

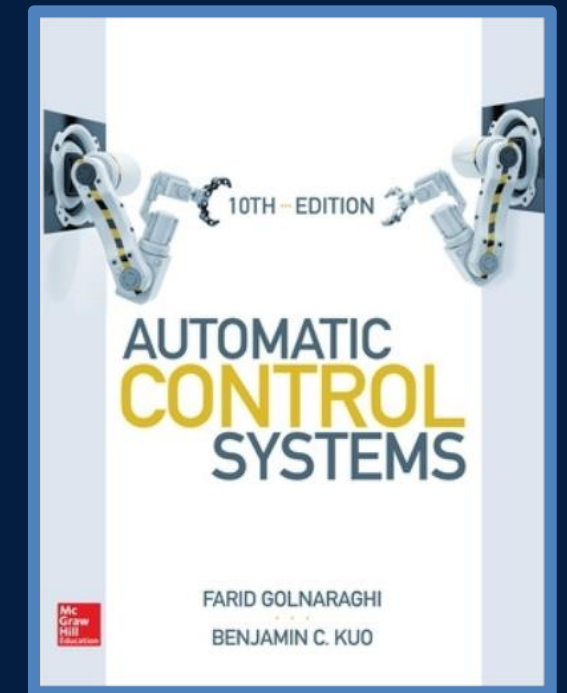
HUMBER ENGINEERING

MENG 3510 – Control Systems
LECTURE 7

LECTURE 7

Root Locus Design

- Properties of Root-Locus
 - Magnitude and Angle Conditions
- Control System Design via Root-Locus
 - Static Feedback Design
 - Effect of Adding a Pole/Zero to Root-Locus
 - Dynamic Compensator Design
 - Lead & Lag Compensators
 - PD & PI Controller

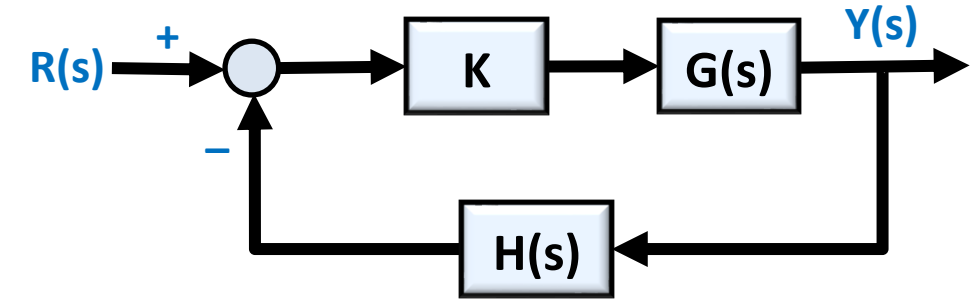


Chapter 9 & 11

Properties of Root Locus

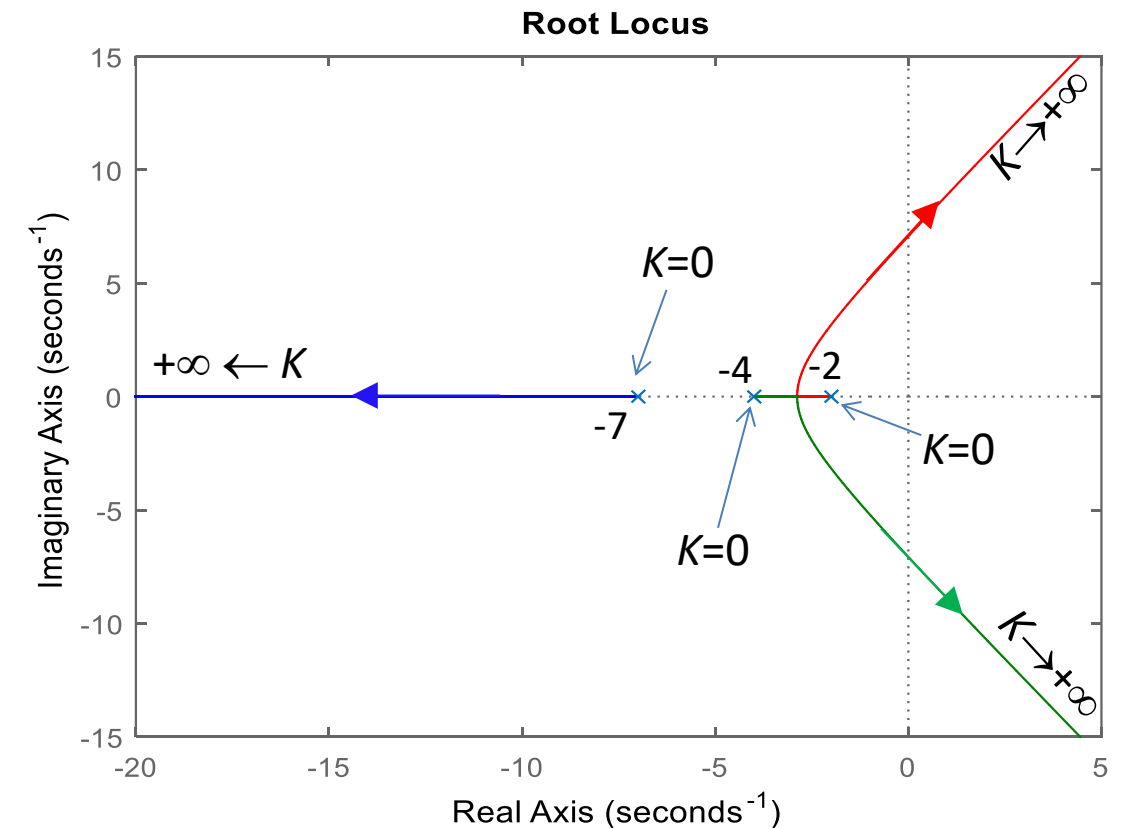
- Root-locus** is a graphical technique to show the **closed-loop pole locations** by variation of a certain **parameter**, such as **loop-gain** K in the following closed-loop system:

$$T(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1 + KG(s)H(s)}$$



- Stability** and **performance** of the closed-loop system depends on the closed-loop poles location.

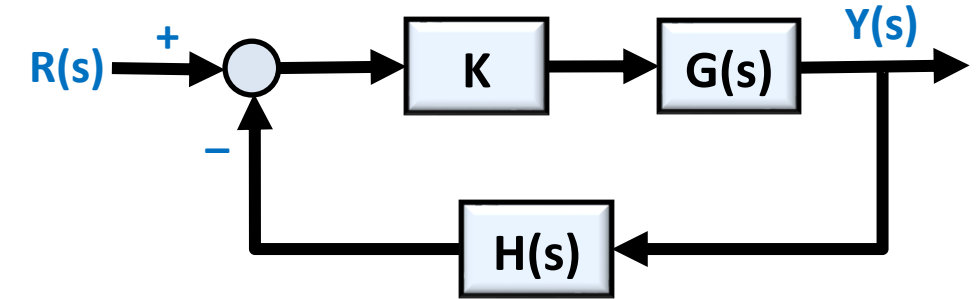
How to design a controller via root-locus to achieve the desire performance?



Properties of Root Locus

- Consider the following closed-loop system with adjustable gain K

$$T(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1 + KG(s)H(s)}$$



- The closed-loop characteristic equation is

$$1 + KG(s)H(s) = 0 \rightarrow KG(s)H(s) = -1$$

- Since $KG(s)H(s)$ is a complex quantity, it can be written as

$$KG(s)H(s) = \underbrace{|KG(s)H(s)|}_{\text{magnitude}} \underbrace{\angle(KG(s)H(s))}_{\text{angle}} = -1$$

- Therefore, the closed-loop poles must satisfy the following magnitude and angle conditions:

Magnitude and Angle Conditions for $K \in [0 + \infty)$:

$$|KG(s)H(s)| = 1$$

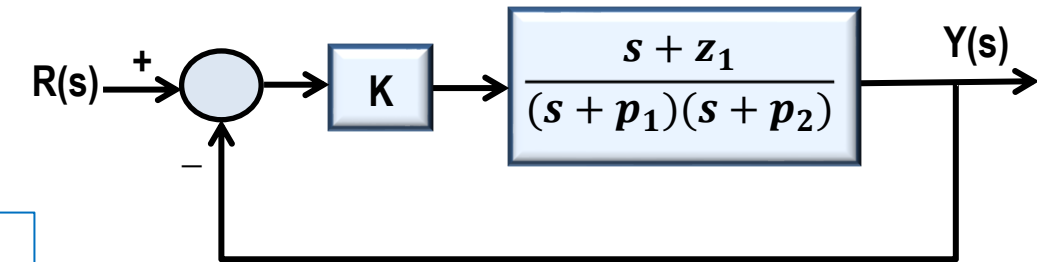
$$\angle(KG(s)H(s)) = \pm(2i + 1)180^\circ, \quad i = 0, 1, 2, \dots$$

The values of s that fulfill the magnitude and angle conditions are poles of the closed-loop system and located on the root-locus.

Properties of Root Locus - Example

Example 1

Consider the following system with arbitrary poles and zeros at $-p_1$, $-p_2$ and $-z_1$



Assume the point A as an arbitrary point in the s-plane.

Angle Condition →

$$\angle(KG(s)H(s)) = \pm(2i + 1)180^\circ$$

Check the angle condition for point A.

$$\angle\left(K \frac{s + z_1}{(s + p_1)(s + p_2)}\right) = (\angle K + \angle(s + z_1)) - (\angle(s + p_1) + \angle(s + p_2)) = \pm(2i + 1)180^\circ$$

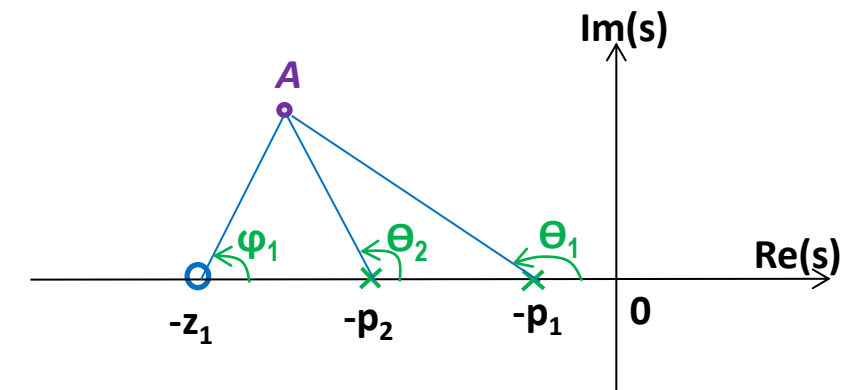
Method 1: Calculation the angles by evaluation at point A

$$\left(\tan^{-1} \left(\frac{\text{Im}[K]}{\text{Re}[K]} \right) + \tan^{-1} \left(\frac{\text{Im}[s_A + z_1]}{\text{Re}[s_A + z_1]} \right) \right) - \left(\tan^{-1} \left(\frac{\text{Im}[s_A + p_1]}{\text{Re}[s_A + p_1]} \right) + \tan^{-1} \left(\frac{\text{Im}[s_A + p_2]}{\text{Re}[s_A + p_2]} \right) \right) = \pm(2i + 1)180^\circ$$

Method 2: Geometrically by measuring the angles

$$(0 + \varphi_1) - (\theta_1 + \theta_2) = \pm(2i + 1)180^\circ$$

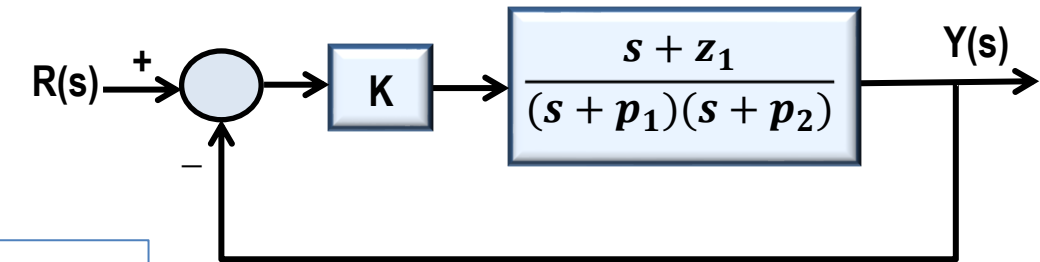
If point A satisfies the **angle condition** it means it is located **on the root-locus of this system**.



Properties of Root Locus - Example

Example 1

Consider the following system with arbitrary poles and zeros at $-p_1$, $-p_2$ and $-z_1$



Next, we can use the **magnitude condition** to find the magnitude of K at point A.

Magnitude Condition →

$$|KG(s)H(s)| = 1 \rightarrow |K| = \frac{1}{|G(s)H(s)|}$$

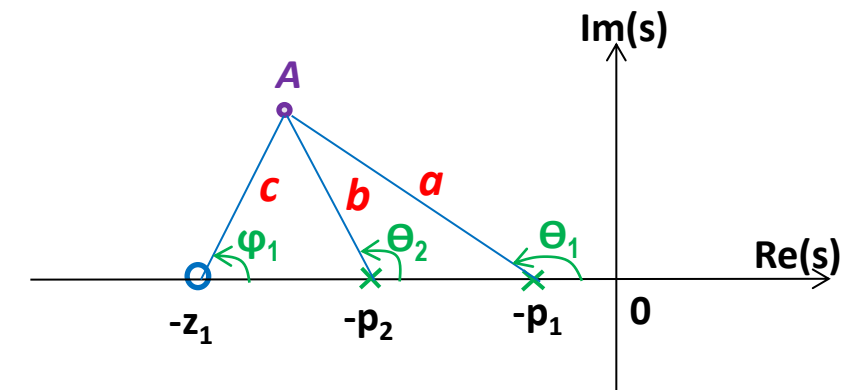
Method 1: Calculation the gain by evaluation at point A

$$|K| = \frac{1}{|G(s)H(s)|} = \frac{|s + p_1||s + p_2|}{|s + z_1|} \Bigg|_{\text{at point A}} = \frac{|s_A + p_1||s_A + p_2|}{|s_A + z_1|}$$

Method 2: Geometrically by measuring length of the vectors

$$|K| = \frac{1}{|G(s)H(s)|} = \frac{|s + p_1||s + p_2|}{|s + z_1|} \Bigg|_{\text{at point A}}$$

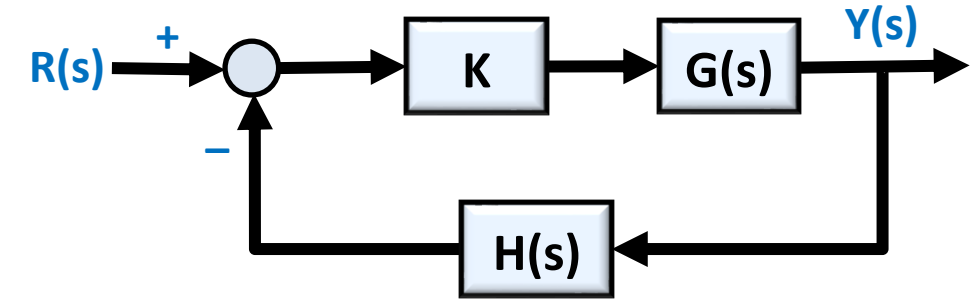
$$|K| = \frac{a \times b}{c}$$



Control System Design via Root-Locus

- Consider the following closed-loop system with adjustable gain K

$$T(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1 + KG(s)H(s)}$$



- There are two approaches to **design a control system** by using the **Root-locus diagram**:

❑ Static Feedback Design

- Selecting the value of K from root-locus in order to **place the closed-loop poles** at the **desired locations** and satisfy the desired performance criteria.
- The technique is similar to the **Proportional Controller** design. Since the control signal $u(t)$ is proportional to the error signal $e(t)$ via the **static gain K** .

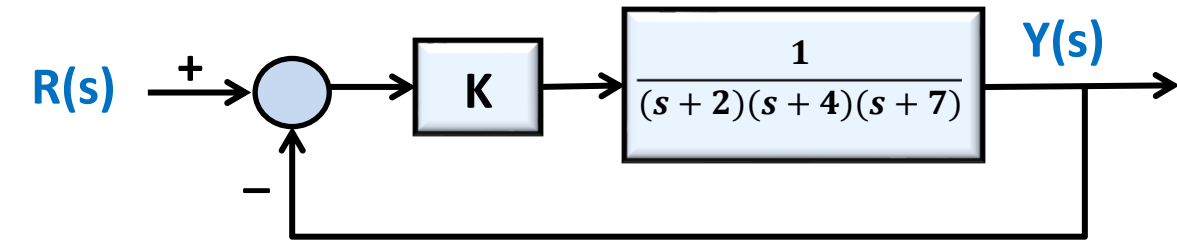
❑ Dynamic Compensator Design

- If the desired performance criteria cannot be obtained by adjusting the gain K *only*, then we need to **reshape the root-locus** by **adding some poles/zeros** as a **compensator**, such as:
 - Lead & Lag Compensators**
 - PD & PI Controllers**

Static Feedback Design via Root Locus

Example 2

Consider the following third-order system. Determine the K value so that the maximum overshoot of unit-step response is 10%.



First, calculate the **desired damping ratio** from the desired maximum overshoot value,

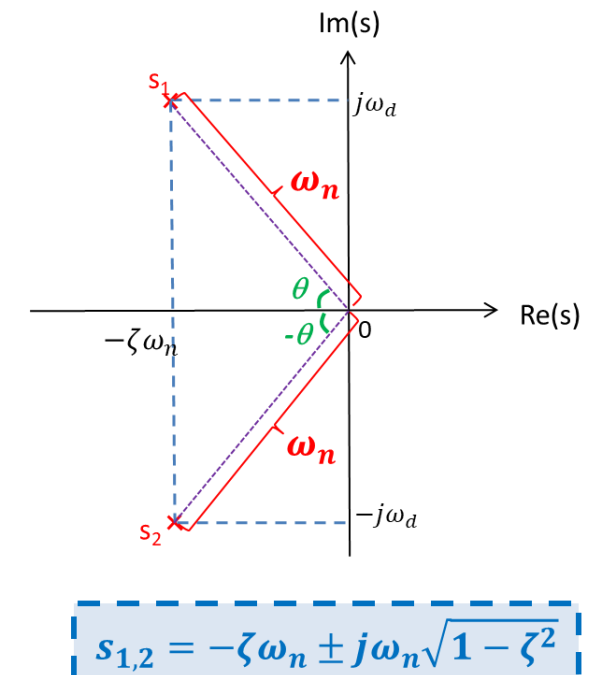
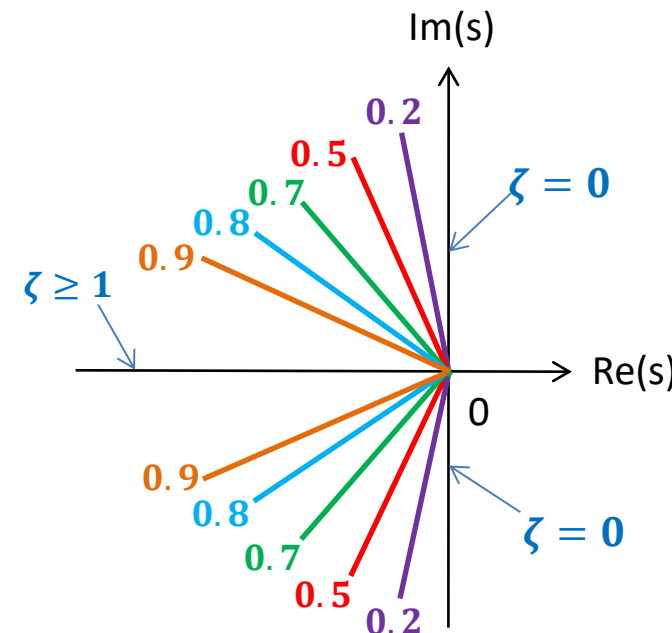
$$\zeta = \frac{-\ln(O.S.)}{\sqrt{\pi^2 + \ln^2(O.S.)}} \rightarrow \zeta = \frac{-\ln(0.1)}{\sqrt{\pi^2 + \ln^2(0.1)}} \rightarrow \boxed{\zeta = 0.5912} \quad \text{Desired Damping Ratio}$$

Next, plot the **root-locus** for this system, and sketch the **constant-damping-ratio** lines of $\zeta = 0.5912$.

The **intersection** of the lines with root-locus will be the **desired pole locations**.

- Recall that the **constant-damping-ratio ζ loci** in the s-plane are **radial lines** passing through the origin.

$$\boxed{\zeta = \cos \theta \rightarrow \theta = \cos^{-1}(\zeta)}$$



Static Feedback Design via Root Locus

Example 2

Consider the following third-order system. Determine the K value so that the maximum overshoot of unit-step response is 10%.

poles $\rightarrow p_1 = -2, p_2 = -4, p_3 = -7$

zeros \rightarrow No finite zeros, three zeros at infinity

$$\zeta = 0.5914 \rightarrow \theta = \cos^{-1}(\zeta) = 53.76^\circ \approx 54^\circ$$

From the graph the desired pole locations are: $s_d = -2.1 \pm j3$

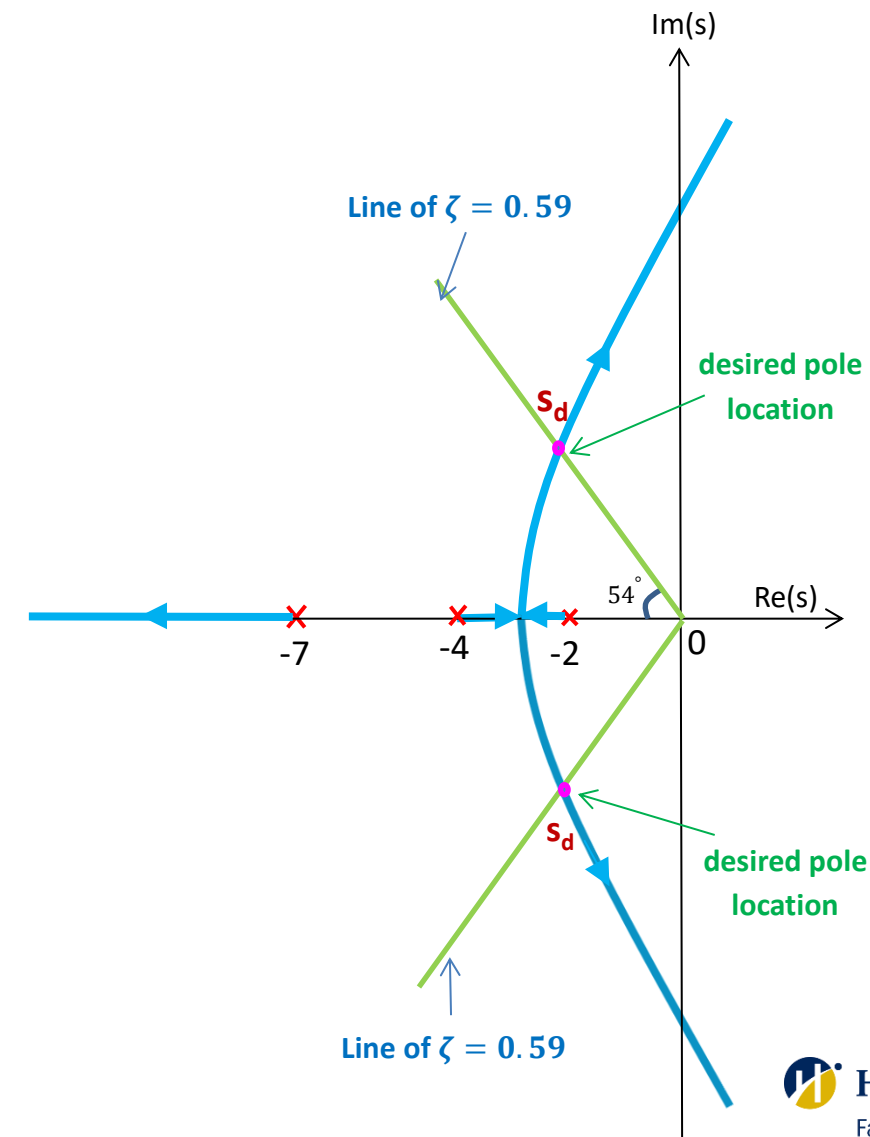
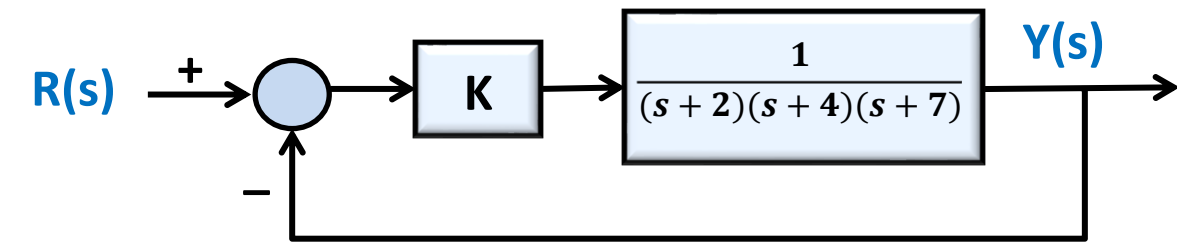
Find the gain K at the desired pole locations by using the magnitude condition:

$$|KG(s)H(s)|_{s=s_d} = 1 \rightarrow |K| = \frac{1}{|G(s_d)H(s_d)|}$$

$$|KG(s)H(s)| = 1 \rightarrow \left| \frac{K}{(s+2)(s+4)(s+7)} \right| = 1$$

$$|K| = |(s+2)(s+4)(s+7)|_{s=s_d}$$

$$|K| = |s_d + 2||s_d + 4||s_d + 7|$$



Static Feedback Design via Root Locus

Example 2

Consider the following third-order system. Determine the K value so that the maximum overshoot of unit-step response is 10%.

