

SESA3029 Aerothermodynamics

Lecture 3.4

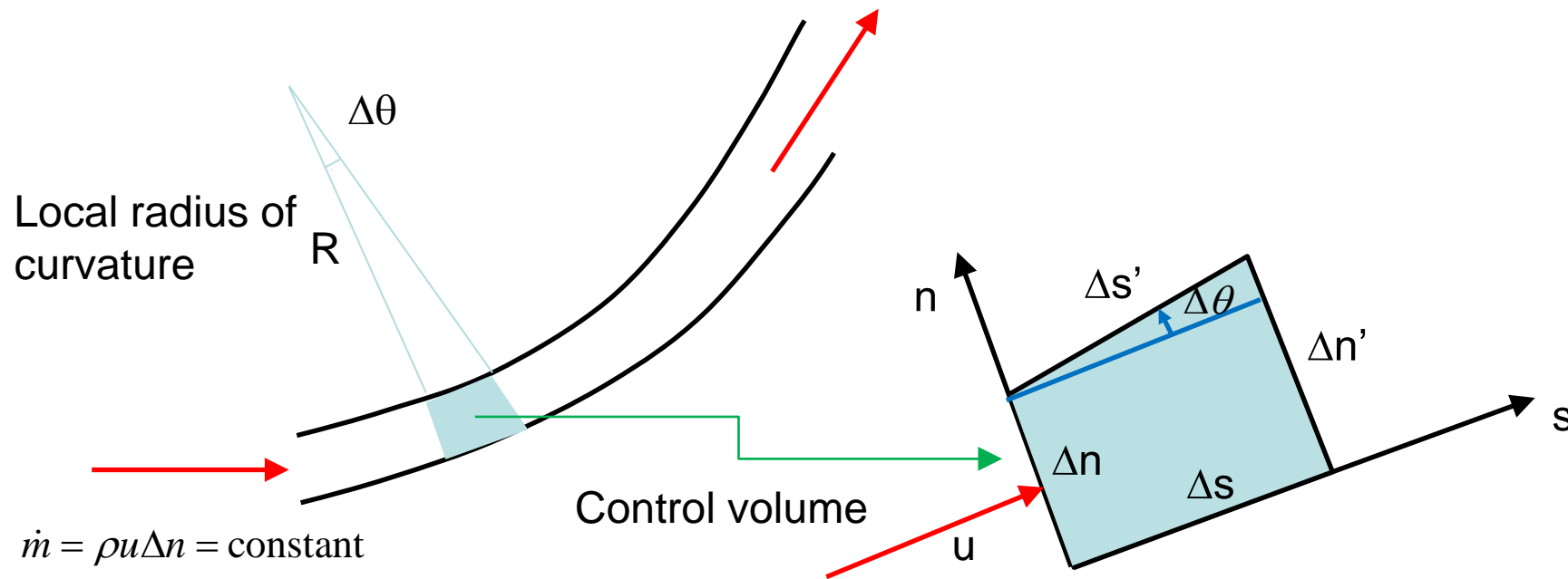
Crocco's theorem for supersonic flow
Coursework Q&A

Outline

- Complete set of equations for curved streamtube
 - connecting stagnation pressure entropy and vorticity
- Coursework Q&A

Recap: Curved streamtube

No flow across streamlines, therefore for steady flow
mass flowrate in = mass flowrate out



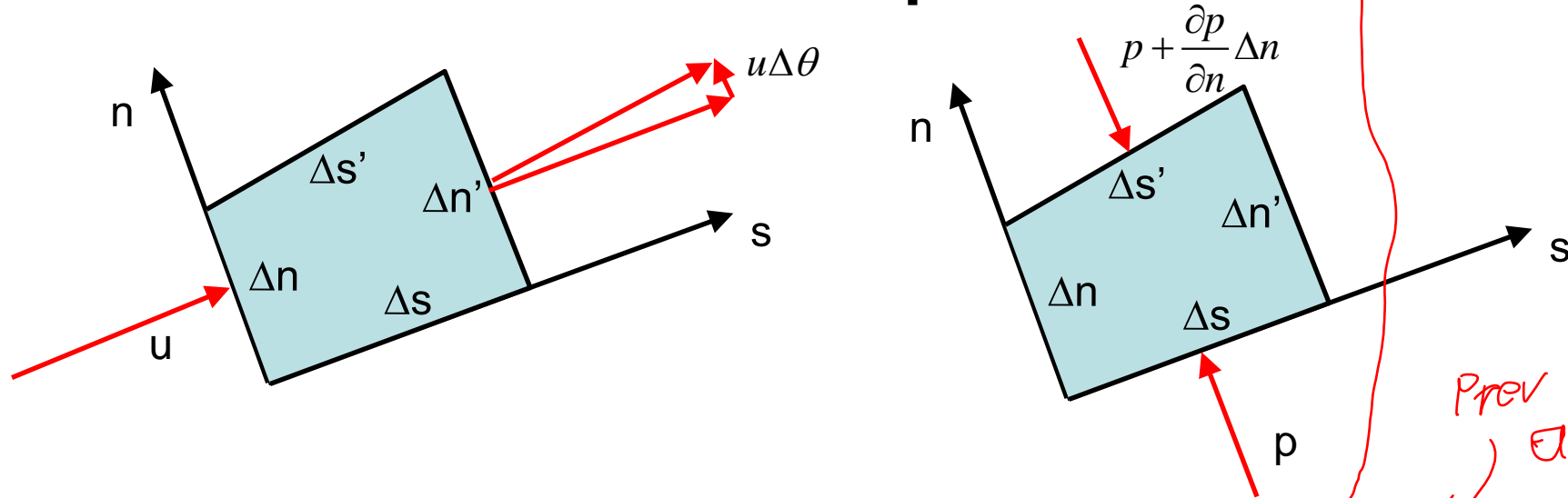
$$\Delta s = R \Delta \theta$$

$$\frac{1}{R} = \frac{\partial \theta}{\partial s} \quad (1)$$

$$\Delta n' - \Delta n = \Delta \theta \Delta s = \left(\frac{\partial \theta}{\partial n} \Delta n \right) \Delta s$$

$$\frac{1}{\Delta n} \frac{\partial \Delta n}{\partial s} = \frac{\partial \theta}{\partial n} \quad (2)$$

