

EE5103R/ME5403 Computer Control Systems: Homework #3 Solution

Semester 1 Y2021/2022

Q1

The controller is required to be in form of

$$R(q)u(k) = T(q)u_c(k) - S(q)y(k), \quad (0.1)$$

whose z transform is

$$U(z) = \frac{T(z)}{R(z)}U_c(z) - \frac{S(z)}{R(z)}Y(z). \quad (0.2)$$

Therefore, the block diagram for this control system is

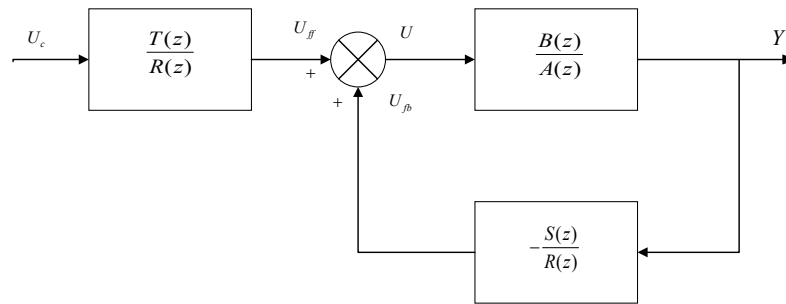


Figure 1 Block diagram of the closed-loop system.

And we have

$$\frac{B(z)}{A(z)} = H(z) = \frac{z + 0.8}{z^2 - 1.5z + 0.5} \quad (0.3)$$

By designing the above two degree-of-freedom controller, we aim to obtain a closed-loop system with the following characteristic polynomial

$$A_m(z) = z^2 - 1.8z + 0.9 \quad (0.4)$$

To match the specified closed-loop denominator (poles), we mainly need to manipulate the feedback term $S(z)/R(z)$ since the closed-loop transfer function caused from the control law (1.2) is computed to be

$$\frac{Y(z)}{U_c(z)} = \frac{B(z)T(z)}{A(z)R(z) + B(z)S(z)} \triangleq \frac{B(z)T(z)}{A_{cl}(z)}. \quad (0.5)$$

Now the question is: what is the closed-loop polynomial $A_{cl}(z)$ in (1.5) ? Since we are

required to achieve $A_m(z)$ as the closed-loop denominator at the end, $A_{cl}(z)$ must originally be in some form of $A_{cl}(z) = A_o(z)A_m(z)$ and then reduced to $A_m(z)$ by cancelling $A_o(z)$. Now the remaining task is to determine $A_o(z)$. Generally, $A_o(z)$ may stem simply from polynomial order matching (when $A_{cl}(z)$ has a higher order than the required $A_m(z)$) or from additional requirements like zero cancellation, disturbance rejection etc. This is a basic principle for model matching of lecture 5, which applies to all the problems in this assignment.

a)

If the process zero is cancelled, that is, there is no $B(z)$ in the final form of (1.5), then it can be inferred that the closed-loop denominator before cancellation is

$$A_{cl}(z) = B(z)A_m(z) = (z + 0.8)A_m(z) = (z + 0.8)(z^2 - 1.8z + 0.9) \quad (0.6)$$

Obviously, we have $\deg(R(z)) = \deg(A_{cl}(z)) - \deg(A(z)) = 1$. Therefore, the general form of $R(z) = z + r_1$. Additionally, since there is a factor $(z + 0.8)$ in $A_{cl}(z)$, the $R(z)$ must be in this form $R(z) = (z + 0.8)$. Further, suppose $S(z) = s_0z + s_1$ due to $\deg(S(z)) \leq \deg(R(z))$, then we have

$$\begin{aligned} A_{cl}(z) &= A(z)R(z) + B(z)S(z) \\ &= (z^2 - 1.5z + 0.5)(z + 0.8) + (z + 0.8)(s_0z + s_1) \\ &= (z + 0.8)[z^2 + (s_0 - 1.5)z + 0.5 + s_1]. \end{aligned} \quad (0.7)$$

