

Problem 1) Design economical built up column to carry 1100 KN using two channels back to back. The length of column is 5 mt. with both ends are held in position and only one end is restrained against rotation. Design a suitable **Lacing system** ? Take $f_y = 250 \text{ Mpa}$

Answer : Given ;

a) Length of member (clear) = 5.00 mt = $5.00 \times 10^3 \text{ mm}$

Given case is **One End Fix And Other Is Hinge (case 02)**

$$\therefore l_{eff} = KL = 0.8 \times 5 \times 10^3 = 4.00 \times 10^3 \text{ mm}$$

..... IS 800-2007; pg. 45

Let. λ (slenderness ratio) = 70

For $f_y=250$; (compressive stress) $\sigma_{ac} = 152 \text{ N/mm}^2$

..... IS 800-2007; pg. 42

Area Req'd. For the strut (A_{Req}) = $\frac{1100 \times 10^3}{152} = 7236.84 \text{ mm}^2$

$$\text{Area of Each angle} = \frac{7236.84}{2} = 3618.42 \text{ mm}^2$$

Refer the steel table and select rolled steel channel ISLC 350

@ $38.8 \frac{\text{kg}}{\text{m}}$

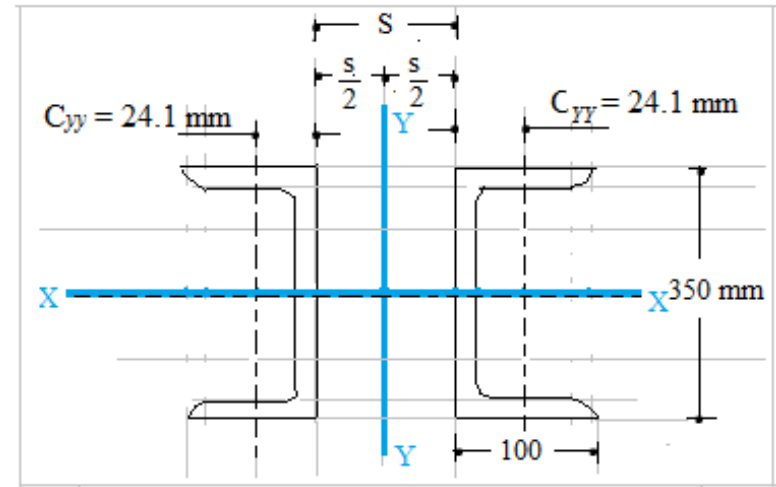
For this channel ; $A = 4947 \text{ mm}^2$; width (b) = 100 mm ;

$t_f = 12.5 \text{ mm}$; $t_w = 7.4 \text{ mm}$; **Center of gravity** ; $C_{yy} = 24.1 \text{ mm}$

; guage distance (g^*) = $g = 60 \text{ mm}$; $r_{xx} = 137.2 \text{ mm}$

$$I_{xx} = 9312.6 \times 10^4 \text{ mm}^4, I_{yy} = 394.6 \times 10^4 \text{ mm}^4;$$

$$r_{xx} = 137.2 \text{ mm}, r_{yy} = 28.2 \text{ mm}$$



For economical Built up column ; We have

$$I_{xx(\text{whole s/c})} = I_{yy(\text{whole s/c})}$$

$$\therefore I_{xx(\text{self})} = \{I_{yy(\text{self})} + A(h)^2\} \times 2$$

$$9312.6 \times 10^4 = \{394.6 \times 10^4 + 4947(C_{yy} + \frac{S}{2})^2\}$$

$$\therefore S = 220.32 \text{ mm say } 222 \text{ mm}$$

Step i) Actual L.C.C. Provided :

$$I_{xx(\text{whole s/c})} = 2 \times 9312.6 \times 10 = 186.252 \times 10^6 \text{ mm}^4$$

$$I_{yy(\text{whole s/c})} = \left\{ 394.6 \times 10^4 + 4947 \left(C_{yy} + \frac{S}{2} \right)^2 \right\} \times 2 = 188.47 \times 10^6 \text{ mm}^4$$

$$I_{xx(\text{whole s/c})} < I_{yy(\text{whole s/c})} \quad \therefore I_{\min} = I_{xx(\text{whole s/c})} = 186.252 \times 10^6$$

$$r_{\min} = \sqrt{\frac{I_{\min}}{A_{\text{gross}}}} = \sqrt{\frac{186.252 \times 10^6}{2 \times 4947}} = 133.20 \text{ mm}$$

$$\text{Now, } (\lambda) = \frac{l_{\text{eff}}}{r_{\min}} = \frac{4 \times 10^3}{133.20} = 30.03 \text{ say } 30$$

Table 9(c) Design Compressive Stress, f_{cd} (MPa) for Column Buckling Class c
IS 800-2007 ; Pg. 42; (Clause 7.1.2.1)

$(\lambda) = \frac{l_{\text{eff}}}{r_{\min}} = \frac{KL}{r}$	Yield Stress, f_y (MPa) = 250 Mpa
30.00	211

Interpolation :

$$\left\{ \left(\frac{\quad}{\quad} \right) \times \left(\quad \right) \right\}$$

$$\therefore (f_{cd}) = 211 \frac{\text{N}}{\text{mm}^2}$$

$$\begin{aligned} \therefore \text{Compressive strength} &= f_{cd} \times A_{\text{gross}} \\ &= 211 \times (2 \times 4947) = 2087.63 \text{ KN} \\ &> 1100 \text{ KN ; Safe} \end{aligned}$$

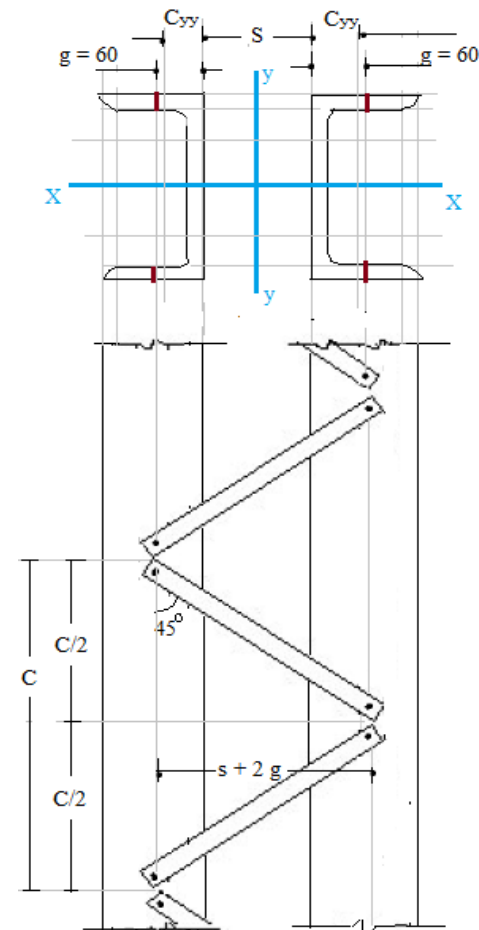
Step ii) Design Of Lacing :

Let us provide single lacing system with bolted connection.

