

# Take Home Final

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December 2, 2021

## 1 (30pts)

A High performance helicopter has a model shown in Figure 1. The goal is to control the pitch angle  $\theta$  of the helicopter by adjusting the rotor thrust  $\delta$ . The equations of motion of the helicopter are

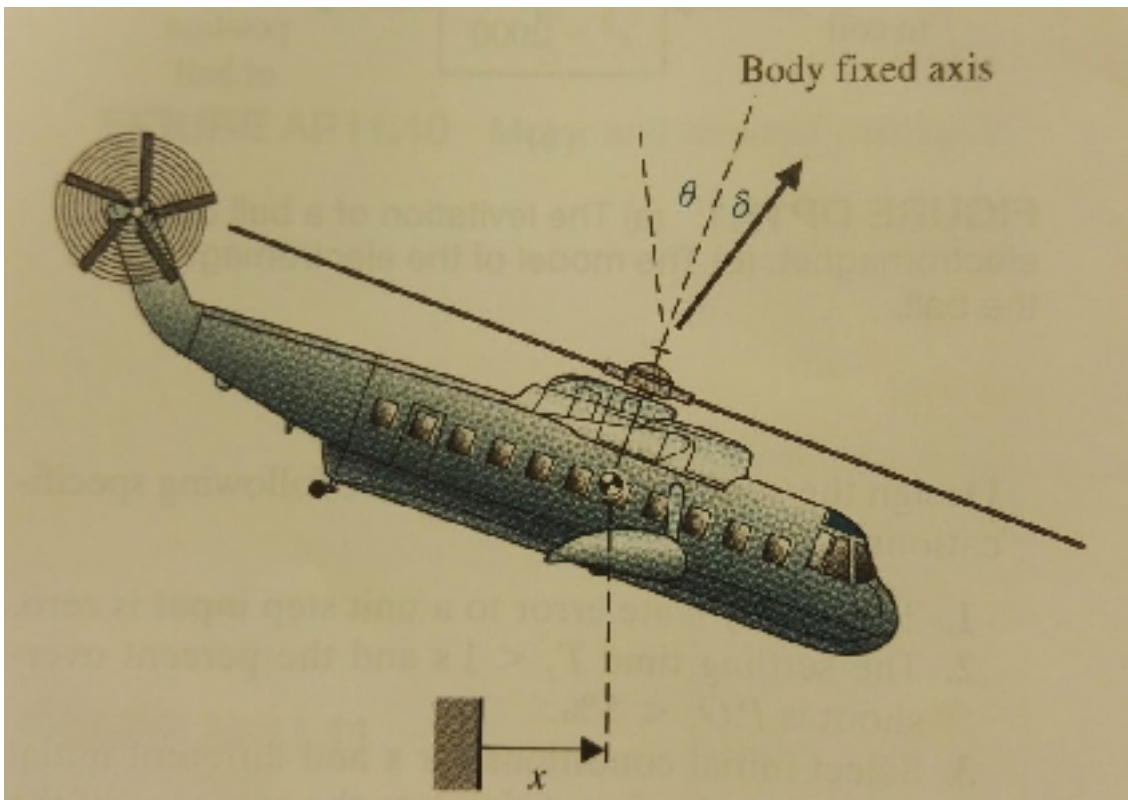


Figure 1:

$$\frac{d^2\theta}{dt^2} = -\sigma_1 \frac{d\theta}{dt} - \alpha \frac{dx}{dt} + n\delta \quad (1)$$

$$\frac{d^2x}{dt^2} = g\theta - \alpha_2 \frac{d\theta}{dt} - \sigma_2 \frac{dx}{dt} + g\delta \quad (2)$$

Where  $x$  is the translation in the horizontal direction. For a military high-performance helicopter we find:  $\sigma_1 = 0.415$ ,  $\sigma_2 = 0.0198$ ,  $\alpha_1 = 0.0111$ ,  $\alpha_2 = 1.43$ ,  $n = 6.27$ ,  $g = 9.8$  all in appropriate SI units. Find:

- (a) A state variable representation of this system
- (b) The transfer function representation for  $\frac{\theta(s)}{\delta(s)}$
- (c) Use state variable feedback to achieve adequate performances for the controlled system. Desired specifications include:
  - (1) A steady-state for an input step command for  $\theta_d(s)$ , the desired pitch angle, less than 20% of the input step magnitude
  - (2) An overshoot for a step input command is less than 20%
  - (3) a settling (with a 2% criterion) time for a step command of less than 1.5 seconds
- (d) If the state variable is not available, design the observer and control law to meet the design specifications included in part (c)

work:

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Final Answer:

## 2 (30pts)

The open loop system

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -1 & 4 & -3 \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} u \quad (3)$$

$$y = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} x \quad (4)$$

$$x(0) = \begin{bmatrix} 1 \\ -1 \\ 1 \end{bmatrix} \quad (5)$$

- 1) Assume that  $x$  is available for state feedback. Design an LQR control law by letting  $R = 1$  and choosing  $Q$  so that all the elements of the feedback gain  $K$  have absolute value less than 50.  
Requirement:  $|y_1(t)| \leq 0.05, |y_2(t)| \leq 0.05$ , for all  $t > 5$ . Plot  $y_1(t)$  and  $y_2(t)$  in the same figure for  $t \in [0, 20]$

- 2) Assume that only the output  $y$  is available. Design an observer so that the poles of the observer are  $-5 \pm j5, -10$ . Choose the observer gain so that all the elements have absolute value less than 80. Form a closed loop system along with the LQR controller in step 1). Plot  $y_1(t)$  and  $y_2(t)$  in the same figure for  $t \in [0, 20]$

work:

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For the system described by:

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -1 & 4 & 5 \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ 2 \end{bmatrix} u \quad (6)$$

$$\dot{x} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \end{bmatrix} u \quad (7)$$

and  $Q = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}, R = [1]$

The following is the outputs of the LQR system assuming the inputs are 0

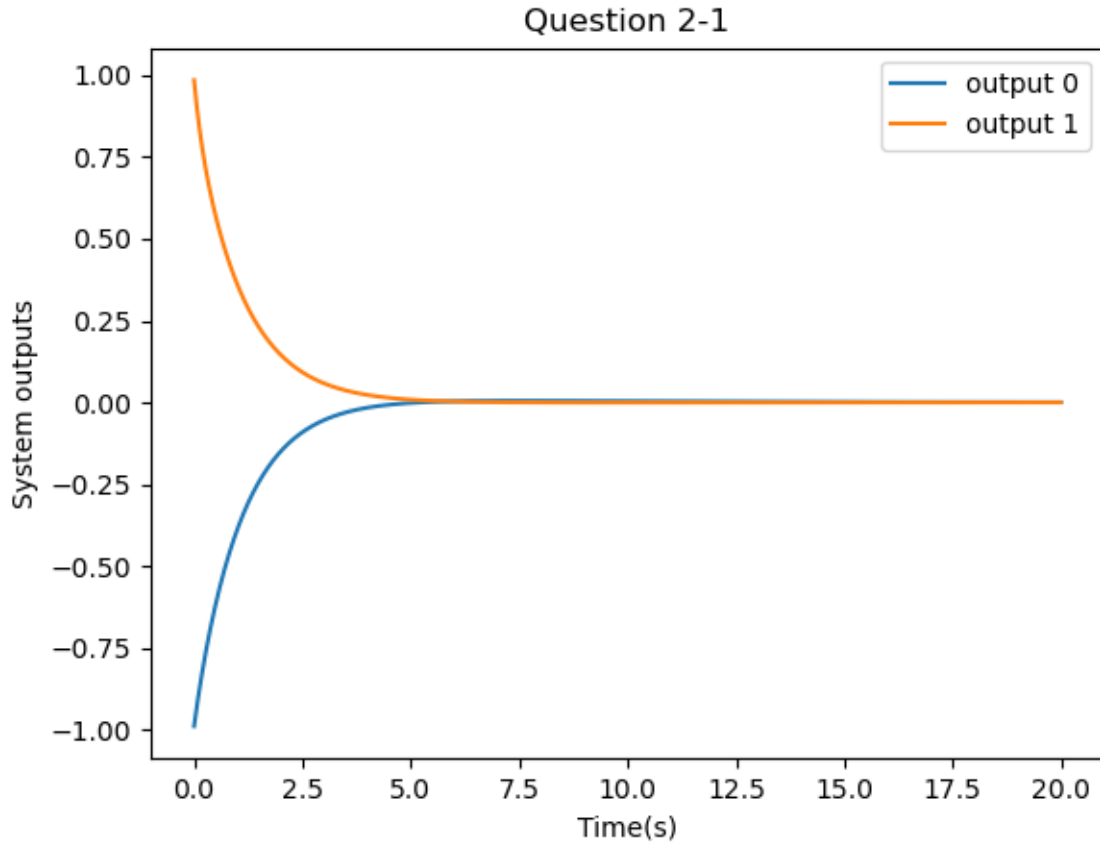


Figure 2: LQR system

- 2) Assume that only the output  $y$  is available. Design an observer so that the poles of the observer are  $-5 \pm j5, -10$ . Choose the observer gain so that all the elements have absolute value less than 80. Form a closed loop system along with the LQR controller in step 1). Plot  $y_1(t)$  and  $y_2(t)$  in the same figure for  $t \in [0, 20]$

