

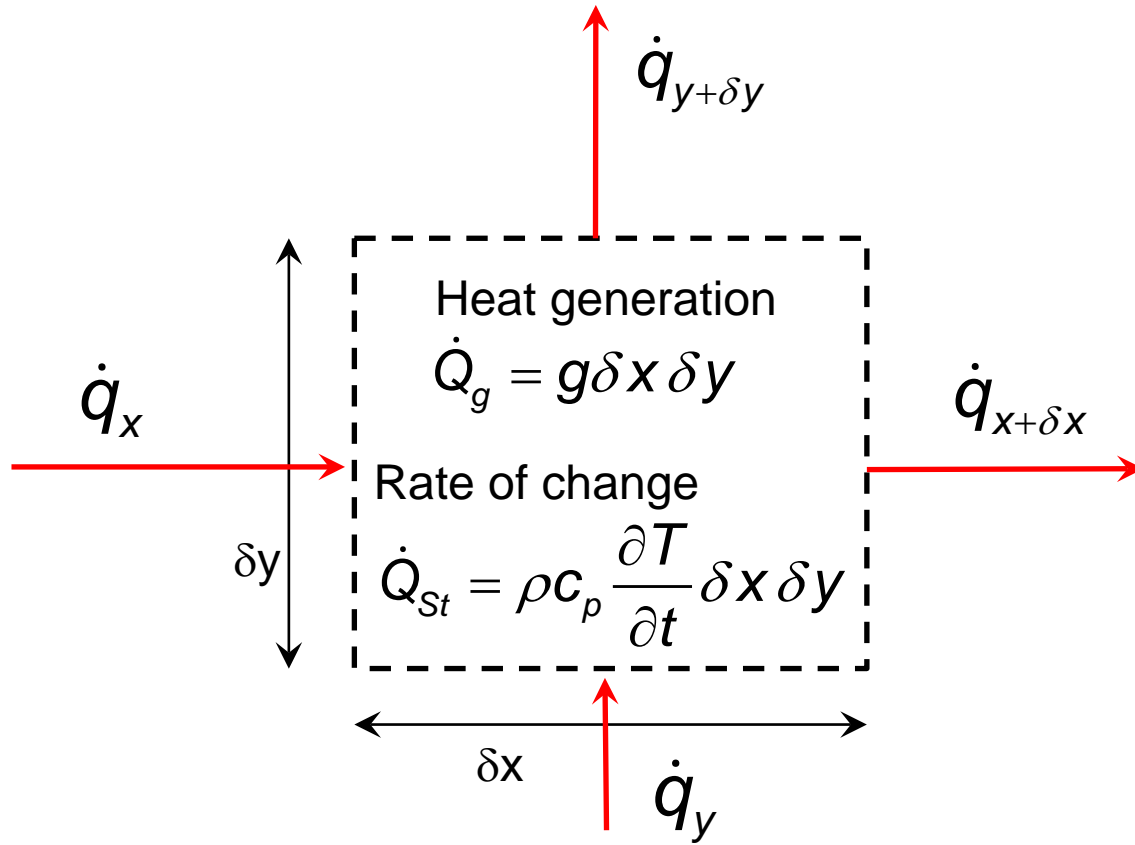
# SESA3029

# Aerothermodynamics

## Lecture 5.5

heat diffusion equation, 1D finite  
difference methods for conduction

# Heat equation



$$\dot{q}_{x+\delta x} = \dot{q}_x + \frac{\partial \dot{q}_x}{\partial x} \delta x$$

$$\dot{q}_{y+\delta y} = \dot{q}_y + \frac{\partial \dot{q}_y}{\partial y} \delta y$$

Energy balance

$$\dot{Q}_{in} - \dot{Q}_{out} + \dot{Q}_g = \dot{Q}_{St}$$

$$\underbrace{(\dot{q}_x - \dot{q}_{x+\delta x})}_{\partial \dot{q}_x} \delta y + \underbrace{(\dot{q}_y - \dot{q}_{y+\delta y})}_{\partial \dot{q}_y} \delta x + g \delta x \delta y = \rho c_p \frac{\partial T}{\partial t} \delta x \delta y$$

$$-\frac{\partial \dot{q}_x}{\partial x} - \frac{\partial \dot{q}_y}{\partial y} + g = \rho c_p \frac{\partial T}{\partial t}$$

↓ ∴  $\partial \gg \partial y$

Use Fourier's law  $\dot{q}_i = -k \frac{\partial T}{\partial x_i}$  to obtain

$$k \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + g = \rho c_p \frac{\partial T}{\partial t}$$

the heat diffusion equation

$$\Delta T + \frac{g}{k} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$$

with thermal diffusivity  $\alpha = \frac{k}{\rho c_p}$

# Typical boundary conditions

For simplicity: 1D case

- Constant temperature:  $T(0, t) = T_s$
- Constant surface heat flux:  $-k \frac{\partial T}{\partial x} \Big|_{x=0} = \dot{q}_x$
- Adiabatic or insulated surface:  $\frac{\partial T}{\partial x} \Big|_{x=0} = 0$
- Convection surface condition:  $-k \frac{\partial T}{\partial x} \Big|_{x=0} = h[T_\infty - T(0, t)]$

