

# Root Locus

## II

# Outline

## Root Locus Design Method

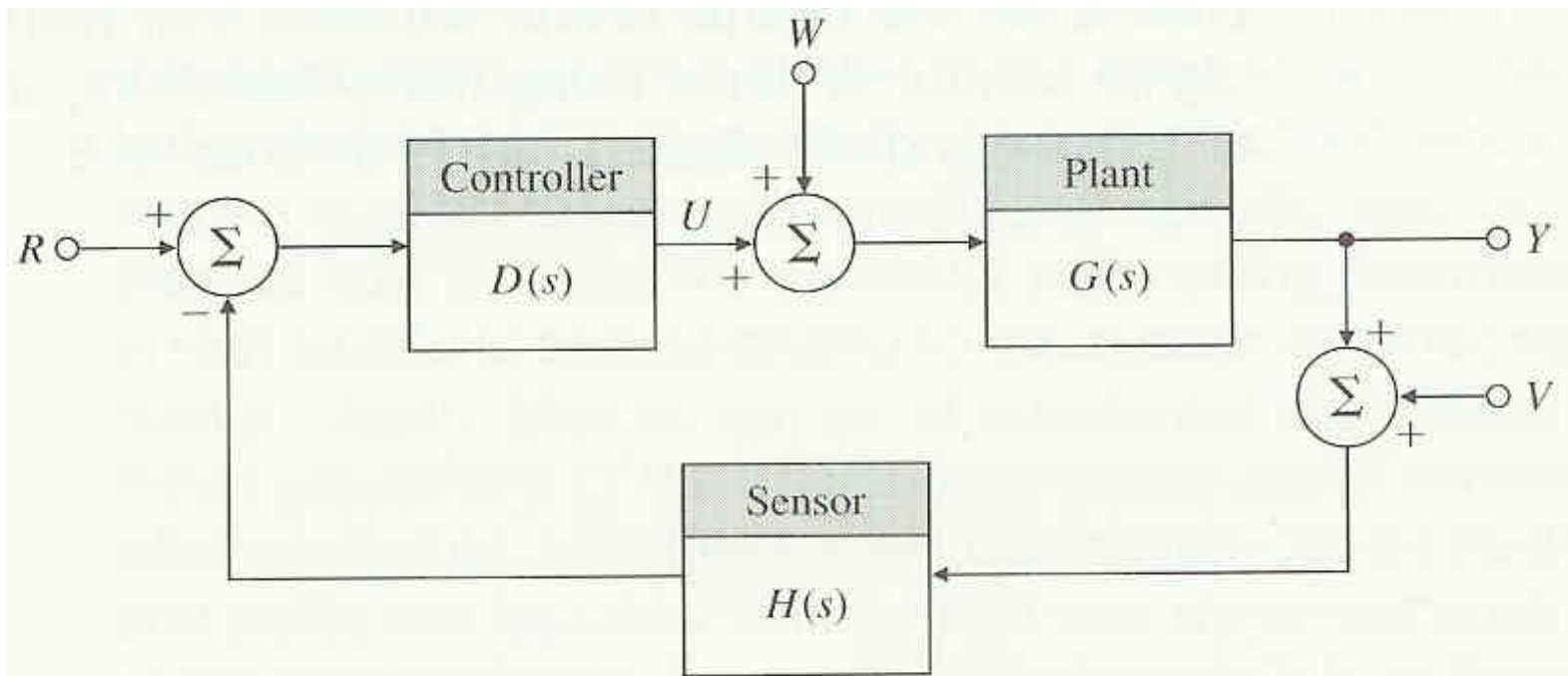
- ❑ Root locus (review)
- ❑ Dynamic Compensation
  - Lead compensation
  - Lag compensation
  - Notch compensation
- ❑ Time Delay (In AnaDig 3)
  - Padé approximation
  - Direct application
- ❑ The Discrete Root Locus

# Root Locus

Closed loop transfer function  $\frac{Y(s)}{R(s)} = T(s) = \frac{D(s)G(s)}{1 + D(s)G(s)H(s)}$

Characteristic equation,  
roots are poles in  $T(s)$

$$1 + D(s)G(s)H(s) = 0$$



# Root Locus

## Definition 1

- The root locus is the values of  $s$  for which  $1+KL(s)=0$  is satisfied as  $K$  varies from 0 to infinity (pos.).

## Definition 2

- The root locus of  $L(s)$  is the points in the  $s$ -plane where the phase of  $L(s)$  is  $180^\circ$ .
  - The angle to the test point from zero number  $i$  is  $\psi_i$ .
  - The angle to the test point from pole number  $i$  is  $\phi_i$ .
  - Therefore,  $\sum \psi_i - \sum \phi_i = 180^\circ + 360^\circ(l-1)$

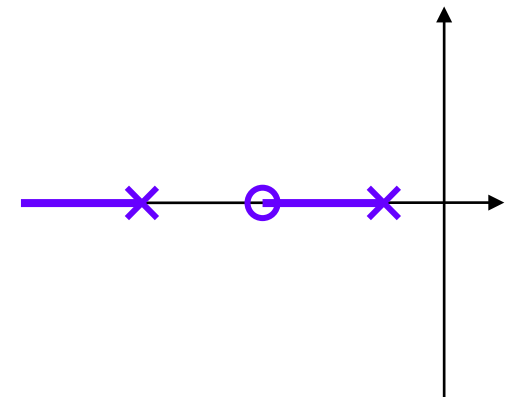
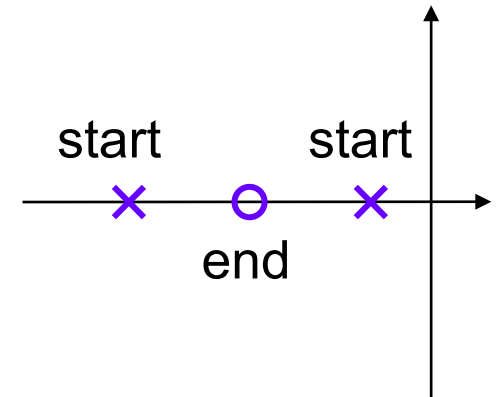
In def. 2, notice,  $L(s) = -1/K$ ,  $\angle(-1/K) = 180^\circ$

# Root Locus

$n$  poles  
 $m$  zeroes

## Rules for sketching a root locus

- Rule 1: The  $n$  branches of the locus start at the poles  $L(s)$  and  $m$  of these branches end on the zeros of  $L(s)$ .
- Rule 2: The loci on the real axis are to the left of an odd number of (real axis) poles plus zeros.

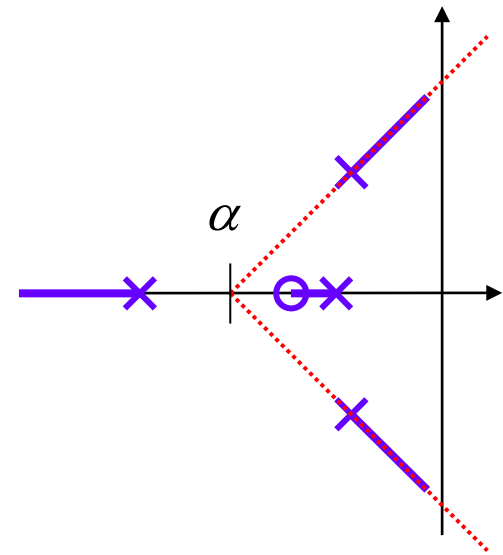


# Root Locus

- Rule 3: For large  $s$ , the loci are asymptotic to lines at angles  $\phi_l$  radiating out from the center point  $s = \alpha$ .

$$\phi_l = \frac{180^\circ + 360^\circ(l-1)}{n-m}, \quad l = 1, 2, \dots, n-m$$

$$\alpha = \frac{\sum p_i - \sum z_i}{n-m}$$



$$\phi_1 = 60 \text{ deg.}$$

$$\phi_2 = 180 \text{ deg.}$$

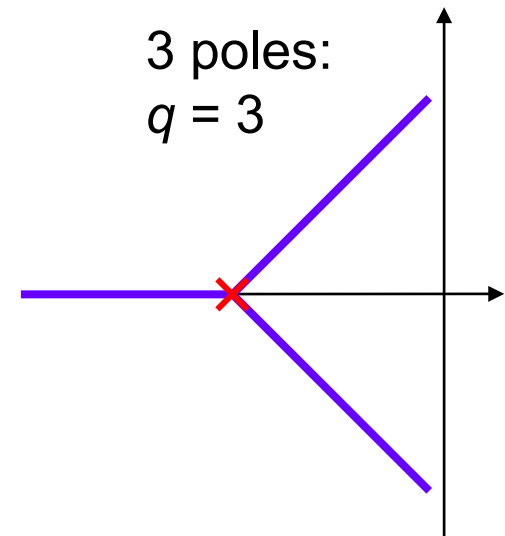
$$\phi_3 = 300 \text{ deg.}$$

# Root Locus

- Rule 4: The angles of departure (arrival) of a branch of the locus from a pole (zero) of multiplicity  $q$ .

$$q\phi_{l,dep} = \sum \psi_i - \sum_{i \neq l} \phi_i - 180^\circ - 360^\circ(l-1)$$

$$q\psi_{l,dep} = \sum \phi_i - \sum_{i \neq l} \psi_i + 180^\circ + 360^\circ(l-1)$$



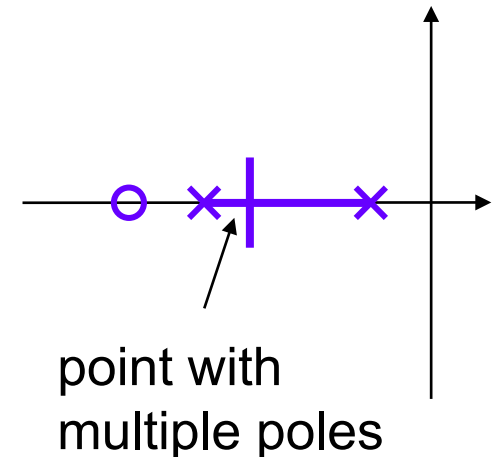
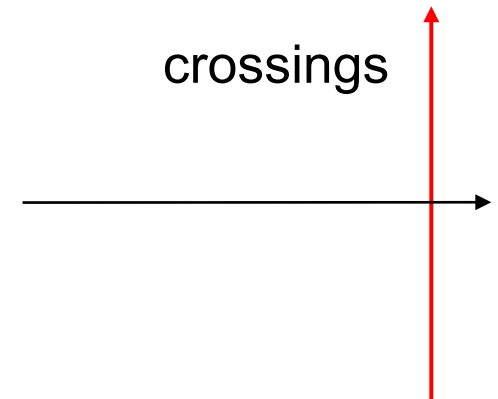
$$\begin{aligned}\phi_{1,dep} &= 60 \text{ deg.} \\ \phi_{2,dep} &= 180 \text{ deg.} \\ \phi_{3,dep} &= 300 \text{ deg.}\end{aligned}$$

# Root Locus

- Rule 5: The locus crosses the  $j\omega$  axis at points found by Routh's criterion.
- Rule 6: Identification of multiple roots on the locus.

$$(1) \quad \left( b \frac{da}{ds} - a \frac{db}{ds} \right) = 0$$

$$(2) \quad \frac{180^\circ + 360^\circ(l-1)}{q}$$





# Dynamic Compensation

## Some facts

- We are able to determine the roots of the characteristic equation (closed loop poles) for a varying parameter  $K$ .
- The location of the roots determine the dynamic characteristics (performance) of the closed loop system.
- It might not be possible to achieve the desired performance with  $D(s) = K$ .

