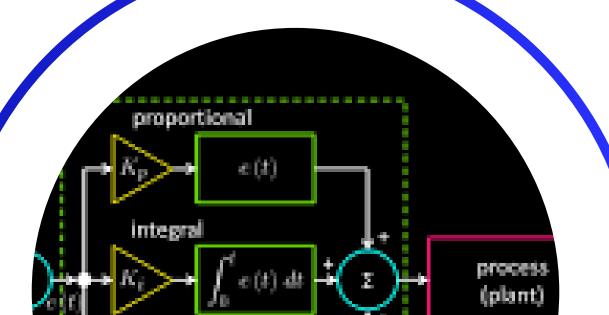


# Project 8: Magnetic Levitation Train Control

Control Systems - MENG-3510-0NA



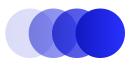
#### **Group Members:**

Michael McCorkell

Joshua Mongal

Mikaeel Khanzada

#### **Agenda**



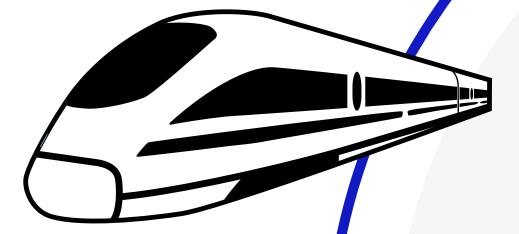
- 01 Problem and Objectives
- O2 System Operation review and Applications
- O3 System to be controlled and its significance
- 04 System modeling and representation
- O5 Analysis of system stability and performance
- 06 Control Strategy Design
- 07 Tuning and Implementation
- 08 Results

#### Problem statement

Investigate the suspension bogie to maintain a proper air gap between the rail and electromagnet despite external forces (e.g. Crosswinds, Weight, and more) being applied on the system.

## Objective

Design a PID control system capable of receiving feedback from external forces and providing a response that regulates the air gap over time with a transient settling time of no more than 1 second and a maximum variation of less than 10%.

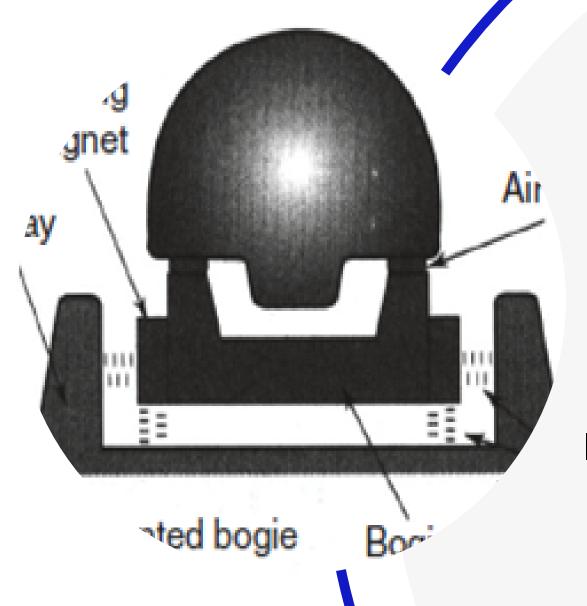


## Why?

Maglev trains use magnetic forces to levitate and move without touching the track, allowing high speeds and smooth, quiet rides. They reduce travel time and emissions but face high costs, complex controls, and sensitivity to power loss and weather. Despite challenges, they offer a promising future for fast, clean transport.

Real-world applications China and Japan

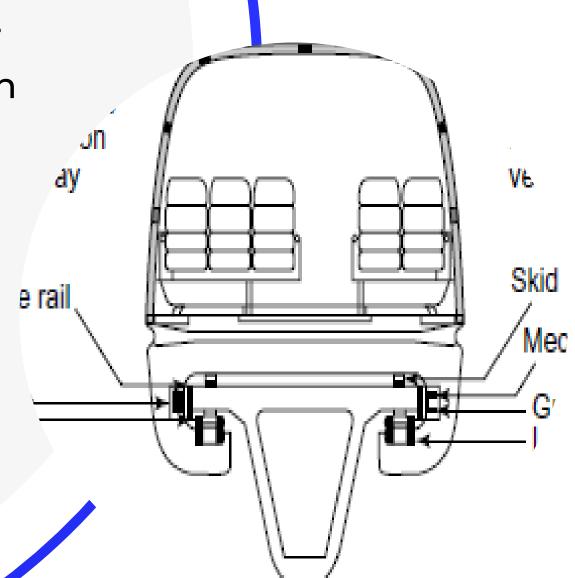
- significant benefits in reducing commute times
- environmental impact



## Suspension Bogie

This system is a critical component especially in maglev trains that allow for levitation, guidance and shock absorption

This allows the train to float above the track and move smoothly.



#### How it works

A Maglev train carries multiple bogies with their own levitation system.

