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ME469: A Verification and Validation (V&V) Methodology (Review)

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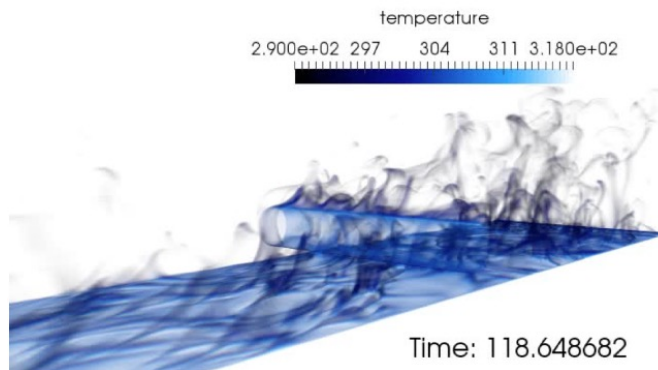
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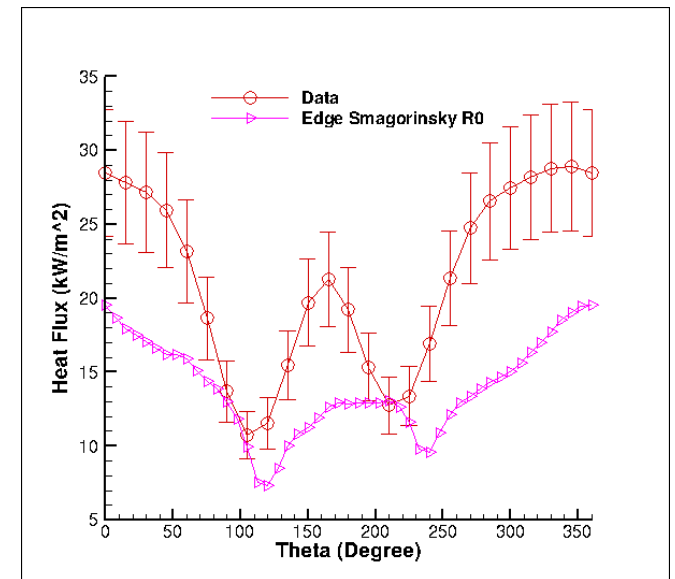


Challenge: Understanding Errors/Uncertainties....

- One mesh, one model, unknown code/numerical pedigree...
- We need to distinguish the types of errors/uncertainties:
 - Conceptual uncertainty, δ_{input}
 - Model-form error/uncertainty, δ_{model}
 - Discretization Error, $\delta_{\text{numerical}}$
 - Code Error, $\delta_{\text{numerical}}$



Heat flux to the cylinder
Volume-rendered temperature



Time-averaged heat flux to cylinder

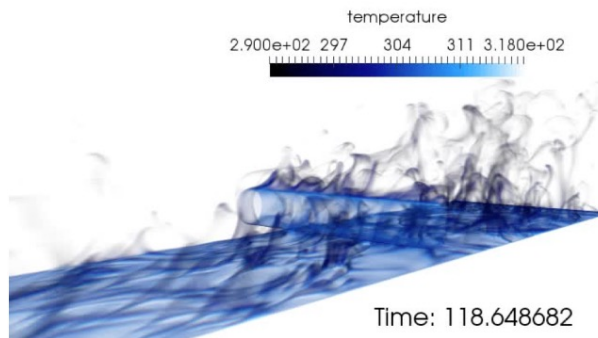
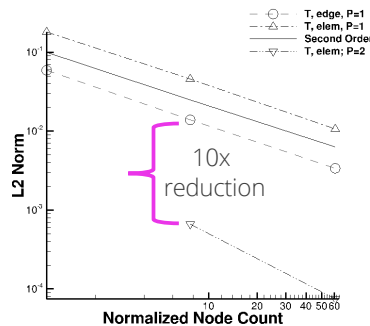
- What credible scientific hypothesis can be tested in this context?



Review of a Strong V&V Process

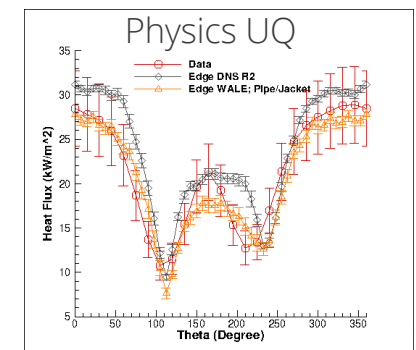
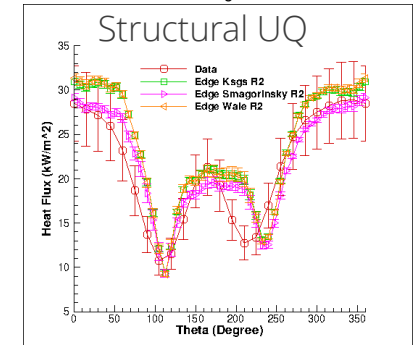
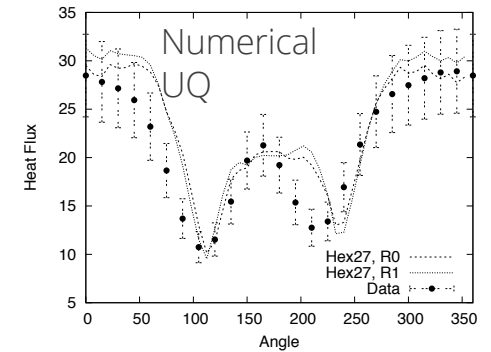
Establish a sound LES-based V&V process (with uncertainty quantification) that includes the following attributes:

- Phenomena Identification and Ranking (PIRT)
- Code and solution verification (numerical error, $\delta_{\text{numerical}}$)
- Validation including solution sensitivity to model inputs (δ_{input})
- Structural uncertainty (model form error, δ_{model})
- Physics assumptions (your conceptual model)



Sources of error and
uncertainty in
simulation

$\delta_{\text{numerical}}$, δ_{input} , δ_{model}



"An assessment of atypical mesh topologies for low-Mach LES", Domino et al., *Comp & Fluids*, 2019



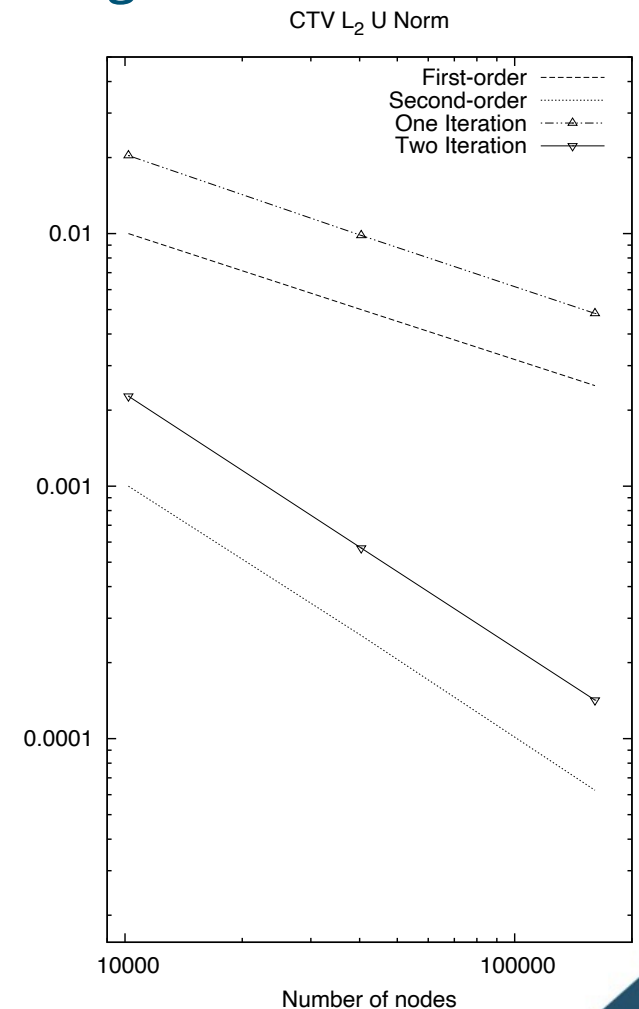
Code or Conceptual Error? Part 1: Time Splitting

