

Experimental Methods

Lecture 9

October 8th, 2020

Invertible operator, the role of input and output, and free evolution 1/3

- Some linear systems are described by operators that have an inverse:

$$o(t) = \mathfrak{I}[i(t)] \quad i(t) = \mathfrak{I}^{-1}[o(t)]$$

- That is, the equation has just one solution

$$o(t) = \mathfrak{I}[i(t)]$$

- For these systems the role of input and output can be interchanged

- Example 1: multiplication by a constant

$$o(t) = ci(t) \quad i(t) = \frac{1}{c}o(t)$$

- Example 2: Fourier transform

$$o(\omega) = \int_{-\infty}^{\infty} i(t) e^{-i\omega t} dt \quad i(t) = \frac{1}{2\pi} \int_{-\infty}^{\infty} o(\omega) e^{i\omega t} d\omega$$

Invertible operator, the role of input and output, and free evolution 2/3

- Some linear systems (e.g. those described by differential equations) are better described by

$$D[o(t)] = i(t)$$

- For these system there may be n solutions:

$$D[o_j(t)] = 0$$

- Because the operator is linear

$$D\left[o(t) + \sum_{j=1}^n c_j o_j(t)\right] = D[o(t)] + \sum_{j=1}^n c_j D[o_j(t)] = i(t)$$

- (c_j arbitrary constants)

Invertible operator, the role of input and output, and free evolution 3/3

$$D\left[o(t) + \sum_{j=1}^n c_j o_j(t)\right] = D[o(t)] + \sum_{j=1}^n c_j D[o_j(t)] = i(t)$$

- For all functions such that

$$\tilde{o}(t) \neq \sum_{j=1}^n c_j o_j(t)$$

- $D[\tilde{o}(t)] = i(t)$ has an inverse

$$\tilde{o}(t) = D^{-1}[i(t)] = \int_{-\infty}^{\infty} h(t, t') i(t') dt'$$

- Finally

$$o(t) = \underbrace{\sum_{j=1}^n c_j o_j(t)}_{\text{Free evolution: output without an input}} + \int_{-\infty}^{\infty} h(t, t') i(t') dt'$$

Free evolution: output without an input

An example: the pendulum

- Equation of motion for the x coordinate

$$m\ddot{x} = -mg\sin(\theta)\cos(\theta) + F_{\text{ex}}$$

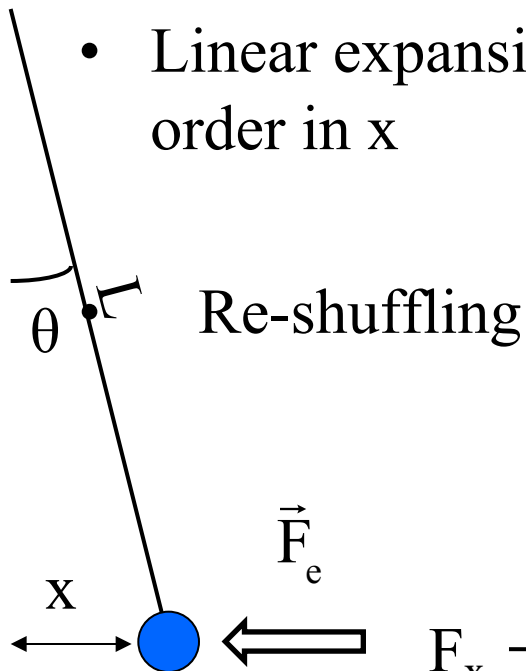
- Converting angles

$$m\ddot{x} = -mg\frac{x}{L}\sqrt{1 - \left(\frac{x}{L}\right)^2} + F_{\text{ex}}$$

- Linear expansion to first order in x

$$m\ddot{x} = -mg\frac{x}{L} + F_{\text{ex}}$$

$$\ddot{x} + \frac{g}{L}x \equiv \ddot{x} + \omega_0^2 x = F_{\text{ex}}/m$$



A basic example: the pendulum

- Linearized equation for the x coordinate $\ddot{x} + \omega_0^2 x = F_{\text{ex}}/m$
 - Special solution $\tilde{x}(t) = \frac{1}{m \omega_0} \int_0^\infty \text{Sin}(\omega_0 t') F_{\text{ex}}(t - t') dt'$
 - Indeed $\dot{\tilde{x}}(t) = -\frac{1}{m \omega_0} \int_0^\infty \text{Sin}(\omega_0 t') \frac{dF_{\text{ex}}(t-t')}{dt'} dt' =$

$$= \frac{1}{m} \int_0^\infty \text{Cos}(\omega_0 t') F_{\text{ex}}(t - t') dt'$$
 - and $\ddot{\tilde{x}}(t) = -\frac{1}{m} \int_0^\infty \text{Cos}(\omega_0 t') \frac{dF_{\text{ex}}(t-t')}{dt'} dt' =$

$$= -\frac{1}{m} \text{Cos}(\omega_0 t') F_{\text{ex}}(t - t') \Big|_0^\infty - \frac{\omega_0}{m} \int_0^\infty \text{Sin}(\omega_0 t') F_{\text{ex}}(t - t') dt' =$$

$$= \frac{F_{\text{ex}}(t)}{m} - \omega_0^2 \tilde{x}(t)$$
 - then:
- $$\ddot{\tilde{x}}(t) + \omega_0^2 \tilde{x}(t) = \frac{F_{\text{ex}}(t)}{m} - \omega_0^2 \tilde{x}(t) + \omega_0^2 \tilde{x}(t) = \frac{F_{\text{ex}}(t)}{m}$$

A basic example: the pendulum

- Response to input: $\tilde{x}(t) = \frac{1}{m \omega_o} \int_0^\infty \text{Sin}(\omega_o t') F_{\text{ex}}(t - t') dt'$
- In addition, for arbitrary values of x_p and x_q :

$$x(t) = x_p \text{Sin}(\omega_o t) + x_q \text{Cos}(\omega_o t) \rightarrow \ddot{x}(t) + \omega_o^2 x(t) = 0$$
- The most general solution is then:

$$x(t) = \underbrace{x_p \text{Sin}(\omega_o t) + x_q \text{Cos}(\omega_o t)}_{\text{Free evolution}} + \underbrace{\frac{1}{m \omega_o} \int_0^\infty \text{Sin}(\omega_o t') F_{\text{ex}}(t - t') dt'}_{\text{Response}}$$
- Note: for an undamped pendulum, the free evolution never decays.
- Passive undamped pendulums do not exist in nature.