poles $\rightarrow p_1 = -2, p_2 = -4, p_3 = -7$

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From the graph the desired pole locations are: $s_d = -2.1 \pm j3$

Method 1: Calculation the gain by evaluation at point s_d

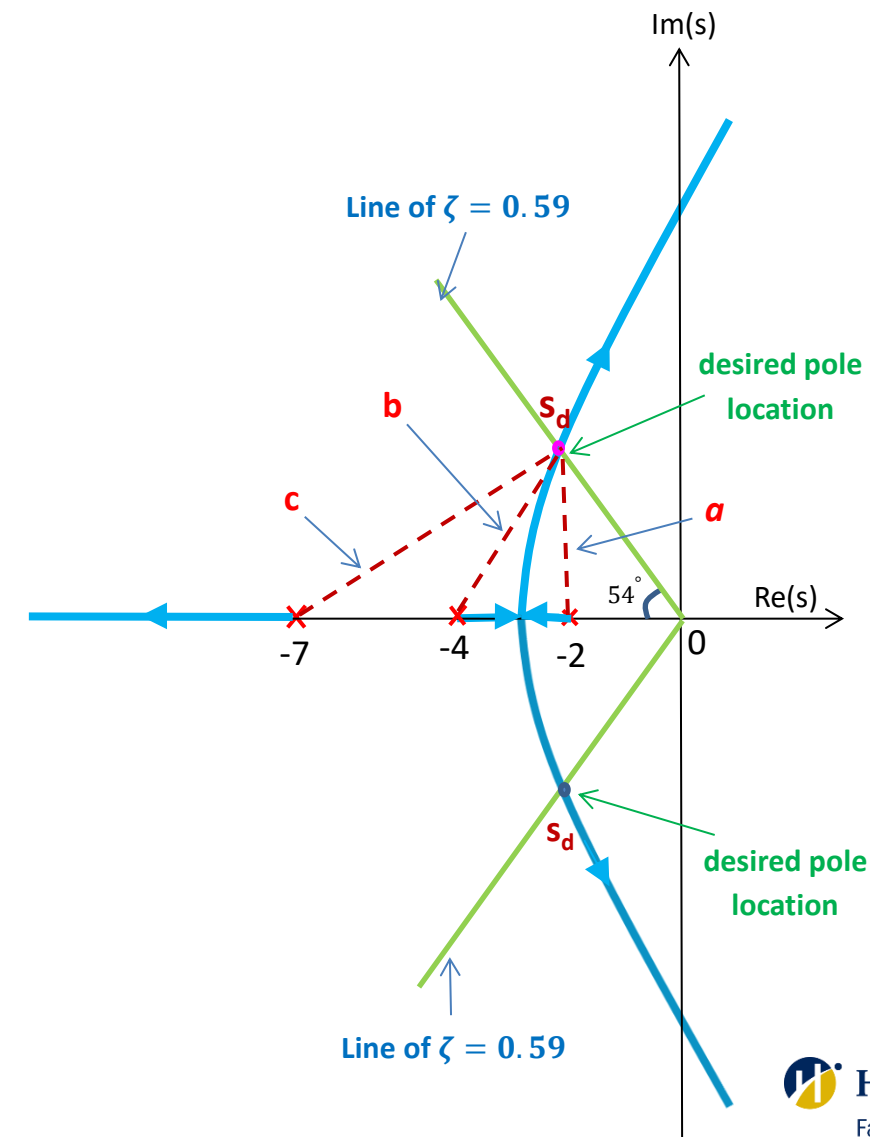
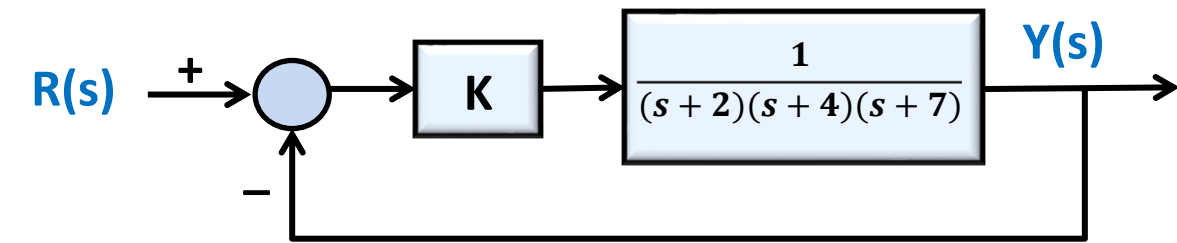
$$|K| = |s_d + 2||s_d + 4||s_d + 7|$$

$$|K| = |-0.1 + j3||1.9 + j3||4.9 + j3| = 3 \times 3.6 \times 5.7 = 61.56$$

Method 2: Geometrically by measuring length of the vectors

$$|K| = |s_d + 2||s_d + 4||s_d + 7|$$

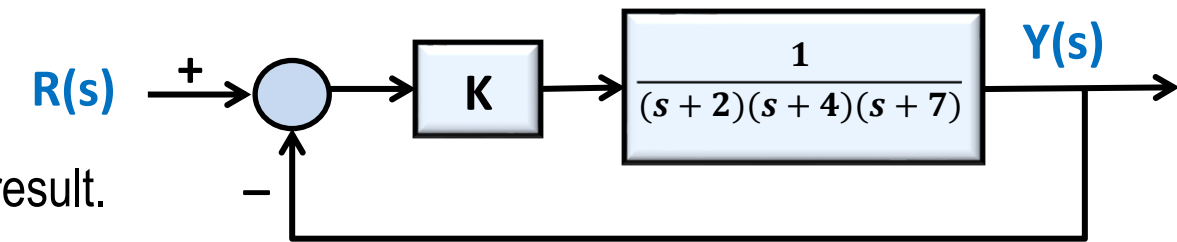
$$|K| = a \times b \times c = (3.1)(3.5)(5.7) = 61.87$$



Static Feedback Design via Root Locus

Example 2

Consider the following third-order system. Determine the K value so that the maximum overshoot of unit-step response is 10%.



We can also plot the unit-step response of the closed-loop system in MATLAB to check the result.

The open-loop transfer function for $K = 61.56$ is

$$KG(s)H(s) = \frac{61.56}{s^3 + 13s^2 + 50s + 56}$$

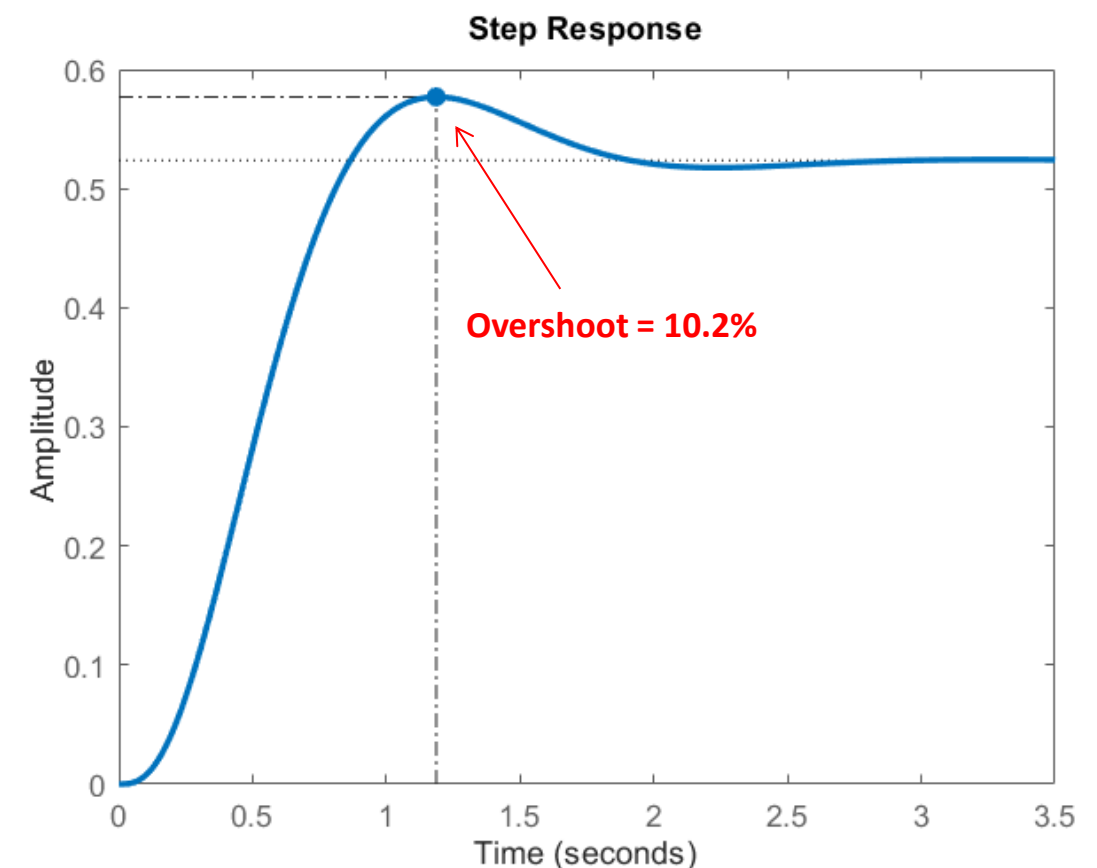
```
num = [61.56];  
den = [1 13 50 56];  
sys_OL = tf(num,den);  
sys_CL = feedback(sys_OL,1);  
step(sys_CL)
```

The closed-loop transfer function, and the closed-loop poles are:

$$\frac{Y(s)}{R(s)} = \frac{61.56}{s^3 + 13s^2 + 50s + 117.6}$$

$$s_{1,2} = -2.071 \pm j2.9965, \quad s_3 = -8.8518$$

Dominant poles



Design Aspects of Root Locus

- The general problem of controller design in control systems may be treated as an investigation of the effects to the root loci when **poles** and **zeros** are added to the **open-loop transfer function $G(s)H(s)$** .

Consider the following **second-order** system.

The goal is to find the K value so that the desired closed-loop poles have a **damping-ratio of 0.5** and **undamped natural frequency of 8 rad/sec**.

First, calculate the **desired pole locations** based on the given damping ratio and the undamped natural frequency values.

$\zeta = 0.5$

$\omega_n = 8$

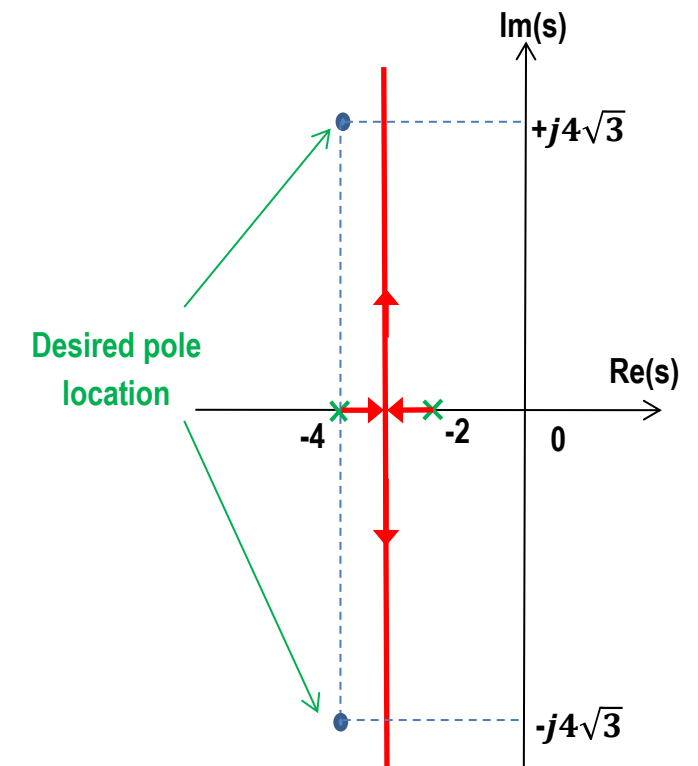
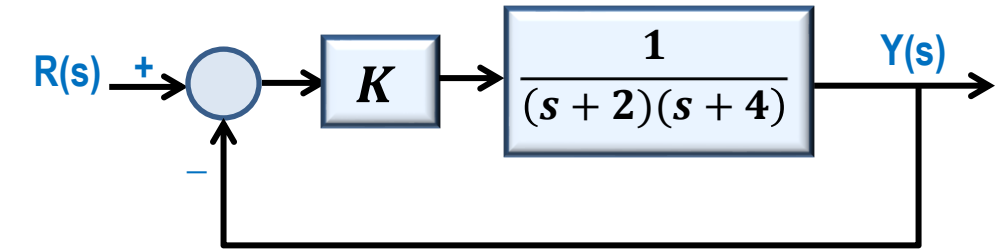
→

$s = -\zeta\omega_n \pm j\omega_n\sqrt{1 - \zeta^2} \rightarrow s = -4 \pm j4\sqrt{3}$ **Desired Poles**

Next, plot the **root-locus** for this system, and locate the desired poles.

- The **root-locus** does not pass the desired pole locations.
- The desired characteristics are **not achievable** by only adjusting the gain K value.

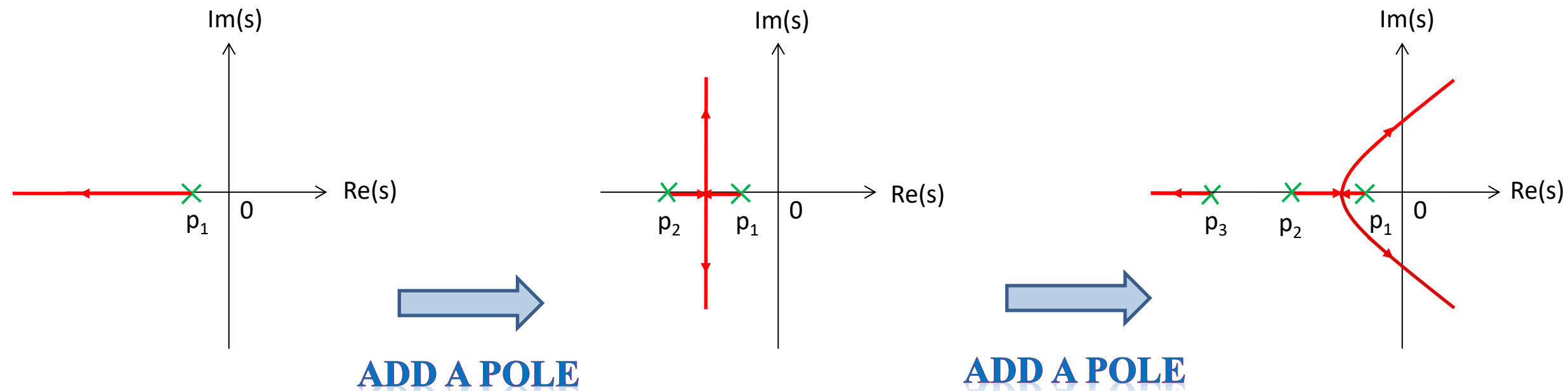
Reshape the root-locus by adding poles and zeros.



Design Aspects of Root Locus

- The general problem of controller design in control systems may be treated as an investigation of the effects to the root loci when **poles** and **zeros** are added to the **open-loop transfer function $G(s)H(s)$** .

□ Effect of Adding Poles on Root-Locus

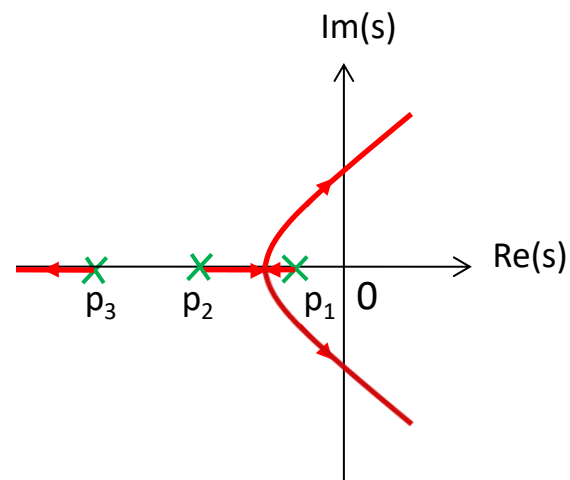


- Pulling the **root-locus** to the **right**
- Decreasing** the relative **stability** of closed-loop system
- Increasing** the **overshoot** of closed-loop response
- Slow down** the **settling** of the response

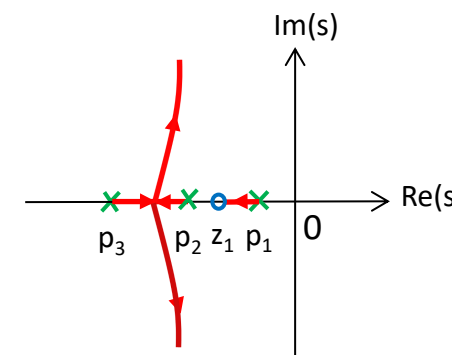
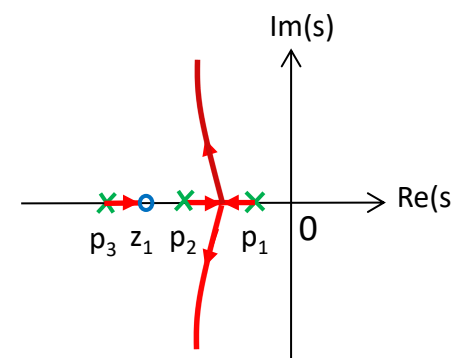
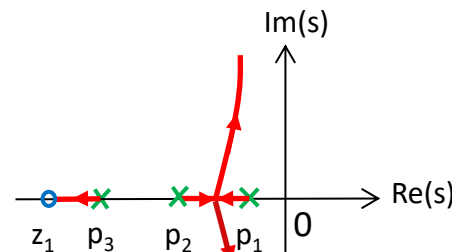
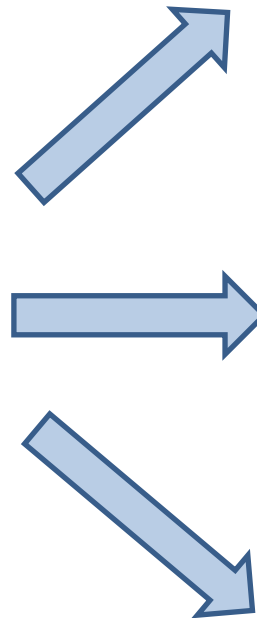
Design Aspects of Root Locus

- The general problem of controller design in control systems may be treated as an investigation of the effects to the root loci when **poles** and **zeros** are added to the **open-loop transfer function $G(s)H(s)$** .

□ Effect of Adding Zeros on Root-Locus



ADD A ZERO



- Pulling the **root-locus** to the **left**
- Increasing** the relative **stability** of closed-loop system
- Speed up** the **settling time** of the response
- Amplifies** the high-frequency **noise**

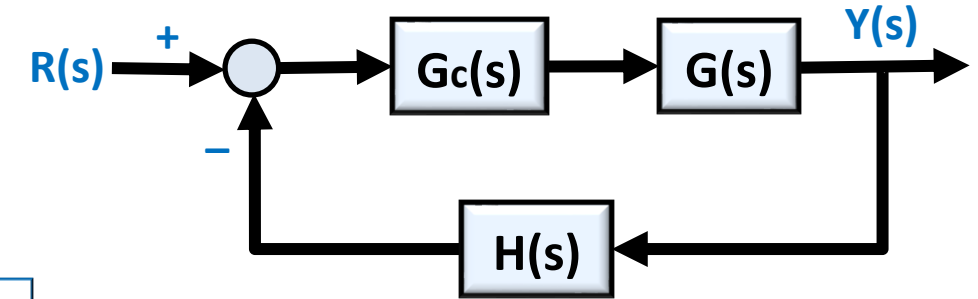
- Based to the effect of adding poles/zeros on root-locus:
- Adding only zero** is often problematic, because it **amplifies the high-frequency noise**.
- Adding only pole** generates a **less stable system** by moving the root-locus (closed-loop poles) to the right.
- Therefore, we need to **add both zero** and **pole** to design a **compensator**.

Dynamic Compensator Design

□ Lead Compensator & Lag Compensator

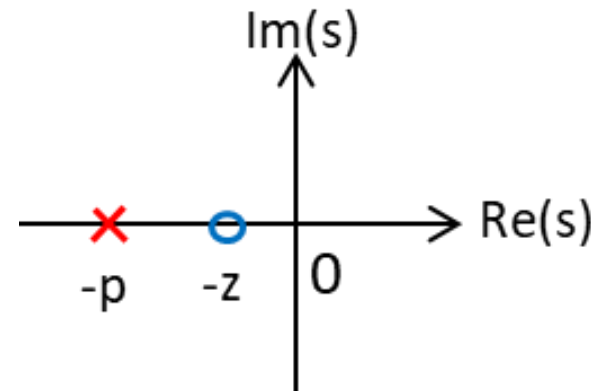
- In general, a **compensator** has the following transfer function

$$G_c(s) = K_c \frac{s + z}{s + p}, \quad z > 0, \quad p > 0$$

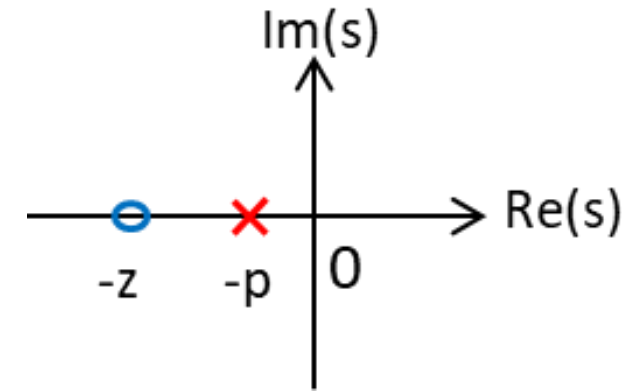


- According to the **pole/zero locations** we have the following structures:

- To improve transient response and stability.
- Similar to PD controller.
- Positive angle contribution.



Lead Compensator



Lag Compensator

- To improve steady-state error.
- Similar to PI controller.
- Negative angle contribution.

- Combined **lead-lag compensator** can improve both **transient response** and **steady-state response**, similar to a **PID controller**.

Lead Compensator Design via Root Locus

Step 1: Determine **desired location** of the dominant **closed-loop poles**.

Step 2: Plot the axes of **s-plane** and mark **open-loop poles/zeros** and the **desired closed-loop poles**.

Step 3: Find the **sum of the angles** at the desired location of the dominant closed-loop pole.

Step 4: Determine the **angle deficiency**, ϕ , which is necessary to satisfy the angle condition.

Step 5: Design a **lead compensator** to compensate the angle deficiency by determining the **locations of the pole/zero** as below:

- P is the desired pole location
- Draw lines PA and PO
- Draw bisector line PB

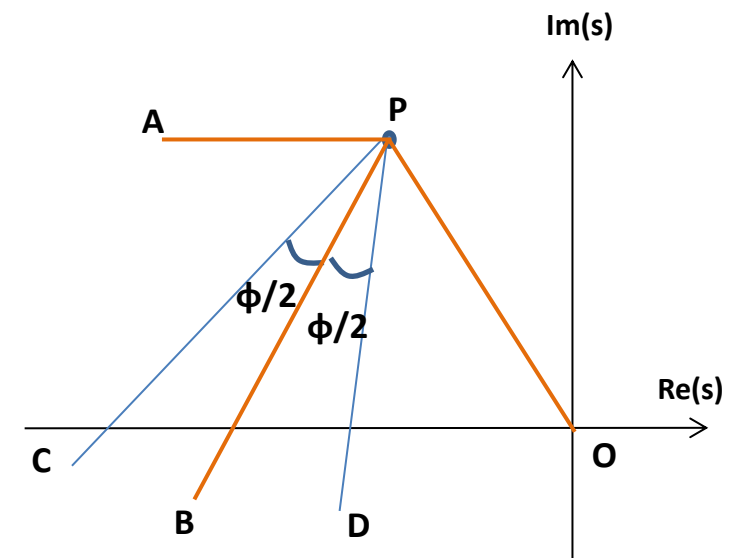
$$\angle APB = \angle BPO = \frac{\angle APO}{2}$$

- Draw lines PC and PD so that

$$\angle CPB = \angle BPD = \phi/2$$

- Pole and zero are the intersections of PC and PD with real axis

$$G_c(s) = K_c \frac{s + z}{s + p}$$

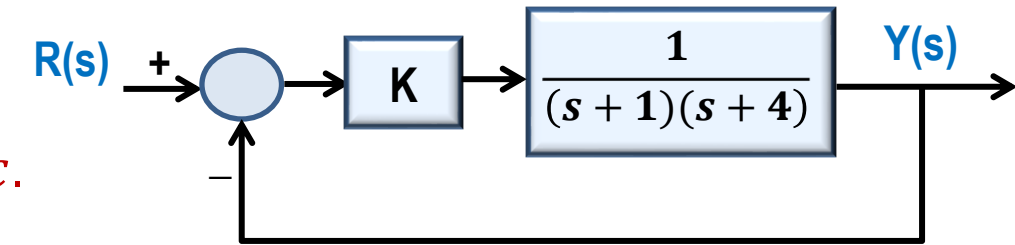


Step 6: Determine **gain** of the compensator by using the root-locus **magnitude condition** or the given **steady-state error condition**.

Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system
Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



Step 1: Determine the desired dominant closed-loop pole locations

First, determine the desired damping ratio and undamped natural frequency based on the desired maximum overshoot and settling time value:

$$\zeta = \frac{-\ln(O.S.)}{\sqrt{\pi^2 + \ln^2(O.S.)}} \rightarrow \zeta = \frac{-\ln(0.1)}{\sqrt{\pi^2 + \ln^2(0.1)}} \rightarrow \boxed{\zeta = 0.5912} \quad \text{Desired Damping Ratio}$$

$$t_s \approx \frac{4}{\zeta\omega_n} \rightarrow 1 = \frac{4}{0.6\omega_n} \rightarrow \boxed{\omega_n = 6.7659} \quad \text{Desired Natural Frequency}$$

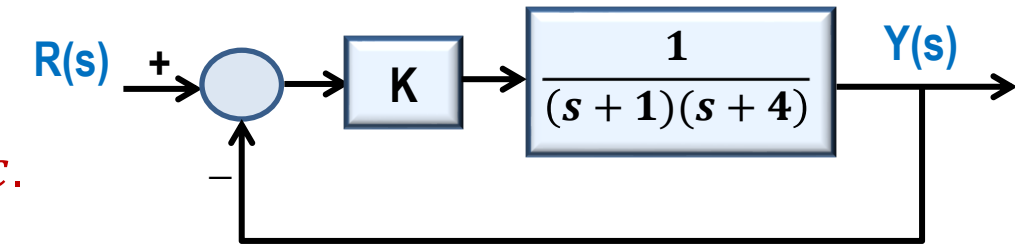
The desired closed-loop poles location $\rightarrow s = -\zeta\omega_n \pm j\omega_n\sqrt{1 - \zeta^2} \rightarrow \boxed{s_d = -4 \pm j5.5}$
Desired Closed-loop Poles

Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system

Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



Step 2: Plot the axes of **s-plane** and mark **open-loop poles/zeros** and the **desired closed-loop poles**.

poles $\rightarrow p_1 = -1, p_2 = -4$

zeros \rightarrow No finite zeros

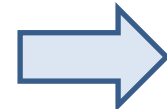
Desired closed-loop poles $\rightarrow s_d = -4 \pm j5.5$

Step 3: Find **sum of the angles** at the desired closed-loop poles location.

- Apply the **angle condition** to check that if the desired poles are on the root-locus or not

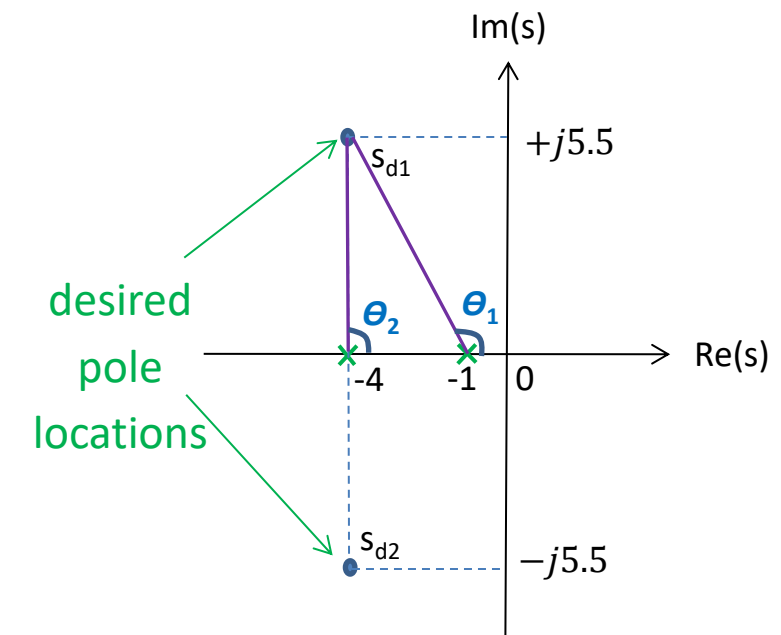
$$\begin{aligned} \angle \left(\frac{K}{(s+1)(s+4)} \right) \bigg|_{s=s_{d1}} &= \angle K - \angle(s+1) - \angle(s+4) \bigg|_{s=-4+j5.5} \\ &= 0 - \angle\theta_1 - \angle\theta_2 \\ &= 0 - 120^\circ - 90^\circ = -210^\circ \end{aligned}$$

$$-210^\circ \neq \pm(2i+1)180^\circ$$



Desired poles are not on the root-locus

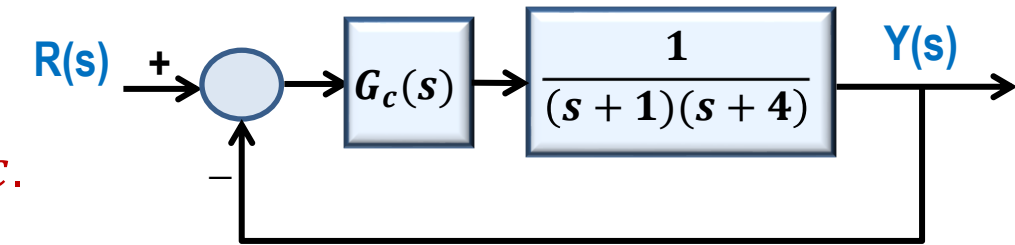
The angle condition is not satisfied



Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system
Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



$$G_c(s) = K_c \frac{s + z}{s + p}$$

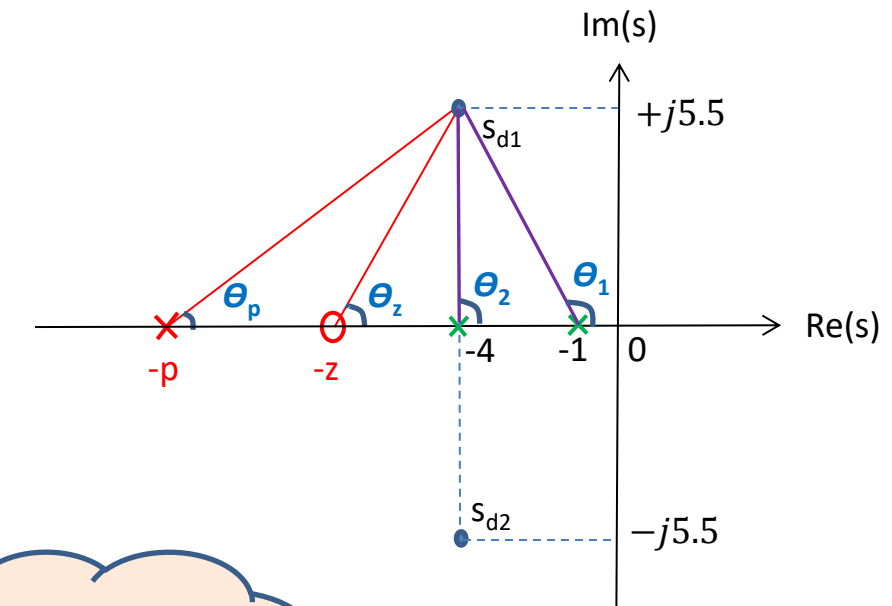
Step 4: Determine the required **angle deficiency** to satisfy the root-locus angle condition.

The **angle deficiency** is calculated as

$$-210^\circ + \phi = -180^\circ \rightarrow \phi = 210^\circ - 180^\circ = 30^\circ$$

Design a **lead compensator** to contribute the angle of $\phi = 30^\circ$ at the desired pole locations.

$$\begin{aligned} \angle G_c(s)G(s) \Big|_{s=s_{d1}} &= \angle \left(\frac{K_c(s+z)}{s+p} \cdot \frac{1}{(s+1)(s+4)} \right) \Big|_{s=s_{d1}} \\ &= \underbrace{\angle K_c}_{0^\circ} + \underbrace{\angle \theta_z - \angle \theta_p}_{30^\circ} - \underbrace{\angle \theta_1 - \angle \theta_2}_{-210^\circ} = -180^\circ \end{aligned}$$



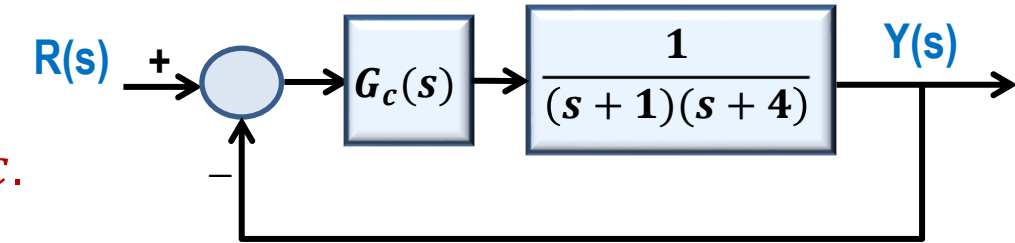
How to select the pole/zero locations?

Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system

Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



$$G_c(s) = K_c \frac{s + z}{s + p}$$

Step 5: Determine pole/zero locations of the **lead compensator** to compensate the angle deficiency.

- Draw lines PA and PO

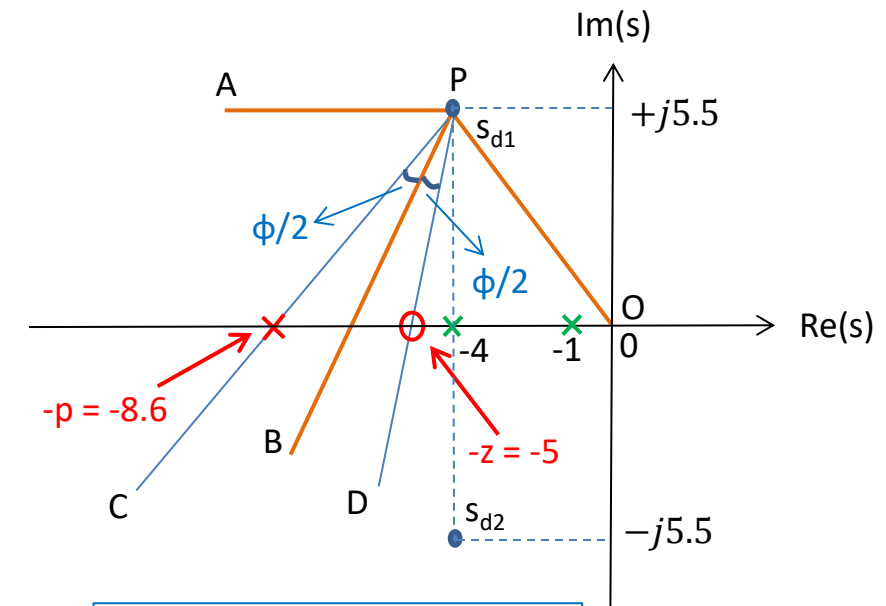
- Draw bisector line PB $\angle APB = \angle BPO = \frac{\angle APO}{2}$

- Draw lines PC and PD so that

$$\angle CPB = \angle BPD = \frac{\phi}{2} = \frac{30^\circ}{2} = 15^\circ$$

- Pole and zero are the intersections of PC and PD with real axis

$$G_c(s) = K_c \frac{s + 5}{s + 8.6}$$



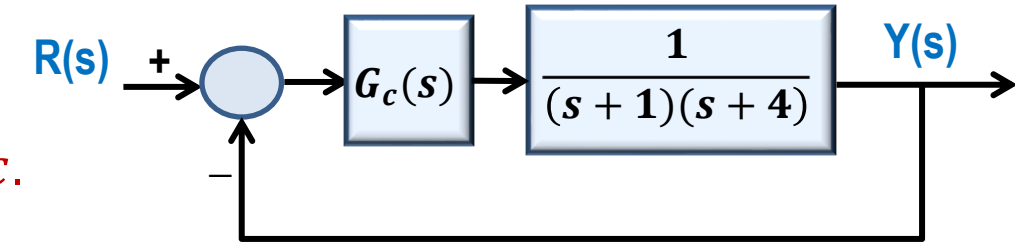
$$z = 5, \quad p = 8.6$$

Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system

Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



$$G_c(s) = K_c \frac{s + z}{s + p}$$

Step 6: Determine gain of the **lead compensator** from magnitude condition.

Next, calculate the **gain** K_c using the **magnitude condition**

$$|G_c(s)G(s)|_{s=s_d} = 1$$

$$\left| K_c \frac{s + 5}{s + 8.6} \cdot \frac{1}{(s + 1)(s + 4)} \right|_{s=-4+j5.5} = 1$$

$$|K_c| = \frac{|-3 + j5.5||j5.5||4.6 + j5.5|}{|1 + j5.5|} = \frac{\sqrt{(-3)^2 + (5.5)^2} \times 5.5 \times \sqrt{(4.6)^2 + (5.5)^2}}{\sqrt{(1)^2 + (5.5)^2}} = 44.32$$

$$K_c = 44.32$$

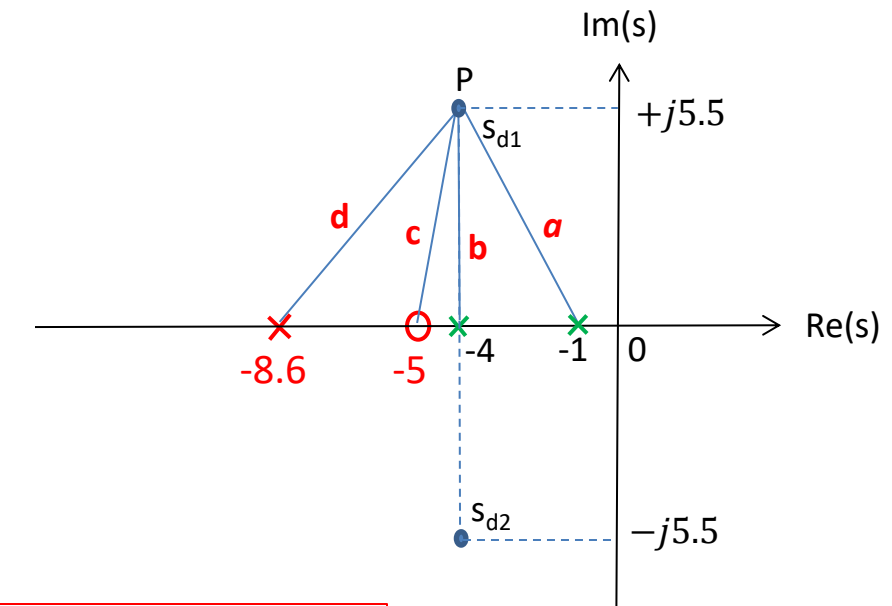
We can also find the **gain** K_c by measuring the length of the vectors connected from the desired pole to the other poles and zeros.

$$K_c = \frac{a \times b \times d}{c}$$



$$G_c(s) = K_c \frac{s + z}{s + p} = 44.32 \frac{s + 5}{s + 8.6}$$

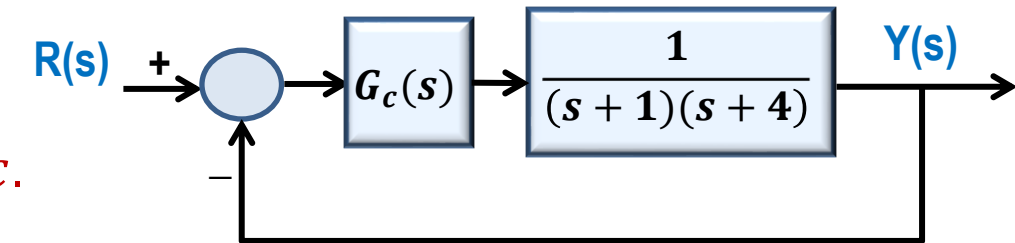
The designed lead compensator



Lead Compensator Design via Root Locus

Example 3

Consider the following second-order system
Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



Step 7: Analyze and verify the designed compensator.

Determine closed-loop transfer function of the compensated system and check the pole locations

$$T(s) = \frac{Y(s)}{R(s)} = \frac{G_c(s)G(s)}{1 + G_c(s)G(s)} \longrightarrow T(s) = \frac{44.32(s + 5)}{s^3 + 13.6s^2 + 91.32s + 256}$$

The closed-loop poles are located at

$$s_{1,2} = -4.07 \pm j5.51,$$

Dominant Poles

$$s_3 = -5.46$$

Close to the Zero

Desired Closed-loop Poles

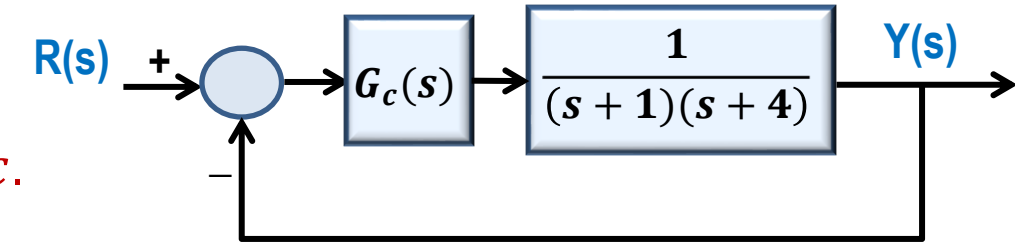
$$s = -4 \pm j5.5$$

- The dominant closed-loop poles are almost located at the desired places in the s-plane.
- The third pole at $s_3 = -5.46$ is very close to the added zero at $s = -5$. Therefore, the effect of this pole on the transient response is negligible.

Lead Compensator Design via Root Locus

Example 3

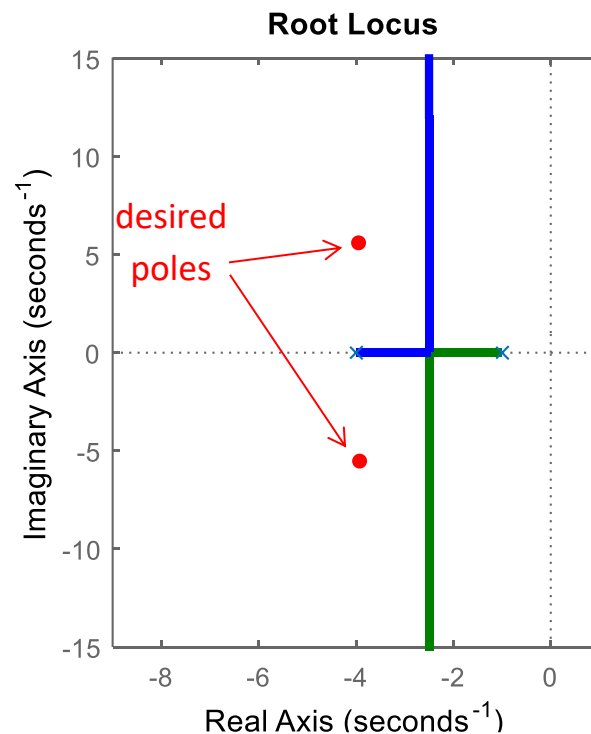
Consider the following second-order system
Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



Step 7: Analyze and verify the designed compensator.

Uncompensated System

$$G(s) = \frac{1}{(s+1)(s+4)}$$



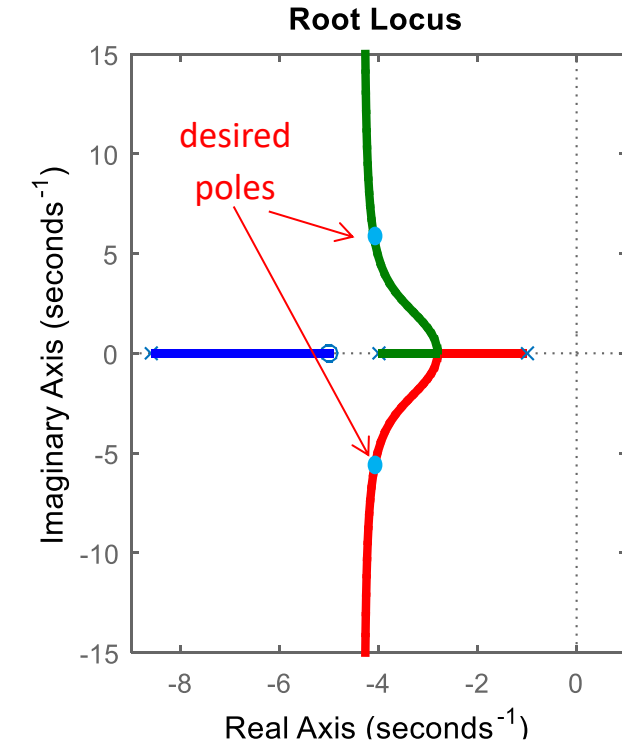
Desired Closed-loop Poles

$$s = -4 \pm j5.5$$

Lead compensator pulls the root-locus to the left and improves the relative stability of the system.

Compensated System

$$G_c(s)G(s) = \frac{44.32(s+5)}{(s+8.6)(s+1)(s+4)}$$

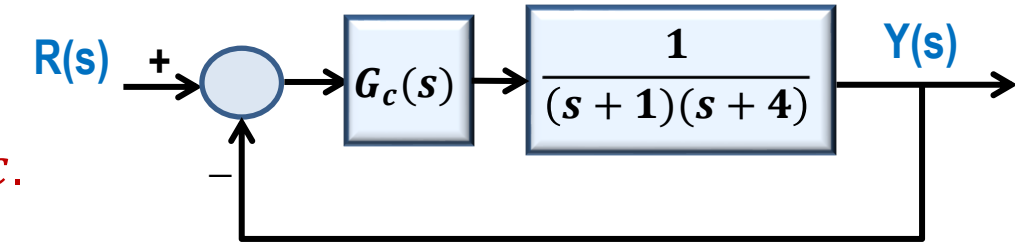


Lead Compensator Design via Root Locus

Example 3

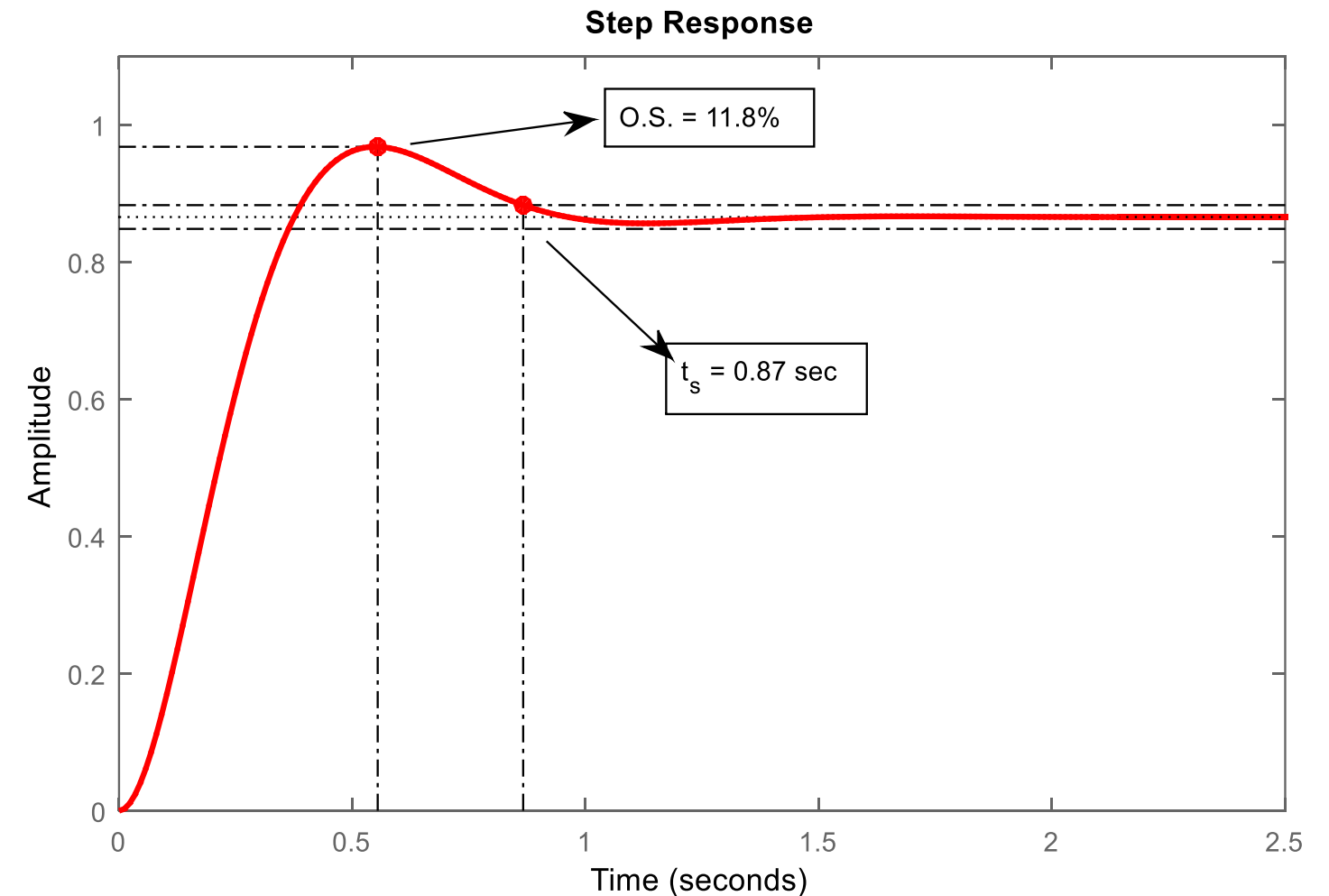
Consider the following second-order system

Design a lead compensator to speed up unit-step response with maximum overshoot of about 10% and settling time of less than 1sec.



