

Type of flow	Velocity	$\phi$	$\psi$
→ Uniform flow in $x$ direction	$u = V_\infty$	$V_\infty x$	$V_\infty y$
Source	$V_r = \frac{\Lambda}{2\pi r}$	$\frac{\Lambda}{2\pi} \ln r$	$\frac{\Lambda}{2\pi} \theta$
Vortex	$V_\theta = -\frac{\Gamma}{2\pi r}$	$-\frac{\Gamma}{2\pi} \theta$	$\frac{\Gamma}{2\pi} \ln r$
Doublet	$V_r = -\frac{\kappa}{2\pi} \frac{\cos \theta}{r^2}$  $V_\theta = -\frac{\kappa}{2\pi} \frac{\sin \theta}{r^2}$	$\frac{\kappa}{2\pi} \frac{\cos \theta}{r}$	$-\frac{\kappa}{2\pi} \frac{\sin \theta}{r}$

# NON-LIFTING FLOW OVER A CIRCULAR CYLINDER

## NONLIFTING FLOW OVER A CIRCULAR CYLINDER

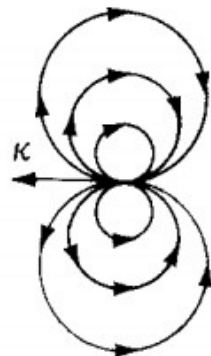
The combination of a **uniform flow** and a **doublet** produces the **flow over a circular cylinder**



Uniform flow

$$\psi = V_{\infty} r \sin \theta$$

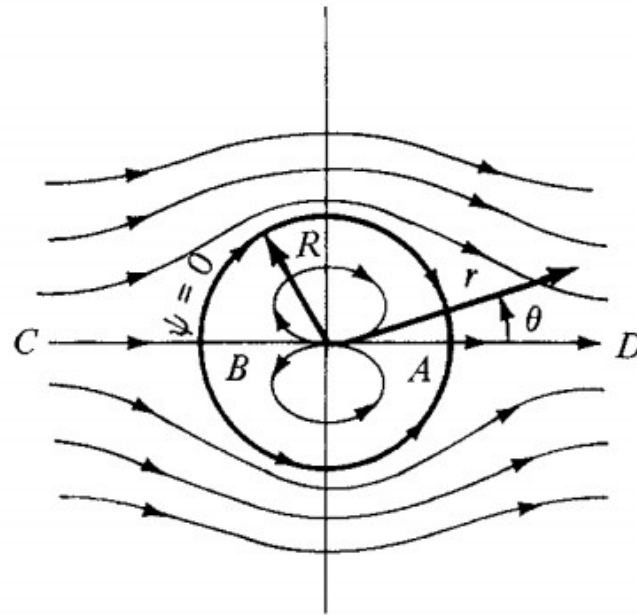
+



Doublet

$$\psi = \frac{-\kappa}{2\pi} \frac{\sin \theta}{r}$$

=



Flow over a cylinder

$$\psi = V_{\infty} r \sin \theta - \frac{\kappa}{2\pi} \frac{\sin \theta}{r}$$

$$\psi = V_{\infty} r \sin \theta - \frac{\kappa}{2\pi} \frac{\sin \theta}{r}$$

$$\psi = V_{\infty} r \sin \theta \left( 1 - \frac{\kappa}{2\pi V_{\infty} r^2} \right)$$

Let  $R^2 \equiv \kappa / 2\pi V_{\infty}$ .

$$\psi = (V_{\infty} r \sin \theta) \left( 1 - \frac{R^2}{r^2} \right)$$

It is the stream function for a uniform flow-doublet combination

When  $r=R \Rightarrow \psi = 0$

$r=R$  is the equation of a circle

Therefore,  $\psi = 0$  is the stream function for the flow over a circle of radius  $R$  as shown