Bogies are designed to absorb shocks, vibrations, and relies on electromagnets that generate attractive & repulsive forces.

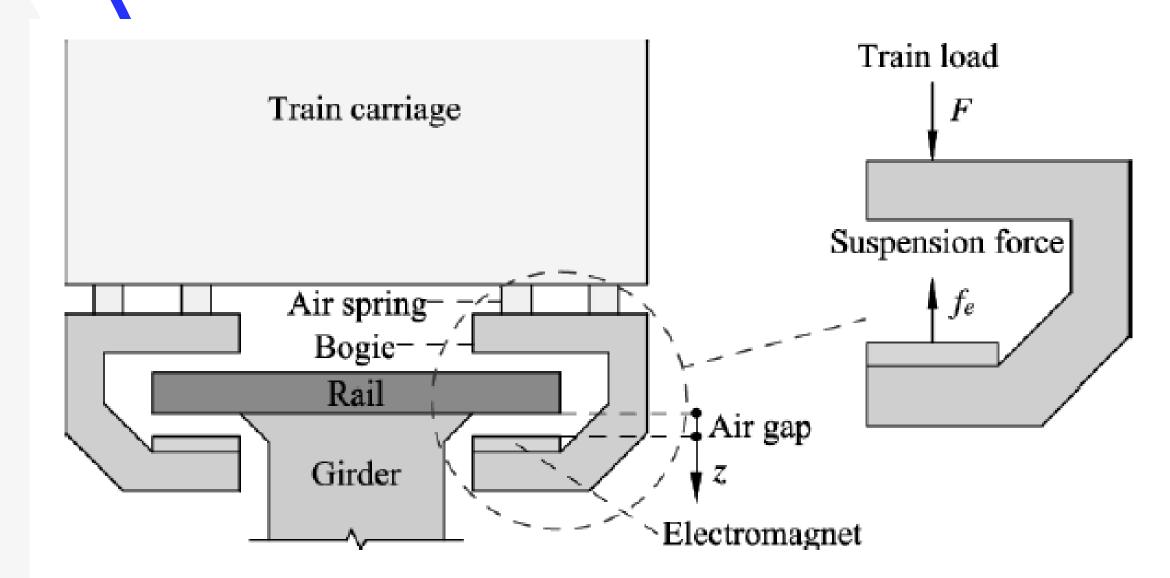


Figure 2: Schematic diagram of the EMS maglev train



**Actuators and Sensors** 

#### **Sensors**

Lateral displacement sensor Weight sensor

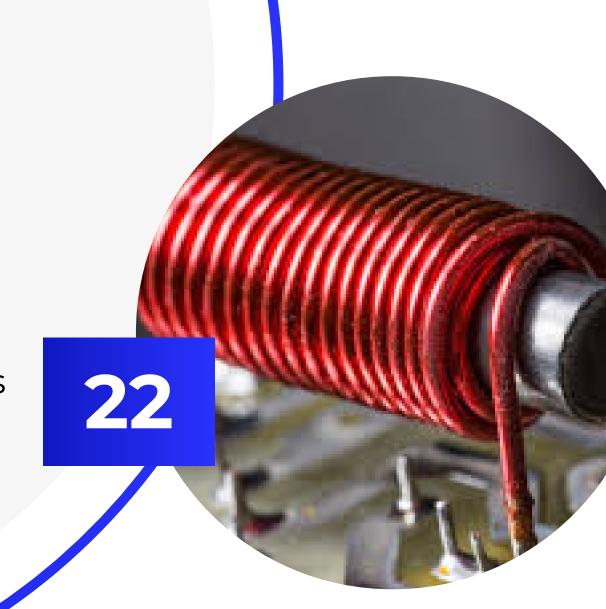
Current sensor

Speedometer

Vibration Sensor

#### **Actuator:**

Electromagnetic Coil
Dynamic Load Balancing Actuators
Aerodynamic Flaps



#### **Transfer functions**



$$m\frac{d^{2}z(t)}{dt^{2}} = \left(\frac{\mu_{0}AN^{2}i_{0}^{2}}{2z_{0}^{3}}\right)z(t) - \left(\frac{\mu_{0}AN^{2}i_{0}}{2z_{0}^{2}}\right)i(t) + F$$

$$L_{0}\frac{di(t)}{dt} = V(t) - Ri(t) + \left(\frac{\mu_{0}AN^{2}i_{0}}{2z_{0}^{2}}\right)\frac{dz(t)}{dt}$$

$$(-k2/m)/Lo$$

 $Lo/Lo \cdot s^3 + R/Lo \cdot s^2 - ((1/m) * ((k1 * Lo) - (k2^2)))/Los - ((k1 * R)/m)/Lo$ 



$$\frac{((1/m) * Lo)/Los + ((1/m) * R)/Lo}{Lo \cdot s^3 + R \cdot s^2 - ((1/m) * ((k1 * Lo) - (k2^2)))s - ((k1 * R)/m)}$$