Inclination of lacing with vertical is $\nless 40^\circ$ and $\ngtr 70^\circ$

.....cl 7.6.4 ; IS 800-2007; pg. 50

Let it be $\theta = 45^\circ$



$$\tan 45^\circ = \frac{opp}{adj} = \frac{S+2(g)}{C/2} = \frac{222+(2 \times 60)}{C/2}$$

$$\therefore \text{Spacing (c)} = 684 \text{ mm}$$

Step iii) Check for spacing as per IS 800-2007:

$$\left(\frac{C}{r_{min}}\right) = \left(\frac{C}{r_{YY}}\right) = \frac{684}{28.2} = 24.25$$

$$\geq i) 50 \text{ and}$$

$$\geq i) 0.7 \times (\lambda)_{group} = 0.7 \times 30 = 21 ; \text{Not safe}$$

.....cl 7.6.5 ; IS 800-2007; pg. 50

$$\text{Revise spacing ; } \therefore \left(\frac{C_{req}}{r_{min}}\right) = \left(\frac{C_{req}}{r_{YY}}\right) = \frac{C_{req}}{28.2} = 21$$

$$\therefore C_{req} = 28.2 \times 21 = 592 \text{ mm and}$$

$$\left(\frac{C_{req}}{r_{min}}\right) = \left(\frac{C_{req}}{r_{YY}}\right) = \frac{C_{req}}{28.2} \geq 50$$

$$\therefore C_{req} = 28.2 \times 50 = 1410 \text{ say } 1000 \text{ mm}$$

$$\therefore C_{prov} = 1000 \text{ mm}$$

By mistake , I have taken $C_{req} = 576.70$ mm in following calculations, you take 1000 mm

$$\tan \theta = \frac{opp}{adj} = \frac{S+2(g)}{C/2} = \frac{222+(2 \times 60)}{576.70/2} = \frac{342}{288} = 1.185$$

$$\therefore \text{Angle of Inclination (} \theta \text{)} = \tan^{-1} (1.185) = 50$$

$$\therefore \text{length of lacing ; } Z = \sqrt{(s + 2g)^2 + C/2^2}$$

$$= \sqrt{(342)^2 + (288)^2} = 447 \text{ mm}$$

Step iii) Check for thickness as per IS 800-2007:

$$i) t \geq \frac{Z}{40} = \frac{447}{40} = 11.175 \text{ say } 12 \text{ mm (Single lacing)}$$

$$i) t \geq \frac{Z}{60} \text{ (Double lacing) ; Safe}$$

.....cl 7.6.3 ; IS 800-2007; pg. 50

Step iv) Check for width of lacing as per IS 800-2007:

$$b_{lacing} = 3 \times \phi_{bolt}$$

.....cl 7.6.2 ; IS 800-2007; pg. 50

Let us provide 20 mm ϕ bolt of grade 4.6

$$\therefore b_{lacing} = 3 \times \phi_{bolt} = 3 \times 20 = 60 \text{ mm}$$

Step v) Load / Force calculation as per IS 800-2007:

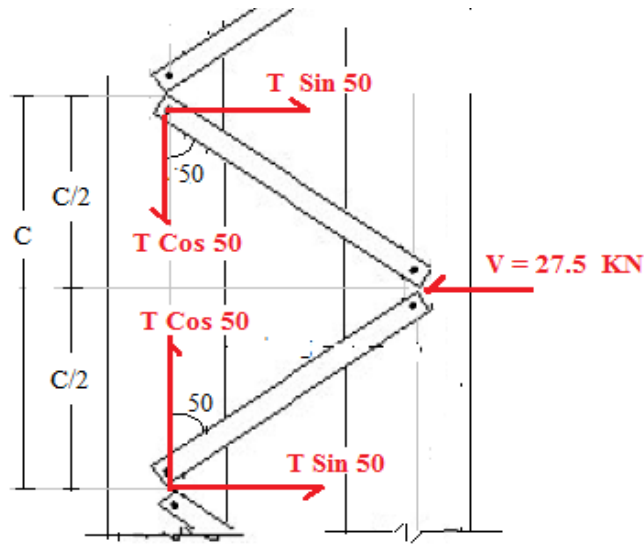
$$a) \text{ Transverse shear (v)} = 2.5 \% \text{ of Axial Load}$$

$$= \left(\frac{2.5}{100}\right) \times 1100 = 27.5 \text{ KN}$$

$$b) \text{ Transverse shear In each lacing ;}$$

$$\text{Equating } \sum X = \sum Y$$

$$2 T \sin 50 = 27.5 \dots \dots \dots \text{Refer Fig.}$$



$$\therefore T = \left(\frac{27.5}{2} \right) \times \left(\frac{1}{\sin 50} \right) = 17.94 \text{ kN}$$

Step v) Check For L.C.C. Of lacing bar :

Properties ; $b = 60 \text{ mm}$; $t = 12 \text{ mm}$; $Z = 447 \text{ mm}$

a) In Compression ;

$$\text{We have ; } r_{min} = \sqrt{\frac{I}{A_{net}}} = \sqrt{\frac{\frac{b \times t^3}{12}}{b \times t}} = \frac{t}{\sqrt{12}} = \frac{12}{\sqrt{12}} = 3.46 \text{ mm}$$

$$\lambda = \frac{l_{eff}}{r_{min}} = \frac{Z}{r_{min}} = \frac{447}{3.46} = 129.19$$

Table 9(c) Design Compressive Stress, f_{cd} (MPa) for Column Buckling Class c
IS 800-2007 ; Pg. 42; (Clause 7.1.2.1)

$(\lambda) = \frac{l_{eff}}{r_{min}} = \frac{KL}{r}$	Yield Stress, f_y (MPa) = 250 Mpa
120	83.7
129.19	???
130	74.3

Interpolation :

$$\left\{ \frac{(83.7 - 74.3)}{(130 - 120)} \times (129.19 - 120) \right\} - 83.7$$

$$\therefore (f_{cd}) = 75.06$$

$$\therefore \text{Compressive strength} = f_{cd} \times A_{gross}$$

$$75.06 \times b \times t = 75.06 \times 60 \times 12$$

$$= 54.04 \text{ kN}$$

Here ; $54.0 \text{ kN} > 17.94 \text{ kN}$; Safe

b) In Tension ;

i) Yielding Consideration : $T_{dg} = 1/\gamma (A_g F_y)$

.....IS 800-2007; cl. 6.2 ; pg. 32

Where, A_g = Gross Area Of bar S/c = $(60 \times 12) = 720 \text{ mm}^2$

F_y = Yield Stress Of the bar = 250 N/mm^2

γ = Partial Safety Factor for failure by Yielding = 1.10

.....IS 800-2007 ; Table 5 ; pg. 30

$$\therefore T_{dg} = 163.636 \text{ kN} > 17.94 \text{ kN} ; \text{ Safe}$$

ii) Rupture Consideration : $T_{dn} = 1/\gamma (0.9 A_n F_u)$

..... IS 800-2007; cl. 6.3.1 ; pg. 32

γ = Partial Safety Factor for failure by Rupture = 1.25

..... IS 800-2007 ; Table 5 ; pg. 30

A_n = Net effective area of the member = $((b) - (n d_o)) \times t$

Where , n = no. of Bolt Holes in weakest s/c = 1

Dia. Of Bolt Hole = $d_0 = \phi + 2 = 20 + 2 = 22$ mm

$$A_n = (60 - 22) \times 12 = 456 \text{ mm}^2$$

Ultimate stress of laced bar material $F_u = F_{up} = 410$

$$N/mm^2$$

$$\therefore T_{dn} = 134.61 \text{ KN}$$

Since ; $T_{dn} < T_{dg}$

\therefore Design strength of bar in tension $T_{dn} = 134.61 \text{ KN}$

c) Design Strength of bar in Shear (Double shear) (V_{ds});

$$V_{ds} = \left\{ 1/\gamma \left[\frac{F_u}{\sqrt{3}} (n_n A_{nb}) \right] \right\}$$

..... IS 800-2007; cl. 10.3.3 ; pg. 75

Ultimate stress for bolt material $F_u = F_{ub} = 400 \text{ N/mm}^2$

n_n = No. of shear planes with threads intercepting shear planes.