# Discretising the heat equation in 1D

1D heat equation, stationary, no heat source  $\frac{\partial^2 T}{\partial x^2} = 0$

on mesh with uniform stepsize  $\Delta x$



The gradients can be approximated as

$$\left. \frac{\partial T}{\partial x} \right|_{i-1/2} \approx \frac{T_i - T_{i-1}}{\Delta x}, \quad \left. \frac{\partial T}{\partial x} \right|_{i+1/2} \approx \frac{T_{i+1} - T_i}{\Delta x}$$

yielding the approximated equation

$$\left. \frac{\partial^2 T}{\partial x^2} \right|_i \approx \frac{\left. \frac{\partial T}{\partial x} \right|_{i+1/2} - \left. \frac{\partial T}{\partial x} \right|_{i-1/2}}{\Delta x} \approx \frac{T_{i+1} - 2T_i + T_{i-1}}{\Delta x^2} = 0$$

second  
you didn't

gradient (assuming linear gradient)  
here

derived numerical  
approximation for  
second derivative  
from some  
arbitrary  
discretised temperature  
line

# Discrete boundary conditions

as is typical, there are issues at the boundary

Constant temperature:  $T_2 - 2T_1 = -T_{sl}$ ,  $-2T_{l-1} + T_{l-2} = -T_{sr}$

Surface convection on the left:  $-k \frac{\partial T}{\partial x} \Big|_0 = h[T_\infty - T_0]$

start of line  
end of line (boundary)  
T<sub>sl</sub> --- T<sub>sr</sub>  
surface touching on left

Using 
$$\frac{\frac{\partial T}{\partial x} \Big|_{1/2} - \frac{\partial T}{\partial x} \Big|_0}{1/2 \Delta x} = \frac{\frac{T_1 - T_0}{\Delta x} - \left( -\frac{h}{k} [T_\infty - T_0] \right)}{1/2 \Delta x} = 0$$

gives 
$$T_1 - \left( 1 + \frac{h \Delta x}{k} \right) T_0 = -\frac{h \Delta x}{k} T_\infty \quad (2)$$

Surface convection on the right (note sign!):  $k \frac{\partial T}{\partial x} \Big|_l = h[T_\infty - T_l]$

gives 
$$T_{l-1} - \left( 1 + \frac{h \Delta x}{k} \right) T_l = -\frac{h \Delta x}{k} T_\infty \quad (3)$$

Adiabatic boundary: Simply use  $h = 0$  in Eqs. (2), (3)

Constant surface heat flux: Use  $h = 0$ ,  $T_\infty = \frac{\dot{q}_x}{h}$  in Eqs. (2), (3)

# Solution process

Equation (1) gives tridiagonal linear systems.  
Constant temperature boundary conditions:

$$\begin{pmatrix} -2 & 1 & 0 & \cdots & 0 \\ 1 & -2 & 1 & 0 & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & 0 & 1 & -2 & 1 \\ 0 & \cdots & 0 & 1 & -2 \end{pmatrix} \begin{pmatrix} T_1 \\ \vdots \\ T_{l-1} \end{pmatrix} = \begin{pmatrix} -T_{sl} \\ 0 \\ \vdots \\ 0 \\ -T_{sr} \end{pmatrix}$$

Surface convection boundary conditions:

$$\begin{pmatrix} -\left(1 + \frac{h_l \Delta x}{k}\right) & 1 & 0 & \cdots & 0 \\ 1 & -2 & 1 & 0 & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & 0 & 1 & -2 & 1 \\ 0 & \cdots & 0 & 1 & -\left(1 + \frac{h_r \Delta x}{k}\right) \end{pmatrix} \begin{pmatrix} T_0 \\ \vdots \\ T_l \end{pmatrix} = \begin{pmatrix} -\frac{h_l \Delta x}{k} T_{\infty l} \\ 0 \\ \vdots \\ 0 \\ -\frac{h_r \Delta x}{k} T_{\infty r} \end{pmatrix}$$

*Handwritten red annotations:*

- A red arrow points from the word "zeros" to the top-right corner of the coefficient matrix, specifically to the zeros in the first row.
- A red arrow points from the word "zeros" to the bottom-left corner of the coefficient matrix, specifically to the zeros in the last row.

# Order of accuracy of finite differences

Finite difference representations are based on Taylor series. For instance

$$T_{i+1} = T_i + \left( \frac{\partial T}{\partial x} \right)_i \Delta x + \left( \frac{\partial^2 T}{\partial x^2} \right)_i \frac{\Delta x^2}{2} + \left( \frac{\partial^3 T}{\partial x^3} \right)_i \frac{\Delta x^3}{3!} + \dots$$

can be rearranged to

$$\left( \frac{\partial T}{\partial x} \right)_i = \underbrace{\frac{T_{i+1} - T_i}{\Delta x}}_{\text{Finite difference}} - \underbrace{\left( \frac{\partial^2 T}{\partial x^2} \right)_i \frac{\Delta x}{2} - \left( \frac{\partial^3 T}{\partial x^3} \right)_i \frac{\Delta x^2}{3!} + \dots}_{\text{Truncation error}} = \frac{T_{i+1} - T_i}{\Delta x} + O(\Delta x)$$

Similarly, we can show that

$$\left( \frac{\partial^2 T}{\partial x^2} \right)_i = \frac{T_{i+1} - 2T_i + T_{i-1}}{\Delta x^2} + O(\Delta x^2)$$

second order accuracy



# Accuracy of basic scheme by Taylor series

Error  $\sim \Delta x^p$       Grid size  $\Delta x$ , order of accuracy  $p$

1<sup>st</sup> order: error decreases by a factor of 2 for twice the number of grid points

2<sup>nd</sup> order: error decreases by a factor of 4 for twice the number of grid points

4<sup>th</sup> order: error decreases by a factor of 16 for twice the number of grid points

Fewer grid points needed for high order methods for a given accuracy, but does depend on the function being approximated