Documentation for RVG maneuvering model in MCsim Python library

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EXAMPLE DOCUMENTATION, WILL BE UPDATED ONCE MODEL IS COMPLETE.

1 Introduction

This note serves as documentation for the RVG maneuvering model.

2 Overview

The toolbox is currently separated into two main libraries:

- MClib: library of general functions.
- MCmod: library of model-specific functions.

Additional libraries, such as **MCcon** (control systems) are added as needed. Model and simulation parameters are identified by dictionary objects. Currently there are three main dictionaries:

- parV: vessel parameters, e.g. rigid body inertia, fluid inertia, viscous load parameters.
- parA: actuator model parameters, e.g. placement of azimuth thrusters, rudder force coefficients.
- parS: simulation parameters, e.g. time step, current speed and direction. (Environmental parameters are included in parS, since they are considered simulation specific.)

Additional dictionary objects, such as **parC** (control system parameters), are added as needed.

2.1 Model functions

Consider a differential equation

$$\dot{x}(t) = f(x(t), u(t), w(t), p) \tag{1}$$

with state x, control input u, disturbance input w and fixed parameters p. Model functions, contained in \mathbf{MCmod} , solve a single time step integration of (1), i.e. the solve the equation

$$x(t_{i+1}) = \int_{t_i}^{t_{i+1}} f(x(\tau), u(t_i), w(t_i), p) d\tau,$$
 (2)

where t_i is current time step, and t_{i+1} is next time step. Note that the inputs u and w are assumed constant over the time interval.

2.2 Using the toolbox

We illustrate the use of the toolbox by an example. The function

$$x_next = int_RVGMan3_lq(x,u,w,parV,parA,parS),$$

performs single time step integration of a 3DOF manuevering model of Research Vessel Gunnerus (RVG) using a "linear+quadratic" damping formulation, with commanded thruster states as control input. Here, \mathbf{x} is the state vector, u is control input, w is disturbance, \mathbf{parV} and \mathbf{parA} are model parameters and \mathbf{parS} are simulation parameters.

To simulate the response of RVG using int_RVGMan3_lq, create an initialization script consisting of the following main steps:

- 1. Load model parameters **parV** and **parA** from relevant pickle file.
- 2. Specify simulation parameters (including environmental parameters) and store in **parS**.
- 3. Simulate the response using a for loop that calculates control inputs and calls the function int_RVGMan3_lq at each time step.
- 4. Plot the results.

Model parameters (**parV** and **parA**) are generated by separate scripts and stored using the pickle package. Note that **int_RVGMan3_lq** can model any vessel with two azimuth thrusters as actuators, simply by changing the model parameters.

2.3 Programming tips

As far as possible, abide by the PEP 8 style rules [3]. If you are coming from MATLAB, useful tips are given in [2]. Highlighted tips are listed below:

- Do not use numpy.matrix objects or associated functions.
- Avoid nested numpy arrays. Ensure that functions do not unintentionally return nested numpy arrays.
- For vectors: be aware of the difference between 1D arrays (vectors without orientation), and 2D vertical or horizontal vectors, i.e. 2D arrays with shape (n,1) or (1,n). Extracting a column from a 2D array returns a 1D array. For this reason, aim at using 1D arrays to represent vectors.

3 Mathematical preliminaries

The following definitions are used in the remainder of this document. For a vector $x = [x_1 \ x_2 \ x_3]^{\top} \in \mathbb{R}^3$ we define the skew-symmetric matrix cross-product operator $S : \mathbb{R}^3 \to \mathbb{R}^{3 \times 3}$ as

$$S(x) := \begin{bmatrix} 0 & -x_3 & x_2 \\ x_3 & 0 & -x_1 \\ -x_2 & x_1 & 0 \end{bmatrix}.$$
 (3)

We define the selection vectors

$$\varepsilon_1 := \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}, \quad \varepsilon_2 := \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix}, \quad \varepsilon_3 := \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}.$$
(4)

4 General 3DOF maneuvering models

The dynamics of 3DOF manuevering models are calculated using:

- State
 - eta: $\eta := [x \ y \ \psi]^{\top} \in \mathbb{R}^2 \times [-\pi, \pi]$: Ship pose (position and heading), nu: $\nu := [u \ v \ r]^{\top} \in \mathbb{R}^3$: Body-fixed velocities.
- Input: \mathbf{F} : $F \in \mathbb{R}^3$ external force vector, including acutator loads
- Parameters: parV, parS: vessel and simulation parameters.