Case Study: An Algorithm is thought to be second-order-in-time accurate with one nonlinear iteration: True or False?

- Issa, "Solution of the implicitly discretized fluid flow equations by operator splitting", JCP (1985).
 - Advent of the "Pressure-implicit with Splitting of Operators", or PISO
- PISO is a scheme that defines a series of predictors and correctors in the context of a fully implicit solve

Conclusion?

- Sometimes we code a method correctly, however, have a conceptual error in our understanding of whether or not a scheme is design-order accurate when run in the suggested manner

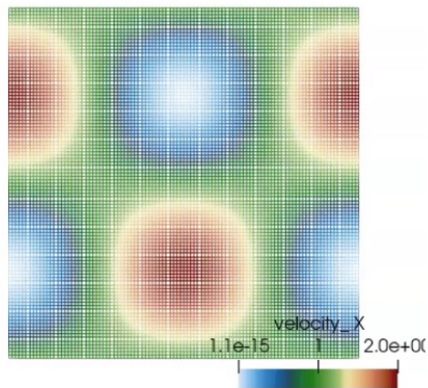




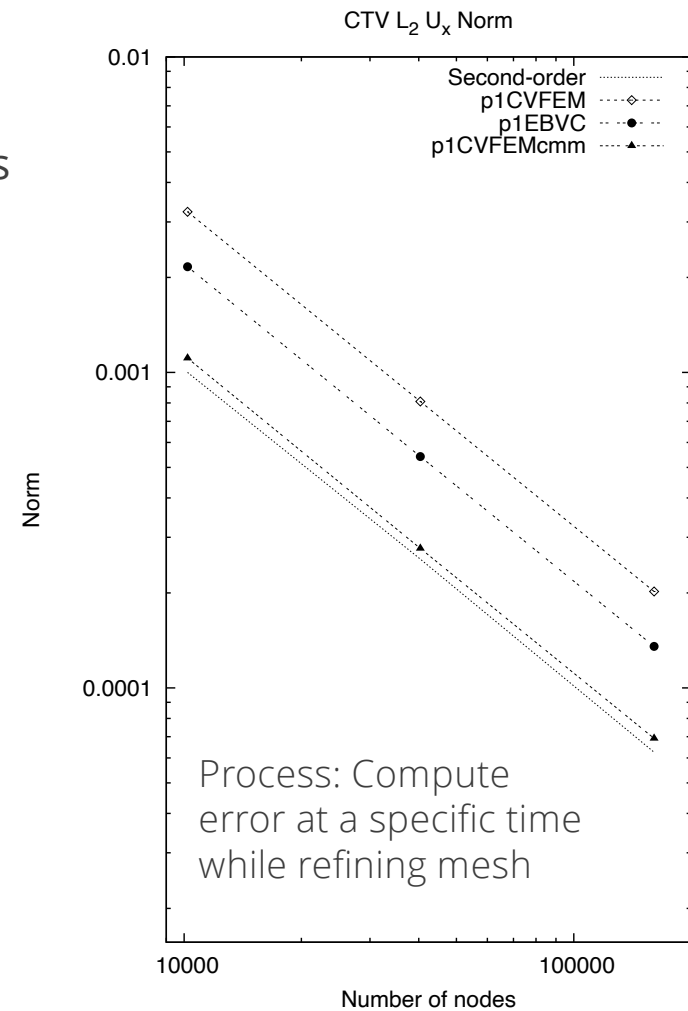
Consistent vs Lumped Mass Matrix

Goal: Explore the time accuracy of the consistent mass matrix approach on a structured mesh

- Convecting Taylor Vortex Case Study: Analytical, transient verification problem



$$\begin{aligned}u &= u_o - \cos(\pi(x - u_o t)) \sin(\pi(y - v_o t)) e^{-2.0\omega t} \\v &= v_o + \sin(\pi(x - u_o t)) \cos(\pi(y - v_o t)) e^{-2.0\omega t} \\p &= -\frac{P_o}{4} (\cos(2\pi(x - u_o t)) + \cos(2\pi(y - v_o t))) e^{-4\omega t}\end{aligned}$$