Inverting the relation to obtain the input

- Notice that the equation of motion $D[o(t)] = i(t)$ allows for calculating the value of $i(t)$ from $o(t)$ even if this one includes some free evolution
- For the pendulum for instance by calculating $m\ddot{x}(t)$ at time t and adding up the value of $mgx(t)/L$ at the same time, one obtains $F_{ex}(t) = m\ddot{x}(t) + mgx(t)/L$, no matter what's the value of the free evolution.
- A practical example with a torsion pendulum

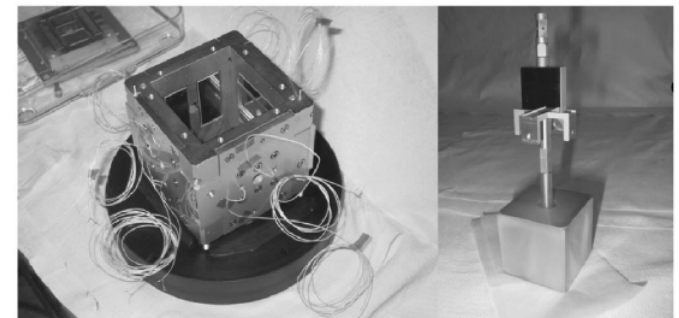
B. EVALUATION OF TORQUE AND BACKGROUND SUBTRACTION

The angular motion of the pendulum $\phi(t)$ is converted into an instantaneous applied torque $N(t)$ as

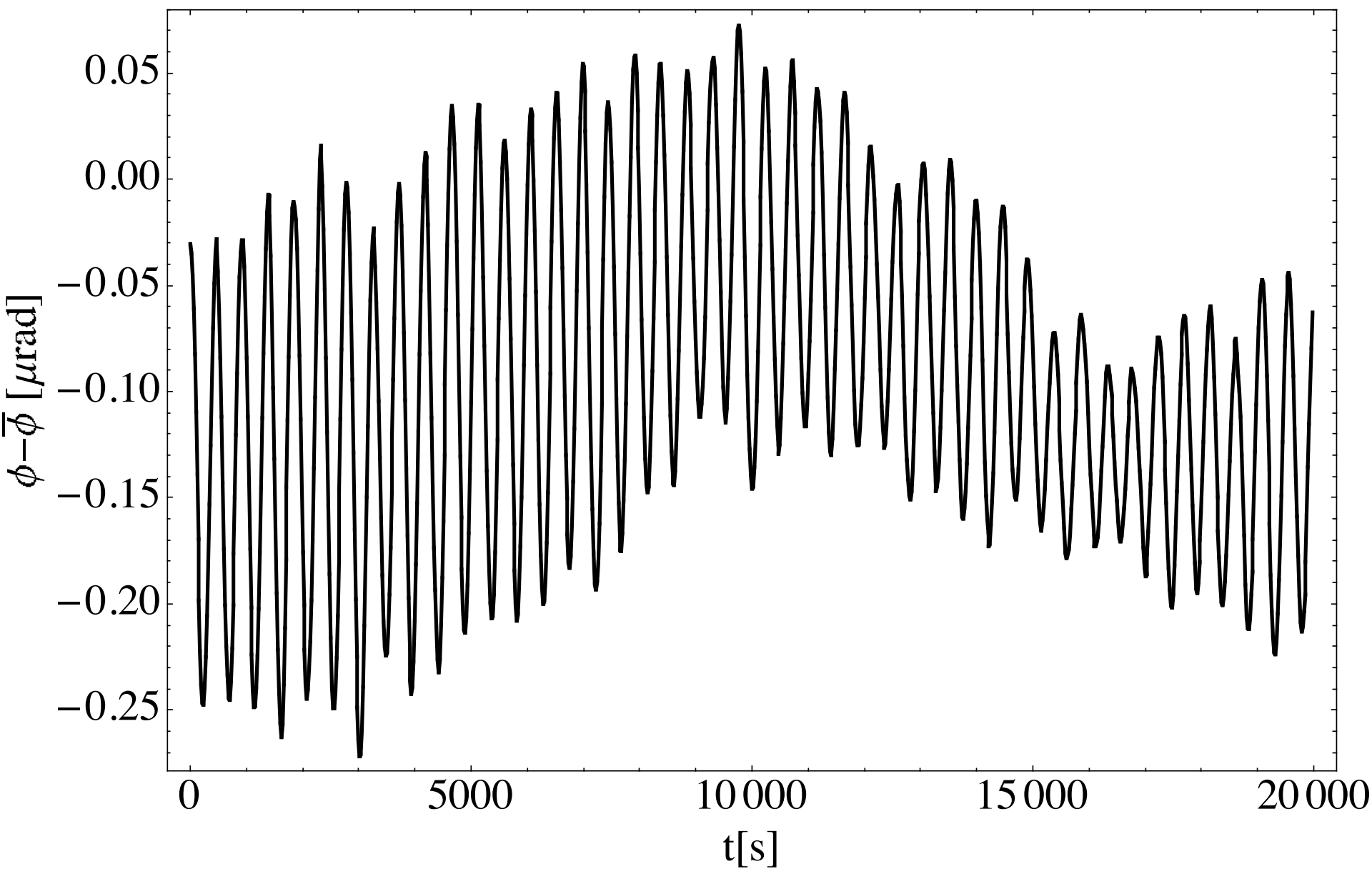
$$N(t) = I_o \{ \ddot{\phi}(t) + (2\pi/T_o Q) \dot{\phi}(t) + (2\pi/T_o)^2 \phi(t) \}. \quad (1)$$

The derivatives are estimated from a sliding second order fit to 5 adjoining data.