Euler n-equation



n-momentum out – n-momentum in = force applied in n direction

$$\cancel{\rho u \Delta n} \left[u \frac{\partial \theta}{\partial s} \Delta s \right] - 0 = - \left(\frac{\partial p}{\partial n} \Delta n \right) \Delta s$$

$$\dot{m} = \rho u \Delta n$$

$$\rho u^2 \frac{\partial \theta}{\partial s} = - \frac{\partial p}{\partial n}$$

hence

$$\boxed{\rho u^2 \frac{\partial \theta}{\partial s} = - \frac{\partial p}{\partial n}}$$

$$\text{or } \frac{\rho u^2}{R} = - \frac{\partial p}{\partial n}$$

← direction change is direct func of normal pressure using (1) (as expected)

Full set of governing equations

Euler-s
equation

$$\rho u \frac{\partial u}{\partial s} = - \frac{\partial p}{\partial s}$$

(1)

} compressible
flow relation

Euler-n
equation

$$\rho u^2 \frac{\partial \theta}{\partial s} = - \frac{\partial p}{\partial n}$$

(2)

} derived
recently

Mass
conservation

$$\frac{\partial \theta}{\partial n} + \frac{1}{u} \frac{\partial u}{\partial s} + \frac{1}{\rho} \frac{\partial \rho}{\partial s} = 0$$

(3)

Energy
conservation

$$h + \frac{u^2}{2} = h_0$$

(4)

} obvious
(stagnation
enthalpy)

Entropy

Recall Gibbs' equation
(with capital S for entropy)

specific

$$T dS = dh - \frac{dp}{\rho}$$

Substitute for enthalpy from
energy equation

$$T dS = dh_0 - u du - \frac{dp}{\rho}$$

Along our streamtube (using
s-momentum)

$$T \frac{dS}{ds} = \frac{dh_0}{ds} - \left(u \frac{\partial u}{\partial s} + \frac{1}{\rho} \frac{\partial p}{\partial s} \right) = 0$$

Handwritten red annotations:
- A bracket under $\frac{dh_0}{ds}$ with an arrow pointing to $=0$.
- A bracket under $\left(u \frac{\partial u}{\partial s} + \frac{1}{\rho} \frac{\partial p}{\partial s} \right)$ with an arrow pointing to $=0$.
- To the right, $\therefore \frac{dS}{ds} = 0$ is written in red.
- A red arrow points from this result down to the concluding text.

i.e. entropy is constant along an (adiabatic) streamtube

Entropy normal to streamline

Along our streamtube

$$T \frac{dS}{dn} = \frac{dh_0}{dn} - \left(u \frac{\partial u}{\partial n} + \frac{1}{\rho} \frac{\partial p}{\partial n} \right)$$

Using Euler n-equation

$$= \frac{dh_0}{dn} - \left(u \frac{\partial u}{\partial n} - u^2 \frac{\partial \theta}{\partial s} \right)$$

from derived equation
at start

$$T \frac{dS}{dn} = \frac{dh_0}{dn} + u\omega$$

with the vorticity
given by

$$\omega = u \frac{\partial \theta}{\partial s} - \frac{\partial u}{\partial n}$$

can be non zero
(eg exhaust)

Entropy, vorticity and stagnation pressure

$$T \frac{dS}{dn} = \frac{dh_0}{dn} + u\omega$$

vorticity can be created from a curved shock wave!

For constant h_0 , the vorticity is directly related to the entropy gradient

Also (from lecture 1.1)

$$S_2 - S_1 = c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{p_2}{p_1}\right)$$

For constant h_0 , let states 1 and 2 be brought to rest isentropically

$$S_2 - S_1 = -R \ln\left(\frac{p_{0,2}}{p_{0,1}}\right)$$

i.e. entropy gradient is also linked to changes in *stagnation* pressure

Crocco's theorem

Special form
(derived here)

$$T \frac{dS}{dn} = \frac{dh_0}{dn} + u\omega$$

General form
(unsteady)

$$T\nabla S + \mathbf{u} \times \boldsymbol{\omega} = \nabla h_0 + \frac{\partial \mathbf{u}}{\partial t}$$

slippery doesn't prove

If h_0 is constant everywhere and the flow is irrotational and steady, then the entropy is constant everywhere.

This is called **homentropic** flow

In a homentropic flow, the isentropic flow relation holds everywhere (not just along streamlines), i.e.

$$\frac{p}{p_0} = \left(\frac{\rho}{\rho_0} \right)^\gamma$$

You should now have an appreciation of:

- A theoretical framework which links the effects of entropy, vorticity and stagnation pressure in compressible flows
 - For example the link between entropy and stagnation pressure is the basis for discussion of 'loss' in turbomachinery
- A deeper understanding of what is meant by the homentropic flow assumption that we use to develop MoC