4.1 Kinematics

Define the 3DOF rotation matrix about the vertical axis as

$$R_z(\psi) := \begin{bmatrix} \cos(\psi) & -\sin(\psi) & 0\\ \sin(\psi) & \cos(\psi) & 0\\ 0 & 0 & 1 \end{bmatrix}.$$
 (5)

The kinematic equation relating ship pose with body-fixed velocities is given by

$$\dot{\eta} = R_z(\psi)\nu. \tag{6}$$

A steady (time-invariant) and uniform (in space) current is described by

• Current speed: $U_c \in \mathbb{R}$.

• Flow direction: $\beta_c \in [-\pi, \pi]$.

The inertial frame current velocity vector $\nu_c^n \in \mathbb{R}^3$ is given by

$$\nu_c^n := U_c \begin{bmatrix} \cos(\beta_c) & \sin(\beta_c) & 0 \end{bmatrix}^\top. \tag{7}$$

The body-fixed current velocity $\nu_c \in \mathbb{R}^3$ becomes

$$\nu_c := [R_z(\psi)]^\top \nu_c^n. \tag{8}$$

The derivative of the body-fixed current velocity is given by

$$\dot{\nu}_c = \frac{d[R_z(\psi)]^\top}{dt} \nu_c^n = r[S(\varepsilon_3)R_z(\psi)]^\top \nu_c^n. \tag{9}$$

4.2 Kinetics

Define the fluid relative velocity $\nu_r \in \mathbb{R}^3$ as

$$\nu_r := \nu - \nu_c. \tag{10}$$

The nonlinear equation of motion of a ship is commonly stated as

$$M_{rb}\dot{\nu} + M_{rb}\dot{\nu}_r + C_{rb}(\nu)\nu + C_a(\nu_r) + D(\nu_r)\nu_r = F,$$
 (11)

where M_{rb} , $M_a \in \mathbb{R}^{3\times3}$ are the rigid body and added mass matrices, respectively, C_{rb} , $C_a : \mathbb{R}^3 \to \mathbb{R}^{3\times3}$ are the rigid body and added mass Coriolis and centripetal force matrices, and $D : \mathbb{R}^3 \to \mathbb{R}^{3\times3}$ is the hydrodynamic damping matrix. F collects all external forces, e.g. actuator forces, wave and wind loads.

Eq. (11) can be solved using either ν or ν_r as the state. Choosing ν we obtain

$$\dot{\nu} = [M_{rb} + M_a]^{-1} (F - C_{rb}(\nu)\nu - C_a(\nu_r) - D(\nu_r)\nu_r + M_a\dot{\nu}_c), \tag{12}$$

where $\dot{\nu}_c$ is given by (9).

4.2.1 Coriolis and centripetal forces

We assume ships with port-starboard symmetry, and reference point along the center line. This gives inertia matrices of the form

$$M = \begin{bmatrix} m_{11} & 0 \\ 0 & m_{22} & m_{23} \\ 0 & m_{32} & m_{33} \end{bmatrix}. \tag{13}$$

The rigid body mass matrix is symmetric, while the added mass matrix is in general non-symmetric. The Coriolis and centripetal force matrices are parametrized as

$$C(\nu) := \begin{bmatrix} 0 & 0 & -m_{22}v - 0.5(m_{23} + m_{32})r \\ 0 & 0 & m_{11}u \\ m_{22}u + 0.5(m_{23} + m_{32})r & -m_{11}u & 0 \end{bmatrix}.$$
(14)

