

Masses  $M$   
carré plein homogène  
côté  $l$

Bilan des Forces:

Gravité:  $\vec{F}_g = Mg \vec{e}_x$

→ Par symétrie,  $C$  = centre  
de masse.  $C = (\frac{l}{\sqrt{2}} \cos(\theta) \vec{e}_y$

$+ \frac{l}{\sqrt{2}} \sin(\theta) \vec{e}_y)$



$$x^2 = \frac{l^2}{2} + \frac{l^2}{2}$$

$$x^2 = \frac{l^2}{2} \Rightarrow x = \frac{l}{\sqrt{2}}$$

Moment cinétique

$$\frac{dI_0}{dt} = \sum \vec{O} \vec{P}_i \wedge \vec{F}_i = \vec{OC} \wedge Mg \vec{e}_x = \frac{l}{\sqrt{2}} \sin(\theta) Mg \vec{e}_z$$

$$\frac{dI_0}{dt} = M_0 = I_0 \ddot{\theta} \vec{e}_z$$

$$\text{On a alors } \frac{dI_0}{dt} = M_0 \Leftrightarrow \frac{l}{\sqrt{2}} \sin(\theta) Mg + I_0 \ddot{\theta} = 0$$

$$\Rightarrow \ddot{\theta} + \frac{l \sin(\theta) Mg}{\sqrt{2} I_0} = 0 \quad (1)$$

Moment d'inertie d'un carré plein

Notons  $I_0$  passant par son sommet  $O$  (quelconque)

- Masse  $M$
- épaisseur négligeable
- Longueur / largeur =  $l$ .

$$\text{Par def: } I_0 = \int d^2 m = \int_S d^2 dm$$

$$= \rho \int d^2 dV \quad \text{avec } \rho = \frac{M}{l^2} \text{ pour un carré}$$

$\rho$  = densité surfacique.

$$= \rho \int_0^l \int_0^l d^2 dx dy$$

$$= \rho \int_0^l \int_0^l (x^2 + y^2) dx dy = \rho \int_0^l \left[ \frac{x^3}{3} + xy^2 \right]_0^l dy$$

$$= \rho \left[ \frac{y l^3}{3} + \frac{y^3}{3} \right]_0^l$$

$$= \rho \left[ \frac{l^4}{3} + \frac{l^4}{3} \right] = \rho \frac{2 l^4}{3} = \frac{M}{l^2} \cdot \frac{2 l^4}{3} = \frac{2}{3} M l^2 \quad (2)$$

cnl devient:

$$\ddot{\theta} + \frac{Mg \cdot \frac{3}{2\sqrt{2}} \sin(\theta)}{\frac{2}{3} M l^2} = 0$$

$\downarrow$  petit  $\theta$   
 $\sin(\theta) \approx \theta$

$$\Leftrightarrow \ddot{\theta} + \frac{3}{2\sqrt{2}} \frac{g}{l} \theta = 0 \quad (3)$$

$$\text{On pose } \omega_0^2 = \frac{3}{2\sqrt{2}} \frac{g}{l}$$

$$\Rightarrow \omega = \frac{2\pi}{T} \Rightarrow T = \frac{2\pi}{\omega} = \frac{2\pi}{\sqrt{\frac{3}{2\sqrt{2}} \frac{g}{l}}} = \frac{2\pi}{\sqrt{\frac{3}{2\sqrt{2}}}} \cdot \sqrt{\frac{l}{g}}$$

→ pendule équivalent de longueur  $l' = \frac{2\sqrt{2}}{3} l$ .

