

## Dynamics 2 – Tutorial 8

Kinetic Energy and the Power Method for Systems

### Outline Solutions

1(a) Disc Mass = 35kg; R = 0.45m; v = 2.5m/s

For general plane motion

$$KE = \frac{1}{2} M v^2 + \frac{1}{2} I_o \Omega^2$$

But for rolling without slip

$$R\Omega = v$$

Hence

$$KE = \frac{1}{2} M v^2 + \frac{1}{2} I_o \frac{v^2}{R^2}$$

Also, for a uniform disc

$$I_o = \frac{1}{2} M R^2 = 3.544$$

Giving

$$KE = \frac{1}{2} M v^2 + \frac{1}{2} \left( \frac{1}{2} M R^2 \right) \frac{v^2}{R^2} = \frac{3}{4} M v^2 = 164.07 \text{ N}$$

(b) Let M = total mass; m = wheel mass; Mo = body mass

Use result from 1(a) for wheels

$$KE = \frac{1}{2} M_o v^2 + 4 \left( \frac{3}{4} m v^2 \right) = \frac{1}{2} (M_o + 6m) v^2 = 235 v^2$$

(c) M = 250 kg; Io = 45 kgm<sup>2</sup>; R = 0.8 m

Kinetic Energy of System

$$KE = \frac{1}{2} M v^2 + \frac{1}{2} I_o \Omega^2$$

But

$$R\Omega = 2v$$

Hence

$$KE = \frac{1}{2} (250) v^2 + \frac{1}{2} (45) \frac{4v^2}{0.8^2} = 265.6 v^2$$

2. Find downwards acceleration of mass M. Let Q = bearing friction torque on Drum

Net system power is power in from falling mass minus friction power

$$\text{Net power} = Mgv - Q\Omega$$

or

$$= \left( Mg - \frac{2Q}{R} \right) v$$

Kinetic Energy (from 1(c))

$$KE = 265.6v^2$$

Thus Net power =  $\frac{d}{dt}(KE)$  gives

$$\left( Mg - \frac{2Q}{R} \right) v = 531.2 va$$

Gives

$$a = 4.05 \text{ m/s}^2$$

Velocity of M after 6m fall

$$v_2^2 = v_1^2 + 2as = 0 + 2 \times 4.05 \times 6$$

$$\Rightarrow v_2 = 6.97 \text{ m/s}$$

3. Let  $v$  = upwards velocity of load mass

$v$  = downwards velocity of counterweight

$\Omega_1$  = angular velocity of drum (clockwise)

$\Omega_2$  = angular velocity of motor (anti-clockwise)

M = load mass

m = counterweight mass

$I_D$  = Drum Inertia

$I_M$  = Motor Inertia

Kinetic Energy

$$KE = \frac{1}{2}Mv^2 + \frac{1}{2}mv^2 + \frac{1}{2}I_D\Omega_1^2 + \frac{1}{2}I_M\Omega_2^2$$

Kinematics:

$$\Omega_1 R_D = v \quad R_D = 0.6\text{m}$$

$$\Omega_2 = 9\Omega_1$$

Giving

$$KE = \frac{1}{2} \left[ M + m + \frac{I_D}{R_D^2} + 81 \frac{I_M}{R_D^2} \right] v^2 = 4048v^2$$

Power Equation:

$$\text{Net power} = \frac{d}{dt}(KE)$$

Let  $\tau$  = Motor Torque

$$\tau\Omega_2 - Mgv + mgv = 8096va$$

$$\tau\left(9\frac{v}{R_D}\right) - Mgv + mgv = 8096va$$

$v$  cancels

$$9\frac{\tau}{R_D} = 8096a + Mg - mg$$

Gives

$$\tau = 2574 \text{ Nm} \quad (\text{for } a = 3.8 \text{ m/s}^2)$$

This is “slightly” less than for problem using D’Alembert (must be due to rounding errors!)

4. From 1(a) the KE for disc rolling at velocity  $v$

$$KE = \frac{3}{4}Mv^2$$

As the wheel is in pure rolling, the friction force acting at the contact point (where velocity is zero) does not dissipate energy.

The power source is due to the weight

$$\text{Net power} = (Mg \sin\theta)v$$

Hence

$$Mg \sin\theta v = \frac{d}{dt}(KE) = \frac{6}{4}Mva$$

$$\Rightarrow a = \frac{2}{3}g \sin\theta$$

5. Similarly to Question 4, the contact friction forces at the wheels do not dissipate energy.

Power source is due to weight  $(Mg \sin\theta)v$

Power sink is due to brake torque  $Q$  on rear wheels:  $-2Q\Omega = -2Q\frac{v}{R}$

If speed is constant  $\frac{d}{dt}(KE) = 0$ , hence Net Power = 0 giving

$$Mg \sin\theta = 2\frac{Q}{R}$$

$$\Rightarrow Q = 177.7 \text{ Nm}$$

6. Let  $M$  = total Mass;  $m$  = wheel mass;  $I_w$  = wheel inertia;  $I_E$  = Engine Inertia;  $\Omega$  = Wheel Angular Velocity;  $\beta$  = Overall Gear Ratio;  $\beta = \frac{\Omega_E}{\Omega}$ ;  $R$  = wheel rolling radius

From Kinematics,  $R\Omega = v$  hence

$$\text{KE} = \underbrace{\frac{1}{2}Mv^2}_{\text{Translational KE}} + \underbrace{4\left\{\frac{1}{2}I_w\Omega^2\right\}}_{\text{Wheel KE}} + \underbrace{\frac{1}{2}I_E\Omega_E^2}_{\text{Engine KE}}$$

$$\text{KE} = \frac{1}{2}\left[M + 4\frac{I_w}{R^2} + \beta^2\frac{I_E}{R^2}\right]v^2$$

$$\text{KE} = \frac{1}{2}[1278 + 29\beta^2]v^2$$

Power supply from Engine

$$\tau\Omega_E = \tau\beta\frac{v}{R}$$

Use Power Law

$$\tau\frac{\beta}{R}v = \{1278 + 29\beta^2\}va$$

$$\Rightarrow a = \frac{800\beta}{1278 + 29\beta^2}$$

Third gear     $\beta = 5.29$     :-     $a = 2.025 \text{ m/s}^2$

Fifth gear     $\beta = 3.1$     :-     $a = 1.593 \text{ m/s}^2$

Air resistance: Drag force is of the form  $F_D = \text{Area} \times C_D \times \frac{1}{2}\rho v^2$ , so power loss to Air Resistance is proportional to  $v^3$