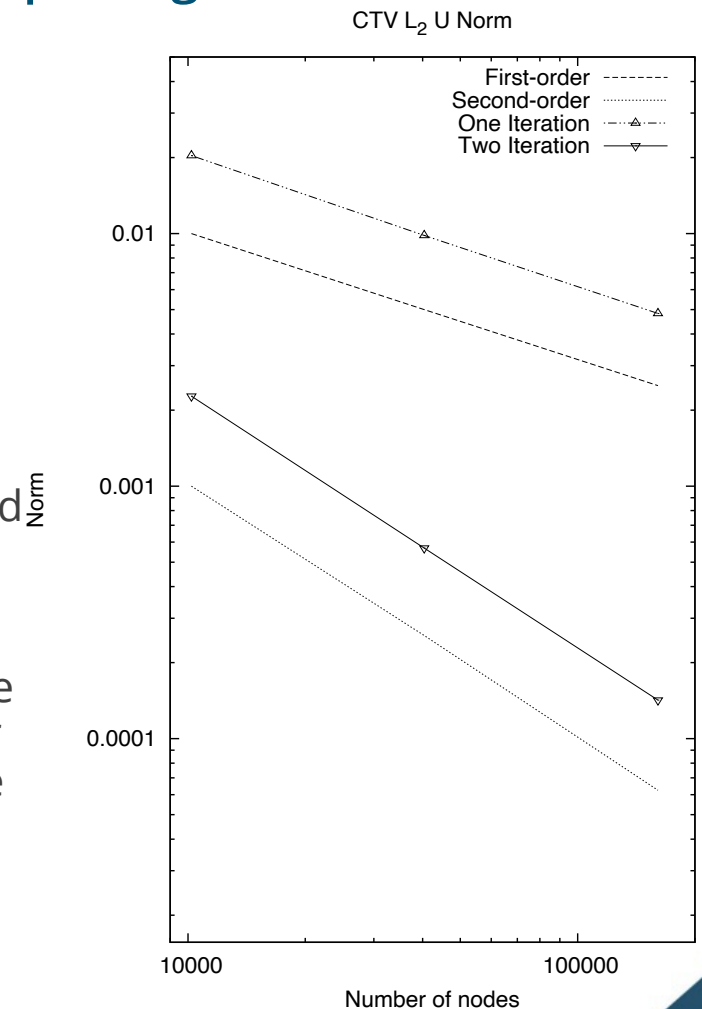
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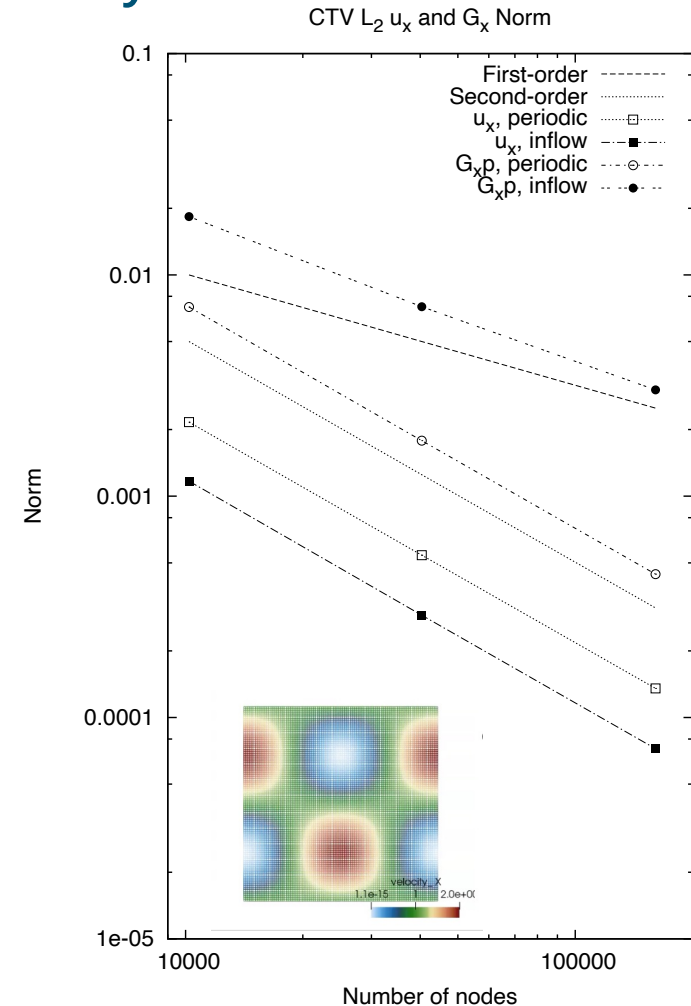




Code or Conceptual Error? Part 2: Boundary Conditions

Case Study: Using the convecting Taylor Vortex case, explore boundary conditions

- Choice 1: Periodic with specified initial condition
 - Requires no formal boundary conditions
- Choice 2: Inflow
 - Easy, velocity is specified as a Dirichlet
- What about pressure? The default approach is to assume a zero normal pressure gradient (no-op) – similar to what would be found at a wall
- A notion of a “spurious numerical boundary layer” (Gresho, 1995) shows a characteristic length $\delta \sim \sqrt{\nu \Delta t}$ error
- We also need to make sure that we are adequately converging the nonlinear system within the time step





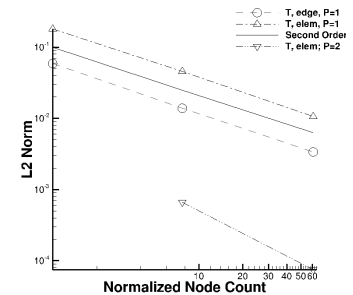
Spatial Verification Process

Process:

1. Start with a mesh of given resolution, Δx_0 , or a given total node/cell-centered, etc. count, N
2. Refine the mesh, uniformly, i.e., h-refinement or in polynomial order, p-refinement
3. Run the series of refinement/promoted meshes and compute an integrated norm over the domain, L_∞, L_1, L_2 , for example, below shown to be based on a nodal DOF prediction,

$$L_\infty = \sum_i \max |\epsilon_i| \quad L_1 = \sum_i \frac{|\epsilon_i|}{N} \quad L_2 = \sum_i \frac{\epsilon_i^2}{N^2}$$

4. Evaluate error relative to what the precise solution think it should be
 - Generally easiest to observe the magnitude of reduction
 - For example, $\Delta x_1 = \frac{\Delta x_0}{2}$, for a first-order in space scheme reduces the error by 2 while for second-order, 4, and third-order, 8, etc.



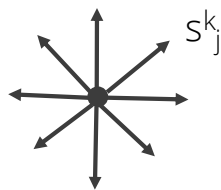


Lurking, or Hidden Errors...

- In some numerical implementations, an error can exist that is not easily found at a given mesh resolution

For example:

- There may be a boundary error that manifests itself locally at a very small subset of the mesh
- There may be an error that is driven by the fidelity of the mesh (consider our specified pressure drop in a pipe example)
- Other errors, e.g., for a discrete-ordinate method, we have an underlying *quadrature* error


$$s_j \frac{\partial I}{\partial x_j} + (\mu_a + \mu_s) I = \frac{\mu_a \sigma T^4}{\pi} + \frac{\mu_s G}{4\pi}$$
$$s_j^k \frac{\partial I^k}{\partial x_j} + (\mu_a + \mu_s) I^k = \frac{\mu_a \sigma T^4}{\pi} + \frac{\mu_s G}{4\pi} \quad G \approx \sum_k w_k I^k$$

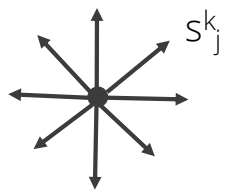


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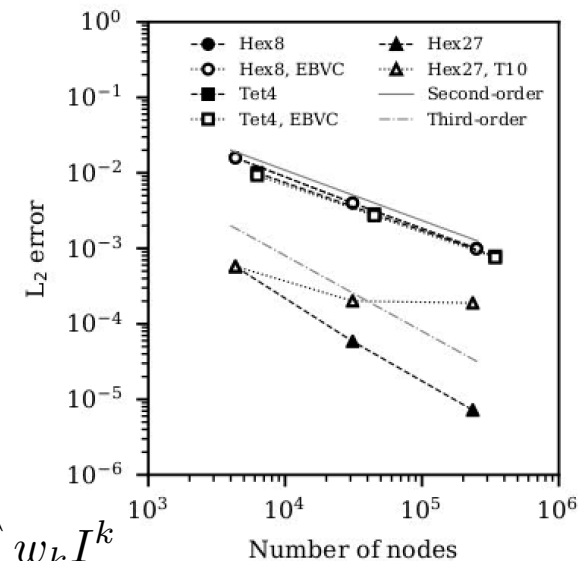
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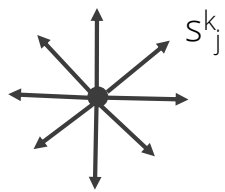