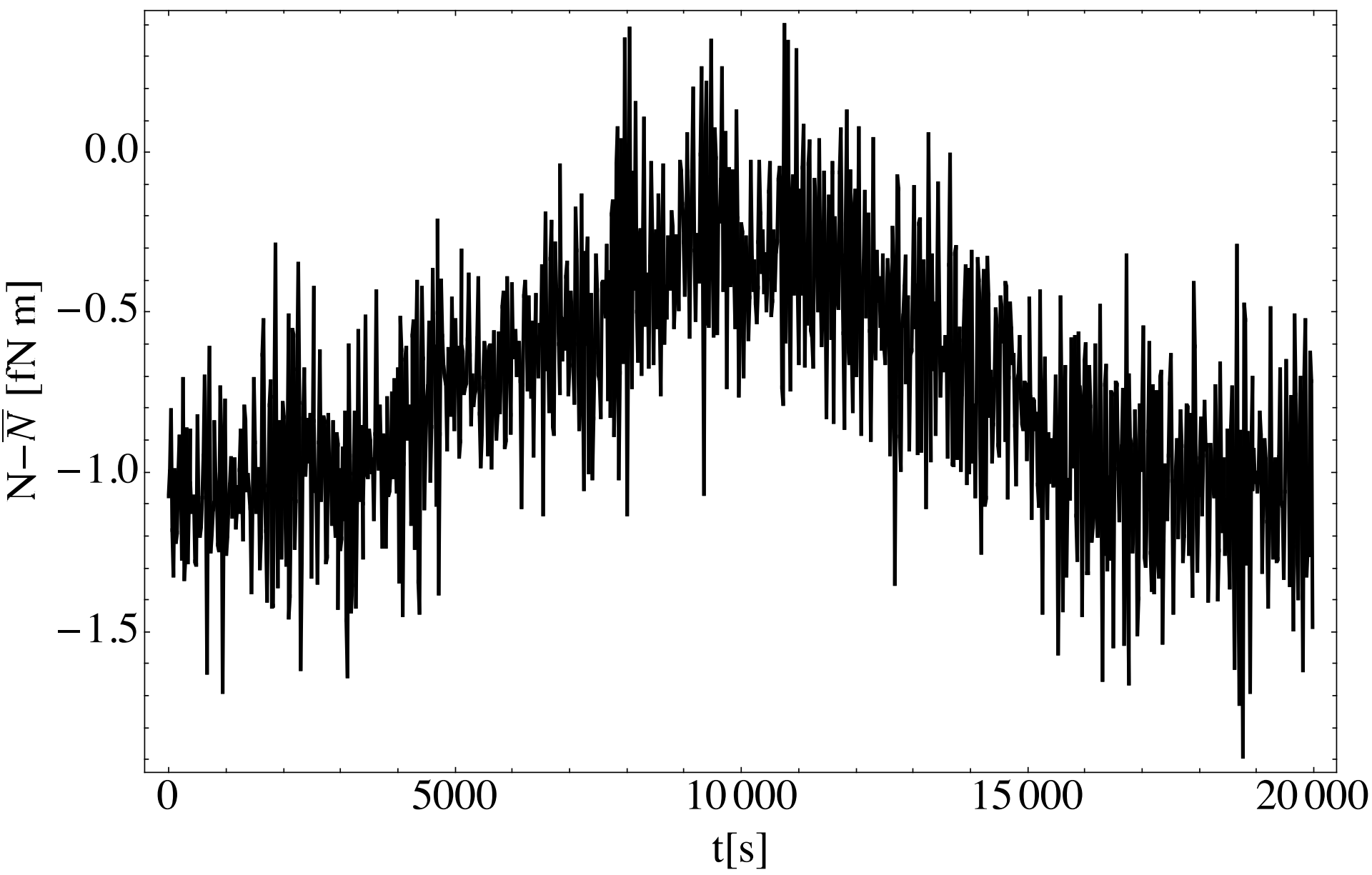
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Angular data



Torque data



Step response

- From
$$o(t) = \int_{-\infty}^{\infty} h(t')i(t-t')dt'$$

- We derived that $h(t)$ is the impulse response

$$o(t) = \int_{-\infty}^{\infty} h(t')\delta(t-t')dt' = h(t)$$

- Consider now the *step response* defined as

$$o_{-1}(t) = \int_{-\infty}^t h(t')\Theta(t-t')dt' = \int_{-\infty}^t h(t')dt'$$

- We can derive o_{-1}

$$do_{-1}(t)/dt = h(t)$$

- Thus $h(t)$ is both the impulse response and the derivative of the step response

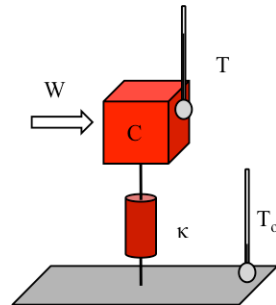
The importance of the step response

- As delta impulse cannot be obtained in reality, $h(t)$ cannot be measured as the response of the system to an impulse.
- High but finite amplitude pulses, easily drive the system out of linearity
- On the contrary steps can be made very small and fed to the system under study.
- The derivative of the system response is proportional to h
- Example: the calorimeter. Response to a step at $t=0$

