, 'DefaultFigureWindowStyle', 'docked');
set (0, 'DefaultLineLineWidth', 0.8);
28
   set( 0, 'DefaultTextInterpreter', 'Latex' );
29
   format ShortG;
   pause( 1 );
   PRINT FIGURES = 0;
38
39 %% Constants and Anonymous Functions
40
   rho0 = 1.21; % Density of air (kg per cubic-meter).
41
42
   c = 343; % Speed of sound in air (meters per second).
43
47
48
   \% See Equation 8.34 on page 479 of Bies et al (2024).
   %% Define Shape
54 \quad L_mouth\_piece = 0.09; \quad \% Meters
   pipe.inner diameter = 0.009; % Meters
   pipe.thickness = 0.004; % Meters
58
59
   % The recorder is unflanged.
60
   hole diameter = 0.006; % Meters
61
62
63
64
65 \frac{7}{2} Part a
67
   % The estimated total length of the recorder is 0.325 meters.
68
69
   % The estimated length of the pipe extension is 0.235 meters.
   \% Estimation was done by trial-and-error.
72
73
   pipe\_net\_length = 0.325;
    pipe area = pi*0.009^2/4;
```

```
76
    flanged = false;
78
79
    frequency set = 1:1:2e3;
80
81
    nFreq = length ( frequency set );
82
        A = zeros(nFreq, 1);
83
    \label{eq:formula} \textbf{for} \hspace{0.2cm} \texttt{frequency\_index} \hspace{0.2cm} = \hspace{0.2cm} 1 \colon \! 1 \colon \! n \, \texttt{Freq}
84
85
86
        f = frequency set (frequency index);
87
88
        T \text{ total} = [1 0; 0 1];
89
        T_segment = duct_segment_transfer_matrix( f, rho0, c, pipe_net_length, pipe_area );
91
        T_total = T_segment * T_total;
        Z = open end impedance (f, rhoo, c, 0, pipe area, flanged);
        T11 = T_total(1, 1); T12 = T_total(1, 2);
96
            A( frequency_index ) = -10*log10 ( abs( T11 + T12 / Z )^2 );
97
99
    end
    A_parta = A; clear A;
    % figure(); ...
          104
              legend ( 'C5');
    \% return
108
   %% Part b
    epsilon = 0.006 \ / \ 0.004; \ \% \ \textit{Diameter of the hold divided by diameter of pipe section (1.5)} \ .
114
    switch (3)
116
        case 0 % Original Value
            L \ o = 0.00001; \ \% \ Estimate
118
                L_e = 0.004 + 2*L_o;
            ) / 0.004 * 100 );
        case 1 % Ingard (2010)
            a = 0.006 / 2;
            L_o = (0.6*a + 0.85*a) / 2;
                L_e = 0.004 + 2*L_o;
128
            fprintf( 1, '\nIngard (2010) - Percentage change in pipe thickness: %3.1f%%.\n\n', ( L e
         -0.004 ) /0.004 * 100 );
        case 2 % Kurze and Riedel (2013)
            e = epsilon^2;
            a = 0.006 / 2;
138
            L o = pi*a*(1 - 1.47*e^0.5 + 0.47*e^1.5);
                L_e = 0.004 + 2*L_o;
            fprintf( 1, '\nKurze and Riedel (2013) - Percentage change in pipe thickness: %3.1f%%.\n
        n', ( L_e - 0.004 ) / 0.004 * 100 );
        case 3 % Ji (2005)
144
            a = 0.006 / 2;
            L o = a*(0.9326 - 0.6196*epsilon);
                L_e = 0.004 + 2*L_o;
149
```

```
fprintf(1, '\nJi(2005) - Percentage change in pipe thickness: <math>\%3.1f\%\%.\n\n', (Le-
         0.004 ) / 0.004 * 100 );
         otherwise
              error ( '*** Invalid SWITCH Index *** );
154
    end
    \% \ \ All \ \ holes \ \ covered \ .