Step 7: Analyze and verify the designed compensator.

- Step responses of the compensated closed-loop system has the overshoot of about 11.8% and settling time of 0.87sec.



PD Controller Design via Root Locus

- In designing the **lead compensator**, the pole-zero locations of the compensator are determined to compensate the **angle deficiency** of the desired closed-loop pole location.

$$G_c(s) = K_c \frac{s + z}{s + p}$$

$$\phi = \angle \theta_z - \angle \theta_p$$

Angle Deficiency

- The **angle deficiency**, ϕ , can also be compensated by adding an **only zero without pole**.

$$G_c(s) = K_c(s + z)$$

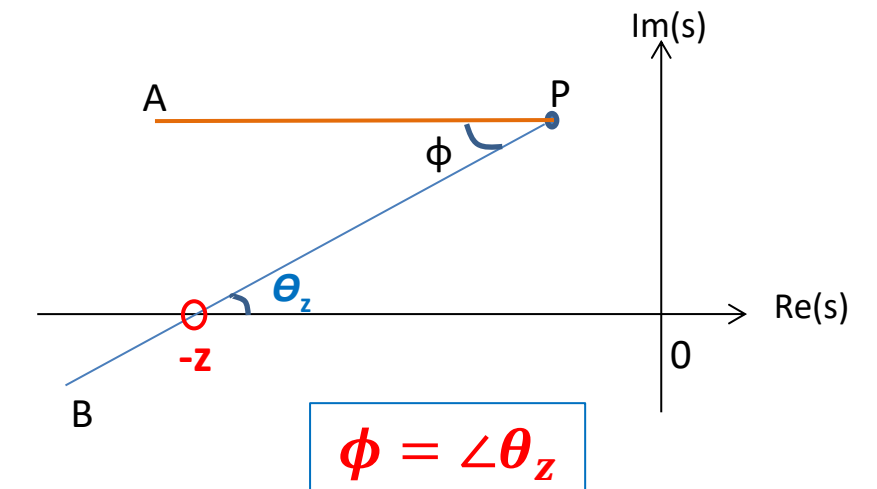
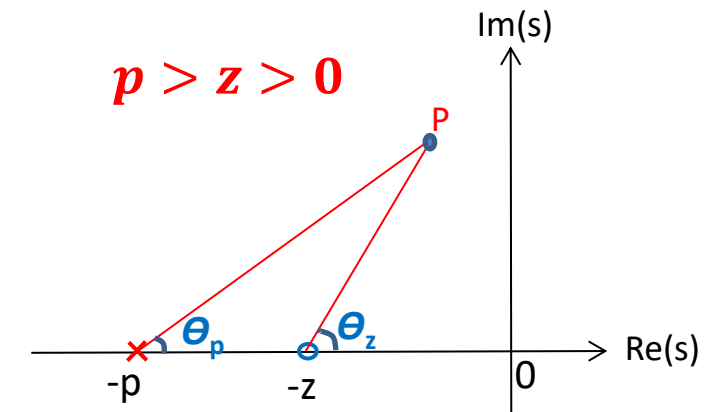
- P is the desired pole location
 - Draw line PA
 - Draw line PB such that $\angle APB = \phi$
 - The zero is located at the intersection of PB with real axis
 - Determine K_c by using the **magnitude condition**
- The **lead compensator without pole** can also be shown as a **PD controller**:

$$G_c(s) = K_c(s + z) = K_c z \left(\frac{s}{z} + 1 \right) \rightarrow G_c(s) = K_p (T_d s + 1)$$

where, the **proportional gain** K_p and the **derivative time-constant** T_d are defined as

$$T_d = \frac{1}{z}$$

$$K_p = K_c z$$

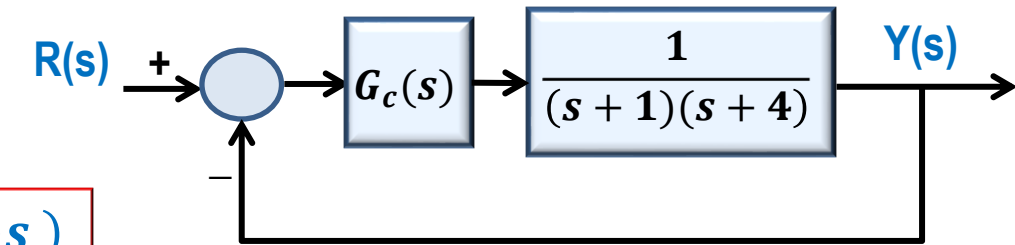


PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.

$$G_c(s) = K_p(1 + T_d s)$$



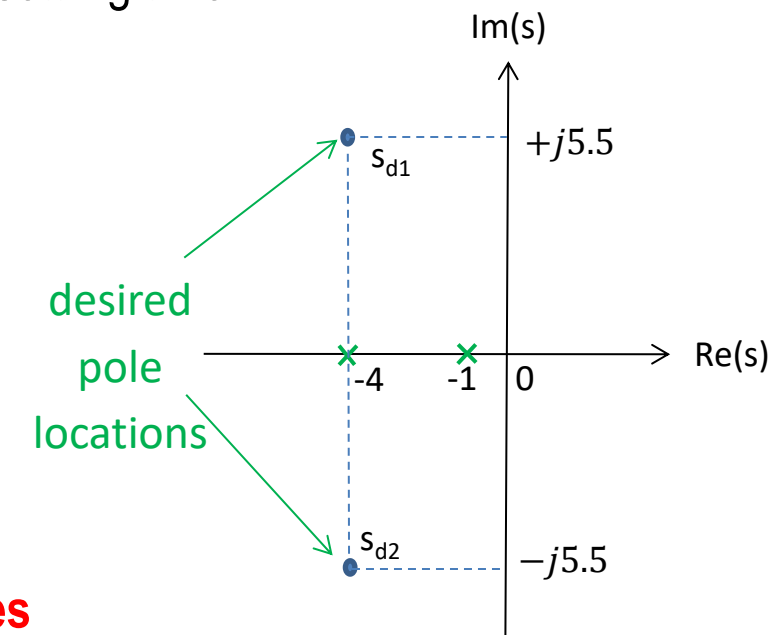
Step 1: Determine the desired dominant closed-loop pole locations

First, find the desired damping ratio and undamped natural frequency based on the desired overshoot and the settling time.

$$\zeta = \frac{-\ln(O.S.)}{\sqrt{\pi^2 + \ln^2(O.S.)}} \rightarrow \zeta = \frac{-\ln(0.1)}{\sqrt{\pi^2 + \ln^2(0.1)}} \rightarrow \boxed{\zeta = 0.5912} \quad \text{Desired Damping Ratio}$$

$$t_s \approx \frac{4}{\zeta \omega_n} \rightarrow 1 = \frac{4}{0.6 \omega_n} \rightarrow \boxed{\omega_n = 6.7659} \quad \text{Desired Natural Frequency}$$

The desired closed-loop poles location $\rightarrow s = -\zeta \omega_n \pm j \omega_n \sqrt{1 - \zeta^2} \rightarrow \boxed{s_d = -4 \pm j5.5}$
Desired Closed-loop Poles



Step 2: Plot the axes of s-plane and mark open-loop poles/zeros and the desired closed-loop poles.

poles $\rightarrow p_1 = -1, p_2 = -4$

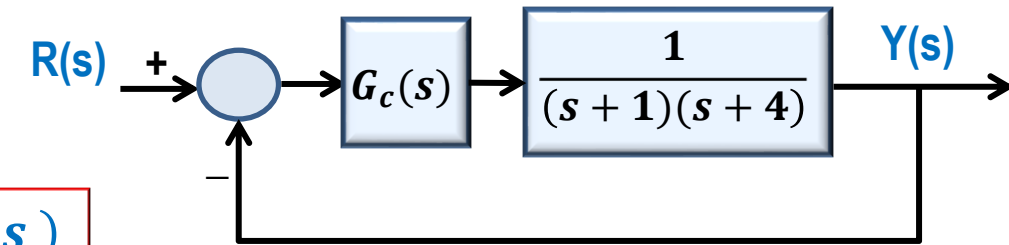
zeros \rightarrow No finite zeros

PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.

$$G_c(s) = K_p(1 + T_d s)$$



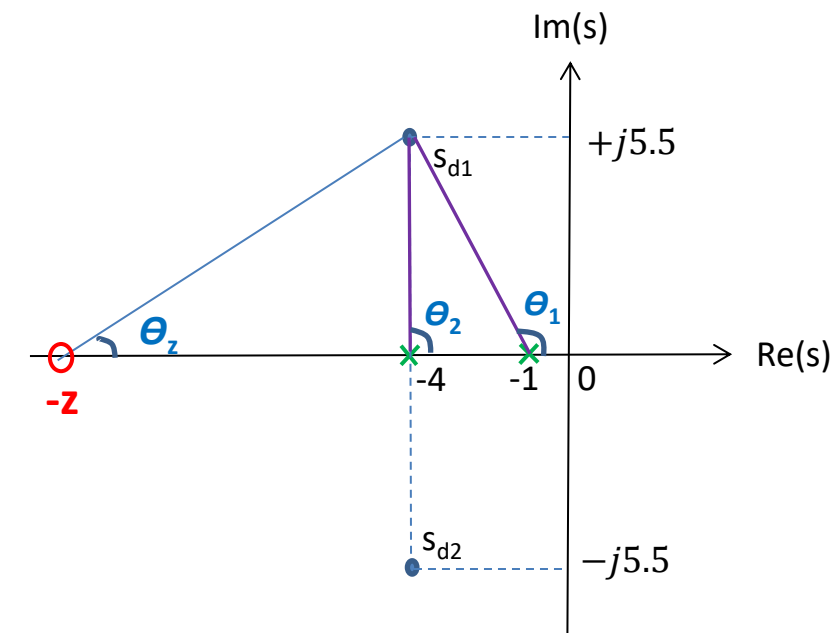
Step 3: Find **sum of the angles** at the desired closed-loop poles location

- Apply the **angle condition** to check that if the desired poles are on the root-locus or not

$$\begin{aligned} \angle \left(\frac{K}{(s+1)(s+4)} \right) \bigg|_{s=s_{d1}} &= \angle K - \angle(s+1) - \angle(s+4) \bigg|_{s=-4+j5.5} \\ &= 0 - \angle\theta_1 - \angle\theta_2 = 0 - 120^\circ - 90^\circ = -210^\circ \end{aligned}$$

The angle condition is not satisfied

Desired poles are not on the root-locus



Step 4: Determine the required **angle deficiency** to satisfy the root-locus angle condition.

The **angle deficiency** is calculated as

$$-210^\circ + \phi = -180^\circ \rightarrow \phi = 210^\circ - 180^\circ = 30^\circ$$

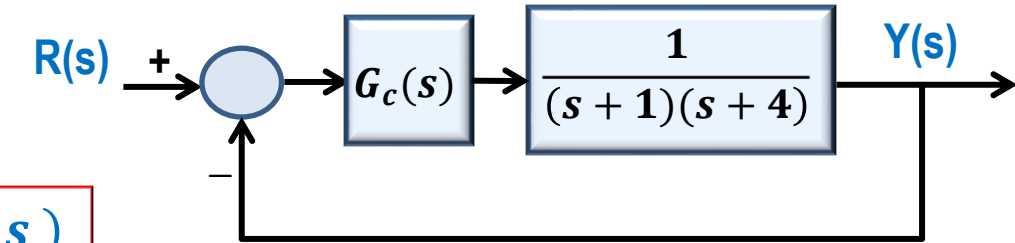
The **PD controller** must contribute the angle of $\phi = 30^\circ$ at the desired pole locations.

PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.

$$G_c(s) = K_p(1 + T_d s)$$



Step 5: Determine **zero** location of the **PD controller** to compensate the angle deficiency

- P is the desired pole location
- Draw line PA
- Draw line PB such that

$$\angle APB = \phi = 30^\circ$$
- The **zero** is located at the intersection of PB with real axis

$$z = 13.8$$

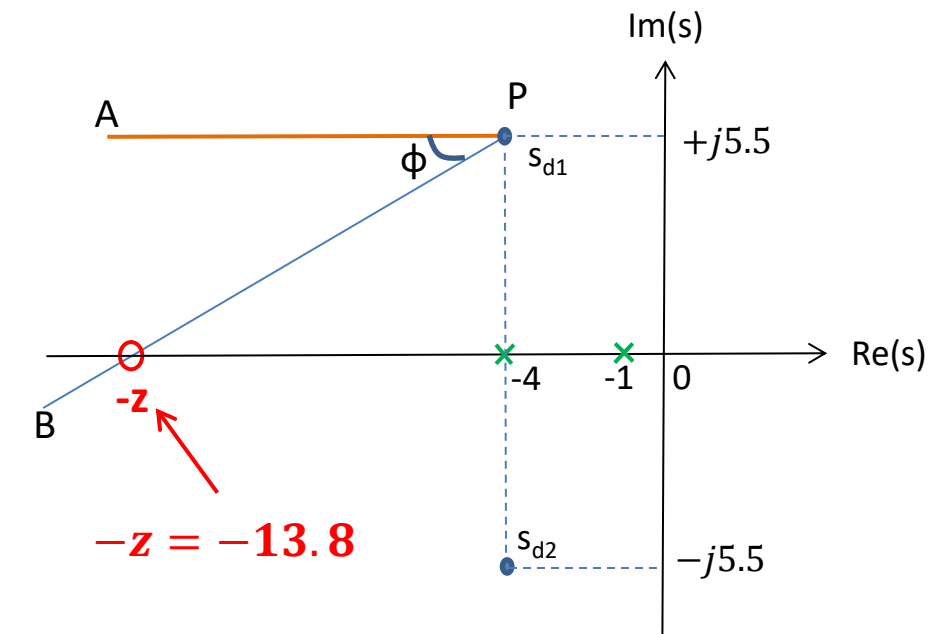
- Determine the **derivative time-constant** T_d

$$T_d = \frac{1}{z}$$

$$\rightarrow T_d = \frac{1}{13.8} = 0.074 \rightarrow$$

$$T_d = 0.074$$

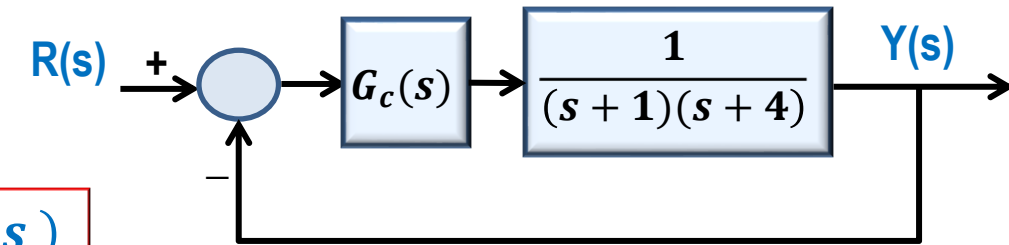
$$G_c(s) = K_p(1 + 0.074s)$$



PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.



$$G_c(s) = K_p(1 + T_d s)$$

Step 6: Determine proportional gain K_p from the magnitude condition.

Next, calculate the K_p using the magnitude condition

$$|G_c(s)G(s)|_{s=s_d} = 1$$

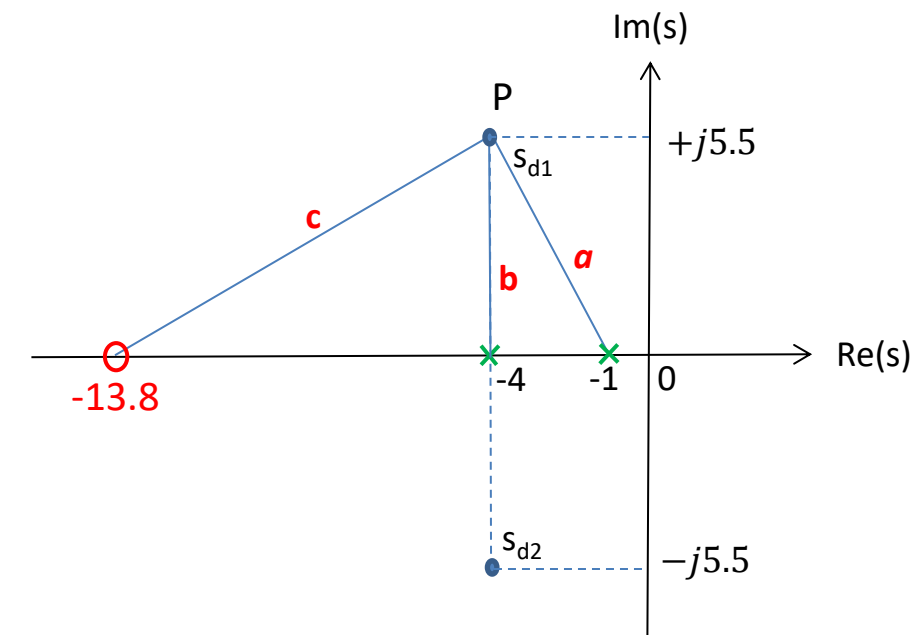
$$\left| K_p(1 + 0.074s) \cdot \frac{1}{(s+1)(s+4)} \right|_{s=-4+j5.5} = 1$$

$$|K_p| = \frac{|-3 + j5.5||j5.5|}{|0.704 + j0.407|} = \frac{6.26 \times 5.5}{0.81} = 42.51$$

$$K_p = 42.51$$

We can also find the gain K_p by measuring the length of the vectors connected from the desired pole to the other poles and zeros.

$$K_c = \frac{a \times b}{c} \rightarrow K_p = K_c z$$



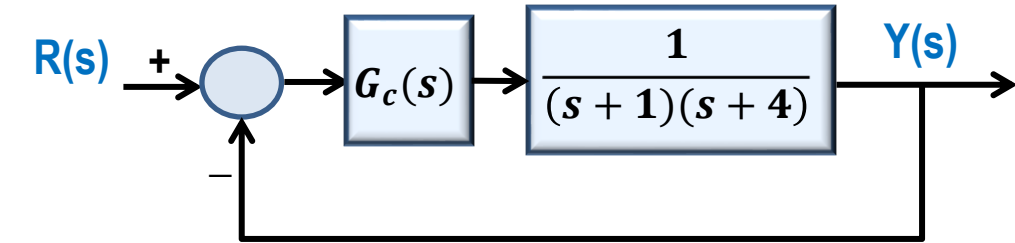
$$G_c(s) = 42.51(1 + 0.074s)$$

The designed PD controller

PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.



$$G_c(s) = 42.51(1 + 0.074s)$$

Step 7: Analyze and verify the designed compensator

Determine closed-loop transfer function of the compensated system and check the pole locations

$$T(s) = \frac{Y(s)}{R(s)} = \frac{G_c(s)G(s)}{1 + G_c(s)G(s)} \longrightarrow T(s) = \frac{42.51(1 + 0.074s)}{s^2 + 8.146s + 46.51}$$

The closed-loop poles are located at

$$s_{1,2} = -4.073 \pm j5.47,$$

Dominant Poles

Desired Closed-loop Poles

$$s = -4 \pm j5.5$$

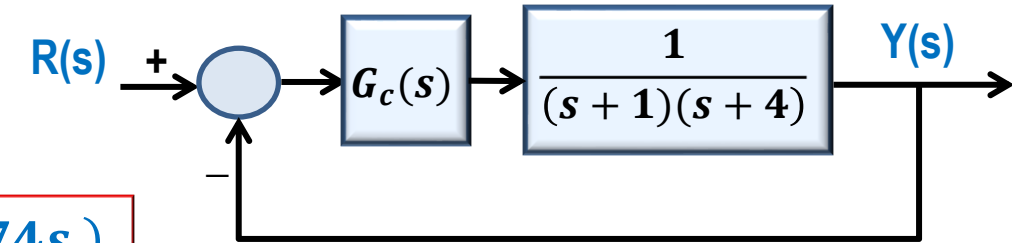
- The dominant closed-loop poles are almost located at the desired places in the s-plane.

PD Controller Design via Root Locus

Example 4

Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.

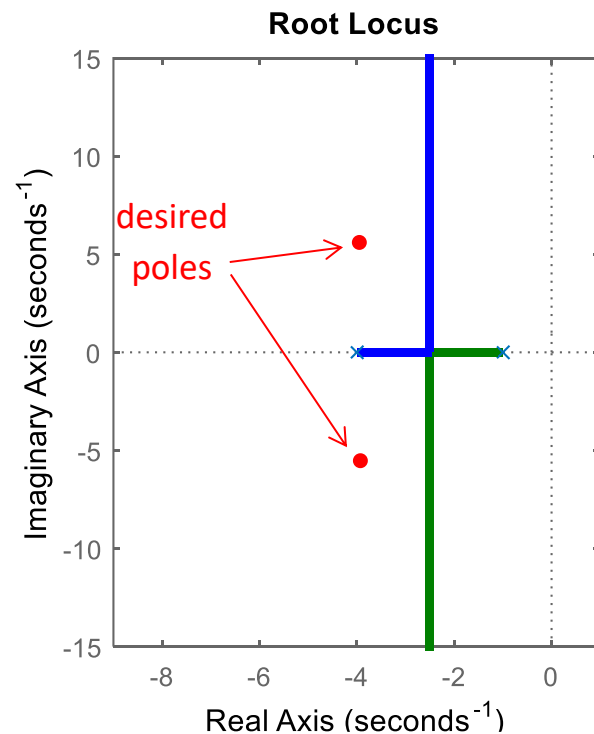
$$G_c(s) = 42.51(1 + 0.074s)$$



Step 7: Analyze and verify the designed compensator

Uncompensated System

$$G(s) = \frac{1}{(s+1)(s+4)}$$



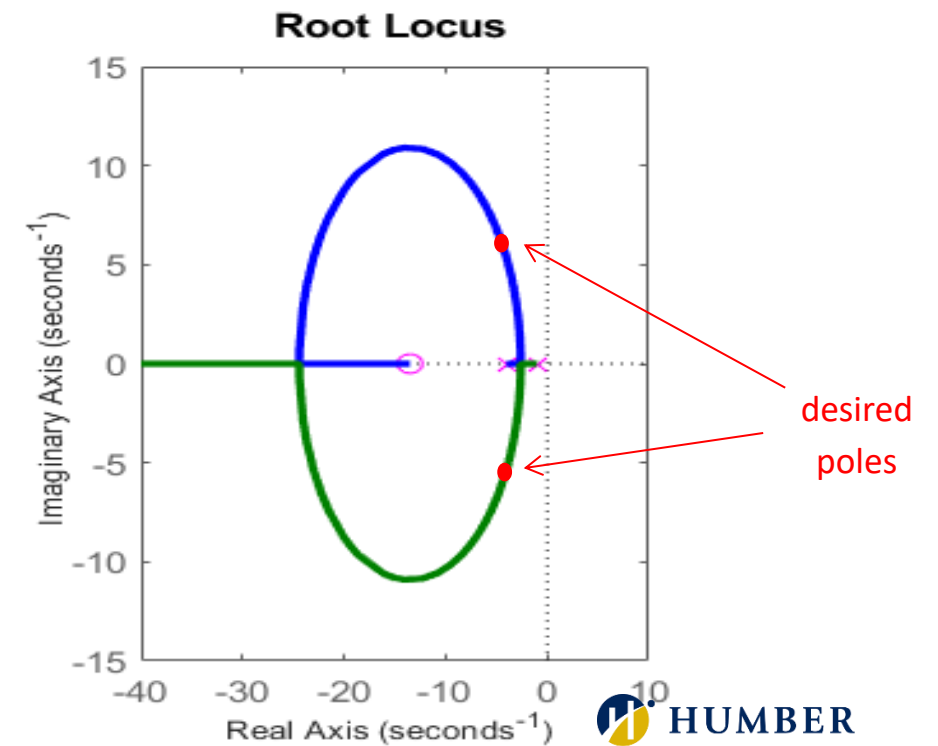
Desired Closed-loop Poles

$$s = -4 \pm j5.5$$

PD controller pulls the root-locus to the left and improves the relative stability of the system.