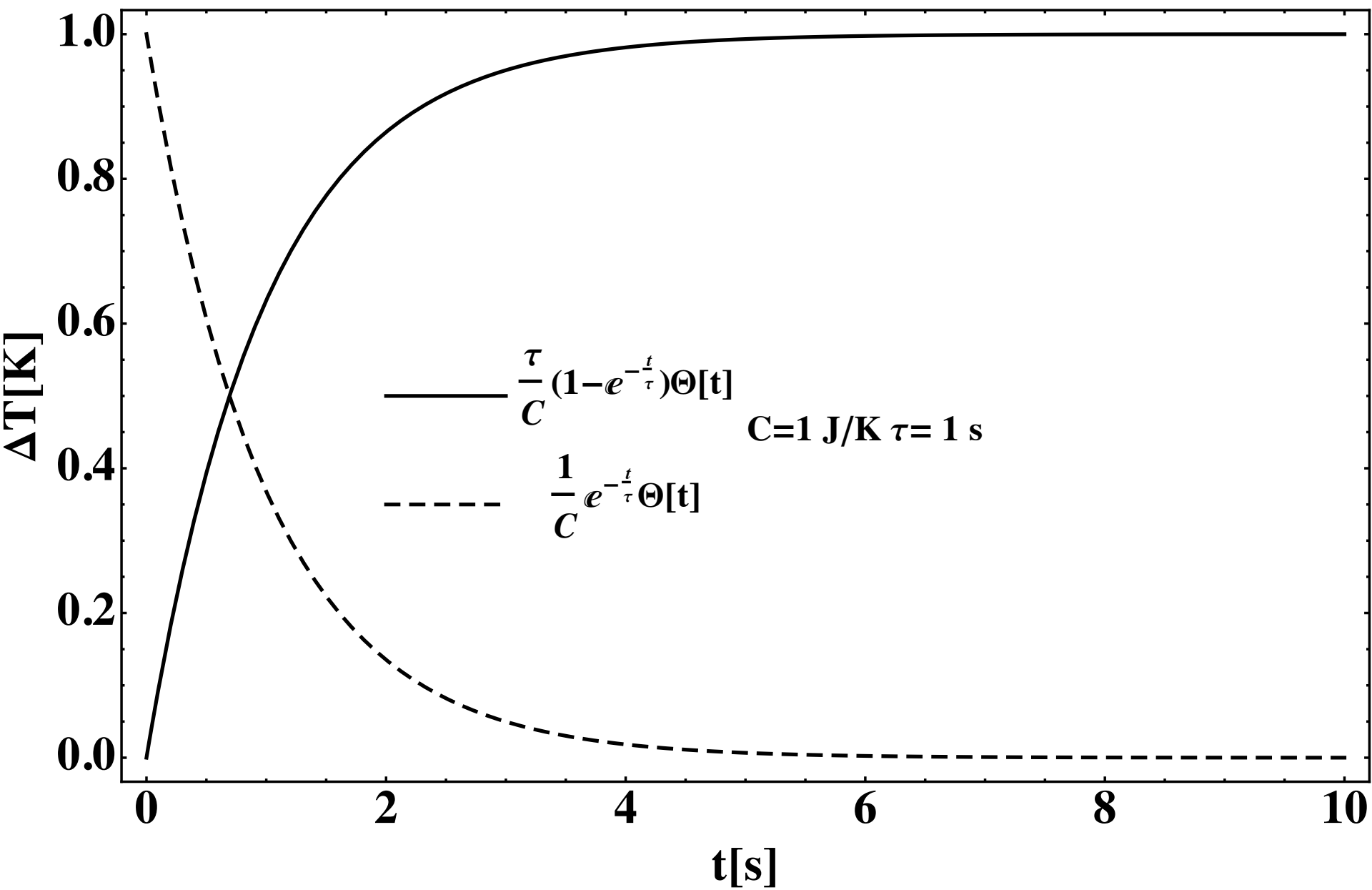
$$\Delta T(t) = (W_o / \kappa) \left(1 - e^{-\frac{t}{\tau}} \right) \Theta(t)$$

- Derivative

$$d\Delta T(t)/dt = (W_o / \tau \kappa) e^{-\frac{t}{\tau}} \Theta(t) = (W_o / C) e^{-\frac{t}{\tau}} \Theta(t) = W_o h(t)$$



- Thus $h(t)$ can be measured with W_o as small as the measurement noise allows for.



Summary: linear stationary systems, continuous

- A linear stationary system $o(t) = \int_{-\infty}^{\infty} i(t')h(t-t')dt' = \int_{-\infty}^{\infty} h(t')i(t-t')dt'$

- A linear, stationary, and causal system

$$o(t) = \int_0^{\infty} h(t')i(t-t')dt'$$

- A linear, stationary system with free evolution (not all systems have free evolution)

$$o(t) = \sum_{j=1}^n c_j o_j(t) + \int_{-\infty}^{\infty} h(t')i(t-t')dt'$$

Linear response and Fourier Transforms

- Fourier Transforms convert input-output relations into algebraic equations
- Within linear response, frequencies don't mix!

Linear system in the frequency domain

- Output is the convolution between input and impulse response

$$o(t) = \int_{-\infty}^{\infty} h(t') i(t - t') dt'$$

- Using convolution theorem

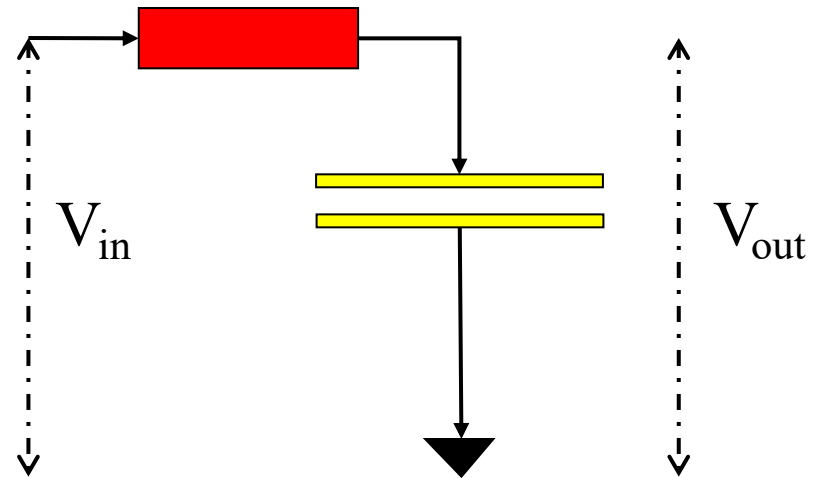
$$o(\omega) = h(\omega) i(\omega)$$

- $h(t)$: impulse response
- $h(\omega)$: *frequency response*

Linear systems do not mix frequencies

A remarkable example: differential equations

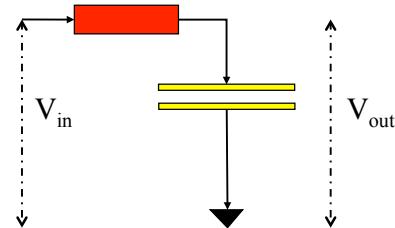
- Consider a low pass electrical circuit
- The circuit equation
$$C dV_{\text{out}}(t)/dt = [V_{\text{in}}(t) - V_{\text{out}}(t)]/R$$
- That is (define $\tau = RC$)
$$dV_{\text{out}}(t)/dt + V_{\text{out}}(t)/\tau = V_{\text{in}}(t)/\tau$$
- Take the Fourier Transform
$$i\omega V_{\text{out}}(\omega) + V_{\text{out}}(\omega)/\tau = V_{\text{in}}(\omega)/\tau$$
- That is $V_{\text{out}}(\omega) = V_{\text{in}}(\omega)/(1 + i\omega\tau)$
- Or
$$h(\omega) = 1/(1 + i\omega\tau)$$