Final Answer:

**3 (40pts)**

A cart with an inverted pendulum as seen in Figure 3

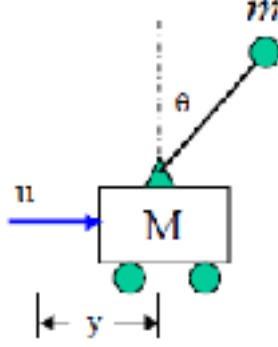


Figure 3:

$u$	control input(Newtons)
$y$	displacement of the cart(meters)
$\theta$	angle of the pendulum(radians)

$$x = \begin{bmatrix} y \\ \dot{y} \\ \theta \\ \dot{\theta} \end{bmatrix} \quad (8)$$

The control problems are

- 1: Stabalization: Design a feedback law  $u = Fx$  such that  $x(t) \rightarrow 0$  for  $x(0)$  close to the diagram
- 2: For  $x(0) = (0, 0, -\pi, 0)$ , apply an impulse force  $u(t) = u_{max}$  for  $t \in [0, 0.1]$  to bring  $\theta$  to a certain range and then switch to the linear controller so that  $x(t) \rightarrow 0$ .

Assume that there is no friction or damping. The nonlinear model is as follows.

$$\begin{bmatrix} M + m & ml \cos(\theta) \\ \cos(\theta) & l \end{bmatrix} \begin{bmatrix} \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} u + ml\dot{\theta}^2 \sin \theta \\ g \sin \theta \end{bmatrix} \quad (9)$$

with

$m = 1kg$	mass of the pendulum
$l = 0.2m$	length of the pendulum
$M = 5kg$	mass of the cart
$g = 9.8 \frac{m}{s^2}$	mass of the cart

Linearize the system at  $x = 0$

$$\begin{bmatrix} M + m & ml \cos(\theta) \\ \cos(\theta) & l \end{bmatrix} \begin{bmatrix} \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} u \\ g\theta \end{bmatrix} \quad (10)$$

the state space description for the linearized system.

$$\dot{x} = Ax + Bu \quad (11)$$

Problems:

1. Find matrices  $A$ ,  $B$  for the state space equation.
2. Design a feedback law  $u = -F_1x$  so that  $A + BF_1$  has eigenvalues as  $-3 \pm j3, -6, -8$ . Build a simulink model for the closed loop linear system. Plot the response under initial condition  $x(0) = (-1.5, 0, 1, 3)$ .

3. Build a simulink model for the original nonlinear system, verify that stabilization is achieved by  $u = F_1 x$  when  $x(0)$  is close to the origin. Find the maximal  $\theta_0$  so that nonlinear system can be stabilized from  $x_0 = (0, 0, \theta_0, 0)$
4. For  $x(0) = (0, 0, \frac{\pi}{5}, 0)$ , compare the response  $y(t)$  and  $\theta(t)$  for the linearized system and the nonlinear system under the same feedback  $u = F_1 x$

work:

1. Find matrices  $A$ ,  $B$  for the state space equation.

$$\dot{x} = \begin{bmatrix} \dot{y} \\ \ddot{y} \\ \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} y \\ \dot{y} \\ \theta \\ \dot{\theta} \end{bmatrix} \quad (12)$$

2. Design a feedback law  $u = F_1 x$  so that  $A + BF_1$  has eigenvalues as  $-3 \pm j3, -6, -8$ . Build a simulink model for the closed loop linear system. Plot the response under initial condition

$$x(0) = \begin{bmatrix} -1.5 \\ 0 \\ 1 \\ 3 \end{bmatrix}.$$

3. Build a simulink model for the original nonlinear system, verify that stabilization is achieved by  $u = F_1 x$  when  $x(0)$  is close to the origin. Find the maximal  $\theta_0$  so that nonlinear system can be stabilized from  $x_0 = \begin{bmatrix} 0 \\ 0 \\ \theta_0 \\ 0 \end{bmatrix}$

4. For  $x(0) = \begin{bmatrix} 0 \\ 0 \\ \frac{\pi}{5} \\ 0 \end{bmatrix}$ , compare the response  $y(t)$  and  $\theta(t)$  for the linearized system and the nonlinear system under the same feedback  $u = F_1 x$

Final Answer: