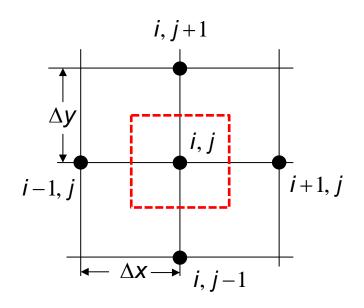
SESA3029 Aerothermodynamics

Lecture 5.7

Finite difference methods for the heat diffusion equation in 2D, example case

2D Discretisation



2D heat diffusion equation, stationary, no heat source

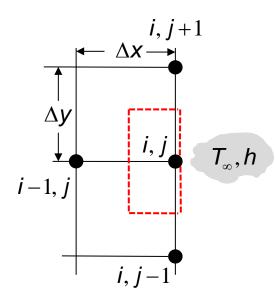
on mesh with uniform **but different** stepsizes $\Delta x, \Delta y$

$$\frac{\partial^2 \mathbf{T}}{\partial \mathbf{x}^2} + \frac{\partial^2 \mathbf{T}}{\partial \mathbf{y}^2} = 0$$

$$\frac{\partial T/\partial \mathbf{x}\Big|_{_{i+1/2,j}} - \partial T/\partial \mathbf{x}\Big|_{_{i-1/2,j}}}{\Delta \mathbf{x}} + \frac{\partial T/\partial \mathbf{y}\Big|_{_{i,j+1/2}} - \partial T/\partial \mathbf{y}\Big|_{_{i,j-1/2}}}{\Delta \mathbf{y}} = 0$$

$$\frac{1}{\Delta \mathbf{x}^{2}} \left(T_{i+1,j} - 2T_{i,j} + T_{i-1,j} \right) + \frac{1}{\Delta \mathbf{y}^{2}} \left(T_{i,j+1} - 2T_{i,j} + T_{i,j-1} \right) = 0$$
 (5)

2D Boundary conditions



Node at a plane surface with convection

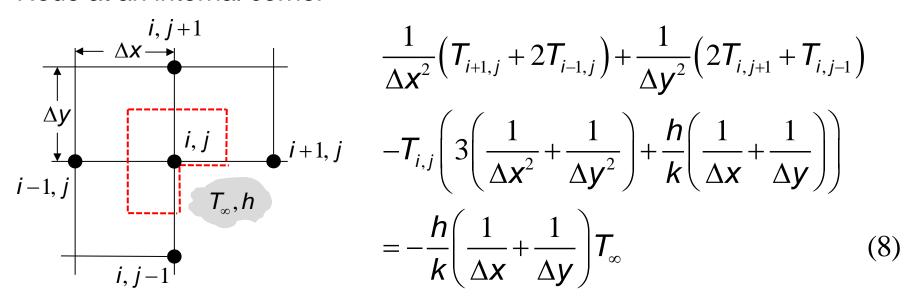
$$\frac{2}{\Delta x^{2}} T_{i-1,j} + \frac{1}{\Delta y^{2}} \left(T_{i,j+1} + T_{i,j-1} \right) -2T_{i,j} \left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{h}{k\Delta x} \right) = -\frac{2h}{k\Delta x} T_{\infty} \tag{6}$$

$$\frac{1}{\Delta x^{2}} \left(T_{i+1,j} + T_{i-1,j} \right) + \frac{1}{\Delta x^{2}} \left(T_{i+1,j} + T_{i+1,j} \right) + \frac{1}{\Delta x^{2}} \left(T_$$

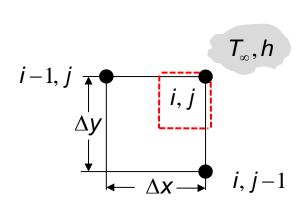
$$\frac{1}{\Delta x^{2}} \left(T_{i+1,j} + T_{i-1,j} \right) + \frac{2}{\Delta y^{2}} T_{i,j+1}$$

$$-2T_{i,j} \left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{h}{k\Delta y} \right) = -\frac{2h}{k\Delta y} T_{\infty} \tag{7}$$

Node at an internal corner



Node at an external corner with convection

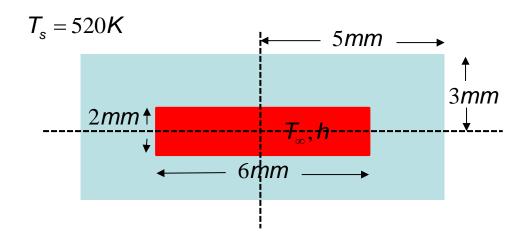


$$\frac{1}{\Delta x^{2}} T_{i-1,j} + \frac{1}{\Delta y^{2}} T_{i,j-1}$$

$$-T_{i,j} \left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{h}{k} \left(\frac{1}{\Delta x} + \frac{1}{\Delta y} \right) \right)$$

$$= -\frac{h}{k} \left(\frac{1}{\Delta x} + \frac{1}{\Delta y} \right) T_{\infty}$$
(9)

2D heat diffusion example



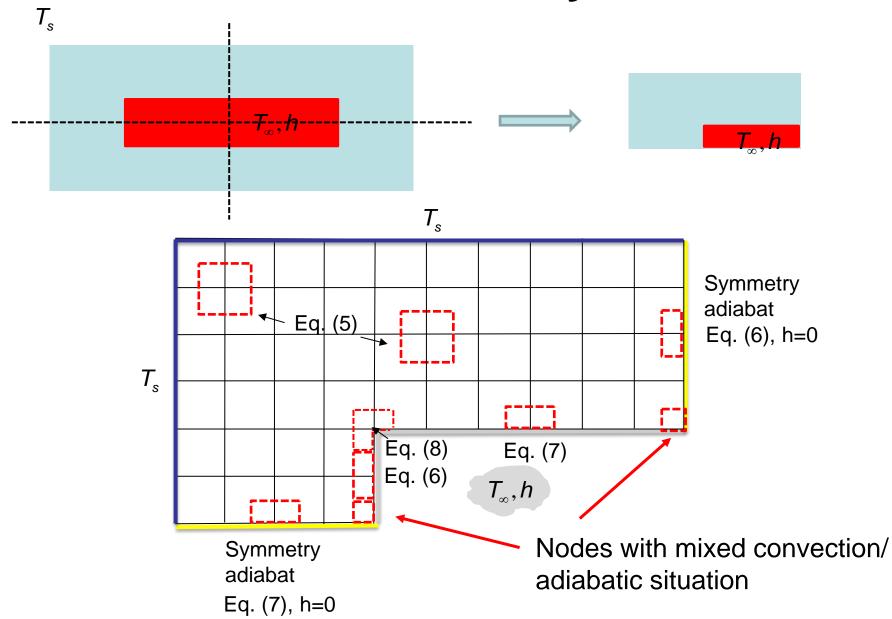
A rectangular glas channel (k = 2 W/(mK)) with 2mm wall thickness is exposed to an outside constant temperature $T_s = 520 \text{K}$.

The channel is water cooled at constant temperature $T_{\infty} = 300 K$ with a constant convective heat transfer coefficient $h = 150 \text{ W/(m}^2 \text{K})$.

Approximate the temperature field in the walls and study the convergence of the predicted values at selected channel locations.

See also Bergman et al., page 251ff / Incropera et al. page 231ff (different values)

Domain and boundary conditions



Node at an external corner with mixed convection

$$\frac{1}{\Delta x^{2}} T_{i-1,j} + \frac{1}{\Delta y^{2}} T_{i,j+1}$$

$$-T_{i,j} \left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{1}{k} \left(\frac{h_{1}}{\Delta x} + \frac{h_{2}}{\Delta y} \right) \right)$$

$$= -\frac{1}{k} \left(\frac{h_{1}}{\Delta x} + \frac{h_{2}}{\Delta y} \right) T_{\infty}$$

Lower left: $h_1 = h$, $h_2 = 0$

$$\frac{1}{\Delta x^{2}} T_{i-1,j} + \frac{1}{\Delta y^{2}} T_{i,j+1} - T_{i,j} \left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{h}{k \Delta x} \right) = -\frac{h}{k \Delta x} T_{\infty}$$

Upper right : $h_1 = 0$, $h_2 = h$

$$\frac{1}{\Delta x^{2}}T_{i-1,j} + \frac{1}{\Delta y^{2}}T_{i,j+1} - T_{i,j}\left(\frac{1}{\Delta x^{2}} + \frac{1}{\Delta y^{2}} + \frac{h}{k\Delta y}\right) = -\frac{h}{k\Delta y}T_{\infty}$$

Matrix organization

Define an index function to transform the 2D solution into a 1D vector:

def index(i,j,Nx):
 return j*Nx+i

Matrix assembly:

```
A[index(i,j,Nx),index(i,j,Nx)]=-2.0*(dx2i+dy2i)
A[index(i,j,Nx),index(i+1,j,Nx)]=dx2i
A[index(i,j,Nx),index(i-1,j,Nx)]=dx2i
A[index(i,j,Nx),index(i,j+1,Nx)]=dy2i
A[index(i,j,Nx),index(i,j-1,Nx)]=dy2i
C[index(i,j,Nx)]=0.0
```