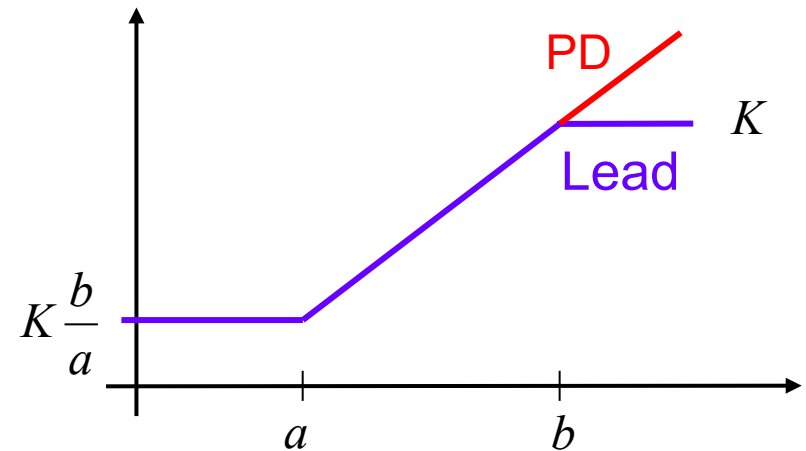
## Controller design using root locus

- Lead compensation (similar to PD control)
  - overshoot, rise time requirements
- Lag compensation (similar to PI control)
  - steady state requirements
- Notch compensation

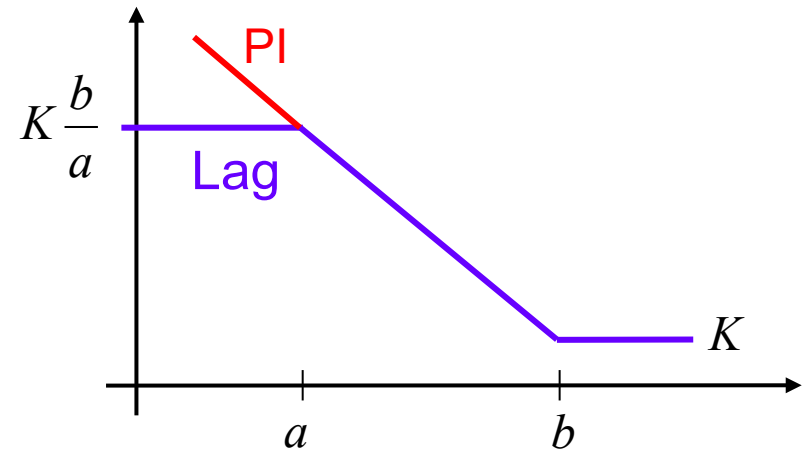
# Dynamic Compensation

$$D(s) = K \frac{s+a}{s+b} = K \frac{b}{a} \frac{(s/a+1)}{(s/b+1)}$$

Lead compensation      $a < b$



Lag compensation      $a > b$



# Lead Compensation

Example system

$$G(s) = \frac{1}{s(s+1)}$$

(blue) with requirement

$$\omega_n \cong 1.9$$

(green circle)

P - control

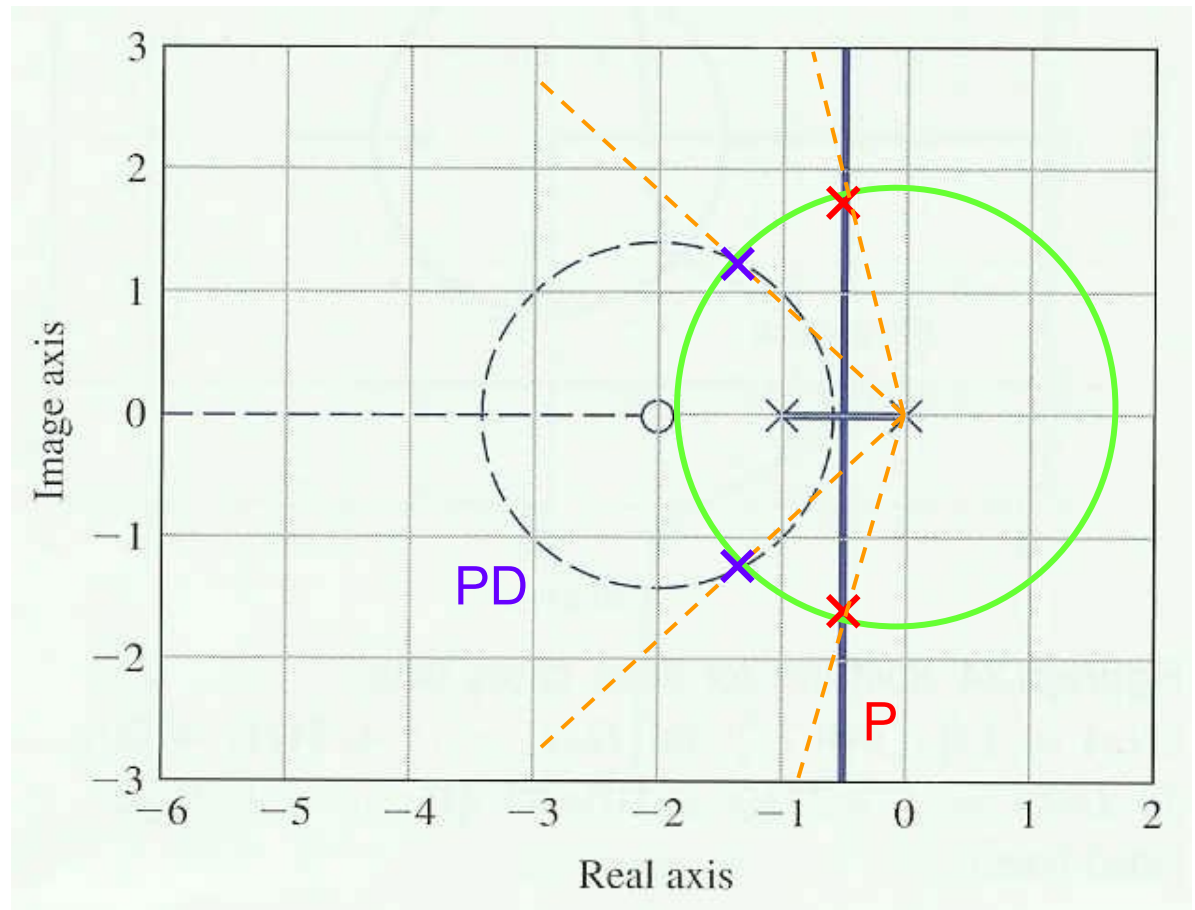
$$D(s) = K$$

(red marks)

PD - control

$$D(s) = K(s+2)$$

(blue marks)

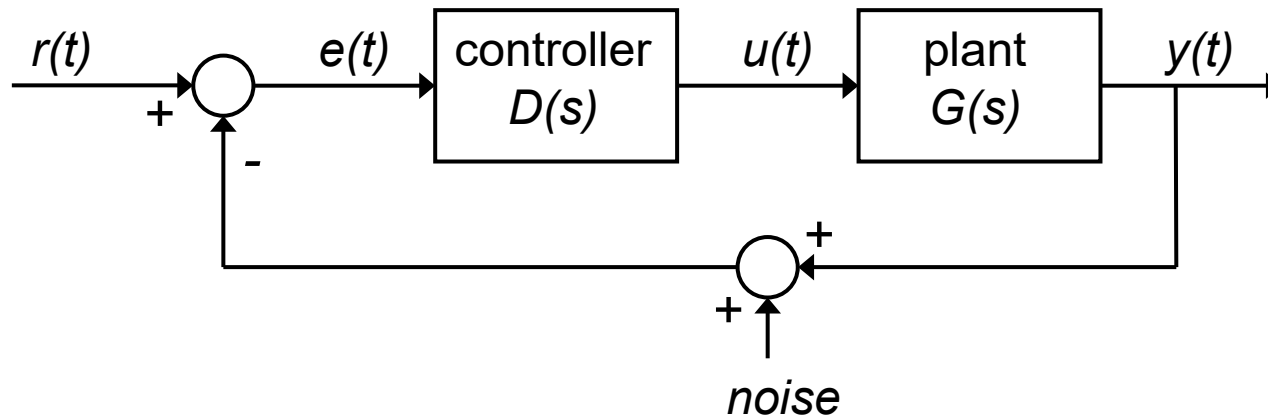


Lower damping ratio with PD !

# Lead Compensation

## PD control

- ❑ Pure differentiation
- ❑ Output noise has a great effect on  $u(t)$
- ❑ Solution: Insert a pole at a higher frequency



# Lead Compensation

$$G(s) = \frac{1}{s(s+1)}$$

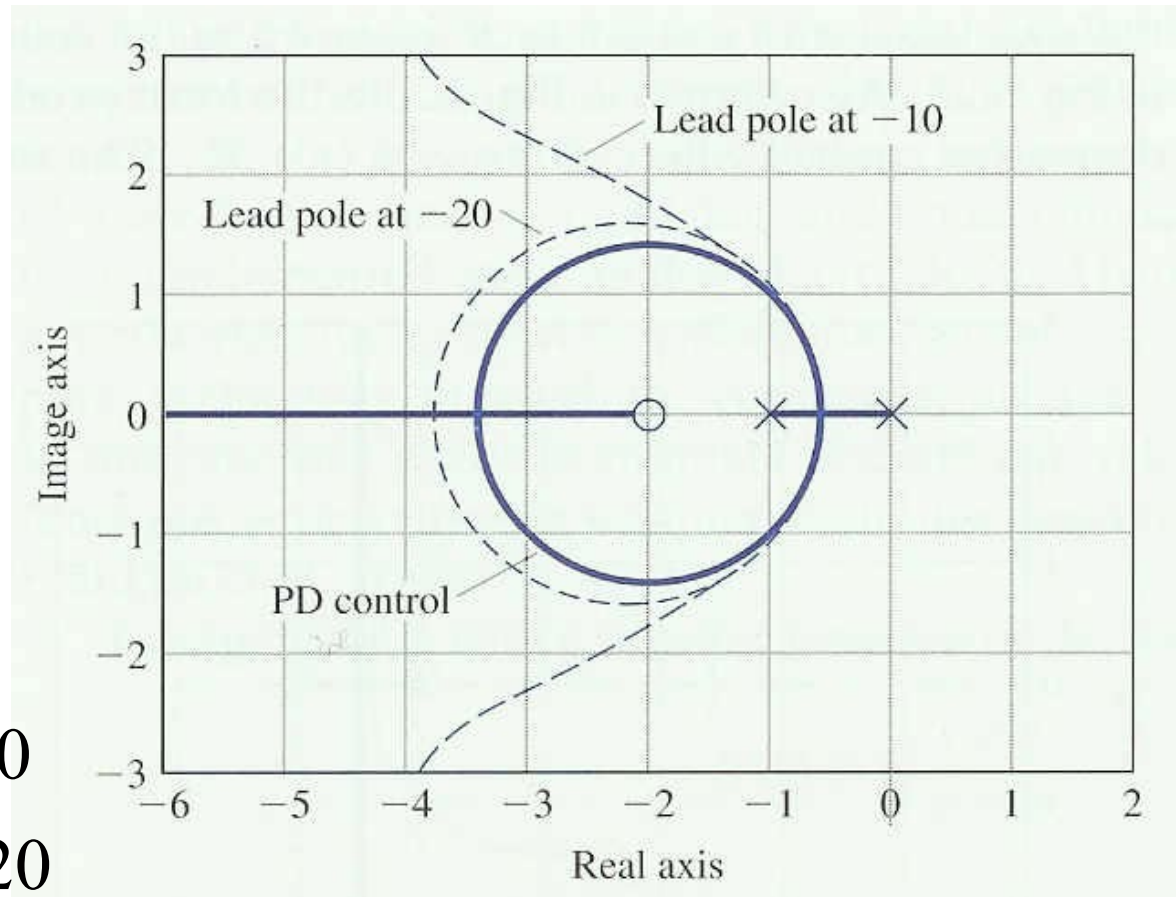
PD - control

$$D(s) = K(s+2)$$

Lead compensation

$$D(s) = K \frac{s+2}{s+p},$$

$$\text{try: } p = \begin{cases} 5 & z = 10 \\ 10 & z = 20 \end{cases}$$



Almost identical for small gains, for calculations look ex 5.4

# Lead Compensation

## Selecting $p$ and $z$

- ❑ Usually, trial and error
- ❑ The placement of the zero is determined by dynamic requirements (rise time, overshoot).
- ❑ The exact placement of the pole is determined by conflicting interests.
  - suppression of output noise
  - effectiveness of the zero
- ❑ In general,
  - the zero  $z$  is placed around desired closed loop  $\omega_n$
  - the pole  $p$  is placed between 5 and 20 times of  $z$