**Velocity field** (Radial Velocity  $V_r$  and Tangential velocity  $V_\theta$ )

**Radial Velocity**

$$\begin{aligned} V_r &= \frac{1}{r} \frac{\partial \psi}{\partial \theta} = \frac{1}{r} \frac{\partial}{\partial \theta} \left[ V_\infty r \sin \theta \left( 1 - \frac{R^2}{r^2} \right) \right] \\ &= \frac{1}{r} (V_\infty r \cos \theta) \left( 1 - \frac{R^2}{r^2} \right) = \left( 1 - \frac{R^2}{r^2} \right) V_\infty \cos \theta \end{aligned}$$

$$V_r = \left( 1 - \frac{R^2}{r^2} \right) V_\infty \cos \theta$$

**Tangential Velocity**

$$\begin{aligned} V_\theta &= -\frac{\partial \psi}{\partial r} = -\frac{\partial}{\partial r} \left[ V_\infty r \sin \theta \left( 1 - \frac{R^2}{r^2} \right) \right] \\ &= - \left[ (V_\infty r \sin \theta) \frac{2R^2}{r^3} + \left( 1 - \frac{R^2}{r^2} \right) (V_\infty \sin \theta) \right] \\ &= - \left( 1 + \frac{R^2}{r^2} \right) V_\infty \sin \theta \end{aligned}$$

$$V_\theta = - \left( 1 + \frac{R^2}{r^2} \right) V_\infty \sin \theta$$

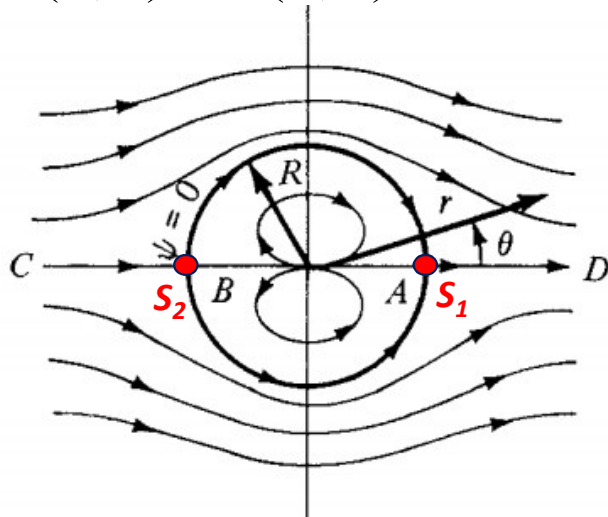
To locate the stagnation points

Stagnation points are the points in the flow where the *VELOCITY* is *ZERO*

$$V_r = 0 \Rightarrow \left(1 - \frac{R^2}{r^2}\right) V_\infty \cos \theta = 0$$

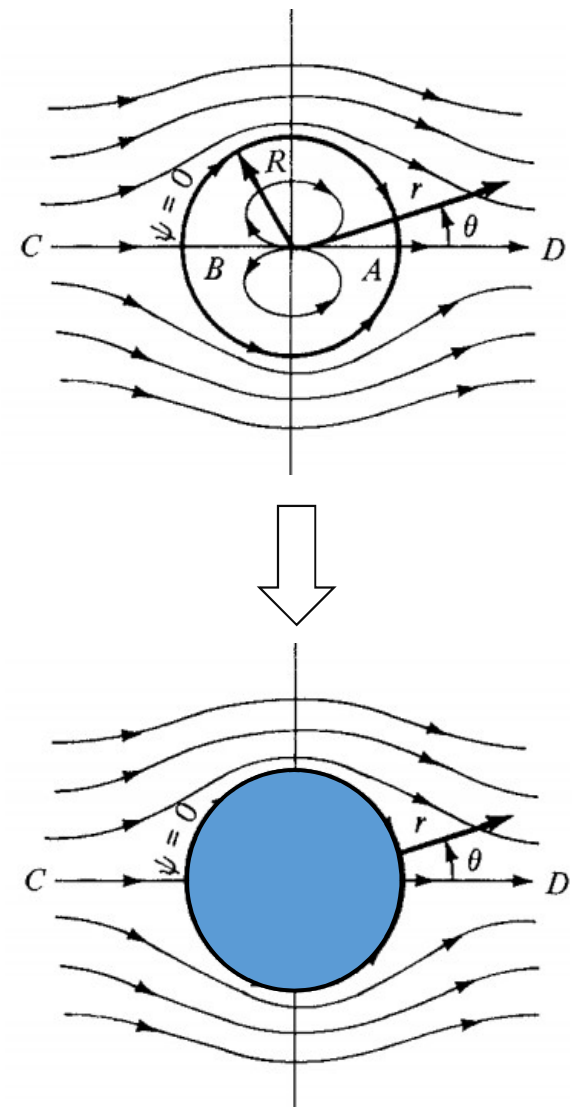
$$V_\theta = 0 \Rightarrow \left(1 + \frac{R^2}{r^2}\right) V_\infty \sin \theta = 0$$