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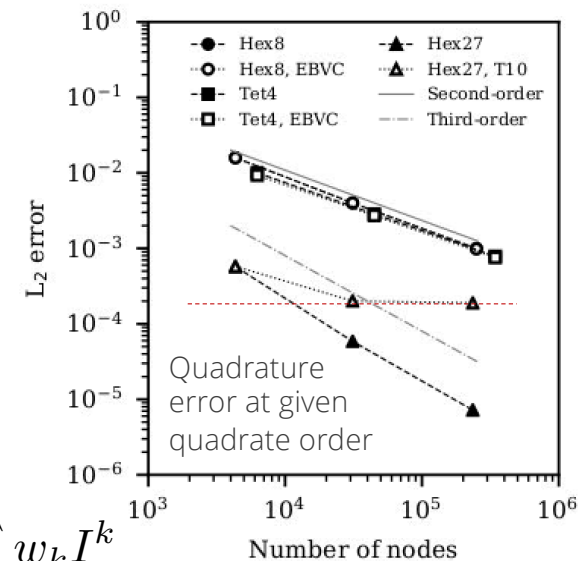
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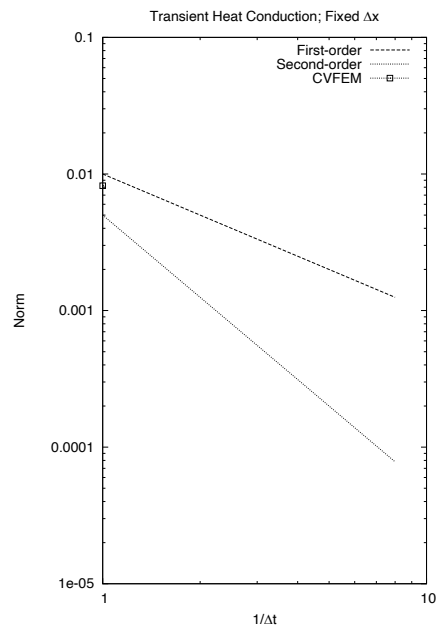
$$G \approx \sum_k w_k I^k$$





Temporal Verification: Fixed Spatial Resolution; Reduced Timestep

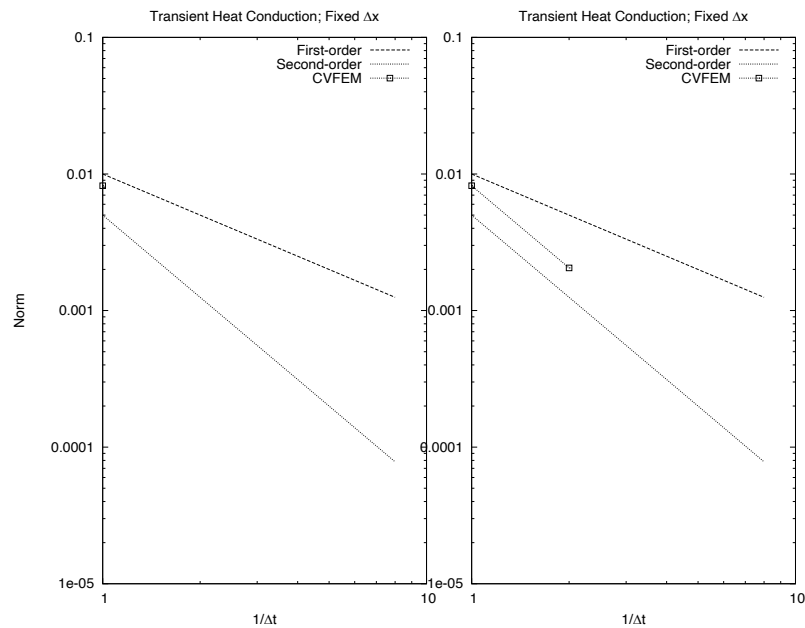
On a given mesh, compute error at a given time: $\epsilon \propto k_1 \Delta x^n + k_2 \Delta t^p$





Temporal Verification: Fixed Spatial Resolution; Reduced Timestep

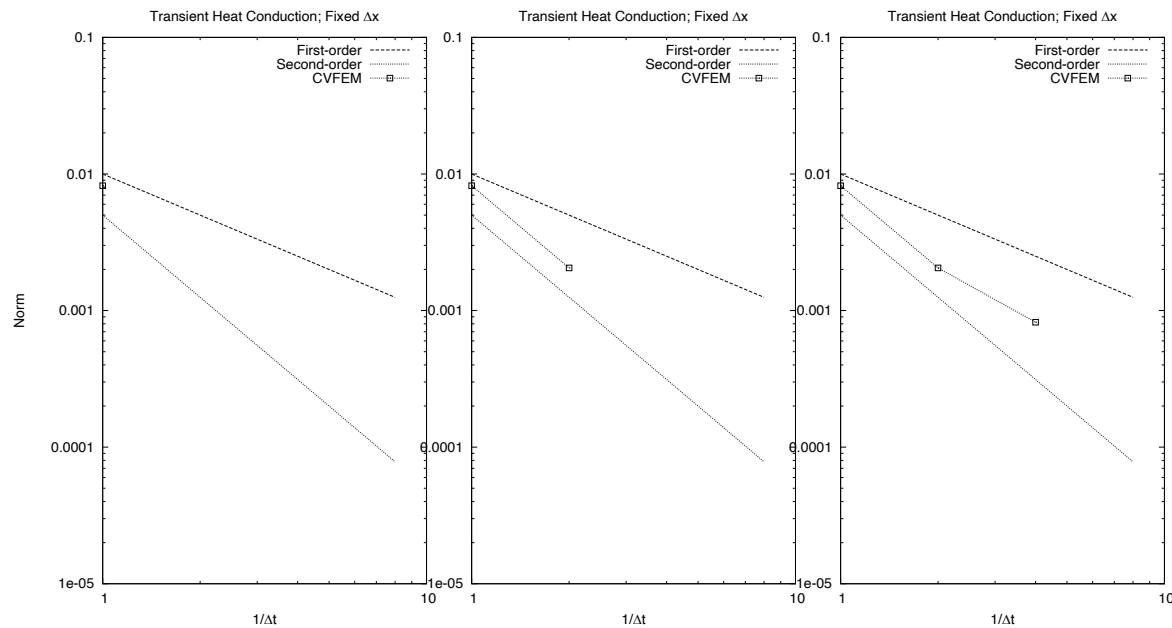
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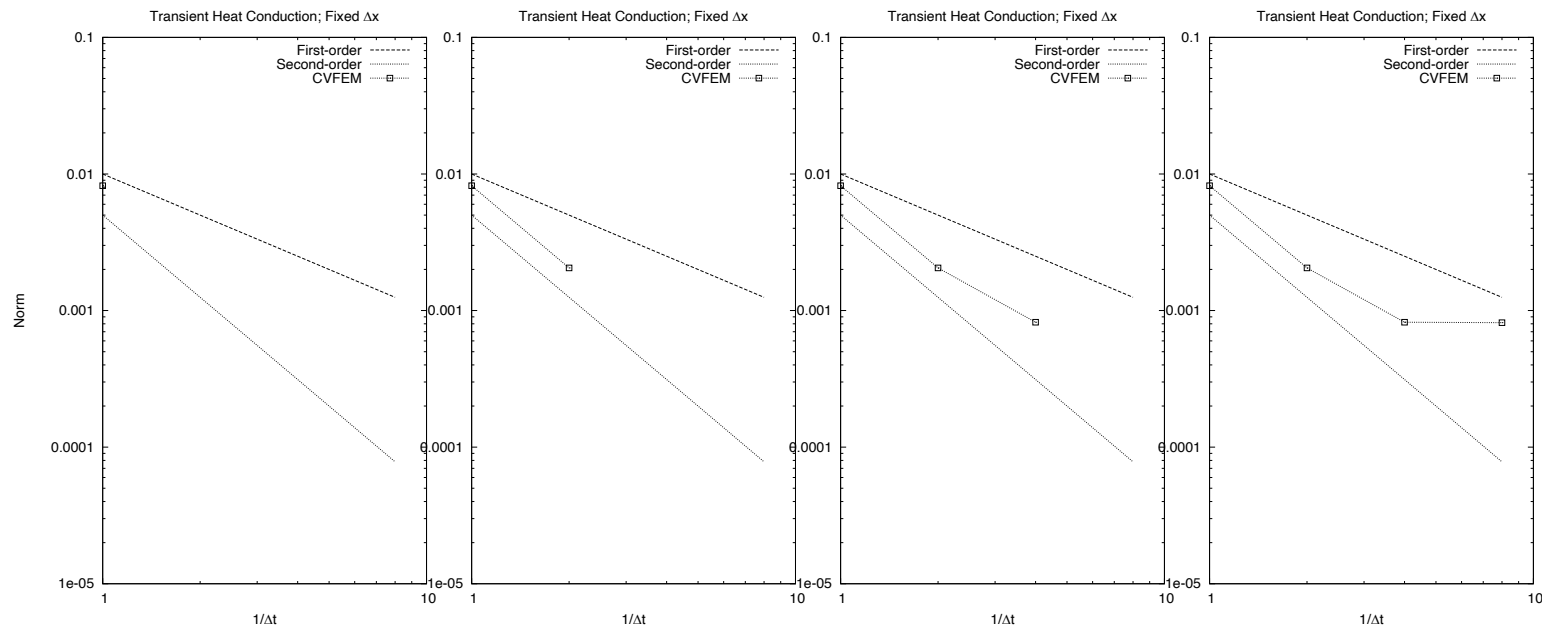