# Lead Compensation

$$G(s) = \frac{1}{s(s+1)}$$

Requirements

Overshoot:  $\zeta \geq 0.5$

Rise time:  $\omega_n \approx 7$

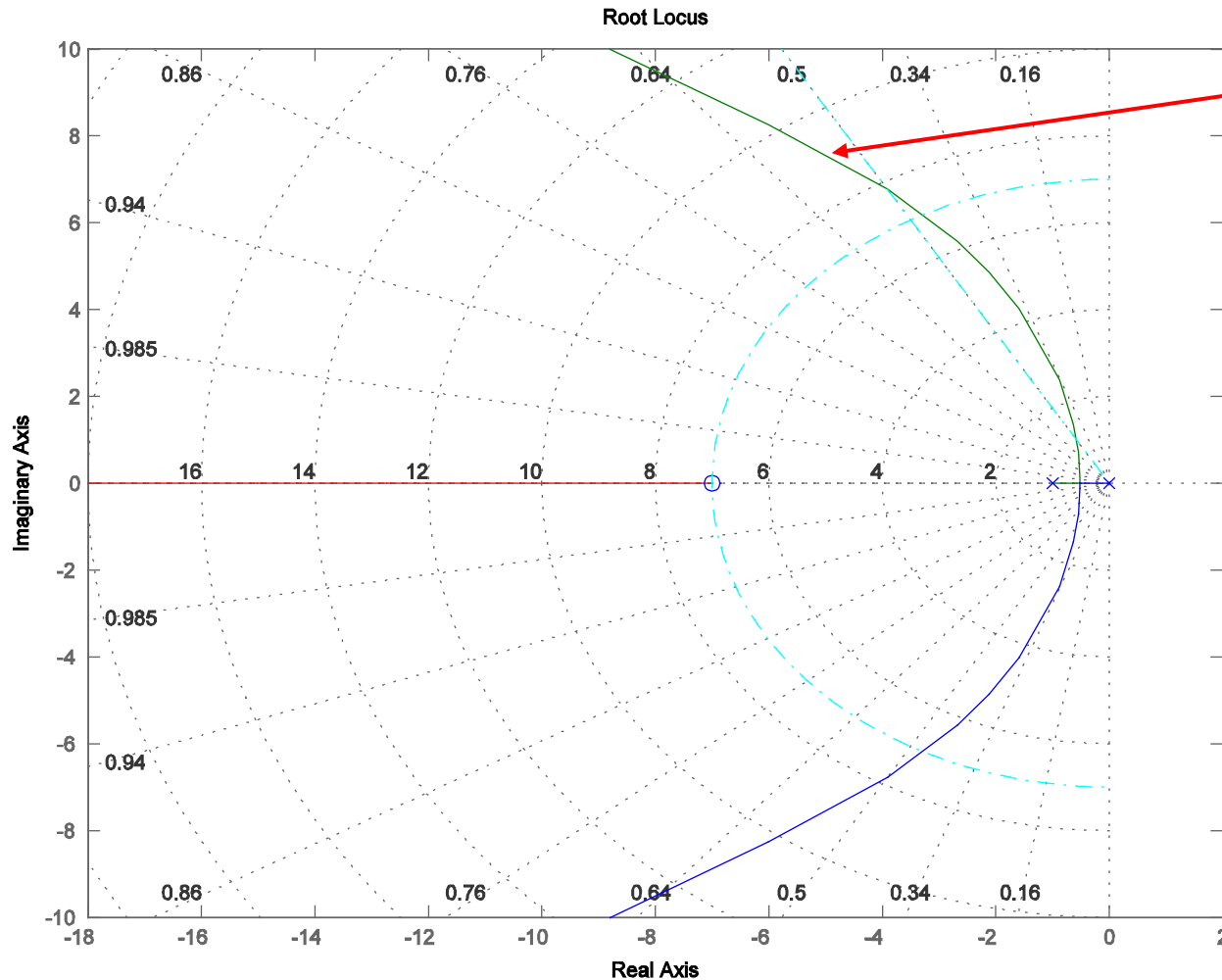
Lead compensation

$$D(s) = K \frac{s+z}{s+p},$$

## Design approach (A)

- ❑ Initial design  $z=\omega_n$ ,  $p=5z$
- ❑ Noise: additional requirement,  $\max(p)=20z$
- ❑ Iterations
  - First design:  $z=\omega_n$ ,  $p=5z$  (neglecting the add. req.)
  - Second design:  $z=\omega_n$ ,  $p=20z$
  - Third design: new  $z$ ,  $p=20z$

# Lead Compensation



Possible  
pole location

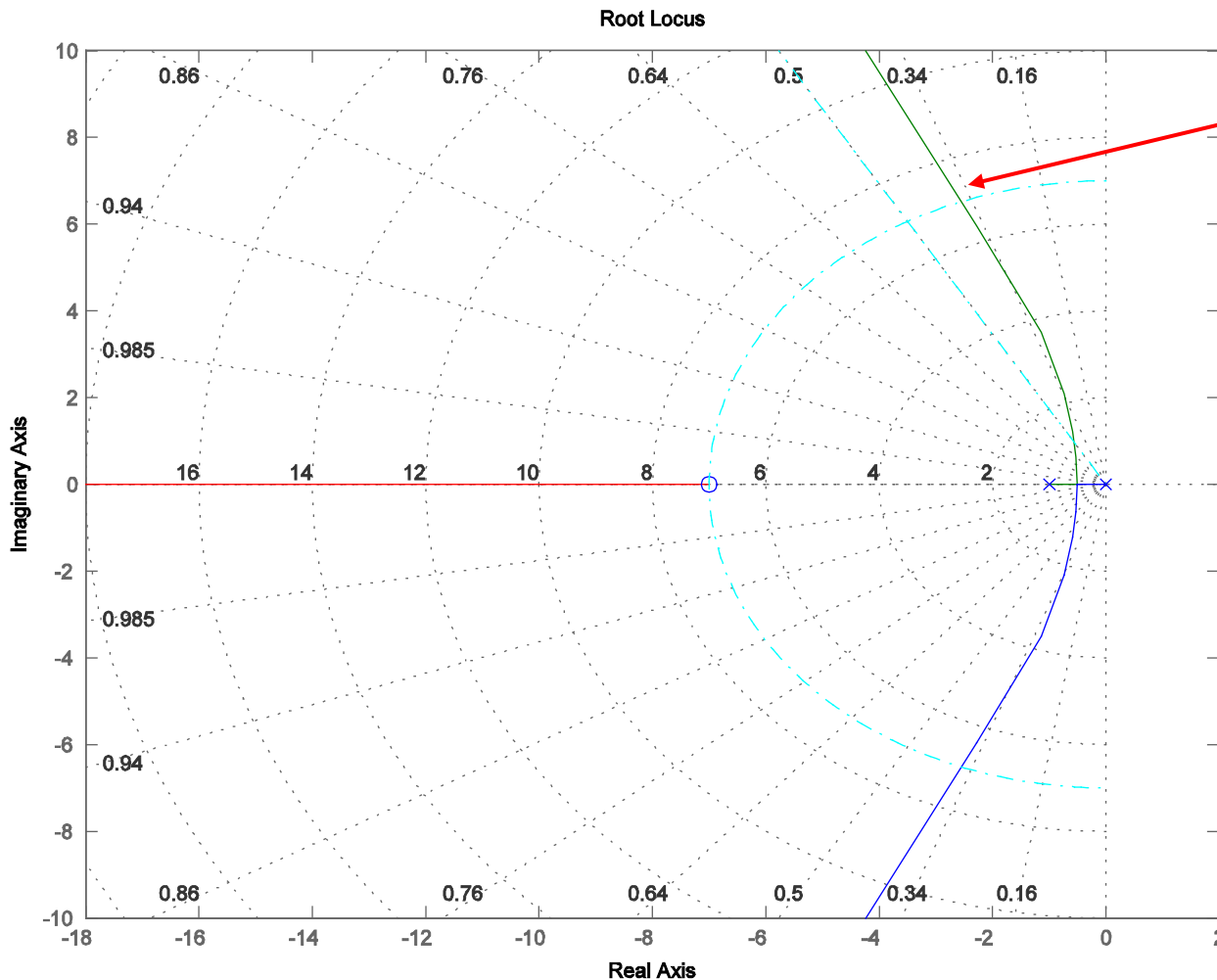
First design:  
 $z=wn = 7$

$p=5z=35$

Matlab – design 1  
`sysG = tf([1],[1 1 0])`  
`sysD = tf([1 7],[1 5*7])`  
`rlocus(sysD*sysG)`



# Lead Compensation

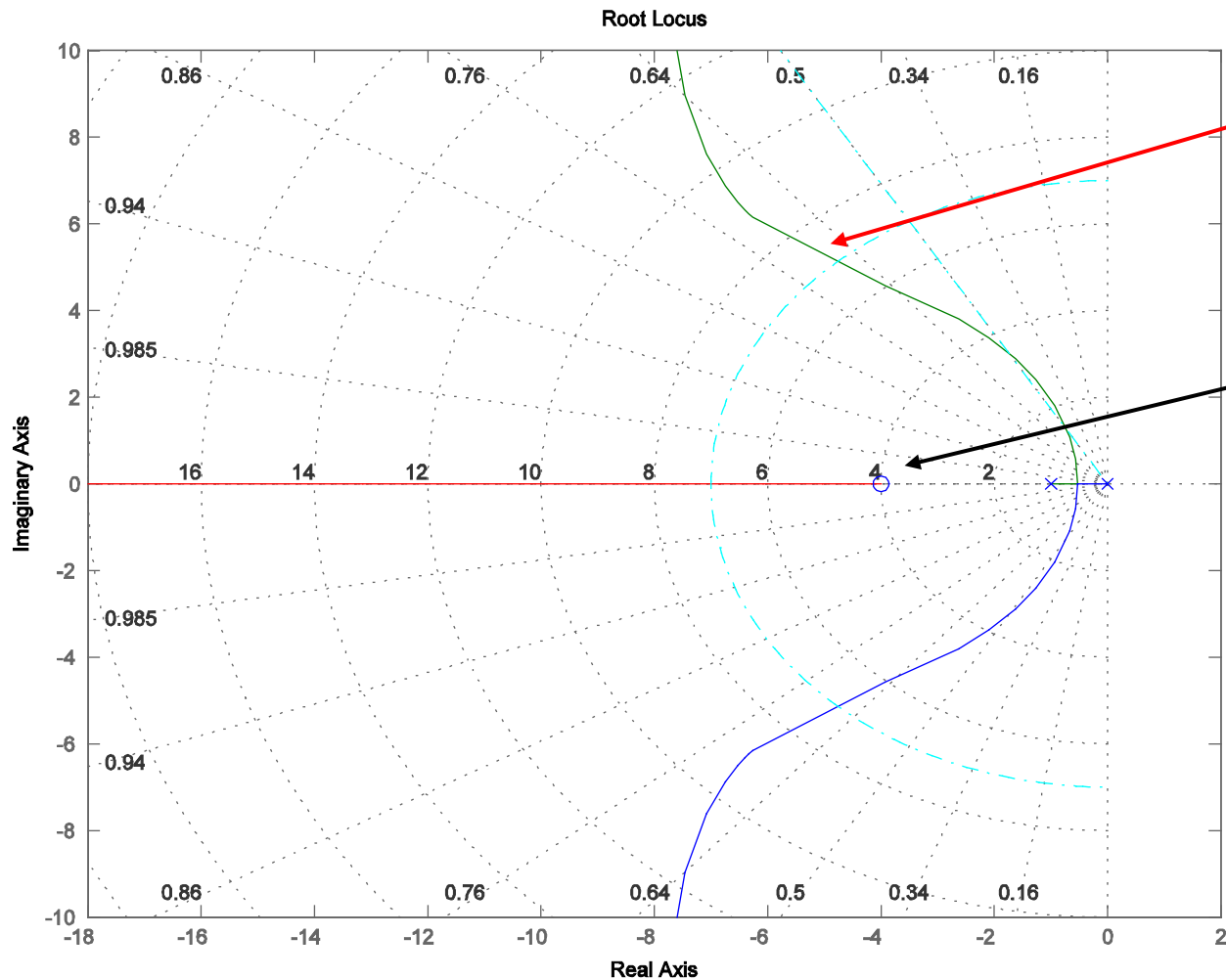


No possible  
pole location !