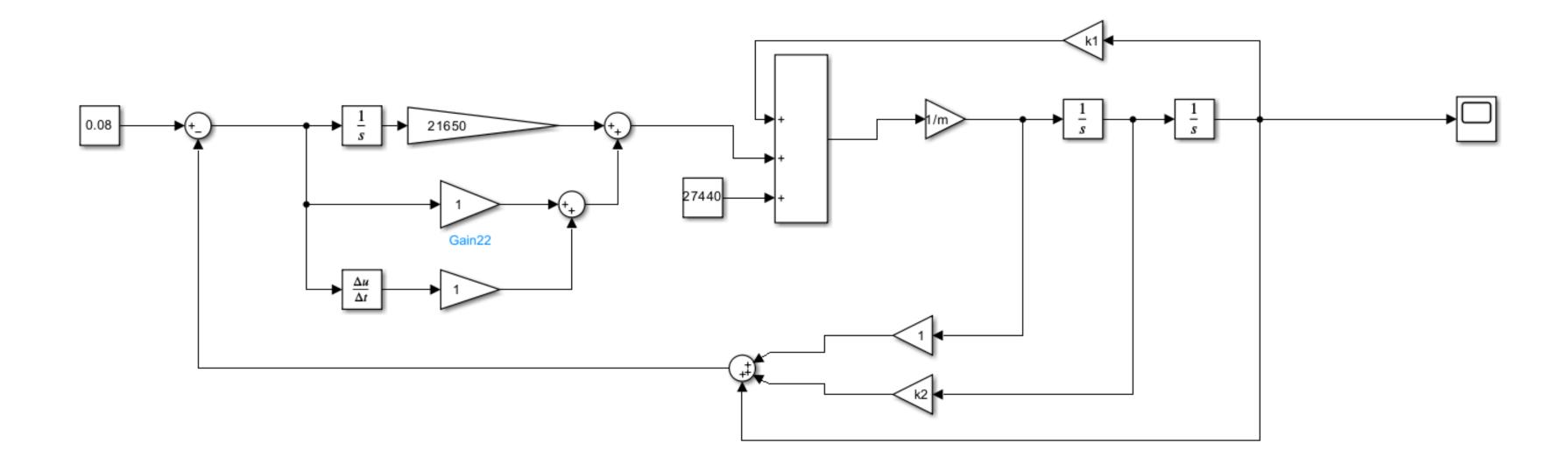


$$\frac{-0.60714}{s^3 + 31.438s^2 - 0.0076s - 309.64}$$

$$\frac{0.0014286s + 0.044911}{s^3 + 31.438s^2 - 0.0076s - 309.64}$$