Compensated System

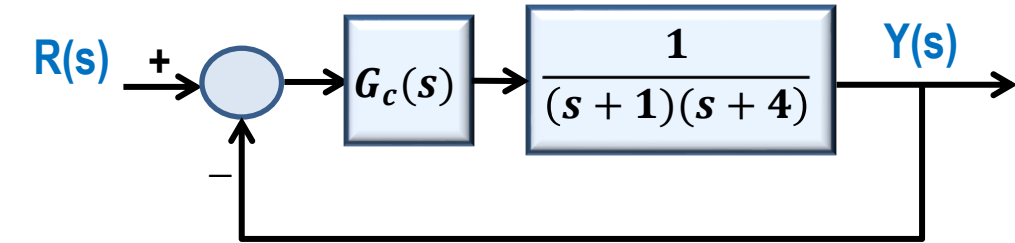
$$G_c(s)G(s) = \frac{42.51(1 + 0.074s)}{(s+1)(s+4)}$$



PD Controller Design via Root Locus

Example 4

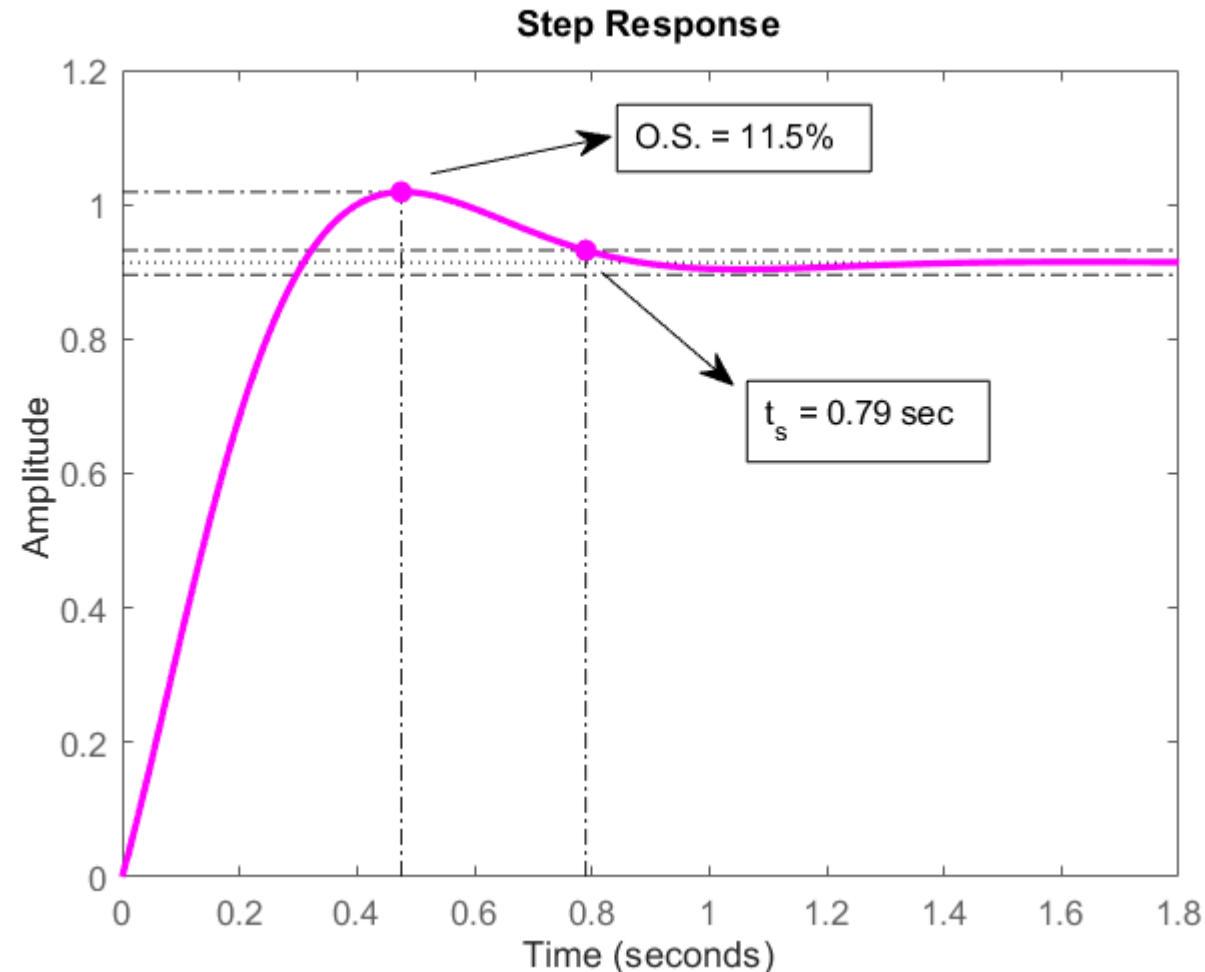
Consider the following closed-loop system with PD controller.
Design PD controller to achieve a percentage of overshoot about 10% and settling time less than **1sec**.



$$G_c(s) = 42.36(1 + 0.074s)$$

Step 7: Analyze and verify the designed compensator

- Step responses of the compensated closed-loop system has the overshoot of about 11.5% and settling time of 0.79sec.

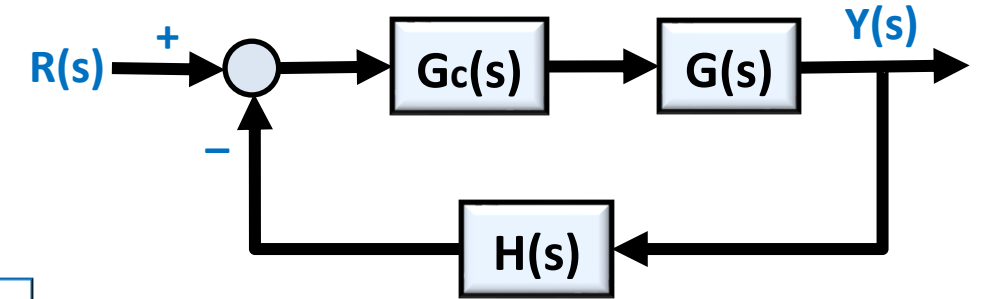


Dynamic Compensator Design

□ Lead Compensator & Lag Compensator

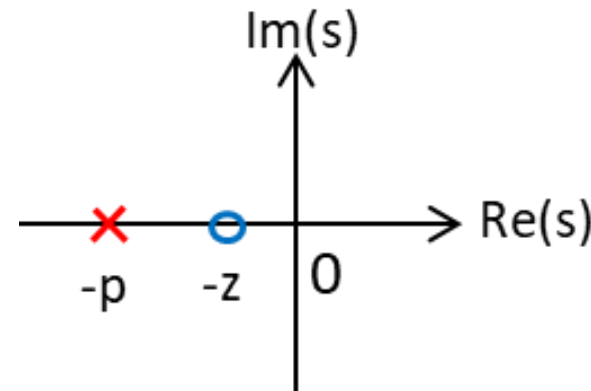
- In general, a **compensator** has the following transfer function

$$G_c(s) = K_c \frac{s + z}{s + p}, \quad z > 0, \quad p > 0$$

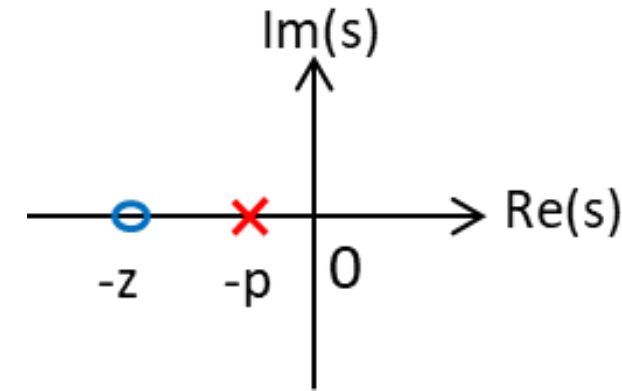


- According to the **pole/zero locations** we have the following structures:

- To improve transient response and stability.
- Similar to PD controller.
- Positive angle contribution.



Lead Compensator



Lag Compensator

- To improve steady-state error.
- Similar to PI controller.
- Negative angle contribution.

- Combined **lead-lag compensator** can improve both **transient response** and **steady-state response**, similar to a **PID controller**.

Lag Compensator Design via Root Locus

Assume that the uncompensated system meets the desired transient response specifications by simple gain adjustment.

Step 1: Determine **desired location** of the dominant **closed-loop poles**. If requires determine the **open-loop gain** at the location of the closed-loop poles from **magnitude condition**.

Step 2: Calculate the **desired error-constant** (k_p , k_v or k_a), based on the desired steady-state error.

$$k_p = \lim_{s \rightarrow 0} G(s)H(s)$$

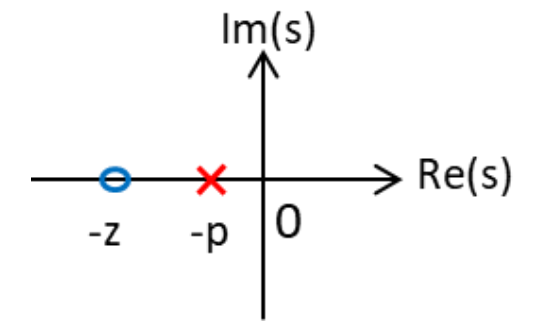
$$k_v = \lim_{s \rightarrow 0} sG(s)H(s)$$

$$k_a = \lim_{s \rightarrow 0} s^2 G(s)H(s)$$

Step 3: Design a **lag compensator** to increase the error constant (k_p , k_v or k_a) to the desired value without significantly altering the original root-locus and the dominant pole locations.

$$G_c(s) = K_c \frac{s + z}{s + p}$$

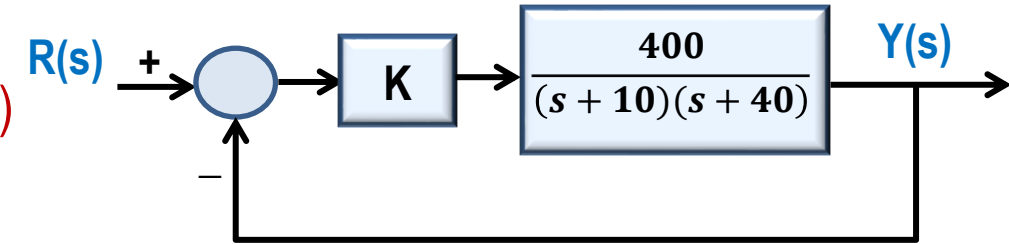
Step 4: Verify your design by comparing the dominant closed-loop pole locations with the desired poles by calculation or plotting the root-locus. If needs, adjust **open-loop gain** of the compensated system from the root-locus magnitude condition.



Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.
It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$)
without altering the transient response with $K = 2$.



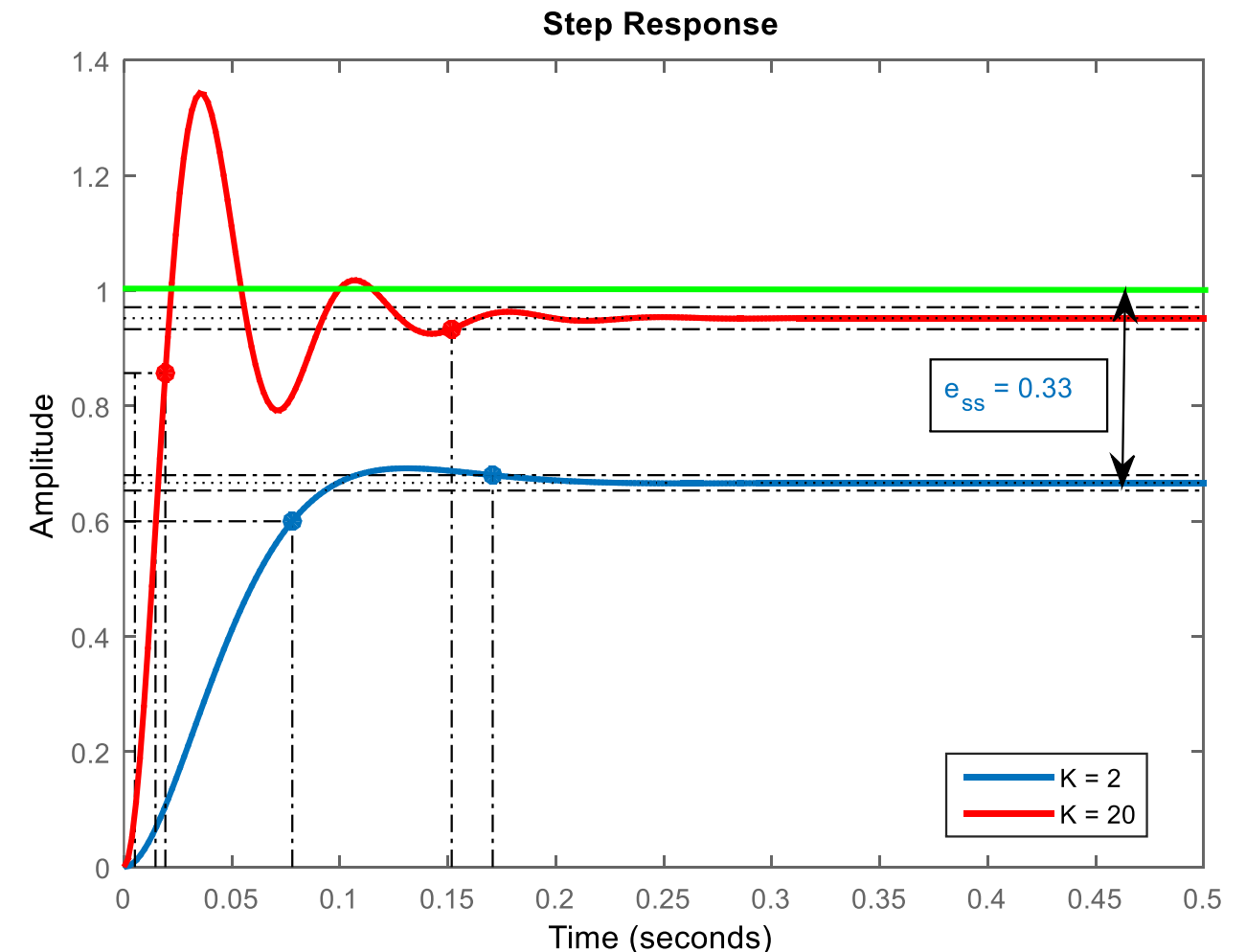
Pre-design Performance Study

- The graph shows that $G(s)$ is a **fast system**.

$K = 2$

$O.S. = 3.8\%$
 $t_s = 0.17 \text{ sec}$
 $t_r = 0.063 \text{ sec}$
 $e_{ss} = 0.33 = 33\%$

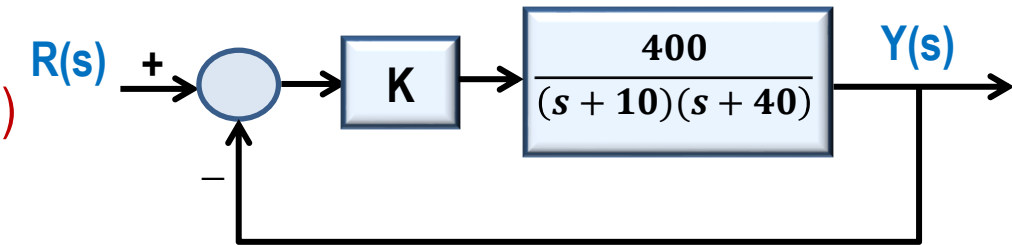
- The transient response specifications are good, but the steady-state error of 33% is not acceptable.
- The **goal** is to **decrease the e_{ss} to 3%** without altering the transient response specifications.



Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.
It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$)
without altering the transient response with $K = 2$.

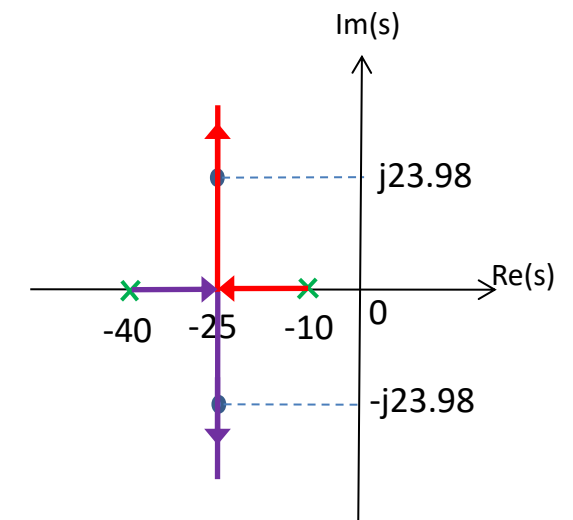


Step 1: Determine **desired dominant closed-loop pole locations** and the corresponding **open-loop gain K**

First, determine location of the **dominant poles** for closed-loop system with $K = 2$.

$$T(s) = \frac{Y(s)}{R(s)} = \frac{KG(s)}{1 + KG(s)} \rightarrow T(s) = \frac{800}{s^2 + 50s + 1200} \rightarrow \boxed{s = -25 \pm j23.98}$$

Desired Closed-loop Poles



$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

Find the corresponding **open-loop gain** from **magnitude condition**:

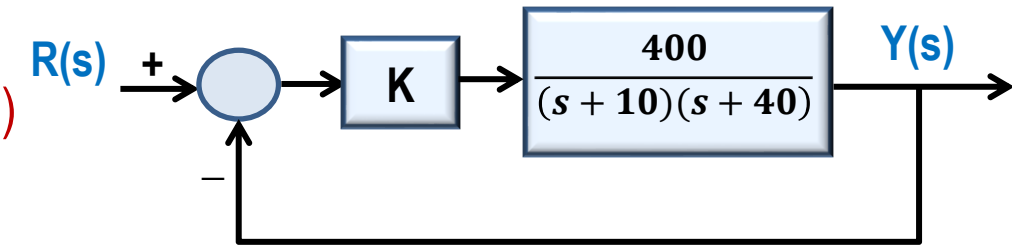
$$|KG(s)H(s)| = 1 \rightarrow |K| = \frac{|s + 10||s + 40|}{400} \Big|_{s=s_d} = \frac{|-15 + j23.98||15 + j23.98|}{400}$$

$$K = \frac{\sqrt{(15)^2 + (23.98)^2} \times \sqrt{(15)^2 + (23.98)^2}}{400} \rightarrow \boxed{K = 2} \quad \text{Open-loop gain}$$

Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.
It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$)
without altering the transient response with $K = 2$.



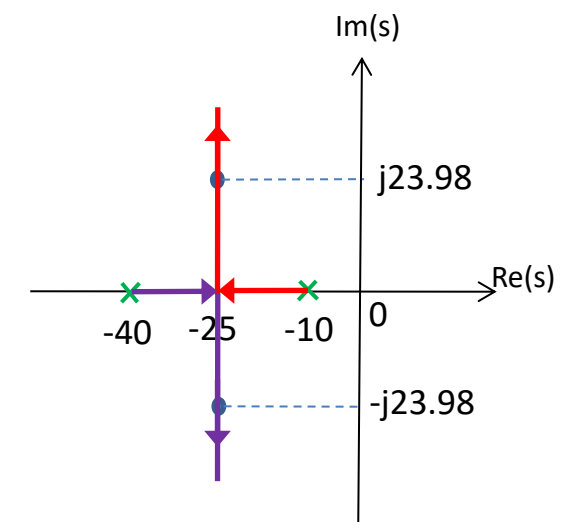
Step 2: Calculate the **desired error-constant**, from the **given e_{ss}** .

Calculate the **desired step-error constant**

$$e_{ss} = 0.03 \rightarrow e_{ss} = \frac{1}{1 + k_p} = 0.03 \rightarrow k_p = 32.3$$

Desired Steady-state Error

Desired Step-error Constant



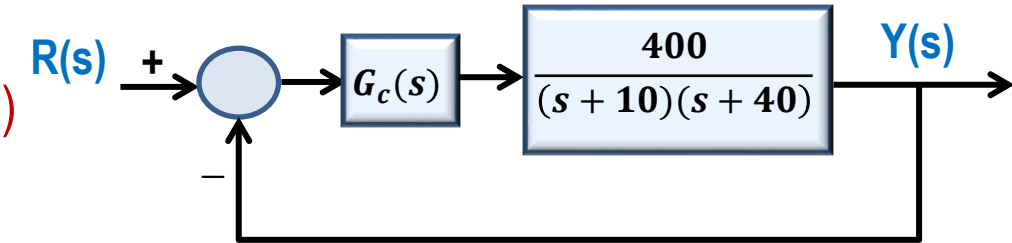
$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.

It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$) without altering the transient response with $K = 2$.



Step 3: Design a **lag compensator** to achieve the desired error value without altering the dominant poles.

To not change the location of the dominant closed-loop poles, the compensator's gain must be selected equal to $K = 2$

$$K_c = K = 2$$

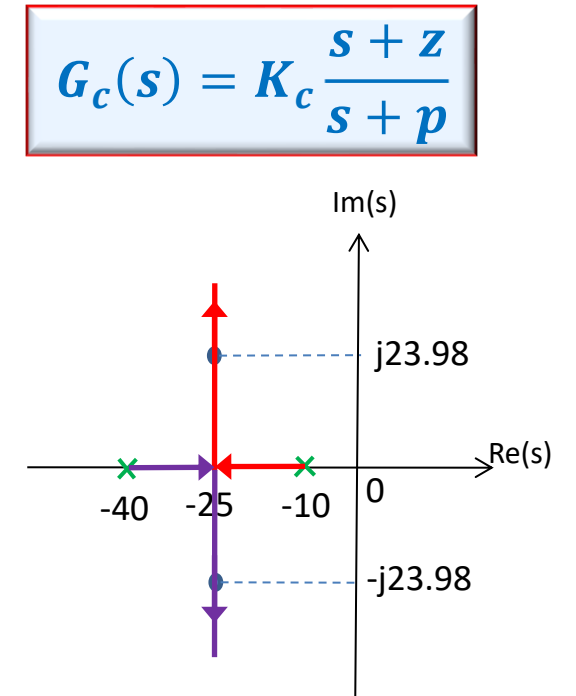
The step-error constant for the compensated system is,

$$k_p = \lim_{s \rightarrow 0} G_c(s)G(s) = \lim_{s \rightarrow 0} K_c \frac{s + z}{s + p} \cdot \frac{400}{(s + 10)(s + 40)} = K_c \frac{z}{p}$$

$$k_p = 32.3$$

$$K_c = 2$$

$$\Rightarrow 32.3 = 2 \times \frac{z}{p} \Rightarrow z \approx 16p$$



$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

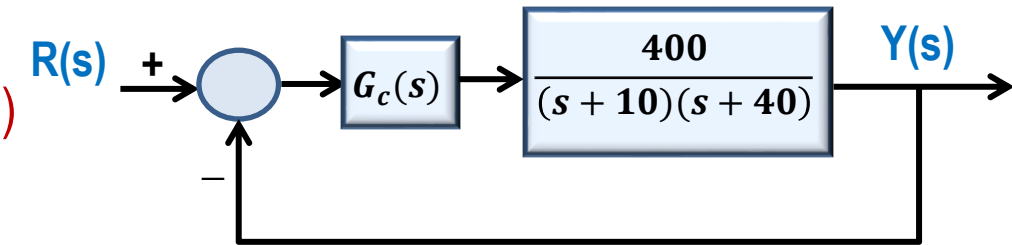
- Pole/zero of lag compensator must be selected far enough from the dominant closed-loop poles and close to the origin. However, settling time increases by selecting them too close to the origin.

Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.

It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$) without altering the transient response with $K = 2$.