From (1.6) and (1.7), it can be inferred that

$$z^2 + (s_0 - 1.5)z + 0.5 + s_1 = z^2 - 1.8z + 0.9 \quad (0.8)$$

And it can be solved as $s_0 = -0.3$, $s_1 = 0.4$. Hence

$$\frac{S(z)}{R(z)} = \frac{-0.3z + 0.4}{z + 0.8} \quad (0.9)$$

After we get $R(z)$ and $S(z)$, it is time now to choose $T(z)$. The requirement is that the steady state gain from $U_c(z)$ to $Y(z)$ is one. For the closed-loop transfer function (1.5), the steady state gain can be computed by $Y(1)/U_c(1)$. We can assume the simplest form for $T(z) = t_0$. Then, we have

$$\frac{Y(1)}{U_c(1)} = \frac{B(z)T(z)}{A_{cl}(z)} \Big|_{z=1} = \frac{t_0}{1 - 1.8 + 0.9} = 1, \quad (0.10)$$

which further gives

$$t_0 = 0.1. \quad (0.11)$$

Thus, $T(z) = 0.1.$ (0.12)

The whole controller is designed as follows:

$$\begin{aligned} R(z) &= z + 0.8 \\ S(z) &= -0.3z + 0.4 \\ T(z) &= 0.1 \\ U(z) &= \frac{T(z)}{R(z)} U_c(z) - \frac{S(z)}{R(z)} Y(z) \end{aligned} \quad (0.13)$$

In this case, from (1.5), the closed-loop transfer function is

$$G(z) = \frac{B_m(z)}{A_m(z)} = \frac{Y(z)}{U_c(z)} = \frac{0.1}{z^2 - 1.8z + 0.9}. \quad (0.14)$$

The controller is:

$$u(k) = 0.2188u(k-1) + 0.0556u_c(k) + 0.0812y(k) - 0.1368y(k-1)$$

b)

If the process zero is not cancelled, based on the above analysis, the order of the closed-loop characteristic polynomial is $\deg(A_{cl}(z)) = \deg(A(z)R(z) + B(z)S(z)) > \deg(A(z)) = 2$ since the order of $R(z)$ is at least one. Then, because the desired one is just $\deg(A_m(z)) = 2$, we have to include another auxiliary factor $A_o(z)$ such that the closed-loop characteristic polynomial can be matched as

$$A_{cl}(z) = A_o(z)A_m(z) = zA_m(z) = z(z^2 - 1.8z + 0.9) \quad (0.15)$$

Here we choose $A_o(z) = z$ because we want the lowest order one whose poles are all at the origin.

Now it can be inferred that $\deg(R(z)) = \deg(A_{cl}(z)) - \deg(A(z)) = 1$. Similarly, we shall assume $R(z) = r_0z + r_1$, $S(z) = s_0z + s_1$. Then, we notice that the leading coefficient of both $A_{cl}(z)$ and $A(z)$ is 1, which indicates that $r_0 = 1$ since there exists $A(z)R(z) + B(z)S(z) = A_{cl}(z)$. Now we have $R(z) = z + r_1$.

Then, note that the additional $A_o(z)$ in (1.15) must be cancelled in the final form because the

$A_m(z)$. This is the responsibility of $T(z)$,

which is designed to be $T(z) = t_0 z$ since $T(z)$ should include the factor $A_o(z) = z$.

Finally, the closed-loop transfer function is

$$G(z) = \frac{Y(z)}{U_c(z)} = \frac{B(z)T(z)}{A(z)R(z) + B(z)S(z)} = \frac{t_0 z(z + 0.8)}{(z^2 - 1.5z + 0.5)(z + r_1) + (z + 0.8)(s_0 z + s_1)} \quad (0.16)$$

$$= \frac{t_0 z(z + 0.8)}{z^3 + (r_1 + s_0 - 1.5)z^2 + (0.5 - 1.5r_1 + s_1 + 0.8s_0)z + 0.5r_1 + 0.8s_1}$$

According to (1.15) and (1.16), we have

$$z^3 + (r_1 + s_0 - 1.5)z^2 + (0.5 - 1.5r_1 + s_1 + 0.8s_0)z + 0.5r_1 + 0.8s_1 = z(z^2 - 1.8z + 0.9) \quad (0.17)$$

Thus, the following equations are derived

$$\begin{cases} r_1 + s_0 - 1.5 = -1.8 \\ 0.5 - 1.5r_1 + s_1 + 0.8s_0 = 0.9 \\ 0.5r_1 + 0.8s_1 = 0 \end{cases} \quad (0.18)$$

whose solutions is

$$\begin{cases} r_1 = -0.2188 \\ s_0 = -0.0812 \\ s_1 = 0.1368 \end{cases} \quad (0.19)$$