= 2 (one bolt is passing from two lacing bars)

A_{nb} = Net shear area of bolts at threads

$$A_{nb} = 0.78 \text{ to } 0.80 \times \frac{\pi}{4} \times d^2$$

$$= 0.78 \text{ to } 0.80 \times \frac{\pi}{4} \times 20^2 = 251.32 \text{ mm}^2$$

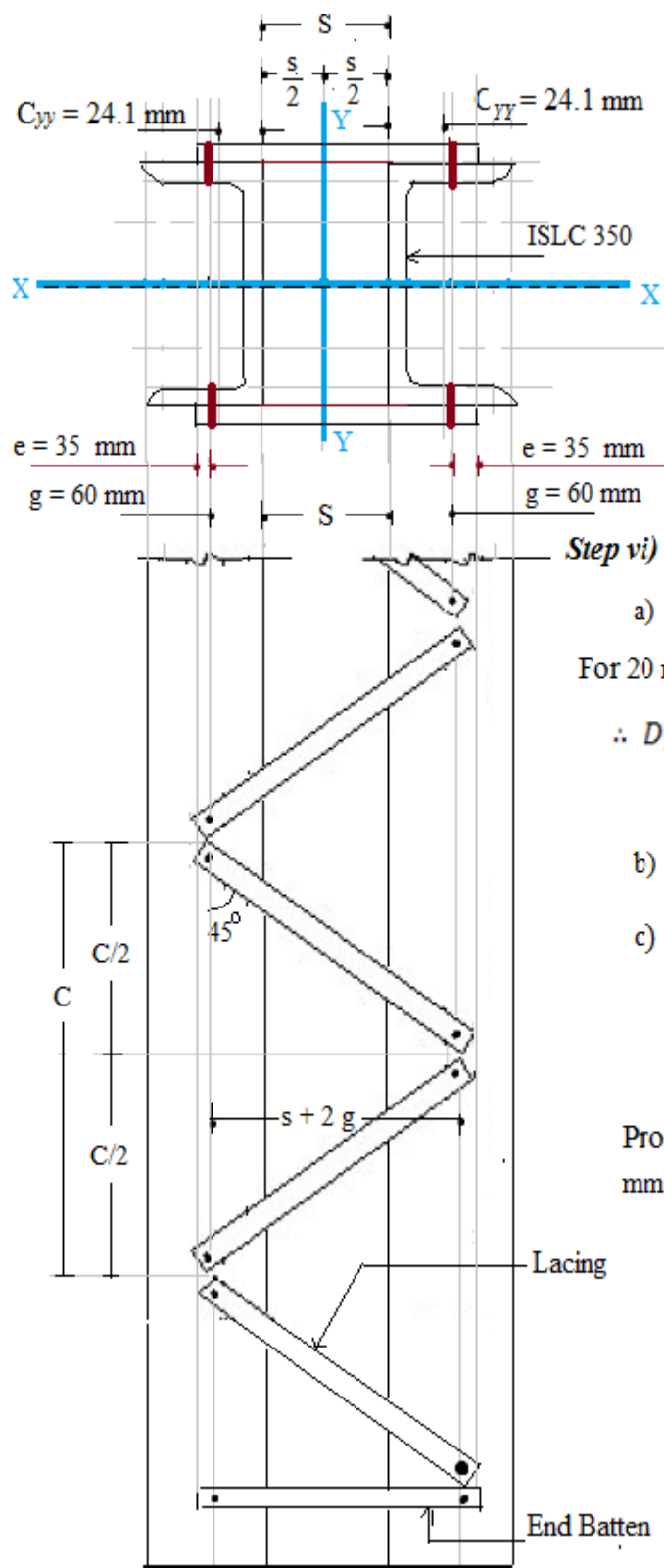
$$\therefore V_{ds} = 92.865 \text{ KN}$$

Since ; Compressive strength $< T_{dn} < V_{ds}$

\therefore Design strength of lacing bar = **54.04 KN**

$$\text{No. Of Bolts Required} = \frac{T}{\text{Design strength}}$$

$$= \frac{17.94 \times 10^3}{54.04 \times 10^3} = 0.33 \text{ say 01 No.}$$



Step vi) Design Of End Battens (Tie plate) :

a) Effective depth = $d = S + 2(g) = 222 + 2(60) = 342 \text{ mm}$

For 20 mm \varnothing bolt ; Edge distance = $e = 1.5 \times d_0 = 1.5 \times 22 = 33$

$\therefore D_{\text{overall}} = 342 + 2(33) = 408 \text{ say } 410 \text{ mm}$

$\leq (S + 2(g)) = 342 \text{ mm} ; \text{ Safe}$

b) Width $\geq S = 222 \text{ say } 225 \text{ mm}$

c) Thickness of End Battens = $\frac{1}{50} \times (S + 2g)$

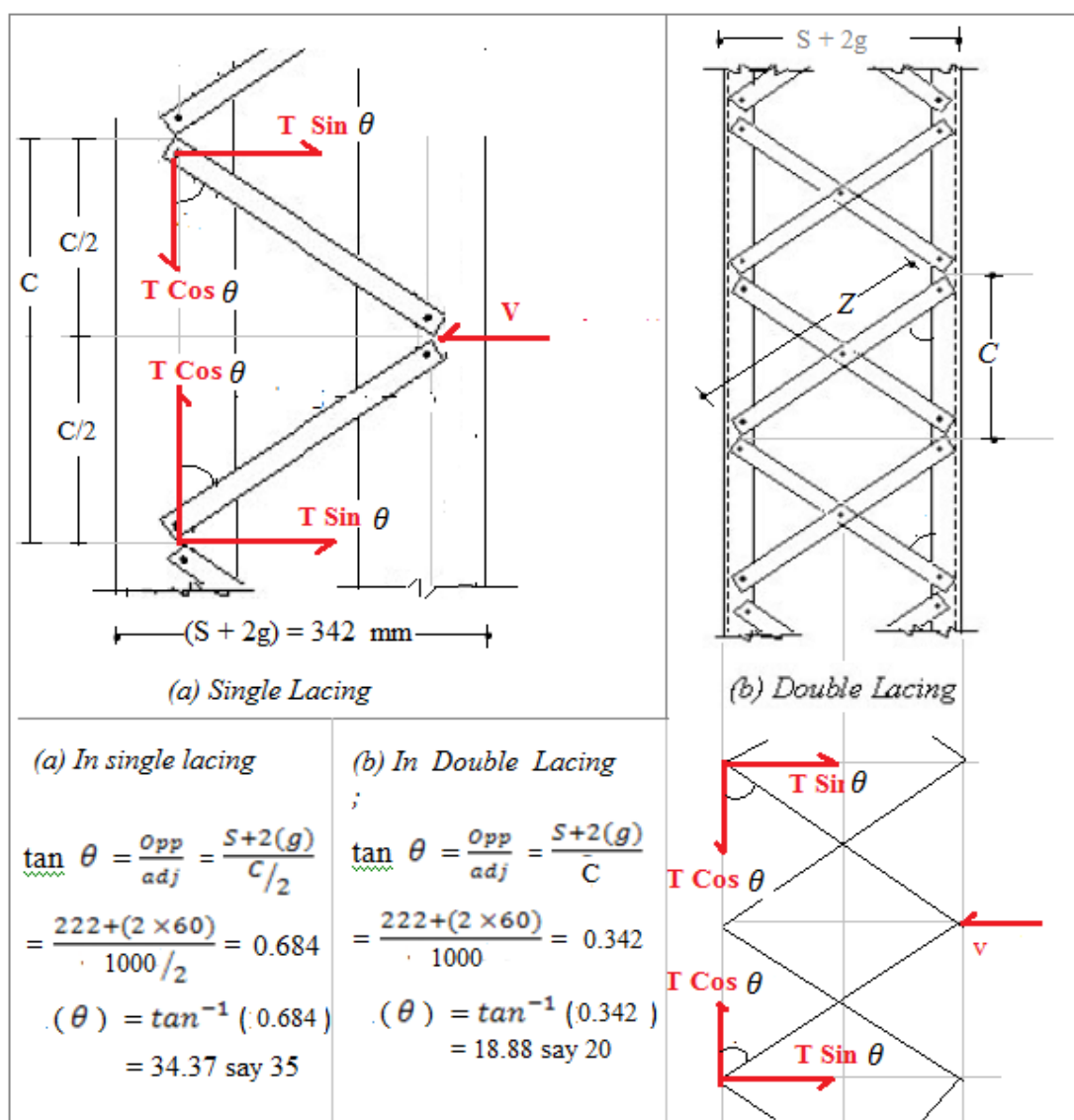
$= \frac{342}{50} = 6.84 \text{ say } 8 \text{ mm}$

