Mechanical vibration - System identification and modal analysis of 3-DOF linear system

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1 Dynamical system

1.1 The linear model

The chosen model is a linear plant consisting of 3 masses, 3 springs between them, and 3 dampers between each mass and the ground. The model is shown in Figure 1.

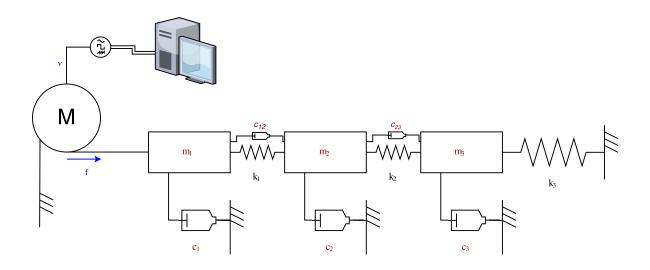


Figure 1: The chosen plant, in red the unknown parameters

1.2 equation of motion

$$\begin{cases}
m_1\ddot{x}_1 = +k_1(x_2 - x_1) + c_{12}(\dot{x}_2 - \dot{x}_1) - c_1\dot{x}_1 + k_v v \\
m_2\ddot{x}_2 = +k_1(x_1 - x_2) + k_2(x_3 - x_2) + c_{12}(\dot{x}_1 - \dot{x}_2) + c_{23}(\dot{x}_3 - \dot{x}_2) - c_2\dot{x}_2 \\
m_3\ddot{x}_3 = +k_2(x_2 - x_3) + c_{23}(\dot{x}_2 - \dot{x}_3) - c_3\dot{x}_3 - k_3x_3
\end{cases} \tag{1}$$

In the classical matrix form:

$$\mathbf{M}\ddot{x} + \mathbf{C}\dot{x} + K\dot{x} = [f \ 0 \ 0]^{\mathsf{T}} \tag{2}$$

where:

$$K = cacca$$
 (3a) $C = dampers$ (3b) $M = \begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix}$ (3c)

1.3 experimental setup

1.4 parameters and data available

1.5 initial hypothesis and approximations

• rectilinear motion (all perfect aligned)

• inerta and damping of the motor are merged respectively into m_1 and c_1 .

$$\begin{cases}
m_1 = m_{block} + \frac{J_{motor}|_{zz}}{r^2} \\
c_2 = c_{block} + \frac{c_{motor}}{r^2}
\end{cases}$$
(4)

where r is the radius of the gear-rack coupling (gear wheel), $J_{motor}|_{zz}$ is the inertia of the motor , c_{motor} the rotational damping and "block" quantities are the ones stricly related to the physical first mass.

2 System identification

2.1 step response analysis

First of all, the step response analysis can be performed. In this analysis the "static" coefficients can be estimated, they are:

- ullet voltage to force k_v
- ullet springs' stiffness k_i with $i\in 1,2,3$

The coefficient to be estimated is the voltage-to-force coefficient

$$f = (k_a \cdot k_t \cdot k_{mp})v = k_v v \tag{5}$$

2.2 voltage-to-force coefficient estimation

2.3 Parameters estimation

$$ciao$$
 (6a)