And then from (1.16) and (1.17) we get

$$G(z) = \frac{t_0(z + 0.8)}{z^2 - 1.8z + 0.9} \quad (0.20)$$

Besides, it is required that $G(1) = 1$ for a unit steady-state gain:

$$G(1) = \frac{1.8t_0}{0.1} = 1 \Rightarrow t_0 = \frac{1}{18} \approx 0.0556 \quad (0.21)$$

After all, the controller is designed as follows

$$\begin{aligned} R(z) &= z - 0.2188 \\ S(z) &= -0.0812z + 0.1368 \\ T(z) &= \frac{1}{18}z \\ U(z) &= \frac{T(z)}{R(z)}U_c(z) - \frac{S(z)}{R(z)}Y(z) \end{aligned} \quad (0.22)$$

And the closed-loop transfer function in this case is

$$G(z) = \frac{Y(z)}{U_c(z)} = \frac{z + 0.8}{18(z^2 - 1.8z + 0.9)} \quad (0.23)$$

The controller is:

$$u(k) = 0.2188u(k-1) + 0.0556u_c(k) + 0.0812y(k) - 0.1368y(k-1)$$

c)

After we finished the design, a Simulink model can be easily built according to Figure 1, which is exhibited in Figure 2. Run the simulation with a sampling period of 1ms.

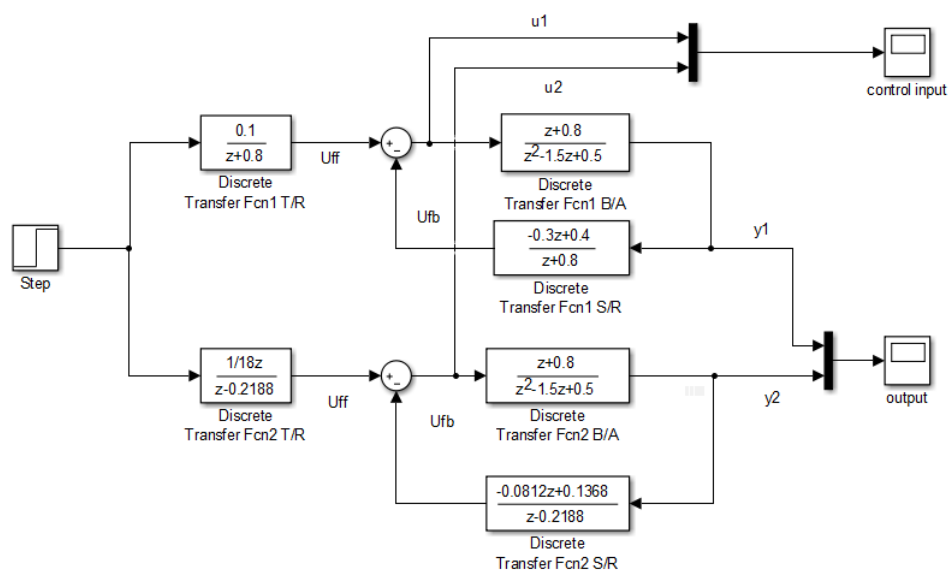


Figure 2 Simulink model.

The closed-loop output and control input profiles are shown in Figure 3 and Figure 4 respectively.

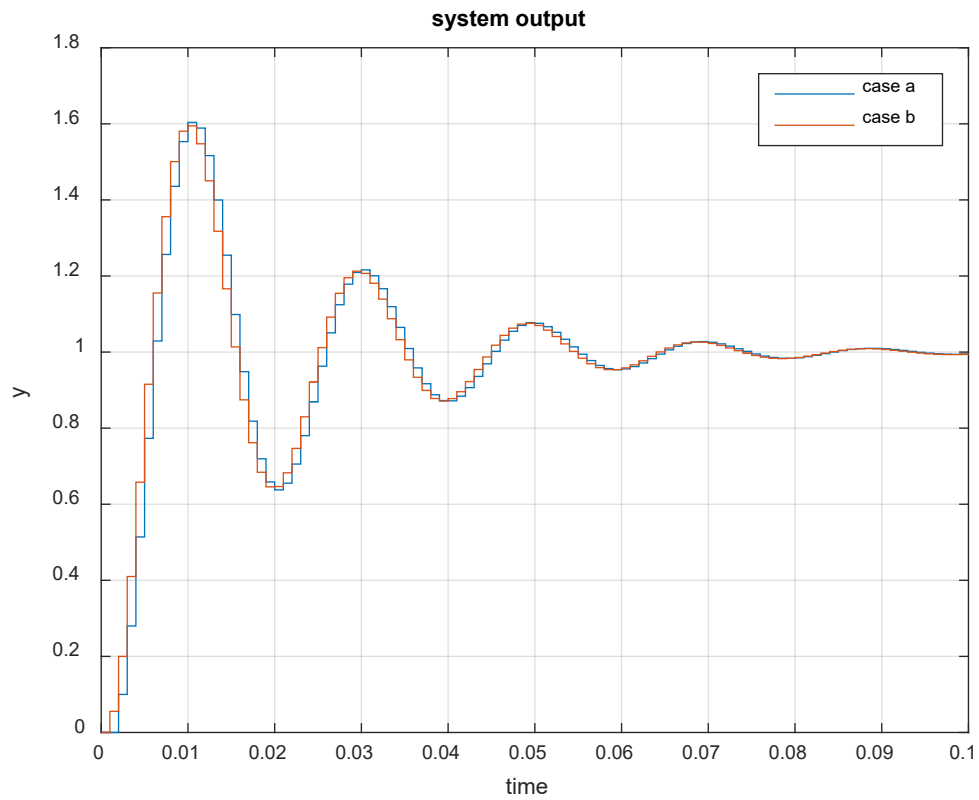


Figure 3 System output for the two cases.

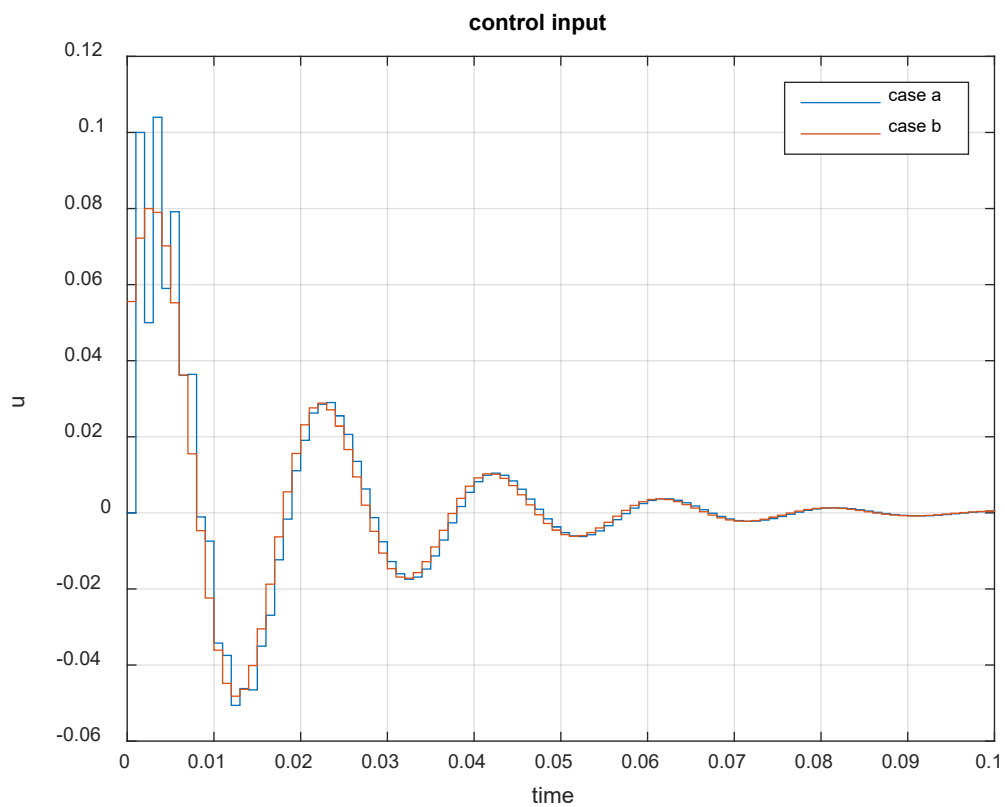


Figure 4 Control input signals for the two cases.

As can be seen, the outputs in the two cases are quite similar. However, in Figure 4 it is noted that the control input signal for case 1 (with zero cancellation) is more seriously oscillating before arriving at the steady state. Besides, the control input for case 1 has a slightly larger magnitude. What is the reason for such behavior of the control input? Since we are studying the control input $u(k)$, we first write down the transfer function from the command signal $u_c(k)$ to the control input $u(k)$ as follows (for both the two cases)

$$\begin{cases} U(z) = \frac{T(z)}{R(z)} U_c(z) - \frac{S(z)}{R(z)} Y(z) \\ \frac{Y(z)}{U(z)} = \frac{B(z)}{A(z)} \end{cases} \Rightarrow H_u(z) = \frac{U(z)}{U_c(z)} = \frac{A(z)T(z)}{A(z)R(z) + B(z)S(z)} = \frac{A(z)T(z)}{A_{cl}(z)}. \quad (0.24)$$

By inserting $A_{cl}(z)$ from (1.6) and (1.15) into the above equation for case 1 and case 2 respectively, we get $H_u(z)$ for the two cases:

$$\begin{aligned} \text{case } a \quad H_u(z) &= \frac{0.1(z^2 - 1.5z + 0.5)}{(z + 0.8)(z^2 - 1.8z + 0.9)} \\ \text{case } b \quad H_u(z) &= \frac{0.0556z(z^2 - 1.5z + 0.5)}{z(z^2 - 1.8z + 0.9)} = \frac{0.0556(z^2 - 1.5z + 0.5)}{z^2 - 1.8z + 0.9} \end{aligned} \quad (0.25)$$

The poles of the above two transfer functions differ from each other, as follows

$$\begin{aligned} \text{case } a \text{ poles} & \quad [-0.8, \quad 0.9 + 0.3i, \quad 0.9 - 0.3i] \\ \text{case } b \text{ poles} & \quad [0.9 + 0.3i, \quad 0.9 - 0.3i] \end{aligned} \quad (0.26)$$

The additional pole of case a has a negative real part. We know that a pole with a negative real part in z transfer function would cause oscillation and this is the main reason for the above phenomenon we have observed.

In practice, we would favor a smoother control input with a smaller magnitude, which is easier to implement and more cost-friendly. Hence with the comparison, the second controller (without zero cancellation) is preferred.

Q2

a)

Applying z -transform to the state-space representation with zero initial conditions yields:

$$\begin{cases} zX(z) = \Phi X(z) + \Gamma U(z) + \Phi_{xv} V(z) \\ Y(z) = CX(z) \end{cases}$$

Where $\Phi = \begin{bmatrix} 0.5 & 1 \\ 0.5 & 0.8 \end{bmatrix}$, $\Gamma = \begin{bmatrix} 0.2 \\ 0.1 \end{bmatrix}$, $\Phi_{xv} = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$, $C = \begin{bmatrix} 1 & 0 \end{bmatrix}$.

Then we can get the transfer function

$$\begin{aligned} Y(z) &= C(zI - \Phi)^{-1}\Gamma U(z) + C(zI - \Phi)^{-1}\Phi_{xv}V(z) \\ &= G_u U(z) + G_v V(z) \end{aligned}$$

By superposition principle, we can firstly assume the disturbance to be zero, so the transfer function from input to output is

$$G_u(z) = \frac{Y(z)}{U(z)} = C(zI - \Phi)^{-1}\Gamma = \frac{0.2z - 0.06}{z^2 - 1.3z - 0.1} \triangleq \frac{B(z)}{A(z)}$$

Secondly, we will assume the input is zero, so the transfer function from the disturbance to the output is

$$G_v(z) = \frac{Y(z)}{V(z)} = C(zI - \Phi)^{-1}\Phi_{xv} = \frac{z - 0.8}{z^2 - 1.3z - 0.1} \triangleq \frac{B_v(z)}{A(z)}$$

Then the disturbance rejection controller can be designed with output feedback as shown in Figure 5.

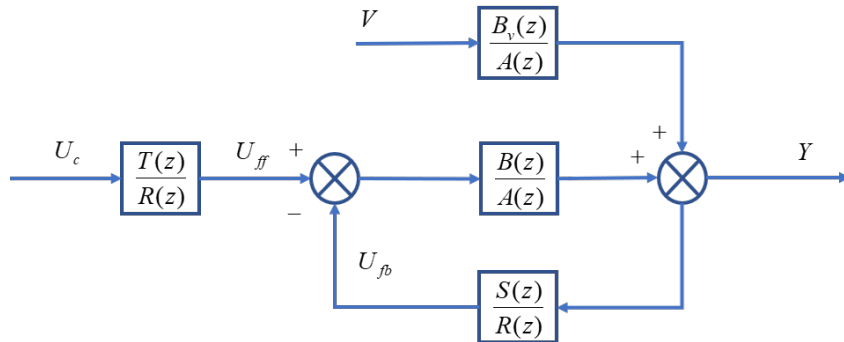


Figure 5 Diagram graph of the control system.

Assuming the $U_c(z) = 0$, we will get the closed-loop transfer function between the disturbance and the output:

$$H_v(z) = \frac{Y(z)}{V(z)} = \frac{B_v(z)R(z)}{A(z)R(z) + B(z)S(z)} \quad (0.27)$$

To eliminate the disturbance effect at steady state, we require the steady state gain of $H_v(z)$ to be zero, that is, $H_v(1) = 0$. This further request $R(z)$ to contain a factor $(z - 1)$. We suppose that $R(z) = (z - 1)(z + r)$ and $S(z) = s_0 z^2 + s_1 z + s_2$ to match the desired control performance. Since the problem did not give the desired closed-loop characteristic polynomial, we can choose the

simplest one as $A_m(z) = z^4$. Therefore to match $A_{cl}(z) = A_m(z)$,

$$\begin{aligned} A_{cl}(z) &= A(z)R(z) + B(z)S(z) \\ &= z^4 + (-2.3 + r_1 + 0.2s_0)z^3 + (1.2 - 2.3r_1 - 0.06s_0 + 0.2s_1)z^2 \\ &\quad + (0.1 + 1.2r_1 - 0.06s_1 + 0.2s_2)z + (0.1r_1 - 0.06s_2) \\ &= A_m(z) = z^4 \end{aligned}$$

We will have

$$\begin{cases} -2.3 + r_1 + 0.2s_0 = 0 \\ 1.2 - 2.3r_1 - 0.06s_0 + 0.2s_1 = 0 \\ 0.1 + 1.2r_1 - 0.06s_1 + 0.2s_2 = 0 \\ 0.1r_1 - 0.06s_2 = 0 \end{cases} \Rightarrow \begin{cases} r_1 = -0.2711 \\ s_0 = 12.8554 \\ s_1 = -5.2608 \\ s_2 = -0.4518 \end{cases}$$

Choose $T(z)$ to guarantee the steady state gain to be 1, so choose $T(z) = t_0 z^2$

$$H_{cl}(z)\Big|_{z=1} = \frac{B(z)T(z)}{A_{cl}(z)} = \frac{t_0 z^2 (0.2z - 0.06)}{z^4} \Big|_{z=1} = 1 \Rightarrow t_0 = 7.1428 \quad (0.28)$$

Therefore, the controller is

$$\boxed{\begin{cases} R(z) = (z-1)(z-0.2711) \\ S(z) = 12.8554z^2 - 5.2608z - 0.4518 \\ T(z) = 7.1428z^2 \end{cases}} \quad (0.29)$$

So, the desired controller is:

$$\begin{aligned} u(k) &= 7.1428u_c(k) + 1.2711u(k-1) - 0.2711u(k-2) - 12.8554y(k) \\ &\quad + 5.2608y(k-1) + 0.4518y(k-2), \quad k \geq 2 \end{aligned}$$

Remark:

You can also choose $R(z)$ as $R(z)=(z-1)B(z)$ for both zero cancellation and disturbance rejection, then calculate $S(z)$ and $T(z)$ accordingly.

b)

For this problem, the polynomial approach is simpler because we just need to choose appropriate R, S and T to make the closed loop system stable and to eliminate constant disturbance. There is no need to build the observer.

Q3

a)

The structure of the closed-loop system is the same with the one in Figure 1. The obtained closed-loop transfer function is

$$\frac{Y(z)}{U_c(z)} = \frac{B(z)T(z)}{A(z)R(z)+B(z)S(z)} \triangleq \frac{B(z)T(z)}{A_{cl}(z)}.$$

Comparing the transfer function with the reference model $A_m(z)$, we can see that the factor $B(z) = z - 0.5$ disappears in the reference model, which implies zero cancellation. Therefore, $R(z)$ must be comprised of $B(z)$ and possibly another factor.

Furthermore, here we should pay attention to another requirement on constant disturbance rejection. Assume the disturbance is $v(k)$. Then, from Figure 5, the transfer function from the disturbance v to the output y is resolved to be

$$G_v(z) = \frac{Y(z)}{V(z)} = \frac{B(z)R(z)}{A(z)R(z) + B(z)S(z)}$$

To eliminate the disturbance effect at steady state, we require the steady state gain of $G_v(z)$ to be zero, that is, $G_v(1)=0$. This further request $R(z)$ to contain a factor $(z-1)$. Therefore, $R(z)$ should be the form $(z-1)(z-0.5)$. Similarly, since there exists zero cancellation, the closed-loop polynomial is in the form of $A_{cl}(z)=(z-0.5)A_m(z)A_o(z)$.