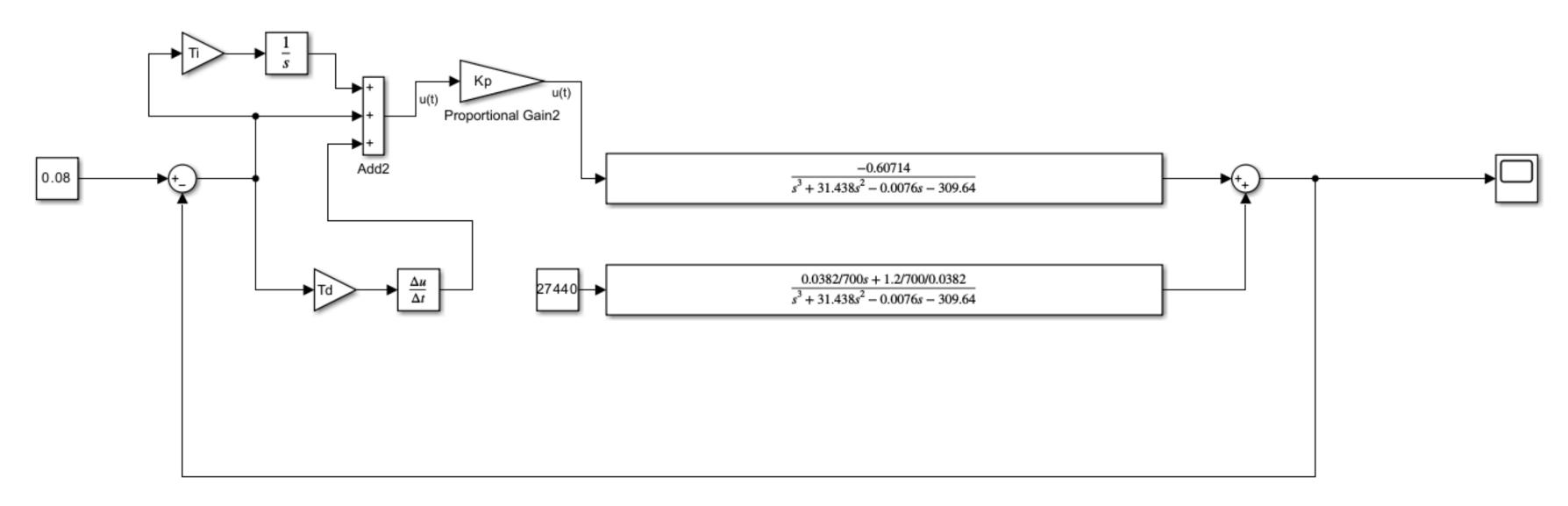
# **Block Diagram**



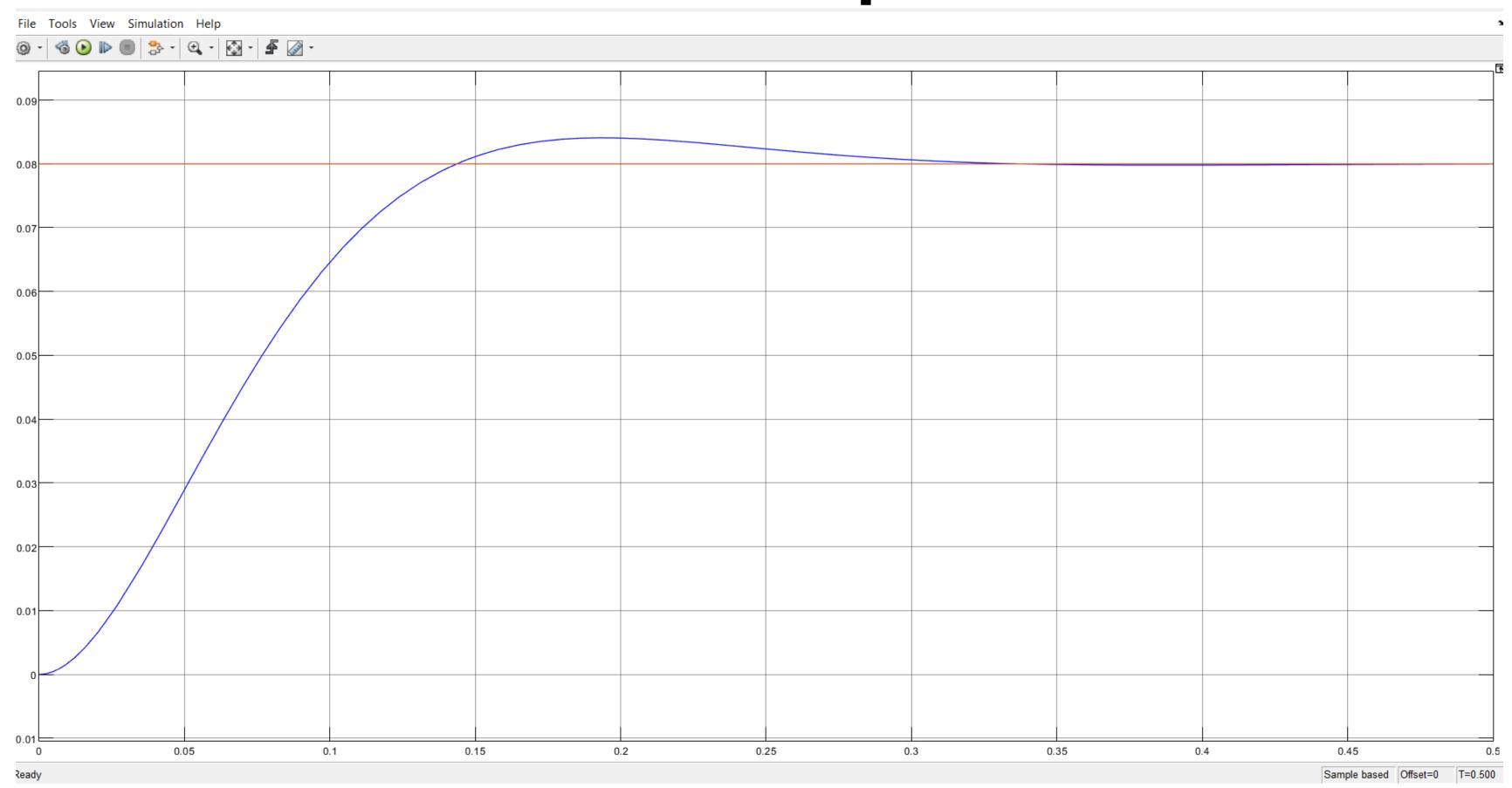


# **Block Diagram**





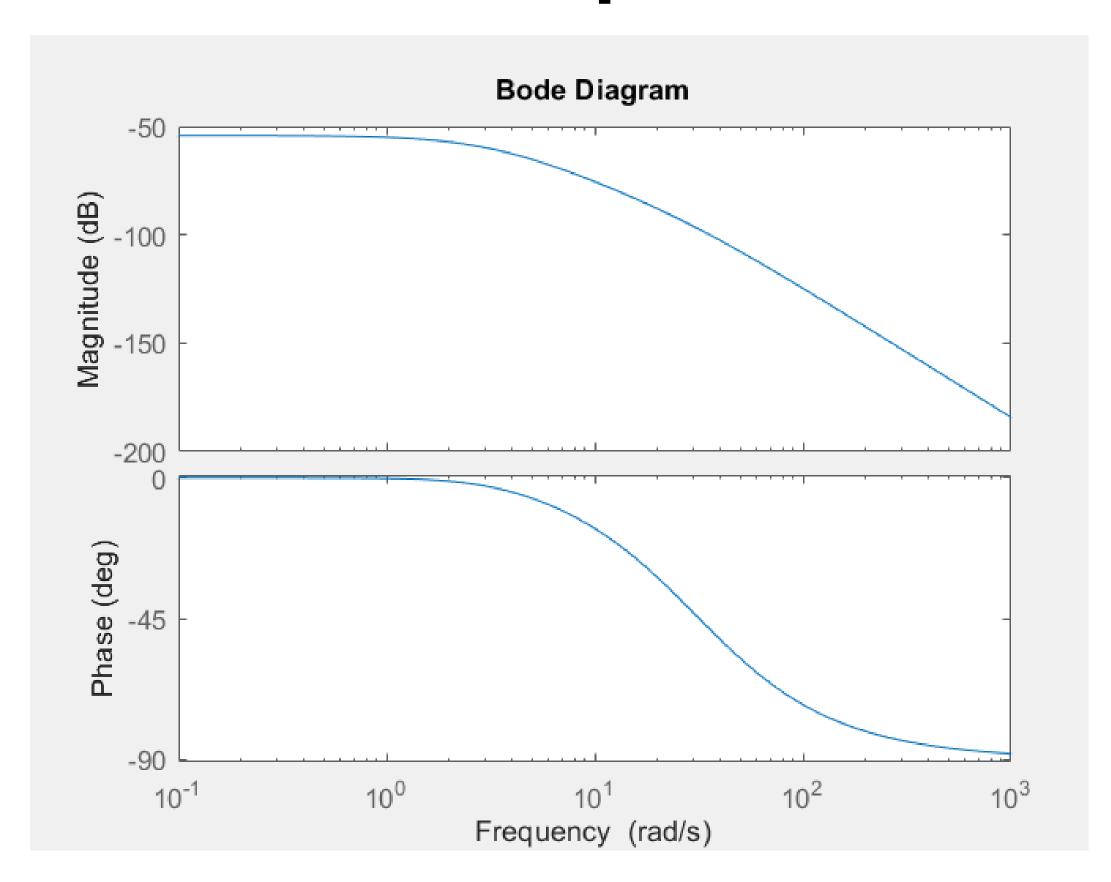
# **Desired Response**