158
    \% 523 Hz - Length of pipe is 0.235 meters.
     duct lengths = 0.235/4 * ones(4, 1); \% 523 Hz
162
163
    frequency\_set = 0:1:2e3;
     nFreq = length ( frequency_set );
         TL = zeros(nFreq, 1);
         ZA\_real = zeros(nFreq, 1); ZA\_imaginary = zeros(nFreq, 1);
170
     f = frequency_set( frequency_index );
174
         T \text{ total} = [1 0; 0 1];
176
         Z A = 1j * rho0 * (2 * pi * f) * L e / ( pi*0.006^2/4 );
178
             ZA_{imaginary}(frequency_index) = Z_A;
179
         R\_A = h\_R\_A ( \text{ rho0} \; , \; c \; , \; \text{pi/4*(0.006)^2} \; , \; 2*\text{pi*f/c} \; , \; \text{sqrt( (2 * 1.83\,e-5 ) / ( 2*\text{pi*f * rho0 } ) )} \; , \; \\
         p\overline{i} * 0.\overline{0}0\overline{6}, 2*pi*f, 1.4, 0.3, epsilon, 0);
181
             ZA_real(frequency_index) = R_A;
182
         Z_A = Z_A + R_A;
183
         \sqrt[\infty]{Z} A = R A;
184
185
             Z_A = Z_A * 1e-7;
186
                  T_{Hole} = [ 1 0; 1/Z_A 1 ];
         if (0) % Hole 1-1 open and 0 closed.
              T2 = T Hole;
                 \overline{OFFSET} = 0.0430; \% 696
193
              T1 = duct\_segment\_transfer\_matrix( f, rho0, c, duct\_lengths(4) + OFFSET, pipe\_area ); ~\%
         Duct - Outlet
             T3 = duct\_segment\_transfer\_matrix(f, rho0, c, duct\_lengths(3) - OFFSET, pipe\_area); \%
         Duct
         else
             T2 = [1 0; 0 1]; \% Hole
             T1 = duct_segment_transfer_matrix( f, rho0, c, duct_lengths(4), pipe_area ); % Duct -
         Outlet
             T3 = duct_segment_transfer_matrix( f, rho0, c, duct_lengths(3), pipe_area ); % Duct
         if (0) % Hole 2-1 open and 0 closed.
              T4 = T Hole;
                 OFFSET = 0.0576; \% 880
              T5 = duct\_segment\_transfer\_matrix( f, rho0, c, duct\_lengths(4) - OFFSET, pipe\_area ); ~\%
         Duct
              T4 = [ 1 0; 0 1 ]; \% Hole
              T5 = duct\_segment\_transfer\_matrix(\ f\ ,\ rho0\ ,\ c\ ,\ duct\_lengths(2)\ ,\ pipe\_area\ )\ ; \quad \mbox{\it\% Duct}
208
209
         end
         if (0) % Hole 3 - 1 open and 0 closed.
              T6 = T Hole;
                 \overline{OFFSET} = 0.00100; \%
              T7 = duct\_segment\_transfer\_matrix( \ f \ , \ rho0 \ , \ c \ , \ duct\_lengths(4) - OFFSET, \ pipe\_area \ ); \ \%
         Duct
216
         else
              T6 = [ 1 0; 0 1 ]; \% Hole
218
              \mathrm{T7} = \mathrm{duct} segment transfer matrix (f, rho0, c, duct lengths (1), pipe area ); \% Duct
         end
```

```
T8 = duct_segment_transfer_matrix( f, rho0, c, 0.09, pipe_area ); % Duct - Inlet
224
          T \text{ total} = T8 * T7 * T6 * T5 * T4 * T3 * T2 * T1 * T \text{ total};
          Z = open end impedance (f, rhoo, c, 0, pipe area, flanged);
          T11 \, = \, T\_total\,(1\,,\ 1)\,; \quad T12 \, = \, T\_total\,(1\,,\ 2)\,;
228
229
               A( frequency index ) = -10 \times \log 10 (abs( T11 + T12 / Z)^2);
     end
234
     [\max \text{ value}, \max \text{ index}] = \max(A);
           frequency_set ( max_index )
     figure(); ...