Simultaneously solving these two equations for  $r$  and  $\theta$ , we find that there are two stagnation points, located at  $(r, \theta) = (R, 0)$  and  $(R, \pi)$



- Note that the  $\psi = 0$  streamline, since it goes through the stagnation points, is the dividing streamline.
- That is, all the flow inside  $\psi = 0$  (inside the circle) comes from the doublet, and all the flow outside  $\psi = 0$  (outside the circle) comes from the uniform flow.
- Therefore, we can replace the flow inside the circle by a solid body, and the external flow will not know the difference.
- Consequently, the inviscid, irrotational, incompressible flow over a circular cylinder of radius  $R$  can be synthesized by adding a uniform flow with velocity  $V_\infty$  and a doublet of strength  $\kappa$ , where  $R$  is related to  $V_\infty$  and  $\kappa$  through

$$R = \sqrt{\frac{\kappa}{2\pi V_\infty}}$$



## Velocity distribution over the cylinder

$$V_r = \left(1 - \frac{R^2}{r^2}\right) V_\infty \cos \theta$$

$$V_\theta = -\left(1 + \frac{R^2}{r^2}\right) V_\infty \sin \theta$$

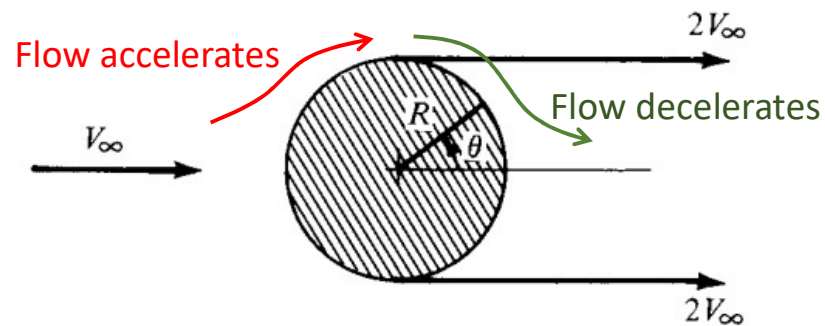
Substituting  $r=R$



$$V_r = 0$$

$$V_\theta = -2V_\infty \sin \theta$$

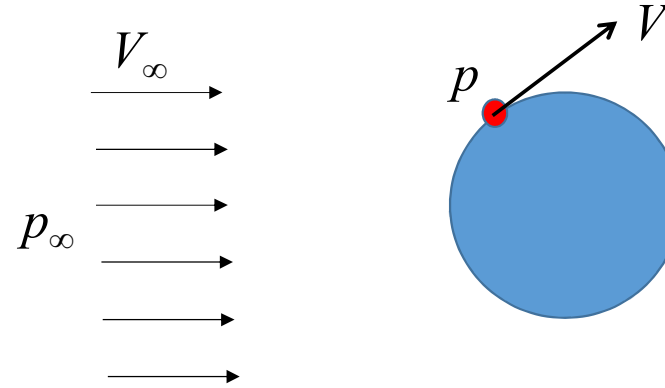
On the surface of the cylinder  $V = V_\theta = -2V_\infty \sin \theta$





## Pressure distribution over a cylinder

Coefficient of Pressure  $C_p = \frac{p - p_\infty}{\frac{1}{2} \rho V_\infty^2}$



$p_\infty, V_\infty$ : Pressure and Velocity of free-stream

$p, V$ : Pressure and Velocity at any point on the cylinder

Applying Bernoulli's equation b/w 1 and 2

$$p_\infty + \frac{1}{2} \rho V_\infty^2 = p + \frac{1}{2} \rho V^2$$

$$\Rightarrow p - p_\infty = \frac{1}{2} \rho V_\infty^2 - \frac{1}{2} \rho V^2 \Rightarrow \frac{p - p_\infty}{\frac{1}{2} \rho V_\infty^2} = 1 - \left( \frac{V}{V_\infty} \right)^2$$

$$\Rightarrow p - p_{\infty} = \frac{1}{2} \rho V_{\infty}^2 - \frac{1}{2} \rho V^2$$

$$\Rightarrow \frac{p - p_{\infty}}{\frac{1}{2} \rho V_{\infty}^2} = 1 - \frac{\frac{1}{2} \rho V^2}{\frac{1}{2} \rho V_{\infty}^2} = 1 - \left( \frac{V}{V_{\infty}} \right)^2$$