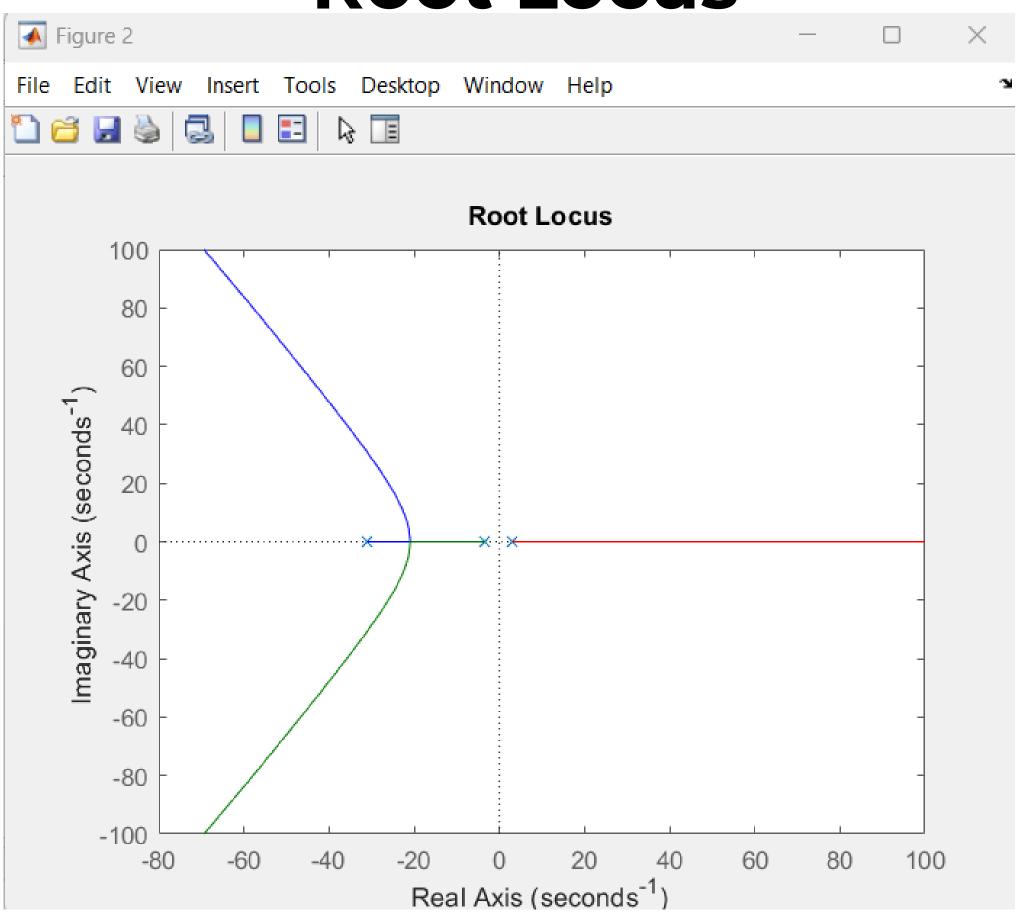
#### Roots

- -31.1185
  - -3.3182
    - 2.9987

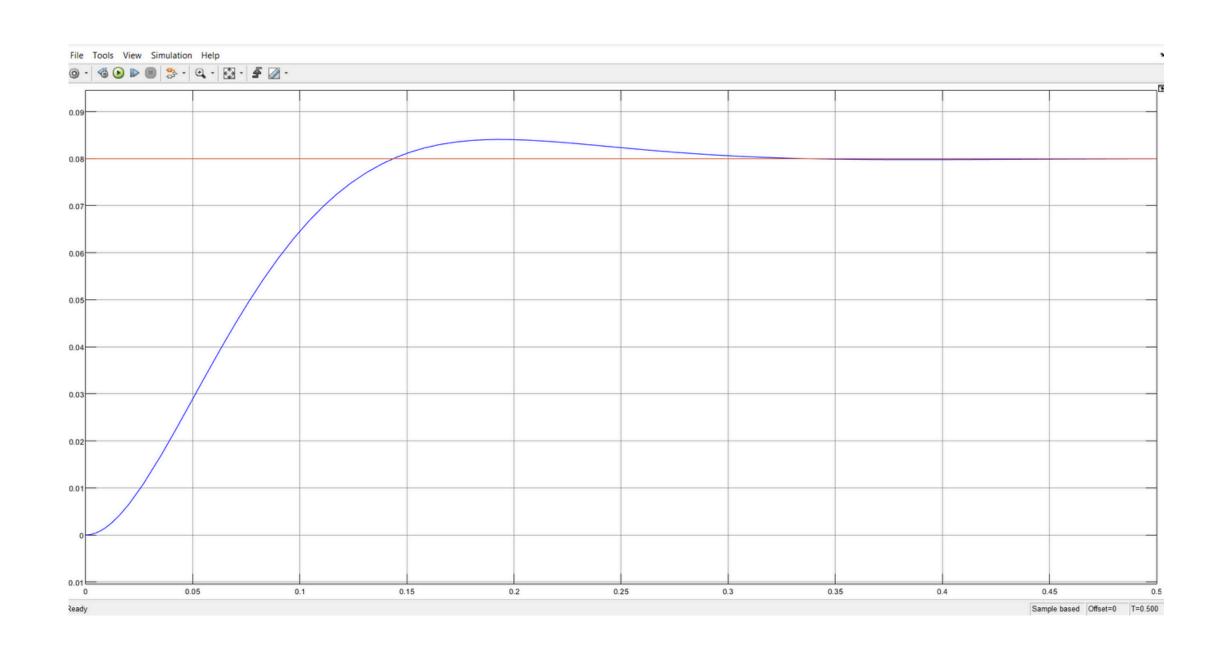
# **Bode plot**



#### **Root Locus**



# Performance

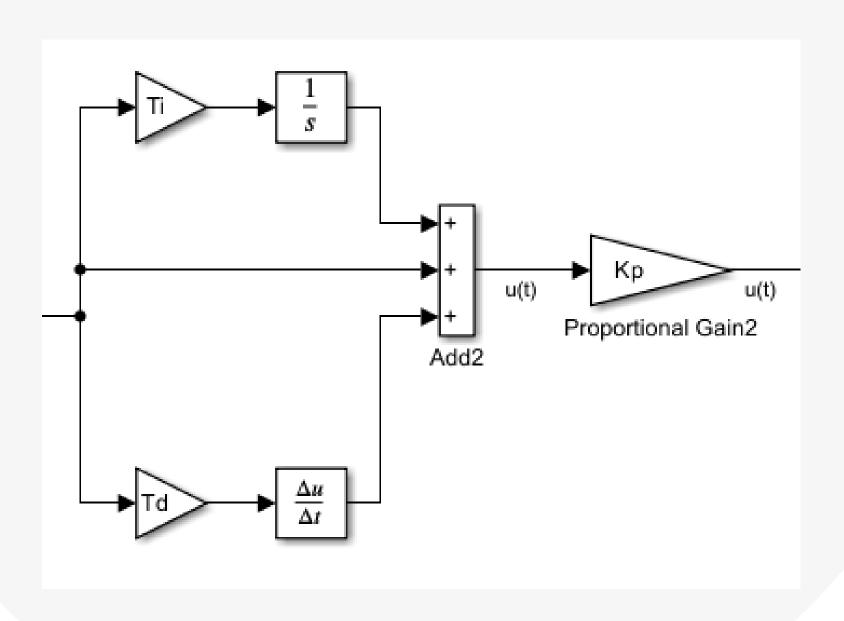


**Overshoot: 5%** 

Settling time: 0.324 sec

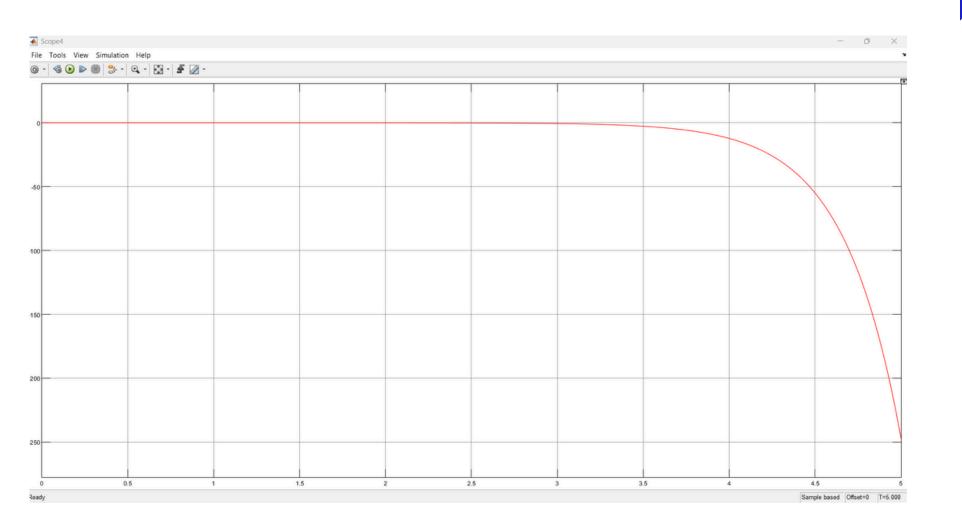
# Control Strategy Design

The control strategy design we went with is