Second design:  
 $z=wn$ ,  $p=20z$

Matlab – design 2  
`sysG = tf([1],[1 1 0])`  
`sysD = tf([1 7],[1 20*7])`  
`rlocus(sysD*sysG)`

# Lead Compensation



Possible  
location

New zero  
location

Matlab – design 3  
 $\text{sysG} = \text{tf}([1],[1 \ 1 \ 0])$   
 $\text{sysD} = \text{tf}([1 \ 4],[1 \ 20*4])$   
 $\text{rlocus}(\text{sysD}*\text{sysG})$

# Lead Compensation

$$G(s) = \frac{1}{s(s+1)}$$

Requirements

Overshoot:  $\zeta \geq 0.5$

Rise time:  $\omega_n \approx 7$

Lead compensation

$$D(s) = K \frac{s+z}{s+p},$$

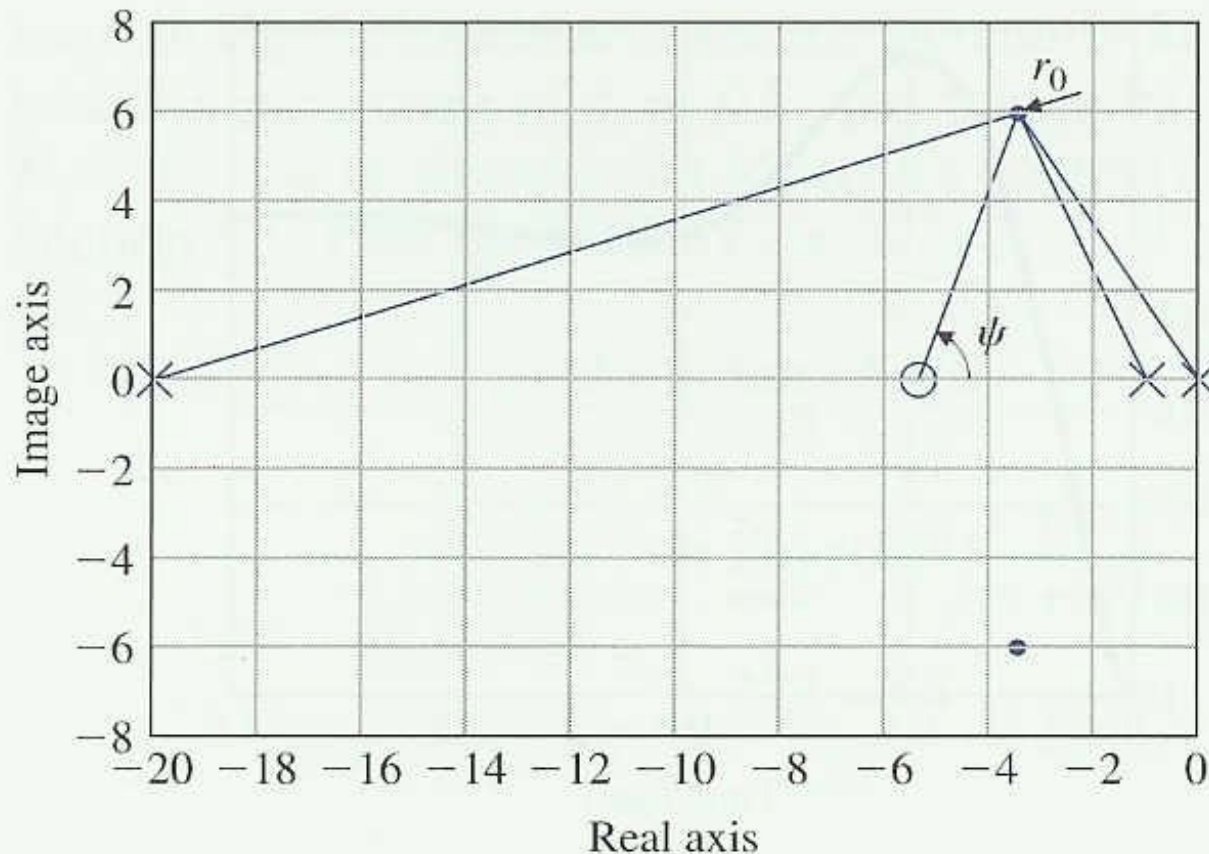
Design approach (B)

- ❑ Pole placement from the requirements

$$r_0 = -3.5 + j3.5\sqrt{3}$$

- ❑ Noise: Additional requirement,  $\max(p)=20$ . So,  $p=20$  to minimize the effect of the pole.

# Lead Compensation



All poles are fixed.

$r_0$  is fixed.



Possible to calculate  $\psi$   
as  $\angle D(s)G(s) = 180^\circ$



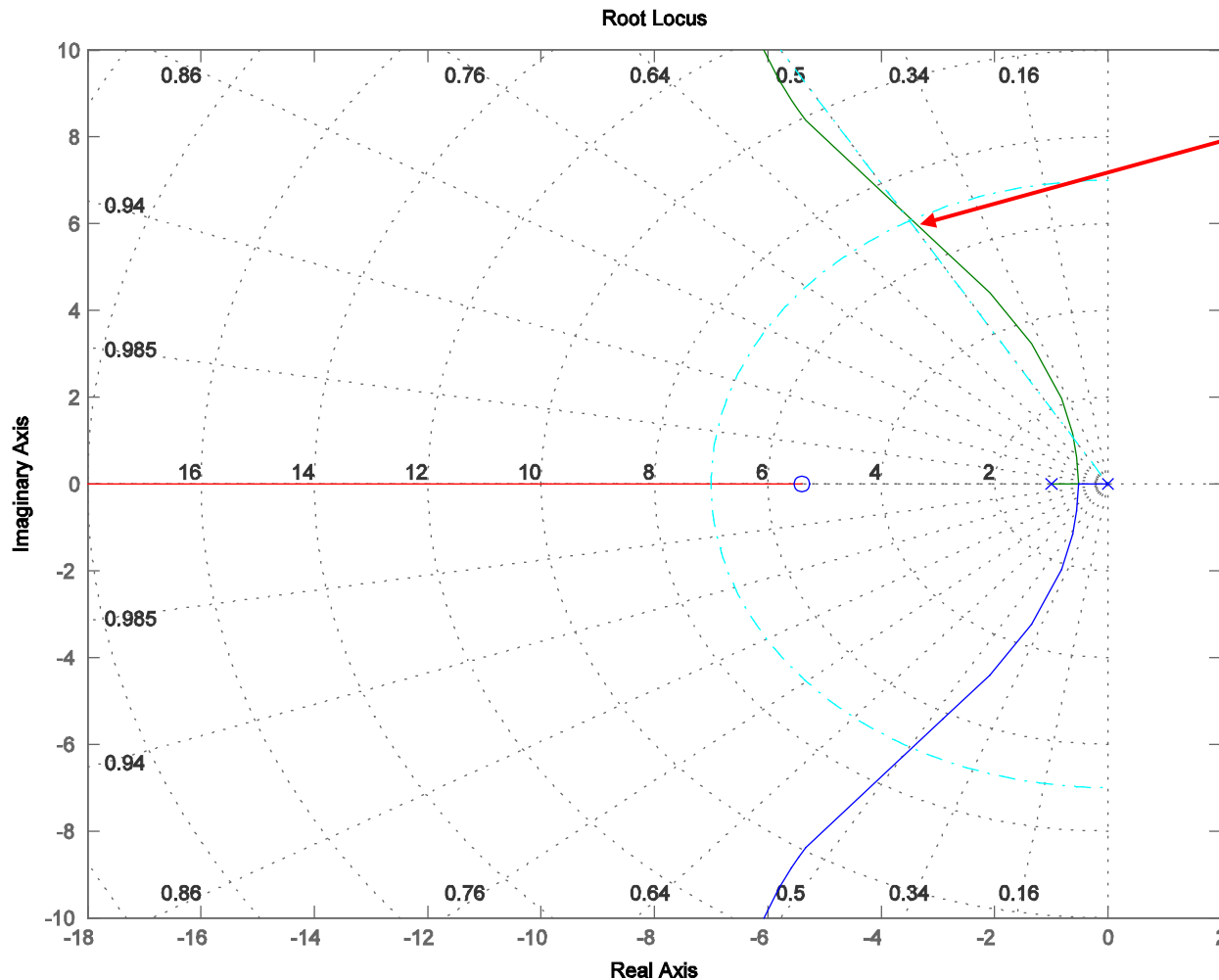
$\psi = 72.6^\circ$



Gives location of  $z$

Result :  $z = 5.4$

# Lead Compensation



Pole location

Matlab – design B

```
sysG = tf([1],[1 1 0])
```

```
sysD = tf([1 5.4],[1 20])
```

```
rlocus(sysD*sysG)
```

Final Controller

$$D(s) = 127 \frac{s + 5.4}{s + 20}$$

# Lead Compensation

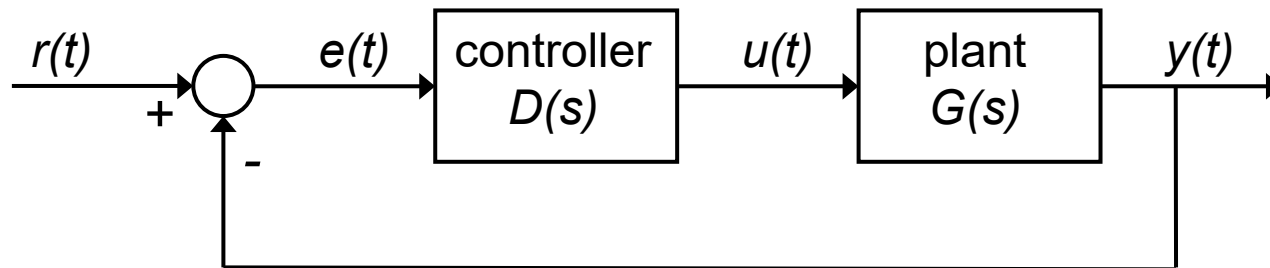
## Exercise

- Design a lead compensation  $D(s)$  to the plant  $G(s)$  so that the dominant poles are located at  $s = -2 \pm 2j$

$$D(s) = K \frac{s + z}{s + p}$$

$$G(s) = \frac{1}{s^2}$$

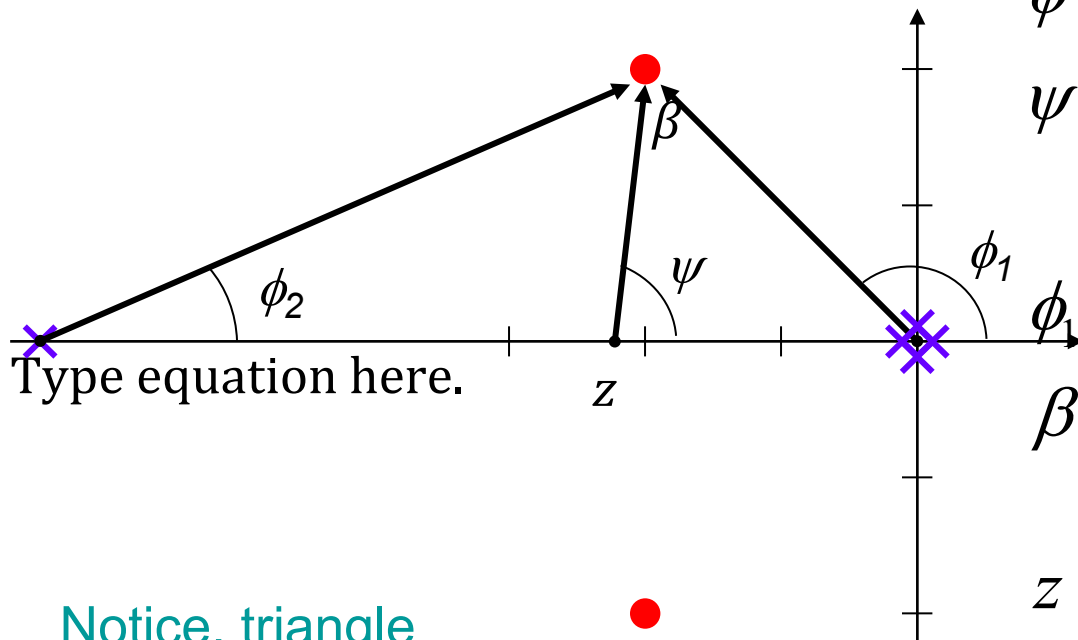
You can use, triangle  
 $a/\sin(A) = b/\sin(B)$



# Lead Compensation

## Answer

- we have three poles 0,0,20 and one zero



Notice, triangle  
 $a/\sin(A) = b/\sin(B)$

## Root Locus Condition

$$\angle D(s)G(s) = 180^\circ$$

$$\phi_1 = 135^\circ, \phi_2 \approx 10^\circ$$

$$\psi - 2\phi_1 - \phi_2 = 180^\circ \Rightarrow$$

$$\psi = 180^\circ + 2 \cdot 135^\circ$$

$$+ 10^\circ = 460^\circ = 100^\circ$$

$$\phi_1 = 135^\circ \Rightarrow$$

$$\beta = 180^\circ - (180^\circ - 135^\circ) = 35^\circ$$

$$z = -\frac{\sqrt{2^2 + 2^2}}{\sin(100^\circ)} \sin(35^\circ)$$

$$= -1.65$$

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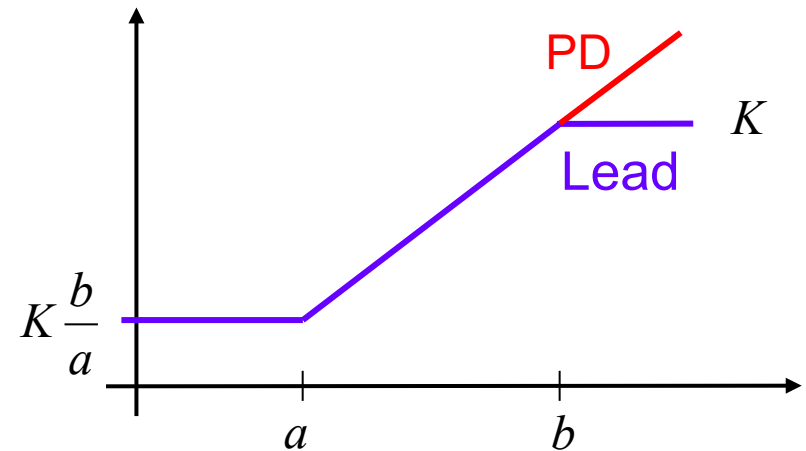
# BREAK