          plot(frequency_set, A); grid on; xlabel('Frequency [Hz]'); ylabel('Amplitude [dB]'); title('Amplification Versus Recorder Length');
238
243 % figure(); ...
             semilogy (\ frequency\_set\ ,\ ZA\_real\ );\ hold\ on;\\ semilogy (\ frequency\_set\ ,\ imag(\ ZA\_imaginary\ )\ );\ grid\ on;\\ legend (\ 'Real\ Part'\ ,\ 'Imaginary\ Par'\ );\\ xlabel (\ 'Frequency\ [Hz]'\ );\ ylabel (\ 'Amplitude\ [dB]'\ );
244 %
245 %
246 %
248
250
251 \frac{\%\%}{Clean-up}
252
254 %
255 %
                 if (size(monitors, 1) == 1)
                      autoArrangeFigures (2, 2, 1);
256 %
                  elseif (1 < \overline{size(monitors, 1)})
     %
     %
258
                       autoArrangeFigures(2,2,1);
259 %
260 % end
264
266
267
    %% Reference(s)
```

```
%% Synopsis
    % Question 4 - Intake Duct
9
   %% To Do
10
   % Check loss profile.
   % Implement the loss Helmholtz resonator.
14
16
   %% Note(s)
18
19
   % Search for FIXMEs.
    % In class note, the areas for the impedance might have bee wrong; switch them?
24
    % Use negative Mach numbers in the equations. The analysis for this case
26
   % is the same as for the horn example. Inlet on the left, outlet on the
    % right.
28
29
   %% Environment
   close all; clear; clc;
34
    % restored efault path;
   \%~addpath (~genpath (~''~)~,~'-begin~'~)~; \\ addpath (~genpath (~'.../40~Assignments/00~Support~'~)~,~'-begin~'~)~; \\
38
   \% \ set (\ 0, \ 'DefaultFigurePosition', [\ 400\ 400\ 900\ 400\ ]\ ); \ \% \ [\ left\ bottom\ width\ height\ ] \\ set (\ 0, \ 'DefaultFigurePaperPositionMode', \ 'manual'\ );
39
40
   set(0, 'DefaultFigureWindowStyle', 'normal');
set(0, 'DefaultLineLineWidth', 1.5);
set(0, 'DefaultTextInterpreter', 'Latex');
41
43
45
   format ShortG;
46
47
    pause ( 1 );
48
49
   PRINT FIGURES = 0;
   % Constants
54
   {
m rho0} = 1.21; % Density of air (kg per cubic-meter).
    c=3\,43; % Speed of sound in air (meters per second).
56
58
   h area = @(diameter) pi * diameter^2 / 4;
59
60
61
   %% Define Shape
62
64 % Source
   67
68
69 % Outlet
70 duct 2.diameter meters = 0.1016; % 4 inches
    duct_2.length_meters = 0.127; % 5 inches
    duct 2. area = h_area ( duct 2. diameter meters );
73
    \% Flanged.
74
```

```
76
              %
             flow_rate_cubic_meters_per_second = 1.04772 / 60; % or 37 cubic-feet per minute
  78
  79
  80
             % return
  81
  82
             %% Part a
  83
  84
              % Calculate the Mach number of the flow in both pip sections.
  85
  86
              duct_1.Mach = -1.0 * flow_rate_cubic_meters_per_second / (pi * duct_1.diameter_meters^2 / 4 ) / c
                           ; \% 0.100 unitless
              duct 2.Mach = -1.0 * flow rate cubic meters per second / (pi * duct 2.diameter meters^2 / 4) / c
  87
                            ; % 0.00628 unitless
  88
              \% return
  89
             %% Part b
  91
              % No flow.