A remarkable example: differential equations

- Input output relation in the frequency domain

$$V_{\text{out}}(\omega) = V_{\text{in}}(\omega) / (1 + i\omega\tau) = h(\omega) V_{\text{in}}(\omega)$$



- From convolution theorem (\mathcal{F}^{-1} = inverse Fourier transform)

$$V_{\text{out}}(t) = \int_0^{\infty} \mathcal{F}^{-1}\{h(\omega)\}(t') V_{\text{in}}(t - t') dt'$$

- Let MathematicaTM calculate the inverse Fourier transform

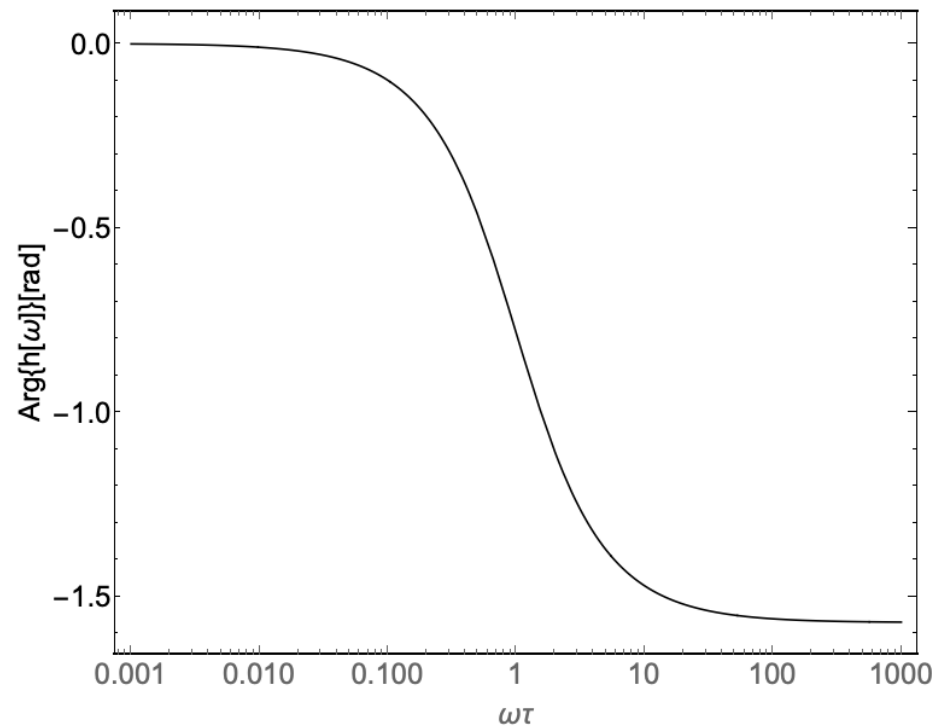
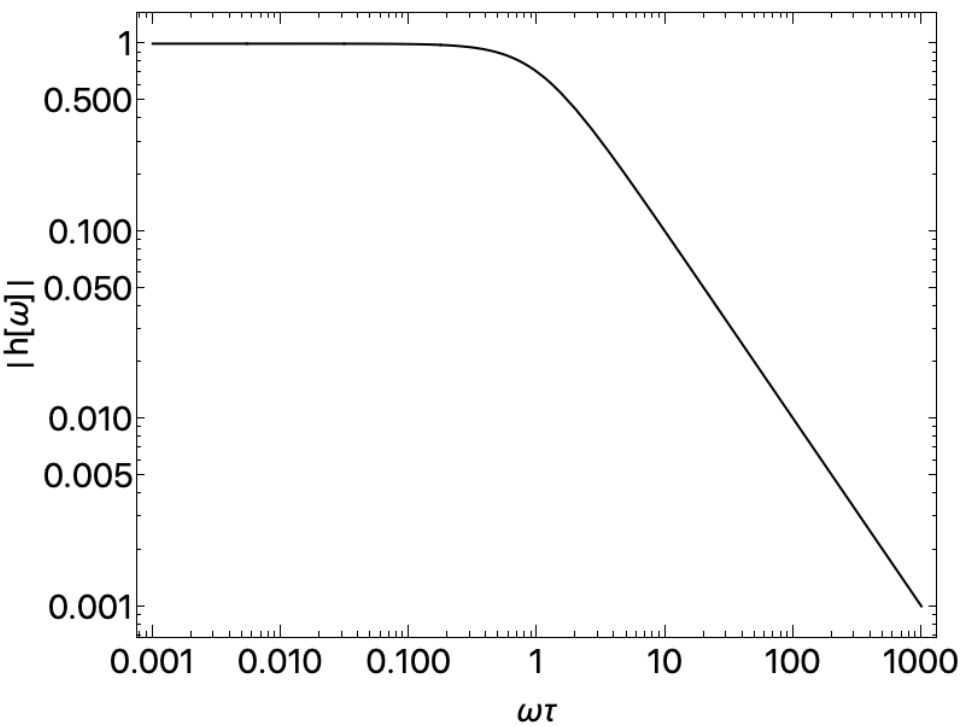
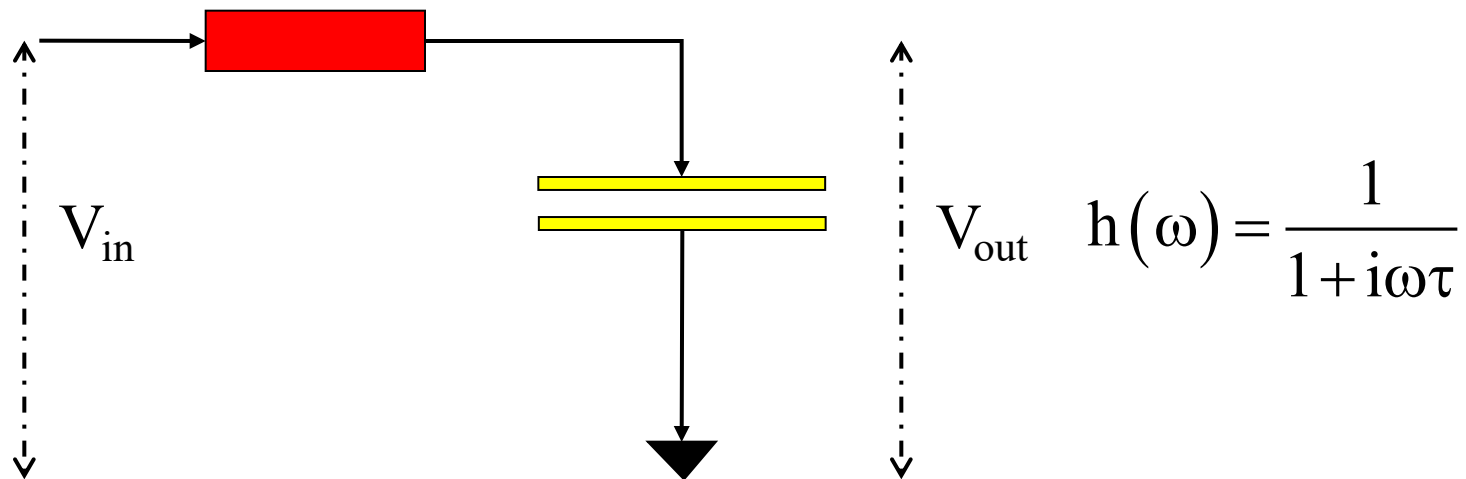
$$\text{Simplify}\left[\text{InverseFourierTransform}\left[\frac{1}{1 + i \omega \tau}, \omega, t\right], \tau > 0\right]$$

