Pandit Deendayal Energy University Computational Engineering Laboratory Matlab Case study Report

Ву,

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Case Study 1

Problem Statement:-

To Solve a problem on Heat Transfer of Natural Convection in H – Shaped Cavity.

Derivation:-

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \tag{1}$$

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = -\frac{1}{\rho_{lmf}}\frac{\partial p}{\partial x} + \frac{\mu_{lmf}}{\rho_{lmf}}\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right) \tag{2}$$

$$u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} = -\frac{1}{\rho_{lmf}}\frac{\partial p}{\partial y} + \frac{\mu_{lmf}}{\rho_{lmf}} \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) + g\beta_{T,lmf} (T - T_c) + g\beta_{c,lmf} (c - c_c)$$
(3)

$$u\frac{\partial T}{\partial x} + v\frac{\partial T}{\partial y} = \alpha_{hnf} \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) \tag{4}$$

$$u\frac{\partial c}{\partial x} + v\frac{\partial c}{\partial y} = D\left(\frac{\partial^2 c}{\partial x^2} + \frac{\partial^2 c}{\partial y^2}\right) \tag{5}$$

Introducing the following dimensionless variables:

$$U = \frac{uL}{\alpha_f}, V = \frac{vL}{\alpha_f}, \theta = \frac{(T - T_c)}{\Delta T}, C = \frac{(c - c_c)}{\Delta c}, P = \frac{pL^2}{\rho_f \alpha_f^2}$$

$$\tag{6}$$

The non-dimensional shape of governing equations becomes:

$$U\frac{\partial U}{\partial X} + V\frac{\partial U}{\partial Y} = -\frac{\rho_f}{\rho_{hnf}}\frac{\partial P}{\partial X} + \Pr{\frac{\rho_f}{\rho_{hnf}}}\frac{\mu_{hnf}}{\mu_f}\left(\frac{\partial^2 U}{\partial X^2} + \frac{\partial^2 U}{\partial Y^2}\right) \tag{7}$$

$$U\frac{\partial V}{\partial X} + V\frac{\partial V}{\partial Y} = -\frac{\rho_f}{\rho_{hnf}}\frac{\partial P}{\partial Y} + \Pr\frac{\rho_f}{\rho_{hnf}}\frac{\mu_{hnf}}{\mu_f}\left(\frac{\partial^2 V}{\partial X^2} + \frac{\partial^2 V}{\partial Y^2}\right) + \frac{\beta_{hrf}}{\rho_{hnf}}Ra\Pr(\theta + NC)$$
(8)

$$U\frac{\partial\theta}{\partial X} + V\frac{\partial\theta}{\partial Y} = \frac{\left(k_{hmf}/k_f\right)}{\left(\left(\rho C_p\right)_{hnf}/\left(\rho C_p\right)_f\right)} \left(\frac{\partial^2\theta}{\partial X^2} + \frac{\partial^2\theta}{\partial Y^2}\right) \tag{9}$$

$$U\frac{\partial C}{\partial X} + V\frac{\partial C}{\partial Y} = \frac{1}{Le} \left(\frac{\partial^2 C}{\partial X^2} + \frac{\partial^2 C}{\partial Y^2} \right) \tag{10}$$

Where;

$$Ra = \frac{g\beta_f \Delta T L^3}{\nu_f \alpha_f}, \text{Pr} = \frac{v_f}{\alpha_f}, \text{Le} = \frac{\alpha_f}{D}$$
 (11)

Boundary conditions:

Bottom wall:
$$\frac{\partial \theta}{\partial Y} = 0, U = 0, V = 0, \Psi = 0, \frac{\partial C}{\partial Y} = 0$$

Bottom rib:
$$\frac{\partial \theta}{\partial n} = -1, U = 0, V = 0, \Psi = 0, \frac{\partial C}{\partial n} = -1$$

Right and left walls: $\theta = 0, U = 0, V = 0, \Psi = 0, C = 0$

Top wall:
$$\frac{\partial \theta}{\partial Y} = 0, U = 0, V = 0, \Psi = 0, \frac{\partial C}{\partial Y} = 0$$

Top rib:
$$\frac{\partial \theta}{\partial n} = -1, U = 0, V = 0, \Psi = 0, \frac{\partial C}{\partial n} = -1$$

Baffle:
$$\frac{\partial \theta}{\partial u} = -1$$
, $U = 0$, $V = 0$, $\Psi = 0$, $\frac{\partial C}{\partial u} = -1$ (12)

The target physical parameters in this paper are the local Nusselt and Sherwood numbers. These parameters are defined as:

$$Nu = \left(\frac{k_{lnf}}{k_f}\right) \left(\frac{1}{\theta}\right) \tag{13}$$

$$Sh = \left(\frac{1}{C}\right) \tag{14}$$

And the average ones are:

$$Nu_{avg} = \frac{1}{s} \int Nu.ds \tag{15}$$

$$Sh_{avg} = \frac{1}{s} \int Sh.ds \tag{16}$$

Where s is the length of the bottom rib.

