

Open Source Ventilator Pakistan

Modeling of C-7 Open Lung Ventilator Radial Arm De-pressor

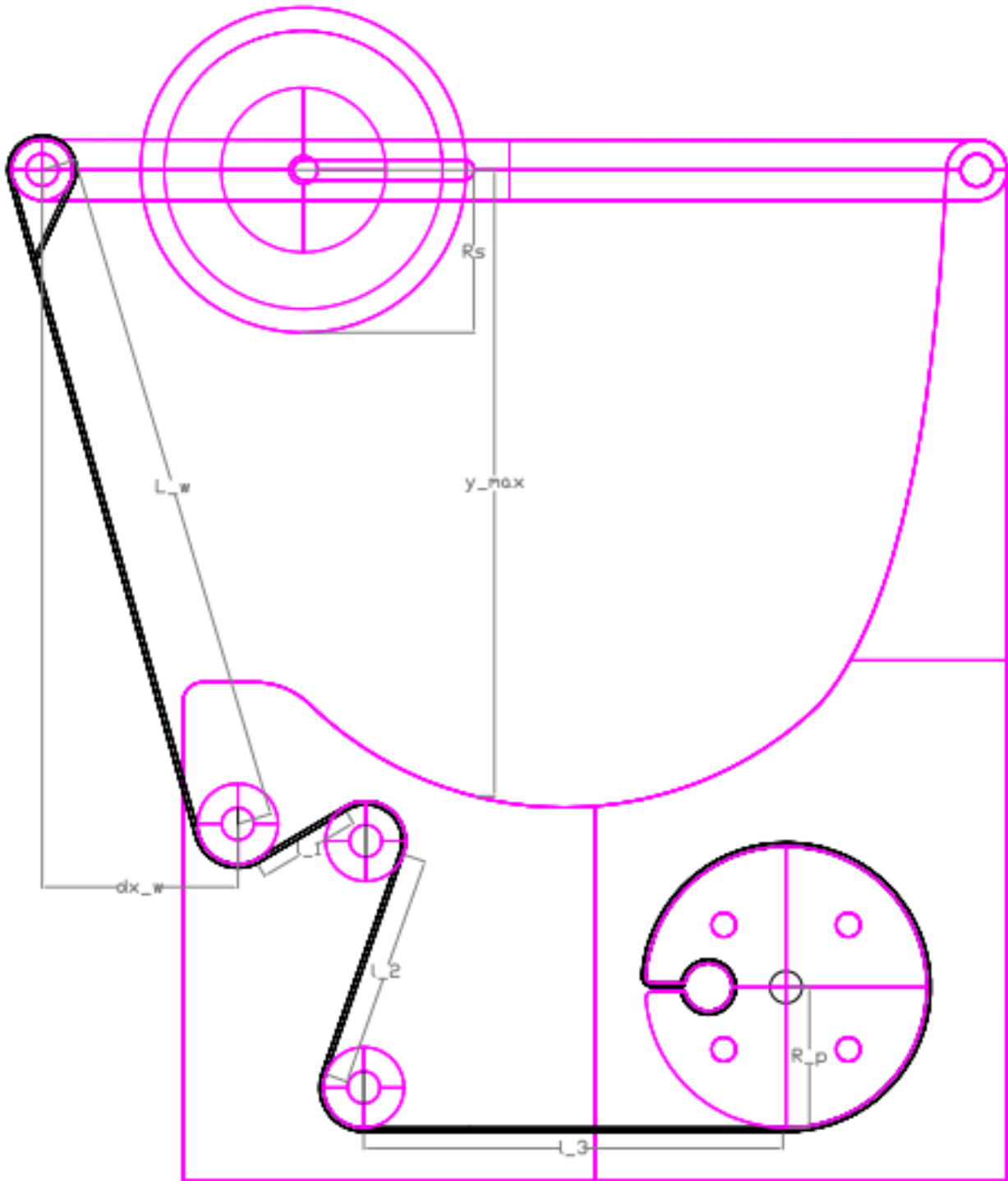
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1. Kinematics Modeling



Assuming linearity and establishing cartesian kinematics, the geometry above provides the equation:

$$\theta^\circ = \frac{180(y_{max} - R_s)}{\pi R_M * \cos(\alpha_{web})}$$

where,

$$\alpha_{web} = \sin^{-1} \left(\frac{dx_w}{L_w} \right) = 16.67^\circ$$

$$R_S = \text{Radius of Sticky Roller}$$

$$R_M = \text{Radius of Motor Roller}$$

Since the sticky roller is free to roller, we can neglect large adhesion; hence, approximating using Hertz's equation, and assuming incompressibility for the given temperature and pressure withing BVM, the following equations can be used to estimate the belt velocity and hence the required angular velocity from the time derivation of equation (1), assuming steady state:

$$v_b = \frac{Q_{BVM}}{A_{cont}}$$

$$A_{cont} = 2a * w_S$$

where,

$$Q = \text{Volume flow rate from BVM}$$

$$w_S = \text{width of sticky roller}$$

$$A_{cont} = \text{Contact area}$$

$$a = \frac{4FR_S}{\pi E^*}$$

where,

$$F = m_s g = \rho_s * V_s * g$$

$$E^* = \frac{1 - \nu_S}{E_S} + \frac{1 - \nu_{BVM}}{E_{BVM}}$$

thus, the rotational velocity at motor is given by:

$$v_b = R_M \frac{d\theta}{dt}$$

$$\omega = \frac{v_b}{R_M}$$

where,

$$v_b = \text{approx. belt velocity}$$

The technical drawing illustrates a mechanical assembly within a rectangular frame. A black line represents a lever or belt path. It starts at a top-left roller, goes down and right to a middle roller, then up and right to a bottom roller. A horizontal segment connects the bottom roller to a large circular pressure roller on the right. Dimensions are provided in millimeters: 40.00, 197.06, 138.24, 165.00, 18.70, 15.40, 10.00, 101.30, and 30.00. A detailed view of the pressure roller shows it has four small internal circles. To the right of the drawing, a list of calculations is provided.

- Vertical max travel by sticky roller ≈ 116
- since the relative length of lever and relative positions of rollers does not change, we assume the lever travels uniformly vertical, hence the angle of belt at webbing = 16.65 deg
- Thus the dL (length of belt being rolled) ≈ $116 / \cos(16.65) = 121$
- It is assumed that the belt remains taut the entire duration of travel. hence the length between the triple rollers and the pressure roller remains constant, thus, angular displacement by pressure roller = $dL / \text{radius of pressure roller} = \frac{121}{33} = 3.46$ rad
- Turn by stepper in degrees = $3.46 * 180 / \pi() = 198.24$ deg

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- It is assumed that the belt remains taut the entire duration of travel. hence the length between the triple rollers and the pressure roller remains constant, thus, angular displacement by pressure roller = $dL / \text{radius of pressure roller} = \frac{121}{35} = 3.46 \text{ rad}$
- Turn by stepper in degrees = $3.46 * 180 / \pi = 198.24^\circ$

- The model only provides an approximation and since the fluid dynamics consists of non-linearity, it is best to create a prototype and measure the appropriate flow, pressure and volume that is being output for various cases. This should then be tabulated for the purpose of calibration and further testing.