Now what is $A_o(z)$? We know that if we choose $R(z) = (z-1)(z-0.5)$, then the general form of $S(z)$ is $S(z) = s_0z^2 + s_1z + s_2$. Since $A(z)R(z)$ has a degree of 4, we may choose $A_o(z) = z$. Now we have

$$A(z)R(z) + B(z)S(z) = (z^2 - 1)(z - 1)(z - 0.5) + (z - 0.5)(s_0z^2 + s_1z + s_2)$$

$$A_{cl}(z) = (z - 0.5)A_m(z)A_o(z) = (z - 0.5)z^3$$

To match corresponding coefficients, we can solve $S(z) = z^2 + z - 1$.

Now the closed-loop transfer function is

$$\frac{Y(z)}{U_c(z)} = \frac{B(z)T(z)}{A(z)R(z) + B(z)S(z)} = \frac{T(z)}{z^3}$$

To eliminate the additional $A_o(z)$ and get $H_m(z)$, it is obvious that $T(z) = z$.

Therefore, the controller is

$$\begin{cases} R(z) = (z-1)(z-0.5) = z^2 - 1.5z + 0.5 \\ S(z) = z^2 + z - 1 \\ T(z) = z \end{cases}$$

Therefore, the controller is

$$u(k) = 1.5u(k-1) - 0.5u(k-2) + u_c(k-1) - y(k) - y(k-1) + y(k-2), \quad k \geq 2$$

b)

The closed-loop transfer function is

$$G(z) = \frac{Y(z)}{U_c(z)} = \frac{B(z)R(z)}{A(z)R(z) + B(z)S(z)} H_{ff}(z). \quad (0.30)$$

The closed-loop transfer function from the disturbance to the output is still

$$G_v(z) = \frac{Y(z)}{V(z)} = \frac{B_v(z)R(z)}{A(z)R(z) + B(z)S(z)}. \quad (0.31)$$

To reject constant disturbance, we need $R(z)$ involves a $(z-1)$ factor. If $R(z)$ is chosen to be of degree one, then we have $S(z) = s_0z + s_1$. Since the degree of $A(z)R(z)$ is 3, to match the closed-loop polynomials there are 3 equations (the leading coefficient is fixed to be 1) but only 2 unknowns s_0 and s_1 , which usually has no solutions. Therefore, $R(z)$ cannot be 1st order and $R(z)$ should have a degree of at least 2. Supposing $R(z) = (z-r_1)(z-1)$ and $S(z) = s_0z^2 + s_1z + s_2$, we can get

$$A(z)R(z) + B(z)S(z) = (z^2-1)(z-r_1)(z-1) + (z-0.5)(s_0z^2 + s_1z + s_2) \quad (0.32)$$

Since $A_m(z) = z^2$ is only second order, we need another term $A_0(z) = z^2$ to match the equation, and get

$$A_{cl}(z) = A_m(z)A_0(z) = z^4 \quad (0.33)$$

Comparing (1.32) and (1.33), we obtain

$$\begin{cases} -1 - r_1 + s_0 = 0 \\ -1 + r_1 - 0.5s_0 + s_1 = 0 \\ 1 + r_1 - 0.5s_1 + s_2 = 0 \\ -r_1 - 0.5s_2 = 0 \end{cases} \Rightarrow \begin{cases} r_1 = 0.3333 \\ s_0 = 1.3333 \\ s_1 = 1.3333 \\ s_2 = -0.6667 \end{cases} \quad (0.34)$$

Therefore, the feedback controller is

$$\boxed{\frac{S(z)}{R(z)} = \frac{1.3333z^2 + 1.3333z - 0.6667}{(z - 0.3333)(z - 1)}} \quad (0.35)$$

Now we have $R(z)$ and $S(z)$. From (1.30) we further obtain

$$\frac{B(z)R(z)}{A(z)R(z) + B(z)S(z)} H_{ff}(z) = \frac{B(z)R(z)}{A_{cl}(z)} H_{ff}(z) \quad (0.36)$$

To match the closed-loop transfer function (1.36) with the reference model $H_m(z) = B_m(z)/A_m(z)$, we get

$$\frac{B(z)R(z)}{A_{cl}(z)} H_{ff}(z) = \frac{B_m(z)}{A_m(z)} \Rightarrow H_{ff}(z) = \frac{B_m(z)A_{cl}(z)}{A_m(z)B(z)R(z)} = \frac{B_m(z)A_0(z)}{B(z)R(z)} \quad (0.37)$$