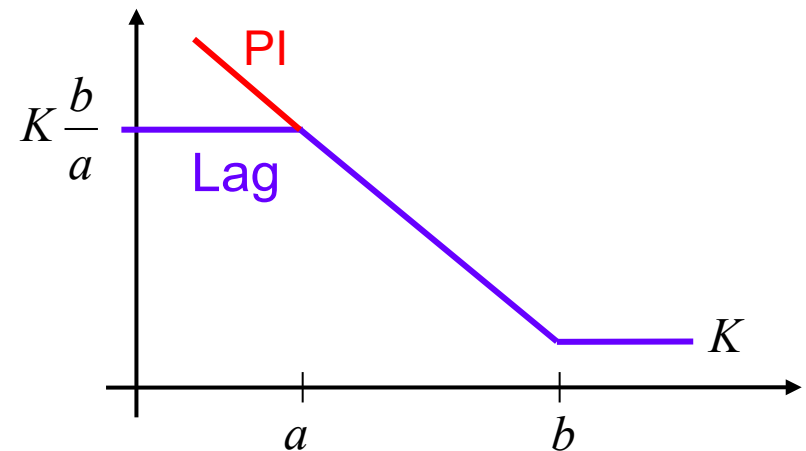
# Dynamic Compensation

$$D(s) = K \frac{s+a}{s+b} = K \frac{b}{a} \frac{(s/a+1)}{(s/b+1)}$$

Lead compensation  $a < b$



Lag compensation  $a > b$



# Lag Compensation

## Lag compensation

- Steady state characteristics (requirements)
  - Position error constant  $K_p$
  - Velocity error constant  $K_v$
- Let us continue using the example system

$$G(s) = \frac{1}{s(s+1)}$$

- and the designed controller (lead compensation)

$$D(s) = 127 \frac{s + 5.4}{s + 20}$$

# Lag Compensation

## Calculation of error constants

- Suppose we want  $K_v \geq 100$
- We need additional gain at low frequencies !

$$D(s)G(s) = 127 \frac{s + 5.4}{s + 20} \frac{1}{s(s + 1)}$$

$$K_p = \lim_{s \rightarrow 0} D(s)G(s) = \infty$$

$$\text{Step input : } e_{ss} = \frac{1}{1 + K_p} = 0$$

$$K_v = \lim_{s \rightarrow 0} sD(s)G(s) = 127 \frac{5.4}{20} = 34.3$$

$$\text{Ramp input : } e_{ss} = \frac{1}{K_v} = 0.03$$

# Lag Compensation

## Selecting $p$ and $z$

- ❑ To minimize the effect on the dominant dynamics  $z$  and  $p$  must be chosen as low as possible (i.e. at low frequencies)
- ❑ To minimize the settling time  $z$  and  $p$  must be chosen as high as possible (i.e. at high frequencies)
- ❑ Thus, the lag pole-zero location must be chosen at as high a frequency as possible without causing any major shifts in the dominant pole locations

# Lag Compensation

$$G(s) = \frac{1}{s(s+1)}$$

Requirement

Velocity error

constant  $K_v \geq 100$

Lag compensation

$$D(s) = K \frac{s+z}{s+p}$$

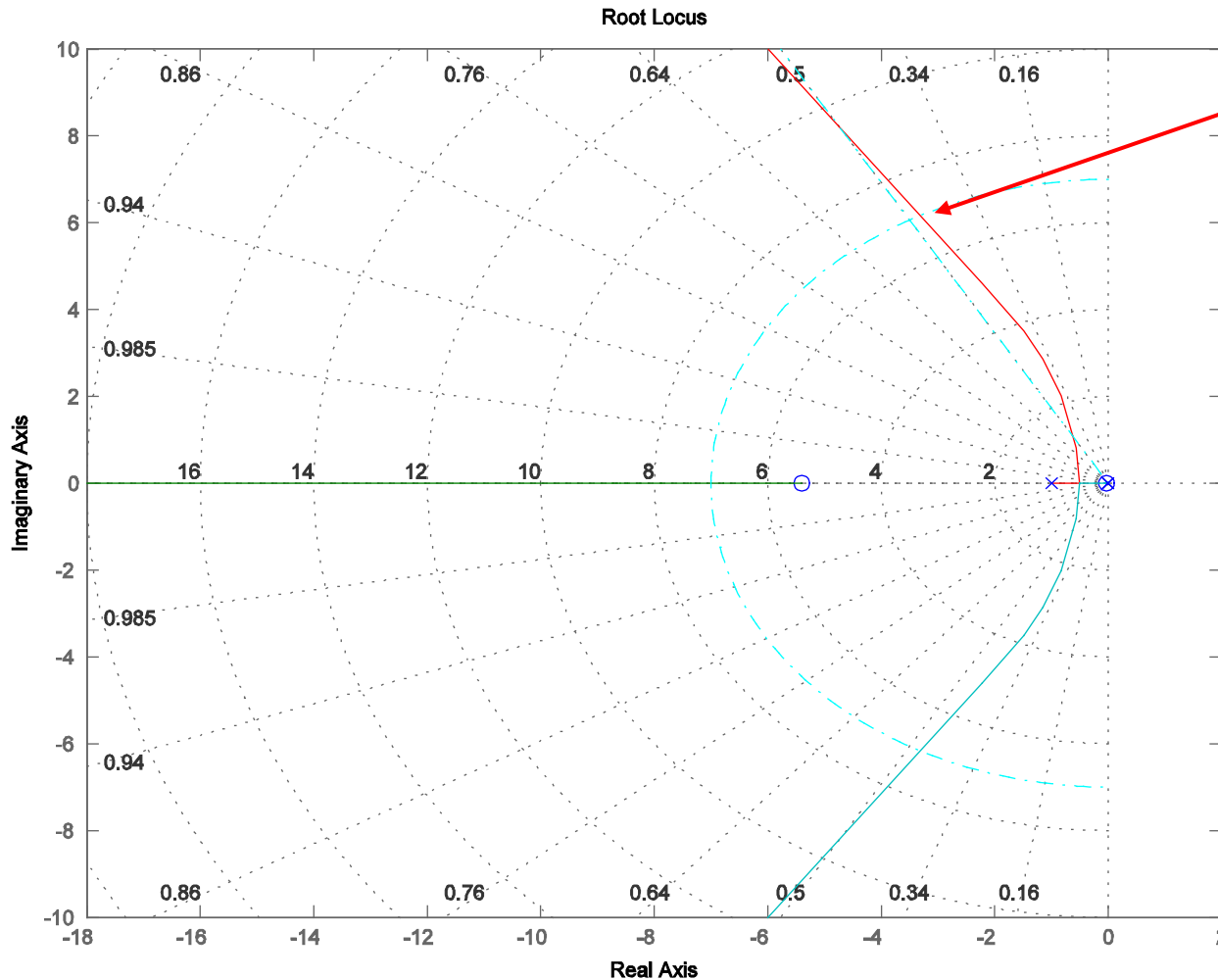
Low freq. gain

$$D(0) = K \frac{z}{p}$$

## Design approach

- Increase the error constant by increasing the low frequency gain
- Choose  $z$  and  $p$  somewhat below  $\omega_n$
- Example system:
  - $K = 1$  (the proportional part has already been chosen)
  - $z/p = 3$ , with  $z = 0.03$ , and  $p = 0.01$ .

# Lag Compensation

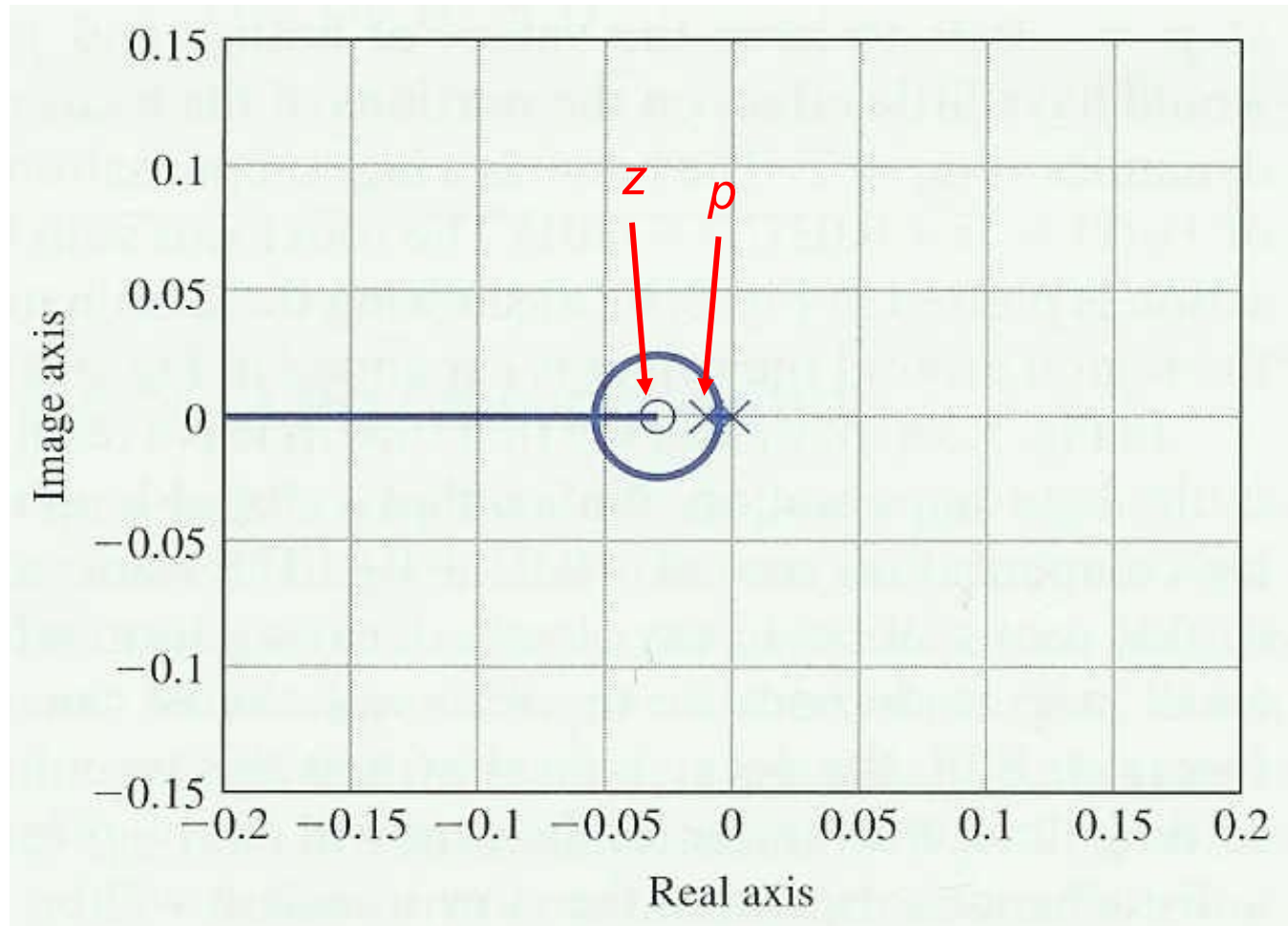


Notice, a slightly different pole location

Lead-lag Controller

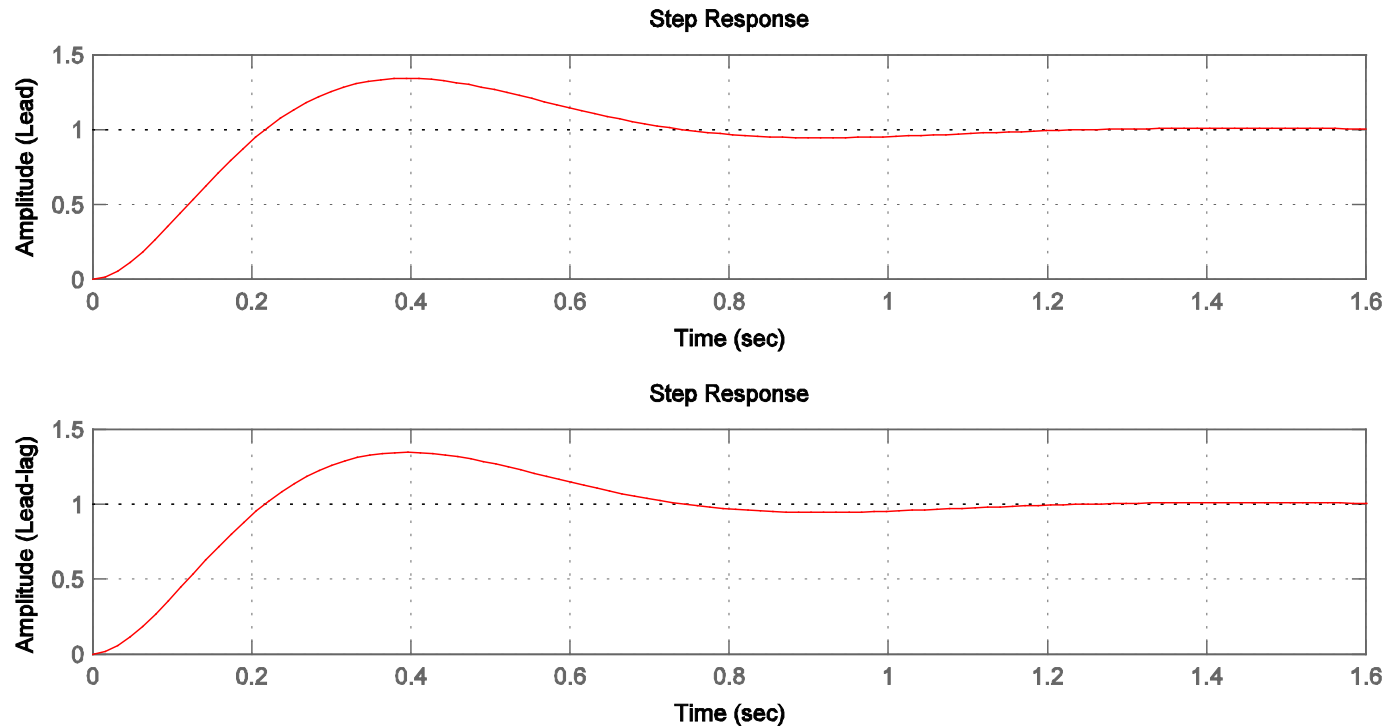
$$D(s) = 127 \frac{s + 5.4}{s + 20} \frac{s + 0.03}{s + 0.01}$$