  95
              outlet_flanged = true; % Flanged end.
  96
  98
             TEST FLAG = 1; \% 1: right-to-left.
 99
              frequency \_set = 0:0.1:2.5e3;
                            nFreq = length ( frequency s
TL = zeros ( nFreq , 1 );
                                                                                                                   set );
104
               f = frequency\_set(frequency\_index);
                           T \text{ total} = [1 0; 0 1];
                            if ( TEST FLAG == 1 )
                                         \% \ Right-to-left \ .
                                         118
119
                                          T_inlet = duct_segment_transfer_matrix( f, rho0, c, duct_1.length_meters, duct_1.area );
                                         Z = open\_end\_impedance(f, rho0, c, duct\_2.length\_meters, duct\_2.area, outlet\_flanged);
                            else
                                          \% Left-to-right.
                                          Т
                                                 _outlet = duct_segment_transfer_matrix(f, rho0, c, duct_1.length_meters, duct_1.area);
                                          T contraction = \begin{bmatrix} 1 & 0; 0 & duct_1 \cdot area/duct_2 \cdot area \end{bmatrix};
                                          T_inlet = duct_segment_transfer_matrix( f, rho0, c, duct_2.length_meters, duct_2.area );
                                         Z = open end impedance( f, rho0, c, duct 1.length meters, duct 1.area, outlet flanged );
                            end
                            T \hspace{0.1in} net \hspace{0.1in} = \hspace{0.1in} T \hspace{0.1in} inlet \hspace{0.1in} * \hspace{0.1in} T \hspace{0.1in} contraction \hspace{0.1in} * \hspace{0.1in} T \hspace{0.1in} outlet \hspace{0.1in} * \hspace{0.1in} T \hspace{0.1in} total;
138
139
                           T11 = T \text{ net } (1, 1); \quad T12 = T \text{ net } (1, 2); \quad T21 = T \text{ net } (2, 1); \quad T22 = T \text{ net } (2, 2);
                             if ( TEST FLAG == 1 )
                                         TL(\  \, frequency\_index\  \, )\  \, =\  \, 10\  \, *\  \, log10\,(\  \, abs(\  \, (\  \, T11\  \, +\  \, duct\_1.area*T12/(rho0*c)\  \, +\  \, (rho0*c)*T10, area*T12/(rho0*c) \, +\  \, (rho0*c)*T10, area*T10/(rho0*c) \, +\  \, (rho0*c)*T10/(rho0*c) \, +
                            T21/duct_2.area + T22) / 2)^2;
                                        TL( frequency\_index ) = 10 * log10( abs( (T11 + duct\_2.area*T12/(rho0*c) + (rho0*c)* + (
                            T21/duct 1.area + T22) / 2)^2;
                            end
148
              end \% End: for f = frequency set
149
```

```
TL part b = TL;
         % return
154
        %% Part c
156
         % Flow present (use Mach numbers).
158
         outlet flanged = true; % Flanged end.
