

# Control LII

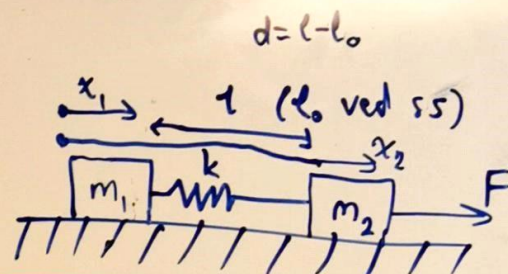
## 1. Derive State Space Model

F: input

$y = \dot{x}_1$ : output

$\dot{x}_2 = v_2$ ,  $v_1, v_2$

$$E_s = \frac{1}{2} k d^2, F_{s1} = k d, F_{s2} = -k d$$



$$\dot{x}_1 = v_1, \dot{x}_2 = v_2$$

$$\dot{v}_2 = \frac{1}{m_2} F - \frac{k}{m_2} x_2 + \frac{k}{m_2} x_1$$

$$\dot{v}_1 = \frac{k}{m_1} x_2 - \frac{k}{m_1} x_1$$

## 2) Eigenvalues (Proof by Matlab™)

$$|A - \lambda I| = 0;$$

$$\lambda_1 = 0, \lambda_2 = 0, \lambda_3 = \frac{\sqrt{-m_2 m_1 k (m_1 + m_2)}}{m_2 m_1}, \lambda_4 = -\lambda_3$$

if  $m_1 = m_2 = k = 1$

then  $\lambda_1 = 0, \lambda_2 = 0, \lambda_3 = \sqrt{2}j, \lambda_4 = -\sqrt{2}j$

$\lambda_1, \lambda_2$  ss response;  $\lambda_3, \lambda_4$  swingy response no damping

## 3) Transfer Function:

$$G(s) = C(sI - A)^{-1} B + D$$

$$\begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{v}_1 \\ \dot{v}_2 \end{pmatrix} = \begin{pmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{k}{m_1} & \frac{k}{m_1} & 0 & 0 \\ \frac{k}{m_2} & -\frac{k}{m_2} & 0 & 0 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ v_1 \\ v_2 \end{pmatrix} + \begin{pmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{m_2} \end{pmatrix} (F)$$

$$G(s) = \begin{pmatrix} 0 & 0 & 1 & 0 \end{pmatrix} \begin{pmatrix} s & 0 & -1 & 0 \\ 0 & s & 0 & -1 \\ \frac{1}{m_1} & -\frac{1}{m_1} & 0 & 0 \\ -\frac{1}{m_2} & \frac{1}{m_2} & 0 & s \end{pmatrix}^{-1} \begin{pmatrix} 0 \\ 0 \\ 0 \\ \frac{1}{m_2} \end{pmatrix} + 0$$

MATLAB™ take the wheel!

$$G(s) = \frac{k}{m_1 m_2 s^3 + k m_1 s + k m_2 s}$$

$$y = \begin{pmatrix} 0 & 0 & 1 & 0 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ v_1 \\ v_2 \end{pmatrix} + \begin{pmatrix} 0 \end{pmatrix} (F)$$