Temporal Verification: Fixed Spatial Resolution; Reduced Timestep

On a given mesh, compute error at a given time: $\epsilon \propto k_1 \Delta x^n + k_2 \Delta t^p$

At a given mesh, the mesh spacing is fixed and, eventually, manifests itself

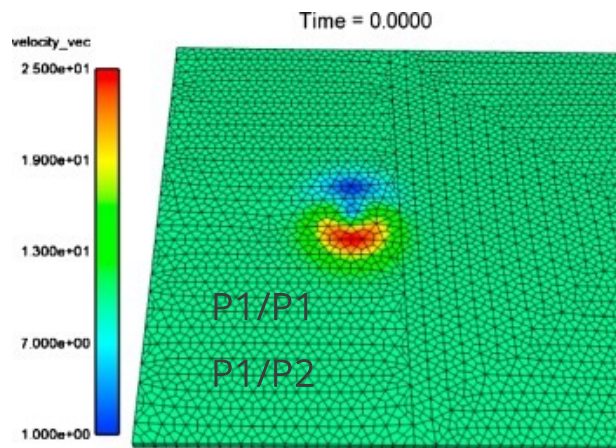




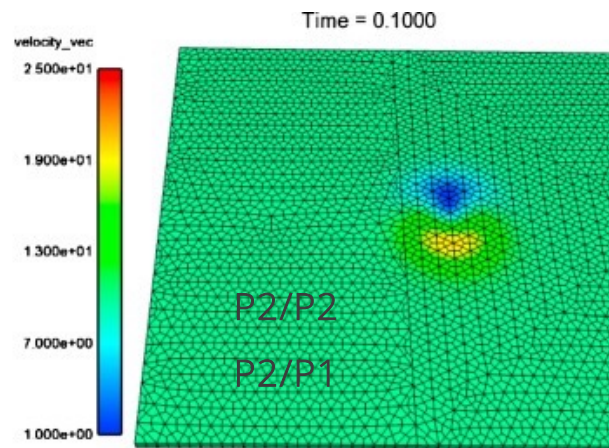
Temporal Verification: Reduction of Timestep as Mesh Is Refined

On a series of uniform mesh refinements, compute error at a given time while reducing time step

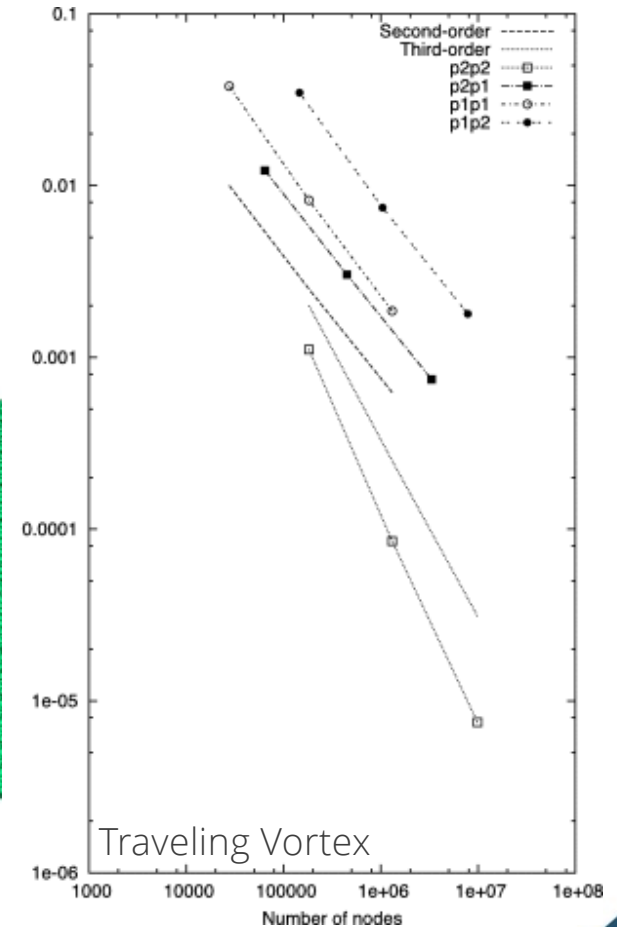
- Must ensure that the spatial error does not dominate: $\epsilon \propto k_1 \Delta x^n + k_2 \Delta t^p$



(a) Time = 0.0 seconds.



(b) Time = 0.10 seconds.