the PID design which is shown as follows

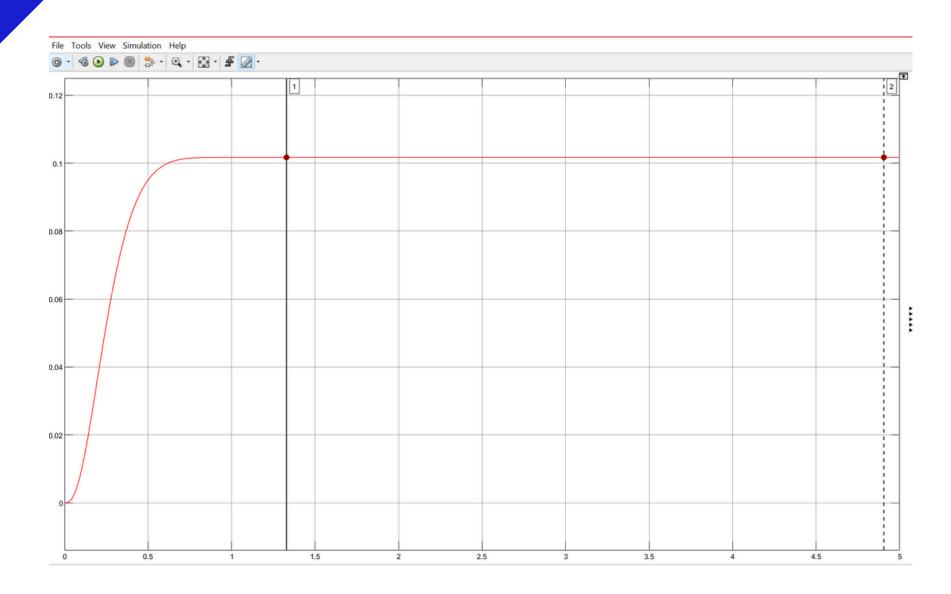




#### Without tuning



#### With tuning



# Reason for chosen tuning parameters

Chosen parameters:

# Calculating the PD controller values

From transfer function

$$G(s) = rac{Z(s)}{V(s)} = rac{-0.6071}{s^3 + 31.44s^2 - 0.0076s - 309.64}$$

$$G(s) = rac{num}{(s+3.3184)(s-2.9986)} = rac{num}{s^2 + 0.319s - 9.95}$$

$$\lim_{s=0} G(s) = rac{-0.6071}{0^3 + 31.44 * 0^2 - 0.0076 * 0 - 309.64} \ \lim_{s=0} G(s) = rac{-0.6071}{-309.64} \ DC \; gain = 0.00196$$

$$G(s) = rac{0.0195}{s^2 + 0.319s - 9.95}$$

$$\zeta = rac{-\ln(O.\,S)}{\sqrt{\pi^2 + \ln^2(O.\,S))}} \ \zeta = rac{-\ln(0.\,01)}{\sqrt{\pi^2 + \ln^2(0.\,01)}} \ \zeta = 0.\,826$$

$$ts=rac{4}{\zeta*wn}$$
  $wn=rac{4}{\zeta*ts}$   $wn=rac{4}{0.\,826*0.\,8}$   $wn=6.\,053$ 

$$G(s) = s^2 + 2*wn*\zeta + wn^2$$

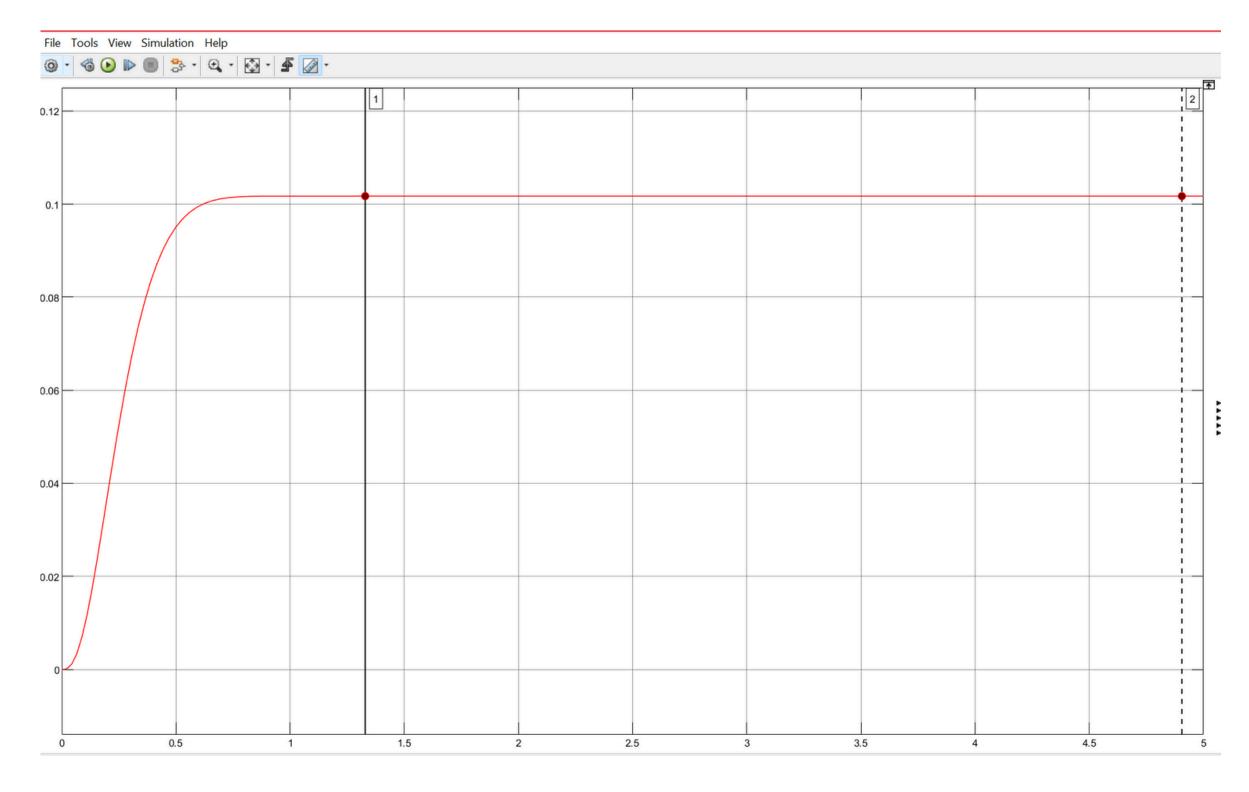
$$G(s) = s^2 + 2 * 6.053 * 0.826 + 6.053^2$$

$$G(s) = s^2 + 9.963s + 36.6388$$

$$Kp = -2389.18$$

$$Td = 0.207$$

$$G(s) = rac{-0.0195 - 0.0195 * Kp * Tds}{s^2 + (0.319 - 0.0195 * Kp * Td)s - 9,95 - 0.0195 * Kp}$$



Overshoot: 1%
Settling time: 0.8 sec

### Calculating the PID

$$G(s) = s^2 + 9.963s + 36.6388$$



$$Ti \geq rac{2}{|Re\{pcl\}|}$$

## Challenges

- Developing the Block Diagram
- Tuning of the PID controller

# Conclusion