Provide end Battens of 410 x 225 x 8 mm connected using 20 mm \varnothing bolt.

Thickness of End Batten = $\frac{1}{50} \times (S + 2g)$ IS 800-2007; cl. 7.7.2.3 ; pg. 51

Problem 2) Design economical built up column to carry 1100 KN using two channels placed *toe to toe*. The length of column is 5 mt. with both ends are held in position and only one end is restrained against rotation. Design a suitable **Double Lacing** system ? Take $f_y = 250 \text{ Mpa}$

Answer :



See the difference Of C (Spacing) in above cases and carry design steps as in Problem no.

Problem 3) Design economical built up column to carry 1100 KN using two channels placed *toe to toe*. The length of column is 5 mt. with both ends are held in position and only one end is restrained against rotation. Design a suitable **Lacing system** ? Take $f_y = 250 \text{ Mpa}$

Answer : Given ;

a) Length of member (clear) = 5.00 mt = $5.00 \times 10^3 \text{ mm}$

Given case is **One End Fix And Other Is Hinge (case 02)**

$$\therefore l_{eff} = KL = 0.8 \times 5 \times 10^3 = 4.00 \times 10^3 \text{ mm}$$

..... IS 800-2007; pg. 45

Let. λ (slenderness ratio) = 70

$$\text{For } f_y = 250; (\text{compressive stress}) \sigma_{ac} = 152 \text{ N/mm}^2$$

..... IS 800-2007; pg. 42

$$\text{Area Req'd. For the strut } (A_{Req}) = \frac{1100 \times 10^3}{152} = 7236.84 \text{ mm}^2$$

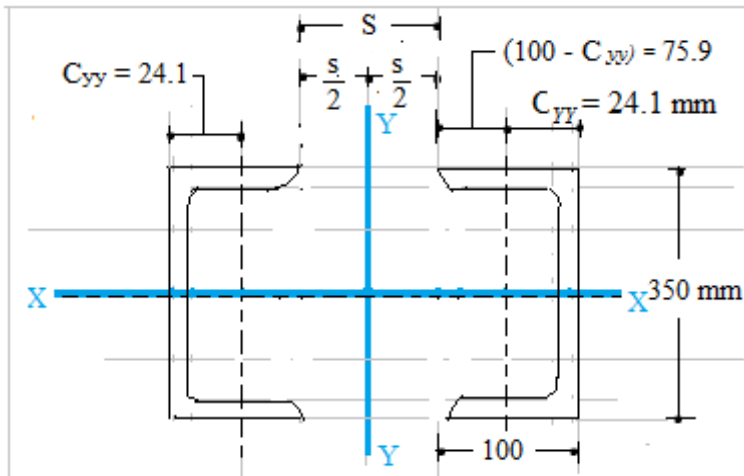
$$\text{Area of Each angle} = \frac{7236.84}{2} = 3618.42 \text{ mm}^2$$

Refer the steel table and select rolled steel channel ISLC 350 @ $38.8 \frac{\text{kg}}{\text{m}}$

For this channel ; $A = 4947 \text{ mm}^2$; width (b) = 100 mm ;

$t_f = 12.5 \text{ mm}$; $t_w = 7.4 \text{ mm}$; **Center of gravity** ; $C_{yy} = 24.1 \text{ mm}$

; guage distance (g^*) = $g = 60 \text{ mm}$



For economical Built up column ; We have

$$I_{xx(whole\ s/c)} = I_{yy(whole\ s/c)}$$

$$\therefore I_{xx(self)} = \{ I_{yy(self)} + A(h)^2 \} \times 2$$

$$9312.6 \times 10^4 = \{ 394.6 \times 10^4 + 4947 (75.9 + \frac{S}{2})^2 \}$$

$$\therefore S = 116.73 \text{ mm}$$

Say S = 117 mm

Follow steps same as in Problem no. 01

Step i) Actual L.C.C. Provided :

$$I_{xx(whole\ s/c)} = 2 \times 9312.6 \times 10 = 186.252 \times 10^6 \text{ mm}^4$$

$$I_{yy(whole\ s/c)} = \{ 394.6 \times 10^4 + 4947 (\frac{S}{2})^2 \} \times 2 = 128.91 \times 10^6 \text{ mm}^4$$

$$I_{yy(whole\ s/c)} < I_{xx(whole\ s/c)} \therefore I_{min} = I_{yy(whole\ s/c)} = 128.91 \times 10^6$$

$$r_{min} = \sqrt{\frac{I_{min}}{A_{gross}}} = \sqrt{\frac{128.91 \times 10^6}{2 \times 4947}} = 114.14 \text{ mm}$$

$$\text{Now, } (\lambda) = \frac{l_{eff}}{r_{min}} = \frac{4 \times 10^3}{114.14} = 35$$

Table 9(c) Design Compressive Stress, f_{cd} (MPa) for Column **Buckling Class c**
IS 800-2007 ; Pg. 42; (Clause 7.1.2.1)

$(\lambda) = \frac{l_{eff}}{r_{min}} = \frac{KL}{r}$	Yield Stress, f_y (MPa) = 250 Mpa
30	211
35	???
40	198

Interpolation :

$$\left\{ \frac{(211 - 198)}{(40 - 30)} \times (35 - 30) \right\} - 211$$

$$\therefore (f_{cd}) = 204.5 \frac{\text{N}}{\text{mm}^2}$$

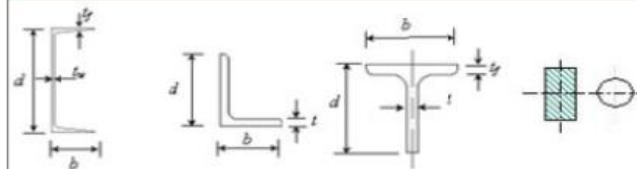
$$\text{Compressive strength} = f_{cd} \times A_{gross}$$

$$= 204.5 \times (2 \times 4947) = 2023.32 \text{ KN}$$

$$> 1100 \text{ KN ; Safe}$$

IS 800 - 2007; Pg. 44; Table 5.3 Buckling class of cross sections
(Section 7.1.2.2)

Cross Section	Limits	Buckling about axis	Buckling Class
Channel, Angle, T and Solid Sections		Any	c



Problem 4) Design economical built up column to carry 1100 KN using two channels back to back. The length of column is 5 mt. with both ends are held in position and only one end is restrained against rotation. Design a suitable **Battening system** ? Take $f_y = 250 \text{ Mpa}$

Answer : Given ;

a) Length of member (clear) = 5.00 mt = $5.00 \times 10^3 \text{ mm}$

Given case is **One End Fix And Other Is Hinge (case 02)**

$$\therefore l_{eff} = KL = 0.8 \times 5 \times 10^3 = 4.00 \times 10^3 \text{ mm}$$

..... IS 800-2007; pg. 45

Let. λ (slenderness ratio) = 70

For $f_y=250$; (compressive stress) $\sigma_{ac} = 152 \text{ N/mm}^2$

..... IS 800-2007; pg. 42

Area Reqd. For the strut (A_{Req}) = $\frac{1100 \times 10^3}{152} = 7236.84 \text{ mm}^2$

$$\text{Area of Each angle} = \frac{7236.84}{2} = 3618.42 \text{ mm}^2$$

Refer the steel table and select rolled steel channel ISLC 350

@ $38.8 \frac{\text{kg}}{\text{m}}$

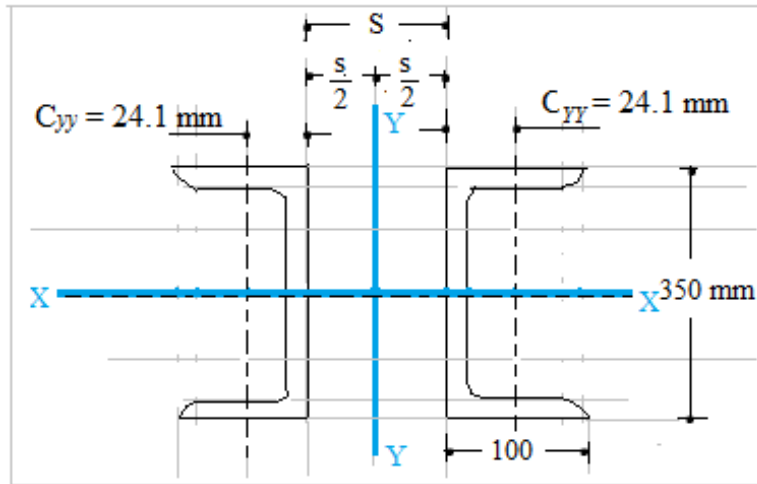
For this channel ; $A = 4947 \text{ mm}^2$; width (b) = 100 mm ;

$t_f = 12.5 \text{ mm}$; $t_w = 7.4 \text{ mm}$; Center of gravity ; $C_{yy} = 24.1 \text{ mm}$

; guage distance (g^*) = $g = 60 \text{ mm}$

$$I_{xx} = 9312.6 \times 10^4 \text{ mm}^4, I_{yy} = 394.6 \times 10^4 \text{ mm}^4;$$

$$r_{xx} = 137.2 \text{ mm}, r_{yy} = 28.2 \text{ mm}$$



For economical Built up column ; We have

$$I_{xx(whole\ s/c)} = I_{yy(whole\ s/c)}$$

$$\therefore \cancel{I} (I_{xx(self)}) = \{ I_{yy(self)} + A (ht)^2 \} \times \cancel{2}$$

$$9312.6 \times 10^4 = \{ 394.6 \times 10^4 + 4947 (C_{yy} + \frac{S}{2})^2 \}$$

$$\therefore S = 220.32 \text{ mm say } 222 \text{ mm}$$

Step i) Actual L.C.C. Provided :

$$I_{xx(whole\ s/c)} = 2 \times 9312.6 \times 10 = 186.252 \times 10^6 \text{ mm}^4$$

$$I_{yy(whole\ s/c)} = \{ 394.6 \times 10^4 + 4947 (C_{yy} + \frac{S}{2})^2 \} \times 2 = 188.47 \times 10^6 \text{ mm}^4$$

$$I_{xx(whole\ s/c)} < I_{yy(whole\ s/c)} \therefore I_{min} = I_{xx(whole\ s/c)} = 186.252 \times 10^6$$

$$r_{min} = \sqrt{\frac{I_{min}}{A_{gross}}} = \sqrt{\frac{186.252 \times 10^6}{2 \times 4947}} = 133.20 \text{ mm}$$

$$\text{Now, } (\lambda) = \frac{l_{eff}}{r_{min}} = \frac{4 \times 10^3}{133.20} = 30.03 \text{ say } 30$$

Table 9(c) Design Compressive Stress, f_{cd} (MPa) for Column Buckling Class c
IS 800-2007 ; Pg. 42; (Clause 7.1.2.1)

$(\lambda) = \frac{l_{eff}}{r_{min}} = \frac{KL}{r}$	Yield Stress, f_y (MPa) = 250 Mpa
30.00	211

Interpolation :

$$\left\{ \left(\frac{\quad}{\quad} \right) \times \left(\quad \right) \right\}$$

$$\therefore (f_{cd}) = 211 \frac{\text{N}}{\text{mm}^2}$$

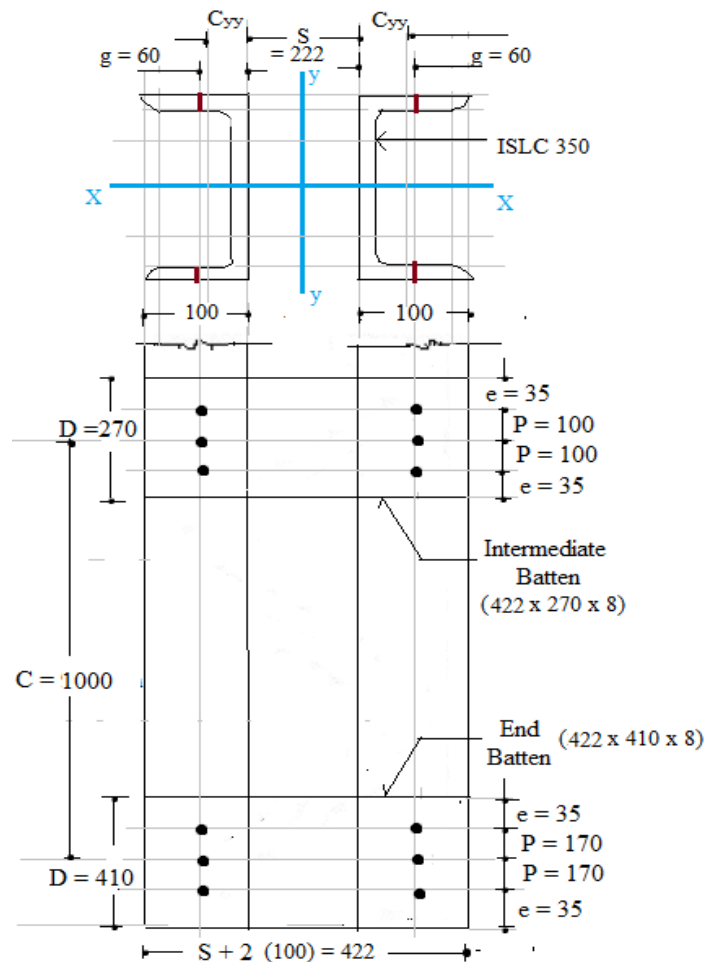
$$\therefore \text{Compressive strength} = f_{cd} \times A_{gross}$$

$$= 211 \times (2 \times 4947) = 2087.63 \text{ KN}$$

$$> 1100 \text{ KN ; Safe}$$

Step ii) Design Of Battening :

Let us provide single Battening system with bolted connection.