K, (ρC_p) , μ , $(\rho \beta)$, and ρ of hybrid nanoliquid could be represented as:

$$\frac{k_{lmf}}{k_f} = \left[(1 - \lambda)\varphi(k_f - k_s) + k_s - (1 - \lambda)k_f \right] \times \left[k_s - (1 - \lambda)k_f + \varphi(k_f - k_s) \right]^{-1}$$

$$(17)$$

$$(\rho C_p)_{hnf} = \varphi_{Cu}(\rho C_p)_{Cu} + \varphi_{Al_2O_3}(\rho C_p)_{Al_2O_3} + (1 - \varphi)(\rho C_p)_f$$
(18)

$$\mu_{hmf} = (1 - \varphi)^{-2.5} \mu_f$$
 (19)

$$(\rho\beta)_{hnf} = (1 - \varphi)(\rho\beta)_f + \varphi_{Cu}(\rho\beta)_{Cu} + \varphi_{Al_2O_3}(\rho\beta)_{Al_2O_3}$$
(20)

$$\rho_{hnf} = (1 - \varphi)\rho_f + \varphi_{Cu}\rho_{Cu} + \varphi_{Al_2O_3}\rho_{Al_2O_3}$$
(21)

And the correlations of entropy generation would be as:

$$S_{gen} = S_{HT} + S_{FF} + S_{Mass} \tag{22}$$

In which

$$S_{HT} = \frac{k_{lnf}}{k_f} \left(\left(\frac{\partial \theta}{\partial X} \right)^2 + \left(\frac{\partial \theta}{\partial Y} \right)^2 \right) \tag{23}$$

1.1

$$S_{FF} = \varphi_1 \frac{\mu_{hnf}}{\mu_f} \left[2 \left(\frac{\partial V}{\partial Y} \right)^2 + 2 \left(\frac{\partial U}{\partial X} \right)^2 + \left(\frac{\partial U}{\partial Y} + \frac{\partial V}{\partial X} \right)^2 \right]$$
 (24)

$$S_{Mass} = \varphi_2 \left(\left(\frac{\partial C}{\partial X} \right)^2 + \left(\frac{\partial C}{\partial Y} \right)^2 \right) + \varphi_3 \left(\frac{\partial C}{\partial X} \frac{\partial \theta}{\partial X} + \frac{\partial C}{\partial Y} \frac{\partial \theta}{\partial Y} \right)$$
 (25)

And the Bejan number is:

$$Be = \frac{S_{HT} + S_{Mass}}{S_{gen}} \tag{26}$$

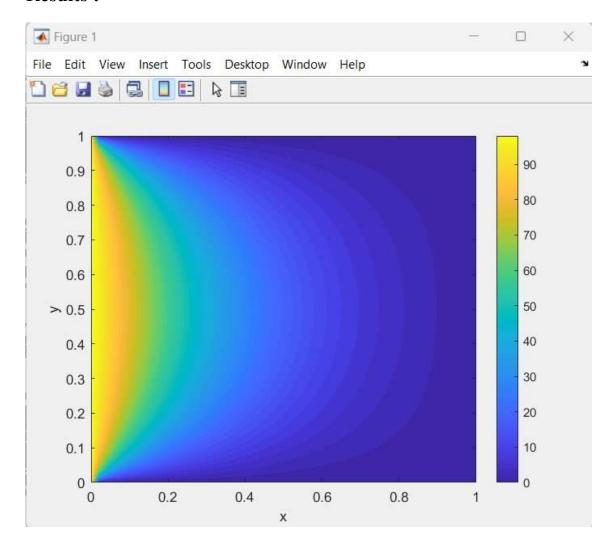
Code

```
title('Temperature Distribution in H-Shaped Cavity');
% Parameters
L = 1; % Length of cavity
W = 1; % Width of cavity
H = 0.1; % Height of cavity
T hot = 100; % Temperature of hot wall
T cold = 0; % Temperature of cold walls
nx = 51; % Number of grid points in x-direction
ny = 51; % Number of grid points in y-direction
dx = L/(nx-1); % Grid spacing in x-direction
dy = W/(ny-1); % Grid spacing in y-direction
max iter = 1000; % Maximum number of iterations
tolerance = 1e-5; % Convergence tolerance
% Initialization
T = ones(nx, ny) * T_cold; % Initialize temperature array
% Main loop (iterate until convergence)
for iter = 1:max_iter
    % Boundary conditions
    T(:,1) = T_cold; % Left cold wall
    T(:,end) = T_cold; % Right cold wall
    T(1,:) = T \text{ hot}; % Hot wall}
    % Compute temperature field using simple averaging
    T_{new} = T;
    for i = 2:nx-1
        for j = 2:ny-1
            % Average of neighboring points
            T_{\text{new}(i,j)} = 0.25*(T(i+1,j) + T(i-1,j) + T(i,j+1) + T(i,j-1));
        end
    end
    % Check for convergence
    if max(abs(T_new(:) - T(:))) < tolerance
        disp(['Converged at iteration ', num2str(iter)]);
        break;
    end
```

```
% Update temperature array
T = T_new;
end

% Plotting the results
[X, Y] = meshgrid(linspace(0,L,nx), linspace(0,W,ny));
contourf(X, Y, T', 50, 'LineColor', 'none');
colorbar;
xlabel('x');
vlabel('v'):
```

Results:-



Case Study 2

Problem Statement:-

To Update the code on the problem on Heat Transfer of Natural Convection in H – Shaped Cavity.

Code:-

```
% Parameters
 L = 1;
                                            % Length of cavity
W = 1;
                                           % Width of cavity
W = 1; % Width of cavity
H = 0.1; % Height of cavity
T_hot = 100; % Temperature of hot wall
T_cold = 0; % Temperature of cold walls
nx = 51; % Number of grid points in x-direction
                                 % Number of grid points in y-direction
 ny = 51;
 dx = L/(nx-1); % Grid spacing in x-direction
 dy = W/(ny-1); % Grid spacing in y-direction
 max_iter = 100; % Maximum number of iterations
 tolerance = 1e-5;% Convergence tolerance
 alpha = 0.1; % Thermal diffusivity
% Initialize temperature array
 T = ones(nx, ny) * T_cold;
 % Set up VideoWriter object
 writerObj = VideoWriter('temperature_evolution.avi'); % Specify the file name
 writerObj.FrameRate = 10; % Set the frame rate
 % Open the video writer object
 open(writerObj);
 % Create a figure for animation
 figure;
 % Main loop (iterate until convergence)
 for iter = 1:max_iter
           % Boundary conditions
                                                                                  % Left cold wall
           T(:,1) = T_{cold};
                                                             % Right cold wall
% Hot wall
           T(:,end) = T_cold;
            T(1,:) = T_hot;
            T(ceil(end/2-round(H/(2*dy))):floor(end/2+round(H/(2*dy))),\ end)\ =\ T\_cold;\ \%\ Middle\ cold\ wall\ Additional to the cold of the cold
           % Compute temperature field using finite difference method
            T_{new} = T;
            for i = 2:nx-1
                      for j = 2:ny-1
                                  % Finite difference equation
```

```
 T_{new(i,j)} = T(i,j) + alpha * ( (T(i+1,j) - 2*T(i,j) + T(i-1,j))/dx^2 + (T(i,j+1) - 2*T(i,j) + T(i,j-1))/dy^2 ); 
     % Check for convergence
if max(abs(T_new(:) - T(:))) < tolerance
    disp(['Converged at iteration ', num2str(iter)]);</pre>
          break;
     end
     % Update temperature array
     T = T_new;
     \% Plot the current temperature distribution in 3D
     [X, Y] = meshgrid(linspace(\theta,L,nx), linspace(\theta,W,ny)); surf(X, Y, T', 'EdgeColor', 'none');
     colorbar;
xlabel('x');
     title(['Iteration: ', num2str(iter)]);
axis([0 L 0 W 0 100]); % Set axis limits for consistency
     % Capture the frame
     frame = getframe(gcf);
     % Write the frame to the video file
     writeVideo(writerObj, frame);
% Close the video writer object
close(writerObj);
```

Results:-