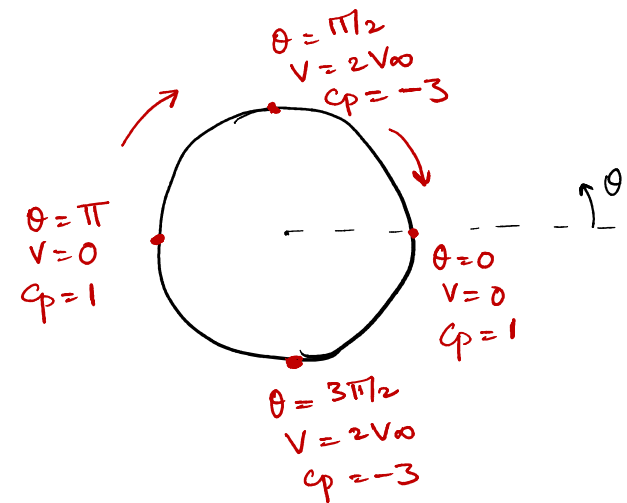
$$\Rightarrow C_p = 1 - \left( \frac{V}{V_{\infty}} \right)^2$$

$$\Rightarrow C_p = 1 - \left( \frac{-2V_{\infty} \sin \theta}{V_{\infty}} \right)^2$$

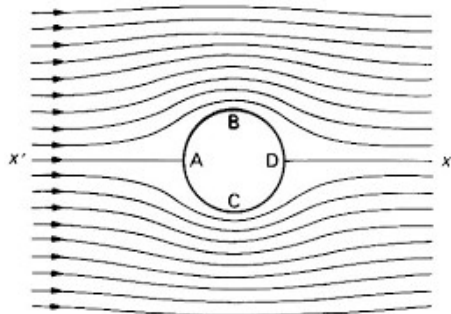
$$\Rightarrow \boxed{C_p = 1 - 4 \sin^2 \theta}$$

on the cylinder surface

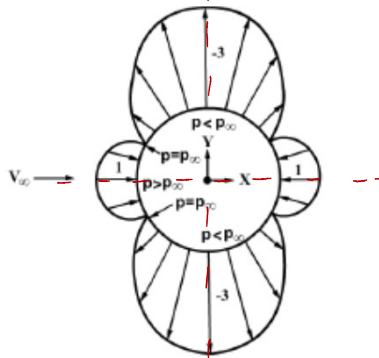
$$V = -2V_{\infty} \sin \theta$$



## Potential flow (Theoretical)

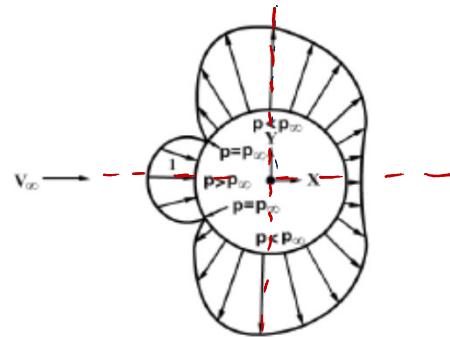
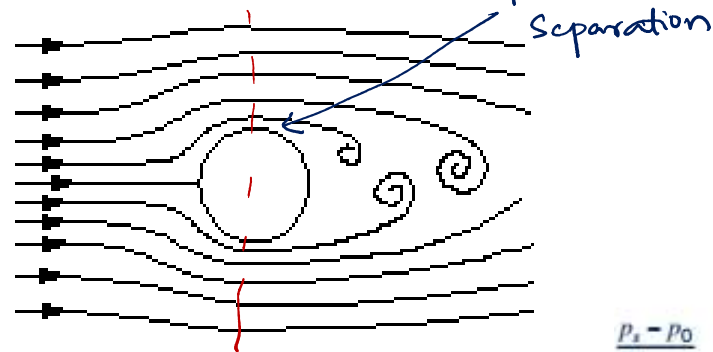