# Lag Compensation



# Lag Compensation

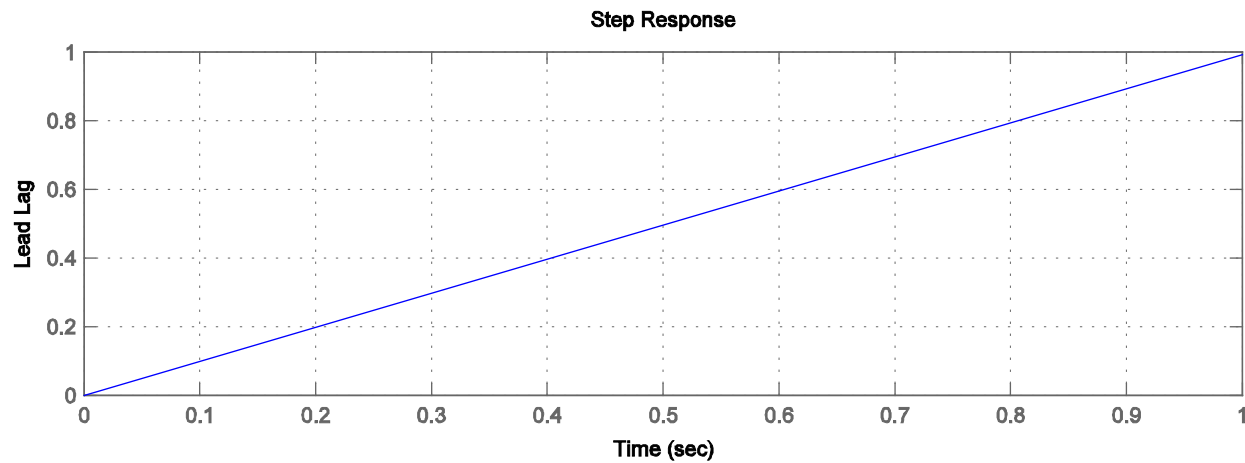
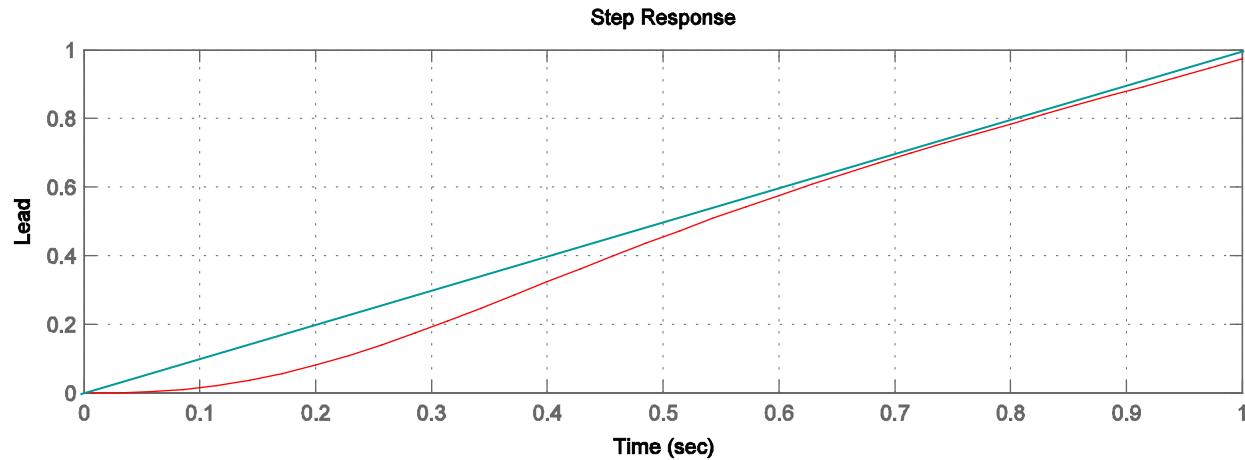
## Step





# Lag Compensation

Ramp



# Notch compensation

## Example system

- We have successfully designed a lead-lag controller

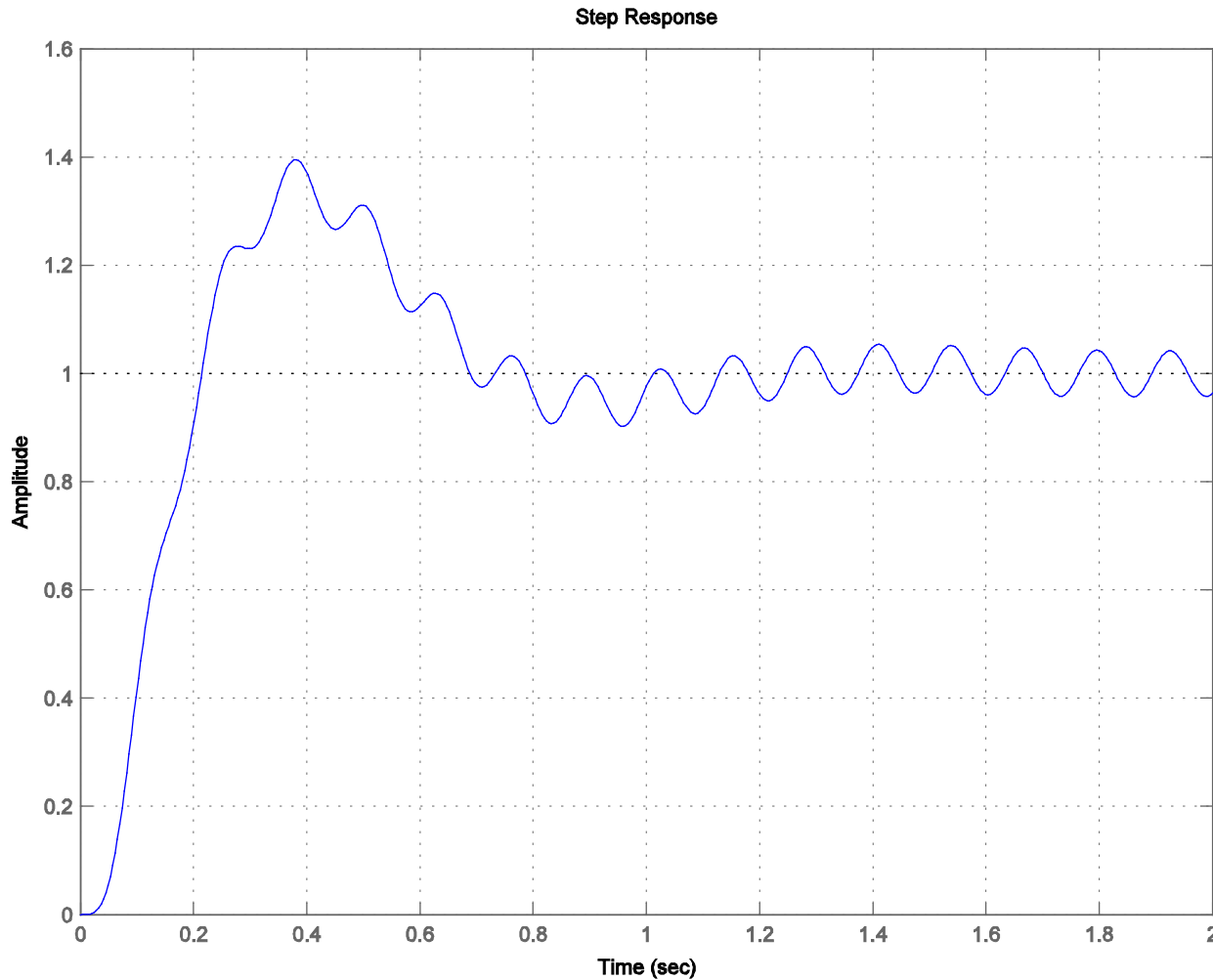
$$D(s) = 127 \frac{s + 5.4}{s + 20} \frac{s + 0.03}{s + 0.01}$$

- Suppose the real system has a rather undampen oscillation about 50 rad/sec.
- Include this oscillation in the model

$$G(s) = \frac{1}{s(s + 1)} \frac{2500}{(s^2 + s + 2500)}$$

- Can we use the original controller ?

# Notch compensation



Step response  
using the lead-  
lag controller

# Notch compensation

Aim: Remove or dampen the oscillations

## Possibilities

- Gain stabilization
  - Reduce the gain at high frequencies
  - Thus, insert poles above the bandwidth but below the oscillation frequency – might not be feasible
- Phase stabilization (*notch compensation*)
  - A zero near the oscillation frequency
  - A zero increases the phase, and  $\zeta \approx PM/100$
  - Possible transfer function

$$D_n(s) = \frac{s^2 + 2\zeta_n\omega_0s + \omega_0^2}{(s + \omega_0)^2}$$

If  $\zeta_n < 1$ , complex zeros and double pole at  $\omega_0$

# Notch compensation

$$G(s) = \frac{1}{s(s+1)} \frac{2500}{(s^2 + s + 2500)}$$
$$= \frac{1}{s(s+1)} \frac{\omega_r^2}{(s^2 + 2\zeta_r \omega_r s + \omega_r^2)}$$