159
161
          frequency\_set = 0:0.1:2.5e3;
                   nFreq = length ( frequency \_set );
                           TL = zeros(nFreq, 1);
164
          for frequency index = 1:1:nFreq
                   f = frequency set (frequency index);
168
                   T_{total} = [1 0; 0 1];
                    T\_outlet = duct\_segment\_transfer\_matrix\_flow( f, rho0, c, duct\_2.length\_meters, duct\_2.area, length\_meters, length\_meters, duct\_2.area, length\_meters, length\_2.area, length\_meters, length\_2.area, length\_2.area, length\_2.area, length\_2.area, length\_2.area, length\_2.area, length\_2
                   duct 2.Mach);
174
                   T_expansion = duct_expansion_connection_transfer_matrix( rho0, c, duct_2.area, duct_1.area,
                   duct 1. Mach);
                   T_inlet = duct_segment_transfer_matrix_flow( f, rho0, c, duct_1.length_meters, duct_1.area,
                   duct 1. Mach);
178
179
                  T net = T inlet * T expansion * T outlet * T total;
180
181
                  Z = open end impedance (f, rhoo, c, duct 2.length meters, duct 2.area, outlet flanged);
182
183
                   T11 = T_{net}(1, 1); T12 = T_{net}(1, 2); T21 = T_{net}(2, 1); T22 = T_{net}(2, 2);
184
                            TL(\ \ frequency\_index\ ) = 10 \ * \ log10 (\ abs(\ (\ \overline{T}11\ +\ duct\_1.area*\overline{T}12/(rho0*c)\ +\ (rho0*c)*
185
                  T21/duct_2.area + T22) / 2)^2;
186
187
188
         end \% End: for f = frequency set
189
         TL\_part\_c = TL;
191
         % return
194
        %% Part d
         % Flow present (use Mach numbers).
197
         % Helmholtz resonator in place (between lefthand duct and expansion).
198
         outlet_flanged = true; % Flanged end.
202
         % Resonance
        w_o = 2*pi*136.6; % Estimated from plot.
        \%\ helmholtz\_\ diameter\_\ cavity =
         \% helmholtz_diameter_neck = 1e-3;
                        helmholtz L01 = 0.82 * (1 - 1.33*(helmholtz diameter neck/helmholtz diameter cavity));
208
         %
210
         %
                        epsilon = helmholtz diameter cavity / duct 1.length meters;
                       \% \ \ helmholtz \ \ L02 =
         %
212
213 % helmholtz volume = 1e-3;
214
         % % keyboard
216
         \% \ Q = 2;
218
219 %
220 \quad \% \ R\_A = rho0*c \ / \ Q \ * \ sqrt( \ L\_e \ / \ ( \ pi*helmholtz\_diameter\_neck^2/4 \ * \ helmholtz\_volume \ ) \ );
```

```
frequency set = 0:0.1:2.5e3;
         nFreq = length ( frequency_set );
              TL = zeros(nFreq, 1);
228
229
     for frequency index = 1:1:nFreq
         f = frequency_set( frequency_index );
         T \text{ total} = [1 0; 0 1];
         T outlet = duct segment transfer matrix flow(f, rho0, c, duct 2.length meters, duct 2.area,
         duct 2. Mach);
          T\_expansion = duct\_expansion\_connection\_transfer\_matrix ( rho0 , c , duct\_2.area , duct\_1.area , \\ 
         duct_1.Mach);
239
         \% Z A = 1j*rho0*2*pi*f(frequency index) * L e / ( pi*helmholtz neck diameter^2/4 ) - 1j*
         rho0*c^2/(helmholtz\_volume*2*pi*f(frequency\_index)) + R\_A;
              % T_Helmholtz = [1 0; 1/Z_A 1];
         T inlet = duct segment transfer matrix flow (f, rho0, c, duct 1.length meters, duct 1.area,
         duct 1.Mach);
         \% \ T \ net = \ T \ inlet \ * \ T \ Helmholtz \ * \ T \ expansion \ * \ T \ outlet \ * \ T \ total;
248
249
         T_net = T_inlet * T_expansion * T_outlet * T_total;
         Z = open end impedance(f, rho0, c, duct 2.length meters, duct 2.area, outlet flanged);
         T11 = T_{net}(1, 1); T12 = T_{net}(1, 2); T21 = T_{net}(2, 1); T22 = T_{net}(2, 2);
              TL(frequency\_index) = 10 * log10(abs((\overline{T}11) + duct\_1.area*\overline{T}12/(rho0*c) + (rho0*c)*
         T21/duct \ 2.area + T22 ) / 2 )^2 );
    end \% End: for f = frequency\_set
258
259
    TL part d = TL;
262
    % return
264
    %% Plot
265
    Y_LIMITS = [0 50];
268
    figure(); ...
         plot (frequency_set, TL_part_b); hold on; plot (frequency_set, TL_part_c, '-'); plot (frequency_set, TL_part_d, '-.'); grid on;
         legend ( ...