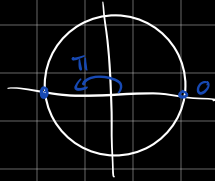
Position d'équilibre:

$$\dot{\theta} = 0$$

On a en général:  $\omega_0^2 \sin(\theta) = 0$

Annulation si:  $\sin(\theta) = 0$

$$\Leftrightarrow \theta = 0 \text{ ou } \pi$$



### Stabilité

$$\theta(t) = \theta_i + \varepsilon_i(t)$$

$$\Leftrightarrow \begin{cases} \ddot{\varepsilon}_1 + \omega_0^2 \sin(\varepsilon_1) = 0 \\ \ddot{\varepsilon}_2 + \omega_0^2 \sin(\pi + \varepsilon_2) = 0 \\ \quad = -\sin(\varepsilon_2) \end{cases} \quad \text{si petites oscillations} \quad \sin(\theta_i) \approx \theta_i$$

$$\begin{aligned} \ddot{\varepsilon}_1 + \omega_0^2 \varepsilon_1 &= 0 \rightarrow p^2 + \omega_0^2 = 0 \Leftrightarrow p^2 = -\omega_0^2 \\ \ddot{\varepsilon}_2 - \omega_0^2 \varepsilon_2 &= 0 \Rightarrow p^2 = \omega_0^2 \end{aligned}$$

$$\Rightarrow p(t) = A e^{i\omega_0 t} + B e^{-i\omega_0 t}$$

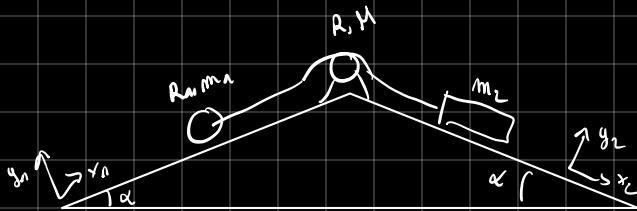
$$= A \cos(\omega_0 t) + B \sin(\omega_0 t)$$

$$\rightarrow p^2 - \omega_0^2 = 0 \Rightarrow p^2 = \omega_0^2$$

$$p = \pm \omega_0$$

$$p(t) = A e^{\omega_0 t} + B e^{-\omega_0 t}$$

$$= A \cosh(\omega_0 t) + B \sinh(\omega_0 t)$$



cylindre roule sans glissement



$$\vec{v}_K = \vec{v}_C + \vec{\omega} \wedge \vec{CK}$$

$$0 = \vec{v}_C - R \dot{\theta}$$

$$\Rightarrow \dot{x}_C = R \dot{\theta}$$

### Moments d'inertie:

Poutre: cylindre plein:

$$\begin{aligned} I_C &= \rho \int_V d^2 dV = \rho \int_0^L \int_0^{2\pi} \int_0^R x^2 dx d\varphi dz \\ &= \frac{R^4}{4} 2\pi \cdot L \cdot \rho \\ &= \frac{R^4}{4L} 2\pi L \cdot \frac{M}{\pi R^2 L} \\ &= \frac{1}{2} M R^2 \end{aligned}$$

• Pour la poutre:  $I_{\text{poutre}} = \frac{1}{2} M R^2$

• Pour le cylindre:  $I_{\text{cyl}} = \frac{1}{2} M_1 R_1^2$

$$\left[ \begin{aligned} \dot{x}_{C,1} &= R_1 \dot{\theta} \\ \dot{x}_{C,\text{poutre}} &= R \dot{\theta} \end{aligned} \right] \quad \frac{\dot{x}_{C,1}}{R_1} = \frac{\dot{x}_{C,\text{poutre}}}{R}$$

$$T = \frac{1}{2} m_A \dot{x}_A^2 + \frac{1}{2} I_A \dot{\theta}_A^2 + \frac{1}{2} M \dot{x}_C^2 + \frac{1}{2} I_P \dot{\theta}_P^2 + \frac{1}{2} m_2 \dot{x}_2^2$$

$$= \frac{1}{2} m_A \dot{x}_A^2 + \frac{1}{2} m_A R_A^2 \dot{\theta} + \frac{1}{2} M \dot{x}_A^2 \left(\frac{R}{R_A}\right)^2 + \frac{1}{2} M R^2 \dot{\theta}^2 + \frac{1}{2} m_2 \dot{x}_2^2 \quad \text{et } \dot{x}_A = \dot{x}_2$$