$$\omega_r = 50 \quad , \quad \zeta_r = 0.01$$

Notch compensation

$$D_n(s) = \frac{s^2 + 2\zeta_n \omega_0 s + \omega_0^2}{(s + \omega_0)^2}$$

Let us choose

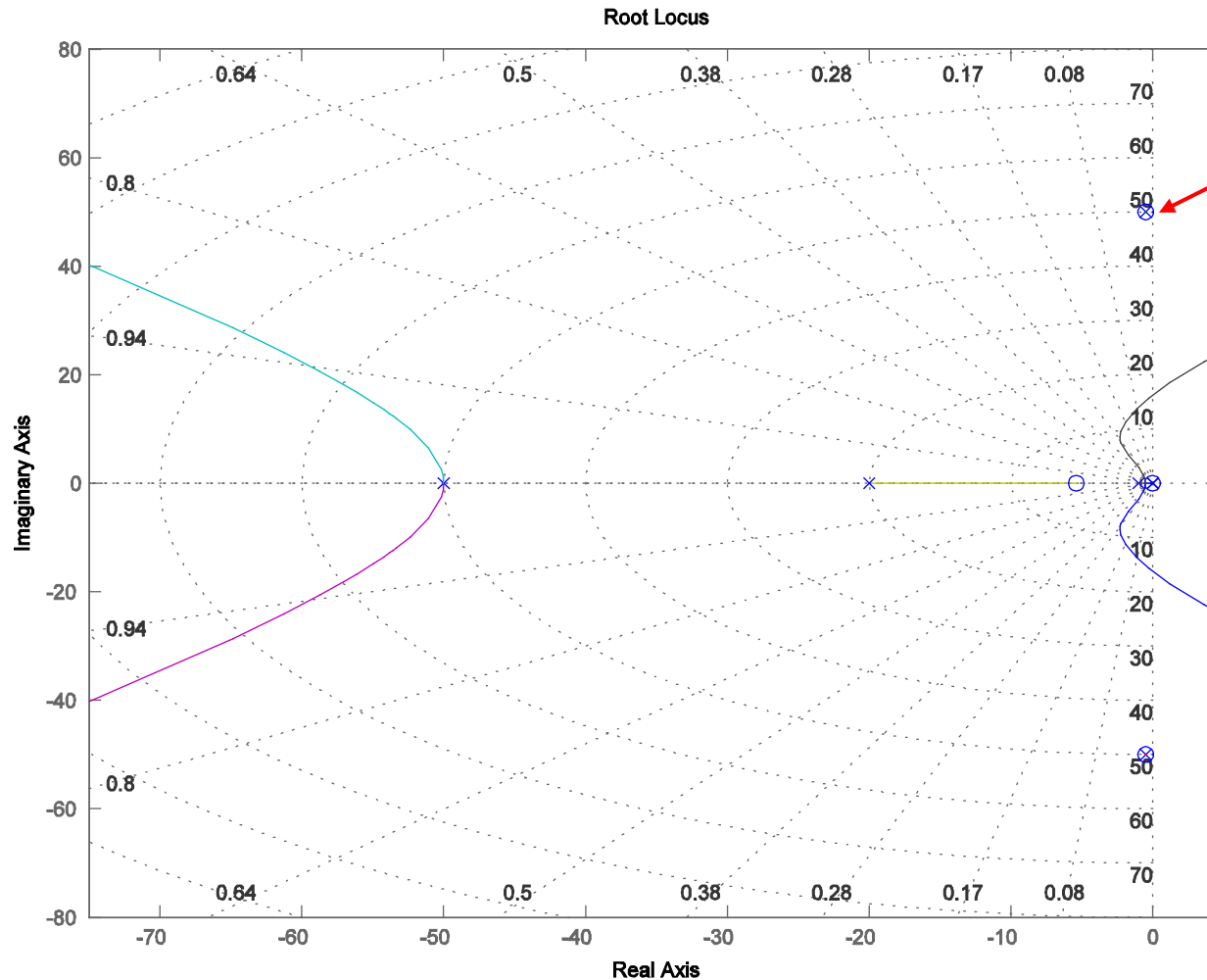
$$\omega_r = \omega_0 \quad , \quad \zeta_r = \zeta_n$$

The undamped poles  
are cancelled

Result

$$D_n(s) G(s)$$
$$= \frac{1}{s(s+1)} \frac{\omega_r^2}{(s + \omega_r)^2}$$

# Notch compensation

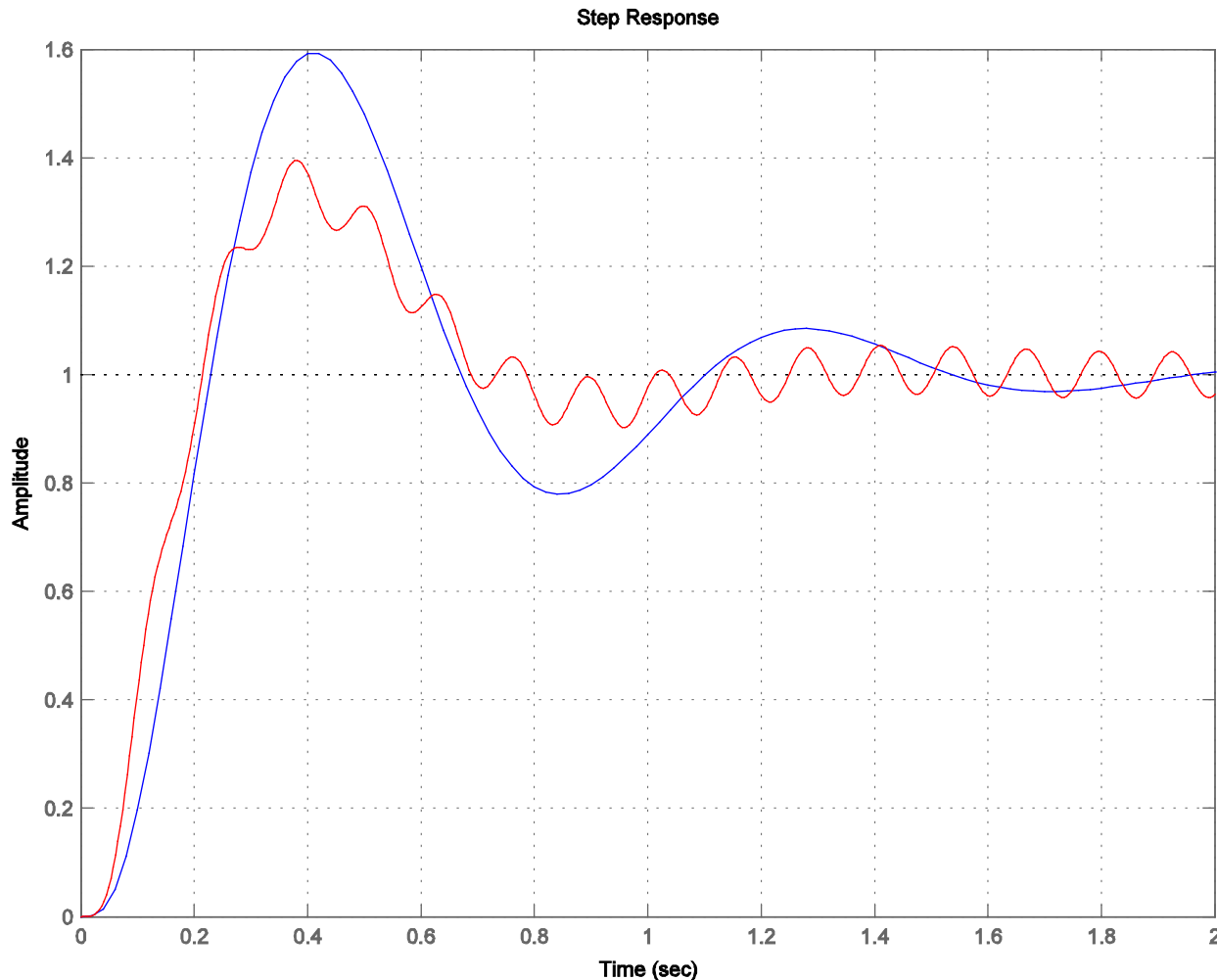


Cancellation

**Root Locus**  
The lead-lag  
controller with  
notch  
compensation

$$\omega_0 = 50$$
$$\zeta = 0.01$$

# Notch compensation



Too much overshoot !!

Also, exact cancellation might cause problems due to modelling errors

Thus, we need different parameters

# Notch compensation

## Modified parameters

- Notch compensation

$$D_n(s) = \frac{s^2 + 2\zeta\omega_0 s + \omega_0^2}{(s + \omega_0)^2}$$

- Dynamic characteristics

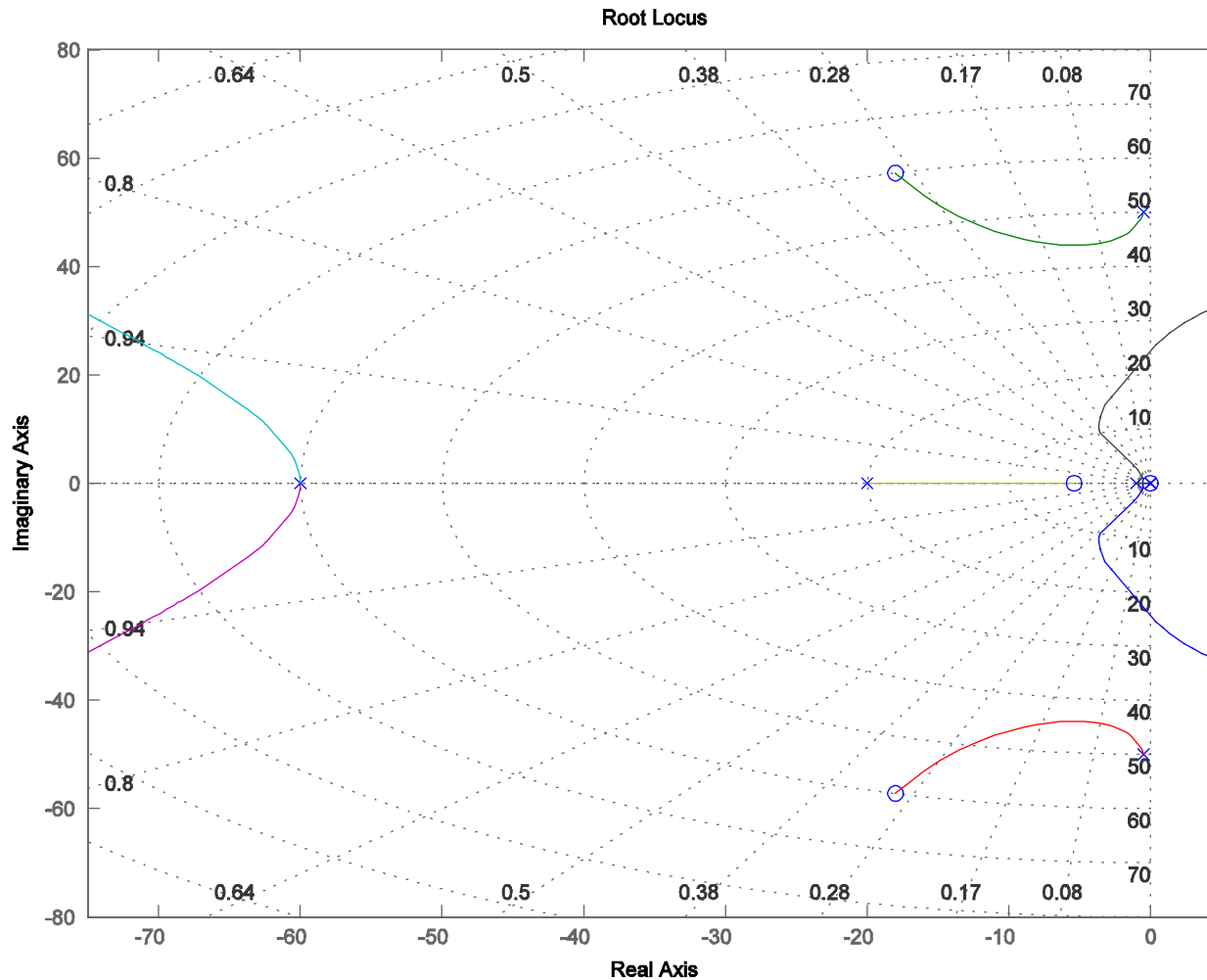
- Increase  $\zeta_n$  to obtain less overshoot
- Make sure that the roots move into the LHP
- In general, obtain a satisfactory dynamic behavior

- After some trial and error

$$\omega_0 = 60 \quad , \quad \zeta_n = 0.3$$



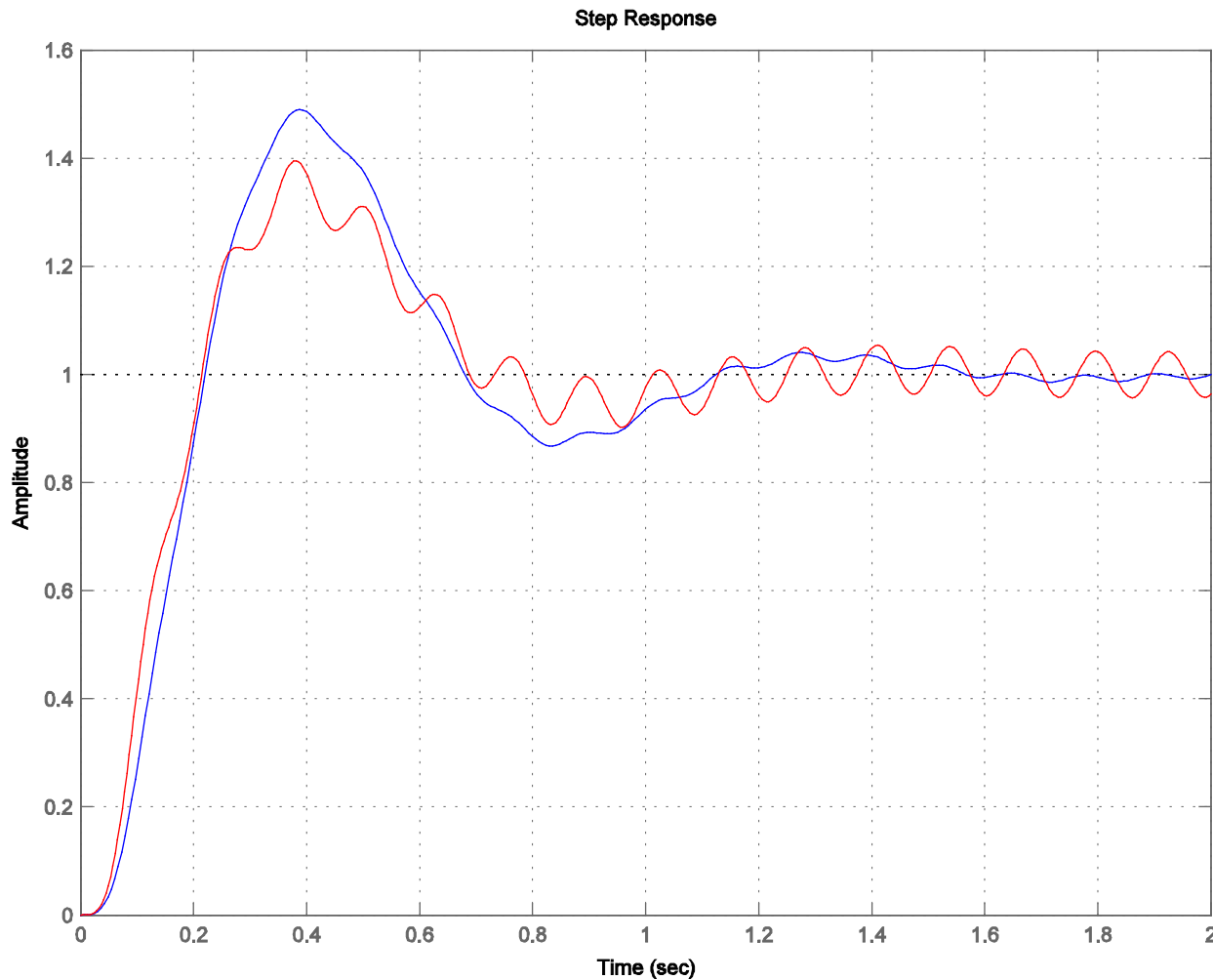
# Notch compensation



**Root Locus**  
The lead-lag  
controller with  
notch  
compensation

$$\omega_0 = 60$$
$$\zeta = 0.3$$

# Notch compensation



The overshoot has been lowered to an acceptable level !