Step iii) Check for spacing as per IS 800-2007:

$$\left(\frac{C}{r_{min}}\right) \not\geq i) \quad 50 \quad \text{and}$$

$$\not\geq ii) \quad 0.7 \times (\lambda)_{group} = 0.7 \times 30 = 21 ; \text{Not safe}$$

.....cl 7.6.5 ; IS 800-2007; pg. 50

$$\therefore \left(\frac{C_{req}}{r_{min}}\right) = \left(\frac{C_{req}}{r_{yy}}\right) = \frac{C_{req}}{28.2} = 21$$

$$\therefore C_{req} = 28.2 \times 21 = 592 \text{ mm} \quad \text{and}$$

$$\left(\frac{C_{req}}{r_{min}}\right) = \left(\frac{C_{req}}{r_{yy}}\right) = \frac{C_{req}}{28.2} \not\geq 50$$

$$\therefore C_{req} = 28.2 \times 50 = 1410 \text{ say } 1000 \text{ mm}$$

$$\therefore C_{prov} = 1000 \text{ mm}$$

\therefore No. of Battens =

$$\frac{l_{eff}}{C} = \frac{4.00 \times 10^3}{1000} = 04 \text{ No.}$$

Step iv) Design Of End Battens (Tie plate) :

a) Effective depth = $d = S + 2(g) = 222 + 2(60) = 342 \text{ mm}$

For 20 mm \varnothing bolt ; Edge distance = $e = 1.5 \times d_0 = 1.5 \times 22 = 33 \text{ mm}$

$\therefore D_{\text{overall}} = 342 + 2(33) = 408 \text{ say } 410 \text{ mm}$

$\nless (S + 2(g)) = 342 \text{ mm} ; \text{ Safe}$

b) Width $\nless S = 222 \text{ say } 225 \text{ mm}$

c) Thickness of End Battens = $\frac{1}{50} \times (S + 2g)$
 $\dots\dots\dots \text{IS 800-2007; cl. 7.7.2.3 ; pg. 51}$
 $= \frac{342}{50} = 6.84 \text{ say } 8 \text{ mm}$

Provide end Battens of 410 x 225 x 8 mm connected using 20 mm \varnothing bolt.

Step v) Design Of Intermediate Batten :

a) Effective depth = $d = \frac{3}{4} \times (S + (2 C_{yy})) = \frac{3}{4} \times 270.2 = 202.65 \text{ mm}$

For 20 mm \varnothing bolt ; Edge distance = $e = 1.5 \times d_0 = 1.5 \times 22 = 33 \text{ mm}$

$\therefore D_{\text{overall}} = 202.65 + 2(33) = 268.6 \text{ say } 270 \text{ mm}$

b) Width $\nless S = 222 \text{ say } 225 \text{ mm}$

c) Thickness of Intermediate batten = $\frac{1}{50} (S + 2g)$
 $\dots\dots\dots \text{IS 800-2007; cl. 7.7.2.3 ; pg. 51}$
 $= \frac{342}{50} = 6.84 \text{ say } 8 \text{ mm}$

Step vi) Load / Force calculation as per IS 800-2007:

a) Transverse shear (v) = 2.5 % of Axial Load
 $= \frac{(2.5}{100}) \times 1100 = 27.5 \text{ KN}$

b) Longitudinal shear (V_1)
 $(V_1) = \frac{V \times C}{N \times S} = \frac{27.5 \times 10^3 \times 1000}{2 \times 342} = 40.20 \text{ KN}$

Where N = No. of parallel planes of battens = 2

c) Moment (M) ;
 $(M) = \frac{V \times C}{2 \times N} = \frac{27.5 \times 10^3 \times 1000}{2 \times 2} = 6875 \text{ KN}$

$\dots\dots\dots \text{cl. 7.7.2.1; IS 800-2007; pg. 51}$

Step vii) Check For Stresses as per IS 800-2007:

a) End Batten :

i) \therefore Shear stress ;
 $\frac{V_1}{d \times t} = \frac{40.20 \times 10^3}{342 \times 8} = 14.70 \text{ N/mm}^2$

$\nless \frac{f_y}{\sqrt{3} \times \gamma} = \frac{250}{\sqrt{3} \times 1.10} = 131.2 \text{ N/mm}^2 ; \text{ Safe}$

ii) : Bending Stress ;

$$\text{We have ; } \frac{M}{I} = \frac{\sigma}{y} \therefore \sigma = \frac{M \times y}{I} = \frac{M \times d}{2 \times \frac{t \times d^3}{12}} = \frac{6 \times M}{t \times d^2}$$

$$= \frac{6 \times 6875 \times 10^3}{8 \times 342^2} = 44.08 \text{ N/mm}^2$$

$$\therefore \frac{f_y}{\gamma} = \frac{250}{1.1} = 227.27 \text{ N/mm}^2 ; \text{ Safe}$$

b) **Intermediate Batten :**

i) : Shear stress ;

$$\frac{V_1}{d \times t} = \frac{40.20 \times 10^3}{202.62 \times 8} = 24.80 \text{ N/mm}^2$$

$$\therefore \frac{f_y}{\sqrt{3} \times \gamma} = \frac{250}{\sqrt{3} \times 1.10} = 131.2 \text{ N/mm}^2 ; \text{ Safe}$$

ii) : Bending Stress ;

$$\text{We have ; } \frac{M}{I} = \frac{\sigma}{y} \therefore \sigma = \frac{M \times y}{I} = \frac{M \times d}{2 \times \frac{t \times d^3}{12}} = \frac{6 \times M}{t \times d^2}$$

$$= \frac{6 \times 6875 \times 10^3}{8 \times 202.65^2} = 125.55 \text{ N/mm}^2$$

$$\therefore \frac{f_y}{\gamma} = \frac{250}{1.1} = 227.27 \text{ N/mm}^2 ; \text{ Safe}$$

Step viii) Design Of Connections :

Let us provide 20 mm \emptyset bolt of grade 4.6

a) **Design Strength of bolt in Shear (Double shear) (V_{ds}) ;**

$$V_{ds} = \left\{ 1/\gamma \left[\frac{F_u}{\sqrt{3}} (n_n A_{nb}) \right] \right\}$$

.....IS 800-2007; cl. 10.3.3 ; pg. 75

Ultimate stress for bolt material $F_u = F_{ub} = 400 \text{ N/mm}^2$

n_n = No. of shear planes with threads intercepting shear planes.