$$= \dot{x}_A^2 \cdot \left(\frac{1}{2} m_A + \frac{1}{2} m_2\right) + \frac{1}{2} m_A \frac{\dot{x}_A^2}{R_A^2} \cdot R_A^2 + \frac{1}{2} M \dot{x}_A^2 \left(\frac{R}{R_A}\right)^2 + \frac{1}{2} M$$

$$T = \frac{1}{2} (m_A \dot{x}_A^2 + I_A \dot{\theta}_A^2 + I_P \dot{\theta}_P^2 + m_2 \dot{x}_2^2) \quad \left| \begin{array}{l} \dot{x}_A = R \dot{\theta} \quad \dot{\theta}_A = \dot{\theta}_P = \dot{\theta} \\ \dot{x}_A = \dot{x}_2 \\ I_P = \frac{1}{2} M R_i^2 \end{array} \right.$$

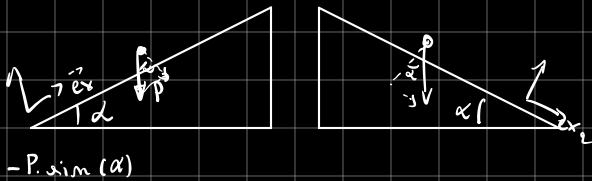
$$\frac{1}{2} \left( m_A \dot{x}_A^2 + \frac{1}{2} m_A R_A^2 \dot{x}_A^2 \cdot \frac{1}{R_A^2} + \frac{1}{2} M R^2 \dot{x}_A^2 \cdot \frac{1}{R^2} + m_2 \dot{x}_A^2 \right)$$

$$= \frac{1}{2} \dot{x}_A^2 \cdot \left( m_A + m_2 + \frac{1}{2} m_A + \frac{1}{2} M \right)$$

$$= \frac{1}{2} \dot{x}_A^2 \left( \frac{3}{2} m_A + m_2 + \frac{1}{2} M \right)$$

$$= M_T$$

On a  $\frac{dT}{dt} = \sum \vec{F}_a \cdot \frac{d\vec{r}_a}{dt} = -m_A g_A \sin(\alpha) \dot{x}_A + m_2 g_C \sin(\alpha) \dot{x}_2 + \underbrace{\vec{F}_L \cdot \vec{x}_L}_{-h \dot{x}_L \cdot \dot{x}_A} = -h \dot{x}_A^2$



Donc on a:  $-m_A g_A \sin(\alpha) \dot{x}_A + m_2 g_C \sin(\alpha) \dot{x}_A - h \dot{x}_A^2$

$\Leftrightarrow \ddot{x}_A M_T = (m_2 - m_A) \sin(\alpha) - h \dot{x}_A$

$\ddot{x}_A = \frac{(m_2 - m_A) \sin(\alpha)}{M_T} - h \dot{x}_A$

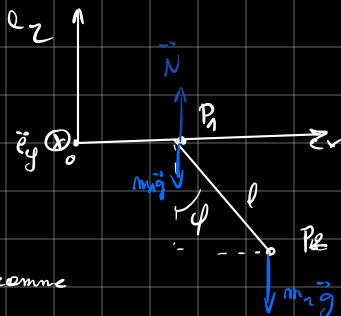
Le pendule d'Euler:

2 Idl. mut selon le plan (x,z)

- $m_1, m_2$
- $l$  const

$S = (O, e_x, e_y, e_z)$  : ref fixe

• Pas de force interne car syst. considéré comme un solide (dist. rel  $P_1$  et  $P_2$  fixe).