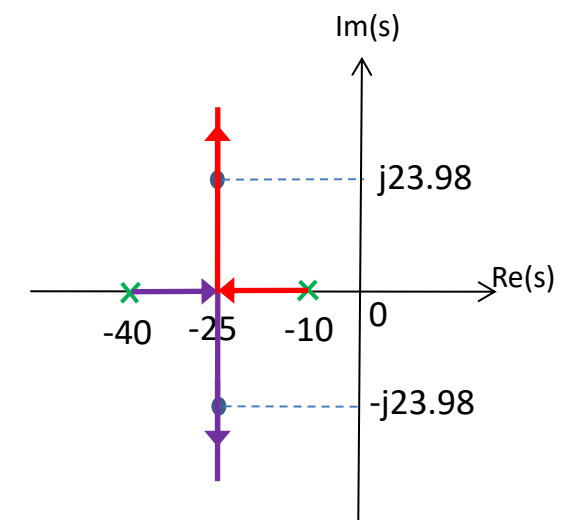
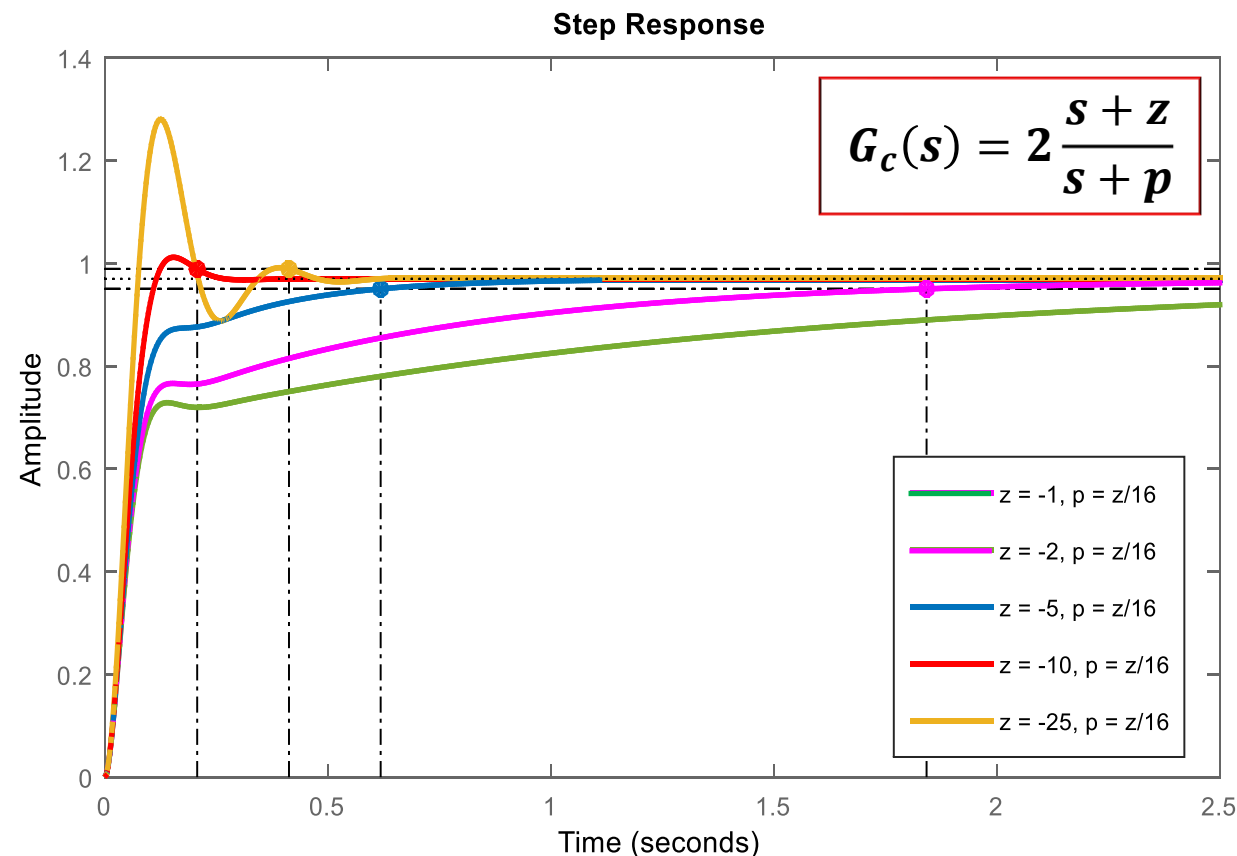
Step 3: Design a lag compensator to achieve the desired error value without altering the dominant poles.

- We can compare the effect of selecting different pole/zero locations for the lag compensator and fine tune the compensator.

$$G_c(s) = K_c \frac{s + z}{s + p}$$

$$\text{If } z = 10 \rightarrow p = \frac{10}{16} = 0.625$$

$$G_c(s) = 2 \frac{s + 10}{s + 0.625}$$

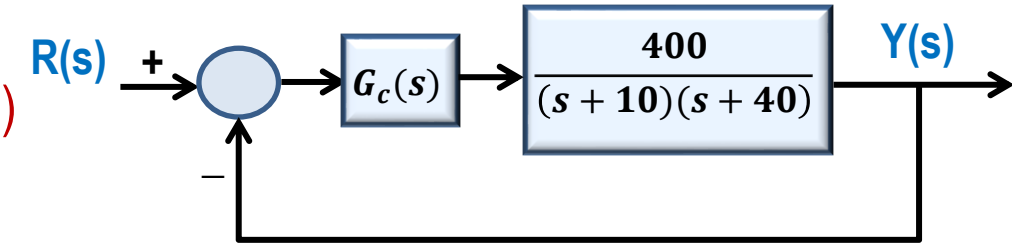


$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

Lag Compensator Design via Root Locus

Example 5

Consider the following second-order system with gain $K = 2$.
It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$) without altering the transient response with $K = 2$.



$$G_c(s) = 2 \frac{s + 10}{s + 0.625}$$

Step 4: Analyze and verify the designed compensator.

Determine closed-loop transfer function of the compensated system and check the pole locations

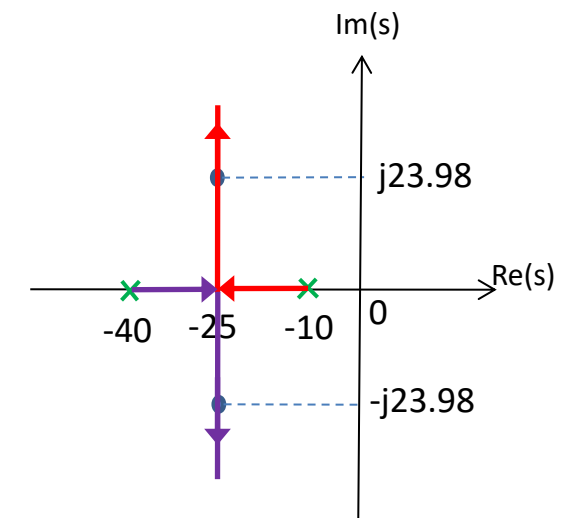
$$T(s) = \frac{Y(s)}{R(s)} = \frac{G_c(s)G(s)}{1 + G_c(s)G(s)} \longrightarrow T(s) = \frac{800(s + 10)}{s^3 + 50.62s^2 + 1231s + 8250}$$

The closed-loop poles are located at

$$s_{1,2} = -20.3125 \pm j20.3077, \quad s_3 = -10$$

Dominant Poles

Close to zero location



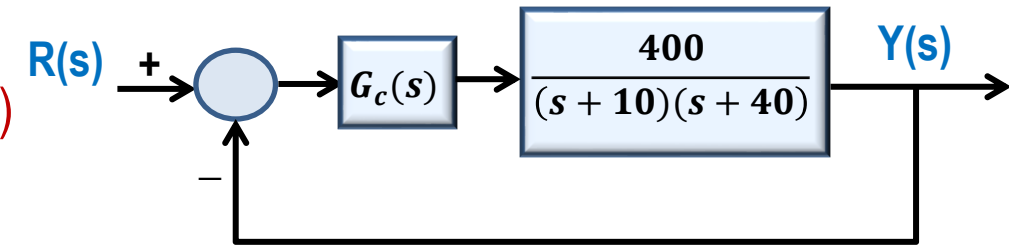
$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

- The dominant closed-loop poles are located at the desired places in the s-plane.
- The third pole at $s_3 = -10$ will be cancelled out with the zero at $s = -10$.

Lag Compensator Design via Root Locus

Example 5

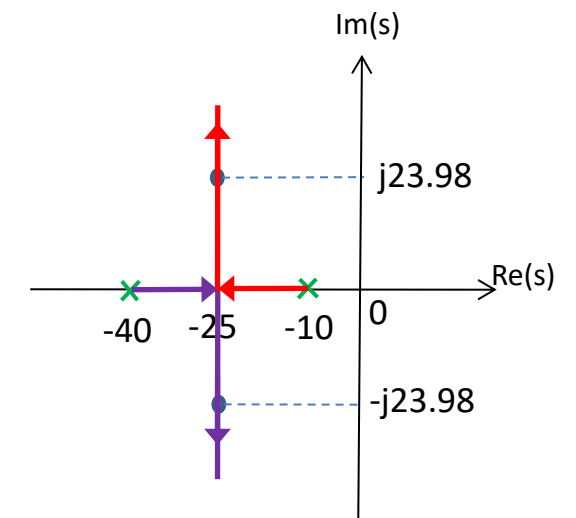
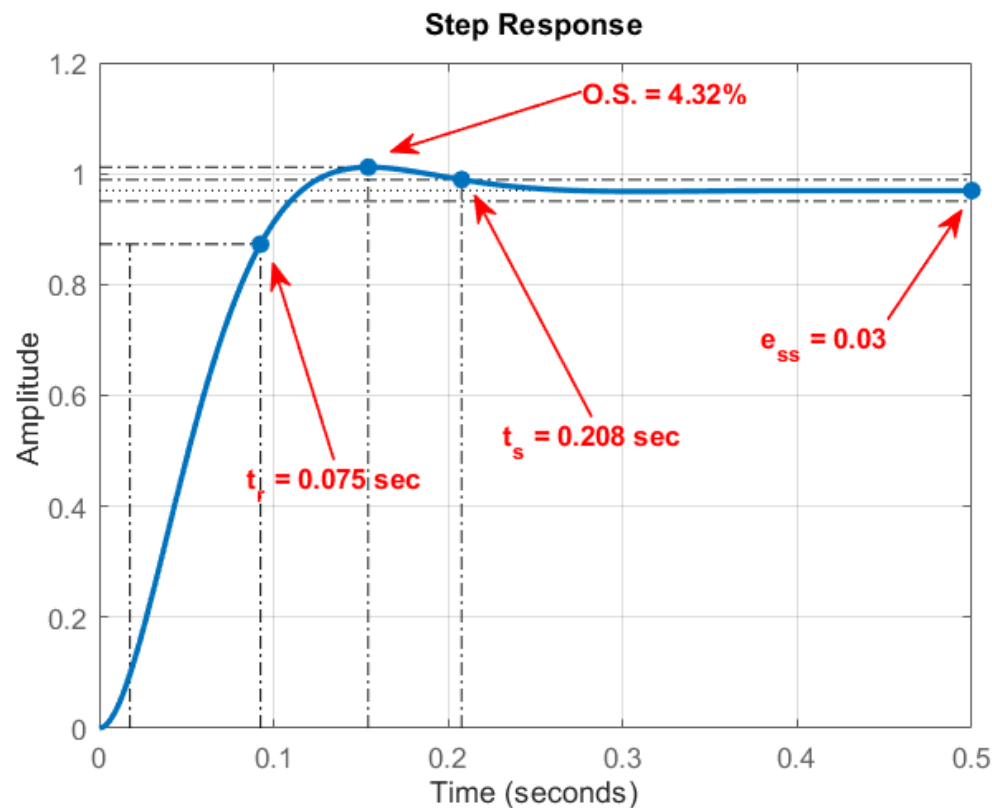
Consider the following second-order system with gain $K = 2$.
It is desired to decrease the steady-state error (to achieve $e_{ss} = 0.03$)
without altering the transient response with $K = 2$.



$$G_c(s) = 2 \frac{s + 10}{s + 0.625}$$

Step 4: Analyze and verify the designed compensator.

- Transient responses are close to the desired values.
- Steady-state error decreases to the desired value of $e_{ss} = 0.03$.



$$\begin{aligned} O.S. &= 3.8\% \\ t_s &= 0.17 \text{ sec} \\ t_r &= 0.063 \text{ sec} \end{aligned}$$

PI Controller Design via Root Locus

- In lag compensator design, the pole and zero have to be located far enough from the dominant poles and close to the origin.

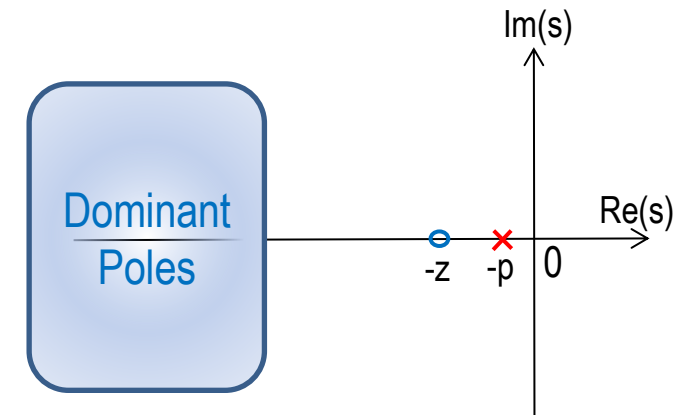
$$G_c(s) = K_c \frac{s + z}{s + p}$$

- The lag compensator can also be designed by placing the pole exactly at the origin

$$p = 0$$



$$G_c(s) = K_c \frac{s + z}{s}$$



- In this case the steady-state error of the compensated closed-loop system for unit-step input will be zero:

$$k_p = \lim_{s \rightarrow 0} G_c(s)G(s) = \lim_{s \rightarrow 0} K_c \frac{s + z}{s} G(s) = \infty \quad \rightarrow \quad e_{ss} = \frac{1}{1 + k_p} = 0$$

- The compensator's zero must be selected far enough from the dominant poles, and close to the origin.
- The gain, K_c is selected to achieve the desired performance of transient response.
- The lag compensator with a pole at the origin can also be shown as a **PI controller**:

$$G_c(s) = K_c \frac{s + z}{s} = K_c \left(1 + \frac{z}{s} \right) \quad \rightarrow \quad G_c(s) = K_p \left(1 + \frac{1}{T_i s} \right)$$

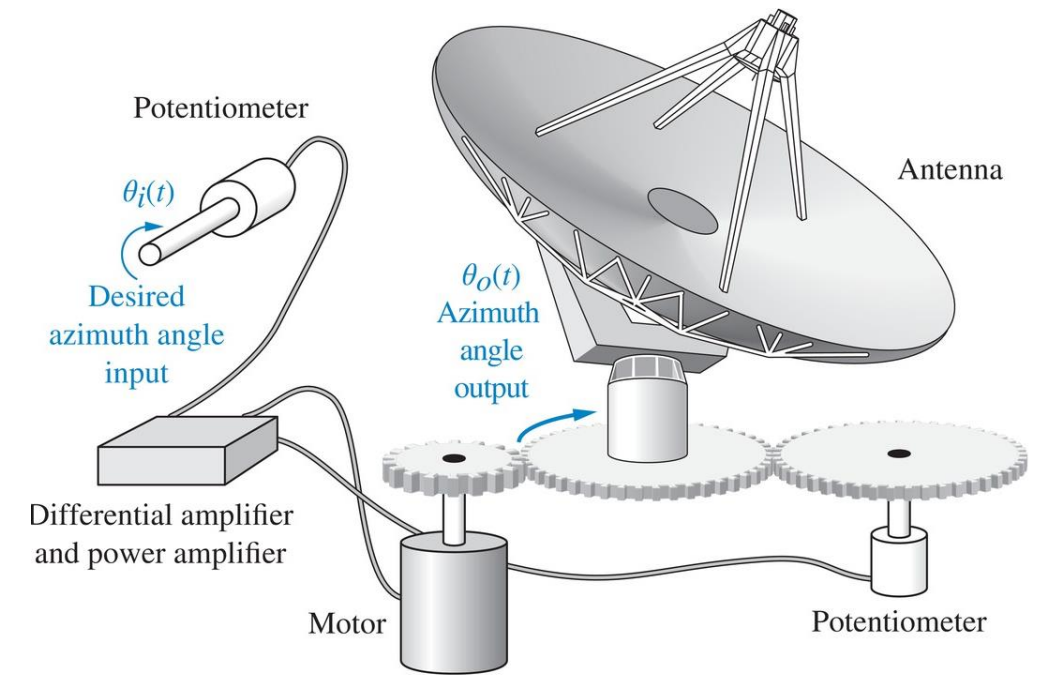
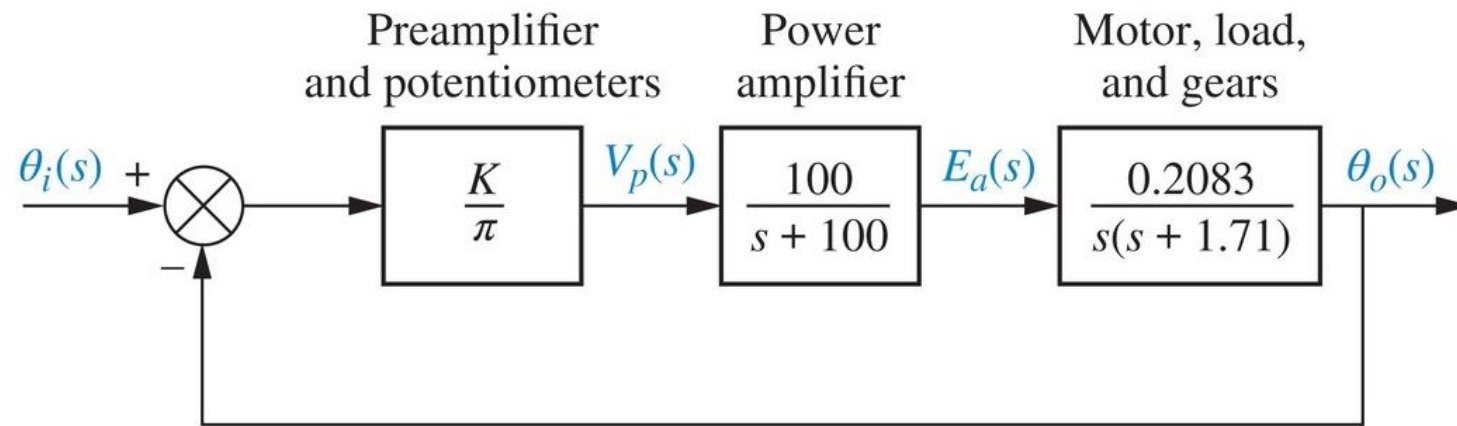
where, the proportional gain K_p and the integral time-constant T_i are defined as

$$T_i = \frac{1}{z}$$

$$K_p = K_c$$

Case Study: Antenna Control System

- Consider the *motor-driven antenna azimuth position control system* example from Lecture 1.
- We determined the block diagram of the control system as below:



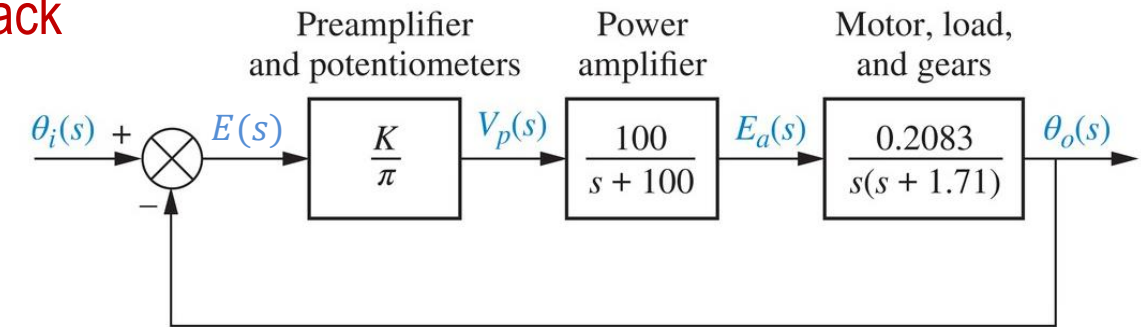
- In this part, we are interested in determining the value of required *gain K* and design a *lead-lag compensator* to meet time response requirements, such as percent overshoot, settling time, peak time and the steady-state error.
- The following case study emphasizes this design procedure, using the *root locus*.

Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

a) Find the preamplifier gain K required for 25% overshoot via root-locus.

Find the overall open-loop transfer function.



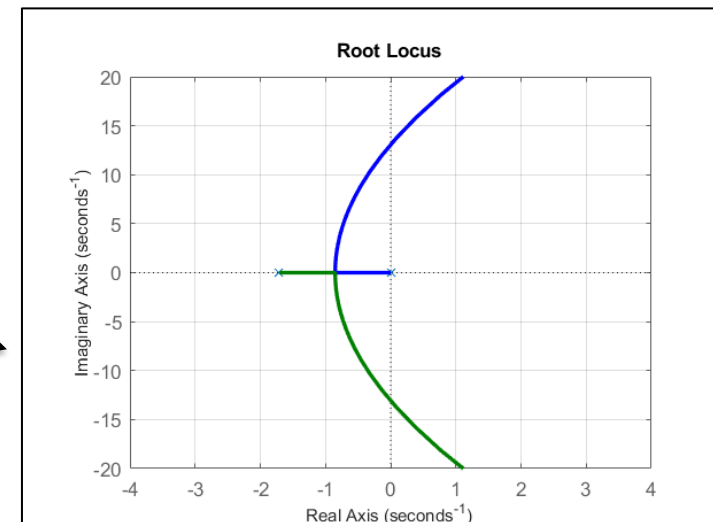
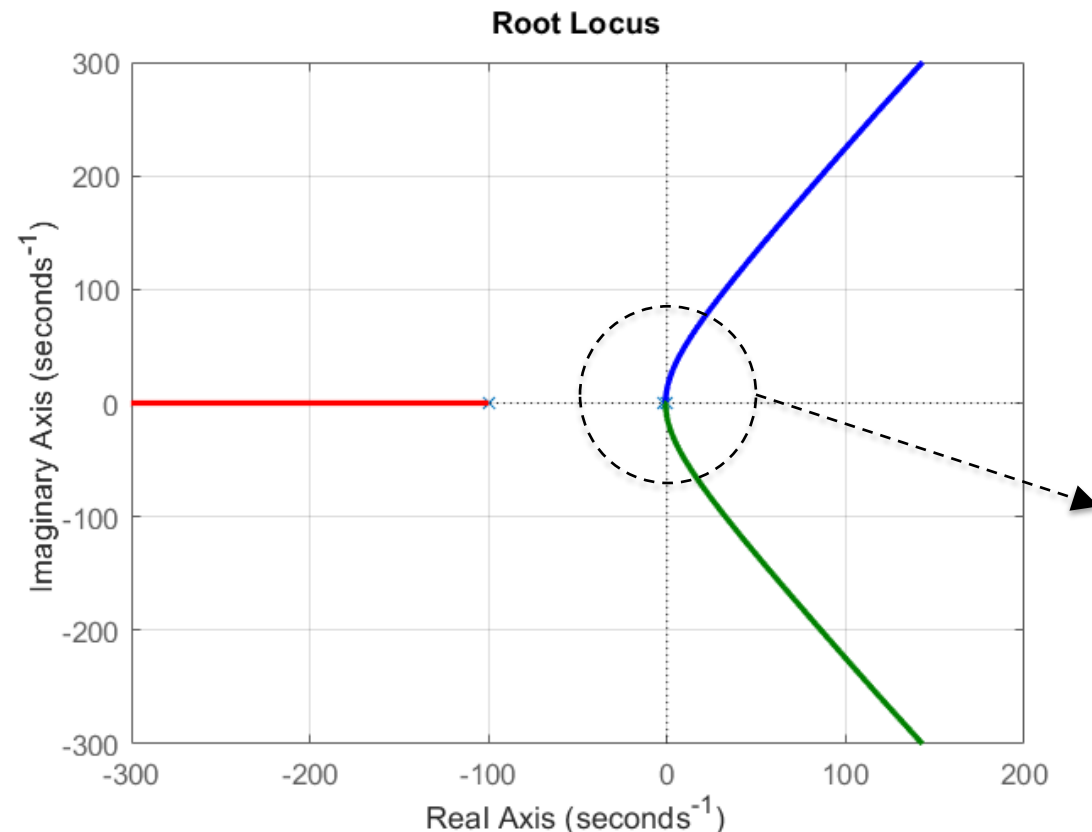
$$G(s) = \frac{6.63K}{s(s + 100)(s + 1.71)}$$

Plot the root-locus of the system.

Poles: $s_1 = 0$, $s_2 = -1.71$, $s_3 = -100$

Zeros: No finite zero. Three zeros at infinity.