$$\frac{e^{-\frac{t}{\tau}} \text{HeavisideTheta}[t]}{\tau}$$

- Then finally

$$V_{\text{out}}(t) = \int_0^{\infty} \frac{1}{\tau} e^{-\frac{t'}{\tau}} \theta(t') V_{\text{in}}(t - t') dt'$$

A remarkable example: differential equations



Frequency response and sinusoidal signals

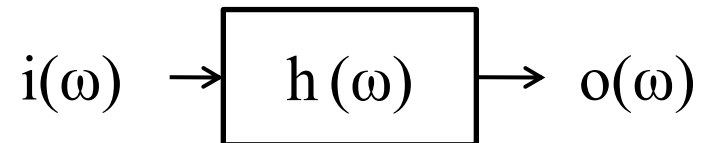
- Take an (ever lasting) sinusoid

$$i(t) = \sin(\omega_0 t)$$

- Its Fourier Transform is

$$i(\omega) = \frac{\pi}{j} [\delta(\omega - \omega_0) - \delta(\omega + \omega_0)]$$

- Let's feed it to a system



- The output is

$$o(\omega) = \frac{\pi}{j} [h(\omega_0) \delta(\omega - \omega_0) - h(-\omega_0) \delta(\omega + \omega_0)]$$

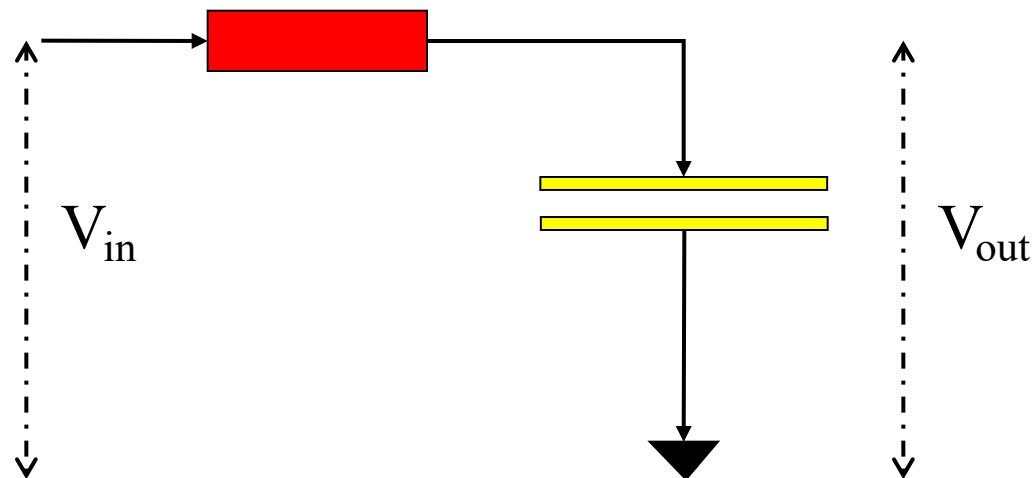
- Using symmetry of h

$$= \frac{\pi}{j} h'(\omega_0) [\delta(\omega - \omega_0) - \delta(\omega + \omega_0)] + \pi h''(\omega_0) [\delta(\omega - \omega_0) + \delta(\omega + \omega_0)]$$

- In the time domain $o(t) = h'(\omega_0) \sin(\omega_0 t) + h''(\omega_0) \cos(\omega_0 t)$

$$= |h(\omega_0)| \sin(\omega_0 t + \text{Arg}\{h(\omega_0)\})$$

The low pass



- The frequency response function

$$h(\omega) = \frac{1}{1 + i\omega\tau}$$

- $\sin(\omega_o t)$ at input

$$o(\omega) = \frac{\pi}{i} \left[\frac{\delta(\omega - \omega_o)}{1 + i\omega_o\tau} - \frac{\delta(\omega + \omega_o)}{1 - i\omega_o\tau} \right]$$

- In the time domain

$$o(t) = \frac{\sin(\omega_o t - \text{ArcTan}[\omega_o\tau])}{\sqrt{1 + \omega_o^2\tau^2}}$$

Measuring the transfer function of a system

- Feed a system with an input $i(t)$ which is wideband enough for your purposes. Measure the response $o(t)$. The transfer function is

$$h(\omega) = o(\omega)/i(\omega) = |o(\omega)|/|i(\omega)| e^{i\{\text{Arg}[o(\omega)] - \text{Arg}[i(\omega)]\}}$$