              'No Flow - Part b', ...
              ^{\scriptscriptstyle |}Flow - Part c^{\scriptscriptstyle |}, ...
274
              'Flow and Resonator - Part d', ...
275
              'Location', 'SouthOutside');
276
         xlabel( 'Frequency [Hz]' ); ylabel( 'Amplitude [dB]' );
title( 'Transmission Loss Profiles' );
278
279
         %
         Ax \ = \ g\,c\,a\ ;
              Ax. XAxis. TickLabelInterpreter = 'latex';
281
              Ax.YAxis.TickLabelInterpreter = 'latex';
282
283
         \% axis ( [ -50 5e3+50 Y_LIMITS ] );
284
         %
286
            ( PRINT FIGURES == 1 )
287
              exportgraphics (gcf, 'Figure TL All Profiles.pdf', 'Append', true );
288
         end
    return
    %% Part c
294
     return
```

23

```
297
       %% Part d
298
299 return
       \%\% Clean-up
302
        if ( ~isempty( findobj( 'Type', 'figure') ) )
   monitors = get( 0, 'MonitorPositions');
   if ( size( monitors, 1 ) == 1 )
      autoArrangeFigures( 2, 2, 1 );
   elseif ( 1 < size( monitors, 1 ) )
      autoArrangeFigures( 2, 2, 1 );</pre>
304
305
306
307
308
                         \mathbf{end}
309
310
        end
311
312
313 fprintf( 1, '\n\n\*** Processing Complete ***\n\n');
314
315
316
317 \frac{\%\%}{Reference(s)}
```

```
%% Synopsis
3
4
   % Question 4 - Intake Duct
9
   %% To Do
10
   % Check loss profile.
13
14
   %% Note(s)
16
   % Search for FIXMEs.
18
19
   % In class note, the areas for the impedance might have bee wrong; switch them?
   \% Use negative Mach numbers in the equations. The analysis for this case
24
    % is the same as for the horn example. Inlet on the left, outlet on the
   \% right.
28 % The S in the diagram (pink area) for the Helmholtz resonator is the
29
   % cross-sectional area of the resonator neck connecting it to the tube.
   % For Lo2, use the value for a quarter-wavelength side tube.
35 % The area expansion ratio is determined using the original duct diameters.
38
39 % Environment
40
   close all; clear; clc;
41
42
   % restored efault path;
43
   % addpath( genpath( ''), '-begin');
45
   addpath ( genpath ( '../40 Assignments/00 Support' ), '-begin');
46
   % set(0, 'DefaultFigurePosition', [ 400 400 900 400 ]); % [ left bottom width height ] set(0, 'DefaultFigurePaperPositionMode', 'manual'); set(0, 'DefaultFigureWindowStyle', 'normal'); set(0, 'DefaultLineUineWidth', 1.5);
47
48
    set( 0, 'DefaultTextInterpreter', 'Latex' );
   format ShortG;
54
   pause( 1 );
56
   PRINT FIGURES = 0;
58
59
61
   %% Define Constants and Anonymous Functions
62
   {
m rho0} = 1.21; % Density of air (kg per cubic-meter).
63
64
   c=3\,43; % Speed of sound in air (meters per second).