$\vec{p}_1 = m_1 \vec{v}_1 \rightarrow \vec{p}_1 = m_1 \dot{x}_1 \vec{e}_x$   
 $\vec{a}_1 = \ddot{x}_1 = \ddot{x}_1 \vec{e}_x$   
 $\vec{p}_2 = m_2 \vec{v}_2$   
 $\vec{a}_2 = \ddot{x}_2 = \frac{d}{dt}(\dot{\vec{r}}_2) = \frac{d}{dt}((x_1 + l \sin(\varphi)) \vec{e}_x - l \cos(\varphi) \vec{e}_y)$

$= \dot{x}_1 + l \dot{\varphi} \cos(\varphi) \vec{e}_x + l \dot{\varphi} \sin(\varphi) \vec{e}_y$   
 $\vec{p}_2 = m_2 \dot{x}_1 + m_2 l \dot{\varphi} \cos(\varphi) \vec{e}_x + m_2 l \dot{\varphi} \sin(\varphi) \vec{e}_y$

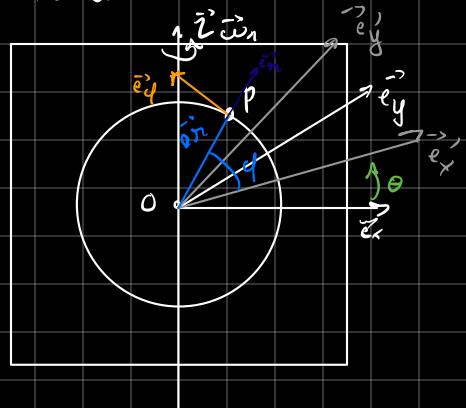
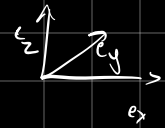
$\vec{p}_{tot} = \vec{p}_1 + \vec{p}_2 = [m_1 \dot{x}_1 + m_2 (\dot{x}_1 + l \dot{\varphi} \cos(\varphi))] \vec{e}_x + m_2 l \dot{\varphi} \sin(\varphi) \vec{e}_y$

$\frac{d\vec{p}_{tot}}{dt} = m_1 \ddot{x}_1 + m_2 \ddot{x}_1 + m_2 l (\ddot{\varphi} \cos(\varphi) - \dot{\varphi}^2 \sin(\varphi)) \vec{e}_x + [m_2 l (\ddot{\varphi} \sin(\varphi) + \dot{\varphi}^2 \cos(\varphi))] \vec{e}_y$   
 $= \sum \vec{F}_{ext} = -m_1 g \vec{e}_y - m_2 g \vec{e}_y + N \vec{e}_y$   
 $= N - (m_1 + m_2) g \vec{e}_y$

projection  
 $\Rightarrow \begin{cases} \ddot{x}_1 - (m_1 + m_2) + m_2 l (\ddot{\varphi} \cos(\varphi) - \dot{\varphi}^2 \sin(\varphi)) = 0 & (1) \\ m_2 l (\ddot{\varphi} \sin(\varphi) + \dot{\varphi}^2 \cos(\varphi)) = N - (m_1 + m_2) g & (2) \end{cases}$

### Bille oscillant sur un cercle tournant

- Anneau circulaire de rayon R.
- $\vec{\omega}_A$  const  $\rightarrow$
- P glisse sans frottement.



$S(0, \vec{e}_x, \vec{e}_y, \vec{e}_z)$  : ref fixe  
 $S'(0, \vec{e}_x', \vec{e}_y', \vec{e}_z')$  : ref lié au plan  
 $S''(P, \vec{e}_x, \vec{e}_y, \vec{e}_z)$  : ref lié à la pat. P  
 avec  $\vec{e}_x \parallel \vec{OP}$

$\rightarrow \vec{\omega}_{S'/S} = \omega_A \vec{e}_z = \dot{\theta} \vec{e}_z$   
 $\rightarrow \vec{\omega}_{S''/S'} = \dot{\varphi} \vec{e}_y'$

Donc  $\vec{\omega}_{S''/S} = \dot{\theta} \vec{e}_z + \dot{\varphi} \vec{e}_y'$   
 $\vec{e}_x = \cos(\varphi) \vec{e}_x' + \sin(\varphi) \vec{e}_z'$   
 $\vec{e}_y = -\sin(\varphi) \vec{e}_x' + \cos(\varphi) \vec{e}_z'$

Calcul de  $\vec{r}$ ,  $\vec{v}$  et  $\vec{a}$

$\vec{r} = R \vec{e}_x = R (\cos(\varphi) \vec{e}_x' + \sin(\varphi) \vec{e}_z')$