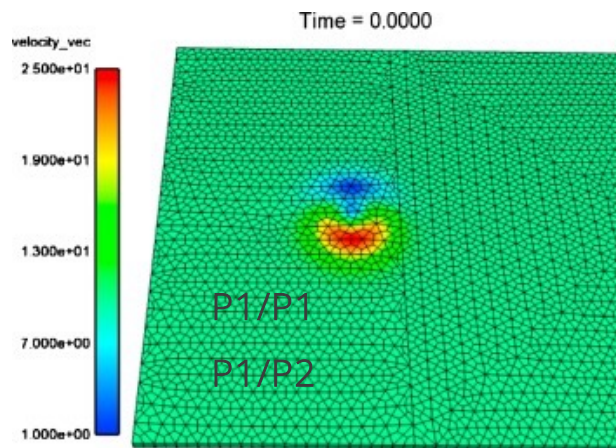




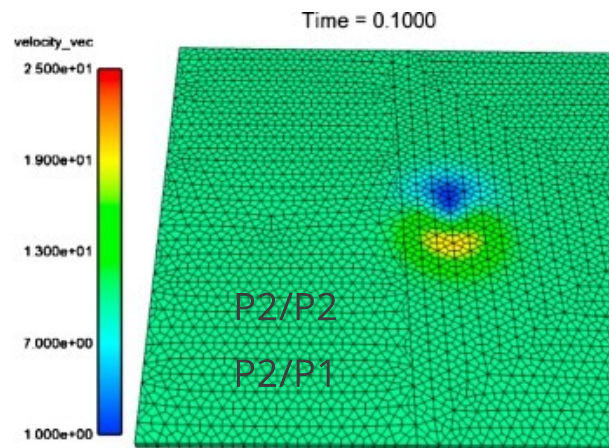
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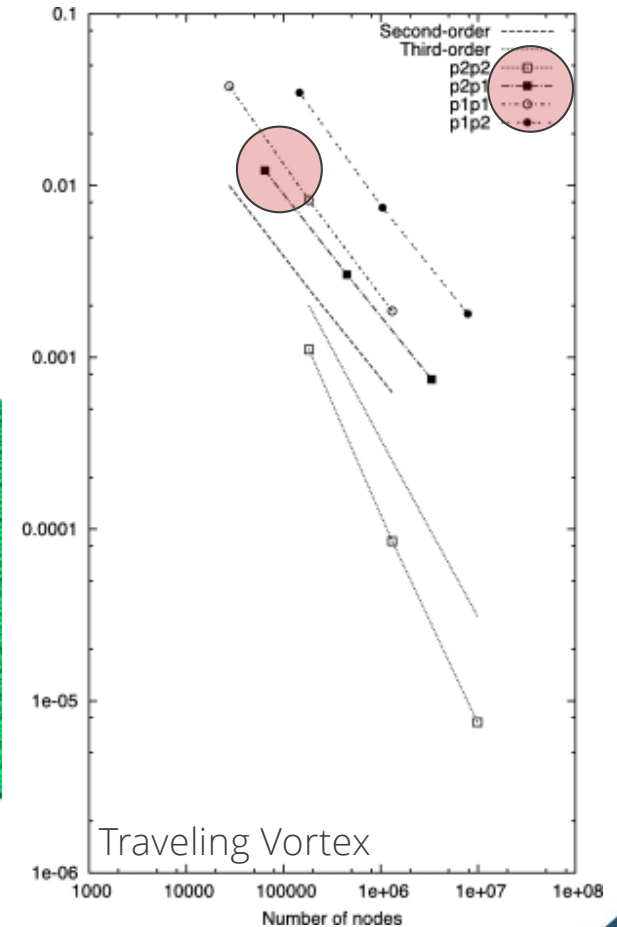
(a) Time = 0.0 seconds.



(b) Time = 0.10 seconds.



P2/P1





Domino et al., Phys. Review Fluids (2025): Fire-engulfed Object in X-wind: Can we predict **windward** → **leeward** migration of peak heat flux (and magnitude) using WMLES?

Figure 11.8 - Test 7 Photograph (low winds)

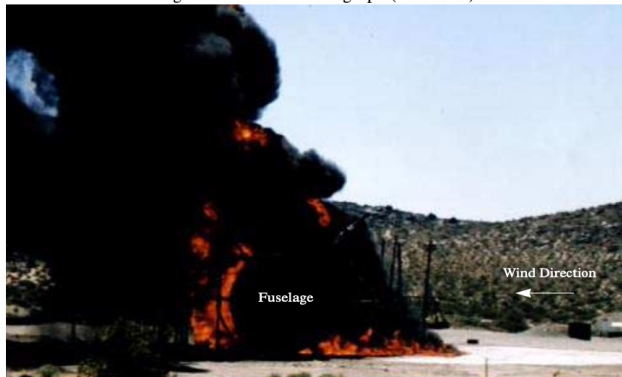
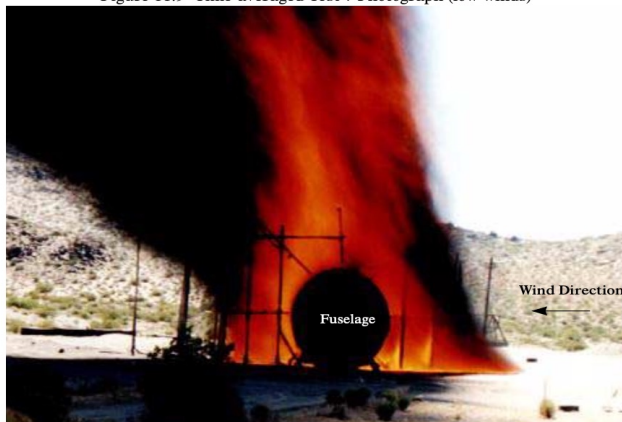


Figure 11.9 - Time-averaged Test 7 Photograph (low winds)



Suo Antilla and Gritzo, "The Effects of Wind on Fire Environments Containing Large Cylinders", Comb. Sci. Tech. (2008)