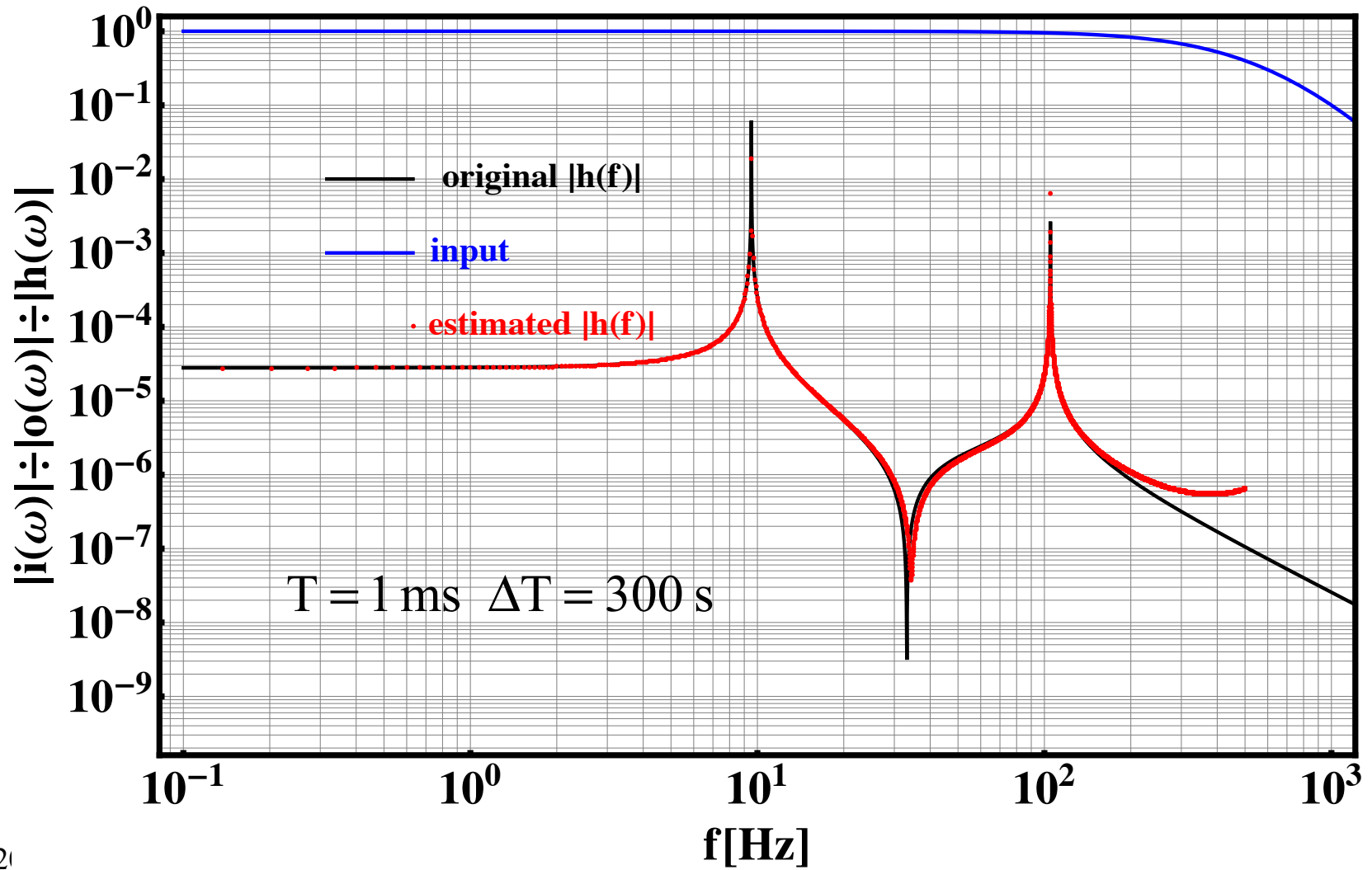
- Notice

$$h(\omega) = o(\omega)/i(\omega)$$

- Holds if $i(\omega)$ and $o(\omega)$ are the true continuous Fourier transform of the continuous signal
- Use of DFT is allowed if:
 - Sampling of $o(t)$ and $i(t)$ is fast enough to produce negligible aliasing
 - Both $i(t)$ and $o(t)$ die out at $t \rightarrow 0$ and $t \rightarrow \Delta T$ to make circular convolution equivalent to standard convolution

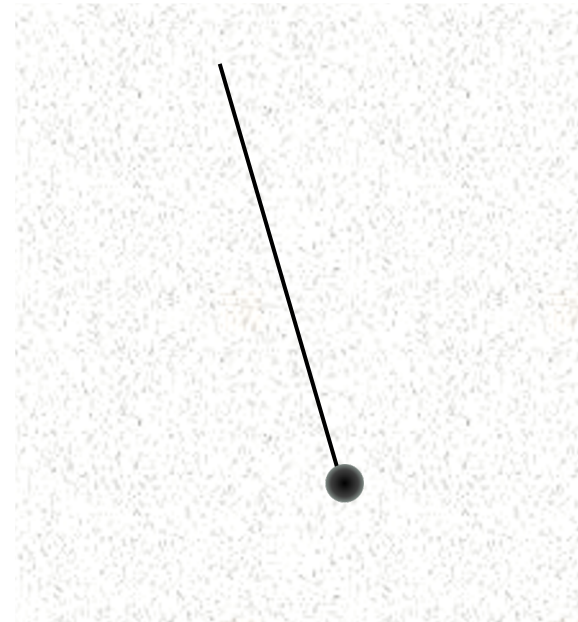
Numerical example

- Input $i(\omega) = \left[1 + \left(\omega / 2\pi \text{ 1 kHz} \right)^2 \right]^{-1} \left[1 + \left(\omega / 2\pi \text{ 500 Hz} \right)^2 \right]^{-1}$
- Frequency response
$$\frac{2.17 \times 10^7 + 62.8i f - 1.97 \times 10^4 f^2}{7.79 \times 10^{11} + 2.04 \times 10^7 i f - 8.65 \times 10^9 f^2 - 2.11 \times 10^3 i f^3 + 7.79 \times 10^5 f^4}$$