67
   h area = @(diameter) pi * diameter^2 / 4;
68
69
70
71
   %% Define Shape
73 % Source
   duct_1.diameter_meters = 0.0254; % 1 inch
74
   duct 1.length meters = 0.1524; % 6 inches
```

```
duct 1.area = h area( duct 1.diameter meters );
 76
 78 % Outlet
 79 \operatorname{duct}_2.\operatorname{diameter}_{\operatorname{meters}} = 0.1016; % 4 \operatorname{inches}
    duct_2.length_meters = 0.127;  % 5 inches
 80
    duct 2. area = h area ( duct 2. diameter meters );
 81
 82
    \% Flanged.
 8.3
 84
 8.5
 86
 87
    flow rate cubic meters per second = 1.04772 / 60; % or 37 cubic-feet per minute
88
 89 % return
 90
91 %% Part a
 92
    \% The -35 dB transmission loss dip | notch is at about 1,150 Hz.
    f = 1150:
96 % Total attenuation required is +35 dB to make it 0 dB.
97
    l = 0.0381; % meters (1.5 inches)
98
    h = 0.0127; % meters (0.5 inches)
    % The total attenuation of the lining from Figure 8.37 (Bies et. al., Fifth Edition),
     \% \ m = (h \ area(\ 0.1016\ ) - h \ area(\ 0.0254\ )) / h \ area(\ 0.0254\ )
    % ASSUMPTION: Cross-section area of lined duct is the same as the inlet.
106
    m = 1;
108
    k = 2 * pi * 1.15 e3;
109
length of expansion chamber = 0.127; % meters
112 \% kL = k*length of expansion chamber
113
    % ASSUMPTION:
114
116
    % Expansion ratio is m = 1;
    % Assume peak difference is 10 dB.
118
    % Total attunation of lining is 10 dB.
119
120 % The attenuation rate is about 10 dB / 0.127 meters or 78.7 dB per meter.
    % return
124 \( \bar{7}\bar{7}\) \( Part \ b \)
126 \quad l = 0.0381; \quad \% \ meters \ (1.5 \ inches)
128
    h = 0.011255; \% meters
          \label{eq:h_validate} \mbox{$h$\_validate} = 0.5* \mbox{$sqrt( (pi*(duct\_2.diameter\_meters - 2*l )^2 ) / 4 ); $\%$ $Same value. } 
133 \quad \frac{\%\% \ Part \ c}{}
135 % The liner thickness ratio is,
136 l / h; % 3.3852 unitless
    % The normalized frequency is,
138
    ( 2 * h ) / ( 343 / \hat{f} ); % 0.075471 unitless
141
143 \frac{7}{2} Part d
145 % Assume the attenuation rate from Part a is 18 dB per meter.
146
    % Use bottom, right subplot (16).
148
149 % The approximate resistivity parameter is 16.
    %% Part e
```

```
% Calculate the flow resistivity.
    R1 = 16 * rho0*c / l; \% 1.74e5 kg / m^3*s
158
159
    %% Placeholder
    % % Flow present (use Mach numbers).
    % Helmholtz resonator in place (between lefthand duct and expansion).
    \% outlet flanged = true; \% Flanged end.
168
    % % Resonance
170 % w \ o = 2*pi*136.6; % Estimated from plot.