Inserting the relevant terms into (1.37) gives

$$\boxed{H_{ff}(z) = \frac{z^2}{(z - 0.5)(z - 0.3333)(z - 1)}} \quad (0.38)$$

Therefore, the two-degree of freedom controller has the form of,

$$U(z) = \frac{z^2}{(z - 0.5)(z - 1)(z - 0.3333)} U_c(z) - \frac{1.3333z^2 + 1.3333z - 0.6667}{(z - 1)(z - 0.3333)} Y(z)$$

Note:

1. For a second thought, you may find that if we keep the $R(z)$ and $S(z)$ in the solution to question a) unchanged and let $H_{ff}(z) = T(z)/R(z)$, we can also meet the requirement. That is to say, the control configuration in a) is exactly a special case of b). However, the $H_{ff}(z)$ structure in b) will give you more freedom since there is no requirement on its denominator form and it is a more general control configuration. For exercise and examination purpose, you should follow different procedures to finish the design in these two cases, although a) can be regarded as a special case of b). In this assignment, if you simply write $H_{ff}(z) = T(z)/R(z)$ by reusing the result obtained in a), you will only get 1 point.
2. In case b), $R(z)$ is only responsible for disturbance rejection since $H_{ff}(z)$ doesn't appear in the transfer function from disturbance to output. All the other work will be done by $H_{ff}(z)$ since it has more freedom than case a).
3. In the above, to make the computation easier, we have chosen a $R(z)$ of a lowest possible degree. Unless explicitly specified, you can of course choose a higher order of $R(z)$. Similarly, we choose a $A_0(z)$ whose poles are all at the origin to facilitate the calculation. You can choose other forms of $A_0(z)$ as long as its poles are stable. Overall, the only

requirement is that the closed-loop transfer function should match the given reference model.

Q4

a)

The given input-output model is

$$y(k+1) = cy(k) + y^2(k-1) + u(k-1) \quad (1.1)$$

Forward shift by one, we get

$$y(k+2) = cy(k+1) + y^2(k) + u(k) \quad (1.2)$$

In order to get a predictor, just replacing $y(k+1)$ by (2.1),

$$y(k+2) = c(cy(k) + y^2(k-1) + u(k-1)) + y^2(k) + u(k) \quad (1.3)$$

In one-step-ahead controller, just make $y(k+2) = r(k+2)$ gives

$$u(k) = r(k+2) - c^2 y(k) - cy^2(k-1) - y^2(k) - cu(k-1) \quad (1.4)$$

b)

We need to assure that the plant output $y(k)$ and the controller output (the plant input) $u(k)$ are both bounded for perfect tracking.

Since the perfect tracking can be achieved, we will have $y(k) = r(k)$, the plant output is bounded as long as the reference $r(k)$ is bounded. In the case of perfect tracking, we have $y(k+2) = r(k+2)$ and $y(k+1) = r(k+1)$. Based on equation (2.2), the control input is

$$u(k) = r(k+2) - cr(k+1) - r^2(k) \quad (1.5)$$

For any reference signal $r(k)$, as long as it is bounded, the right hand side of (2.5), i.e., $r(k+2) - cr(k+1) - r^2(k)$, is also bounded. Therefore, as long as $r(k)$ is bounded, the input $u(k)$ is also bounded. So, there is no constraint on parameter c .

Note:

For the discussion on the perfect tracking, we should derive the condition based on the original input-output equation. This is the key step to get the correct result. The reason lies that, when converting the original input-output model, we are very likely to introduce more common poles/ zeros into the model. For example, let's just consider one simple input-output model as

$$y(k+1) = cy(k) + y(k-1) + u(k-1)$$

Then the transfer function is

$$\frac{Y(z)}{U(z)} = \frac{1}{z^2 - cz - 1}$$

which has a stable inverse.

However, in order to get the predictor form, we have

$$y(k+2) = c(cy(k) + y(k-1) + u(k-1)) + y(k) + u(k)$$

Then the transfer function now becomes

$$\frac{z+c}{z^3 - (c^2+1)z - c} = \frac{z+c}{(z^2 - cz - 1)(z+c)}$$

So, the transfer function of the predictor is the original TF plus extra common poles and zeros ($z+c$). But we cannot use this TF to analyze the stability of the original open-loop system because the extra pole ($z+c$) is imposed by us.