4.3 Hydrodynamic viscous loads

The function **dot_nu3_man_lq** calculates the derivative of $\nu \in \mathbb{R}^3$, using maneuvering theory, and a "linear+quadratic" damping formulation given by

$$D(\nu_r) := D_l + D_u |u_r| + D_v |v_r| + D_r |r|, \tag{15}$$

where $D_l \in \mathbb{R}^{3\times3}$ is the linear damping matrix, and $D_u, D_v, D_r \in \mathbb{R}^{3\times3}$ are matrices of quadratic damping coefficients. The relation to second-order modulus functions using hydrodynamic derivatives is illustrated by an example. The viscous damping formulation proposed by Blanke (see [1, Section 7.1.2]),

$$D(\nu_r) = \begin{bmatrix} -X_{|u|u}|u_r| & 0 & 0\\ 0 & -Y_{|v|v}|v_r| & -Y_{|v|r}|v_r|\\ 0 & -N_{|v|v}|v_r| & -N_{|v|r}|v_r| \end{bmatrix}$$
(16)

is represented by the matrices $D_l = D_r = 0$, and

$$D_{u} = \begin{bmatrix} -X_{|u|u} & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \quad D_{v} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & -Y_{|v|v} & -Y_{|v|r} \\ 0 & -N_{|v|v} & -N_{|v|r} \end{bmatrix}.$$
 (17)

5 Actuator models

5.1 First order actuator models

By first order actuator models, we mean actuator dynamics modelled as

$$\dot{x} = -T(x - x_d),\tag{18}$$

where x is the actuator state (e.g. rudder angle), x_d is the desired actuator state, and T is a diagonal matrix of time constants.

5.2 Azimuth thruster loads

Consider an azimuth thruster located at a position $r_t := [x_t \ y_t \ z_t]^{\top} \in \mathbb{R}^3$ in the body-fixed coordinate system, with thrust force $\tau \in \mathbb{R}$ and azimuth angle $\alpha \in [-\pi, \pi]$. The body-fixed actuator force vector $F : \mathbb{R}^2 \to \mathbb{R}^6$ is given by

$$F(\tau, \alpha) = \tau \begin{bmatrix} R_z(\alpha)\varepsilon_1 \\ S(r_t)R_z(\alpha)\varepsilon_1 \end{bmatrix}, \quad \varepsilon_1 := \begin{bmatrix} 1 & 0 & 0 \end{bmatrix}^\top, \tag{19}$$

which implicitly defines the convention that $\tau > 0$ and $\alpha = 0$ represents a pure surge force in the forward direction. Expanding we obtain

$$F(\tau, \alpha) = \tau \begin{bmatrix} \cos(\alpha) \\ \sin(\alpha) \\ 0 \\ -z_t \sin(\alpha) \\ z_t \cos(\alpha) \\ -y_t \cos(\alpha) + x_t \sin(\alpha) \end{bmatrix}.$$
 (20)

Note that (19) does not contain "rudder" loads, that are important for foil-shaped azimuth thrusters at forward speeds.

6 RVG maneuvering model

The function MCmod.int_RVGMan3_lq models the 3DOF maneuvering equations of RVG, combined with a first order model of the two azimuth thrusters. Thruster forces and azimuth angles are considered control inputs.

Main parameters of RVG are provided in Table 1.

Table 1: Main parameters of RVG

Parameter	Value
Length between perpendiculars	$L_{pp} = 33.9 \mathrm{m}$
Breadth (moulded)	$B_m = 9.6 \text{m}$
Draught (moulded)	$d_m = 2.7 \text{ m}$

6.1 Vessel data

HOLD

6.2 Thruster data

The location of the thrusters, crudely estimated from drawings, are provided in Table 2.

Table 2: Thruster locations of Gunnerus

Parameter	Value	Comment
Longitudinal	$0 \mathrm{m}$	Relative to aft perpendicular
Vertical	$0.3 \mathrm{m}$	Above baseline
Lateral	$\pm 2.7~\mathrm{m}$	Relative to centerline

References

- [1] Thor I Fossen. Handbook of marine craft hydrodynamics and motion control. John Wiley & Sons, 2011.
- [2] numpy for MATLAB users. https://numpy.org/doc/stable/user/numpy-for-matlab-users.html.
- [3] PEP 8 Python style guide. https://www.python.org/dev/peps/pep-0008/.