$$C_p = 1 - 4 \sin^2 \theta$$

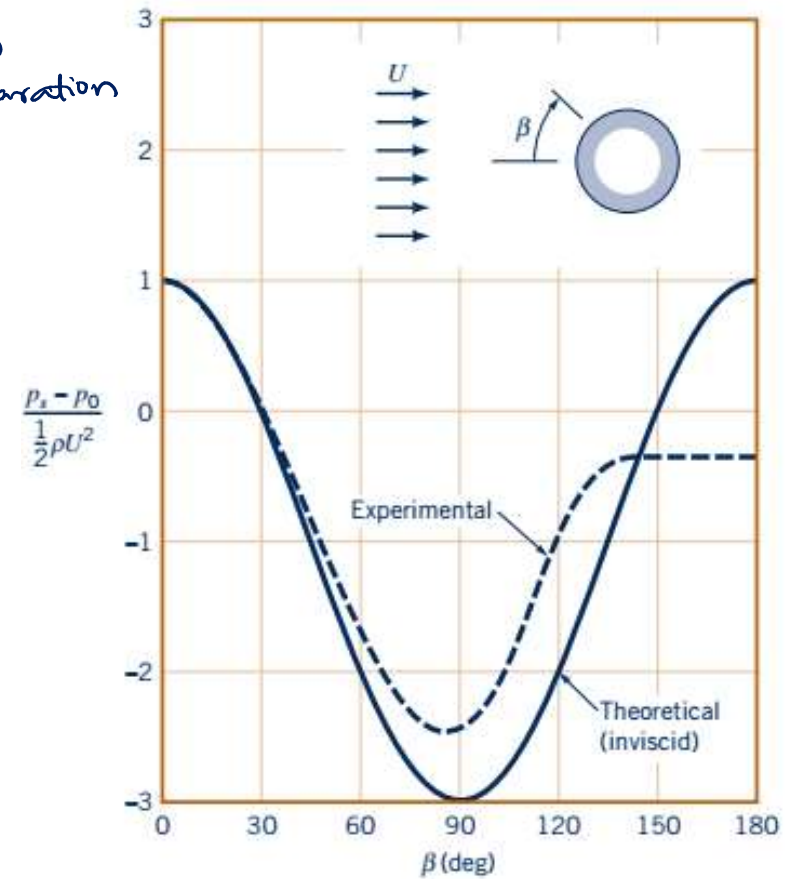


Lift = 0  
Drag = 0

## Real flow (Experimental)

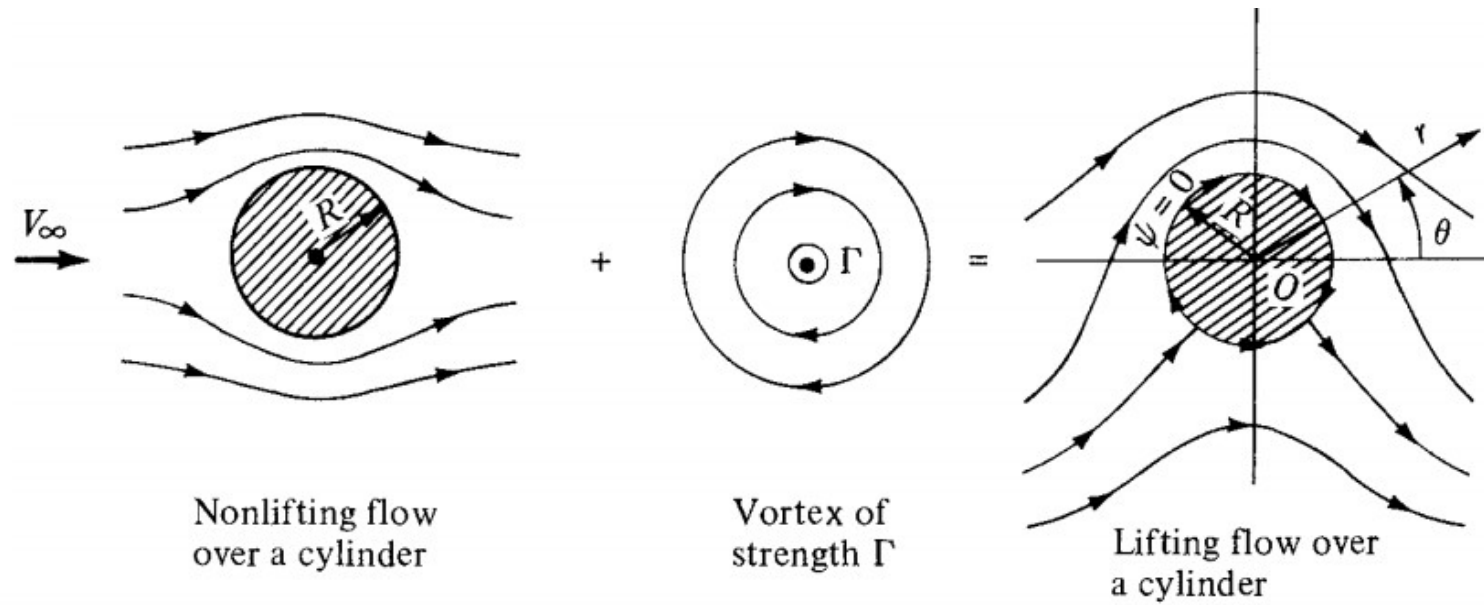


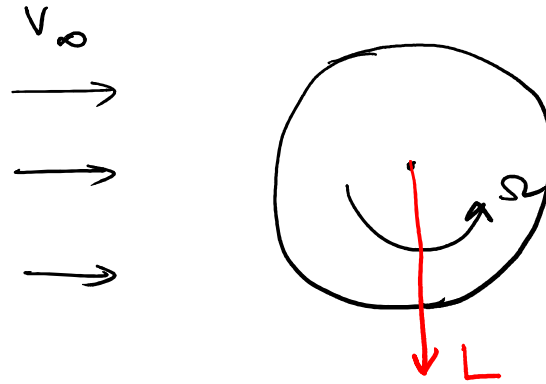
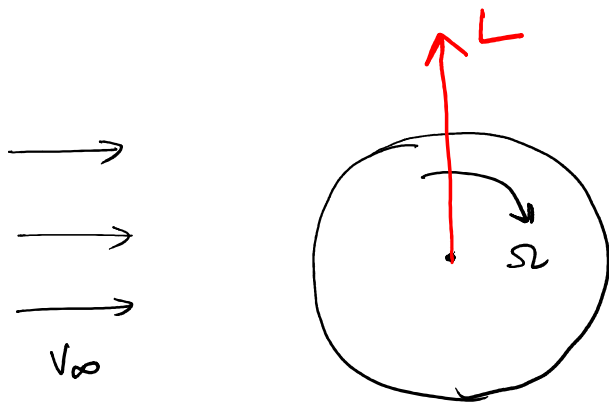
Lift = 0  
Drag ≠ 0