The oscillations have almost been removed !

So, we have reached a final design !

# Time Delay

## Notice

- ❑ Time delay always reduces the stability of a system !
- ❑ Important to be able to analyze its effect
- ❑ In the s-domain a time delay is given by  $e^{-\lambda s}$
- ❑ Most applications contain delays (sampled systems)

## Root locus analysis

- ❑ The original method does only handle polynomials

## Solutions

- ❑ Approximation (Padé) of  $e^{-\lambda s}$
- ❑ Modifying the root locus method (direct application)

# Time Delay

First approximation

(1,1) Padé approximant

$$e^{-s} \approx \frac{b_0 s + b_1}{a_0 s + 1}$$

McLauren series

$$e^{-s} = 1 - s + \frac{s^2}{2} + \frac{s^3}{3!} + \frac{s^4}{4!} + \dots$$

$$\begin{aligned} \frac{b_0 s + b_1}{a_0 s + 1} &= b_1 + (b_0 - a_0 b_1)s \\ &\quad - a_0(b_0 - a_0 b_1)s^2 + a_0^2(b_0 - a_0 b_1)s^3 \end{aligned}$$

$$b_1 = 1$$

$$b_0 - a_0 b_1 = -1$$

$$-a_0(b_0 - a_0 b_1) = \frac{1}{2}$$

$$a_0^2(b_0 - a_0 b_1) = -\frac{1}{6}$$

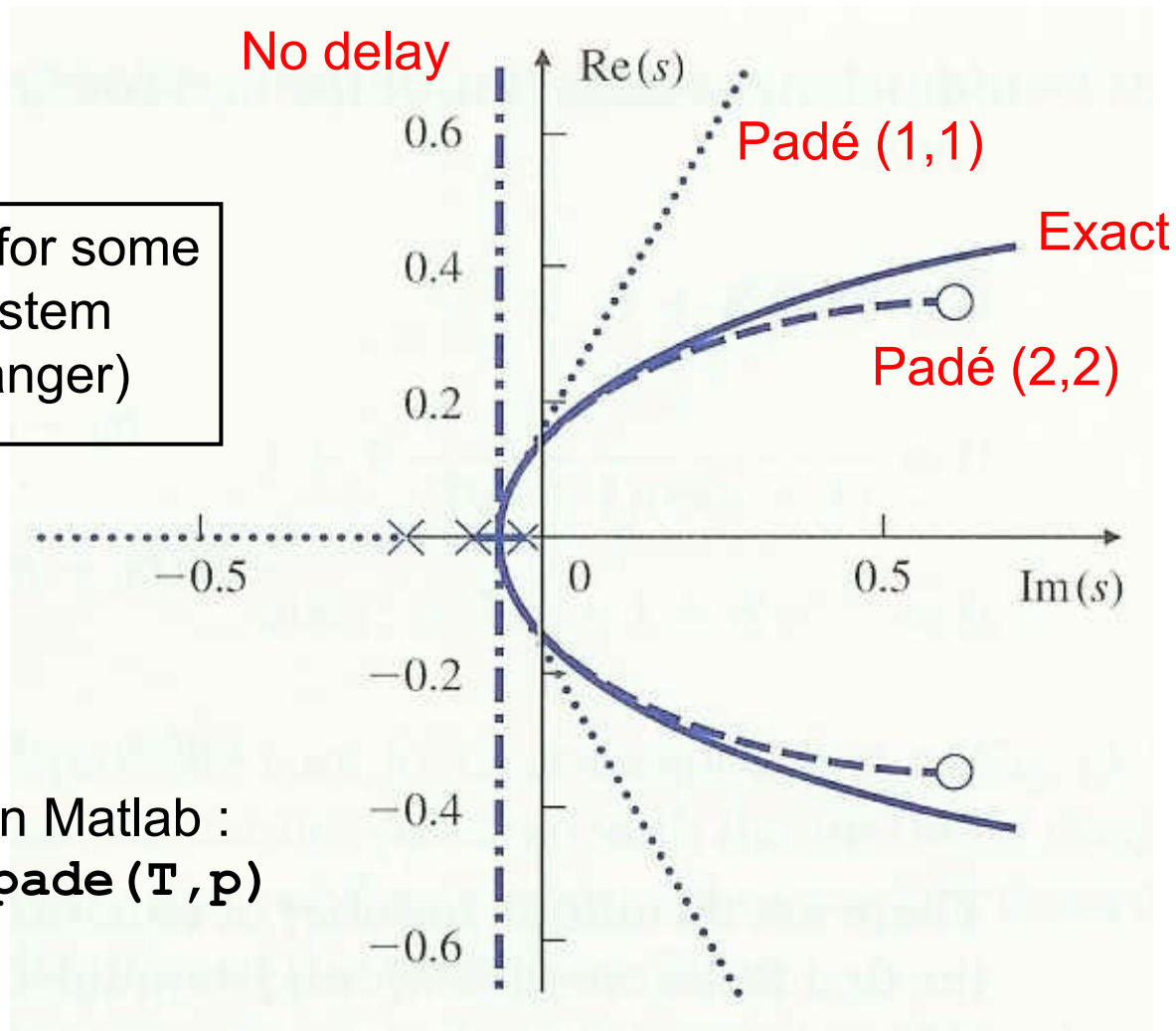
$$\Downarrow \quad \text{with } s \equiv T_d s$$

$$e^{-T_d s} \cong \frac{1 - (T_d s / 2)}{1 + (T_d s / 2)}$$

# Time Delay

Root locus for some  
example system  
(heat exchanger)

(p,p) Approx. in Matlab :  
`[num,den]=pade(T,p)`



# Time Delay

Direct approach (exact calculation)

Process

$$G(s) = e^{-T_d s} G_0(s)$$

Notice,  $s = \sigma + j\omega$

$$\angle e^{-T_d s} = \angle (e^{-T_d \sigma} e^{-jT_d \omega}) = -T_d \omega$$

Modified root locus condition

$$\angle D(s) G(s) = \angle (D(s) G_0(s)) + \angle (e^{-T_d s}) = 180^\circ$$

$\Downarrow$

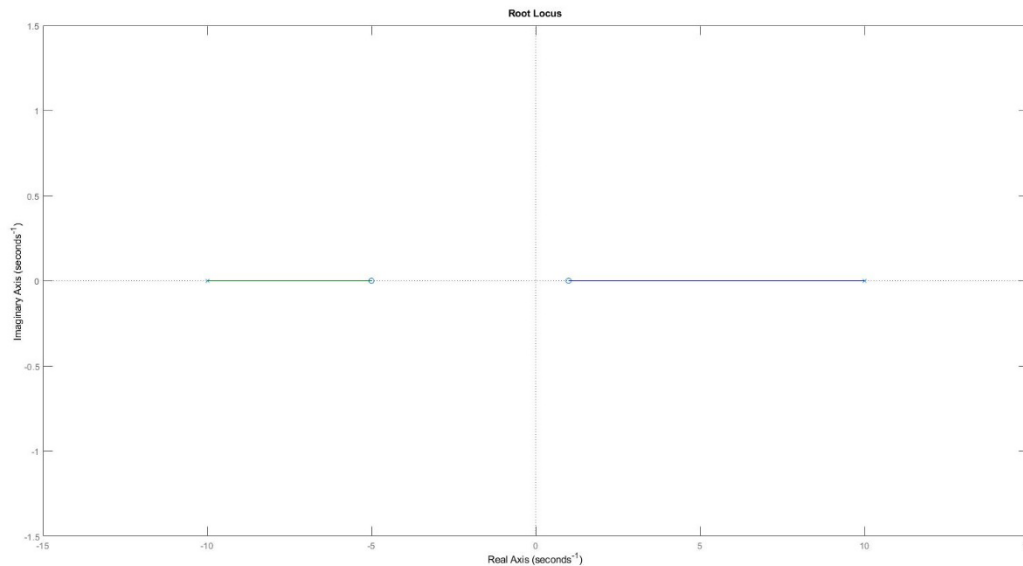
$$\underline{\angle D(s) G(s) = 180^\circ + T_d \omega}$$

However, Matlab  
does not support  
this approach...

# Unstable systems - example

$$G(s) = \frac{(s-1)(s+5)}{(s+10)(s-10)}$$

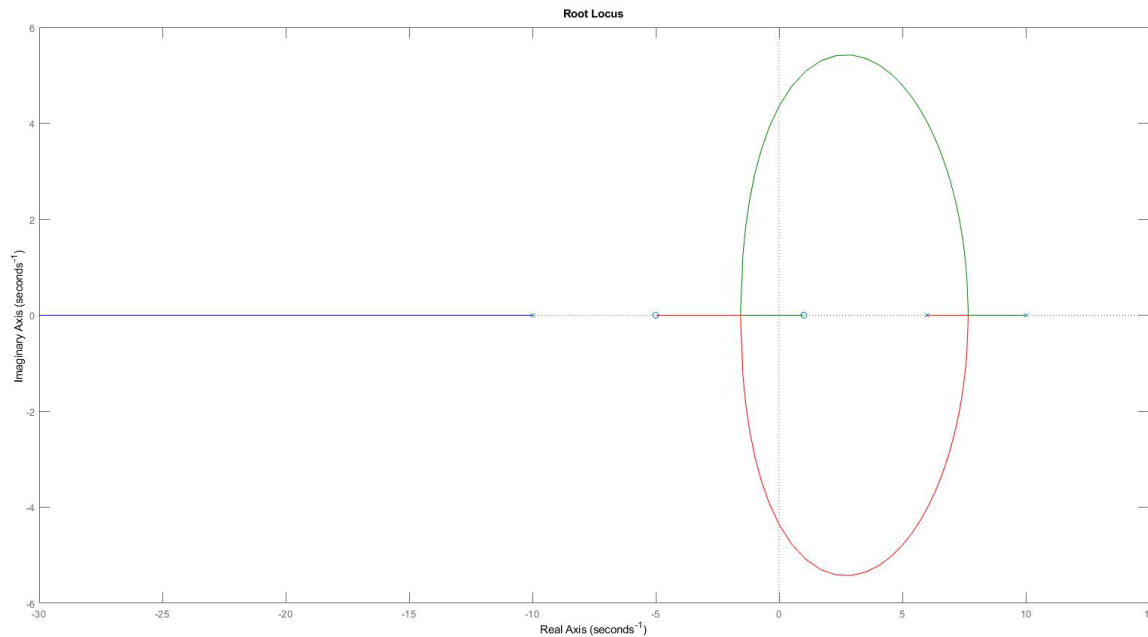
an unstable pole and an unstable zero



How can we stabilize the system

# Dirty trick -Insert an unstable pole

$$G(s) = \frac{(s - 1)(s + 5)}{(s + 10)(s - 10)(s - 6)}$$





# The Discrete Root Locus

## Discrete (z-domain)

- Closed loop transfer function

$$\frac{D(z)G(z)}{1 + D(z)G(z)}$$

- Characteristic equation

$$1 + D(z)G(z) = 0$$

- Thus, same sketching techniques as in the s-domain, the unit circle is the stable region.
- *However, different interpretation !!!*