Wind
←

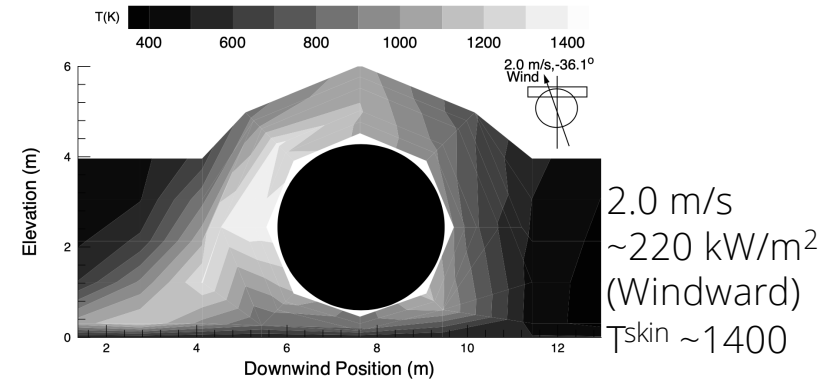


Figure 4.35 - Test 7 Thermocouple Temperature, Windward of Center, 250-500s

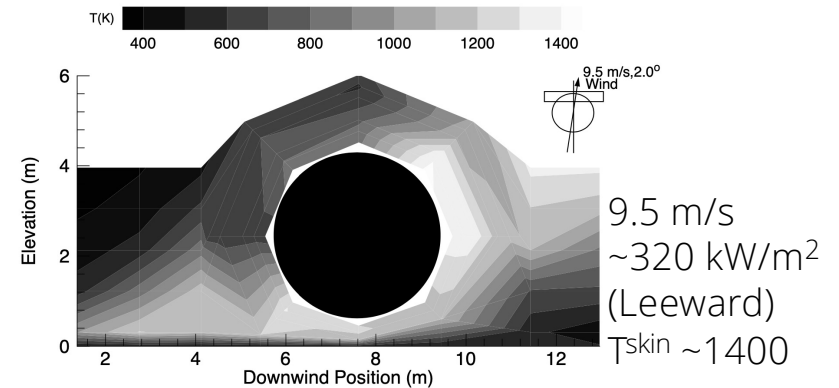


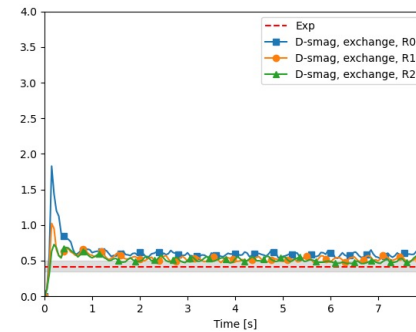
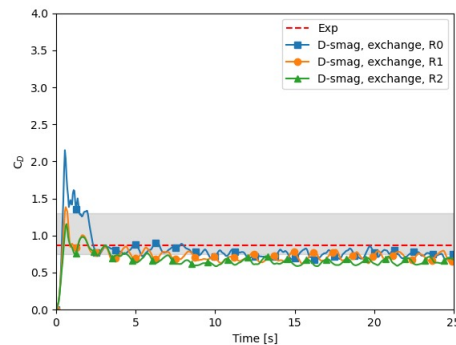
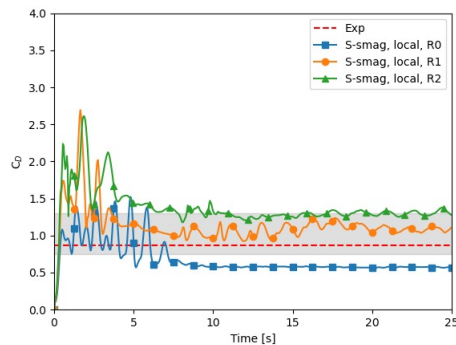
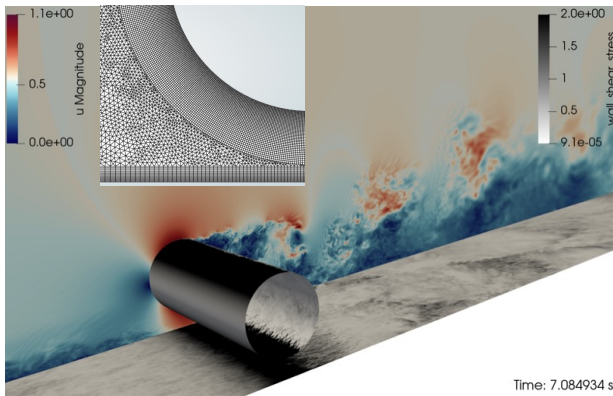
Figure 4.32 - Test 6 Thermocouple Temperature, Leeward of Center, 270-390s

Wind
→

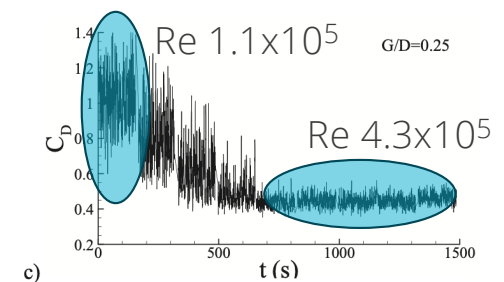


Step 1: Isothermal Crosswind Simulation Structural Uncertainty Study: Re 1.1×10^5 and 4.3×10^5 Simulations Gap (G) Diameter (D): $G/D = 0.25$

- $G/D = 0.25$, Yang et al., Ocean Eng. (2018) mimics the fire-engulfed fuselage use case (sub/supercritical drag regime; drag crisis well-predicted)



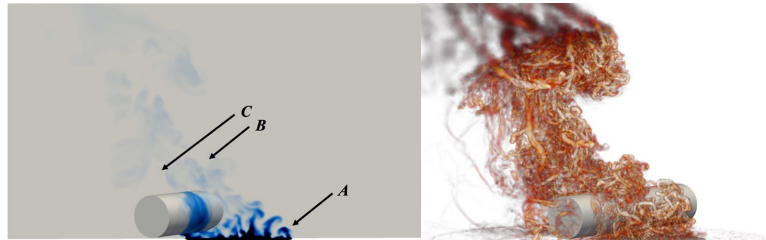
- Finding:** Wall-modeled large-eddy simulation (WMLES) using a dynamic model with exchange-based velocity sampling increases predictivity (C_D & C_p) and convergence as a $f(\Delta x)$
- $O(600)$ million element mesh for Refinement 2!