172
173 \quad \% \ \% \ helmholtz\_diameter\_cavity \ =
    \% % helmholtz\_diameter\_neck = 1e-3;
174
              helmholtz\_L01 = 0.82 * (1 - 1.33*(helmholtz\_diameter\_neck/helmholtz\_diameter\_cavity));
    % %
176
    % %
    % %
              epsilon = helmholtz\_diameter\_cavity / duct\_1.length\_meters;
    % %
178
              \% \ \ helmholtz \ \ L02 =
179
    % %
180 \% \% helmholtz_volume = 1e-3;
181
     % %
    % % % keyboard
182
183
    % %
184
    \% % helmholtz_L_neck = 1e-3;
    \% \ \% \ Q = 2;
185
    % %
187
    \% \ \ R\_A = \ rho \ 0 * c \ / \ Q * \ sqrt( \ L\_e \ / \ ( \ pi * helmholtz\_diameter\_neck \ ^2/4 * \ helmholtz\_volume \ ) \ );
188
189 %
    %
191
    \% \ frequency\_set = 0:0.1:2.5e3;
           nFreq = length (frequency\_set);
    %
                TL = zeros(nFreq, 1);
194
196 % for frequency index = 1:1:nFreq
197
    %
198
            f = frequency\_set(frequency\_index);
    %
    %
           T_total = [1 0; 0 1];
202
    %
    %
            T outlet = duct segment transfer matrix flow (f, rho0, c, duct 2.length meters, duct 2.area
         , duct 2.Mach);
204
    %
    %
           T\_expansion = duct\_expansion\_connection\_transfer\_matrix(\ rho0\ ,\ c\ ,\ duct\_2\ .area\ ,\ duct\_1\ .area\ ,
          duct_1 . Mach);
    %
         \label{eq:sum_energy} \begin{array}{lll} \% \ Z\_A = 1 \ j * rho0 * 2 * pi * f(frequency\_index) & * L\_e \ / \ ( \ pi * helmholtz\_neck\_diameter ^ 2/4 \ ) & - \ 1 \ j * rho0 * c ^ 2/(helmholtz\_volume * 2 * pi * f(frequency\_index)) & + \ R\_A; \end{array}
    %
    %
               \% T Helmholtz = [1 0; 1/Z_A 1];
    %
    %
    %
            T\_inlet = duct\_segment\_transfer\_matrix\_flow(\ f,\ rho0,\ c,\ duct\_1.length\_meters,\ duct\_1.area,
          duct 1. Mach);
    %
214 %
           \% \ T \ net = \ T \ inlet \ * \ T \ Helmholtz \ * \ T \ expansion \ * \ T \ outlet \ * \ T \ total;
    %
           T\_net = T\_inlet * T\_expansion * T\_outlet * T\_total;
216
217
    %
218
    0%
           Z = open\_end\_impedance (\ f,\ rho0\,,\ c\,,\ duct\_2.length\_meters\,,\ duct\_2.area\,,\ outlet\_flanged\ )\,;
219
220 %
           %
    %
         )*T21/duct 2.area + T22 ) / 2 )^2 );
224
    %
    \% end \% End: for f = frequency\_set
```

```
% TL part d = TL;
228
229 \quad \% \ return
    %% Plot
    % Y LIMITS = [ 0 50 ];
    % figure(); ...
           238
239
    %
           % legend(...
    %
                  'No Flow - Part b', ...
240
           %
                  'Flow - Part \ c', \dots
'Flow \ and \ Resonator - Part \ d', \dots
    %
    %
           %
    %
                  'Location', 'SouthOutside');
243
           %
    %
246 %
    %
           Ax = gca;
    %
               Ax. XAxis. TickLabelInterpreter = 'latex';
248
                Ax. YAxis. TickLabelInterpreter = 'latex';
249
   %
    %
           %
    %
           \% axis ( \begin{bmatrix} -50 & 5e3+50 & Y & LIMITS \end{bmatrix} );
    %
    %
%
            if (PRINT\_FIGURES == 1)
253
                export\overline{g}raphics(\ gcf,\ 'Figure\ TL\ All\ Profiles.pdf',\ 'Append',\ true\ );
254
    %
256
    % return
258
259
    %% Clean-up
    if ( ~isempty( findobj( 'Type', 'figure')))
    monitors = get( 0, 'MonitorPositions');
262
              if ( size( monitors, 1 ) == 1 )
   autoArrangeFigures( 2, 2, 1 );
264
              elseif ( 1 < size ( monitors, 1 ) )
265
                  autoArrangeFigures(2,2,1);
267
268
    end
270
    fprintf(\ 1,\ '\backslash n\backslash n ****\ Processing\ Complete\ ***\backslash n\backslash n'\ );
274
275 \%\% Reference(s)
```