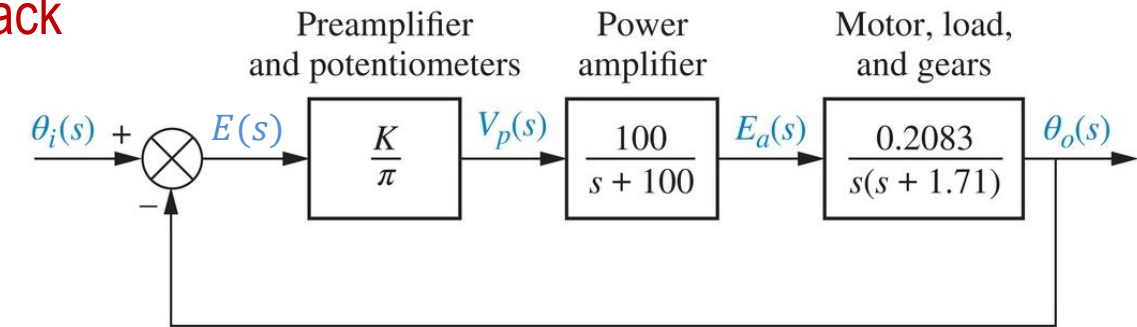
Since the pole at -100 is too far from the other two poles, the poles at 0 and -1.71 are the dominant poles.



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

a) Find the preamplifier gain K required for 25% overshoot via root-locus.



Find the **damping ratio** correspond to 25% overshoot.

$$\zeta = \frac{-\ln(O.S.)}{\sqrt{\pi^2 + \ln^2(O.S.)}} = \frac{-\ln(0.25)}{\sqrt{\pi^2 + \ln^2(0.25)}} \rightarrow \zeta = 0.404$$

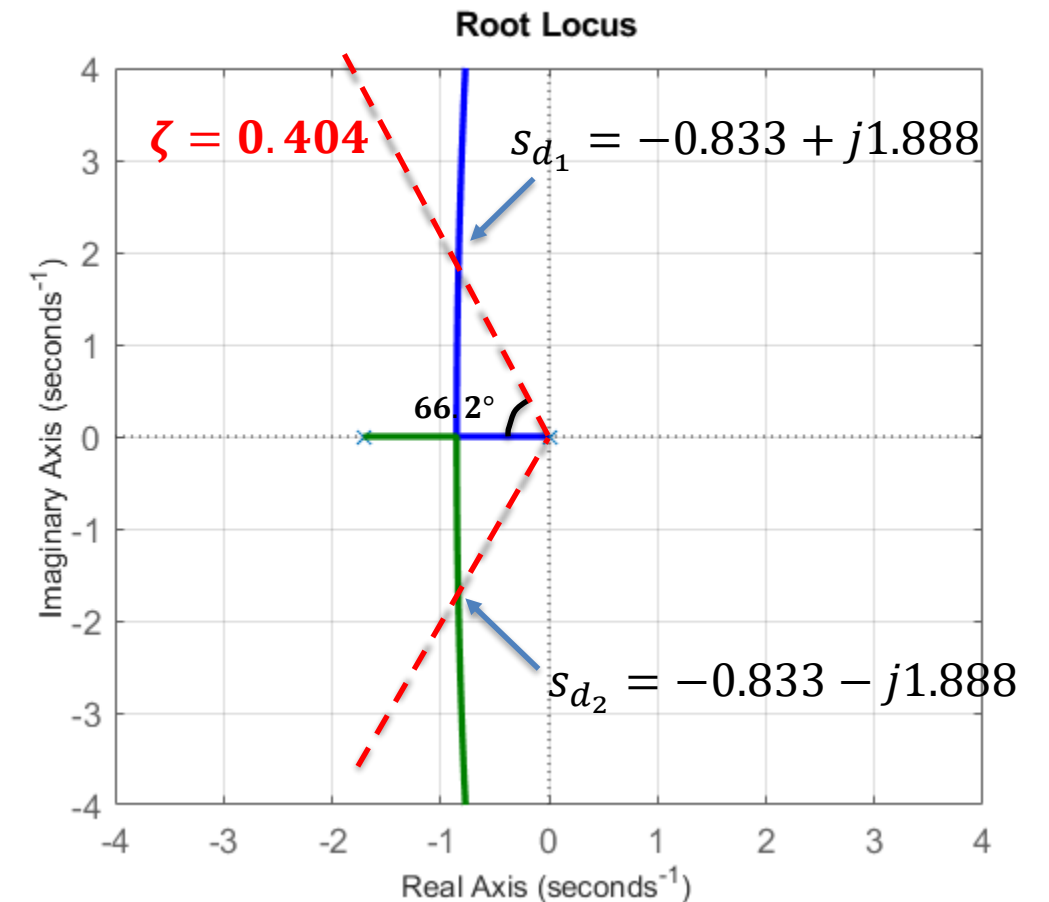
Draw a **radial line** from the origin at an angle of $\theta = \cos^{-1} \zeta$

$$\theta = \cos^{-1} 0.404 = 66.2^\circ$$

The intersection of this line with the root locus locates the systems **dominant closed-loop poles** to have a 25% overshoot.

From the graph the **dominant poles** are at:

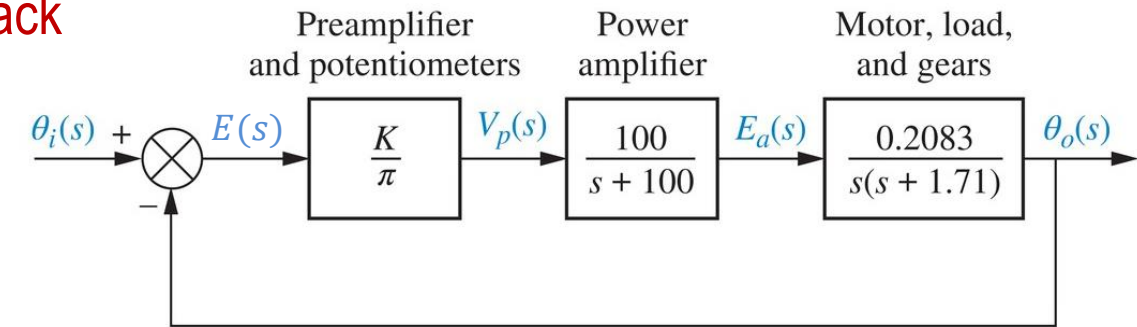
$$s_{d_{1,2}} = -0.833 \pm j1.888$$



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

a) Find the preamplifier gain K required for 25% overshoot via root-locus.



The corresponding gain K is obtained from the root-locus **gain condition**:

$$|KG(s)H(s)| = 1 \rightarrow \left| \frac{6.63K}{s(s+100)(s+1.71)} \right| = 1$$

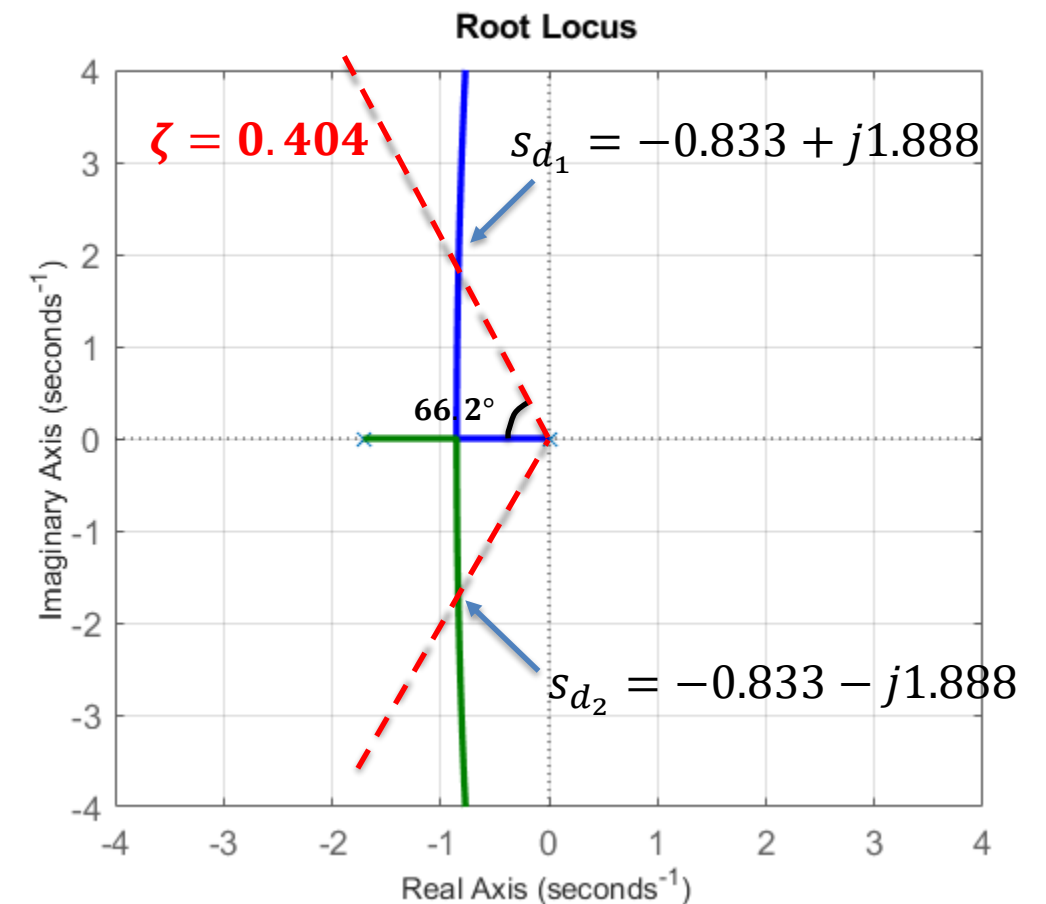
$$|6.63K| = |s(s+100)(s+1.71)|_{s=s_{d1}}$$

$$|6.63K| = |s_d||s_d+100||s_d+1.71|$$

$$|6.63K| = |-0.833 + j1.888||99.167 + j1.888||0.877 + j1.888|$$

$$|6.63K| = (2.064)(99.185)(2.082) = 426.22$$

$$K = \frac{426.22}{6.63} \rightarrow \boxed{K = 64.29}$$

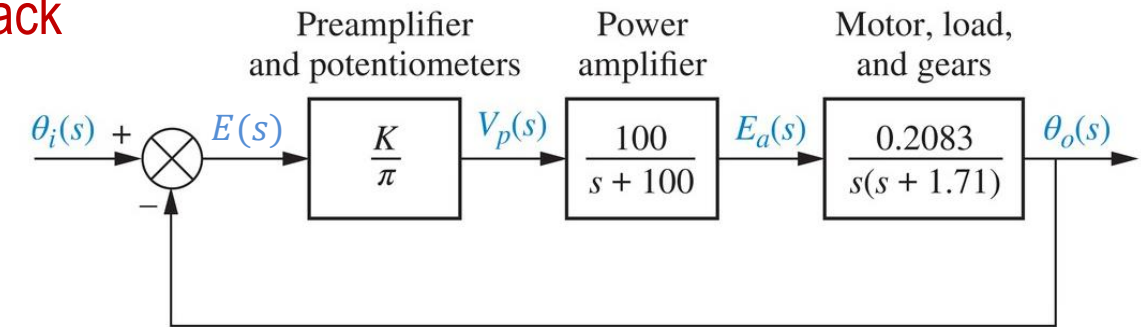


Case Study: Antenna Control System

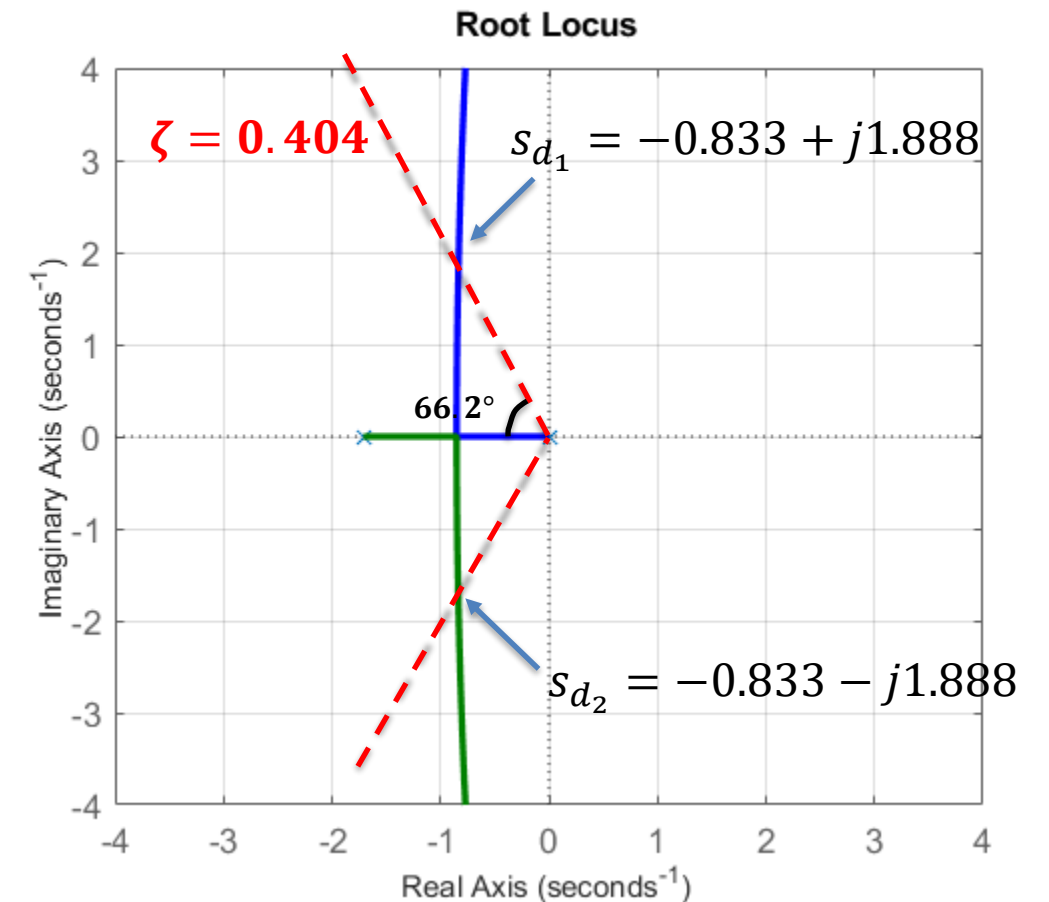
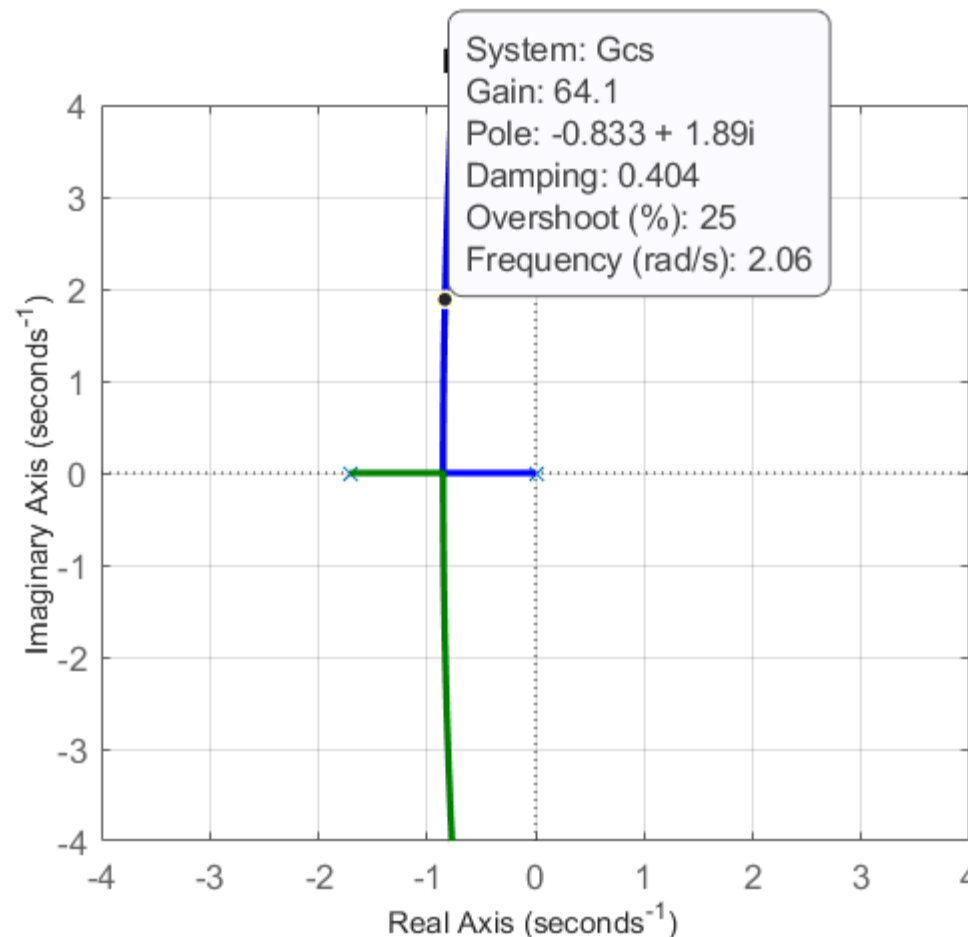
For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

a) Find the preamplifier gain K required for 25% overshoot via root-locus.

We can also determine the required gain K by inspecting the root-locus plot in MATLAB



Gain = 64.1
Pole = $-0.833 + j1.89$
Damping ration = 0.404
Overshoot = 25%
Natural freq. = 2.06



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

b) Design a lead compensator for settling-time of 2 sec and 25% overshoot.

Find the **damping ratio** correspond to 25% overshoot.

$$\zeta = \frac{-\ln(O.S.)}{\sqrt{\pi^2 + \ln^2(O.S.)}} = \frac{-\ln(0.25)}{\sqrt{\pi^2 + \ln^2(0.25)}} \rightarrow \zeta = 0.404$$

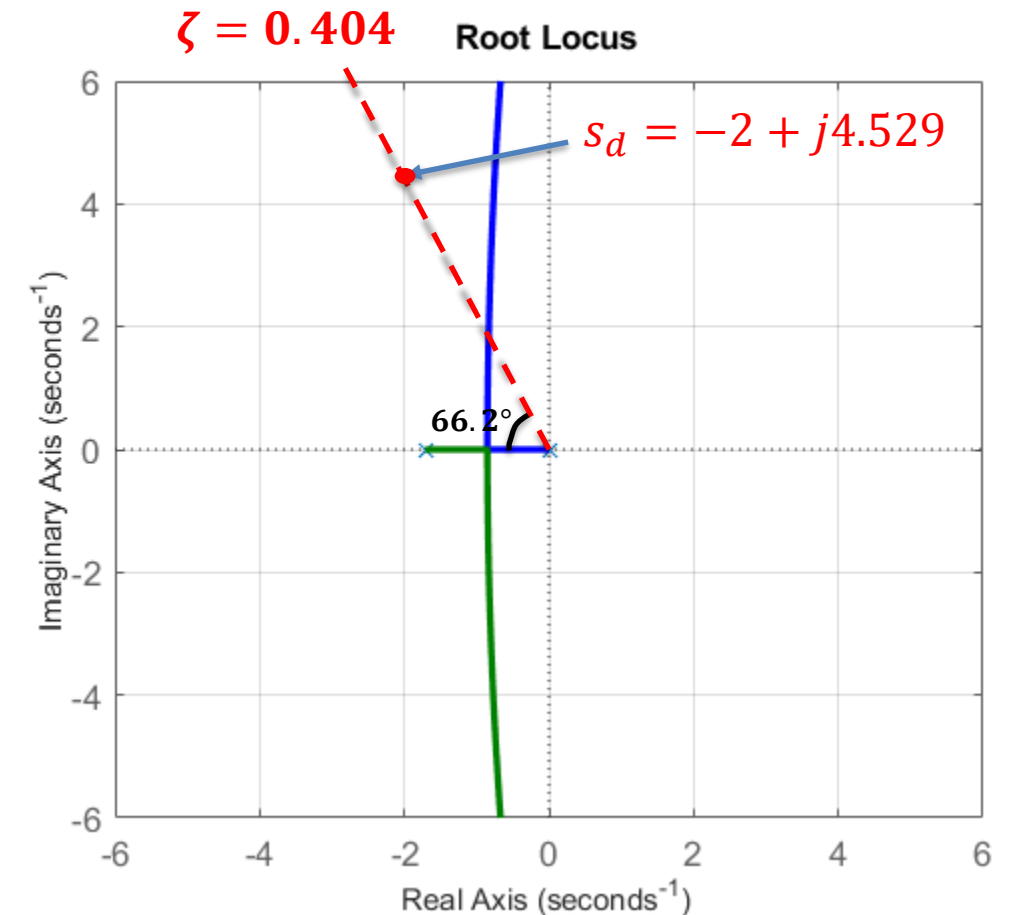
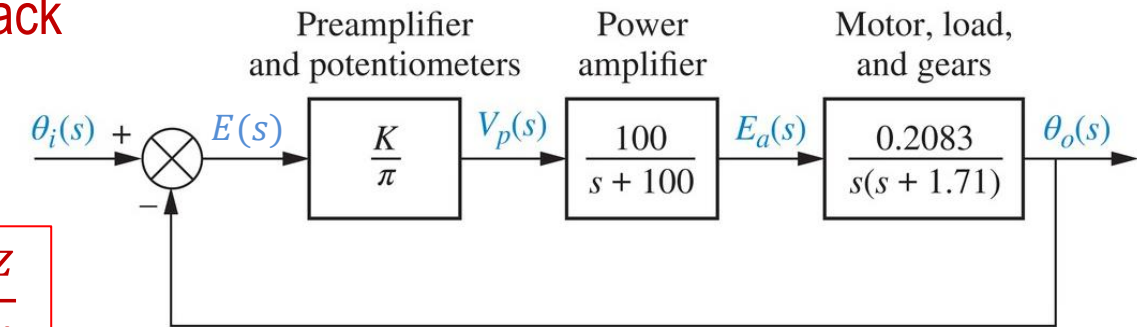
Determine the required **natural frequency** to have settling-time of 2 sec.

$$t_s = \frac{4}{\zeta \omega_n} \rightarrow \omega_n = \frac{4}{t_s \zeta} = \frac{4}{2(0.404)} \rightarrow \omega_n = 4.95 \text{ rad/s}$$

Therefore, the desired **dominant poles** will be at:

$$s_{1,2} = -\zeta \omega_n \pm j \omega_n \sqrt{1 - \zeta^2} \rightarrow s_{1,2} = -2 \pm j4.529$$

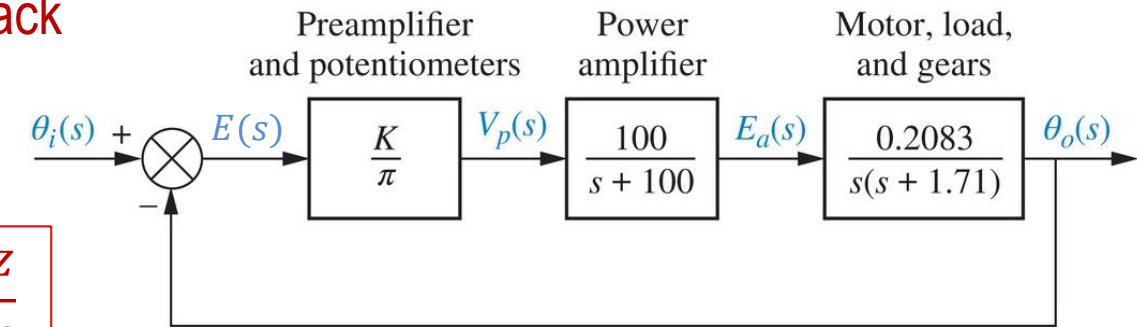
The desired poles are **not** on the root-locus of the system, the desired transient response characteristics are not achievable by a simple gain tuning.



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

b) Design a lead compensator for settling-time of 2 sec and 25% overshoot.