$$\mu_t = \bar{\rho} f_\mu C_{\mu\epsilon} \Delta \sqrt{k_{SGS}}$$



Step 2: Fire V&V

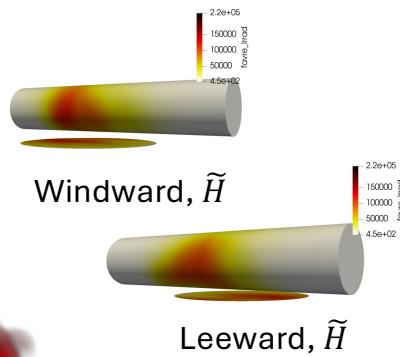


Fuel shadings

VR Q-criterion

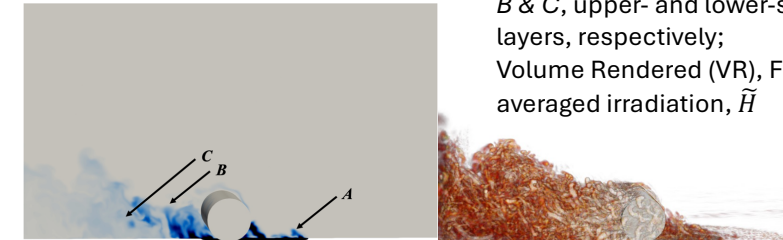


Mean flame contour



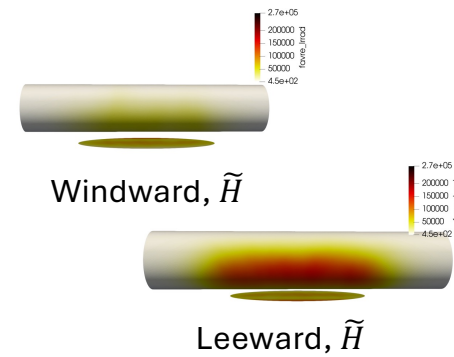
VR Temperature

2.0 m/s



Fuel shadings

VR Q-criterion



VR Temperature

Mean flame contour

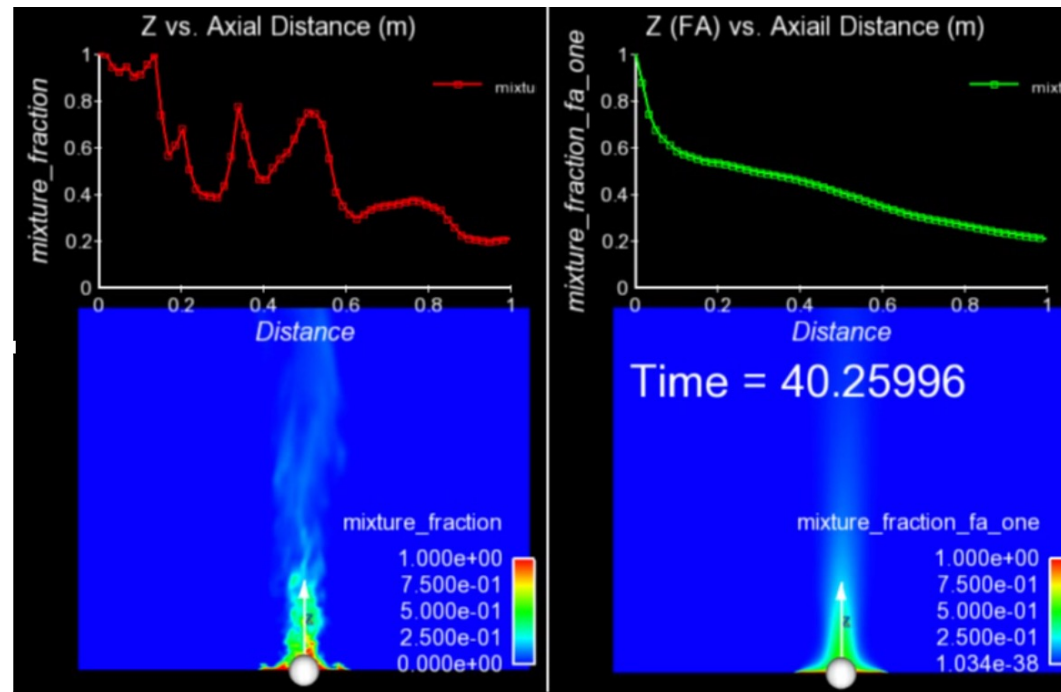
9.5 m/s

Key: A, windward pool structures;
B & C, upper- and lower-shear
layers, respectively;
Volume Rendered (VR), Favre-
averaged irradiation, \tilde{H}



Finally.... For Transient Flows, Averaging is Required

- The **bane** of turbulent validation: converged statistics require many flow-through times
- Statistical convergence of a given simulation may require many flow-through-times; additional source of uncertainty and/or requirement for quantification of solution convergence





Essentials of Code Verification: Review

Taxonomy: One *verifies* code and *validates* models

- Code verification establishes the numerical accuracy of the underlying discretization for the given partial differential equation set
- Code verification seeks to provide the temporal and spatial accuracy of the underlying discretization approach

For temporal discretization error,

- A two-state Backward Euler time integrator should be first-order in time, specifically the error should scale with Δt
- A three-state BDF2 time integrator should scale with Δt^2
- A multi-state Runge-Kutta schemes can achieve higher-order accuracy

For spatial discretization error,

- A method is design-order if the observed order of accuracy is Δx^{P+1} , where P is the underlying basis polynomial order

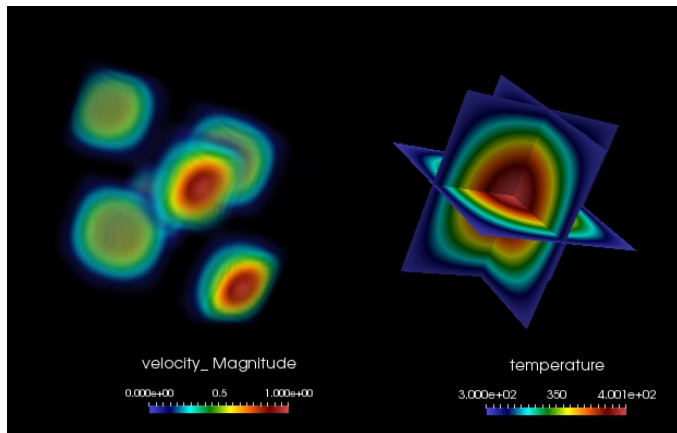
Oberkampf and Trucano, Verification and validation in computational fluid dynamics, Progress in Aerospace Sciences, Volume 38, Issue 3, 2002, [https://doi.org/10.1016/S0376-0421\(02\)00005-2](https://doi.org/10.1016/S0376-0421(02)00005-2).



Spatial Code Verification for a low-Mach, Variable-Density Flow

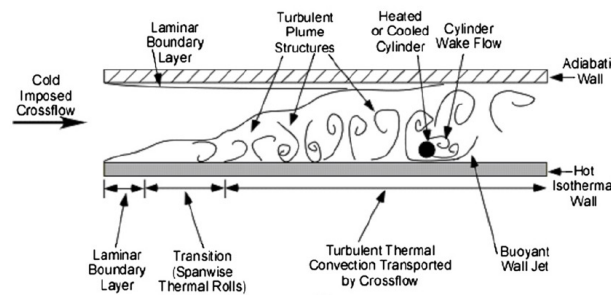
		Import				Adequacy		
		Phen	Mod	Code	Val	Mats		
Convective heat transfer		M	M	M	L			

- Density is a function of static enthalpy transport via the standard ideal gas, $\rho = f(P, M, R, T)$
- Temperature range maps to experiment (see below)
- Arbitrary buoyancy source term via rotated gravity vector
- Collective study now provides confidence in the interplay between numerical and modeling accuracy

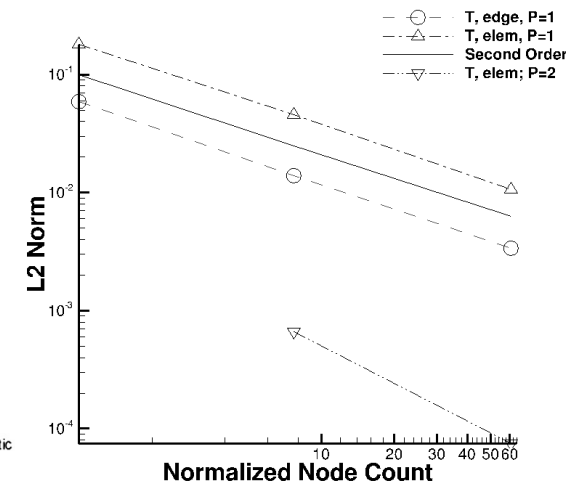


Velocity Mag

Temperature



Kearney experimental configuration



See, "Exploring model-form uncertainties in large-eddy simulations", Domino et al, 2016



Review of the Method of Manufactured Solutions (MMS):

Providing confidence that the code implementation converges to the proper solution

- We understand that the number of analytical solutions to test our code implementation are very few in number
- How can we test the numerical accuracy of our implementation that, in general, solves very complex physics?
- Specifically, as we refine the mesh and time step, how does the error respond?
- New, analytically modified system that includes a new source term that we can implement in the code base:

$$\rho C_p \frac{\partial T^{mms}}{\partial t} - \frac{\partial}{\partial x_j} \lambda \frac{\partial T^{mms}}{\partial x_j} = S^{mms}$$

- The error is computed to be the difference between the analytical, or manufactured solution and our numerical simulation, T^h
- We can now refine the mesh and timestep, while computing the error to ensure that the rate of reduction is expected
- For example, if we believe our scheme is 2nd or 3rd order in space accuracy, one uniform refinement should reduce the error by 4x or 8x, respectively

Consider a simple heat conduction PDE:

$$\rho C_p \frac{\partial T}{\partial t} - \frac{\partial}{\partial x_j} \lambda \frac{\partial T}{\partial x_j} = 0$$

With given [steady] manufactured solution:

$$T^{mms}(x, y, z) = \frac{k}{4\lambda} (\cos(2\pi x) + \cos(2\pi y) + \cos(2\pi z))$$

$$S^{mms}(x, y, z) = k\pi^2 (\cos(2\pi x) + \cos(2\pi y) + \cos(2\pi z))$$