= 2 (one bolt is passing from two Battening bars)

A_{nb} = Net shear area of bolts at threads

$$A_{nb} = 0.78 \text{ to } 0.80 \times \frac{\pi}{4} \times d^2$$

$$= 0.78 \text{ to } 0.80 \times \frac{\pi}{4} \times 20^2 = 251.32 \text{ mm}^2$$

$$\therefore V_{ds} = 92.865 \text{ KN}$$

$$\therefore \text{Strength Of bolt} = V_{ds} = 92.865 \text{ KN}$$

$$\text{No. Of Bolts Required} = \frac{V_1}{\text{Design strength}} = \frac{40.20}{92.865}$$

$$= 0.432 \text{ say } 03$$

Step ix) Design for Bolts against

A) End Batten :

i) : Longitudinal (Vertical) shear :

$$\frac{V_1}{\text{no. of bolts}} = \frac{40.20 \times 10^3}{03} = 13.4 \text{ KN}$$

ii) : Horizontal shear due to moment ;

For 20 mm \emptyset bolts ;

$$d_0 = \text{diameter of Bolt Hole} = \emptyset + 2 = 20 + 2 = 22 \text{ mm}$$

$$\text{Edge Distance} = e = 1.5 \times d_0 = 1.5 \times 22 = 33 \text{ say } 35 \text{ mm}$$

$$\text{Pitch} = \frac{d_{\text{overall}} - 2(e)}{2} = \frac{410 - 2(35)}{2} = 170 \text{ mm}$$

$$\begin{aligned} \therefore \text{Horizontal shear} &= \frac{M}{\sum \text{Pitch}^2} \times (\text{Pitch}) \\ &= \frac{6875 \times 10^3}{2(170^2 + 170^2)} \times (170) = 10.110 \text{ KN} \end{aligned}$$

$$\begin{aligned} \text{Resultant} &= \sqrt{13.4^2 + 10.11^2} = 16.78 \text{ KN} < V_{ds} = 92.865 \text{ KN} \\ &\therefore \text{ Safe} \end{aligned}$$

B) Intermediate Batten :

i) : Longitudinal (Vertical) shear :

$$\frac{V_1}{\text{no. of bolts}} = \frac{40.20 \times 10^3}{03} = 13.4 \text{ KN}$$

ii) : Horizontal shear due to moment ;

For 20 mm \emptyset bolts ;

$$d_0 = \text{diameter of Bolt Hole} = \emptyset + 2 = 20 + 2 = 22 \text{ mm}$$

$$\text{Edge Distance} = e = 1.5 \times d_0 = 1.5 \times 22 = 33 \text{ say } 35 \text{ mm}$$

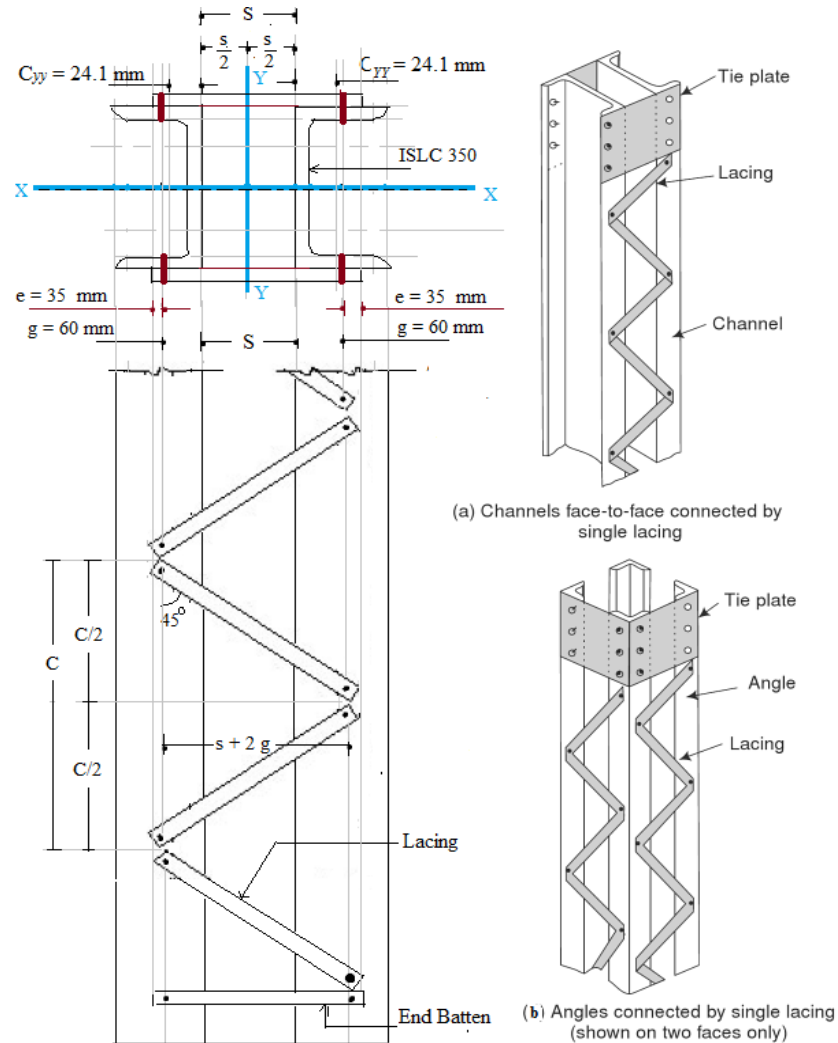
$$\text{Pitch} = \frac{d_{\text{overall}} - 2(e)}{2} = \frac{270 - 2(35)}{2} = 100 \text{ mm}$$

$$\begin{aligned} \therefore \text{Horizontal shear} &= \frac{M}{\sum \text{Pitch}^2} \times (\text{Pitch}) \\ &= \frac{6875 \times 10^3}{2(100^2 + 100^2)} \times (100) = 17.187 \text{ KN} \end{aligned}$$

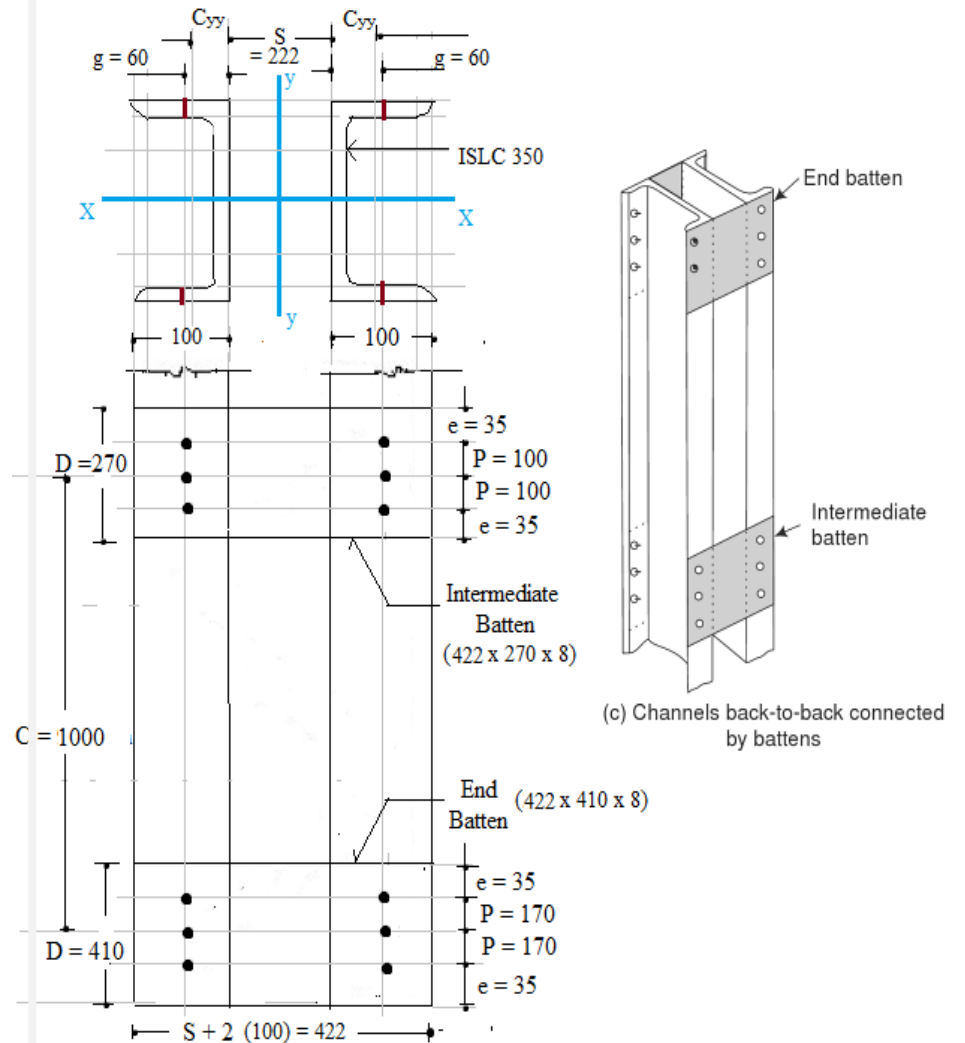
$$\begin{aligned} \text{Resultant} &= \sqrt{13.4^2 + 17.187^2} = 21.80 \text{ KN} < V_{ds} = 92.865 \text{ KN} \\ &\therefore \text{ Safe} \end{aligned}$$

SEE FIGURE

Sheet NO. 02 (Lacing & Battening)



8.