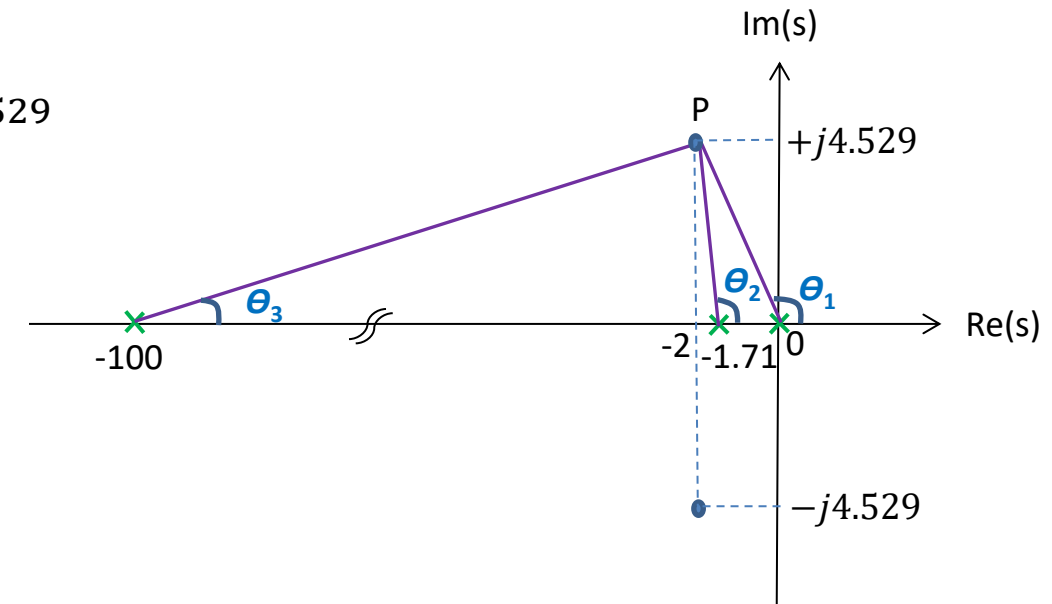


Find **sum of the angles** at the desired closed-loop poles location and determine the **angle deficiency**.

$$G_c(s) = K_c \frac{s + z}{s + p}$$

$$\angle \left(\frac{6.63K}{s(s+1.71)(s+100)} \right) \bigg|_{s=s_{d1}} = \angle 6.63K - \angle(s) - \angle(s+1.71) - \angle(s+100) \bigg|_{s=-2+j4.529}$$

$$= 0 - \angle\theta_1 - \angle\theta_2 - \angle\theta_3 = 0 - 113.83^\circ - 93.66^\circ - 2.64^\circ = -210.13^\circ$$



The **angle deficiency** is calculated as

$$-210.13^\circ + \phi = -180^\circ \rightarrow \phi = 210.13^\circ - 180^\circ = 30.13^\circ$$

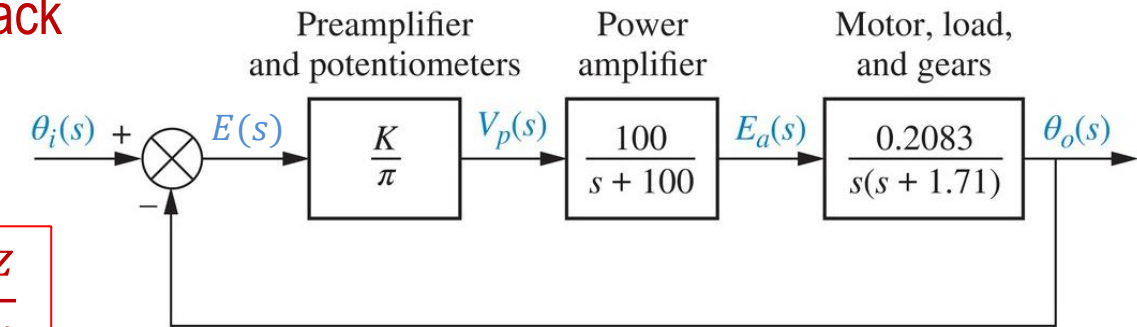
The **lead compensator** must contribute the angle of $\phi = 30.13^\circ$ at the desired pole locations.

Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

b) Design a lead compensator for settling-time of 2 sec and 25% overshoot.

Determine pole/zero locations of the lead compensator to compensate the angle deficiency



$$G_c(s) = K_c \frac{s + z}{s + p}$$

- Draw lines PA and PO

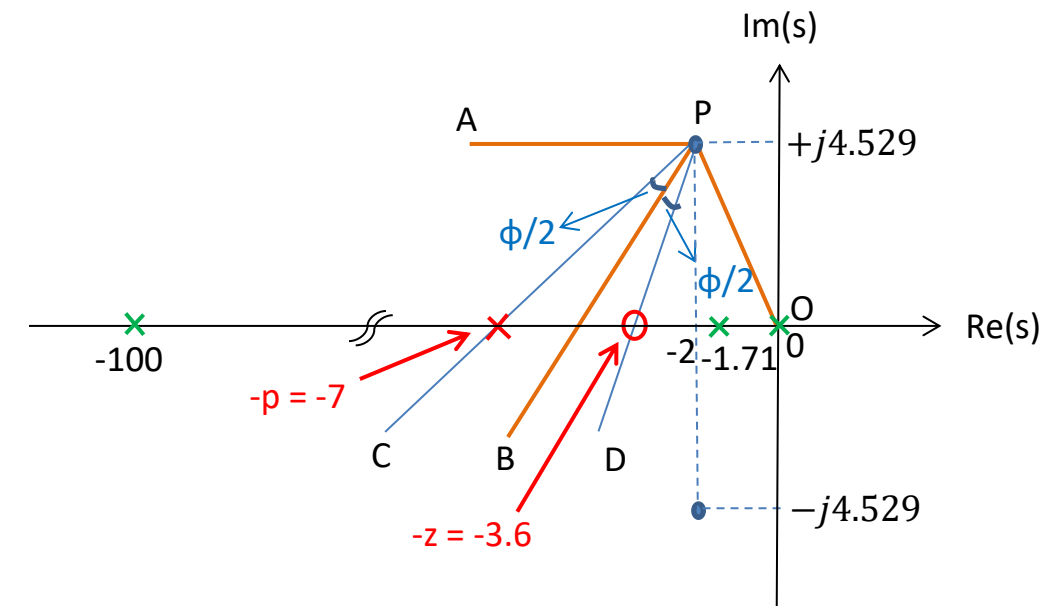
- Draw bisector line PB $\angle APB = \angle BPO = \frac{\angle APO}{2}$

- Draw lines PC and PD so that

$$\angle CPB = \angle BPD = \frac{\phi}{2} = \frac{30.13^\circ}{2} = 15.065^\circ$$

- Pole and zero are the intersections of PC and PD with real axis

$$G_c(s) = K_c \frac{s + 3.6}{s + 7}$$



$$z = 3.6, \quad p = 7$$

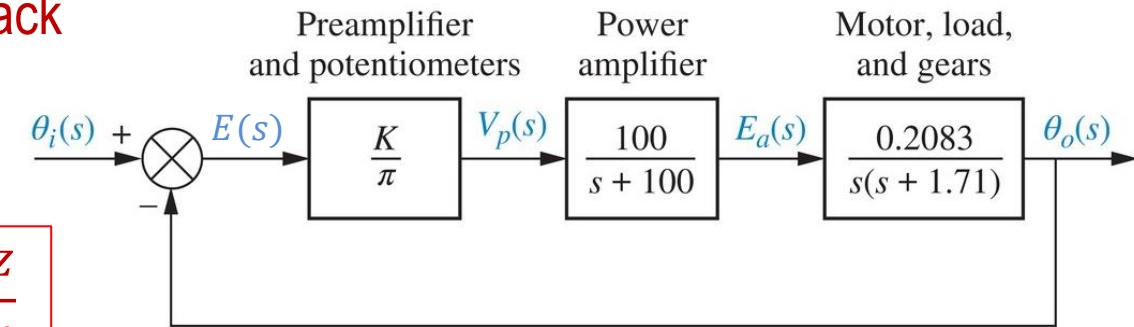
Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

b) Design a lead compensator for settling-time of 2 sec and 25% overshoot.

Next, calculate the gain K_c using the magnitude condition

$$G_c(s) = K_c \frac{s + z}{s + p}$$

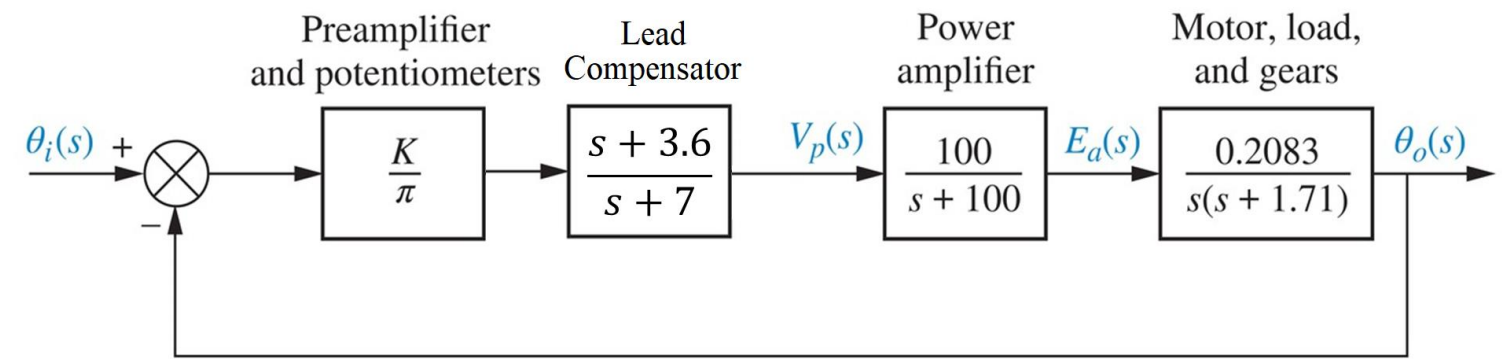


$$\left| K_c \frac{s + 3.6}{s + 7} \cdot \frac{6.63}{s(s + 100)(s + 1.71)} \right|_{s=-2+j4.529} = 1$$

$$|K_c| = \left| \frac{|s||s + 100||s + 1.71||s + 7|}{|6.63||s + 3.6|} \right|_{s=-2+j4.529} = \frac{|-2 + j4.529||98 + j4.529||-0.29 + j4.529||5 + j4.529|}{|6.63||1.6 + j4.529|}$$

$$|K_c| = \frac{(4.95)(98.1)(4.54)(6.75)}{(6.63)(4.80)} = 433.6$$

$$G_c(s) = 433.6 \frac{s + 3.6}{s + 7} \quad \text{Lead compensator}$$

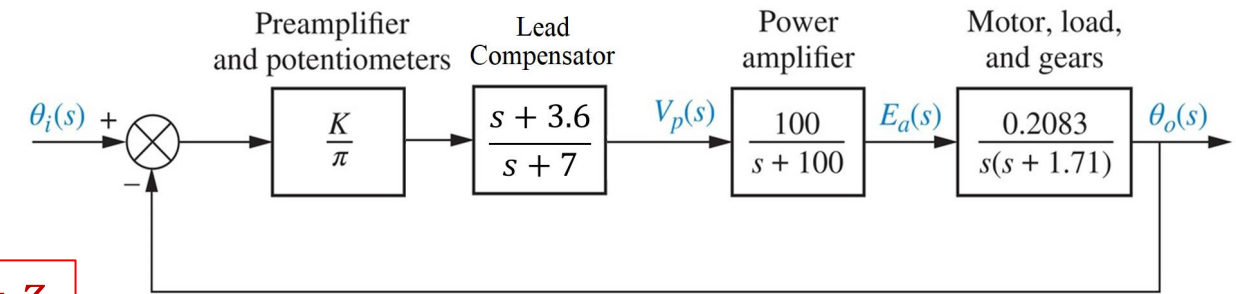


The preamplifier gain should be selected as: $K = 433.6$

Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

c) Design a lag compensator to have a steady-state error of 0.05 for unit-ramp input without changing the transient response characteristics.



$$G_c(s) = \frac{s+z}{s+p}$$

Calculate the desired ramp error-constant

$$e_{ss} = \frac{1}{k_v} \rightarrow 0.05 = \frac{1}{k_v} \rightarrow k_v = 20$$

The ramp error-constant for the lead-lag compensated system is,

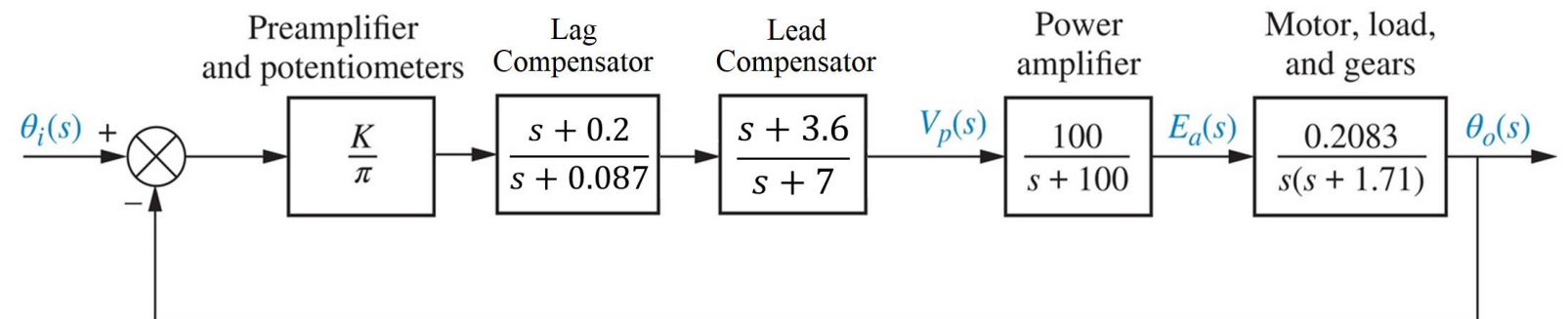
$$k_v = \lim_{s \rightarrow 0} s G_c(s) G(s) = \lim_{s \rightarrow 0} s \cdot \frac{s+z}{s+p} \cdot \frac{6.63(433.6)(s+3.6)}{s(s+100)(s+1.71)(s+7)} \rightarrow 20 = 8.65 \times \frac{z}{p} \rightarrow z \approx 2.31p$$

Since the desired dominant poles are at $s_{1,2} = -2 \pm j4.529$, we can select the pole and zero of the lag compensator as:

$$z = 0.2 \rightarrow p = \frac{0.2}{2.31} = 0.087$$

$$G_c(s) = \frac{s+0.2}{s+0.087}$$

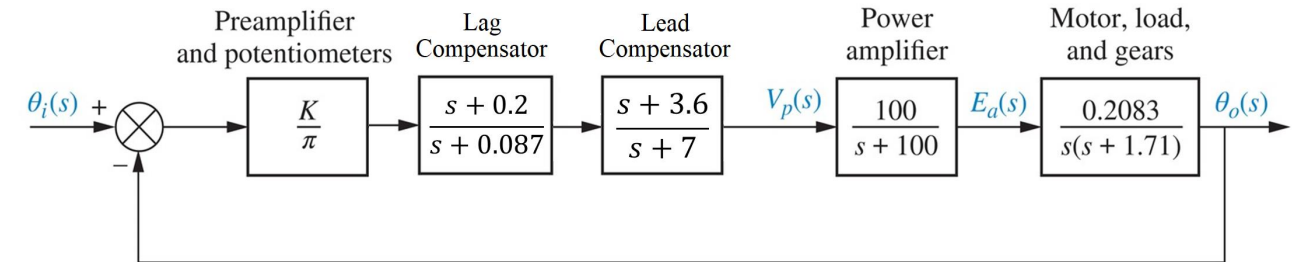
Lag compensator



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

c) Design a lag compensator to have a steady-state error of 0.05 for unit-ramp input without changing the transient response characteristics.



The complete **lead-lag compensated open-loop system** transfer function is

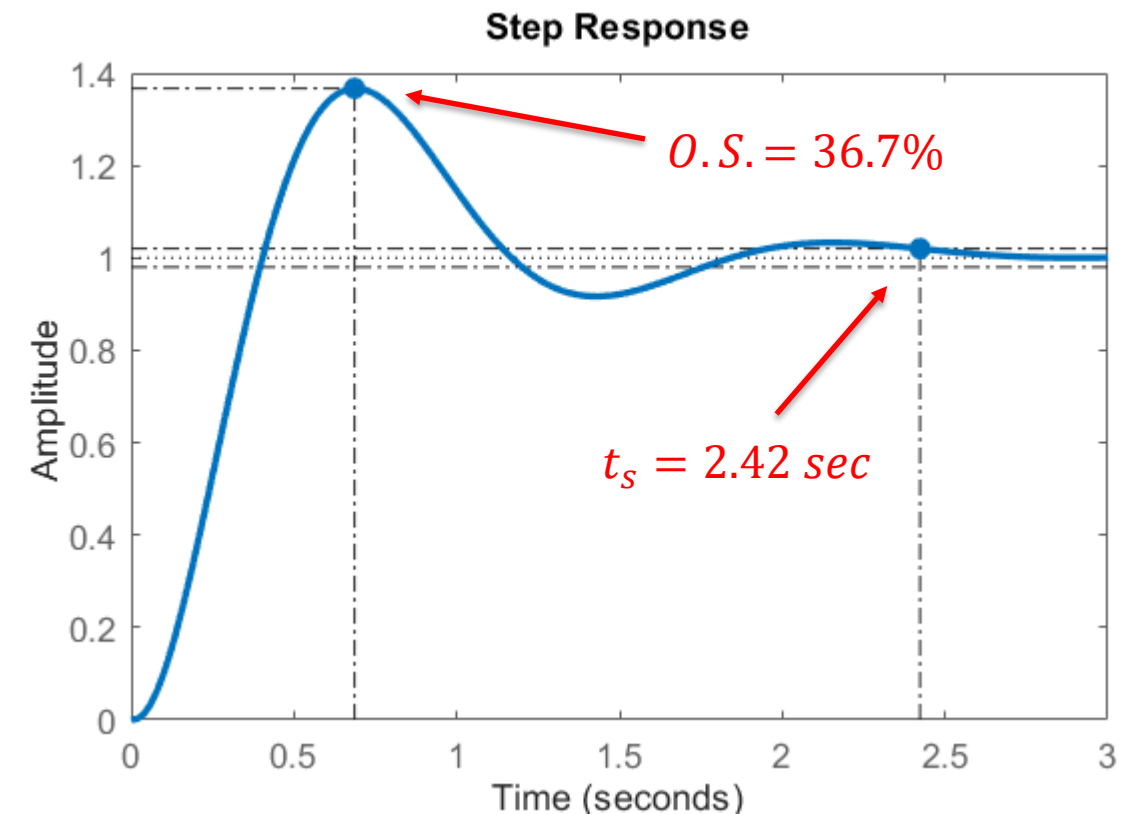
$$G(s) = \frac{6.63K(s + 0.2)(s + 3.6)}{s(s + 100)(s + 1.71)(s + 0.087)(s + 7)}$$

We can plot the **step response of closed-loop system** for $K = 433.6$ and check for the **settling-time** and the **overshoot**.

$$O.S. = 36.7\%$$

$$t_s = 2.42 \text{ sec}$$

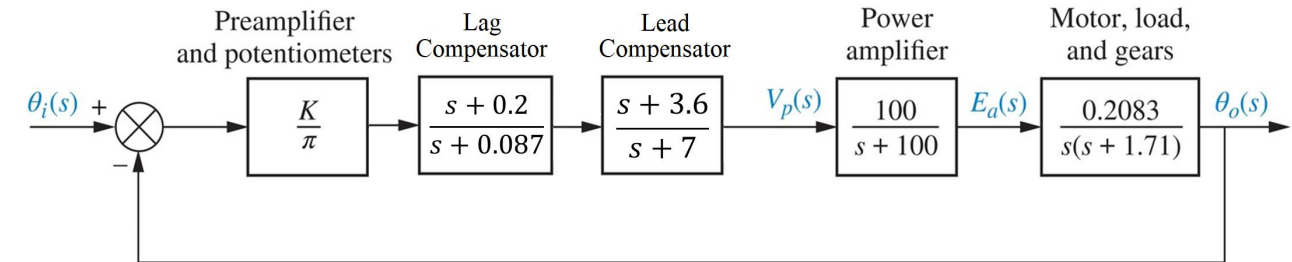
We can plot **root-locus** of the **lead-lag compensated system** and **fine tune the gain K** to achieve the desired transient response of $t_s = 2 \text{ sec}$, $O.S. = 25\%$.



Case Study: Antenna Control System

For the antenna azimuth position control system with the given simplified unity-feedback block diagram model:

c) Design a lag compensator to have a steady-state error of 0.05 for unit-ramp input without changing the transient response characteristics.



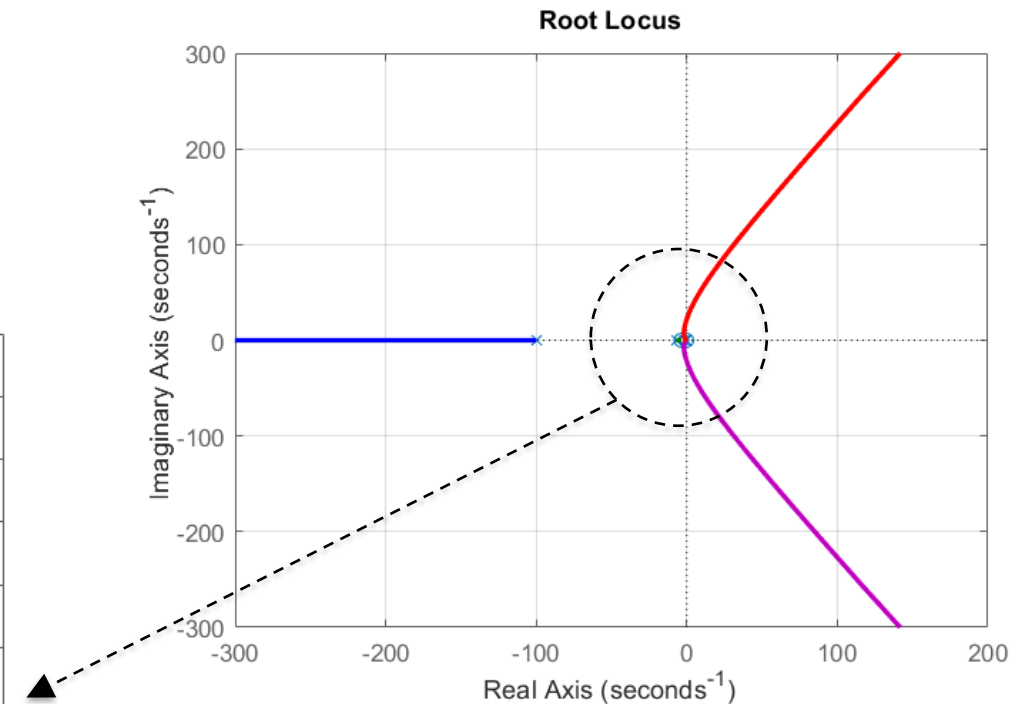
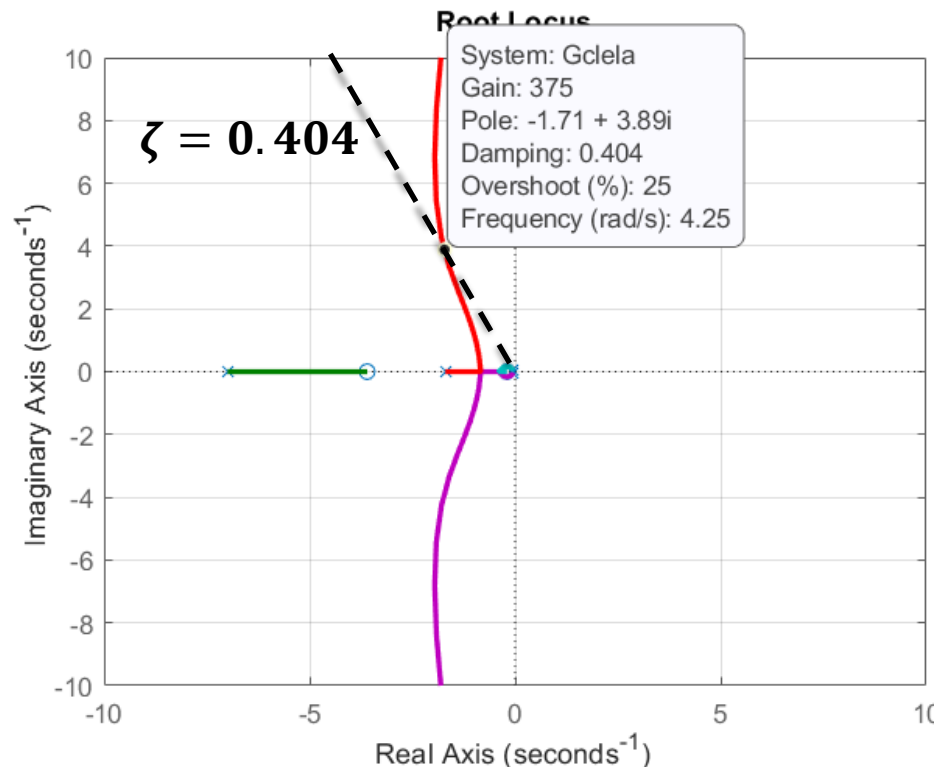
The complete lead-lag compensated open-loop system transfer function is

$$G(s) = \frac{6.63K(s + 0.2)(s + 3.6)}{s(s + 100)(s + 1.71)(s + 0.087)(s + 7)}$$

We can plot the root-locus of the lead-lag compensated system and search for the damping ratio of $\zeta = 0.404$, which represent $O.S. = 25\%$.

Then, find the desired poles and calculate the required gain K for it.

$$K = 375$$



THANK YOU