$\vec{v} = \frac{d\vec{r}}{dt} = \underbrace{\frac{dR}{dt}}_{=0 \text{ car R const sur disque}} \vec{e}_x + R \vec{\omega}_{S''/S} \wedge \vec{e}_x = R (\dot{\theta} \vec{e}_z + \dot{\varphi} \vec{e}_y') \wedge \vec{e}_x$   
 $= R \dot{\theta} (\vec{e}_z \wedge \vec{e}_x) + R \dot{\varphi} (\vec{e}_y' \wedge \vec{e}_x)$   
 $= R \dot{\theta} (\vec{e}_z \wedge \cos(\varphi) \vec{e}_x' + \sin(\varphi) \vec{e}_z')$   
 $= R \dot{\theta} (\vec{e}_y' \wedge \cos(\varphi) \vec{e}_x' + \sin(\varphi) \vec{e}_z')$

$$= R \dot{\theta} \cos(\varphi) \vec{e}_y' + R \dot{\varphi} \cos(\varphi) \vec{e}_z' - R \dot{\varphi} \sin(\varphi) \vec{e}_x'$$

$$\vec{\omega}_{S',S} \text{ can } \vec{e}_x', \vec{e}_y', \vec{e}_z' \in S'$$

$$\downarrow \begin{cases} \dot{\theta} = \text{const} \\ \dot{\varphi} = 0 \end{cases}$$

$$\begin{aligned} \vec{a} &= \frac{d\vec{v}}{dt} = -R \dot{\varphi} \dot{\varphi} \sin(\varphi) \vec{e}_y' + R \dot{\theta} \cos(\varphi) \dot{\theta} (\vec{e}_z' \wedge \vec{e}_y') \\ &\quad + R \dot{\varphi} \cos(\varphi) \vec{e}_z' - R \dot{\varphi}^2 \sin(\varphi) \vec{e}_z' \\ &\quad - R \dot{\varphi} \sin(\varphi) \vec{e}_x' - R \dot{\varphi}^2 \cos(\varphi) \vec{e}_x' - R \dot{\varphi} \sin(\varphi) \dot{\theta} (\vec{e}_z' \wedge \vec{e}_x') \\ &= \vec{e}_x' \cdot (-R \cos(\varphi) (\dot{\theta}^2 + \dot{\varphi}^2) - R \dot{\varphi} \sin(\varphi)) \\ &\quad - \vec{e}_y' \cdot 2 R \dot{\varphi} \dot{\theta} \sin(\varphi) \\ &\quad + \vec{e}_z' \cdot (R \dot{\varphi} \cos(\varphi) - R \dot{\varphi}^2 \sin(\varphi)) \end{aligned}$$

• Case 1: plan fixe  $\Rightarrow \dot{\theta} = 0$

$$\begin{aligned} \vec{v} &= -R \dot{\varphi} \sin(\varphi) \vec{e}_x' + R \dot{\varphi} \cos(\varphi) \vec{e}_z' \\ &= R \dot{\varphi} \cdot (-\sin(\varphi) \vec{e}_x' + \cos(\varphi) \vec{e}_z') = R \dot{\varphi} \vec{e}_\varphi \end{aligned}$$

$$\begin{aligned} \vec{a} &= \vec{e}_x' \cdot (-R \cos(\varphi) \dot{\varphi}^2 - R \dot{\varphi} \sin(\varphi)) \\ &\quad + \vec{e}_z' \cdot (R \dot{\varphi} \cos(\varphi) - R \dot{\varphi}^2 \sin(\varphi)) \end{aligned}$$

$$\begin{aligned} &= -R \dot{\varphi}^2 \cdot (\cos(\varphi) \vec{e}_x' + \sin(\varphi) \vec{e}_z') + R \dot{\varphi} \cdot (\sin(\varphi) \vec{e}_x' + \cos(\varphi) \vec{e}_z') \\ &= -R \dot{\varphi}^2 \vec{e}_\rho + R \dot{\varphi} \vec{e}_\varphi \end{aligned}$$