

Continuous-Time Dynamics

- Most general / generic for smooth system:

$$\dot{x} = f(x, \overset{\curvearrowleft}{u}) \quad \text{"input" } \in \mathbb{R}^m$$

"dynamics" \nearrow "state" $\in \mathbb{R}^n$ \nwarrow

- For a mechanical system $x = \begin{bmatrix} q \\ v \end{bmatrix}$
- "configuration" \downarrow
 (not always vector)
 "velocity" \swarrow

- Example (Pendulum):



$$(Ml^2\ddot{\theta} + mgl\sin(\theta)) = \sum \overset{\curvearrowleft}{u}$$

$$q = \theta, \quad v = \dot{\theta}$$

$$x = \begin{bmatrix} \theta \\ \dot{\theta} \end{bmatrix} \Rightarrow \dot{x} = \begin{bmatrix} \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l}\sin(\theta) + \frac{1}{ml^2}u \end{bmatrix}$$

$f(x, u)$

$q \in S^1$ (circle), $x \in S^1 \times \mathbb{R}$ (cylinder)

* Control-Affine Systems

$$\dot{x} = \underbrace{f_0(x)}_{\text{drift}} + \underbrace{B(x)u}_{\text{input Jacobian}}$$

- most systems can be put in this form
- Pendulum:

$$f_0(x) = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) \end{bmatrix}, \quad B_{ext} = \begin{bmatrix} 0 \\ \frac{1}{ml^2} \end{bmatrix}$$

* Manipulator Dynamics:

$$\underbrace{M(q)\ddot{v}}_{\text{"Mass matrix"} \atop \text{(Controls+Gravity)}} + \underbrace{(C(q,v))}_{\text{"Dynamic Bias"} \atop \text{(Controls+Gravity)}} = \underbrace{B(q)u}_{\text{"Input Jacobian"}}$$

$$\underbrace{\dot{q} = G(q)v}_{\text{"Velocity Kinematics"}}$$

$$\dot{x} = f(x,u) = \begin{bmatrix} G(q)v \\ -M(q)^{-1}(B(q)u - C) \end{bmatrix}$$

- Pendulum:

$$M(q) = ml^2, \quad (C(q,v)) = gl \sin(\theta), \quad B = I, \quad G = I$$

- All mechanical systems can be written in this form.
- This is just a way of re-writing the Euler-Lagrange equations for:

$$L = \frac{1}{2} v^T M(q) v - V(q)$$

(Bonus Points for showing this)

* Linear Systems

$$\dot{x} = Ax + Bu$$

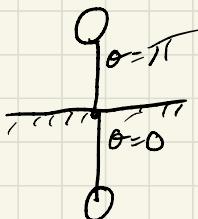
- Called "time invariant" if $A(t)=A$, $B(t)=B$
- Called "time varying" otherwise
- Super important in control
- We often approximate Nonlinear Systems with linear ones:

$$\dot{x} = f(x, u) \Rightarrow A = \frac{\partial f}{\partial x}, B = \frac{\partial f}{\partial u}$$

Equilibria:

- A point where the system will "remain at rest"
 $\Rightarrow \dot{x} = f(x, u) = 0$
- Algebraically, roots of the dynamics
- Pendulum:

$$\dot{x} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) \end{bmatrix} \rightarrow \begin{bmatrix} 0 \\ 0 \end{bmatrix} \Rightarrow \begin{array}{l} \dot{\theta} = 0 \\ \theta = 0, \pi \end{array}$$



* First Control Problem:

- Can I move the equilibria?

$$\theta = \frac{\pi}{2}$$

$$\dot{x} = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) + \frac{1}{m l^2} u \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

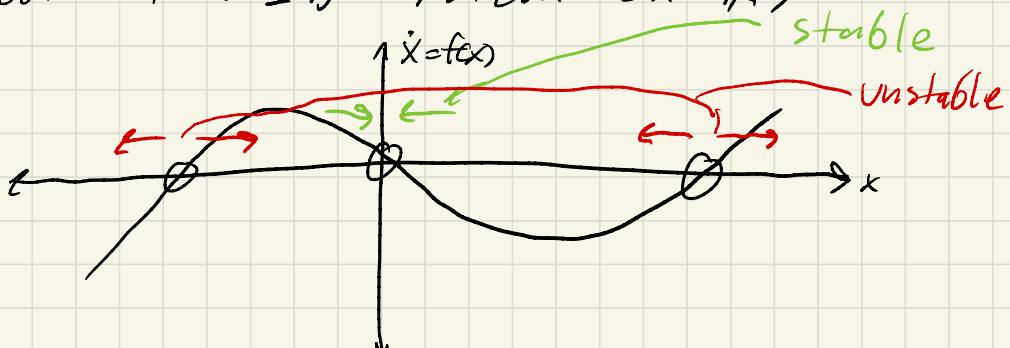
$$\frac{1}{m l^2} u = \frac{g}{l} \underbrace{\sin(\theta)}_{\approx 1} \Rightarrow u = m g l$$

- In general, we get a root-finding problem in u :

$$f(x^*, u) = 0$$

Stability of Equilibria:

- When will we stay "near" an equilibrium point under perturbations.
- Look at a 1D system ($x \in \mathbb{R}$)



$$\frac{\partial f}{\partial x} < 0 \Rightarrow \text{stable} \quad , \quad \frac{\partial f}{\partial x} > 0 \Rightarrow \text{unstable}$$

- In higher dimensions:

$\frac{\partial f}{\partial x}$ is a Jacobian matrix

- Take an eigendecomposition \Rightarrow decouple into n 1D systems

$$\operatorname{Re}[\operatorname{eig}\left(\frac{\partial f}{\partial x}\right)] < 0 \Rightarrow \text{stable}$$

otherwise \Rightarrow unstable

- Pendulum:

$$f(x) = \begin{bmatrix} \dot{\theta} \\ -\frac{g}{l} \sin(\theta) \end{bmatrix} \Rightarrow \frac{\partial f}{\partial x} = \begin{bmatrix} 0 & 1 \\ -\frac{g}{l} \cos(\theta) & 0 \end{bmatrix}$$

$$\frac{\partial f}{\partial x} \Big|_{\theta=\pi} = \begin{bmatrix} 0 & 1 \\ \frac{g}{l} & 0 \end{bmatrix} \Rightarrow \operatorname{eig}\left(\frac{\partial f}{\partial x}\right) = \pm \sqrt{\frac{g}{l}}$$

\Rightarrow unstable

$$\frac{\partial f}{\partial x} \Big|_{\theta=0} = \begin{bmatrix} 0 & 1 \\ -\frac{g}{l} & 0 \end{bmatrix} \Rightarrow \operatorname{eig}\left(\frac{\partial f}{\partial x}\right) = \pm i \sqrt{\frac{g}{l}}$$

- Pure imaginary case is called "marginally stable"
 \Rightarrow Undamped oscillations

- Add damping (e.g. $\ddot{\theta} = -K_d \dot{\theta}$) results in strictly negative real part.