Problem 5) Design economical built up column to carry 2000 KN using Four angles. The length of column is 6.4 mt. with both ends are held in position and one end is restrained against rotation. Design a suitable **Lacing / Battening** also ? Take $f_y = 250 \text{ Mpa}$ [Summer 2013]

Answer : Given ;

a) Length of member (clear) = 6.4 mt = $6.4 \times 10^3 \text{ mm}$

Given case is **One End Fix And Other Is Hinge (case 02)**

$$\therefore l_{eff} = KL = 0.8 \times 6.4 \times 10^3 = 5.12 \times 10^3 \text{ mm}$$

..... IS 800-2007; pg. 45

Let. λ (slenderness ratio) = 70

For $f_y = 250$; (compressive stress) $\sigma_{ac} = 152 \text{ N/mm}^2$

..... IS 800-2007; pg. 42 (See table)

Area Req'd. For the strut

$$(A_{Req}) = \frac{2000 \times 10^3}{152} = 13.57 \times 10^3 \text{ mm}^2$$

$$\text{Area req'd. of Each angle} = \frac{13.57 \times 10^3}{4} = 3289.47 \text{ mm}^2$$

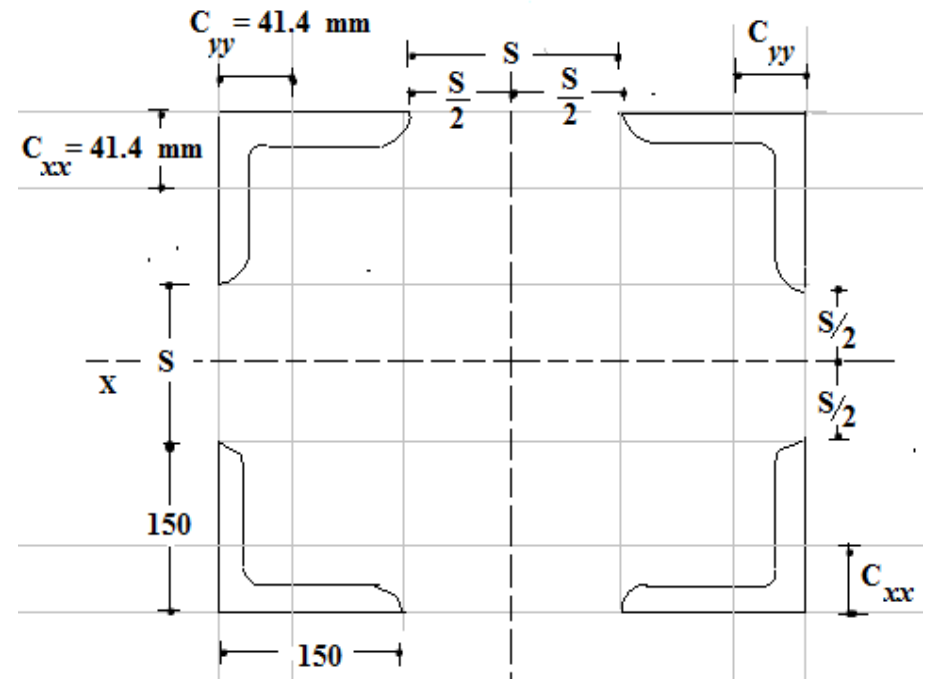
Note : Always select Equal angle s/c for easy calculations

Refer the steel table and select rolled steel equal angle such that $A_{prov} > A_{req}$ thus selecting ISA 150 x 150 x 12

For this Angle ; Area = 3459 mm^2 ; width (b) = 150 mm ;

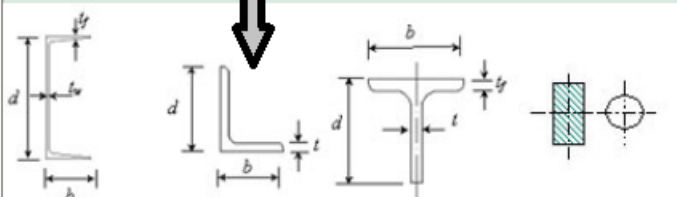
Center of gravity ; $C_{yy} = C_{xx} = 41.4 \text{ mm}$

Here, Area = $3459 > 3289.47 \text{ mm}^2$



IS 800 - 2007; Pg. 44; Table 5.3 Buckling class of cross sections (Section 7.1.2.2)

Cross Section	Limits	Buckling about axis	Buckling Class
Channel, Angle, T and Solid Sections		Any	C



(λ) Table 9(c) Design Compressive Stress, f_{cd} (MPa) for Column Buckling Class c (Clause 7.1.2.1) ; Pg. 42 ; IS 800 - 2007

KL/r ↓	Yield Stress, f_y (MPa) = 250													
	200	210	220	230	240	250	260	280	300	320	340	360	380	400
10	182	191	200	209	218	226	236	255	273	291	309	327	345	364
20	182	190	199	207	216	223	233	250	266	283	299	316	332	348
30	172	180	188	196	204	219	234	249	264	278	293	307	321	321
40	163	170	177	184	191	205	218	231	244	256	268	280	292	292
50	153	159	165	172	178	189	201	212	222	232	242	252	261	261
60	142	148	153	158	163	173	182	191	199	207	215	222	228	228
70	136	142	147	152	157	166	175	183	191	199	207	215	222	228
80	120	123	127	130	133	136	139	145	149	154	158	162	165	169

M. I. ; $I_{xx} = I_{yy} = 735.4 \times 10^4 \text{ mm}^4$;

Radius Of Gyration ; $r_{xx} = r_{yy} = 46.1 \text{ mm}$

We have ; $\lambda = \frac{l_{eff}}{r_{min}} ; 70 = \frac{5.12 \times 10^3}{r_{min}}$

$r_{min} = 73.142 \text{ mm}$

Again We have ; $r_{min} = \sqrt{\frac{I_{min}}{A}} \therefore I_{min} = A \times r_{min}^2$

$\therefore I_{min} = (4 \times 3459) \times 73.142^2 = 74.01 \times 10^6 \text{ mm}^4$

Now, Finding actual M. I. of the S/C And Comparing

With $I_{min} = 74.01 \times 10^6 \text{ mm}^4$; We will get spacing (S)

$\therefore 74.01 \times 10^6 = 4 [I_{xx(self)} + \text{Area} (\text{distance}^2)]$

$74.01 \times 10^6 = 4 [735.4 \times 10^4 + 3459 (\frac{S}{2} + (150 - 41.4))^2]$

Find S = spacing and carry steps in lacing and battening