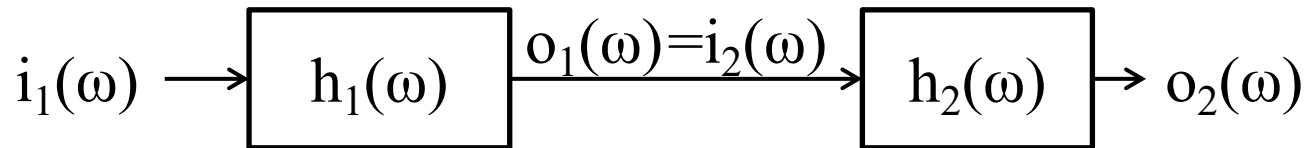
Exercise: the pendulum

- Take the standard, *viscously damped* pendulum
- Consider torque as input and angle as output
- Write the input-output relation
- Linearize for small angles
- Find the frequency response
- Find the impulse response
- Find the free evolution
- Calculate the response to a Gaussian pulse



Linear systems in series

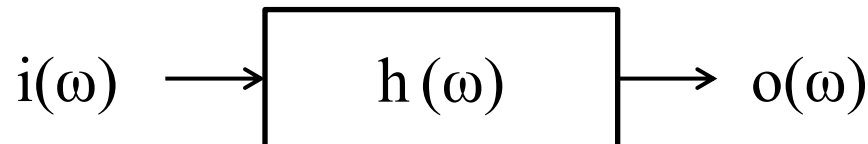
- Two systems in series: the output of the first system is the input to the second one



- It follows that

$$o_2(\omega) = h_2(\omega)i_2(\omega) = h_2(\omega)o_1(\omega) = h_2(\omega)h_1(\omega)i_1(\omega)$$

- Thus the system series is equivalent to

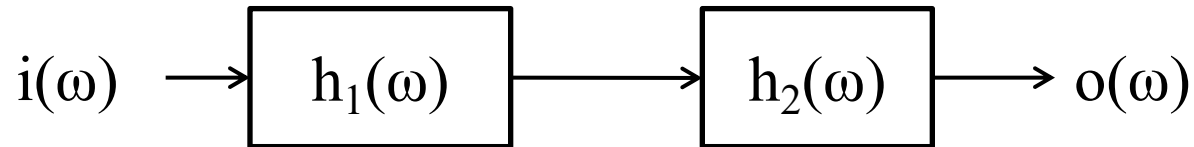


- with

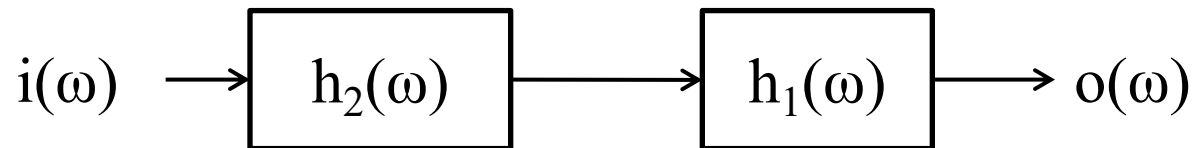
$$h(\omega) = h_1(\omega)h_2(\omega)$$

Linear systems in series

- Notice



- Is equivalent to



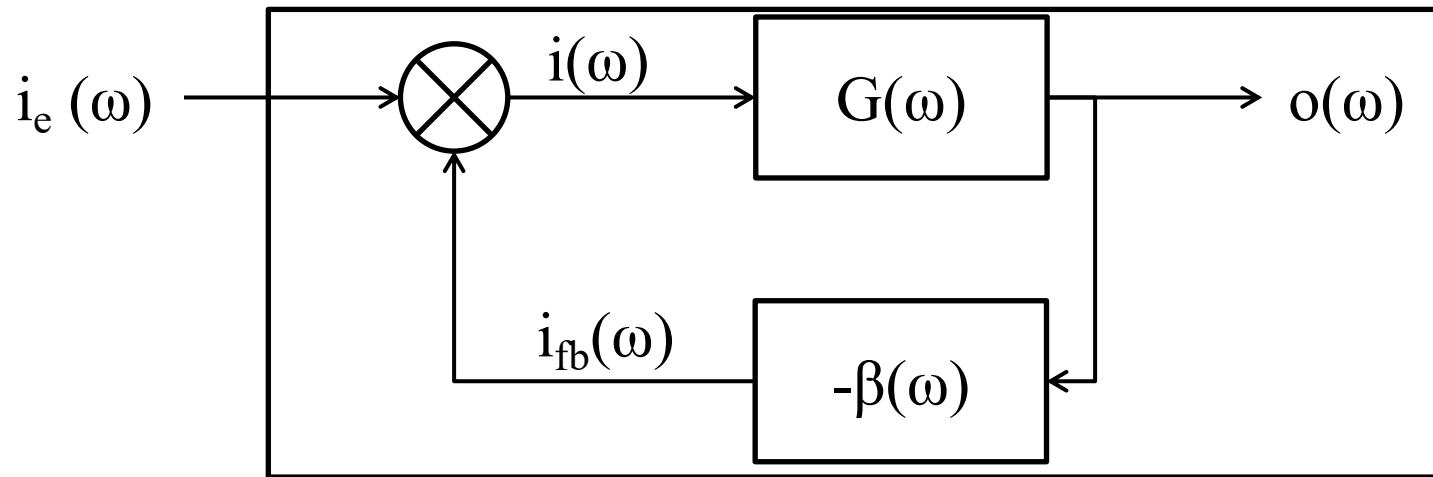
- As product is commutative

$$h_2(\omega)h_1(\omega) = h_1(\omega)h_2(\omega)$$

- Thus the order of the systems within the series is irrelevant
- Warning: physical interchanging two systems may alter their respective frequency response

A remarkable example: the feedback loop

- The output of a system is fed back and added to external input via another linear system:



- Deriving the input output relations of the full system $i_e \rightarrow o$

$$o(\omega) = G(\omega)i(\omega) = G(\omega)[i_e(\omega) + i_{fb}(\omega)]$$

- But:

$$i_{fb}(\omega) = -\beta(\omega)o(\omega)$$

- Thus

$$o(\omega) = G(\omega)[i_e(\omega) - \beta(\omega)o(\omega)]$$