# LIFTING FLOW OVER A CIRCULAR CYLINDER

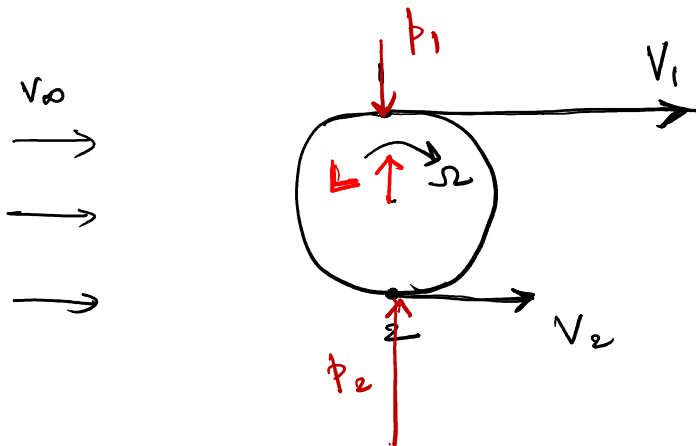
## LIFTING FLOW OVER A CIRCULAR CYLINDER – Flow over a rotating cylinder





\* A rotating cylinder or sphere moving in a fluid produces lift force  $\rightarrow$  MAGNUS EFFECT

ROBIN - MAGNUS EFFECT



$$V_1 > V_2$$

$$\Rightarrow p_1 < p_2$$